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POSI-SEAL INTERNATIONAL, INC.

North Stonington, CT 06359

LOCA & SEISMIC ANALYSIS

<input checked="" type="checkbox"/>	APPROVED
<input type="checkbox"/>	APPROVED EXCEPT AS NOTED
<input type="checkbox"/>	NOT APPROVED-REVISE AND RESUBMIT
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By PC-RM Date NOV 22 1985

CUSTOMER: Texas Utilities Generating Company

UNIT: Comanche Peak Steam Electric Station  
Units 1 & 2

P.O. NO.: CPF-12296-S

SPEC. NO.: 2323-SS-20

REPORT NO.: 34977SL-001

CALCULATED BY: John B. Rodger

DATE: 1/25/85

APPROVED BY: John W. Cony #9093

DATE: Jan 25, 1985

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A PDR

REVISION A - Changed Pages 5,6,7,15,19,20,21,41,42  
44,47,48,53,56,C-2,C-4,C-6,C-8,C-10,  
D-2,E-1,E-3,E-4,E-6,E-7,E-9,E-10 and  
E-12.

Added Pages 42A and E-13 thru E-18  
Revision A was made to incorporate  
Comments 1 thru 7,8a,9,10,12a and  
13 of Texas Utilities letter TSG-  
8637, dated March 13, 1985.

REVISION B - Changed the Table of Contents, Pages  
2,5,6,7,8,15,41,42,43,44,54,56,C-2,  
C-4 thru C-10, Appendix E and  
Enclosure (5).

Added Pages 8A, 42B and 48A-E and  
C-11 thru C-16.

Revision B was made to incorporate  
comments made by Mr. R. Manuenyan  
of Texas Utilities and to upgrade  
Enclosure (5).



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<u>APPENDIX</u> A - Schematics of the Piping System	
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- ENCLOSURE
- (1) Valve Assembly Dwgs 14758-2 Rev. D  
14759-2 Rev. C
  - (2) Posi-Seal Technical Bulletin No. 2,  
dated June 1982
  - (3) Derivation of Hydrodynamic Torque Curves
  - (4) Posi-Seal Technical Bulletin No. 1A,  
dated June 1982
  - (5) Comparison of Calculated Aerodynamic Torques  
with Test Data

- REFERENCES
- (a) DuPont Tefzel Catalog, A-95151, dated 1973
  - (b) "Flow of Fluids through Valves, Fittings  
and Pipe", Technical Paper No. 410, Crane
  - (c) "Formulas for Stress and Strain" by R. J. Roark,  
McGraw Hill, 4th Edition
  - (d) "Mechanical Vibrations" by Church, John Wiley  
& Sons
  - (e) "Advanced Strength of Materials" by Seeley &  
Smith, John Wiley & Sons
  - (f) "Eccentrically Loaded Joints", Machine Design,  
August 1967
  - (g) Texas Utilities Generating Company, TSG-7086,  
Mr. Fred Madden to Mr. Stites, dated Nov. 7, 1984
  - (h) TUSI Nuclear Power Plant, Instructure Response  
Spectra for Containment Building, FRB-5R,  
August, 1976
  - (i) TUGCO Dwg BRP-VA-1-RE-004, Rev. CP-1

SUMMARY

Due to the design of the Posi-Seal butterfly valve with the disc being asymmetrical, flow in the preferred direction tends to close the valve. In the nonpreferred direction the disc tends to stay in the open position until it reaches an angle of approximately 75 degrees open, then tends to close. See Figure 1 on Page 2 for an illustration of what direction is preferred.

For the subject valves, those inside containment are in the nonpreferred direction and those outside containment are in the preferred direction.

If a LOCA does occur the scenario given below describes what effect the large flows resulting from the LOCA will have on the subject valves.

A. If all the subject valves are fully open

Under normal flow conditions the subject valves would pass 2000 SCFM with an approximate pressure drop of .008 in. water.

If a LOCA were to occur the initial flow due to a LOCA will create an aerodynamic torque of such a magnitude that the valve inside containment will remain open.

For the valves outside containment the aerodynamic torque will override the actuators and cause the valves to partially close and damage the valve disc pins. Upon receiving the signal to close, the valves outside containment will close, however, due to the damage to the disc pins a tight shut off will not be obtained.

- B. If the amount of valve opening for all the subject valves is restricted to 65 degrees.

Under normal flow conditions the subject valves would pass 2000 SCFM with an approximate  $\Delta P$  of .03 in. water. The requirement of the specification requires a  $\Delta P$  of .03 in. of water.

Once the LOCA occurs all the valves would override their actuators and tend to close a few degrees. Once the signal to close was received, the valves would fully close providing a proper shut off.

Preferred Direction

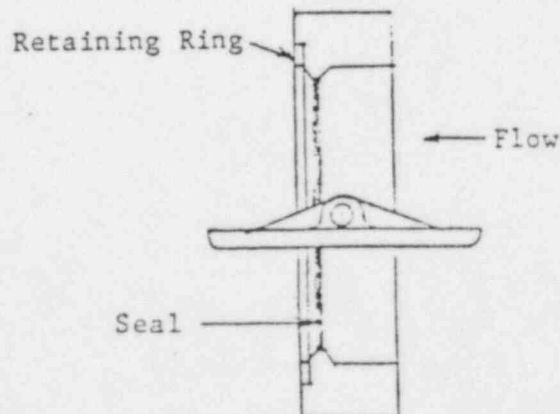


Figure 1

## INTRODUCTION

The objective of this analysis is to show that the subject containment isolation valves can withstand a Loss of Coolant Accident (LOCA) as well as a seismic event and still maintain operability.

The escape of containment atmosphere during a LOCA will result in aerodynamic torques acting on the valve assembly if it is in the open position. These torques are a result of the disc acting like an airfoil wanting to rotate about the axis of the stem. This analysis will determine the magnitude of the aerodynamic torque and its effect on the operation of the valve assembly.

The seismic aspect of the analysis will consist of first determining the natural frequencies of the valve assembly. If any of these frequencies is less than 33 Hz then the maximum accelerations of the appropriate RRS curves will be used for the stress analysis. If not, then the ZPA will be used.

The stress analysis will be performed based on the accelerations statically acting at the CG of the actuator. Critical sections of the valve assembly such as bolting, neck, stem and disc pins will be analyzed. Seismic stresses will be combined with stresses due to the LOCA torque and the operating pressure.

All equations are either straight forward or from Reference (c) unless otherwise noted.

The valves to be qualified by this analysis are as follows:

<u>PSI Item No.</u>	<u>Description</u>	<u>Tag No.</u>
14758-2	18"-150 Cl. with Matryx 26072-SR80	CP1-VADPBC-09 CP1-VADPBC-10
14759-2	18"-150 Cl. with Matryx 26072-SR80	CP2-VADPBC-09 CP2-VADPBC-10

The assembly drawings of these valves are shown in Enclosure (1).



RESULTSMaximum Torques Resulting from a LOCA  
and Closing Times

Tag No.	<u>VADPBC-09</u>	<u>VADPBC-10</u>
Valve	18"	18"
Class	150	150
Matryx Actuator Model	26072-SR80	26072-SR80
Location	Inside Containment	Outside Containment
Torques (in-lbs)		
Max Torque Resulting from a LOCA		
Valve opened to 90 degrees	15906 (1)	20609 (1)
Valve opened to 70 degrees	13350 (2)	12198 (2)
Valve opened to 65 degrees	11401 (2)	7677 (2)
Actuator Max Allowable Torque	28286	28286
Actuator Spring Beginning Torque	10400	10400
Actuator Spring Ending Torque	6420	6420
Closing Times (Sec)		
No Flow	1.65 (3)	1.65 (3)
Valve opened to 90 degrees	Will not close	.65 (1)
Valve opened to 70 degrees	.57 (2)	.65 (1)
Valve opened to 65 degrees	.57	.65
Required Closing Time	5	5

- Notes
- (1) Piping upstream of valve VADPBC-09 breaks off, Case 3
  - (2) Piping upstream of elbow breaks off, Case 2
  - (3) See Page 54



LOCA AND SEISMIC STRESSES (PSI)

Tag No. & Item No.	Valve Opening	Actuator Bolt		Bracket Bolt		Bracket		Valve Neck		Stem		Disc Pin	
		Calc	Allow	Calc	Allow	Calc	Allow	Calc	Allow	Calc	Allow	Calc	Allow
CP1 & 2 VADPBC 10	90°	42328	94500	21523	94500	23458	41400	2923	26250	22023	52500	35790	21000
CP1 & 2 VADPBD 10	70°	36970	94500	20104	94500	19079	41400	2651	26250	14122	52500	21183	21000
CP1 & 2 VADPBC 09	70°	39640	94500	21382	94500	25270	41400	2918	26250	15233	52500	23184	21000
CP1 & 2 10	65°	34400	94500	12478	94500	16844	41400	2577	26250	9954	52500	13332	21000
CP1 & 2 09	65°	38431	94500	21064	94500	24324	41400	2862	26250	13417	52500	19799	21000

Tag No. & Item No.	Valve Opening		
		Calc	Allow
CP1 & 2 VADPBC 10	90°	4856	24750
CP1 & 2 VADPBC 10	70°	5094	24750
CP1 & 2 VADPBC 09	70°	4948	24750
CP1 & 2 VADPBC 10	65°	6141	24750
CP1 & 2 VADPBC 09	65°	5792	24750

RECOMMENDATIONS AND CONCLUSIONS

Based on the results of this analysis Fosi-Seal recommends that the amount of valve opening for all the subject valves be restricted to 65 degrees. This can be accomplished by bolting a stop to the internals of the Matryx piston cartridge which would limit the stroke of the actuator. See Figure 2 on Page 9.

The conclusion that the amount of valve opening be restricted to 65 degrees is based on:

1. The LOCA flow tending to close the valves irregardless of whether they are in the preferred direction or nonpreferred direction.
2. As a result of the flow assisting the closing, closing times will be less than those for no flow and will be less than the required closing time.
3. The yield strength of disc pin material per the ASME specification is 3.28 times greater than the allowable strength used based on the ASME Code.
4. The maximum torque of the actuator is 2.48 times the maximum torque calculated based on a LOCA flow with the valve at 65 degrees.

5. With the valves at 65 degrees the valves can still pass the required flow under normal conditions with the required pressure drop.

The ability of the Tefzel seals to provide a proper shut off is based on the following:

1. Posi-Seal's experience with Tefzel seals in applications where the velocities are comparable to those the subject valves will experience during a LOCA.
2. Radiation testing performed by DuPont as reported in Reference (a). For Posi-Seal's application we have rated the Tefzel seals to  $3 \times 10^8$  rads.
3. The ability of Tefzel to withstand 340 degrees F. temperatures for a short period of time as also reported in Reference (a).

ALLOWABLE STRESSES

Per Paragraph 6.2 of Specification 2323-SS-20, which addresses seismics, the stress limit should be 90 percent of the yield strength unless otherwise set forth in the appropriate design standards and codes specified in the equipment specifications.

Posi-Seal used this criteria for the bolting and bracket stresses.

As for the parts subject to pressure, Section 3.7, 16.5 of the subject specification states that the stress limits for the upset, emergency and faulted conditions are the primary pressure rating corresponding to the pressure temperature classification as defined by ANSI B16.5. These pressures, however, cannot be translated into allowable stresses for the purpose of this report. It should also be noted that Paragraph NC-3500 of Section III of the ASME Boiler and Pressure Vessel Code, 1974 Edition, including the Winter of 1975 Addenda, only states that the valve shall be designed in accordance with ANSI B16.5.

As a consequence of the above, Posi-Seal decided to use 0.6 S for the allowable disc pin stress and 1.5 S for the remaining allowable stresses. S is the allowable stress for Class 2 and 3 components given in Appendix I, Tables 1-7.1, 2 and 3 of the Code. This decision was based on using the service limit Level A given in the present Code. The 0.6 factor was used for the disc pins since they are in pure shear. The 1.5 factor for the body, disc and stem was used since the stress of these components includes bending.

MATRYX ACTUATOR  
Mechanical Stop  
for  
Restricting Valve Opening to 65°

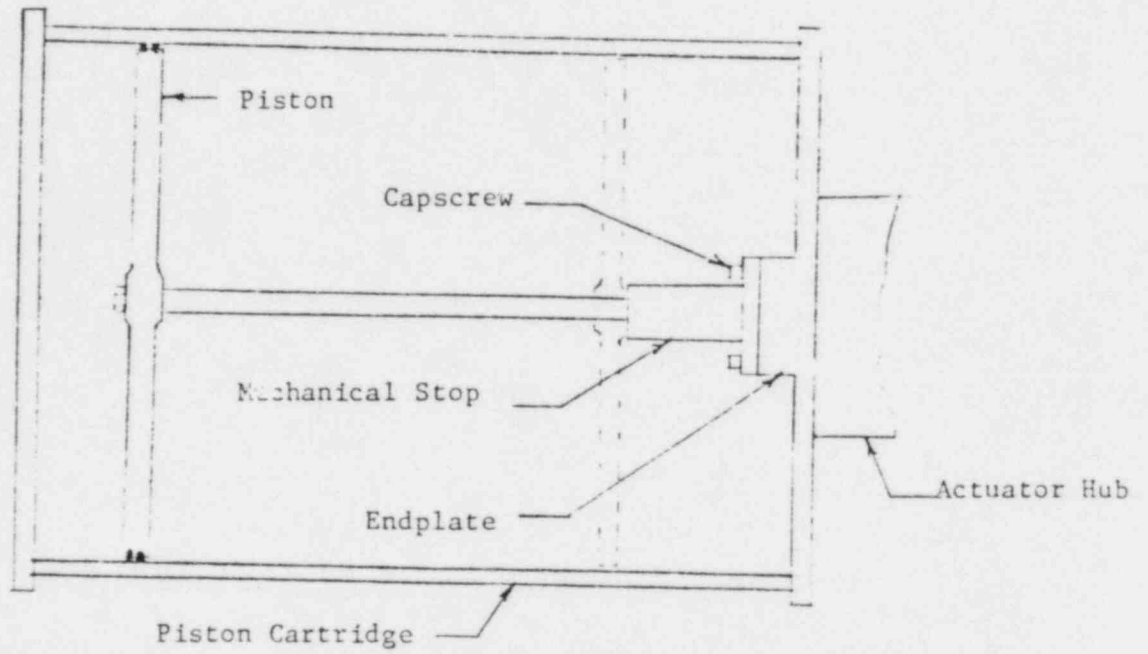


Figure 2

## LOCA ANALYSIS

The purpose of this analysis is to determine what effects the aerodynamic torque resulting from a LOCA will have on a valve assembly. Since aerodynamic torque is dependent upon the flow conditions and the valve angle, computer programs are developed which:

1. Models the piping system
2. Determines the flow at various valve angles
3. Simulates the actuator as it strokes the valve from fully open to fully close.

## MODELING THE PIPING SYSTEM

For this LOCA analysis three cases are investigated to determine which piping system results in the largest aerodynamic torques resulting from a LOCA.

The piping systems are:

- Case 1 The piping breaks off downstream of Valve VADPBC-10
- Case 2 The piping breaks off upstream of 90 degree bend inside containment and downstream of Valve VADPBC-10
- Case 3 The piping breaks off upstream of Valve VADPBC-9 and downstream of Valve VADPBC-10

Shown in Appendix A are schematics of the piping systems with each system broken down into the individual components with its corresponding resistance factor. These factors are inputted into the computer program either as a K value, as a length of pipe, a change in pipe diameter, or as a valve  $C_v$ . The K values are obtained from Reference (b), the  $C_v$  values from Posi-Seal Technical Bulletin No. 2, Enclosure (2).

Thus, with the piping system modeled, and with the upstream and downstream conditions known, the flow conditions can be determined.

### Determination of Flow Conditions

Derivation of equations

Bernoulli's Equation

$$Z_1 + \frac{144 P_1}{\rho_1} + \frac{V_1^2}{2g} = Z_2 + \frac{144 P_2}{\rho_2} + \frac{V_2^2}{2g} + h_L$$

Since the flow investigated will either be steam or air the height terms ( $Z_1, Z_2$ ) can be ignored.

$$\frac{144 P_1}{\rho_1} + \frac{V_1^2}{2g} = \frac{144 P_2}{\rho_2} + \frac{V_2^2}{2g}$$

where P = Pressure PSIG

$\rho$  = Density lb/ft<sup>3</sup>

V = Velocity ft/sec

g = Gravitational constant = 32.2 ft/sec<sup>2</sup>

$h_L$  = Head Loss

Since the piping systems are relatively short the flow is assumed to be adiabatic.

$$\rho_2 = \rho_1 \left( \frac{P_2'}{P_1'} \right)^{1/K_1} \quad \text{per Ref. (b)}$$

$$T_2 = T_1 \left( \frac{P_2'}{P_1'} \right)^{\frac{K_1-1}{K_1}}$$



where  $K_1$  = Ratio of specific heats

$P'$  = Pressure PSIA

$T$  = Absolute temperature  $^{\circ}\text{R}$

Flow equations

In pipe

$$Q = \frac{694.3 P' V D^2}{T}$$

where  $Q$  = Flow SCFH

$D$  = Diameter in<sup>2</sup>

In valve

$$Q = 1360 C_V P_1' Y \sqrt{\frac{X}{GTZ}}$$

per Encl. (2)

where  $C_V$  = Valve coefficient

$X = \Delta P/P'$

$\Delta P$  = Pressure drop across valve PSI

$$Y = 1 - \frac{X}{3F_K X_T}$$

$F_K$  = Ratio of specific heat factors

$X_T$  = Rated pressure drop ratio factor

$G$  = Specific gravity

$Z$  = Compressibility factor

For choke flow in valve

$$Q = 907.1 C_V P_1' \sqrt{\frac{F_K X_T}{GTZ}}$$

per Encl. (2)

$$\Delta P \text{ choked} = F_K X_T P_1'$$



## Sonic Velocity Equation

$$V_S = \sqrt{\frac{4637 K_1 P'}{\rho}} \quad \text{per Ref. (b)}$$

Determination of the flow conditions will be performed as follows:

1. Calculate density at the end condition

$$\rho_{N+1} = \left( \frac{P'_{N+1}}{P'_N} \right) \rho_1$$

2. Calculate initial velocity based on beginning and end conditions.

$$V(1) = \sqrt{\frac{\left( \frac{P'_{N+1}}{\rho_{N+1}} - \frac{P'_1}{\rho_1} \right) 2885}{\left( 1 - \left( \frac{D_1}{D_{N+1}} \right)^4 - K \right)}}$$

where  $K = K(1) + K(2) + \dots + K(N+1)$

3. Using the initial velocity  $V(1)$ , calculate  $\Delta P$  for all the stations as shown below

For  $I = 1$  to  $N$

$$\rho = \rho(I)$$

$$\rho(I+1) = \rho$$

$$V(I+1) = D(I)^2 V(I) \rho(I) / D(I+1)^2 \rho(I+1)$$

$$P(I) = P(I) - 14.7$$

$$P(I+1) = \rho(I+1) \left( \frac{P(I)}{\rho(I)} + \frac{V(I)^2 (1 - K(I))}{9274} \right) - \frac{V(I+1)^2}{9274}$$

$$P(I) = P(I) + 14.7 \quad P(I+1) = P(I+1) + 14.7$$

$$\rho(I+1) = \rho(I) \left( \frac{P(I+1)}{P(I)} \right)^{1/K_1}$$

$$\text{If } |\rho(I+1) - \rho| > .0005 \text{ then } \rho = \rho - .0005$$

and recalculate  $P(I+1)$

Note: This is done since  $\rho(I+1)$  is a function of  $P(I+1)$  and vice versa.

$$T(I+1) = T(I) \left( \frac{P(I+1)}{P(I)} \right)^{(K_1-1)/K_1}$$

For determining the  $\Delta P$  across the valves, the equation for  $Q$  given on the preceding page is used. Solving for  $\Delta P$  from this equation results in a cubic equation with the smallest root being equal to the actual drop across the valve.

4. With the final pressure  $P(N+1)$  calculated, this pressure is compared to the final pressure given. For this particular study the final pressure is atmospheric.

If the calculate pressure is less than the given final pressure then the initial velocity is decreased and Step 3 is repeated. The initial velocity is increased if the calculated final pressure is greater than the given final pressure.

5. Steps 3 and 4 are repeated until the calculated final pressure approximately equals the given final pressure.
6. If sonic velocity is encountered at any of the stations the initial velocity is decreased until Step 5 is achieved or until the calculated sonic velocity approximately equals the actual sonic velocity.

If the latter is the case then the given final pressure is assumed and the pressures at the stations between the outlet and the station at which sonic flow occurs are determined by using the equation given in Step 3 in reverse order and using the flow,  $Q$ , based on the sonic velocity.

7. If choke flow is encountered in any of the valves then the same approach is taken as given in Step 6.
8. To determine the flow conditions for the various valve angles, the  $C_v$  of the valve closing is determined for the angle of interest and Steps 1 thru 7 are repeated.

The above is formulated into the computer program "FLOW-GAS."

SIMULATION OF THE ACTUATOR STROKING THE VALVE CLOSE

In order to simulate the closing of the valve, an equation which describes the torques acting on the valve stem has to be defined. This equation is given below:

$$T_{TTO} = T_{flow} + T_{air} + T_{spring} + T_{packing \text{ and } seal} + T_{bearing}$$

Where  $T_{TTO}$  = The net torque tending to open the valve (equals zero when the valve starts to close).

$T_{flow}$  = The torque due to aerodynamic flow caused by the LOCA can act in either direction depending on flow direction and valve angle.

$T_{air}$  = The torque exert by the actuator as a result of the air acting on the actuator piston tending to open the valve acts in the positive direction.

$T_{spring}$  = The torque exerted by the actuator spring tending to close the valve acts in the negative direction.

$T_{packing \text{ \& } seal}$  = Torque of the packing and the seal resisting the closing motion of the valve. The seal torque does not take affect until the disc begins to seal which occurs at approximately 3 degrees from fully closed. Resists the motion of the actuator. Positive when valve closes, negative when the valve opens.

$T_{bearing}$  = Torque due to the  $\Delta P$  acting across the valve which forces the stem/disc assembly into the bearings. Resists the motion of the actuator. Positive when valve closes, negative when the valve opens.

### Derivation of Torque Equations

#### Aerodynamic Torque ( $T_{\text{flow}}$ )

Since Posi-Seal has only determined hydrodynamic torques for water based on testing, see Encl. (3), a way to determining aerodynamic torques for air and steam from those for water has to be derived.

The resultant drag and lift forces acting on the disc are as follows:

$$F_D = C_D \rho \frac{V^2 A}{2} \quad \text{Resultant Drag Force}$$

$$F_L = C_L \rho \frac{V^2 A}{2} \quad \text{Resultant Lift Force}$$

The resultant torque is the resultant force times the length from  $\bar{C}$  of stem to the location of the resultant force.

$$\therefore T_D = C_D L_P \rho \frac{V^2 A}{2} \quad \text{Resultant Drag Torque}$$

$$T_L = C_L L_L \rho \frac{V^2 A}{2} \quad \text{Resultant Lift Torque}$$

$$T_{D,L} = C_{D,L} L_{D,L} \rho \frac{V^2 A}{2}$$

Where  $V$  = Velocity

$A$  = Surface Area

$\rho$  = Density of Fluid

$C_D, C_L$  = Drag and Lift Coefficients (Dependent upon shape and orientation of disc)

$L_D, L_L$  = Length  $\bar{C}$  stem to resultant lift and drag forces

$D, L$  = Combined Subscript

NOTE:  $C_{D,L}$  and  $L_{D,L}$  are the same for the same size and class valve, assuming the same angular position, regardless of fluid, flow, media or temperature.

$$\therefore \frac{T_{\text{fluid}}}{T_{\text{water}}} \approx \frac{\rho_{\text{fluid}} V_{\text{fluid}}^2}{\rho_{\text{water}} V_{\text{water}}^2}$$

$$T_{4F} = \rho_F \frac{T_{4W} V_F^2}{62.4 V_W^2}$$

Where W = Water

F = Fluid

$\rho_{\text{water}} = 62.4 \text{ lbs/ft}^3$

$V_F$  = Calculated in the determination of the flow conditions

$$V_W = .00223 \frac{Q}{A} = .00223 \frac{C_V}{A} \sqrt{\Delta P}$$

$T_{4W}$  = Disc Hydrodynamic Torque per PSI  $\Delta P$  (function of valve angle)

$T_{4F}$  = Disc Aerodynamic Torque per PSI  $\Delta P$

The total aerodynamic torque equals

$$T_F = \rho_F \frac{T_{4W} V_F^2}{62.4 V_W^2} \Delta P$$

$$\text{Since } C_F = \frac{144 P_1}{R T_1} \quad R_{\text{Air}} = 53.34$$

$$V_F = \frac{Q T_1}{127300 P_1 A} \quad R_{\text{Steam}} = 85.76$$

$$V_w = \frac{.00223 C_v \sqrt{\Delta P}}{A}$$

$$\text{Then } T_F = \frac{144 P_1}{R T_1} \left( \frac{T_{LW}}{62.4} \right) \left( \frac{\frac{Q T_1}{127300 P_1 A}}{\frac{.00223 C_v \sqrt{\Delta P}}{A}} \right)^2 \Delta P$$

$$T_{\text{Air}} = .04326 T_{LW} \frac{T_1}{P_1} \left( \frac{Q}{203.0 C_v} \right)^2$$

$$T_{\text{Steam}} = .0269 T_{LW} \frac{T_1}{P_1} \left( \frac{Q}{203.0 C_v} \right)^2$$

Values for  $C_v$  and  $T_{LW}$  can be found in Enclosures (2) and (4) respectively for various valve angles.

For critical flow the equations can be simplified to:

$$T_{\text{Air}} = .441 T_{LW} \frac{F_K X_T P_1}{G Z}$$

$$T_{\text{Steam}} = .274 T_{LW} \frac{F_K X_T P_1}{G Z}$$

The above aerodynamic torque equations have also been incorporated into "FLOW-GAS" computer program such that the torque resulting from a LOCA can be determined for every 10° of valve closure.

Since Posi-Seal has not performed any aerodynamic flow testing, the validity of these equations is determined by comparison to test data given in an ISA transaction. See Enclosure (5)

In performing the LOCA analysis it is assumed that the valves close individually. This assumption is made for two reasons. The first is for ease of analysis. The second reason being, this is considered to be more conservative since if both valves close simultaneously, the resistance in the system will be greater; consequently, the flow will be less and the aerodynamic torque will be less.

Pneumatic Torque ( $T_{air}$ )

$$T_{air} = \frac{A R P_1}{C_2}$$

Where A = Area of piston

$D_c$  = Cubic Displacement

$$A = \frac{D_c}{2R}$$

P = Working Pressure of Actuator

$$A = \frac{1728}{2R} V \left( \frac{14.7}{P+14.7} \right)$$

V = Specific Volume - SCF

R = Radius of Scotch Yoke (See Figure 4)

$P_1$  = Pressure of the air in the piston cylinder

$$= \frac{P_1 (V - \Delta V)}{V}$$

$P_1$  = Previous pressure (See Note Below)

$\Delta V$  = Change in Volume

$$= \frac{dt * Q}{3600}$$

dt = Change in Time

Q = Flow thru solenoid valve or quick exhaust

$$= \frac{963 C_{vs} F_{LS} P_1 \sqrt{1 - .25 (F_L)^2}}{\sqrt{GT}}$$

$C_{vs} = C_v$  of solenoid valve or quick exhaust

$F_{LS}$  = Rated liquid pressure recovery factor of a solenoid valve or quick exhaust = .9

G = Specific Gravity of Air = 1

T = Temperature ° Rankine

$$Q = 774.2 C_v P_1 / \sqrt{T}$$

$C_2$  = Equation describing the advantage of the Scotch yoke as a function of angle.

NOTE: In order to take the effect of the building atmospheric pressure into consideration the initial pressure for  $P_1$  is equal to the working pressure of the actuator minus the building atmospheric pressure.

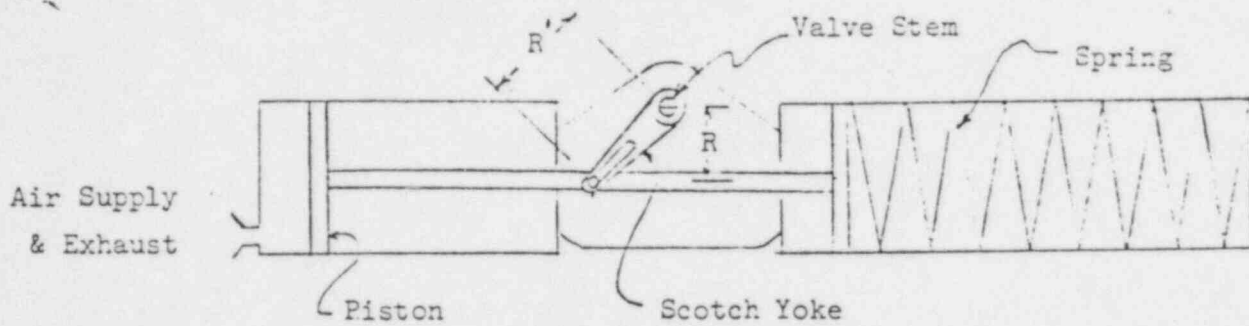


Figure 4

Forces acting on Scotch Yoke Pin

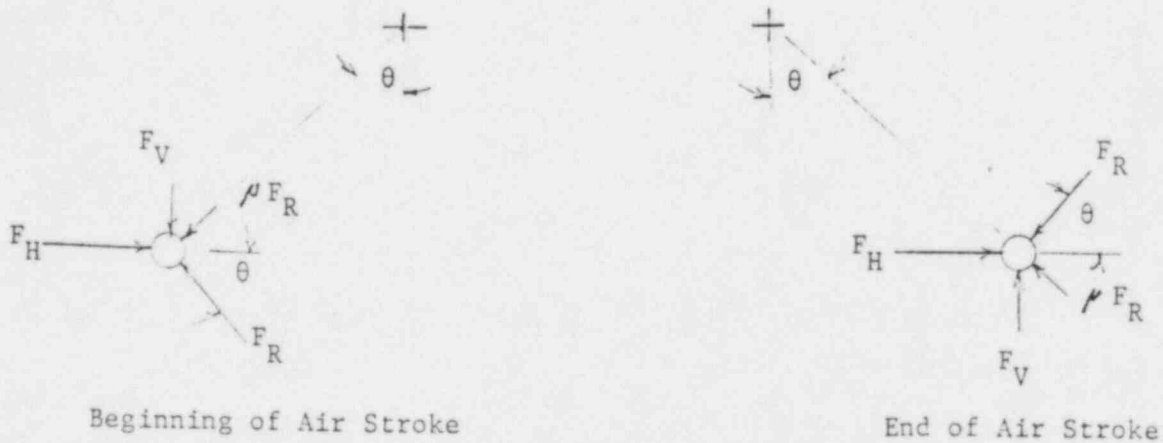


Figure 5

Summing forces in the horizontal direction

$$F_H - \cos |\theta| F_R - \mu \sin |\theta| F_R = 0 \quad \text{For } \theta = -90^\circ \text{ to } +90^\circ$$

$$F_R = F_H / (\cos \theta + \mu \sin |\theta|) \quad \cos \theta = \cos |\theta|$$

 $F_R$  = Resultant Force $F_H$  = Horizontal Force

$$T_R = F_R R' = \frac{F_R R}{\cos \theta} = \text{Resultant Torque}$$

$$\therefore T_R = \frac{T}{\cos \theta (\cos \theta + \mu \sin |\theta|)}$$

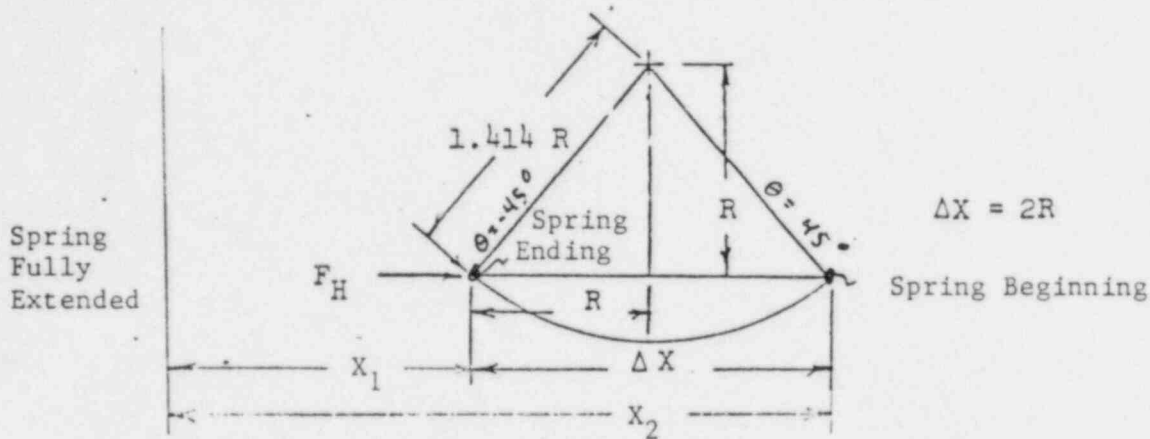
$$C2 = \cos \theta (\cos \theta + \mu \sin |\theta|)$$



Spring Torque ( $T_{\text{spring}}$ )

$$T_{\text{spring}} = \frac{K X_2 R}{C}$$

Figure 6



$K$  = Springrate

$$= \frac{\Delta F}{\Delta X} = \frac{T_{\text{spring beginning}} - T_{\text{spring ending}}}{1.414 (1.414)R \Delta X}$$

$$K = \frac{T_{\text{spring beginning}} - T_{\text{spring ending}}}{4R^2}$$

$$X_2 = X_1 + R (1 + \tan \theta)$$

$$X_1 = \frac{T_{\text{spring ending}} C_2 @ \theta = -45}{KR}$$

$$X_1 = \frac{.571 T_{\text{spring ending}}}{KR}$$

$$X_2 = \frac{.571 T_{\text{spring ending}}}{KR} + R (1 + \tan \theta)$$

Bearing Torque ( $T_{\text{bearing}}$ )

$$T_{\text{bearing}} = \frac{\pi \mu P D^2 d}{8}$$

where  $\mu$  = Coefficient of friction  
 = .059 for bronze bearings  
 $D$  = Disc gage diameter  
 $d$  = Stem diameter

The torque equations are formulated into the computer program "FLOW-CL". This program calculates the various torques acting on the valve and the amount of valve closure as time is incremented until which time the valve is fully closed. In order to determine the aerodynamic torque for angles other than the ten degree increments calculated by "FLOW-GAS" the values for the densities, pressure drops and velocities are taken from "FLOW-GAS", interpolated to correspond to the angle of interest and then the aerodynamic torques are calculated based on those values. The reason for calculating aerodynamic torque in this manner is that density, pressure drop and velocity are more linear between the ten degree increments than is the aerodynamic torque. However, for the case where there is a bend upstream of the valve the torque is inputted directly. The program used for this is "FLOW-CL1".

SEISMIC ANALYSISA. NATURAL FREQUENCIES:LATERAL - Disc/Stem (By Rayleigh's Method)

The natural frequency is calculated for the worse case, that being the valve open where the disc is not supported by the seat.

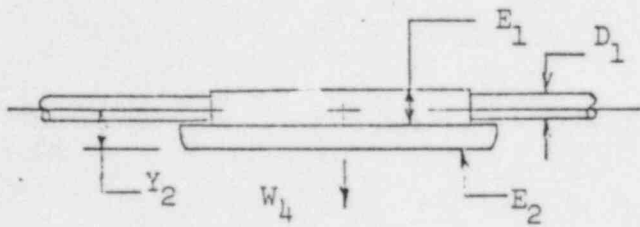
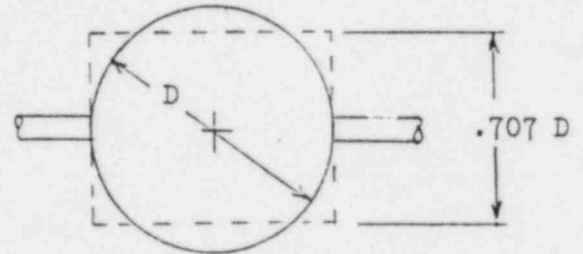


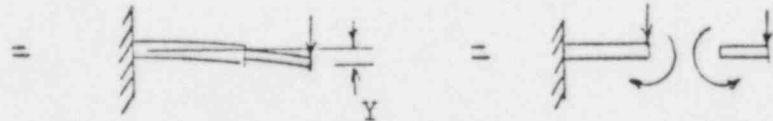
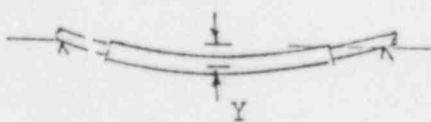
Figure 7



NATURAL FREQUENCY Per Ref. (d)

Figure 8

$$N_C = 3.125 \sqrt{\frac{1}{Y}} \text{ Hz}$$



$$Y_D = \frac{W_4 \left[ \frac{D}{2} \right]^3}{3 EI_D} + \frac{M \left[ \frac{D}{2} \right]^2}{2 EI_D} = \frac{W_4 D^3}{48 EI_D} + \frac{W_4 \left[ \frac{L_1}{3} + L_2 \right] D^2}{16 EI_D} \quad \text{Deflection of Disc}$$

$$\theta_D = \frac{W_4 \left[ \frac{D}{2} \right]^2}{2 EI_D} + \frac{M \frac{D}{2}}{EI_D} = \frac{W_4 D^2}{16 EI_D} + \frac{W_4 \left[ \frac{L_1}{3} + L_2 \right] D}{4 EI_D} \quad \text{Slope at end of Disc}$$

$$Y_S = \frac{W_4 \left[ \frac{L_1}{3} + L_2 \right]^3}{3 EI_S} = \frac{W_4 \left[ \frac{L_1}{3} + L_2 \right]^3}{6 EI_S} \quad \text{Deflection of Stem}$$

$$Y = \frac{W_4}{E} \left[ \frac{D^3}{48 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right] D^2}{16 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right] D^2}{16 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right]^2 D}{4 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right]^3}{6 I_S} \right]$$

$$Y = \frac{W_4}{E} \left[ \frac{D^3}{48 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right] D^2}{8 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right]^2 D}{4 I_D} + \frac{\left[ \frac{L_1}{3} + L_2 \right]^3}{6 I_S} \right]$$

$$I_S = \frac{\pi D_1^4}{64} \quad \text{Moment of Inertia of Stem (in}^4\text{)}$$

$$I_D = \frac{.707 DE^3}{12} + .707 DE \left[ \frac{E}{2} - \bar{Y} \right]^2 + \frac{\pi E^4}{64} + \frac{\pi E^2}{4} \left[ Y_2 - \bar{Y} \right]^2 \quad \text{Moment of Inertia of Disc (in}^4\text{)}$$

$$\bar{Y} = \frac{.3535 DE^2 + .7854 E_1^2 Y_2}{.707 DE_2 + .7854 E_1} \quad \text{Distance to Neutral Axis (in)}$$

WHERE:  $W_4$  = DISC WEIGHT (LBS.)

$D_1$  = STEM DIA. (IN.)

$D$  = DISC GAGE DIA. (IN.)

$L_1$  = EFFECTIVE BRG. LENGTH (IN.) See Page 46 for equation

$L_2$  = THRUST WASHER THICKNESS (IN.)

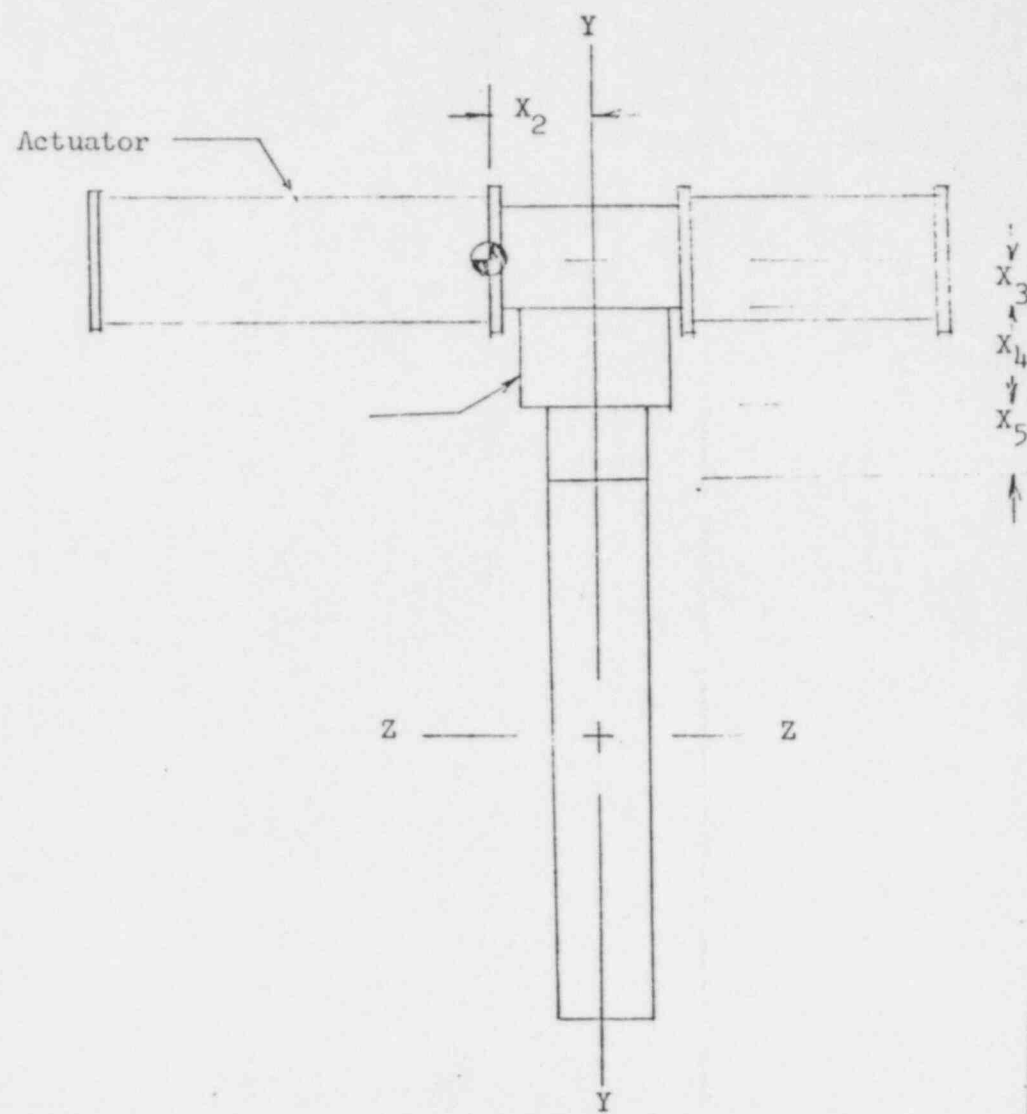
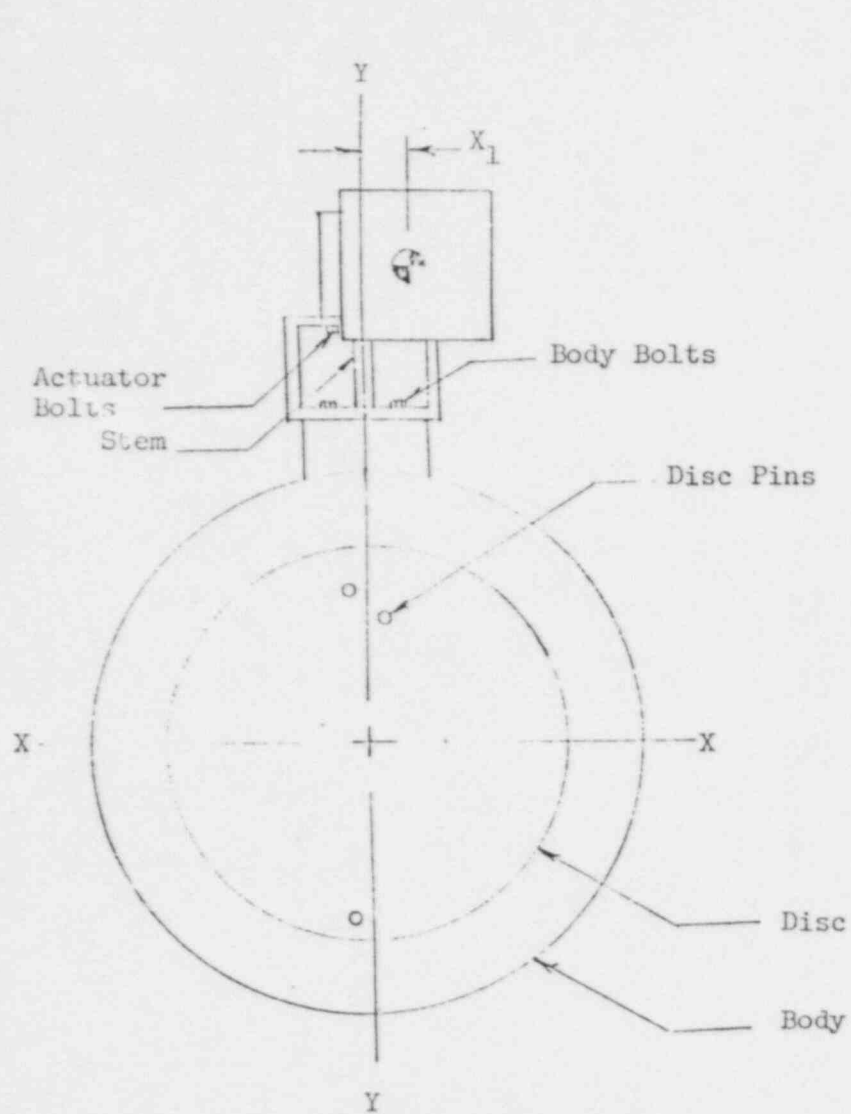
$E_1$  = WIDTH SMALL DIA. BACK OF DISC (IN.)

$E_2$  = WIDTH LARGE DIA. OF DISC (IN.)

$Y_2$  = DIST.  $\phi$  STEM TO FRONT FACE OF DISC (IN.)

NATURAL FREQUENCY ANALYSIS - VALVE/ACTUATOR ASSEMBLY

The natural frequency in the longitudinal direction is that of the actuator rocking about the top of the valve neck in the direction parallel to the pipeline. The transverse natural frequency is that of the actuator rocking about the top of the valve neck in the direction perpendicular to the pipeline.



PICTORAL MODEL OF VALVE ASSEMBLY

Figure 9

# VALVE NECK AND FLANGE DIMENSIONS

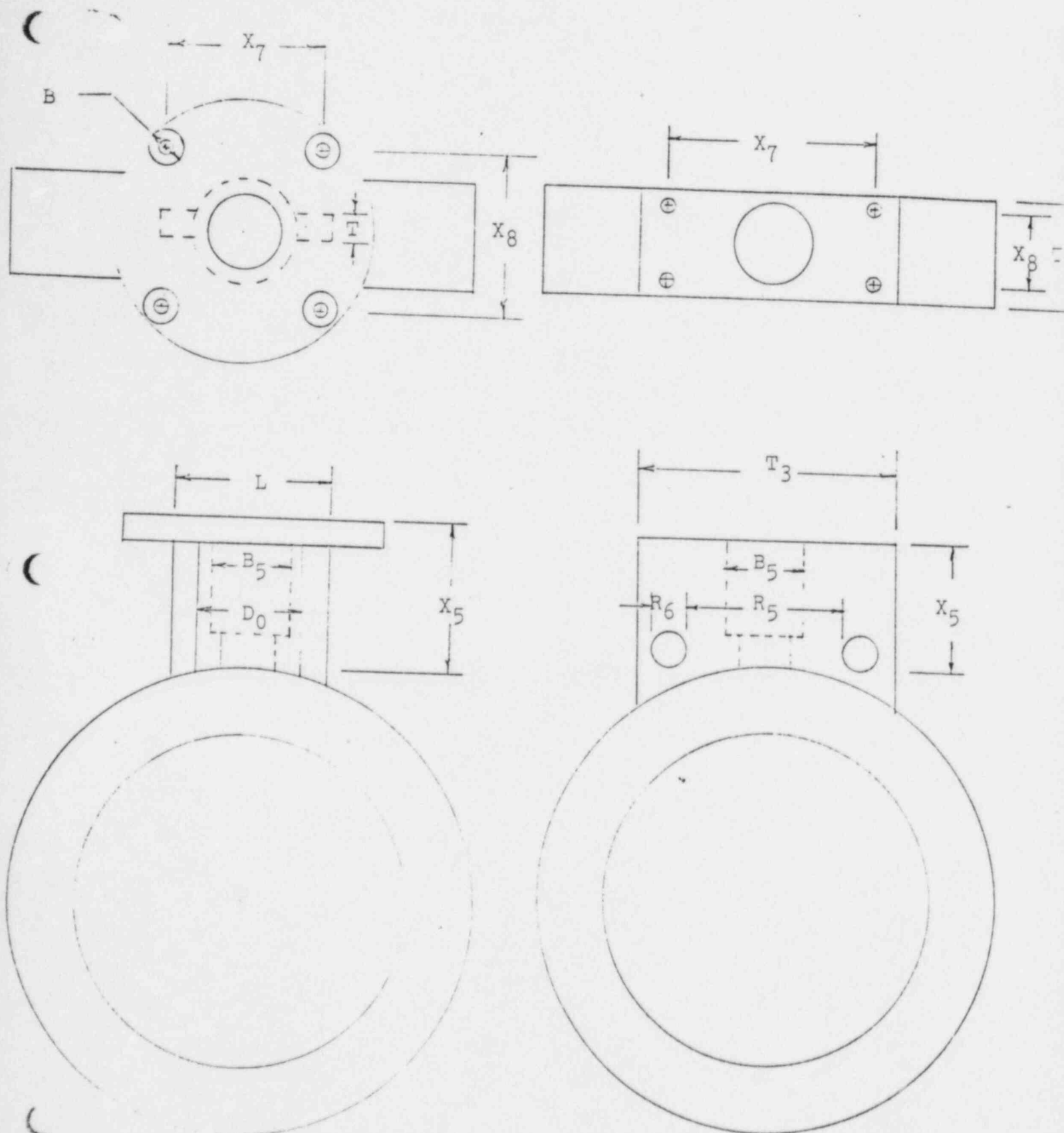


Figure 10

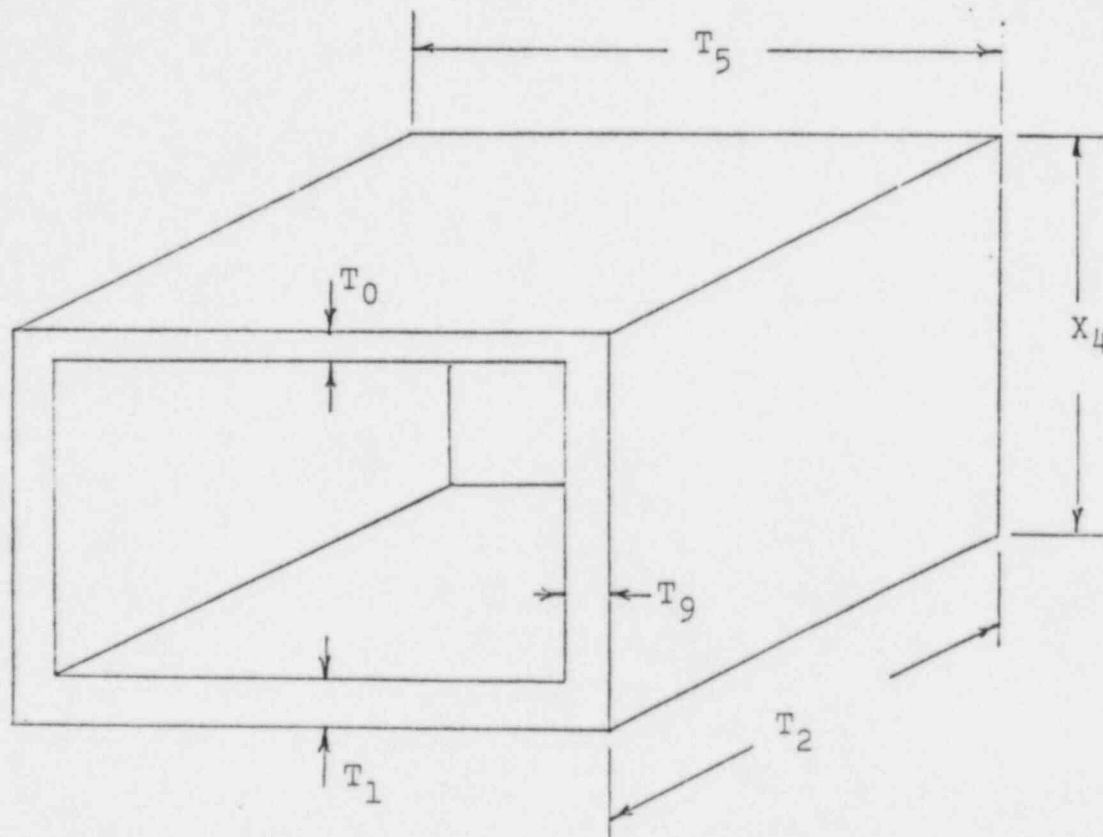
ACTUATOR MOUNTING BRACKET DIMENSIONS

Figure 11



# LONGITUDINAL (Z) NATURAL FREQUENCY

## DETERMINATION OF LONGITUDINAL SPRINGRATE

### Springrate of the Neck

Springrate of the neck due to the shear in the Z direction

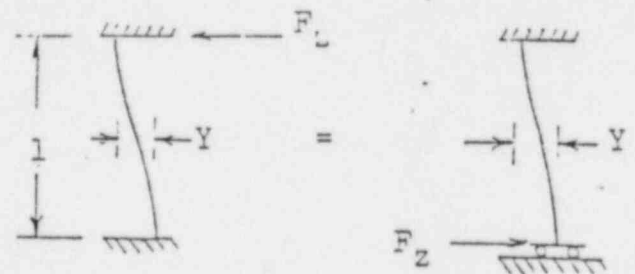
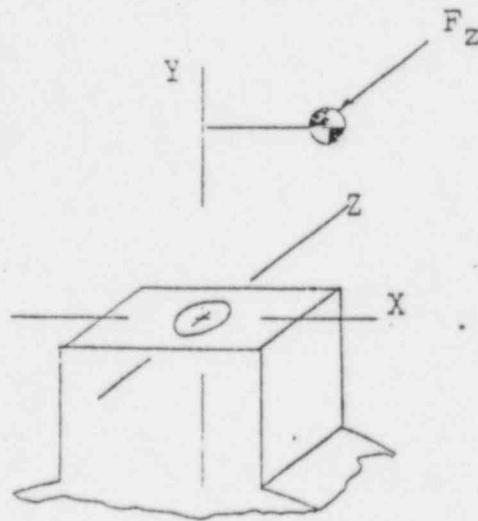
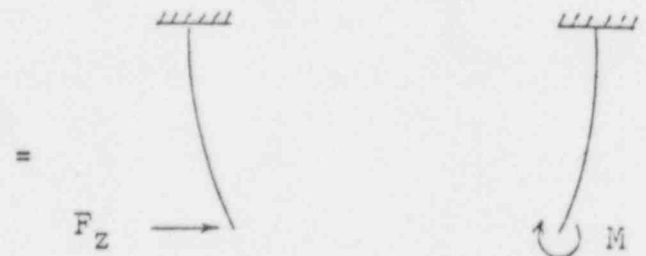


Figure 12



$$Y_F = \frac{F_Z l^3}{3 EI_X}$$

$$\theta_F = \frac{F_Z l^2}{2 EI_X}$$

$$Y = Y_F + Y_M$$

$$\theta = \theta_F + \theta_M$$

$$Y_M = \frac{M l^2}{2 EI_X}$$

$$\theta_M = \frac{M l}{EI_X}$$

$$Y = \frac{F_Z l^3}{3 EI_X} + \frac{M l^2}{2 EI_X}$$

$$\theta = \frac{F_Z l^2}{2 EI_X} + \frac{M l}{EI_X} = 0$$

$$M = - \frac{F_Z l}{2}$$

# LONGITUDINAL SPRINGRATE OF NECK

$$Y = \frac{F_z l^3}{3 EI_x} - \frac{F_z l^3}{4 EI_x}$$

$$Y = \frac{F_z l^3}{12 EI_x}$$

$$K = \frac{F_z}{Y} = \frac{12 EI_x}{l^3}$$

Circular Cross Section

$$I_x = \frac{\pi}{64} (D_0^4 - B_5^4)$$

Rectangular Cross Section

$$I_x = \frac{T_3 T_4^3}{12} - \frac{\pi B_5^4}{64}$$

With Gussets

$$I_x = I_x + \frac{(L - D_0) T^3}{12}$$

For the Neck  $l = X_5$

$$\therefore K_{NF_z} = \frac{12 EI_x}{X_5^3}$$

Springrate of the neck due to the moment about the X - axis

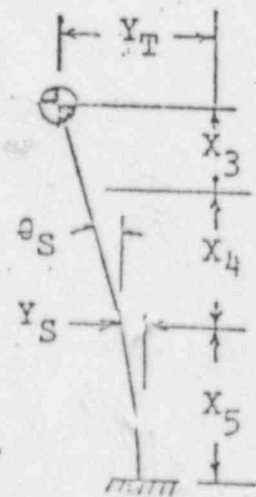
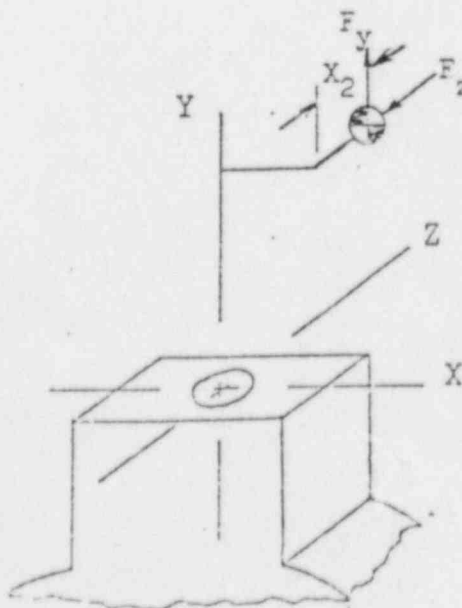


Figure 13

LONGITUDINAL SPRINGRATE OF NECK

$$Y_T = Y_S + e_S (X_3 + X_4)$$

$$Y_S = Y_F + Y_M$$

$$e_S = e_F + e_M$$

$$Y_F = \frac{F_z X_5^3}{3 EI_x}$$

$$Y_M = \frac{M X_5^2}{2 EI_x} \quad M = F_z (X_3 + X_4) + F_y X_2$$

$$F_y = F_z \quad Y_M = \frac{F_z (X_2 + X_3 + X_4) X_5^2}{2 EI_x}$$

$$e_F = \frac{F_z X_5^2}{2 EI_x}$$

$$e_F (X_3 + X_4) = \frac{F_z (X_3 + X_4) X_5^2}{2 EI_x}$$

$$e_M = \frac{M X_5}{EI_x} = \frac{F_z (X_2 + X_3 + X_4) X_5}{EI_x}$$

$$e_M (X_3 + X_4) = \frac{F_z (X_3 + X_4) (X_2 + X_3 + X_4) X_5}{EI_x}$$

$$Y_T = F_z \left[ \frac{X_5^3/3 + (X_3 + X_4) X_5^2/2 + (X_2 + X_3 + X_4) X_5^2/2}{EI_x} + \frac{(X_3 + X_4) (X_2 + X_3 + X_4) X_5}{EI_x} \right]$$

$$K = \frac{F}{Y_T}$$

$$K_{TMX} = \frac{EI_x}{X_5^3/3 + (X_3 + X_4) X_5^2/2 + (X_2 + X_3 + X_4) X_5^2/2 + (X_3 + X_4) (X_2 + X_3 + X_4) X_5}$$

$$K_{TZ} = \frac{K_{TFZ} K_{TMX}}{K_{TFZ} + K_{TMX}}$$

LONGITUDINAL SPRINGRATE OF BRACKETSpringrate of Bracket

Springrate due to twisting of the bottom plates of bracket per Seely and Smith, Page 271, Ref. (e).

$$\phi = \frac{1}{B b h^3} \left[ \frac{T}{G} \right] \quad \text{where } B = .333 = \frac{1}{3} \text{ for } b \gg h$$

$G = \text{Shear Modulus}$

Looking at one side of bracket.

Assume the width of the bracket that resists the twist of the plate is the distance between the bolts.

$$\therefore b = X_8$$

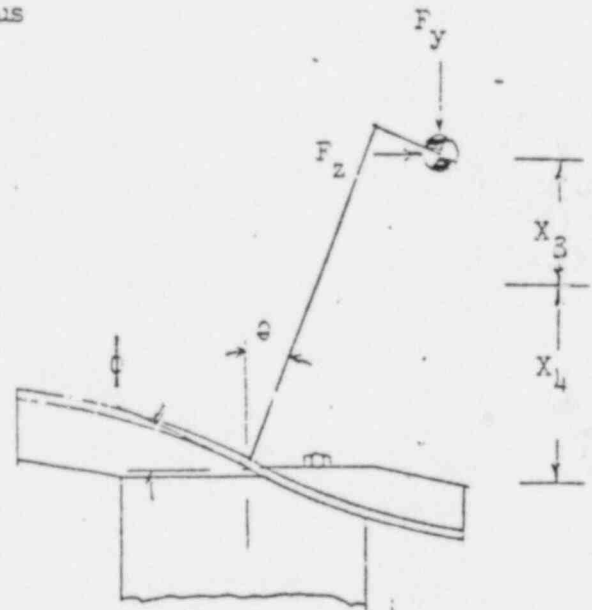


Figure 14

$$G = 12 \times 10^6 \quad h = T_1$$

$$T = M_x = \frac{F_z}{2} (X_4 - T_1 + X_3) + \frac{F_y}{2} X_2$$

$$F_y = F_z \quad \phi = \frac{3}{X_8 T_1^3} \left[ \frac{F_z (X_4 - T_1 + X_3 + X_2)}{2 (12 \times 10^6)} \right] = \frac{F_z (X_4 - T_1 + X_3 + X_2)}{X_8 (4 \times 10^6) T_1^3}$$

$$\theta = \phi d \quad \text{where } d \text{ is assumed to be } = \frac{T_5 - 2T_9 - X_7}{2}$$

$$\theta = F_z \frac{(X_4 - T_1 + X_3 + X_2) (T_5 - 2T_9 - X_7)}{X_8 \cdot 16 \times 10^6 T_1^3}$$

LONGITUDINAL SPRINGRATE OF BRACKET

$$Y = \theta (X_4 - T_1 + X_3)$$

$\tan \theta = \theta$  for small angles

$$Y = \frac{F_z (X_4 - T_1 + X_3) (X_4 - T_1 + X_3 + X_2) (T_5 - 2T_9 - X_7)}{X_8 \cdot 16 \times 10^6 \cdot T_1^3}$$

$$K_{EBM_x} = \frac{X_8 \cdot T_1^3 \cdot 16 \times 10^6}{(X_4 - T_1 + X_3) (X_4 - T_1 + X_3 + X_2) (T_5 - 2T_9 - X_7)}$$

Since the springrate of the bottom plate for one half of the bracket is based on half the force acting on it, the springrate of all of the bottom plate equals

$$K_{EBM_x} = \frac{X_8 \cdot T_1^3 \cdot 16 \times 10^6}{(X_4 - T_1 + X_3) (X_4 - T_1 + X_3 + X_2) (T_5 - 2T_9 - X_7)}$$

Springrate due to the twisting of the top plate of the bracket

$$b = X_6 - \text{DBC of Actuator}$$

$$K_{BTM_x} = \frac{X_6 \cdot T_0^3 \cdot 16 \times 10^6}{X_3 (X_3 + X_2) (T_5 - 2T_9 - X_6)}$$

$$K_{EM_x} = \frac{K_{EBM_x} \cdot K_{BTM_x}}{K_{EBM_x} + K_{BTM_x}}$$

LONGITUDINAL SPRINGRATE OF BRACKET

Springrate of the bracket due to the shear in the Z direction per

Page 29 
$$K_{\text{Shear}} = \frac{12 EI_x}{l^3}$$

$$K_{BF_z} = \frac{12 EI_x}{X_4 - T_1 - T_0}$$

$$I_x = (2) \frac{T_9 T_2^3}{12}$$

$$K_{BF_z} = \frac{12 E T_9 T_2^3}{6 (X_4 - T_1 - T_0)^3}$$

$$K_{BF_z} = \frac{60 \times 10^6 T_9 T_2^3}{(X_4 - T_1 - T_0)^3}$$

Springrate of the bracket due to the bending of the side plates

$$Y_T = Y_S + e_S (X_3 + T_0)$$

$$Y_S = Y_F + Y_M$$

$$e_S = e_F + e_M$$

$$Y_F = \frac{F_z (X_4 - T_0 - T_1)^3}{3 EI_x}$$

$$Y_M = \frac{M (X_4 - T_0 - T_1)^2}{2 EI_x}$$

$$M_x = F_z (X_3 + T_0) + F_y X_2$$

$$F_y = F_z$$

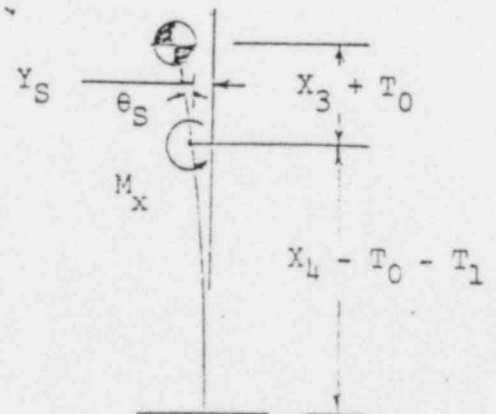


Figure 15

LONGITUDINAL SPRINGRATE OF BRACKET

$$Y_M = \frac{F_z (X_3 + T_0 + X_2) (X_4 - T_0 - T_1)^2}{2 EI_x}$$

$$\theta_F = \frac{F_z (X_4 - T_0 - T_1)^2}{2 EI_x}$$

$$\theta_F (X_3 + T_0) = \frac{F_z (X_3 + T_0) (X_4 - T_0 - T_1)}{2 EI_x}$$

$$\theta_M = \frac{M(X_4 - T_0 - T_1)}{EI} = \frac{F_z (X_3 + T_0 + X_2) (X_4 - T_0 - T_1)}{EI_x}$$

$$\theta_M (X_3 + T_0) = \frac{F_z (X_3 + T_0) (X_3 + T_0 + X_2) (X_4 - T_0 - T_1)^2}{EI}$$

$$Y_T = F_z \left[ \frac{(X_4 - T_0 - T_1)^3}{3 EI_x} + \frac{(X_4 - T_0 - T_1)^2 (X_3 + T_0)}{2 EI_x} + \frac{(X_4 - T_0 - T_1)^2 (X_3 + T_0 + X_2)}{2 EI_x} + \frac{(X_4 - T_0 - T_1) (X_3 + T_0) (X_3 + T_0 + X_2)}{EI_x} \right]$$

$$K_{BSM_x} = \frac{F_z}{Y_T} = \frac{EI_x}{(X_4 - T_0 - T_1)^3/3 + (X_4 - T_0 - T_1)^2(X_3 + T_0)/2 + (X_4 - T_0 - T_1)^2(X_3 + T_0 + X_2)/2 + (X_4 - T_0 - T_1) (X_3 + T_0) (X_3 + T_0 + X_2)}$$

$$I_x = \frac{(2)T_9 T_2^3}{12} = \frac{T_9 T_2^3}{6}$$

$$K_{BS_z} = \frac{K_{BF_z} K_{BSM_x}}{K_{BF_z} + K_{BSM_x}}$$

$$K_B = \frac{K_{BM_x} K_{BS_z}}{K_{BM_x} + K_{BS_z}}$$

$$K_{z1} = \frac{K_B K_{N_z}}{K_B + K_{N_z}}$$

## SPRINGRATE OF VALVE NECK FLANGE

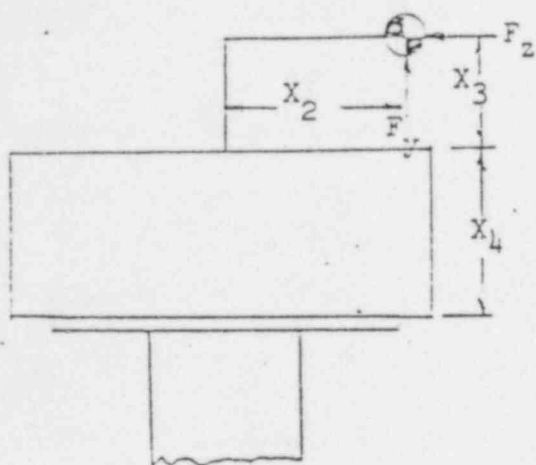
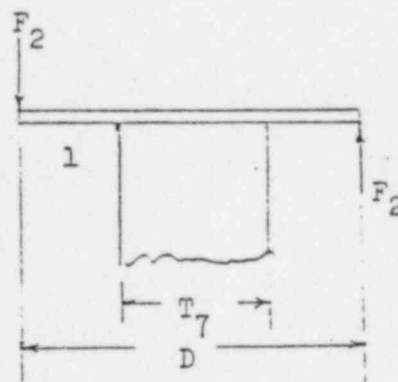


Figure 16



$$T_7 = T_4 \text{ or } D_0$$

Case 22  
Roark Ref. (e)

$$\Sigma M_1 = F_y \left( X_2 + \frac{D}{2} \right) + F_z (X_3 + X_4) - F_2 D$$

$$F_y = F_z$$

$$F_2 = \frac{F_z (X_2 + D/2 + X_3 + X_4)}{D}$$

$$y = f(F_2) = F_2 C$$

$$\frac{y}{D/2} = \frac{X}{(X_3 + X_4)}$$

$$X = \frac{2y (X_3 + X_4)}{D}$$

$$X = \frac{2 (X_3 + X_4) F_z (X_2 + D/2 + X_3 + X_4) C}{D^2}$$

$$K_{FL} = \frac{F}{X} \frac{D^2}{2(X_3 + X_4) (X_2 + D/2 + X_3 + X_4) C}$$

$$K_{FL} = \frac{2.3 D^2 T_6^3 E}{(X_3 + X_4) (X_2 + D/2 + X_3 + X_4)} \left[ \frac{4.33 D^2 + 2.33 T_7^2}{11 D^4 - 2.33 T_7^4 - 8.67 D^2 T_7^2 - 26.8 E^2 T_7^2 \ln \left( \frac{D}{T_7} \right) - 17.3 D^2 T_7^2 (\ln(D/T_7))^2} \right]$$

$$K_z = \frac{K_{Z1} K_{FL}}{K_{Z1} + K_{FL}}$$



LONGITUDINAL (Z) NATURAL FREQUENCY

$$M = \frac{W_4 + W_5}{386}$$

Where  $W_4$  = Weight of Actuator

$W_5$  = Weight of Bracket

$$f_{n_z} = \frac{1}{2\pi} \sqrt{\frac{K_z}{M}} \quad H_z$$

TRANSVERSE (X) NATURAL FREQUENCYDETERMINATION OF TRANSVERSE SPRINGRATESpringrate of Neck

Springrate of the neck due to the shear in the X direction

$$K_{NF_x} = \frac{12 EI_z}{X_5^3}$$

$$I_z = \frac{\pi}{64} (D_0^4 - B_5^4) \quad \text{Circular}$$

$$I_z = \frac{T_4 T_3^3}{12} - \frac{\pi}{64} B_5^4 \quad \text{Rectangular}$$

$$I_z = I_z + \frac{T(L-D_0)^3}{6} + \frac{2(L-D_0)T(L+D_0)}{4}$$

with Gusset

Springrate of the neck due to the moment about the Z - Axis

$$K_{NM_z} = \frac{EI_z}{\frac{X_5^3}{3} + (X_3 + X_4) \frac{X_5^2}{2} + (X_1 + X_3 + X_4) \frac{X_5^2}{2} + (X_3 + X_4) (X_1 + X_3 + X_4) X_5}$$

$$K_{N_x} = \frac{K_{NF_x} K_{NM_z}}{K_{NF_x} + K_{NM_z}}$$

TRANSVERSE SPRINGRATE OF BRACKETSPRINGRATE OF BRACKET

Springrate due to the moment about the Z - axis acting on the bottom plate at the bracket

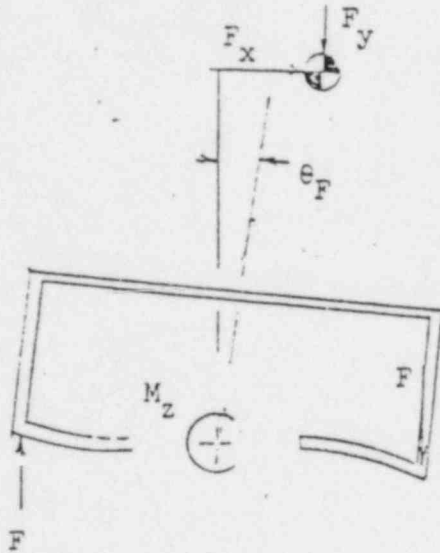


Figure 17

$$Y = \theta_F (X_4 - T_1 + X_3)$$

$$F = \frac{M_z}{T_5 - T_9} \quad M_z = F_x (X_4 - T_1 + X_3) + F_y X_1$$

$$F_y = F_x$$

$$F = \frac{F_x (X_4 - T_1 + X_3 + X_1)}{T_5 - T_9}$$

$$\theta_F = \frac{F l^2}{2 EI} = \frac{F \left[ (T_5 - 2T_9 - X_7) \sqrt{2} \right]^2}{2 EI_z}$$

$$\theta_F = \frac{F_x (X_4 - T_1 + X_3 + X_1)}{2 EI_z (T_5 - T_9)} \left[ (T_5 - 2T_9 - X_7) \sqrt{2} \right]^2$$

$$Y = \frac{F_x (X_4 - T_1 + X_3) (X_4 - T_1 + X_3 + X_1) (T_5 - 2T_9 - X_7)^2}{8 EI_z (T_5 - T_9)}$$

$$K_{EBM_z} = \frac{8 EI_z (T_5 - T_9)}{(X_4 - T_1 + X_3) (X_1 + X_4 - T_1 + X_3) (T_5 - 2T_9 - X_7)^2} \quad I_z = \frac{T_1^3 T_2}{12}$$

TRANSVERSE SPRINGRATE OF BRACKET

$$K_{BBM_z} = \frac{20 \times 10^6 T_2 T_1^3 (T_5 - T_9)}{(X_4 - T_1 + X_3) (X_4 - T_1 + X_3 + X_1) (T_5 - 2T_9 - X_7)^2}$$

Springrate due to the moment about the Z - axis acting on the top plate of the bracket

$$K_{BTM_z} = \frac{20 \times 10^6 T_2 T_0^3 (T_5 - T_9)}{X_3 (X_3 + X_1) (T_5 - 2T_9 - X_6)^2}$$

$$K_{BM_z} = \frac{K_{BBM_z} K_{BTM_z}}{K_{BBM_z} + K_{BTM_z}}$$

Springrate of the bracket due to the shear in the X - direction.

$$K_{BF_x} = K_{BF_z} \quad \text{except } l_z = \frac{T_9^3 T_2}{6}$$

$$K_{BF_x} = \frac{60 \times 10^6 T_2 T_9^3}{(X_4 - T_1 - T_0)^3}$$

TRANSVERSE SPRINGRATE OF BRACKET

Springrate of the bracket due to the bending of the side plates.

$$K_{BSM_z} = K_{BSM_x} \quad \text{except} \quad I_z = \frac{T_9^3 T_2}{6} + 2T_9 T_2 \frac{(T_5 - T_9)^2}{2} \quad \& \text{Substitute } X_1 \text{ for } X_2$$

$$K_{BSM_z} = \frac{EI_z}{(X_1 - T_0 - T_1)^3/3 + (X_4 - T_0 - T_1)^2(X_3 + T_0)/2 + (X_4 - T_0 - T_1)^2(X_3 + T_0 + X_1)/2 + (X_4 - T_0 - T_1)(X_3 + T_0)(X_3 + T_0 + X_1)}$$

$$K_{BS_x} = \frac{K_{BF_x} K_{BSM_z}}{K_{BF_x} + K_{BSM_z}}$$

$$K_{B_x} = \frac{K_{BM_z} K_{BS_x}}{K_{BM_z} + K_{BS_x}}$$

$$K_{x1} = \frac{K_{B_x} K_{N_x}}{K_{B_x} + K_{N_x}}$$

Springrate of valve neck flange

$$T_7 = T_4 \text{ or } D_0$$

$$K_{FT} = \frac{2.3 D^2 T_6^3 E}{(X_3 + X_4)(X_1 - \frac{D}{2} + X_3 + X_4)}$$

See Page 35 for expression inside brackets

$$K_x = \frac{K_{x1} K_{FY}}{K_{x1} + K_{FY}}$$

$$f_{n_x} = \frac{1}{2\pi} \sqrt{\frac{K_x}{M}} \quad H_z$$

1. ACTUATOR/BACKET BOLTING.

## LOCA STRESSES -

$$S_{SOX} = \frac{1.414T}{N_1 A_1 X_6}$$

WHERE: T = LOCA TORQUE (in LBS).  
 $N_1$  = NO. OF ACTUATOR BOLTS

$A_1$  = TENSILE STRESS AREA OF ACTUATOR BOLTS (IN<sup>2</sup>)

$$S_{SOZ} = S_{SOX}$$

$X_6$  =  $D_{BC}$  OF ACTUATOR BOLTS (IN.)

$OL_{T,V}$  = Occasional Load (Transverse, Vertical)

$G_1$  = Transverse Seismic Acceleration

$G_2$  = Vertical Seismic Acceleration

$G_3$  = Longitudinal Seismic Acceleration

## SEISMIC STRESSES -

A. VERTICAL DIRECTION ( $F_y$ )

$$F_y = W_1 * G_2 + OL_V \quad M_z = F_y X_1$$

$$M_x = F_y X_2 \quad W_1 = \text{Actuator Weight}$$

$$W_2 = \text{Bracket Weight}$$

$$F_x = \left[ \frac{4}{X_6 N_1} \cos \frac{\pi}{N_1} \right] M_x \quad \text{MAXIMUM BOLT LOAD (LBS.)}$$

Per Ref (f)

$$F_z = \left[ \frac{4}{X_6 N_1} \cos \frac{\pi}{N_1} \right] M_z$$

$$F_T = \frac{F_y}{N_1} + F_x + F_z \quad \text{(LBS) TOTAL TENSILE LOAD ON BOLT}$$

$$S_{Ty} = \frac{F_T}{A_1} \quad \text{(PSI) TENSILE STRESS}$$

B. TRANSVERSE DIRECTION ( $F_x$ )

$$F_x = W_1 * G_1 + OL_T \quad T_y = F_x X_2 \quad M_z = F_x X_3$$

$$S_{SSX} = \frac{1.414T_y}{N_1 A_1 X_6} + \frac{F_x}{N_1 A_1} \quad \text{(PSI) SHEAR STRESS}$$

$$F_{RZ} = \left[ \frac{4}{X_6 N_1} \cos \frac{\pi}{N_1} \right] M_z \quad \text{MAXIMUM BOLT LOAD (LBS.)}$$

$$S_{TZ} = \frac{F_{RZ}}{A_1} \quad \text{(PSI) TENSILE STRESS}$$

C. LONGITUDINAL DIRECTION ( $F_z$ )

$$F_z = W_1 * G_3 \quad T_y = F_z X_1 \quad M_x = F_z X_3$$

$$S_{SSZ} = \frac{1.414T_y}{N_1 A_1 X_6} + \frac{F_z}{N_1 A_1}$$

$$F_{RX} = \left[ \frac{4}{X_6 N_1} \cos \frac{\pi}{N_1} \right] M_x \quad \text{MAXIMUM BOLT LOAD (LBS.)}$$

$$S_{TX} = \frac{F_{RX}}{A_1} \quad \text{(PSI) TENSILE STRESS}$$

$$S_{SS} = \sqrt{(S_{SOX} + S_{SSX})^2 + (S_{SOZ} + S_{SSZ})^2}$$

$$S_{TS} = \sqrt{S_{TY}^2 + S_{TZ}^2 + S_{TX}^2} \text{ (PSI) TENSILE}$$

MAXIMUM STRESS: MAX. STRESS THEORY

$$S_T = \frac{S_{TS}}{2} + \sqrt{\left(\frac{S_{TS}}{2}\right)^2 + S_{SS}^2}$$

BRACKET

LOCA STRESSES -

$$S_B = \frac{Mc}{I_x} \quad F = T/T_5 \quad M = X_4 T/T_5$$

$$I_x = \frac{T_9 T_2^3}{12} \quad c = \frac{T_2}{2}$$

$$S_B = \frac{6 X_4 T}{T_5 T_9 T_2^2}$$

TENSILE (PSI)

$S_{SSO}$

SHEAR (PSI)  
See Page 42A

SEISMIC STRESSES -

A. VERTICAL DIRECTION ( $F_y$ )

$$F_y = (W_1 + .5W_2) G_2 + OL_V \quad M_Z = F_y X_1 \quad M_X = F_y X_2$$

$$S_{TY} = \frac{F_y}{2T_9 T_2} + \frac{M_Z T_5}{2J_3} + \frac{M_X T_2}{2J_2} \quad \text{TENSILE (PSI)}$$

$$J_2 = 2 \left[ \frac{T_9 T_2^3}{12} \right] \quad J_3 = 2 \left[ \frac{T_2 T_9^3}{12} + T_9 T_2 (T_{5/2} - T_{9/2})^2 \right]$$

B. TRANSVERSE DIRECTION ( $F_x$ )

$$F_x = (W_1 + .5W_2) G_1 + OL_T \quad T_y = F_x X_2 \quad M_Z = F_x (X_3 + X_4)$$

$$S_{SX} = S_{SSX} + \frac{F_x}{T_9 T_2} \quad \text{SHEAR (PSI)}$$

See Page 42A for derivation of  $S_{SSX}$

$$S_{TZ} = \frac{6 X_4 T_y}{T_5 T_9 T_2^2} + \frac{M_Z T_5}{2 J_3} + S_{BX} \quad \text{TENSILE (PSI)}$$

See Page 42B for derivation of  $S_{BX}$

C. LONGITUDINAL DIRECTION ( $F_z$ )

$$F_z = (W_1 + .5W_2) G_3 \quad T_y = F_z X_1 \quad M_X = F_z (X_3 + X_4)$$

$$S_{SZ} = \frac{T_y}{T_9 T_2 T_5} + \frac{F_z}{T_9 T_2} \quad \text{SHEAR (PSI)}$$

$$S_{TX} = \frac{6 X_4 T_y}{T_5 T_9 T_2^2} + \frac{M_X T_2}{2 J_2}$$

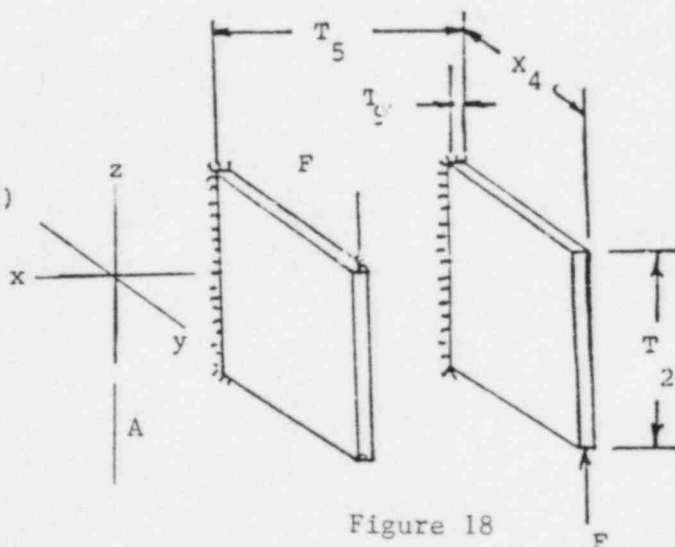
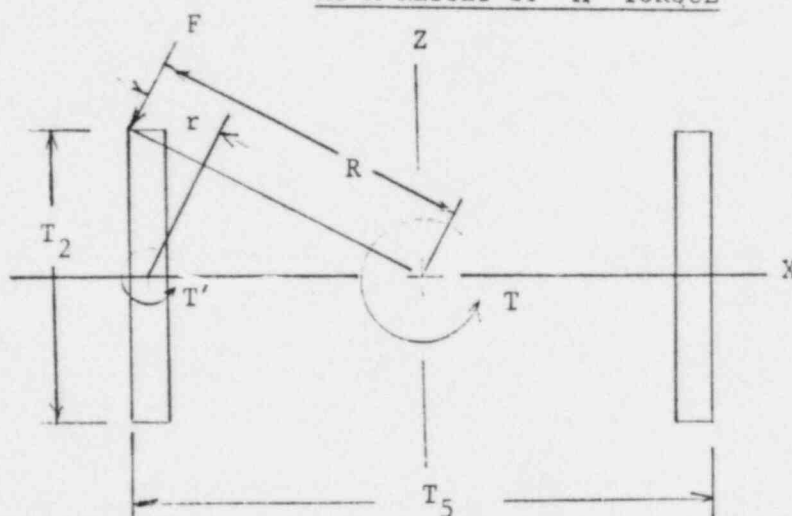


Figure 18

STRESSES ACTING ON THE BRACKET IN THE X DIRECTION  
AS A RESULT OF A TORQUE



$$F = \frac{T}{T_5}$$

$$R = \sqrt{\left(\frac{T_2}{2}\right)^2 + \left(\frac{T_5}{2}\right)^2}$$

$$r = R - \frac{T_5}{2} \cos \theta$$

$$\cos \theta = \frac{T_5}{\sqrt{T_2^2 + T_5^2}}$$

$$r = \frac{\sqrt{T_2^2 + T_5^2}}{2} - \frac{T_5^2}{2 \sqrt{T_2^2 + T_5^2}}$$

$$T' = r F$$

$$S_S = \frac{T(3a + 1.8b)}{8a^2 b^2}$$

per Table IX, Case 4, Ref. (c)

$$a = \frac{T_2}{2} \quad b = \frac{T_9}{2}$$

$$S_S = \frac{T}{T_5} \left( \frac{\sqrt{T_2^2 + T_5^2}}{2} - \frac{T_5^2}{2 \sqrt{T_2^2 + T_5^2}} \right) \left( \frac{3 \frac{T_2}{2} + 1.8 \frac{T_9}{2}}{8 \left(\frac{T_2}{2}\right)^2 \left(\frac{T_9}{2}\right)^2} \right)$$

for  $S_{SSO} - T = \text{Valve Operating Torque}$

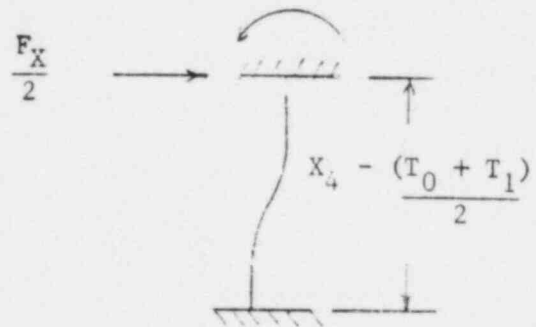
for  $S_{SSX} - T = T_y$



BRACKET STRESS DUE TO TRANSVERSE SHEAR

Maximum Bending Moment

$$M = \frac{\frac{F_X}{2} \left( X_4 - \left( \frac{T_0 + T_1}{2} \right) \right)}{2}$$



$$\text{Section Modulus} = \frac{T_2 T_9^2}{6}$$

$$\text{Bending Stress} = S_{BX} = \frac{1.5 F_X \left( X_4 - (T_0 + T_1)/2 \right)}{T_2 T_9^2}$$

$$S_{SS} = \sqrt{S_{SX}^2 + S_{SZ}^2 + S_S^2} \quad \text{SHEAR (PSI)}$$

$$S_{TS} = \sqrt{S_{TY}^2 + S_{TZ}^2 + S_{TX}^2 + S_B^2} \quad \text{TENSILE (PSI)}$$

MAXIMUM STRESS: MAX. STRESS THEORY

$$S_T = \frac{S_{TS}}{2} + \sqrt{\left[\frac{S_{TS}}{2}\right]^2 + S_{SS}^2}$$

BRACKET/VALVE BOLTING

LOCA STRESSES -

$$S_{SOX} = \frac{2XT}{N_2 A_2 X_9}$$

$$S_{SOZ} = \frac{2 X_7 T}{N_2 A_2 X_9^2}$$

WHERE" T= LOCA TORQUE (IN-LBS)

N<sub>2</sub> = NO. OF BRACKET BOLTS= 4A<sub>2</sub> = TENSILE STRESS AREA OF BOLTS (IN.)

$$X_9 = \sqrt{X_7^2 + X_8^2}$$

= EQUIVALENT BOLT DIA. (IN.)

SEISMIC STRESSES -

A. VERTICAL DIRECTION (F<sub>y</sub>)

$$F_y = (W_1 + W_2) * G_2 + OL_V \quad M_Z = F_y X_1 \quad M_X = F_y X_2$$

$$F_{RX} = \frac{M_X}{2X_8} \quad F_{RZ} = \frac{M_Z}{2X_7} \quad F_T = \frac{F_y}{N_2} + F_{RX} + F_{RZ}$$

$$S_{Ty} = \frac{F_T}{A_2} \quad \text{TENSILE (PSI)}$$

TOTAL TENSILE LOAD ON BOLT (LBS.)

B. TRANSVERSE DIRECTION (F<sub>x</sub>)

$$F_x = (W_1 + W_2) * G_1 + OL_T \quad T_y = F_x X_2 \quad M_Z = F_x (X_3 + X_4)$$

$$S_{SSX} = \frac{2X_7 T_y}{N_2 A_2 X_9^2} + \frac{F_x}{N_2 A_2} \quad \text{SHEAR (PSI)}$$

$$F_{RZ} = \frac{M_Z}{2X_7} \quad S_{TZ} = \frac{F_{RZ}}{A_2} \quad \text{TENSILE (PSI)}$$

C. LONGITUDINAL DIRECTION (F<sub>z</sub>)

$$F_z = (W_1 + W_2) * G_3 \quad T_y = F_z X_1 \quad M_X = F_z (X_3 + X_4)$$

$$S_{SSZ} = \frac{2X_7 T_y}{N_2 A_2 X_9^2} + \frac{F_z}{N_2 A_2}$$

$$F_{RX} = \frac{M_X}{2X_8} \quad S_{TX} = \frac{F_{RX}}{A_2} \quad \text{TENSILE (PSI)}$$

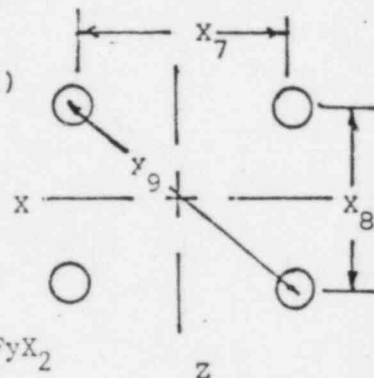


Figure 19

$$S_{SS} = \sqrt{(S_{SSX} + S_{SOX})^2 + (S_{SSZ} + S_{SOZ})^2}$$

$$S_{TS} = \sqrt{S_{TY}^2 + S_{TZ}^2 + S_{TX}^2}$$

MAXIMUM STRESS: MAX. STRESS THEORY

$$S_T = \frac{S_{TS}}{2} + \sqrt{\left(\frac{S_{TS}}{2}\right)^2 + S_{SS}^2}$$

VALVE NECK -

OPERATING &amp; LOCA STRESSES -

$$S_S = \frac{T_C}{J_1} \quad \text{SHEAR}$$

 $W_3$  = Weight of Valve Neck (LBS)

$$J_1 = \frac{8(T_3/2)^2(T_4/2)^3}{3(T_3/2 + 1.8T_4/2)} - \frac{8(R_5/2 + R_6)^2(T_4/2)^3}{3(R_5/2 + R_6) + 1.8T_4/2}$$

$$+ \frac{8(R_5/2)^2(T_4/2)^3}{3(R_5/2) + 1.8T_4/2} - \frac{\pi}{32} Di^4$$

Based on shear stress equation given in Table IX, Case 4, of Ref. (c)

$$\text{OR } J_1 = \frac{\pi}{32} (d_3^4 - Di^4)$$

$$C = \frac{T_4}{2} \quad \text{DIST. TO MAX. FIBER STRESS}$$

$$S_P = P \left[ \frac{T_4^2 + Di^2}{T_4^2 - Di^2} \right] \quad \text{OR } P \left[ \frac{d_3^2 + Di^2}{d_3^2 - Di^2} \right]$$

HOOP (PSI)

Figure 20

Based on hoop stress equation given in Table XIII, Case 33, of Ref. (c).

SEISMIC STRESSES -

A. VERTICAL DIRECTION ( $F_y$ )

$$T_4 \text{ \& } d_3 = 26 \quad Di = 2a$$

$$F_y = (W_1 + W_2 + W_3) * G_2 + OL_V \quad M_Z = F_y X_1 \quad M_X = F_y X_2$$

$$S_{Ty} = \frac{F_y}{A} + \frac{M_Z T_3}{2J_3} + \frac{M_X T_4}{2J_2} \quad \text{TENSILE (PSI)}$$

$$A = (T_3 - 2R_5)T_4 - \pi \frac{Di^2}{4}$$

OR

$$A = \frac{\pi}{4} (d_3^2 - Di^2) \text{ (in}^2\text{)}$$

$$J_2 = \frac{(T_3 - 2R_6)T_4^3}{12} - \frac{\pi}{64} Di^4 \quad \text{OR } \frac{\pi}{64} (d_3^4 - Di^4)$$

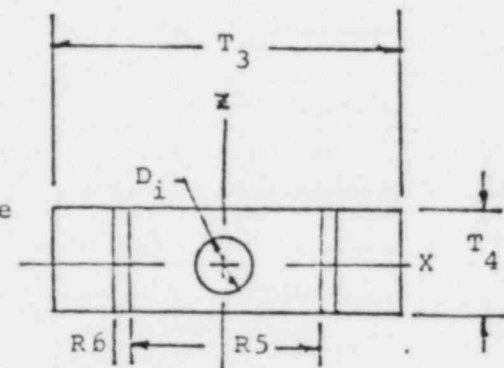
$$J_3 = \frac{T_4 T_3^3}{12} - \frac{2R_5^3 T_4}{12} - 2R_5 T_4 (R_6/2 + R_5/2)^2 - \left( \frac{\pi}{64} Di^4 \right) \text{ OR } \frac{\pi}{64} (d_3^4 - Di^4)$$

B. TRANSVERSE DIRECTION ( $F_X$ )

$$F_X = (W_1 + W_2 + W_3) * G_1 + OL_T \quad T_y = F_X X_2 \quad M_Z = F_X (X_3 + X_4 + X_5)$$

$$S_{SX} = \frac{T_y T_4}{2J_1} + \frac{F_X}{A} \quad \text{SHEAR (PSI)}$$

$$S_{TZ} = \frac{M_Z T_3}{2J_3} \quad \text{TENSILE (PSI)}$$



C. LONGITUDINAL DIRECTION ( $F_z$ )

$$F_z = (W_1 + W_2 + W_3) * G_3 \quad T_y = F_z X_1 \quad M_x = F_z (X_3 + X_4 + X_5)$$

$$S_{SZ} = \frac{T_y T_4}{2J_1} + \frac{F_z}{A} \quad \text{SHEAR (PSI)}$$

$$S_{TX} = \frac{M_x T_4}{2J_2} \quad \text{TENSILE (PSI)}$$

ROOT SUM SQUARE STRESSES:

$$S_{SS} = \sqrt{S_{SX}^2 + S_{SZ}^2 + S_S} \quad \text{SHEAR (PSI)}$$

$$S_{TS} = \sqrt{S_{Ty}^2 + S_{Tz}^2 + S_{TX}^2 + S_P} \quad \text{TENSILE (PSI)}$$

MAXIMUM STRESS: MAX. STRESS THEORY

$$S_T = \frac{S_{TS}}{2} + \sqrt{\left[\frac{S_{TS}}{2}\right]^2 + S_{SS}^2}$$

Valve with Gussets

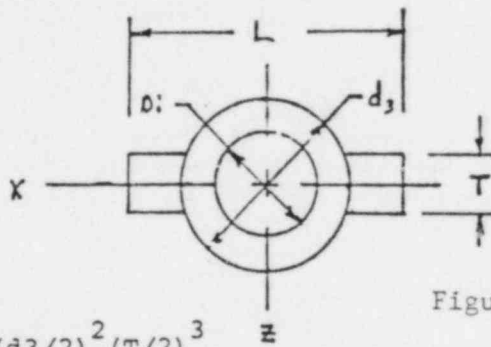


Figure 2

$$J_1 = J_1 + \frac{8 (L/2)^2 (T/2)^3}{3(L/2) + 1.8(T/2)} - \frac{8(d_3/2)^2 (T/2)^3}{3(d_3/2) + 1.8(T/2)}$$

$$C = \frac{d_3}{2}$$

$$J_2 = J_2 + \frac{(L - d_3) T^3}{12}$$

$$J_3 = J_3 + T \frac{\left[\frac{L - d_3}{2}\right]^3}{6} + 2 (L - d_3) T \left[\frac{L + d_3}{4}\right]^2$$

$$A = A + (L - d_3) T$$

$$W_3 = A * X_5 * .283$$

$$S_{Ty} = \frac{F_y}{A} + \frac{M_z L}{2J_3} + \frac{M_x d_3}{2J_2}$$

$$S_{SX} = \frac{T_y d_3}{2J_1} + \frac{F_x}{A} \quad S_{Tz} = \frac{M_z L}{2J_3}$$

$$S_{SZ} = \frac{T_y d_3}{2J_1} + \frac{F_z}{A} \quad S_{TX} = \frac{M_x d_3}{2J_2}$$

VALVE STEM:

## OPERATING &amp; LOCA STRESSES -

$$S_S = \frac{16T}{\pi D_1^3} \quad \text{SHEAR (PSI)}$$

$$F = \frac{1}{2} \frac{P}{A} = \frac{\pi P D^2}{8} \quad \frac{1}{2} \text{ APPLIED FORCE ON DISC (LBS.)}$$

THE EFFECTIVE BEARING LENGTH IS DERIVED AS FOLLOWS:

$$S_y = \frac{F}{A} \quad \begin{array}{l} \text{WHERE } A = \text{PROJECTED BEARING AREA} \\ = L_1 D_1 = \text{EFFECTIVE BEARING LENGTH TIMES THE STEM DIAMETER.} \end{array}$$

$$S_y = \frac{F}{L_1 D_1}$$

SINCE THE LOAD IS ASSUMED TO BE TRIANGULAR, HALF OF THE YIELD STRENGTH (OR 15000 psi) IS USED.

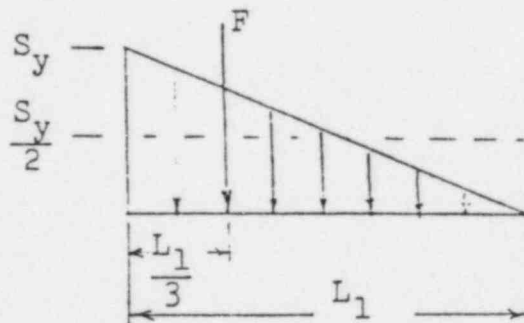


Figure 22

$$\text{THEREFORE, } L_1 = \frac{F}{15000 D_1}$$

AS CAN BE SEEN IN THE FIGURE ABOVE, IT IS ASSUMED THE RESULTANT BEARING REACTION OCCURS AT  $L_1/3$ .

$$S_b = \frac{4PD^2}{D_1^3} \left[ \frac{L_1}{3} + L_2 \right] \quad \text{BENDING DUE TO PRESSURE (PSI)}$$

WHERE  $\left[ \frac{L_1}{3} + L_2 \right]$  EQUALS LENGTH OF STEM FROM THE END OF DISC TO THE BEARING REACTION.

## SEISMIC STRESSES -

$$M = \frac{W L_g}{2} \left[ \frac{L_1}{3} + L_2 \right] \quad \epsilon = \sqrt{\epsilon_x^2 + \epsilon_z^2}$$

$$S_{b1} = \frac{32M}{\pi D_1^3} = \frac{16 W L_g}{\pi D_1^3} \left[ \frac{L_1}{3} + L_2 \right] \quad \text{BENDING (PSI)}$$

$$S_{bs} = S_b + S_{b1}$$

MAXIMUM STRESS: MAX. STRESS THEORY

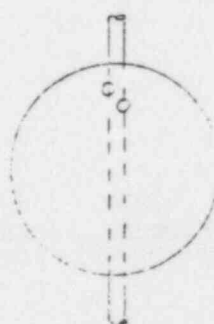
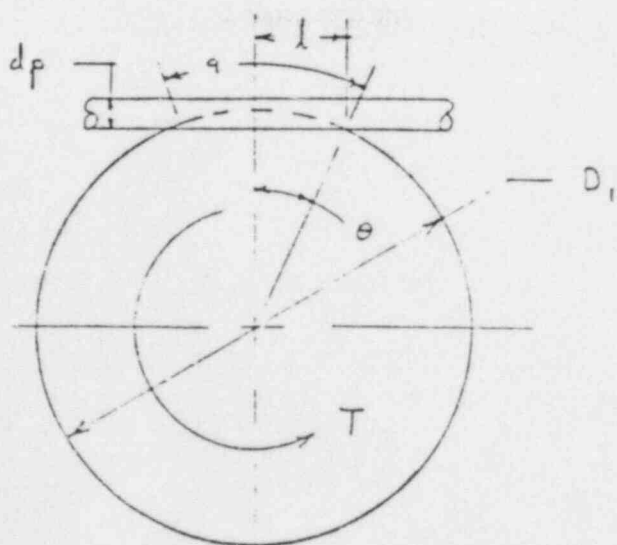
$$S_t = \frac{S_{bs}}{2} + \sqrt{\left[ \frac{S_{bs}}{2} \right]^2 + S_s^2}$$

VALVE DISC PIN:

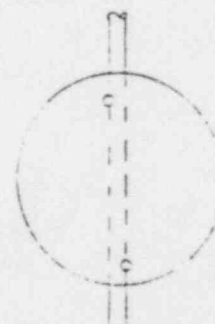
$$S_s = \frac{2(\%/100)T}{D_1 A_s} \quad \text{SHEAR STRESS (PSI)}$$

T = Torque due to LOCA

% = Percentage of torque taken up by the top pin. See the sketch below for the two different pinning configurations.



% = 65



% = 90

Figure 23

$A_s = .7854 \text{ adp}$  SHEAR AREA (IN<sup>2</sup>) along chordal length a

$a = .01745 D_1 \theta$  ARC. LENGTH (IN.)

$\theta = \sin^{-1} \left( \frac{2l}{D_1} \right)$

$l = \sqrt{dp \left( \frac{D_1}{2} \right) - \frac{dp^2}{4}}$  CHORDAL LENGTH

#### VALVE AND PIPING SECTION MODULUS:

$Z_v = \frac{\pi}{32 d_1} (d_1^4 - R_7^4)$  SECTION MODULUS OF VALVE (IN<sup>3</sup>)

$Z_p = \frac{\pi}{32 R_8} (R_8^4 - R_9^4)$  SECTION MODULUS OF ADJACENT PIPING (IN<sup>3</sup>)

$Z_v > 1.1 Z_p$

WHERE:  $d_1$  = VALVE BODY O.D. (IN.)

$R_7$  = VALVE BODY WATERWAY DIA. (IN.)

$R_8$  = O.D. OF ADJACENT PIPING (IN.)

$R_9$  = I.D. OF ADJACENT PIPING (IN.)

#### DEFLECTIONS:

$\Delta X = \frac{(W_1 + .5 W_2) g_{x,y}}{K_x}$

$g_{x,y} = \sqrt{G_1^2 + G_2^2}$

$g_{z,y} = \sqrt{G_3^2 + G_2^2}$

$\Delta Z = \frac{(W_1 + .5 W_2) g_{z,y}}{K_z}$

$K_{x,z}$  = Springrate (lbs/in)

#### VALVE BODY BOLTING:

$F = \frac{4}{X_O N_3} \left[ \cos \frac{\pi}{N_3} \right] M_4$  MAXIMUM BOLT LOAD (LBS)

$S_T = \frac{F}{A_4}$  TENSILE STRESS (PSI)

WHERE:  $M_4$  = MAXIMUM PIPING MOMENT (IN-LBS)

$N_3$  = NO. OF BODY BOLTS

$X_O = D_{BC}$  OF BODY BOLTS (IN)

$A_4$  = ROOT AREA OF BODY BOLTS (IN<sup>2</sup>)

NOTE: NOT APPLICABLE IF BODY IS WAFER STYLE OR THE ALLOWABLE MATERIAL STRENGTH OF VALVE EXCEEDS THAT OF ADJACENT PIPING, OR IF THE VALVE SECTION MODULUS IS 1.1 TIMES GREATER THAN THAT OF THE ADJACENT PIPE AS DETERMINED ABOVE. PIPING MAT'L ASSUMED SA106 GR.B S ALLOW = 15,000

DISC STRESSES

Shown on Pages 48D and E are the equations Posi-Seal uses for analyzing the disc when subject to design pressure.

For the purpose of this report the disc will be analyzed for two cases. For the first case, stresses will be calculated due to the LOCA flow acting on the disc, the pressure drop due to the LOCA flow acting across the disc and due to seismic loads. The second case will analyze the disc stresses due to the full pressure drop acting across the disc in the shut position and due to seismic loads.

Case 1

Moments about the X - axis

A. LOCA

$$M_{XL} = T_{LOCA} \quad (\text{LOCA Torque})$$

B. Pressure Drop

To determine  $M_{XP}$  for this case the flow pressure drop is substituted into the equation for  $M_X$  on Page 48D.

C. Seismic

$$F_S = W_4 \sqrt{G_1^2 + G_3^2} / 2 \quad \text{per side}$$

$$M_{XS} = \frac{4 D F_S}{3\pi}$$



$$\sum M_X = M_X = M_{XL} + M_{XP} + M_{XS}$$

To determine the stress  $M_X$  is substituted into the equation for  $\sigma_X$  on Page 48D.

Moments about the Y - axis

A. LOCA

For this analysis it will be assumed that the bearing reaction is equal to half the LOCA torque divided by the distance from the disc centerline to the centroid. The resulting  $M_Y$  is as follows:

$$M_{YL} = \frac{1}{2} \frac{T_{LOCA}}{\frac{2D}{3\pi}} \left[ \frac{D}{2} + L_2 + \frac{L_1}{3} \right]$$

B. Pressure Drop

$M_{XP}$  is determined by substituting the flow pressure drop into the equation for  $M_Y$  on Page 48E.

C. Seismic

$$M_{YS} = F_S \left[ \frac{D}{2} + L_2 + \frac{L_1}{3} \right]$$

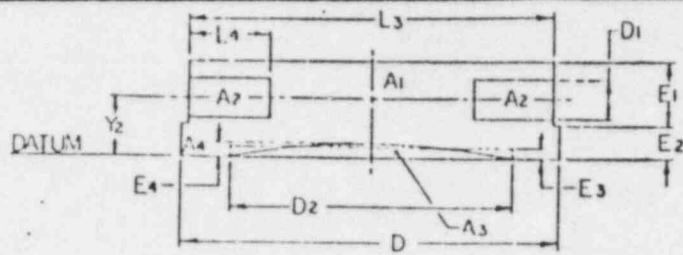
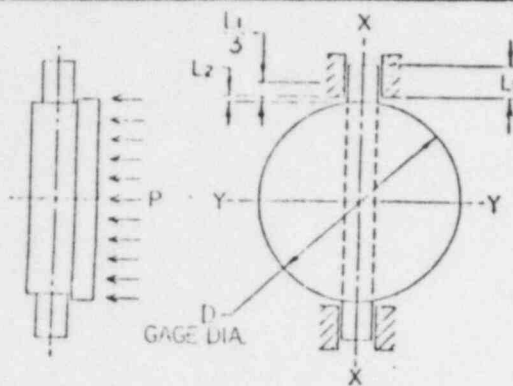
$$\sum M_Y = M_Y = M_{YL} + M_{YP} + M_{YS}$$

The stress due to  $M_Y$  is calculated using the equation for  $\sigma_Y$  on Page 48E.

The combined stress is then calculated using the equation for  $\sigma_C$  on Page 48E.

### Case 2

This case is the same as Case 1 except the moments due to flow are not determined and the pressure used in the equations for  $M_X$  and  $M_Y$  is the pressure when the disc is closed.



NOTE 1: FOR EASE OF CALCULATING  $A_3$  ASSUME AN EQUIVALENT RECTANGLE IN LIEU OF CIRCLE SEGMENT.  
 $E_3 = 0.667 E_4 = 0.667 ( ) = \text{IN}$   
 NOTE 2 FOR THRU SHAFT ONLY USE  $\frac{1}{2} L_3$  FOR  $L_4$ .

D = \_\_\_\_\_ IN.  
 D1 = \_\_\_\_\_ IN.  
 D2 = \_\_\_\_\_ IN.  
 L2 = \_\_\_\_\_ IN.  
 L3 = \_\_\_\_\_ IN.  
 L4 = \_\_\_\_\_ IN.  
 E1 = \_\_\_\_\_ IN.  
 E2 = \_\_\_\_\_ IN.  
 E4 = \_\_\_\_\_ IN.  
 Y2 = \_\_\_\_\_ IN.

CALCULATION OF  $\bar{Y}_x$ :

$$A_1 = L_3 E_1 = ( ) ( ) \quad A_1 = \text{IN}^2$$

$$A_2 = 2 L_4 D_1 = 2 ( ) ( ) \quad A_2 = \text{IN}^2$$

$$A_3 = D_2 E_3 = ( ) ( ) \quad A_3 = \text{IN}^2$$

$$A_4 = D E_2 = ( ) ( ) \quad A_4 = \text{IN}^2$$

$$Y_1 = E_2 + 5 E_1 = ( ) + 5 ( ) \quad Y_1 = \text{IN}$$

$$Y_3 = 5 E_3 = 5 ( ) \quad Y_3 = \text{IN}$$

$$Y_4 = 5 E_2 = 5 ( ) \quad Y_4 = \text{IN}$$

$$\bar{Y}_x = \text{DISTANCE FROM DATUM TO NEUTRAL AXIS} = \frac{A_1 Y_1 - A_2 Y_2 - A_3 Y_3 + A_4 Y_4}{A_1 - A_2 - A_3 + A_4} = \frac{( ) - ( ) - ( ) + ( )}{( ) - ( ) - ( ) + ( )} \quad \bar{Y}_x = \text{IN}$$

CALCULATION OF  $I_x$ :

$$I_x = \frac{L_3 E_1^3}{12} + A_1 (\bar{Y}_x - Y_1)^2 - \frac{2 L_4 D_1^3}{12} - A_2 (\bar{Y}_x - Y_2)^2 - \frac{D_2 E_3^3}{12} - A_3 (\bar{Y}_x - Y_3)^2 + \frac{D E_2^3}{12} + A_4 (\bar{Y}_x - Y_4)^2$$

$$I_x = \frac{( )^3}{12} + ( ) - \frac{( )^3}{12} - ( ) - \frac{( )^3}{12} - ( ) + \frac{( )^3}{12} + ( )$$

$$I_x = \text{IN}^4$$

CALCULATION OF  $M_x$ :

$$M_x = \frac{0.2122 \pi D^3 P}{8} = \frac{0.2122 \pi ( )^3 ( )}{8}$$

$$M_x = \text{IN-LBS.}$$

CALCULATION OF  $\sigma_x$ :

$$\sigma_x = \frac{M_x (E_1 + E_2 - \bar{Y}_x)}{I_x} = \frac{( ) + ( ) - ( )}{( )}$$

$$\sigma_x = \text{PSI}$$

DISC MATERIAL \_\_\_\_\_  
 DESIGN PRESSURE (P) \_\_\_\_\_  
 DESIGN TEMPERATURE \_\_\_\_\_  
 MIN. YIELD \_\_\_\_\_  
 ALLOWABLE STRESS \_\_\_\_\_

SHEET NO 1 OF 2

UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCE ON	DRAWN	DATE	POSI SEAL INTERNATIONAL INC. WESTERLY, R.I.	
DECIMALS $\pm .005$	CHECKED		TITLE	
FRACTIONS $\pm 1/64$	APPROVED		STRESS REPORT	
ANGLES $\pm 1/2^\circ$	APPROVED		DISC - LB - WCG	
DO NOT SCALE ENG BREAK ALL SHARP EDGES R25 TO 0.15 RAD			NOT SCALE	



DETAILED ANALYSISDetermination of LOCA Torques

Per Reference (g) when a LOCA occurs the pressure inside containment will increase to 50 psig and the temperature of 300 degrees F.

Since the make up of the media is not known, two conditions are analyzed for each of the three cases in order to insure that the largest aerodynamic torque is determined. The two conditions are air and steam.

The results of the LOCA analysis are given in Appendix B.

As can be seen in Appendix A, Cases 1 and 2, there is bend upstream of Valve VADPBC-09. Therefore, the effects of these bends on the aerodynamic torques during a LOCA have to be determined.

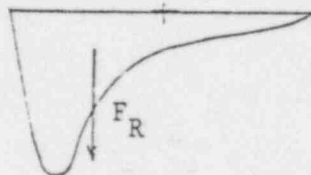
Posi-Seal has performed an investigation to determine what effect a pipe bend will have. To date this investigation has not revealed anything Posi-Seal can use with confidence. Consequently, Posi-Seal has taken the following approach:

1. Assume the flow results in a resultant load acting at the midpoint between the edge of the disc and the stem center line.
2. Determine the load based on the principles of impulse and momentum.
3. Calculate the resulting torque choosing the angle of attack which results in the worst case when added to the aerodynamic torque of the normal flow thru the valve.

The assumption that the flow resultant acts at the midpoint between the edge of the disc and the stem center line with a bend upstream is based on the following:

1. For flow through a straight pipe the largest aerodynamic torque will occur at 80 degrees open for flow in the preferred direction and 90 degrees for flow in the nonpreferred direction. This torque is predominately due to the lift and drag forces acting on the disc. Only a small amount of torque is due to the flow impacting on the disc since the moment arm is relatively small.
2. It is assumed that the resultant of the flow impacts the disc at the midpoint between the disc edge and the stem center line throughout the closure from 90 degrees open to fully closed. Although this may not be conservative at the smaller angles, it will result in very conservative results at 80 and 90 degrees where the maximum torques due to the lift and drag forces occur. See the diagram below.

Figure 24



Velocity profile with resultant acting at midpoint between disc edge and stem  $\bar{C}$



Valve at 80 degrees



Velocity profile acting on the disc at 80 degrees and the corresponding resultant.

Posi-Seal uses the full value of  $F_R$  throughout the valve closure.

The magnitude of the force impacting the disc is determined by using the principles of impulse and momentum as follows:

$$\Delta M V_1 + F \Delta t = \Delta M V_2$$

$$V_2 = 0$$

$$\Delta M V_1 = F \Delta t$$

where  $\Delta M$  = Change in mass - lbm

$V$  = Velocity - ft/sec

$F$  = Force - lbs

$\Delta t$  = Change in time

$$\Delta M = \frac{A \rho V \Delta t}{g}$$

where  $A$  = Area - ft<sup>2</sup>

$\rho$  = Density - lbm/ft<sup>3</sup>

$g$  = Gravitational Constant

$W = A \rho V$  = Flow - lbs/sec

$$\frac{W V_1 \Delta t}{g} = F \Delta t$$

$$F = \frac{W V_1}{g}$$

$$W = \frac{.0764 Q G}{3600}$$

where  $Q$  = Flow - SCFH

$$F = \frac{.0764 Q V}{32.2 (3600)} = \frac{Q V G}{1.517 \times 10^6}$$



DETERMINATION OF AERODYNAMIC TORQUES RESULTING FROM A BEND

The orientation of Valve VADPBC-09 relative to the end is as shown below. Flow is in the nonpreferred direction.

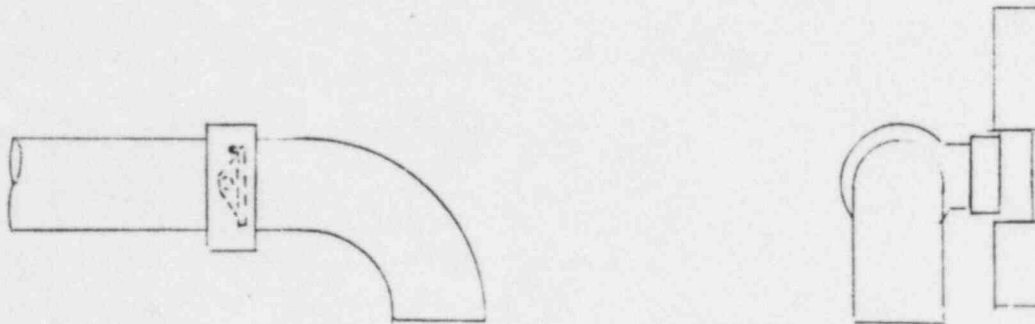


Figure 25

Since Case 2A resulted in much larger flows and aerodynamic torques than did Case 1A just Case 2A will be investigated.

The determination of the flow resultant force for both air and steam is as follows:

Air (G = 1.0)

<u>Degree</u>	<u>Q</u>	<u>V</u>	<u>F</u>
(Per Appendix B, Case 2A)			
90	8,549,000	528	2976
80	8,549,000	558	3144
70	7,760,000	534	2732
60	6,131,000	548	2214
50	4,522,000	512	1526
40	3,042,000	401	804
30	1,860,000	266	495
20	942,000	139	86
10	476,000	71	22



Steam (G = .62)

<u>Degree</u>	<u>Q</u>	<u>V</u>	<u>E</u>
(Per Appendix B, Case 2A, Cond. 1)			
90	10,140,000	627	2598
80	10,140,000	658	2727
70	9,345,000	634	2421
60	7,548,000	666	2055
50	5,578,000	642	1464
40	3,762,000	516	793
30	2,303,000	346	327
20	1,166,000	182	87
10	590,000	93	22

Since the air resulted in higher forces just the resulting torques from the air are calculated.

R = Radius corrected for the component acting normal to the disc at the worst angle of attack.

$$R = \frac{D}{4} \cos(45-\theta) = \frac{16.659}{4} \cos(45-\theta) = 4.16 \cos(45-\theta)$$

See Figure 26 on Page 57 for the determination that the angle of 45 degrees represents the worst case.

<u>Degree</u>	<u>R</u>	<u>T<sub>Bend</sub></u>	<u>T<sub>LOCA</sub></u>	<u>T<sub>T</sub></u>
(Per Appendix B)				
90	2.94	- 8761	15640	6879
80	3.41	-10720	1620	- 9100
70	3.77	-10310	- 3035	-13350
60	4.02	- 8903	- 549	- 9452
50	4.15	- 6329	1566	- 4763
40	4.15	- 3335	2190	- 1145
30	4.02	- 1991	2390	399
20	3.77	- 324	2470	2146
10	3.41	- 75	2485	2410

DETERMINATION OF CLOSING TIMES

Based on the aerodynamic torques determined in Appendix B and the preceding pages, the closing times are calculated using the computer programs "FLOW-CL" and "FLOW-CL1". The results of these calculations are shown in Appendix C.

As can be seen on Pages C-2 Valve VADPBC-09 will not close when subject to a LOCA in the full open position. When all the valves are restricted to 65 degrees open, the closing times are less than the closing times for no flow and less than the required time of 5.0 seconds.

Shown in Appendix D is a comparison of the calculated closing time to that actually measured.

The measured times which were recorded prior to shipment of the valves were as follows:

<u>PSI Item No.</u>	<u>Tag No.</u>	<u>Closing Time</u> (Sec)
14758-2A	CP1-VADPBC-09	1.5
2B	10	1.0
14759-2A	CP2-VADPBC-09	2.5
2B	10	2.5

Posi-Seal believes the difference in closing times between those for Unit 1 and those for Unit 2 to be attributed to the difference in quick exhaust valves. This belief is based on Matryx Certified Actuator Drawings not calling out the specific make or model of the quick exhausts.

Posi-Seal, in performing the closing time analysis, assumed the quick exhausts to be Parker-Hannifin OR-37s. As can be seen in Appendix D the calculated time was approximately the same as for Valve CP1-VADPBC-09. Posi-Seal felt justified in using the  $C_v$  for two OR-37s, since, for the case where flow is resisting closure if the valve could not meet the required closing time with this  $C_v$ , it could not meet it with a smaller  $C_v$ .

As it turns out, with restricting the valve opening to 65 degrees, the LOCA flow will assist closure, thus the closing times will be less than for the no flow condition.

### SEISMIC ANALYSIS

Since the fundamental natural frequency of the subject valves is less than 33 Hz, the input accelerations are taken from Reference (h). Because the valves are located at an elevation of 879'6" and the response accelerations at 905.75' are greater than those at 860', the maximum response accelerations at 905.75' are used for the seismic analysis. These accelerations are 3.2 Gs in the vertical direction and 2.6 in the horizontal direction.

For Valve VADPEC-10 which has its actuator mounted on the bottom, one G is added to the vertical acceleration input to account for the actuator dead weight.

Since the seismic analysis is based on the vertical axis (y) being the axis in which the weight of the actuator acts and the transverse axis (x) being perpendicular to the pipeline, the relative orientation for Valve VADPEC-09 has to be changed for the analysis. The reason being, the actuator for this valve is mounted on the side. See Figure 25 and Page 52.

With the actuator on the side, for the purpose of the analysis the vertical direction becomes the transverse direction and vice versa. As a consequence, one G is added to the transverse acceleration for the actuator dead weight.

It should be noted that 10 lbs. has been added to the actuator weight to take into consideration the weight of the internal stop used to restrict the valve opening to 65 degrees.

Shown on Page E-3 of Appendix E is the seismic analysis for VADPBC-10 fully open with the largest aerodynamic torque calculated applied. As can be seen, the disc pin stress exceeds the allowable.

On Pages E-9 and E-15 are the seismic analyses for Valve VADPBC-10 and 09, respectively, with the amount of valve opening restricted to 65 degrees.

It should be noted that the occasional load is a 500 lb. load which acts in the vertical direction. Consequently, for valve VADPBC-10 which is installed upside down the vertical load acts in the vertical direction for the purpose of the analysis. However, for valve VADPBC-09 which is mounted such that the actuator is on the side, the vertical load acts in the transverse direction.

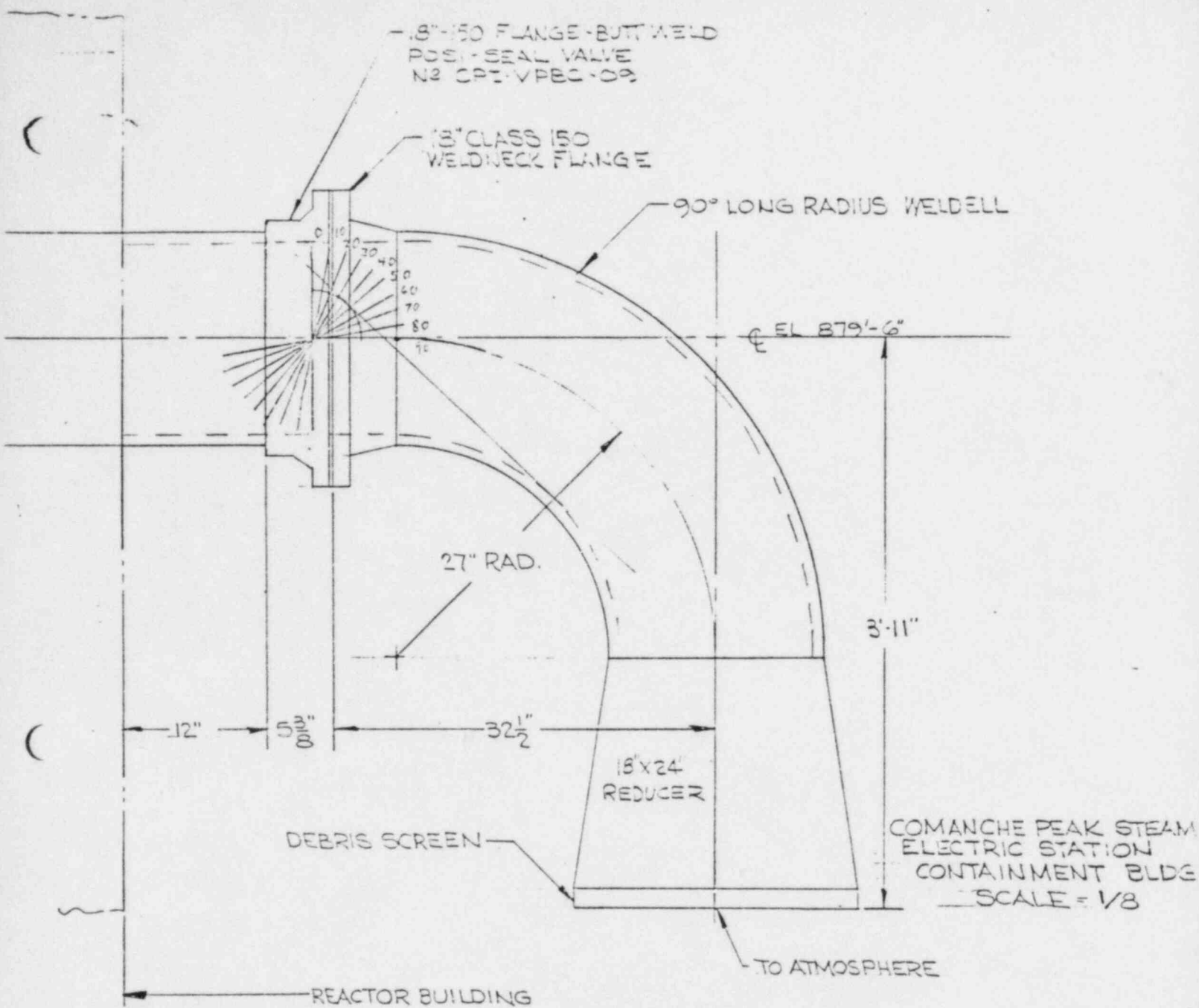


Figure 26

APPENDIX A

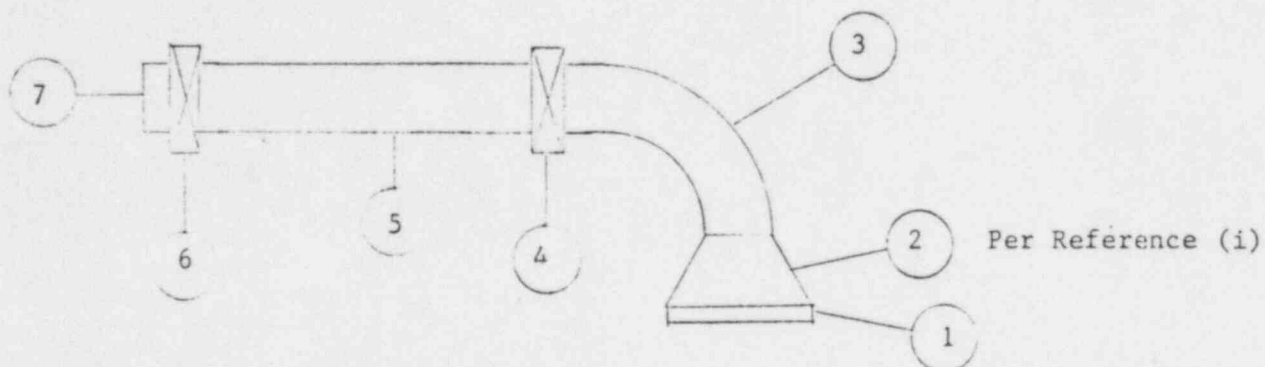
Schematics of the Piping Systems

CASE 1

18" - 150 Class Valves CP1-VADPBC-09 & 10

Piping outboard of Valve CP1-VADPBC-10 breaks off.

The resulting system is as shown below.



Station No.    Type of Resistance (No)

1	Entrance (1)	$K = .5$	$D_{in} = 24"$
2	Reducer (2)	$D_{in} = 24"$	$D_{out} = 18"$
3	90° Bend (5)	$K = 12$	$f_t = .14$
4	Valve (7) Flow in the Nonpreferred Direction	$C_v = 11053$	
5	Straight Pipe (4)	$L = 9.5'$	
6	Valve (7) Flow in the Preferred Direction	$C_v = 11053$	
7	Exit (8)	$K = 1.0$	$D_{out} = 18"$

Case 1 A Valve 9 cycles closed  
B Valve 10 cycles closed

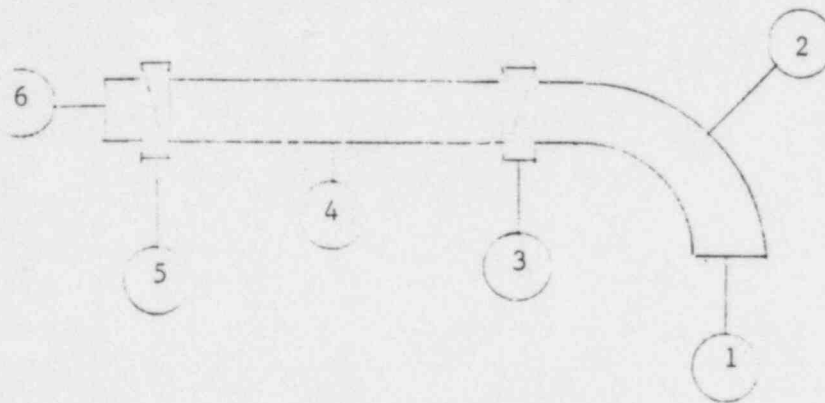


CASE 2

18" - 150 Class Valves CP1-VADPBC-09 & 10

Reducer in front of valve CP1-VADPBC-09 breaks off  
and piping outboard of Valve CP1-VADPBC-10 breaks off.

The resulting system is as shown below.



Station No.	Type of Resistance (No)		
1	Entrance (1)	$K = .5$	$D_{in} = 18"$
2	90° Bend (5)	$K = 12 f_t$	$= .14$
3	Valve (7) Flow in the Nonpreferred Direction	$C_v = 11053$	
4	Straight Pipe (4)	$L = 9.5'$	
5	Valve (7) Flow in the Preferred Direction	$C_v = 11053$	
6	Exit (8)	$K = 1.0$	$D_{out} = 18"$

Case 1 A Valve 9 cycles closed  
B Valve 10 cycles closed

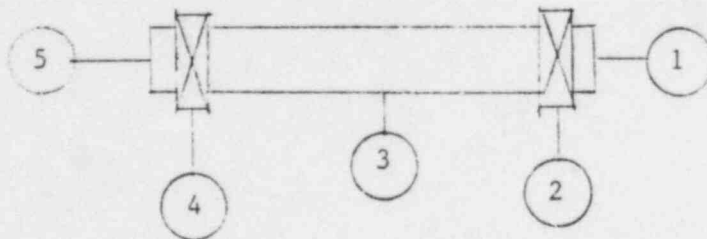


CASE 3

18" - 150 Class Valves CP1-VADPBC-09 & 10

Piping inboard of Valve CP1-VADPBC-09 and outboard of Valve CP1-VADPBC-10 breaks off.

The resulting system is as shown below.



Station No.	Type of Resistance (No)			
1	Entrance (1)	$K = .5$	$D_{in} = 18"$	
2	Valve (7) Flow in the Nonpreferred Direction	$C_v = 11053$		
3	Straight Pipe (4)	$L = 9.5'$		
4	Valve (7) Flow in the Preferred Direction	$C_v = 11053$		
5	Exit (8)	$K = 1.0$	$D_{out} = 18"$	

Case 1 A Valve 9 cycles closed  
B Valve 10 cycles closed

APPENDIX B

Determination of Flow Conditions

CASE 1A

CONDITION 1

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"

VALVE CLASS: 150

ACTUATOR: Matrox 20022 SR-80

UPSTREAM PRESSURE 64.7 PSIA

INITIAL TEMPERATURE 300 °F

SHUT OFF PRESSURE 64.7 PSIA

RATIO OF SP. HEAT 1.4

COMPRESSIBILITY 1

INITIAL DENSITY 1.230 LBS/FT<sup>3</sup>

FINAL PRESSURE 14.7 PSIA

MEDIA A-1

SPECIFIC GRAVITY 1

HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN. LBS  
PSI

STEM DIA. 1.75 IN.

PACKING TORQUE 1058 IN. LBS.

DIRECTION Non preferred

GAGE DIA. 16.654 IN

SEAL TORQUE 2398 IN. LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

VALVE SIZE=18

VALVE CLASS=150

FLOW-GAS

UPSTREAM PRESSURE	INITIAL DENSITY-X10 <sup>4</sup>	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE
64.7	23	300	14.7	64.7
MEDIA	RATIO OF SP. HEAT	SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR
AIR	1.4	1	1	3245
STE- VAL.	CAVE VAL.	PACKING TORQUE	SEAL TORQUE	
1.75	14.554	1000	2096	

STATION NO.	TYPE OF RESISTANCE	DIAMETER (IN)	LENGTH (FT)	RESISTANCE (IN)	CORRECTED RESISTANCE (IN)
1	ENTRANCE	24.0	0.0	0.500	0.50000
2	REDUCER	24.0	0.0	0.499	0.49985
3	PIPE BEND	18.0	0.0	0.140	0.44246
4	VALVE	18.0	0.0	0.713	2.42787
5	STRAIGHT PIPE	18.0	9.5	0.095	0.30024
6	VALVE	18.0	0.0	0.713	2.42787
7	EXIT	18.0	0.0	1.000	3.16049

## FLOW IN NONPREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN

FLOW 6,222,267 SCFH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	203.3
2	62.0	0.2275	298.7	205.1
3	57.0	0.2101	299.3	304.0
4	53.3	0.2002	293.8	410.7
5	49.2	0.1911	279.6	432.3
6	47.2	0.1837	274.2	448.7
7	43.3	0.1720	267.6	477.1
8	14.7	0.0752	193.6	1060.2

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 5,306 IN. LBS

DELTA P=50.00 PSI

# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	Totalling	
90	2,922,253	3.35	10,709	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	203.3
2	63.7	0.2275	298.7	205.1
3	57.0	0.2101	289.3	394.0
4	53.3	0.2000	283.9	412.7
5	49.9	0.1911	278.5	432.3
6	47.2	0.1837	274.2	448.7
7	43.3	0.1723	267.5	477.1
8	44.7	0.0735	193.3	1062.2

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
90	4,708,311	5.15	1,322	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	199.3
2	63.7	0.2275	298.7	201.0
3	57.3	0.2111	289.2	384.5
4	53.3	0.2017	284.6	401.3
5	49.7	0.1977	276.6	431.5
6	45.0	0.1804	272.2	448.0
7	42.2	0.1696	265.5	475.8
8	44.7	0.0725	194.2	1009.7

ANGLE	FLOW	DP ACROSS VALVE	Following	
90	5,193,333	8.20		1,715
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	199.7
2	61.9	0.2230	298.2	191.2
3	58.1	0.2131	290.2	364.1
4	57.7	0.2047	286.3	379.7
5	46.1	0.1905	272.3	429.5
6	43.6	0.1735	269.1	445.2
7	39.9	0.1630	261.4	474.5
8	44.7	0.0825	199.2	921.1

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
90	5,797,315	10.70	192	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	197.5
2	66.1	0.2225	299.2	189.3
3	59.8	0.2175	293.4	313.7
4	57.5	0.2117	287.5	321.8
5	32.7	0.1693	259.0	427.6
6	34.7	0.1535	255.2	442.9
7	33.3	0.1433	248.3	474.4
8	44.7	0.0799	195.5	846.4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing
90	4,392,393	32.07	1,460

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	128.7

3	82.0	0.2332	296.4	234.9
4	50.9	0.2203	294.8	237.8
5	28.8	0.1291	238.1	407.5
6	27.5	0.1252	235.2	420.1
7	25.7	0.1189	232.4	442.4
8	14.7	0.0793	196.4	559.5

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
40	1,907,636	41.93	2,093	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2330	300.0	20.0
2	84.6	0.2327	299.9	22.7
3	84.6	0.2323	298.6	157.9
4	84.6	0.2323	298.0	188.0
5	21.2	0.1039	218.3	745.3
6	20.7	0.1020	214.7	350.7
7	19.9	0.0991	214.2	363.2
8	14.7	0.0793	196.4	451.1

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
30	1,853,338	41.95	2,342	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2330	300.0	24.0
2	84.6	0.2327	299.9	24.0
3	84.6	0.2321	298.6	97.7
4	84.6	0.2323	298.0	97.7
5	17.2	0.0994	208.7	350.3
6	17.1	0.0980	208.0	250.1
7	15.7	0.0897	204.0	255.0
8	14.7	0.0793	196.4	201.1

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
20	947,527	40.15	2,457	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	27.8
2	84.6	0.2299	299.9	27.8
3	84.6	0.2298	299.9	49.4
4	84.6	0.2297	299.8	40.6
5	15.7	0.0827	199.0	137.1
6	15.7	0.0825	198.0	137.2
7	15.2	0.0812	198.6	138.0
8	14.7	0.0793	196.4	140.6

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
10	485,759	49.77	2,482	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	14.2
2	84.6	0.2299	299.9	14.2
3	84.6	0.2299	299.9	25.3

5	14.9	0.0805	197.2	72.3
6	14.8	0.0805	197.1	72.4
7	14.8	0.0803	198.9	72.6
8	14.7	0.0798	198.4	73.1

NOTE: THERE IS CHANGED FLOW AT STATION 4

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 1A

CONDITION 2

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"

VALVE CLASS: 150

ACTUATOR: Matryx 26072 SR-80

UPSTREAM PRESSURE 64.1 PSIA

INITIAL TEMPERATURE 300 °F

SHUT OFF PRESSURE 64.1 PSIA

RATIO OF SP. HEAT 1.329

COMPRESSIBILITY 1

INITIAL DENSITY .159 LBS/FT<sup>3</sup>

FINAL PRESSURE 17.7 PSIA

MEDIA Steam

SPECIFIC GRAVITY 1.62

HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN. LBS  
PSI

STEM DIA. 1.75 IN.

PACKING TORQUE 1058 IN. LBS.

DIRECTION Non preferred

GAGE DIA. 14.654 IN

SEAL TORQUE 2396 IN. LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)



# CONTROL SYSTEM ANALYSIS

Case 1A  
Cond 2

VALVE SIZE-18		VALVE CLASS-150		FLOW-GAS	
UPSTREAM PRESSURE	INITIAL DENSITY-X10 <sup>12</sup>	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE	
44.7	15.9	300	14.7	44.7	
MEDIA	RATIO OF SPEC. HEAT	SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR	
STEAM	1.329	.62	1	E90 DEG 3245	
SEAL	PACKING	SEAL	PACKING		
1.75	18.854	1059	2396		

STATION NO.	TYPE OF RESISTANCE	DIAMETER-IN	LENGTH-FT	RESISTANCE- $\Delta K$	CORRECTED RESISTANCE- $\Delta K$
1	ENTRANCE	24.0	0.0	0.500	0.50000
2	REDUCER	24.0	0.0	0.499	0.49989
3	PIPE BEND	18.0	0.0	0.140	0.44246
4	VALVE	18.0	0.0	0.769	2.42787
5	STRAIGHT PIPE	18.0	9.5	0.095	0.30024
6	VALVE	18.0	0.0	0.769	2.42787
7	EXIT	18.0	0.0	1.000	3.16049

## FLOW IN NONPREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN

FLOW= 8,157,471 GPH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	44.7	0.1590	300.0	239.6
2	50.1	0.1571	299.9	261.9
3	56.9	0.1442	290.6	457.2
4	57.0	0.1367	285.6	491.1
5	50.1	0.1313	281.7	512.1
6	47.4	0.1259	277.8	532.4
7	44.7	0.1203	272.9	562.7
8	14.7	0.0521	207.8	1292.4

NOTE: THERE IS CHOKED FLOW AT STATION 7

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 5,306 IN. LBS

DELTA P=50.00 PSI

# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
90	8,159,671	2.87	10,351	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	239.6
2	63.7	0.1571	299.8	241.9
3	58.8	0.1442	290.5	467.2
4	53.7	0.1369	285.3	491.1
5	50.1	0.1313	281.7	512.1
6	47.4	0.1259	277.8	532.4
7	44.7	0.1203	272.9	562.7
8	14.7	0.0521	207.8	1299.4

NOTE: THERE IS CHOKED FLOW AT STATION 7

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
80	8,159,671	4.55	1,320	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	239.6
2	63.7	0.1571	298.8	241.9
3	58.8	0.1442	290.5	467.2
4	53.8	0.1369	285.3	491.1
5	48.3	0.1277	279.1	526.3
6	45.8	0.1223	275.2	547.7
7	44.7	0.1203	272.9	562.7
8	14.7	0.0521	207.8	1299.4

NOTE: THERE IS CHOKED FLOW AT STATION 7

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
70	8,159,671	9.50	2,153	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	239.6
2	63.7	0.1571	298.8	241.9
3	58.8	0.1442	290.5	467.2
4	53.8	0.1369	285.3	491.1
5	48.5	0.1180	271.9	569.0
6	40.4	0.1117	267.1	600.3
7	44.7	0.1203	260.2	562.7
8	14.7	0.0521	207.8	1299.4

NOTE: THERE IS CHOKED FLOW AT STATION 7

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	8,900,609	17.02	274	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	204.3
2	64.1	0.1580	299.4	204.9
3	59.4	0.1492	293.8	385.0
4	57.1	0.1447	290.8	395.3
5	40.0	0.1107	266.3	517.0
6	37.9	0.1084	262.8	536.4
7	34.9	0.1000	257.6	570.5
8	14.7	0.0517	207.3	1096.5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	5,397,553	30.84	1,397	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	153.0
2	84.4	0.1588	299.7	153.0
3	81.8	0.1537	296.7	289.2
4	80.2	0.1514	295.2	293.0
5	29.7	0.0887	247.5	503.6
6	28.5	0.0858	244.9	520.5
7	26.8	0.0819	241.2	544.9
8	14.7	0.0521	207.8	957.0

NOTE: THERE IS CHOKED FLOW AT STATION 1

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	5,711,524	41.25	2,058	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	109.2
2	84.5	0.1588	299.8	107.2
3	63.8	0.1569	298.7	196.0
4	55.1	0.1550	293.1	196.6
5	21.8	0.0703	229.3	439.1
6	21.2	0.0683	227.8	448.2
7	20.5	0.0670	225.7	460.7
8	14.7	0.0521	207.2	592.2

NOTE: THERE IS CHOKED FLOW AT STATION 1

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	2,583,105	46.81	2,522	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	67.5
2	84.5	0.1587	299.7	67.5
3	84.4	0.1585	299.7	120.1
4	84.3	0.1583	299.5	120.1
5	17.5	0.0595	217.1	320.9
6	17.2	0.0587	216.4	324.1
7	16.9	0.0581	216.4	320.5
8	14.7	0.0521	207.8	336.3

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	1,172,275	49.07	2,656	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	34.6
2	84.5	0.1587	299.7	34.6
3	84.4	0.1588	299.9	61.5
4	84.3	0.1588	299.8	51.5
5	15.5	0.0543	210.7	180.2
6	15.4	0.0541	210.5	180.7
7	15.3	0.0537	210.0	182.0
8	14.7	0.0521	207.8	187.8

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
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	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	17.7
2	64.6	0.1589	299.9	17.7
3	64.6	0.1589	299.9	31.5
4	64.6	0.1589	297.9	31.5
5	14.9	0.0527	208.6	95.3
6	14.9	0.0526	208.5	95.1
7	14.8	0.0525	208.4	95.5
8	14.7	0.0521	207.8	96.3

NOTE: THERE IS CROWNED FLOW AT STATION 4

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 1B

CONDITION 1

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"  
VALVE CLASS: 150  
ACTUATOR: Matrox 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 300 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.4  
COMPRESSIBILITY 1

INITIAL DENSITY 1.230 LBS/FT<sup>3</sup>  
FINAL PRESSURE 14.7 PSIA  
MEDIA Air  
SPECIFIC GRAVITY 1  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3275 IN.LBS  
PSI

STEM DIA. 1.75 IN.  
PACKING TORQUE 1058 IN.LBS.  
DIRECTION Preferred

GAGE DIA. 16.654 IN  
SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

VALVE SIZE=12

VALVE CLASS=150

FLOW-GAS

UPSTREAM PRESSURE	INITIAL DENSITY-X10 <sup>4</sup>	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE
64.7	23	300	14.7	64.7
MEDIA	RATIO OF SPEC. HEAT	SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR
AIR	1.4	1	1	3245
STEM DIA.	GASK DIA.	PACKING TORQUE	SEAL TORQUE	
1.75	12.454	1050	2375	

STATION NO.	TYPE OF RESISTANCE	DIAMETER-(D)	LENGTH-(L)	RESISTANCE-(K)	CORRECTED RESISTANCE-(K)
1	ENTRANCE	24.0	0.0	0.500	0.50000
2	REDUCER	24.0	0.0	0.499	0.49985
3	PIPE BEND	18.0	0.0	0.140	0.44246
4	VALVE	18.0	0.0	0.763	2.42787
5	STRAIGHT PIPE	12.0	9.5	0.095	0.30024
6	VALVE	18.0	0.0	0.763	2.42787
7	EXIT	18.0	0.0	1.000	3.16049

## FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN

FLOW=6,922.253 GPM

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	203.3
2	57.0	0.2291	299.7	201.1
3	57.0	0.2191	299.3	394.0
4	55.0	0.2002	293.3	412.7
5	49.2	0.1911	279.8	432.3
6	47.0	0.1837	274.0	448.7
7	43.3	0.1728	267.6	477.1
8	14.7	0.0780	193.0	1086.1

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 5,306 IN. LBS  
DELTA P=50.00 PSI

# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
70	6,327.53	3.89	- 9,351	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	203.3
2	83.7	0.2275	299.7	205.1
3	82.0	0.2101	289.3	394.0
4	83.3	0.2002	283.8	412.1
5	80.9	0.1911	270.5	432.3
6	81.2	0.1837	274.2	448.7
7	81.1	0.1720	267.0	477.1
8	81.7	0.1768	198.5	1044.2

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
80	5,784,511	5.06	11,762	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	199.3
2	83.7	0.2275	299.7	201.0
3	82.0	0.2111	289.9	384.5
4	83.3	0.2017	284.8	401.5
5	80.9	0.1930	279.7	419.4
6	81.2	0.1855	275.7	434.0
7	81.1	0.1692	265.3	477.5
8	81.7	0.1703	198.5	1017.7

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
70	6,422,833	10.91	7,472	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	190.7
2	83.7	0.2275	299.7	191.9
3	82.0	0.2131	290.9	364.4
4	83.2	0.2047	286.3	378.7
5	80.9	0.1972	282.1	393.1
6	82.2	0.1912	280.6	406.4
7	79.1	0.1607	259.9	401.3
8	81.7	0.1730	198.4	1001.5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
60	5,532,750	22.26	2,417	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	162.5
2	84.1	0.2230	297.2	163.2
3	80.1	0.2132	291.7	303.4
4	80.8	0.2122	290.8	310.5
5	86.2	0.2079	283.1	317.8
6	81.7	0.2041	280.0	323.2
7	80.4	0.1406	246.4	472.4
8	81.7	0.0798	198.4	832.5

NOTE: THERE IS CHECKED FLOW AT STATION 8

ANGLE	FLOW	DP ACROSS VALVE	Tclosing
50	4,277,297	34.21	592



1	64.7	0.2300	300.0	125.6
2	64.5	0.2293	299.7	125.6
3	62.3	0.2237	298.6	220.9
4	61.1	0.2209	295.1	221.4
5	60.0	0.2181	293.7	234.2
6	59.8	0.2173	292.7	235.0
7	25.1	0.1171	229.0	439.4
8	14.7	0.0799	196.4	643.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
40	2,970,115	42.82	1,767	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	87.0
2	64.5	0.2293	299.7	87.0
3	62.3	0.2237	298.6	156.4
4	61.1	0.2209	295.1	156.8
5	60.0	0.2241	297.4	157.6
6	59.8	0.2247	297.2	157.6
7	19.8	0.0987	213.9	361.0
8	14.7	0.0799	196.4	447.0

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
30	1,022,037	47.00	1,400	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	54.3
2	64.5	0.2293	299.7	54.3
3	61.4	0.2224	299.6	96.7
4	61.1	0.2221	299.5	95.7
5	61.1	0.2227	299.3	94.9
6	64.1	0.2225	299.2	95.7
7	16.7	0.0875	203.8	253.7
8	14.7	0.0799	196.4	270.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
20	937,542	49.31	2,449	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	27.5
2	64.5	0.2293	299.7	27.5
3	61.4	0.2228	299.2	49.0
4	61.1	0.2227	299.0	49.0
5	61.1	0.2226	299.2	49.0
6	64.6	0.2226	299.0	49.0
7	15.2	0.0813	198.4	137.0
8	14.7	0.0799	196.4	141.3

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
10	481,745	49.81	2,482	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	14.1



3	64.5	0.2299	299.9	25.1
4	64.5	0.2299	299.9	25.1
5	64.5	0.2299	299.9	25.1
6	64.5	0.2299	299.9	25.1
7	14.2	0.0803	196.9	72.0
8	14.7	0.0798	195.4	72.5

NOTE: THERE IS CHOKED FLOW AT STATION 5

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE Case 1B

CONDITION 2

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"  
VALVE CLASS: 150  
ACTUATOR: Mot-yx 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 200 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.329  
COMPRESSIBILITY 1

INITIAL DENSITY 1.159 LBS/FT<sup>3</sup>  
FINAL PRESSURE 14.7 PSIA  
MEDIA Steam  
SPECIFIC GRAVITY 1.62  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN.LBS  
PSI

STEM DIA. 1.75 IN.  
PACKING TORQUE 1058 IN.LBS.  
DIRECTION Prefered

GAGE DIA. 16.654 IN  
SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

Case 1B  
Cond 2

# CONTROL SYSTEM ANALYSIS

VALVE DESIGN		VALVE CLASSIFICATION		FLOW DATA	
UPSTREAM PRESSURE	INITIAL DENSITY X 10 <sup>42</sup>	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE	
54.7	15.7	100	14.7	54.7	
MEDIA	RATIO OF SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR		
WATER	1.0	1.0	1.0		

STATION NO.	TYPE OF RESISTANCE	DIAMETER-INS	LENGTH-FT	RESISTANCE-INS	RESISTANCE-INS
1	ENTRANCE	24.0	0.0	0.000	0.00000
2	REDUCED	24.0	0.0	0.000	0.00000
3	PIPE BEND	18.0	0.0	0.140	0.44015
4	VALVE	18.0	0.0	0.000	0.00000
5	STRAIGHT PIPE	18.0	0.0	0.000	0.30004
6	VALVE	18.0	0.0	0.000	0.00000
7	EXIT	18.0	0.0	1.000	3.14159

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN  
FLOW = 5.000 GPM

STATION NO.	PRESSURE	DENSITY	TEMPERATURE	VALVE POS.
1	54.7	0.1590	100.0	200.0
2	53.0	0.1591	200.0	200.0
3	52.0	0.1592	200.0	200.0
4	53.0	0.1592	200.0	200.0
5	50.1	0.1593	200.0	200.0
6	47.4	0.1597	200.0	200.0
7	44.7	0.1603	200.0	200.0
8	14.7	0.1601	200.0	200.0

NOTE: THERE IS NO FLOW AT STATION 7

CONDITIONS WITH VALVE SHUT

VALVE TORQUE = 5.306 IN. LBS  
SEAL PRESSURE PSI

# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	INLET P	
30	3,157,671	2.77	11,197	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	239.6
2	83.7	0.1571	298.0	241.0
3	82.8	0.1553	296.5	457.0
4	82.2	0.1537	295.5	491.1
5	80.1	0.1513	291.7	510.1
6	77.6	0.1488	287.0	557.2
7	44.7	0.1203	254.5	562.7
8	14.7	0.0521	207.8	1051.8

NOTE: THERE IS CAVITATION FROM AT STATION 7

ANGLE	FLOW	DP ACROSS VALVE	Tolising	
30	3,159,671	2.77	= 11,197	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	239.6
2	83.7	0.1571	298.0	241.0
3	82.8	0.1553	296.5	457.0
4	82.2	0.1537	295.5	491.1
5	80.1	0.1513	291.7	510.1
6	77.6	0.1488	287.0	557.2
7	44.7	0.1203	254.5	562.7
8	14.7	0.0521	207.8	1051.8

NOTE: THERE IS CAVITATION FROM AT STATION 7

ANGLE	FLOW	DP ACROSS VALVE	INLET P	
30	3,157,671	2.77	11,197	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	239.6
2	83.7	0.1571	298.0	241.0
3	82.8	0.1553	296.5	457.0
4	82.2	0.1537	295.5	491.1
5	80.1	0.1513	291.7	510.1
6	77.6	0.1488	287.0	557.2
7	44.7	0.1203	254.5	562.7
8	14.7	0.0521	207.8	1051.8

NOTE: THERE IS CAVITATION FROM AT STATION 7

ANGLE	FLOW	DP ACROSS VALVE	INLET P	
30	3,157,671	2.77	11,197	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	300.0	199.4
2	84.1	0.1580	299.4	203.1
3	59.7	0.1497	294.1	374.7
4	57.4	0.1453	291.2	388.0
5	55.6	0.1420	289.0	394.1
6	54.1	0.1391	287.0	401.2
7	31.4	0.1011	267.0	557.2
8	14.7	0.0521	207.8	1051.8

ANGLE 50 FLOW 1.0000 DP APPROX VALUE 31.00 TOLERANCE 4.4

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	188.0
2	64.7	0.1588	300.0	188.0
3	64.7	0.1581	300.0	188.0
4	64.7	0.1518	300.0	188.0
5	64.7	0.1500	300.0	188.0
6	64.7	0.1500	300.0	188.0
7	64.7	0.1500	300.0	188.0
8	64.7	0.1500	300.0	188.0

NOTE: THERE IS CHECKED FLOW AT STATION 4

ANGLE 40 FLOW 1.0000 DP APPROX VALUE 10.10 TOLERANCE 1.347

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	107.0
2	64.7	0.1588	300.0	107.0
3	64.7	0.1589	300.0	107.0
4	64.7	0.1581	300.0	107.0
5	64.7	0.1518	300.0	107.0
6	64.7	0.1500	300.0	107.0
7	64.7	0.1500	300.0	107.0
8	64.7	0.1500	300.0	107.0

NOTE: THERE IS CHECKED FLOW AT STATION 4

ANGLE 30 FLOW 1.0000 DP APPROX VALUE 07.00 TOLERANCE 0.7

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	67.0
2	64.7	0.1588	300.0	67.0
3	64.7	0.1589	300.0	67.0
4	64.7	0.1581	300.0	67.0
5	64.7	0.1518	300.0	67.0
6	64.7	0.1500	300.0	67.0
7	64.7	0.1500	300.0	67.0
8	64.7	0.1500	300.0	67.0

NOTE: THERE IS CHECKED FLOW AT STATION 4

ANGLE 20 FLOW 1.0000 DP APPROX VALUE 40.00 TOLERANCE 2.143

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	34.5
2	64.7	0.1588	300.0	34.5
3	64.7	0.1589	300.0	34.5
4	64.7	0.1581	300.0	34.5
5	64.7	0.1518	300.0	34.5
6	64.7	0.1500	300.0	34.5
7	64.7	0.1500	300.0	34.5
8	64.7	0.1500	300.0	34.5

NOTE: THERE IS CHECKED FLOW AT STATION 4



CASE 2ACONDITION 1NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"  
VALVE CLASS: 150  
ACTUATOR: Matryx 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 300 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.4  
COMPRESSIBILITY 1

INITIAL DENSITY .230 LBS/FT<sup>3</sup>  
FINAL PRESSURE 14.7 PSIA  
MEDIA Air  
SPECIFIC GRAVITY 1  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3275 IN.LBS  
PSI

STEM DIA. 1.75 IN.  
PACKING TORQUE 1058 IN.LBS.  
DIRECTION Non preferred

GAGE DIA. 16.654 IN  
SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

CONTROL SYSTEM ANALYSIS

UPSTREAM PRESSURE	VALVE SIZE IN	INITIAL DENSITY-10 <sup>3</sup> LB/FT <sup>3</sup>	VALVE CLASSIFIED	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE
44.7		23		300	14.7	44.7
MEDIA	RATIO OF FLOW	SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR		
AIR	1.0	1	1	0.0001		
WATER	0.001	62.4	0.0001	0.0001		
STEAM	0.001	1000	0.0001	0.0001		

STATION NO.	TYPE OF RESISTANCE	DIAMETER IN	LENGTH-FT	RESISTANCE IN	ELASTICITY IN
1	CHANGE	10.0	0.0	0.000	0.00000
2	PIPE BEND	10.0	0.0	0.100	0.10000
3	VALVE	10.0	0.0	0.700	0.70000
4	STRAIGHT PIPE	10.0	0.0	0.000	0.00000
5	VALVE	10.0	0.0	0.700	0.70000
6	EXIT	10.0	0.0	1.000	1.00000

FLOW IN NONPREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN

FLOW 5.5 GPM

STATION	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	44.7	0.0001	300.0	0.000
2	50.0	0.0001	290.0	0.000
3	50.0	0.0001	290.0	0.000
4	50.0	0.0001	290.0	0.000
5	50.0	0.0001	290.0	0.000
6	44.7	0.0001	300.0	0.000
7	14.7	0.0001	196.4	1000.0

NOTE: THERE IS NO FLOW AT STATION

CONDITIONS WITH VALVE SHUT

VALVE THROTTLED 5.00 IN. L21  
DELTA P=50.00 PSI



# CONDITIONS AS VALVE CLOSING

ANGLE	FLOW	OP. ABOVE VALVE	TELEGRAPH	
80	2,547,000	9.00	15,800	
1	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	56.7	0.2300	300.0	445.4
2	57.4	0.2126	299.7	482.0
3	58.0	0.2076	297.9	492.8
4	58.3	0.1937	298.1	507.7
5	47.7	0.1890	277.2	517.4
6	38.4	0.1793	247.3	526.1
7	14.7	0.0798	196.4	1127.3

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	OP. ABOVE VALVE	TEMPERATURE	
80	2,547,000	9.00	1,800	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	56.7	0.2300	300.0	445.4
2	57.4	0.2126	299.7	482.0
3	58.0	0.2076	297.9	492.8
4	58.3	0.1937	298.1	507.7
5	47.7	0.1890	277.2	517.4
6	38.4	0.1793	247.3	526.1
7	14.7	0.0798	196.4	1127.3

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	OP. ABOVE VALVE	TELEGRAPH	
80	2,547,000	9.00	15,800	
1	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	56.7	0.2300	300.0	445.4
2	57.4	0.2126	299.7	482.0
3	58.0	0.2076	297.9	492.8
4	58.3	0.1937	298.1	507.7
5	47.7	0.1890	277.2	517.4
6	38.4	0.1793	247.3	526.1
7	14.7	0.0798	196.4	1127.3

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	OP. ABOVE VALVE	TELE. GAGE	
80	2,547,000	9.00	15,800	
-----				
1	PRESSURE	DENSITY	TEMPERATURE	
1	56.7	0.2300	300.0	445.4
2	57.4	0.2126	299.7	482.0
3	58.0	0.2076	297.9	492.8
4	58.3	0.1937	298.1	507.7
5	47.7	0.1890	277.2	517.4
6	38.4	0.1793	247.3	526.1
7	14.7	0.0798	196.4	1127.3

NOTE: THERE IS CHOKED FLOW AT STATION 3

50 4,522,440 41.10 1,500

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.02300	200.0	225.1
2	53.8	0.02299	200.1	225.2
3	53.0	0.02297	200.1	225.3
4	51.2	0.02281	200.1	211.3
5	51.0	0.02280	200.1	211.3
6	49.5	0.02279	200.1	214.4
7	48.7	0.02278	198.4	200.5

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE FLOW DP ACROSS VALVE TEMPERATURE  
10 1,200,000 49.10 2,190

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.02300	200.0	225.1
2	54.1	0.02298	200.0	227.5
3	54.0	0.02298	200.1	229.5
4	47.7	0.02211	207.1	400.7
5	47.5	0.02206	206.5	403.9
6	46.1	0.02202	203.5	410.6
7	46.7	0.02298	198.4	457.8

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE FLOW DP ACROSS VALVE TEMPERATURE  
10 1,200,000 49.10 2,190

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.02300	200.0	225.1
2	54.5	0.02297	200.0	225.1
3	54.0	0.02297	200.0	225.1
4	48.7	0.02200	200.1	248.0
5	48.7	0.02200	200.1	248.0
6	48.2	0.02204	199.8	270.9
7	48.7	0.02298	198.4	280.0

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE FLOW DP ACROSS VALVE TEMPERATURE  
10 940,018 49.10 2,470

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.02300	200.0	225.1
2	54.5	0.02299	200.0	225.1
3	54.1	0.02299	200.0	225.1
4	48.7	0.02211	207.1	400.7
5	48.5	0.02211	207.1	400.7
6	47.5	0.02202	203.5	410.6
7	48.7	0.02298	198.4	457.8

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE FLOW DP ACROSS VALVE TEMPERATURE  
10 476,281 49.89 2,485

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.02300	200.0	24.8
2	54.5	0.02299	200.0	24.8

1.1	1.1	0.0001	1.1	1.1
1.1	1.1	0.0001	1.1	1.1
1.1	1.1	0.0001	1.1	1.1
1.1	1.1	0.0001	1.1	1.1

NOTE: HERE IS TURNED FLOW AT STATION 1

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 2A

CONDITION 2

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"

VALVE CLASS: 150

ACTUATOR: Malcoy 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA

INITIAL TEMPERATURE 300 °F

SHUT OFF PRESSURE 64.7 PSIA

RATIO OF SP. HEAT 1.329

COMPRESSIBILITY 1

INITIAL DENSITY .159 LB3/FT<sup>3</sup>

FINAL PRESSURE 14.7 PSIA

MEDIA Steam

SPECIFIC GRAVITY .62

HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN.LBS  
PSI

STEM DIA. 1.75 IN.

PACKING TORQUE 1058 IN.LBS.

DIRECTION Non preferred

GAGE DIA. 10.654 IN

SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

Case 24  
C-2

# CONTROL SYSTEM ANALYSIS

VALVE TYPE-13		VALVE TYPE-13		FLOW-13	
UPSTREAM PRESSURE PSI	INITIAL DENSITY-1000 LB/FT <sup>3</sup>	INITIAL TEMP-DEGREES F	FINAL PRESSURE PSI	TEMP-DEGREES F	INITIAL PRESSURE PSI
14.0	62.0	300	14.0	300	14.0
MEDIA	WATER	WATER	WATER	WATER	WATER
STEAM	1.00	1.00	1.00	1.00	1.00
TEMP	300	300	300	300	300
VAL	1.00	1.00	1.00	1.00	1.00

STATION NO.	TYPE OF RESISTANCE	DIAMETER (IN)	LENGTH (FT)	RESISTANCE-1000	RESISTANCE-1000
1	ENTRANCE	10.0	0.0	0.000	0.00000
2	PIPE LOSS	10.0	0.0	0.100	0.10000
3	PIPE LOSS	10.0	0.0	0.100	0.10000
4	STRAIGHT PIPE	10.0	0.5	0.005	0.00500
5	VALVE	10.0	0.0	0.100	0.10000
6	EXIT	10.0	0.0	1.000	1.00000

## FLOW IN UNREFERENCED DIRECTION

CONDITIONS WITH VALVE OPEN

FLOW=10.00000 GPM

	PRESSURE PSI	DENSITY LB/FT <sup>3</sup>	TEMPERATURE DEGREES F	VELOCITY FT/SEC
1	14.0	62.0	300.0	0.000
2	13.9	62.0	300.0	0.000
3	13.8	62.0	300.0	0.000
4	13.7	62.0	300.0	0.000
5	13.6	62.0	300.0	0.000
6	13.5	62.0	300.0	0.000
7	13.4	62.0	300.0	0.000

NOTE: THERE IS CHANGED FLOW AT STATION 6

CONDITIONS WITH VALVE SHUT

VALVE TORQUE = 5.000 IN-FT

DELTA P=50.00 PSI

# CONDITIONS AS VALVE CLOSES

ANGLE FLOW DP ACROSS VALVE Telesing  
 00 10,1-1,045 7.35 15,571

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.1590	220.0	829.6
2	57.5	0.1486	221.1	872.0
3	58.7	0.1421	226.1	882.9
4	59.1	0.1331	231.1	877.2
5	59.7	0.1210	237.1	871.9
6	59.8	0.1204	243.0	897.0
7	59.7	0.1521	251.0	1115.0

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Telesing  
 00 10,1-1,045 7.35 15,571

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.1590	220.0	829.6
2	57.5	0.1486	221.1	872.0
3	58.7	0.1421	226.1	882.9
4	59.1	0.1331	231.1	877.2
5	59.7	0.1210	237.1	871.9
6	59.8	0.1126	247.6	747.0
7	59.7	0.1521	251.0	1115.0

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Telesing  
 00 10,1-1,045 7.35 15,571

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.1590	220.0	829.6
2	57.5	0.1486	221.1	872.0
3	58.7	0.1421	226.1	882.9
4	59.1	0.1331	231.1	877.2
5	59.7	0.1191	240.0	845.0
6	59.8	0.1080	244.6	710.7
7	59.7	0.1521	251.0	1190.1

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Telesing  
 40 7,349,100 27.35 15,571

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.1590	220.0	829.6
2	57.5	0.1520	224.0	877.2
3	58.7	0.1514	226.1	877.2
4	59.1	0.0940	232.4	888.0
5	59.7	0.0921	236.7	877.2
6	59.8	0.0840	243.0	745.0
7	59.7	0.0521	251.0	1000.0

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Telesing  
 40 7,349,100 27.35 15,571

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.1590	220.0	829.6
2	57.5	0.1520	224.0	877.2
3	58.7	0.1514	226.1	877.2
4	59.1	0.0940	232.4	888.0
5	59.7	0.0921	236.7	877.2
6	59.8	0.0840	243.0	745.0
7	59.7	0.0521	251.0	1000.0

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	200.0	201.0
2	83.1	0.1581	200.0	200.0
3	82.9	0.1587	200.0	200.0
4	81.0	0.1581	200.0	200.0
5	80.1	0.1579	200.0	200.0
6	80.1	0.1555	200.0	200.0
7	19.7	0.0521	207.0	200.0

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	DP ACROSS VALVE	Thickening	
20	2,322,000	47.07	2,400	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	200.0	201.0
2	83.1	0.1581	200.0	199.0
3	82.9	0.1587	200.0	197.0
4	17.9	0.0604	213.2	518.5
5	17.7	0.0600	217.7	520.1
6	16.9	0.0579	215.0	538.7
7	19.7	0.0521	207.0	597.2

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	DP ACROSS VALVE	Thickening	
20	2,322,000	47.07	2,400	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	200.0	200.3
2	83.1	0.1581	200.0	200.3
3	82.9	0.1587	200.0	200.3
4	17.9	0.0604	213.2	518.5
5	17.7	0.0600	217.7	520.1
6	16.9	0.0579	215.0	538.7
7	19.7	0.0521	207.0	597.2

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	DP ACROSS VALVE	Thicking	
20	1,161,000	47.07	2,400	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	200.0	201.0
2	84.6	0.1587	200.0	200.0
3	84.3	0.1587	200.0	200.0
4	17.9	0.0604	213.2	518.5
5	17.7	0.0600	217.7	520.1
6	16.9	0.0579	215.0	538.7
7	19.7	0.0521	207.0	597.2
8	19.7	0.0521	207.0	597.2

NOTE: THERE IS CHOKED FLOW AT STATION 3

ANGLE	FLOW	DP ACROSS VALVE	Thickening	
10	527,740	47.07	2,400	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1590	200.0	30.7
2	83.1	0.1581	200.0	30.7
3	82.9	0.1587	200.0	30.7

10.7	0.0021	203.1	0.01
11.1	0.0022	200.0	0.01
14.1	0.0021	207.0	0.01

NOTE: PRESS IS MONITORED ONLY AT STATION 3

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN



CASE 2 BCONDITION 1NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"  
VALVE CLASS: 150  
ACTUATOR: Mat-yx 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 700 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.4  
COMPRESSIBILITY 1

INITIAL DENSITY .330 LBS/FT<sup>3</sup>  
FINAL PRESSURE 14.7 PSIA  
MEDIA Air  
SPECIFIC GRAVITY 1  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN. LBS  
PSI

STEM DIA. 1.75 IN.  
PACKING TORQUE 1058 IN. LBS.  
DIRECTION Portland

GAGE DIA. 16.654 IN  
SEAL TORQUE 2396 IN. LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

VALVE SIZE-UP		VALVE CLASS-100		FLOW-100	
UPSTREAM	INITIAL	INITIAL	FINAL	SHUT-OFF	
PRESSURE	DENSITY X 10 <sup>12</sup>	TEMPERATURE	PRESSURE	PRESSURE	
81.7	23	300	14.7	86.7	
MEDIA	RATIO OF	SPECIFIC	COMPRESSIBILITY	HYDRODYNAMIC FACTOR	
	10 <sup>12</sup> LBS/FT <sup>3</sup>	DENSITY		290 F/10 <sup>12</sup>	
AIR	1.4	1	1	32.5	
ITEM	CODE	PAVING	SEAL		
1.5	1.5	10000	10000		

STATION NO.	TYPE OF RESISTANCE	DIAMETER IN	LENGTH FT	RESISTANCE-100	COEFF. FLO
1	ENTRANCE	18.0	0.0	0.800	0.50000
2	PIPE BEND	18.0	0.0	0.140	0.14000
3	VALVE	18.0	0.0	0.740	0.74019
4	ORIFICE PIPE	18.0	0.0	0.000	0.00000
5	VALVE	18.0	0.0	0.740	0.74019
6	EXIT	18.0	0.0	1.000	1.00000

## FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN  
FLOW= 8,540,429 SCFH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	81.7	0.0200	300.0	518.7
2	87.9	0.0126	280.7	482.1
3	89.7	0.0112	267.7	480.7
4	90.4	0.1987	260.1	527.0
5	89.7	0.1092	277.8	517.5
6	47.3	0.1724	267.3	594.8
7	16.7	0.0770	191.4	1206.6

NOTE: THERE IS NO FLOW AT STATION 4

CONDITIONS WITH VALVE SHUT  
VALVE TORQUE= 5,386 IN. LBS  
DELTA PRESS. 65 PSI

# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	TELETYPE	
50	8,549,692	12.10	- 15,571	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.2302	300.0	445.1
2	57.7	0.2126	299.7	420.0
3	58.0	0.2076	297.9	492.5
4	58.3	0.1937	288.1	527.9
5	49.4	0.1892	277.2	537.8
6	40.2	0.1724	261.7	594.8
7	14.7	0.0778	175.4	1226.1

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
50	8,549,692	12.04	- 20,571	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.2302	300.0	445.1
2	57.7	0.2126	299.7	420.0
3	58.0	0.2076	297.9	492.5
4	58.3	0.1937	288.1	527.9
5	49.4	0.1892	277.2	537.8
6	37.4	0.1555	256.3	640.0
7	14.7	0.0778	175.4	1226.1

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
50	7,817,327	12.05	10,700	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.2300	300.0	397.5
2	57.5	0.2171	293.1	420.3
3	58.3	0.2127	291.3	428.2
4	54.5	0.2000	280.0	467.7
5	53.7	0.2016	284.5	451.4
6	34.1	0.1473	250.7	620.3
7	14.7	0.0790	175.4	1155.7

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
50	5,917,725	24.35	- 3,716	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.2300	294.2	310.5
2	41.7	0.2220	294.2	312.2
3	51.5	0.2210	295.5	321.0
4	59.3	0.2153	292.7	328.8
5	59.0	0.2153	292.2	328.8
6	24.5	0.1154	227.7	610.5
7	14.7	0.0790	175.4	895.0

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Telesing
50	5,917,725	24.35	- 3,716

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.0222	300.0	231.2
2	53.0	0.0220	299.9	235.2
3	53.1	0.0220	299.9	235.2
4	53.0	0.0220	299.9	235.2
5	51.2	0.0219	298.7	237.0
6	19.3	0.0071	210.3	549.0
7	14.7	0.0070	196.4	552.5

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	TELESING	
90	1,000.00	10.95	1,000	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.0222	299.8	157.1
2	54.5	0.0222	299.9	157.1
3	54.5	0.0222	299.9	158.0
4	53.6	0.0222	299.8	158.0
5	53.5	0.0222	299.8	158.0
6	18.8	0.0071	203.6	415.0
7	14.7	0.0070	196.4	402.6

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
90	1,000.00	10.95	0.000	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.0220	300.0	97.1
2	54.5	0.0220	299.9	97.1
3	54.5	0.0220	299.9	97.1
4	54.5	0.0220	299.9	97.1
5	54.5	0.0220	299.9	97.1
6	18.8	0.0071	199.0	371.0
7	14.7	0.0070	196.4	360.1

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	TELESING	
90	927,086	69.73	0,001	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.0220	299.9	80.9
2	54.5	0.0220	299.9	80.9
3	54.5	0.0220	299.9	80.9
4	54.5	0.0220	299.9	80.9
5	54.5	0.0220	299.9	80.9
6	18.8	0.0071	197.0	127.8
7	14.7	0.0070	196.4	121.8

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
10	473.747	69.73	2,005	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.0220	300.0	24.7
2	54.5	0.0220	299.9	24.7
3	54.5	0.0220	299.9	24.7

5	0.143	0.2252	230.0	24.7
6	14.7	0.2708	198.0	71.1
7	14.7	0.3702	198.1	71.2

NOTE: THERE IS NO FLOW AT STATION 6

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 2 B

CONDITION 2

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18  
VALVE CLASS: 150  
ACTUATOR: Matryx 25072 SA-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 300 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.329  
COMPRESSIBILITY 1

INITIAL DENSITY .155 LBS/FT<sup>3</sup>  
FINAL PRESSURE 17.7 PSIA  
MEDIA steam  
SPECIFIC GRAVITY .62  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN. LBS  
PSI

STEM DIA. 1.75 IN.  
PACKING TORQUE 1058 IN. LBS.  
DIRECTION Preferred

GAGE DIA. 16.657 IN  
SEAL TORQUE 2296 IN. LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

Case 2A  
cmlr

VALVE TYPE-12		VALVE TYPE-12		FLOW-023	
UPSTREAM	INITIAL	INITIAL	FINAL	SHUT-OFF	
PRESSURE	DENSITY-1000	TEMPERATURE	PRESSURE	PRESSURE	
24.7	10.7	300	14.7	24.7	
MEDIA	RATIO OF	SPECIFIC	CUMULATIVE	HYDRODYNAMIC	
	1.000	GRAVITY		LOSS PER	
STEAM	1.124	.62	1	3245	
ITEM	DATA	WORTHING	NEW		
SIG.	DATA	TORQUE	TORQUE		
1.05	10.054	1000	2195		

STATION NO.	TYPE OF RESISTANCE	DIAMETER (IN)	LENGTH (FT)	RESISTANCE	CORRECTED RESISTANCE - PSI
1	ENTRANCE	18.0	0.0	0.100	0.50000
2	PIPE BEND	18.0	0.0	0.140	0.10000
3	VALVE	18.0	0.0	0.748	0.74819
4	STRAIGHT PIPE	18.0	0.0	0.000	0.00000
5	VALVE	18.0	0.0	0.748	0.74819
6	EXIT	18.0	0.0	1.000	1.00000

## FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN  
FLOW-10,113,045 SCFH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	24.7	0.1890	201.0	529.5
2	27.5	0.1406	201.4	575.0
3	25.7	0.1421	200.1	500.7
4	31.0	0.1334	200.0	607.3
5	20.0	0.1310	211.3	431.7
6	44.0	0.1206	207.0	697.0
7	10.7	0.0821	207.0	1615.0

NOTE: THERE IS NO FLOW AT STATION 1

CONDITIONS WITH VALVE SHUT

VALVE TORQUE- 5,106 IN. LBS  
DELTA PRESSURE PSI

# CONDITIONS AS VALVE CLOSES

ANGLE FLOW DP ACROSS VALVE Tclosing  
20 10,143.04g 9.37 - 10,154

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1570	300.0	529.8
2	87.5	0.1456	291.4	576.0
3	88.7	0.1421	287.1	580.9
4	91.2	0.1334	283.2	627.3
5	80.0	0.1310	281.1	630.7
6	40.5	0.1262	277.9	751.7
7	14.7	0.0521	207.8	1670.6

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Tclosing  
20 10,143.04g 9.37 - 10,154

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1570	300.0	529.8
2	87.5	0.1456	291.4	576.0
3	88.7	0.1421	287.1	580.9
4	91.2	0.1334	283.2	627.3
5	80.0	0.1310	281.1	630.7
6	40.5	0.1120	267.1	751.7
7	14.7	0.0521	207.8	1670.6

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Tclosing  
20 7,330.07g 15.28 - 11,083

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1570	300.0	480.2
2	89.1	0.1406	293.4	516.8
3	87.2	0.1441	291.8	521.5
4	91.2	0.1376	287.4	588.2
5	83.4	0.1391	285.4	580.1
6	38.2	0.1071	243.2	715.3
7	14.7	0.0521	207.8	1670.6

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Tclosing  
20 7,330.07g 12.34 - 3,970

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1570	300.0	300.2
2	81.7	0.1576	298.5	375.6
3	91.2	0.1520	295.0	376.8
4	89.2	0.1488	293.5	406.7
5	87.3	0.1483	293.2	405.7
6	25.6	0.0715	240.8	746.9
7	14.7	0.0521	207.8	1158.7

NOTE: THERE IS CHOKED FLOW AT STATION 6

ANGLE FLOW DP ACROSS VALVE Tclosing  
20 1.4g 1.4g -



	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	81.7	0.1597	200.0	208.4
2	82.1	0.1598	198.3	201.3
3	82.9	0.1598	198.3	201.3
4	81.8	0.1598	198.3	201.3
5	81.8	0.1598	198.3	201.3
6	12.1	0.0569	204.0	590.3
7	15.7	0.0521	213.0	573.2

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
30	3,727.71	86.71	1,731	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	81.7	0.1597	200.0	191.2
2	82.1	0.1598	198.3	194.3
3	82.9	0.1598	198.3	198.7
4	81.8	0.1598	198.3	196.3
5	83.8	0.1589	200.7	196.3
6	16.8	0.0570	210.1	531.7
7	15.7	0.0521	207.0	523.2

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
30	2,108.22	8.35	2,312	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	81.7	0.1598	200.0	120.3
2	82.1	0.1598	198.3	120.3
3	82.9	0.1597	198.3	120.3
4	81.8	0.1598	198.3	120.3
5	82.7	0.1583	200.9	120.3
6	15.8	0.0512	210.3	353.0
7	14.7	0.0511	207.8	347.1

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	TELESING	
30	1,140.713	49.71	2,159	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	81.7	0.1597	200.0	60.3
2	84.6	0.1589	200.0	60.3
3	84.3	0.1589	200.0	60.3
4	81.3	0.1579	200.0	60.3
5	81.3	0.1579	200.0	60.3
6	10.3	0.0526	200.0	183.0
7	10.7	0.0521	200.0	183.0

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Telesing	
10	588,002	19.72	2,405	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.1570	220.0	30.4
2	84.5	0.1580	217.9	30.4
3	84.6	0.1580	220.9	30.4

0	0.000	0.0000	0.000	0.000
1	0.001	0.0001	0.001	0.001
2	0.002	0.0004	0.002	0.002
3	0.003	0.0009	0.003	0.003
4	0.004	0.0016	0.004	0.004
5	0.005	0.0025	0.005	0.005
6	0.006	0.0036	0.006	0.006
7	0.007	0.0049	0.007	0.007
8	0.008	0.0064	0.008	0.008
9	0.009	0.0081	0.009	0.009

NOTE: THERE IS NO FLOW AT STATION 8

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 34

CONDITION 1

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"  
VALVE CLASS: 150  
ACTUATOR: Mat-y x 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 300 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.4  
COMPRESSIBILITY 1

INITIAL DENSITY 230 LBS/FT<sup>3</sup>  
FINAL PRESSURE 64.7 PSIA  
MEDIA Air  
SPECIFIC GRAVITY 1  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN.LBS  
PSI

STEM DIA. 1.15 IN.  
PACKING TORQUE 1058 IN.LBS.  
DIRECTION Nonpreferred

GAGE DIA. 16.654 IN.  
SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

VALVE SIZE-12		VALVE CLASS-160		FLUID-OIL	
UPSTREAM PRESSURE	INITIAL DENSITY-110.02	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE	
54.7	23	300	14.7	54.7	
MEDIA	RATIO OF SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR		
AIR	1.0	1	1	32.5	
PIPE	SIZE	ENDING TORQUE	FEEL TORQUE		
1.5	12.564	1275	2395		

STATION NO.	TYPE OF RESISTANCE	DIAMETER (IN)	LENGTH (FT)	RESISTANCE (IN)	CORRECTED RESISTANCE (IN)
1	ENTRANCE	12.0	0.0	0.501	0.50000
2	VALVE	12.0	0.0	0.768	0.76819
3	STRAIGHT PIPE	12.0	9.5	0.095	0.09500
4	VALVE	12.0	0.0	0.768	0.76819
5	EXIT	12.0	0.0	1.000	1.00000

FLOW IN NONREVERSED DIRECTION

CONDITIONS WITH VALVE OPEN

FROM 2,575,100 CFM

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.2300	300.0	454.0
2	57.5	0.2115	280.1	492.6
3	52.3	0.1977	282.4	527.2
4	50.0	0.1930	280.1	536.7
5	44.7	0.1767	269.9	590.7
6	14.7	0.0770	185.4	1303.5

NOTE: THERE IS NO FLOW AT STATION 6

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 5,306 IN. LBS  
DELTA P=50.00 PSI

# CONDITIONS AS VALVE CLOSING

ANGLE	FLOW	DP ACROSS VALVE	TRIPPING	
80	8,475,183	9.01	15,425	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	454.0
2	57.5	0.2118	290.1	492.5
3	50.3	0.1977	282.8	527.2
4	55.1	0.1930	280.1	535.1
5	44.7	0.1767	269.9	590.7
6	14.7	0.0790	195.1	1300.5

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	TRIPPING	
80	8,475,183	9.01	1,425	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	454.0
2	57.5	0.2118	290.1	492.5
3	48.5	0.1874	278.4	556.2
4	67.0	0.1930	270.3	568.4
5	39.8	0.1607	241.0	641.6
6	14.7	0.0790	195.1	1300.5

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	TRIPPING	
70	7,071,110	14.40	- 3,103	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	411.0
2	59.2	0.2160	290.5	436.6
3	44.7	0.1789	270.0	533.4
4	43.5	0.1734	267.9	542.6
5	37.5	0.1550	257.0	605.6
6	14.7	0.0798	196.4	1184.6

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	TRIPPING	
80	4.181.051	31.12	574	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	323.0
2	51.5	0.2001	285.6	333.9
3	37.5	0.1300	247.0	550.4
4	29.6	0.1317	260.0	564.0
5	26.0	0.1170	230.7	622.9
6	14.7	0.0798	196.4	930.9

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	TRIPPING	
50	4,531,403	41.41	1,572	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY

1	1312.8	0.0000	300.0	159.1
2	1312.8	0.0000	300.0	159.1
3	1312.8	0.0000	300.0	159.1
4	1312.8	0.0000	300.0	159.1
5	1312.8	0.0000	300.0	159.1
6	1312.8	0.0000	300.0	159.1

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Tempering	
30	3,049,430	48.42	2,191	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	159.1
2	84.7	0.02300	300.0	159.1
3	15.0	0.02111	207.8	401.3
4	15.0	0.0205	207.8	404.3
5	15.0	0.02111	207.8	410.1
6	15.0	0.02111	207.8	410.1

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	TEMPERING	
30	1,943,934	49.78	2,391	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	97.3
2	84.5	0.02297	299.8	97.0
3	15.0	0.02360	200.0	108.3
4	15.7	0.0237	200.2	107.1
5	15.3	0.0234	199.0	107.3
6	14.7	0.0228	198.4	100.4

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Tempering	
20	943,483	49.61	2,470	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	49.2
2	84.5	0.02299	299.9	49.2
3	15.0	0.02111	197.8	130.5
4	15.0	0.02111	197.7	130.8
5	14.9	0.02001	197.1	140.7
6	14.7	0.0202	196.4	141.9

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Tempering	
10	477.014	49.99	2,485	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	24.9
2	84.5	0.02299	299.9	24.9
3	14.7	0.02001	196.7	71.6
4	14.7	0.0201	196.7	71.4
5	14.7	0.0202	196.8	71.6
6	14.7	0.0202	196.4	71.7

NOTE: THERE IS CHOKED FLOW AT STATION 2

CASE 3A

CONDITION 2

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"

VALVE CLASS: 150

ACTUATOR: Mntryx 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA

INITIAL TEMPERATURE 300 °F

SHUT OFF PRESSURE 64.7 PSIA

RATIO OF SP. HEAT 1.329

COMPRESSIBILITY 1

INITIAL DENSITY 1.159 LBS/FT<sup>3</sup>

FINAL PRESSURE 14.7 PSIA

MEDIA steam

SPECIFIC GRAVITY 1.62

HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN.LBS  
PSI

STEM DIA. 1.75 IN.

PACKING TORQUE 1058 IN.LBS.

DIRECTION non-preferred

GAGE DIA. 14.657 IN

SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

VALVE SIZE-IN		VALVE PLD-150		FLOW-DAS	
UPSTREAM PRESSURE	INITIAL DENSITY-10 <sup>3</sup>	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE	
54.7	18.9	300	14.7	54.7	
REL IN	RATIO OF HEAT	SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR	
STEAM	1.000	.62	1	3245	
STEM DIA.	BASE DIA.	PACKING TORQUE	SEAL TORQUE		
1.75	18.551	1058	2796		

STATION NO.	TYPE OF RESISTANCE	DIAMETER-(D)	LENGTH-(L)	RESISTANCE-(K)	CORRECTED RESISTANCE-(K)
1	ENTRANCE	18.0	0.0	0.500	0.50000
2	VALVE	18.0	0.0	0.768	0.73817
3	STRAIGHT PIPE	18.0	9.8	0.095	0.09500
4	VALVE	18.0	0.0	0.768	0.73817
5	EXIT	18.0	0.0	1.000	1.00000

FLOW IN NONPREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN  
FLOW=10,123,272 SCFH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	54.7	0.1590	300.0	536.7
2	57.3	0.1451	291.1	586.5
3	52.3	0.1365	295.3	623.5
4	51.0	0.1340	291.5	612.8
5	44.5	0.1240	274.2	687.9
6	14.7	0.0521	207.8	1636.9

NOTE: THERE IS CHOKED FLOW AT STATION 5

CONDITIONS WITH VALVE SHUT  
VALVE TORQUE= 5,306 IN. LBS  
DELTA P=50.00 PSI



# CONDITIONS AS VALVE CLOSES

ANGLE 30 FLOW 10,370,979 DP ACROSS VALVE 7.58 Tclosing 1,572

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	538.7
2	57.3	0.1451	291.1	586.5
3	49.7	0.1304	281.0	623.5
4	48.2	0.1275	279.0	655.4
5	42.4	0.1161	270.4	734.2
6	14.7	0.0521	207.8	1636.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE 30 FLOW 10,370,979 DP ACROSS VALVE 7.58 Tclosing 1,572

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	538.7
2	57.3	0.1451	291.1	586.5
3	49.7	0.1304	281.0	623.5
4	48.2	0.1275	279.0	655.4
5	42.4	0.1161	270.4	734.2
6	14.7	0.0521	207.8	1636.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE 70 FLOW 9,428,031 DP ACROSS VALVE 12.55 Tclosing 3,102

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	494.4
2	58.8	0.1480	293.0	527.4
3	46.7	0.1236	276.1	634.2
4	45.1	0.1215	274.3	644.3
5	40.1	0.1109	266.3	708.5
6	14.7	0.0521	207.8	1507.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE 30 FLOW 7,517,402 DP ACROSS VALVE 23.31 Tclosing 708

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	397.7
2	51.2	0.1325	276.0	417.3
3	32.4	0.0951	253.3	566.9
4	31.3	0.0932	251.2	578.2
5	22.1	0.0840	240.1	744.4
6	14.7	0.0521	207.8	1213.0

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE 50 FLOW 5,505,400 DP ACROSS VALVE 40.44 Tclosing 1,494

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	292.7

4	21.7	0.0522	224.4	645.7
5	22.1	0.0510	224.7	645.9
6	20.4	0.0557	225.6	647.7
6	12.1	0.0521	227.0	652.1

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Closing	
10	3,733.9-1	48.51	2,157	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	194.8
2	64.6	0.1580	299.4	197.3
3	17.3	0.0604	210.0	317.8
4	17.7	0.0590	217.7	321.5
5	16.7	0.0580	218.2	329.4
6	14.7	0.0521	227.0	360.1

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Closing	
30	2,300.207	40.71	2,331	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	120.3
2	64.6	0.1587	299.8	120.8
3	15.8	0.0550	211.9	346.6
4	16.7	0.0550	211.5	347.7
5	15.4	0.0542	210.5	352.9
6	14.7	0.0521	207.8	367.1

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Closing	
30	1,167.145	49.37	2,462	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	60.9
2	64.6	0.1587	299.8	50.7
3	15.0	0.0531	209.1	192.3
4	15.0	0.0530	209.1	192.5
5	14.8	0.0526	208.5	194.0
6	14.7	0.0521	207.8	195.0

NOTE: THERE IS CHOKED FLOW AT STATION 2

ANGLE	FLOW	DP ACROSS VALVE	Closing	
10	570.094	47.37	2,485	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	30.8
2	64.6	0.1587	297.7	30.8
3	14.7	0.0523	208.2	93.5
4	14.7	0.0522	208.1	93.5
5	14.7	0.0522	208.0	93.7
6	14.7	0.0521	207.8	93.7

NOTE: THERE IS CHOKED FLOW AT STATION 2

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 3B

CONDITION 1

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"  
VALVE CLASS: 150  
ACTUATOR: Matryx 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA  
INITIAL TEMPERATURE 300 °F  
SHUT OFF PRESSURE 64.7 PSIA  
RATIO OF SP. HEAT 1.4  
COMPRESSIBILITY 1

INITIAL DENSITY 1270 LBS/FT<sup>3</sup>  
FINAL PRESSURE 14.7 PSIA  
MEDIA Air  
SPECIFIC GRAVITY 1  
HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN.LBS  
PSI

STEM DIA. 1.75 IN.  
PACKING TORQUE 1058 IN.LBS.  
DIRECTION Preferred

GAGE DIA. 16.654 IN  
SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

VALVE SIZE-18		VALVE CLASS-150		FLOW-210	
UPSTREAM PRESSURE	INITIAL DENSITY-10142	INITIAL TEMPERATURE	FINAL PRESSURE	SHUT-OFF PRESSURE	
84.7	0.23	300	14.7	84.7	
WATER	RATIO OF SPECIFIC HEAT	CORRECTED DENSITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR	
AIR	1.4	1	1	3245	
STEM DIA.	SALE DIA.	PACKING TORQUE	SEAL TORQUE		
1.75	14.884	1059	2396		

STATION NO.	TYPE OF RESISTANCE	DIAMETER (IN)	LENGTH (FT)	RESISTANCE (IN)	CORRECTED RESISTANCE (IN)
1	ENTRANCE	18.0	0.0	0.500	0.50000
2	VALVE	18.0	0.0	0.750	0.75017
3	STRAIGHT PIPE	18.0	9.5	0.095	0.09500
4	VALVE	18.0	0.0	0.750	0.75017
5	EXIT	18.0	0.0	1.000	1.00000

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN

FLOW = 3,895,103 SCFH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.2300	300.0	454.0
2	57.5	0.2114	299.1	492.5
3	52.3	0.1977	292.4	527.2
4	50.1	0.1938	290.1	535.7
5	44.7	0.1767	269.9	590.7
6	14.7	0.0793	196.4	1302.5

NOTE: THERE IS CHECKED FLOW AT STATION 5

CONDITIONS WITH VALVE SHUT

VALVE TORQUE = 5,306 IN. LBS

DELTA P = 50.00 PSI

# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
80	8,585,183	21.59	- 18,121	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	454.0
2	57.6	0.2116	290.1	490.6
3	52.3	0.1977	292.4	527.2
4	50.9	0.1933	290.1	534.7
5	39.0	0.1603	289.8	651.3
6	14.7	0.0778	175.4	1300.5

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
80	8,585,183	11.59	- 20,609	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	454.0
2	57.6	0.2116	290.1	490.6
3	52.3	0.1977	292.4	527.2
4	50.9	0.1933	290.1	534.7
5	39.0	0.1603	289.8	651.3
6	14.7	0.0778	175.4	1300.5

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
70	7,727,304	18.54	- 12,171	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	403.4
2	57.6	0.2116	292.4	427.6
3	55.7	0.2066	297.4	448.2
4	54.7	0.2042	295.0	452.6
5	36.2	0.1530	254.1	610.2
6	14.7	0.0778	175.4	1152.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	5,792,555	34.70	- 3,791	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	312.9
2	61.9	0.2230	296.3	322.0
3	59.9	0.2177	293.4	326.0
4	57.6	0.2187	292.6	330.6
5	24.2	0.1161	228.2	619.7
6	14.7	0.0778	175.4	901.9

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
50	4,450,040	42.74	395	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.2300	300.0	232.4

3	81.4	0.02227	300.0	157.7
4	81.4	0.02227	300.0	158.4
5	81.4	0.02227	300.0	158.4
6	14.7	0.0824	199.0	271.6
7	14.7	0.0824	199.0	271.6

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
20	3,300,885	46.92	1,728	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	157.7
2	84.4	0.02286	299.8	158.4
3	83.6	0.02274	299.5	159.3
4	83.6	0.02272	298.5	159.2
5	16.6	0.08271	203.6	418.2
6	16.7	0.08280	197.4	151.0

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
30	1,300,740	40.27	2,330	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	97.4
2	84.5	0.02297	299.8	97.4
3	84.4	0.02292	299.5	97.6
4	81.5	0.02281	299.5	97.6
5	15.3	0.0824	199.0	271.6
6	14.7	0.0798	195.4	280.7

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
20	937,313	47.74	2,451	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	49.0
2	84.6	0.02297	299.7	49.0
3	84.6	0.02298	299.8	49.0
4	84.6	0.02297	299.8	49.0
5	14.8	0.0804	197.1	140.1
6	14.7	0.0798	195.4	141.0

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
10	474,865	49.93	2,435	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	84.7	0.02300	300.0	24.7
2	84.6	0.02299	299.9	24.7
3	84.6	0.02299	299.9	24.7
4	84.6	0.02299	299.9	24.7
5	14.7	0.0799	195.6	71.3
6	14.7	0.0798	195.4	71.4

NOTE: THERE IS CHOKED FLOW AT STATION 4

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

CASE 3 B

CONDITION 2

NUCLEAR LOCA ANALYSIS

VALVE SIZE: 18"

VALVE CLASS: 150

ACTUATOR: Mat-yx 26072 SR-80

UPSTREAM PRESSURE 64.7 PSIA

INITIAL TEMPERATURE 300 °F

SHUT OFF PRESSURE 64.7 PSIA

RATIO OF SP. HEAT 1.329

COMPRESSIBILITY 1

INITIAL DENSITY 1.59 LBS/FT<sup>3</sup>

FINAL PRESSURE 17.7 PSIA

MEDIA steam

SPECIFIC GRAVITY 1.62

HYDRODYNAMIC FACTOR  
@ 90 DEG 3245 IN.LBS  
PSI

STEM DIA. 1.75 IN.

PACKING TORQUE 1058 IN.LBS.

DIRECTION rotated

GAGE DIA. 16.654 IN

SEAL TORQUE 2396 IN.LBS

INPUT STATION NO., K FACTORS, ETC.

(See Appendix A)

# CONTROL SYSTEM ANALYSIS

Case 30

Co-12

UPSTREAM PRESSURE PSI	VALVE SIZE-IN	VALVE CLASS-100	FINAL PRESSURE PSI	SHUT-OFF PRESSURE PSI	FLOW-GAS
60.7	15.0	100	14.7	60.7	
MEDIA	RATIO OF SPEC. HEAT	SPECIFIC GRAVITY	COMPRESSIBILITY	HYDRODYNAMIC FACTOR	
STEAM	1.709	1.62	1	3045	
STEM DIA.	GASK DIA.	PACKING TORQUE	SEAL TORQUE		
1.75	15.854	1058	2394		

STATION NO.	TYPE OF RESISTANCE	DIAMETER-INO	LENGTH-FT	RESISTANCE-INO	CORRECTED RESISTANCE-INO
1	ENTRANCE	18.0	0.0	0.503	0.50000
2	VALVE	18.0	0.0	0.768	0.76819
3	STRAIGHT PIPE	18.0	9.5	0.095	0.09500
4	VALVE	18.0	0.0	0.768	0.76819
5	EXIT	18.0	0.0	1.000	1.00000

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE OPEN  
FLOW=10,279,079 SCFH

	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	535.7
2	57.3	0.1451	291.1	585.5
3	52.3	0.1365	285.3	623.5
4	51.5	0.1340	283.5	631.5
5	46.5	0.1240	276.2	687.9
6	14.7	0.0521	207.5	1636.7

NOTE: THERE IS CHOKED FLOW AT STATION 5

CONDITIONS WITH VALVE SHUT  
VALVE TORQUE= 5,305 IN. LBS  
DELTA P=50.00 PSI



# CONDITIONS AS VALVE CLOSES

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
30	10,270,778	9.24	10,270	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	536.7
2	57.1	0.1481	291.1	506.5
3	52.9	0.1365	285.3	423.5
4	51.5	0.1340	293.2	432.8
5	40.3	0.1185	270.0	738.8
6	14.7	0.0821	207.8	1436.9

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
30	10,270,778	9.24	10,270	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	536.7
2	57.1	0.1481	291.1	506.5
3	52.9	0.1365	285.3	423.5
4	51.5	0.1340	293.2	432.8
5	40.3	0.1185	270.0	738.8
6	14.7	0.0821	207.8	1436.9

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
70	9,323,300	15.25	11,727	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	489.3
2	57.1	0.1405	293.3	521.0
3	55.6	0.1419	289.9	545.9
4	54.5	0.1400	287.7	551.7
5	39.4	0.1095	265.2	709.7
6	14.7	0.0821	207.8	1490.9

NOTE: THERE IS CHOKED FLOW AT STATION 5

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
30	7,380,187	32.48	4,007	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	305.3
2	51.5	0.1531	276.3	379.2
3	59.5	0.1494	293.9	409.0
4	59.5	0.1487	293.6	409.0
5	26.2	0.0820	241.3	746.7
6	14.7	0.0821	207.8	1175.3

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Tclosing	
30	5,508,557	41.85	225	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	287.5

3	62.1	0.1592	299.9	205.3
4	61.9	0.1590	299.8	205.2
5	60.1	0.1589	299.7	205.0
6	14.7	0.0521	207.3	184.9

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
40	1,732,332	46.72	1,732	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	194.2
2	64.2	0.1581	299.4	195.5
3	62.7	0.1572	299.9	196.5
4	63.0	0.1571	299.8	196.5
5	14.8	0.0520	215.1	535.2
6	14.7	0.0521	207.3	594.3

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
30	1,305,205	43.37	1,313	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	120.4
2	64.5	0.1587	299.7	120.4
3	64.4	0.1584	299.6	120.6
4	64.3	0.1584	299.3	120.3
5	15.4	0.0542	210.5	353.1
6	14.7	0.0521	207.3	367.2

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
20	1,101,374	47.72	1,453	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	60.6
2	64.5	0.1587	299.7	60.5
3	64.6	0.1588	299.9	60.5
4	64.5	0.1588	299.7	60.5
5	14.8	0.0526	208.5	183.0
6	14.7	0.0521	207.3	184.9

NOTE: THERE IS CHOKED FLOW AT STATION 4

ANGLE	FLOW	DP ACROSS VALVE	Closing	
10	581,978	49.93	1,485	
	PRESSURE	DENSITY	TEMPERATURE	VELOCITY
1	64.7	0.1590	300.0	30.6
2	64.5	0.1589	299.9	30.5
3	64.5	0.1589	299.9	30.6
4	64.5	0.1589	299.9	30.6
5	14.7	0.0522	208.0	93.2
6	14.7	0.0521	207.3	92.4

NOTE: THERE IS CHOKED FLOW AT STATION 4

NOTE: A POSITIVE CLOSING TORQUE INDICATES THAT THE VALVE WILL TEND TO REMAIN OPEN

## APPENDIX C

### Determination of Closing Times

NOTE: Positive torques are tending to open the valve, negative torques are tending to close the valve.

DETERMINATION OF CLOSING TIME

VALVE SIZE 18" VALVE CLASS 150

ACTUATOR Matrox 26072-SR80

AMOUNT OF VALVE OPENING 90°

DIRECTION OF FLOW Non preferred

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420

ACTUATOR VOLUME 2.84 SCF ACTUATOR YOKE RADIUS 2.63 IN.

ACTUATOR PRESSURE 80 PSIG SOLENOID VALVE  $C_v$  8.48

MEDIA A1- VALVE  $C_v$  11053

HYDRODYNAMIC TORQUE @ 90 3245 SHUT OFF PRESSURE DROP 50 PSI

PACKING TORQUE 1058 IN.LBS. SEAL TORQUE 2396 IN.LBS.

STEM DIA. 1.75 IN. GAGE DIA. 16.654 IN.

BUILDING PRESSURE 50 PSIG dt 0.5 SEC.

<u>DEG.</u>	<u>DENSITY (LBS/FT<sup>3</sup>)</u>	<u>VELOCITY (FT/SEC)</u>	<u>PRESSURE DROP (PSI)</u>
10	<u>.0801</u>	<u>71.4</u>	<u>49.9</u>
20	<u>.0811</u>	<u>139</u>	<u>49.6</u>
30	<u>.0840</u>	<u>266</u>	<u>48.8</u>
40	<u>.0912</u>	<u>401</u>	<u>46.4</u>
50	<u>.106</u>	<u>513</u>	<u>41.4</u>
60	<u>.134</u>	<u>552</u>	<u>31.1</u>
70	<u>.177</u>	<u>533</u>	<u>14.5</u>
80	<u>.187</u>	<u>556</u>	<u>9.01</u>
90	<u>.198</u>	<u>527</u>	<u>5.22</u>

# **DETERMINATION OF CLOSING TIME** **18 - 150 CLASS VALVE WITH A 26072-SR90 ACTUATOR**

THE VALVE IS IN THE NON-PREFERRED DIRECTION

Ispring begin= 10400  
 ACT. VOL.= 2.84  
 MEDIA=AIR  
 PACKING TORQUE= 1058  
 BUILDING PRESSURE= 50

Ispring ending= 6420  
 ACT. YOKE RADIUS= 2.63  
 VALVE Cv= 11053  
 SEAL TORQUE= 2396

TEMPERATURE= 300  
 ACT. PRESS= 80  
 HYDRO. TORQUE @ 90= 3245  
 Dstem= 1.75

SOL. VALVE Cv= 8.48  
 SHUT-OFF PRES. DROP= 50  
 Dgage= 16.654

DEG.	10	20	30	40	50	60	70	80	90
DENSITY	.0801	.0811	.0840	.0912	.1060	.1340	.1770	.1870	.1980
VELOCITY	71.4	139.0	266.0	401.0	513.0	552.0	533.0	556.0	527.0
FRES DROP	49.90	49.60	48.80	46.40	41.40	31.10	14.50	9.01	5.22

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	11861	6301	-10390	14743	1058	149	90.00	5.22
0.50	9660	4100	-10390	14743	1058	149	90.00	5.22
1.00	8728	3168	-10390	14743	1058	149	90.00	5.22
1.50	8172	2612	-10390	14743	1058	149	90.00	5.22
2.00	7794	2234	-10390	14743	1058	149	90.00	5.22
2.50	7517	191	-10390	14743	1058	149	90.00	5.22
3.00	7300	1745	-10390	14743	1058	149	90.00	5.22
3.50	7136	1576	-10390	14743	1058	149	90.00	5.22
4.00	6998	1408	-10390	14743	1058	149	90.00	5.22
4.50	6883	1323	-10390	14743	1058	149	90.00	5.22
5.00	6786	1226	-10390	14743	1058	149	90.00	5.22
5.50	6703	1143	-10390	14743	1058	149	90.00	5.22
6.00	6630	1070	-10390	14743	1058	149	90.00	5.22
6.50	6567	1007	-10390	14743	1058	149	90.00	5.22
7.00	6511	951	-10390	14743	1058	149	90.00	5.22
7.50	6460	901	-10390	14743	1058	149	90.00	5.22
8.00	6415	856	-10390	14743	1058	149	90.00	5.22
8.50	6376	815	-10390	14743	1058	149	90.00	5.22
9.00	6338	778	-10390	14743	1058	149	90.00	5.22
9.50	6304	744	-10390	14743	1058	149	90.00	5.22
10.00	6274	714	-10390	14743	1058	149	90.00	5.22

\* 14743 + 1058 + 149 agree with Page B-43

DETERMINATION OF CLOSING TIMEVALVE SIZE 18" VALVE CLASS 150ACTUATOR Matrox 26072-SR80AMOUNT OF VALVE OPENING 70°DIRECTION OF FLOW Non preferred

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420

ACTUATOR VOLUME <u>2.84</u> SCF	ACTUATOR YOKE RADIUS <u>2.63</u> IN.
ACTUATOR PRESSURE <u>80</u> PSIG	SOLENOID VALVE C <sub>v</sub> <u>8.48</u>
MEDIA <u>A1-</u>	VALVE C <sub>v</sub> <u>11053</u>
HYDRODYNAMIC TORQUE @ 90 <u>3275</u>	SHUT OFF PRESSURE DROP <u>50</u> PSI
PACKING TORQUE <u>1058</u> IN.LBS.	SEAL TORQUE <u>2396</u> IN.LBS.
STEM DIA. <u>1.75</u> IN.	GAGE DIA. <u>16.654</u> IN.
BUILDING PRESSURE <u>50</u> PSIG	dt <u>.05</u> SEC.

DEG.	DENSITY (LBS/FT <sup>3</sup> )	VELOCITY (FT/SEC)	PRESSURE DROP (PSI)
10	<u>.0801</u>	<u>71.4</u>	<u>49.9</u>
20	<u>.0811</u>	<u>139</u>	<u>49.6</u>
30	<u>.0840</u>	<u>266</u>	<u>48.8</u>
40	<u>.0912</u>	<u>401</u>	<u>46.4</u>
50	<u>.106</u>	<u>513</u>	<u>41.4</u>
60	<u>.134</u>	<u>552</u>	<u>31.1</u>
70	<u>.177</u>	<u>533</u>	<u>14.5</u>
80	<u>.187</u>	<u>556</u>	<u>9.01</u>
90	<u>.198</u>	<u>527</u>	<u>5.22</u>

# DETERMINATION OF CLOSING TIME

18 - 150 CLASS VALVE WITH A 26072-SR80 ACTUATOR

THE VALVE OPENING IS RESTRICTED TO 70 DEGS.

THE VALVE IS IN THE NON-PREFERRED DIRECTION

Tspring begin= 10400  
ACT. VOL.= 2.84  
MEDIA=AIR  
PACKING TORQUE= 1058  
BUILDING PRESSURE= 50

Tspring ending= 6420  
ACT. YOKE RADIUS= 2.63  
VALVE CV= 11053  
SEAL TORQUE= 2396

TEMPERATURE= 300  
ACT. PRESS= 80  
HYDRD. TORQUE @ 90= 3245  
Dstem= 1.75

SOL. VALVE CV= 8.48  
SHUT-OFF PRES. DROP= 50  
Dgage= 16.654

DEG.	10	20	30	40	50	60	70	80	90
DENSITY	.0801	.0811	.0840	.0912	.1060	.1340	.1770	.1870	.1980
VELOCITY	71.4	139.0	266.0	401.0	513.0	552.0	533.0	556.0	527.0
FRES. DROP	49.90	40.50	48.90	46.40	41.40	31.10	14.50	9.01	5.22

LOCA CLOSES THE VALVE TO 54 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM AERODYNAMIC TORQUE AS VALVE CLOSES IS -3111 IN.LBS. @ 70 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	0	4032	-4993	-1281	1058	1067	54.00	37.28
0.05	0	3067	-4830	-516	1058	1221	47.49	42.65
0.10	0	2659	-4731	-279	1058	1293	42.46	45.16
0.15	0	2417	-4652	-163	1058	1341	38.13	46.84
0.20	0	2308	-4626	-108	1058	1368	34.26	47.77
0.25	0	2255	-4636	-70	1058	1392	30.70	48.63
0.30	0	2257	-4676	-42	1058	1403	27.38	49.00
0.35	0	2296	-4741	-23	1058	1410	24.26	47.75
0.40	0	2365	-4829	-11	1058	1417	21.30	49.49
0.45	0	2464	-4937	-6	1058	1421	18.47	49.64
0.50	0	2587	-5065	-4	1058	1424	15.77	49.72
0.55	0	2729	-5210	-2	1058	1426	13.19	49.80
0.60	0	2888	-5374	-1	1058	1428	10.73	49.87
0.65	0	3066	-5552	-0	1058	1429	8.38	49.91
0.70	0	3258	-5746	-0	1058	1430	6.15	49.93
0.75	0	3464	-5952	-0	1058	1430	4.03	49.95
0.80	0	2317	-6170	-0	2421	1431	2.02	49.97
0.85	0	1781	-6312	-0	3099	1431	0.82	49.99

\* Agrees with Page B-43

DETERMINATION OF CLOSING TIME

VALVE SIZE 18" VALVE CLASS 150

ACTUATOR Matyx 24072-SR80

AMOUNT OF VALVE OPENING 65°

DIRECTION OF FLOW Non preferred

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420

ACTUATOR VOLUME 2.94 SCF ACTUATOR YOKE RADIUS 2.63 IN.

ACTUATOR PRESSURE 80 PSIG SOLENOID VALVE  $C_v$  8.48

MEDIA A1- VALVE  $C_v$  11053

HYDRODYNAMIC TORQUE @ 90 3245 SHUT OFF PRESSURE DROP 50 PSI

PACKING TORQUE 1058 IN.LBS. SEAL TORQUE 2396 IN.LBS.

STEM DIA. 1.75 IN. GAGE DIA. 16.654 IN.

BUILDING PRESSURE 50 PSIG dt .05 SEC.

DEG.	DENSITY (LBS/FT <sup>3</sup> )	VELOCITY (FT/SEC)	PRESSURE DROP (PSI)
10	<u>.0801</u>	<u>71.4</u>	<u>49.9</u>
20	<u>.0811</u>	<u>139</u>	<u>49.6</u>
30	<u>.0840</u>	<u>200</u>	<u>48.8</u>
40	<u>.0912</u>	<u>401</u>	<u>46.4</u>
50	<u>.106</u>	<u>513</u>	<u>41.4</u>
60	<u>.134</u>	<u>552</u>	<u>31.1</u>
70	<u>.177</u>	<u>533</u>	<u>14.5</u>
80	<u>.187</u>	<u>556</u>	<u>9.01</u>
90	<u>.198</u>	<u>527</u>	<u>5.22</u>



# DESCRIPTION OF CLOSING TIME 18 - 150 CLASS VALVE WITH A 26072-SR80 ACTUATOR

THE VALVE OPENING IS RESTRICTED TO 65 DEGS.

THE VALVE IS IN THE NON-PREFERRED DIRECTION

Tspring begin= 10400	Tspring ending= 6420	TEMPERATURE= 300	
ACT. VOL.= 2.84	ACT. YOKE RADIUS= 2.63	ACT. PRESS= 80	SOL. VALVE Cv= 8.48
MEDIA= AIR	VALVE Cv= 11053	HYDRO. TORQUE @ 90= 3245	SHUT-OFF PRES. DROP= 50
PACKING TORQUE= 1058	SEAL TORQUE= 2396	Dstem= 1.75	Dgpg= 16.654
BUILDING PRESSURE= 50			

DEG.	10	20	30	40	50	60	70	80	90
DENSITY	.0801	.0811	.0840	.0912	.1060	.1340	.1770	.1870	.1980
VELOCITY	71.4	139.0	266.0	401.0	513.0	552.0	533.0	556.0	527.0
PRES DROP	49.90	49.60	48.80	46.40	41.40	31.10	14.50	9.01	5.22

LOCA CLOSES THE VALVE TO 53 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM AERODYNAMIC TORQUE AS VALVE CLOSIS IS -1786 IN.LBS. @ 65 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	0	3896	-4960	-1114	1058	1097	53.00	38.31
0.05	0	2999	-4819	-470	1058	1233	46.68	43.05
0.10	0	2607	-4714	-254	1058	1303	41.76	45.51
0.15	0	2396	-4645	-153	1058	1345	37.52	46.99
0.20	0	2297	-4625	-101	1058	1371	33.70	47.90
0.25	0	2252	-4640	-65	1058	1396	30.18	48.75
0.30	0	2261	-4684	-38	1058	1404	26.90	49.04
0.35	0	2305	-4753	-21	1058	1411	23.80	49.29
0.40	0	2378	-4844	-10	1058	1418	20.86	49.53
0.45	0	2481	-4955	-6	1058	1422	18.06	49.65
0.50	0	2607	-5085	-3	1058	1424	15.38	49.73
0.55	0	2752	-5234	-2	1058	1426	12.81	49.81
0.60	0	2914	-5400	-1	1058	1428	10.37	49.88
0.65	0	3094	-5581	-0	1058	1429	8.04	49.91
0.70	0	3288	-5776	-0	1058	1430	5.82	49.94
0.75	0	3496	-5985	-0	1058	1430	3.72	49.96
0.80	0	2159	-6204	-0	2613	1431	1.73	49.98
0.85	0	1716	-6335	-0	3187	1431	0.63	49.99

DETERMINATION OF CLOSING TIMEVALVE SIZE 18" VALVE CLASS 150ACTUATOR Mattys 26072-3R80AMOUNT OF VALVE OPENING 90°DIRECTION OF FLOW Portland

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420

ACTUATOR VOLUME <u>2.84</u> SCF	ACTUATOR YOKE RADIUS <u>2.63</u> IN.
ACTUATOR PRESSURE <u>80</u> PSIG	SOLENOID VALVE $C_v$ <u>8.48</u>
MEDIA <u>A1-</u>	VALVE $C_v$ <u>11053</u>
HYDRODYNAMIC TORQUE @ 90 <u>3245</u>	SHUT OFF PRESSURE DROP <u>50</u> PSI
PACKING TORQUE <u>1058</u> IN.LBS.	SEAL TORQUE <u>2396</u> IN.LBS.
STEM DIA. <u>1.25</u> IN.	GAGE DIA. <u>16.654</u> IN.
BUILDING PRESSURE <u>0</u> PSIG	dt <u>1.05</u> SEC.

<u>DEG.</u>	<u>DENSITY (LBS/FT<sup>3</sup>)</u>	<u>VELOCITY (FT/SEC)</u>	<u>PRESSURE DROP (PSI)</u>
10	<u>.0799</u>	<u>71.3</u>	<u>49.9</u>
20	<u>.0804</u>	<u>140</u>	<u>49.7</u>
30	<u>.0824</u>	<u>272</u>	<u>49.0</u>
40	<u>.0871</u>	<u>416</u>	<u>47.0</u>
50	<u>.0972</u>	<u>550</u>	<u>42.7</u>
60	<u>.116</u>	<u>619</u>	<u>34.7</u>
70	<u>.152</u>	<u>610</u>	<u>18.5</u>
80	<u>.160</u>	<u>651</u>	<u>11.9</u>
90	<u>.177</u>	<u>591</u>	<u>6.17</u>

# **DETERMINATION OF CLOSING TIME** 1B - 150 CLASS VALVE WITH A 26072-SR80 ACTUATOR

THE VALVE IS IN THE PREFERRED DIRECTION

Ispring begin= 10400	Ispring ending= 3470	TEMPERATURE= 70	
ACT. VOL.= 2.94	ACT. YOKE RADIUS= 2.63	ACT. PRESS= 80	SOL. VALVE CV= 8.48
MEDIA= AIR	VALVE CV= 11053	HYDRO. TORQUE @ 90= 3245	SHUT-OFF PRES. DROP= 50
PACKING TORQUE= 1058	SEAL TORQUE= 2394	Dstem= 1.75	Dgage= 16.654
BUILDING PRESSURE= 0			

DEG.	10	20	30	40	50	60	70	80	90
DENSITY	.0799	.0804	.0824	.0871	.0972	.1160	.1520	.1600	.1770
VELOCITY	71.3	140.0	272.0	416.0	550.0	619.0	610.0	651.0	591.0
PRES DROP	49.90	49.70	49.00	47.00	42.70	34.70	18.50	11.90	6.17

LOCA CLOSED THE VALVE TO 65 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM AERODYNAMIC TORQUE AS VALVE CLOSING IS -20599<sup>+</sup> IN.LBS. @ 80 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	0	13170	-5614	-9496	1058	761	65.00	26.60
0.05	0	3111	-4717	-774	1058	1322	41.87	46.19
0.10	0	2553	-4632	-349	1058	1369	35.83	47.83
0.15	0	2339	-4634	-160	1058	1397	31.05	48.78
0.20	0	2300	-4684	-83	1058	1409	26.93	49.21
0.25	0	2338	-4771	-42	1058	1416	23.17	49.47
0.30	0	2427	-4888	-20	1058	1423	19.66	49.70
0.35	0	2562	-5034	-11	1058	1425	16.38	49.77
0.40	0	2727	-5206	-5	1058	1427	13.26	49.83
0.45	0	2918	-5403	-2	1058	1428	10.32	49.87
0.50	0	3135	-5627	-0	1058	1429	7.55	49.92
0.55	0	3372	-5861	-0	1058	1430	4.94	49.95
0.60	0	2651	-6117	-0	2034	1431	2.50	49.97
0.65	0	1778	-6313	-0	3102	1431	0.81	49.99

\* Agrees with Pages B-51 & E-1

DETERMINATION OF CLOSING TIMEVALVE SIZE 18" VALVE CLASS 150ACTUATOR Matvix 26072-SP80AMOUNT OF VALVE OPENING 70°DIRECTION OF FLOW Preferred

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420ACTUATOR VOLUME 2.84 SCFACTUATOR YOKE RADIUS 2.63 IN.ACTUATOR PRESSURE 80 PSIGSOLENOID VALVE C<sub>v</sub> 8.48MEDIA ALVALVE C<sub>v</sub> 11053HYDRODYNAMIC TORQUE @ 90 3245SHUT OFF PRESSURE DROP 50 PSIPACKING TORQUE 1058 IN.LBS.SEAL TORQUE 2396 IN.LBS.STEM DIA. 1.75 IN.GAGE DIA. 16.654 IN.BUILDING PRESSURE 0 PSIGdt .05 SEC.

<u>DEG.</u>	<u>DENSITY (LBS/FT<sup>3</sup>)</u>	<u>VELOCITY (FT/SEC)</u>	<u>PRESSURE DROP (PSI)</u>
10	<u>.0799</u>	<u>7.3</u>	<u>49.9</u>
20	<u>.0804</u>	<u>140</u>	<u>49.7</u>
30	<u>.0824</u>	<u>272</u>	<u>49.0</u>
40	<u>.0871</u>	<u>416</u>	<u>47.0</u>
50	<u>.0972</u>	<u>550</u>	<u>42.7</u>
60	<u>.116</u>	<u>619</u>	<u>34.7</u>
70	<u>.152</u>	<u>610</u>	<u>18.5</u>
80	<u>.160</u>	<u>651</u>	<u>11.9</u>
90	<u>.177</u>	<u>591</u>	<u>6.17</u>

# **DETERMINATION OF CLOSING TIME** 18 - 150 CLASS VALVE WITH A 26072-SR80 ACTUATOR

THE VALVE OPENING IS RESTRICTED TO 70 DEGS.

THE VALVE IS IN THE PREFERRED DIRECTION

Tspring begin= 10400  
ACT. VOL.= 2.94  
MEDIA= AIR  
PACKING TORQUE= 1058  
BUILDING PRESSURE= 0

Tspring ending= 8420  
ACT. YOKE RADIUS= 2.63  
VALVE CV= 11053  
SEAL TORQUE= 2396

TEMPERATURE= 70  
ACT. PRESS= 80  
HYDRO. TORQUE @ 90= 3245  
Dstem= 1.75

SOL. VALVE CV= 8.48  
SHUT-OFF PRES. DROP= 50  
Dgage= 16.654

DEG.	10	20	30	40	50	60	70	80	90
DENSITY	.0799	.0804	.0824	.0871	.0972	.1160	.1520	.1600	.1770
VELOCITY	71.3	140.0	272.0	416.0	550.0	619.0	610.0	651.0	591.0
PRES. DROP	49.90	49.70	49.00	47.00	42.70	34.70	18.50	11.90	6.17

LOCA CLOSES THE VALVE TO 62 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM AERODYNAMIC TORQUE AS VALVE CLOSES IS -12212 IN.LBS. @ 70 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	0	10643	-5394	-7222	1058	900	62.00	31.46
0.05	0	3215	-4735	-851	1058	1313	42.63	45.86
0.10	0	2589	-4635	-379	1058	1366	36.38	47.72
0.15	0	2352	-4631	-174	1058	1394	31.51	48.69
0.20	0	2299	-4677	-88	1058	1408	27.33	49.18
0.25	0	2332	-4760	-46	1058	1416	23.54	49.45
0.30	0	2415	-4875	-21	1058	1423	20.01	49.69
0.35	0	2546	-5018	-1	1058	1425	16.71	49.76
0.40	0	2708	-5187	-6	1058	1426	13.58	49.82
0.45	0	2897	-5381	-2	1058	1428	10.62	49.88
0.50	0	3112	-5598	-1	1058	1429	7.83	49.92
0.55	0	3347	-5835	-0	1058	1430	5.20	49.94
0.60	0	2903	-6089	-0	1755	1431	2.74	49.97
0.65	0	1800	-6305	-0	3073	1431	0.87	49.99

\* Agree with Pages B-33 & E-4

DETERMINATION OF CLOSING TIMEVALVE SIZE 18" VALVE CLASS 150ACTUATOR Matryx 26072-SP80AMOUNT OF VALVE OPENING 65°DIRECTION OF FLOW Preferred

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420

ACTUATOR VOLUME 2.84 SCF  
 ACTUATOR PRESSURE 80 PSIG  
 MEDIA Air  
 HYDRODYNAMIC TORQUE @ 90 3245  
 PACKING TORQUE 1058 IN.LBS.  
 STEM DIA. 1.75 IN.  
 BUILDING PRESSURE 0 PSIG

ACTUATOR YOKE RADIUS 2.63 IN.  
 SOLENOID VALVE  $C_v$  8.48  
 VALVE  $C_v$  11053  
 SHUT OFF PRESSURE DROP 50 PSI  
 SEAL TORQUE 2396 IN.LBS.  
 GAGE DIA. 16.654 IN.  
 dt .05 SEC.

DEG.	DENSITY (LBS/FT <sup>3</sup> )	VELOCITY (FT/SEC)	PRESSURE DROP (PSI)
10	<u>.0799</u>	<u>71.3</u>	<u>49.9</u>
20	<u>.0804</u>	<u>140</u>	<u>49.7</u>
30	<u>.0824</u>	<u>272</u>	<u>49.0</u>
40	<u>.0871</u>	<u>416</u>	<u>47.0</u>
50	<u>.0972</u>	<u>556</u>	<u>42.7</u>
60	<u>.116</u>	<u>619</u>	<u>34.7</u>
70	<u>.152</u>	<u>610</u>	<u>18.5</u>
80	<u>.160</u>	<u>651</u>	<u>11.9</u>
90	<u>.177</u>	<u>591</u>	<u>6.17</u>

EXPERIMENTAL OF CLOSING TIME  
18 - 150 CLASS VALVE WITH A 24072-SR80 ACTUATOR

THE VALVE OPENING IS RESTRICTED TO 65 DEGS.

THE VALVE IS IN THE PREFERRED DIRECTION

Tspring begin= 10400  
ACT. VOL.= 2.84  
MED. A-AIR  
PACKING TORQUE= 1058  
BUILDING PRESSURE= 0

Tspring ending= 6420  
ACT. YOKE RADIUS= 2.63  
VALVE CV= 11053  
SEAL TORQUE= 2396

TEMPERATURE= 70  
ACT. PRESS= 80  
HYDRO. TORQUE @ 90= 3245  
Dste= 1.75

SOL. VALVE Cv= 8.48  
SHUT-OFF PRES. DROP= 50  
Dqage= 16.554

DEG.	10	20	30	40	50	60	70	80	90
DENSITY	.0799	.0804	.0824	.0871	.0972	.1160	.1520	.1600	.1770
VELOCITY	71.3	140.0	272.0	416.0	550.0	619.0	610.0	651.0	591.0
PRES DROP	49.90	49.70	49.00	47.00	42.70	34.70	18.50	11.90	6.17

LOCA CLOSSES THE VALVE TO 62 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM AERODYNAMIC TORQUE AS VALVE CLOSSES IS -7677 IN.LBS. @ 65 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	0	10265	-5394	-7223	1058	900	60.00	31.46
0.05	0	3318	-4753	-927	1058	1304	43.34	45.56
0.10	0	2623	-4639	-405	1058	1363	36.88	47.52
0.15	0	2386	-4629	-187	1058	1392	31.92	48.61
0.20	0	2299	-4671	-93	1058	1407	27.69	49.16
0.25	0	2327	-4751	-49	1058	1415	23.87	49.42
0.30	0	2405	-4863	-22	1058	1422	20.31	49.67
0.35	0	2533	-5003	-12	1058	1424	17.00	49.75
0.40	0	2692	-5171	-6	1058	1426	13.86	49.82
0.45	0	2879	-5363	-2	1058	1428	10.88	49.88
0.50	0	3091	-5577	-1	1058	1429	8.00	49.91
0.55	0	3325	-5813	-0	1058	1430	5.44	49.94
0.60	0	3317	-6066	-0	1317	1430	2.96	49.97
0.65	0	1777	-6313	-0	3104	1431	0.81	49.99

DETERMINATION OF CLOSING TIMEVALVE SIZE 18" VALVE CLASS 150ACTUATOR Matt, xx 26072-SR80AMOUNT OF VALVE OPENING 70°DIRECTION OF FLOW Non preferred

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420ACTUATOR VOLUME 2.81 SCFACTUATOR PRESSURE 80 PSIGMEDIA AirHYDRODYNAMIC TORQUE @ 90 3245PACKING TORQUE 1058 IN.LBS.STEM DIA. 1.75 IN.BUILDING PRESSURE 50 PSIGACTUATOR YOKE RADIUS 2.13 IN.SOLENOID VALVE C<sub>v</sub> 9.48VALVE C<sub>v</sub> 11053SHUT OFF PRESSURE DROP 50 PSISEAL TORQUE 2396 IN.LBS.GAGE DIA. 16.654 IN.dt .05 SEC.

<u>DEG.</u>	<u>AERO TORQUE (IN.LBS.)</u>	<u>PRESSURE DROP (PSI)</u>
10	<u>2410</u>	<u>49.9</u>
20	<u>2146</u>	<u>49.6</u>
30	<u>399</u>	<u>48.8</u>
40	<u>- 1145</u>	<u>46.4</u>
50	<u>- 4763</u>	<u>41.1</u>
60	<u>- 9453</u>	<u>30.6</u>
70	<u>- 13350</u>	<u>14.5</u>
80	<u>- 9100</u>	<u>9.00</u>
90	<u>6879</u>	<u>5.19</u>

NOTE: The AERO TORQUE is the calculated LOCA torque minus the packing and bearing torques.



# DETERMINATION OF CLOSING TIME

1B - 150 CLASS VALVE WITH A 26072-SR80 ACTUATOR

THE VALVE OPENING IS RESTRICTED TO 70 DEGS.

THE VALVE IS IN THE NON-PREFERRED DIRECTION

Ispring begin= 10400  
ACT. VOL.= 2.84  
MEDIA= AIR  
PACKING TORQUE= 1058  
BUILDING PRESSURE= 50

Ispring ending= 6420  
ACT. YOKE RADIUS= 2.63  
VALVE Cv= 11053  
SEAL TORQUE= 2396

TEMPERATURE= 300  
ACT. PRESS= 80  
HYDRO. TORQUE @ 70= 3243  
Dstem= 1.75

SOL. VALVE Cv= 8.48  
SHUT-OFF PRES. DROP= 50  
Dgage= 15.354

DEG.	10	20	30	40	50	60	70	80	90
AERO TORQUE	2410	2146	399	-1145	-4763	-9452	-13350	-9100	6879
PRES DROP	49.90	49.60	48.80	46.40	41.10	30.60	14.50	9.00	5.19

LOCA CLOSING THE VALVE TO 33 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM LOCA TORQUE AS VALVE CLOSING IS -13350 IN. LBS. @ 70 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P sec
0.00	0	4749	-4626	-2499	1058	1376	33.00	48.080
0.02	0	4143	-4648	-1951	1058	1373	29.39	48.848
0.05	0	3668	-4694	-1437	1058	1405	25.41	49.087
0.07	0	3288	-4750	-1007	1058	1411	23.91	49.286
0.10	0	2990	-4812	-642	1058	1416	21.79	49.456
0.12	0	2730	-4876	-331	1053	1420	19.97	49.600
0.15	0	2752	-4941	-291	1058	1421	18.19	49.648
0.17	0	2781	-5010	-252	1058	1423	16.17	49.693
0.20	0	2817	-5084	-215	1058	1424	15.41	49.737
0.22	0	2858	-5162	-179	1058	1425	14.11	49.779
0.25	0	2904	-5244	-144	1058	1426	12.65	49.820
0.27	0	2956	-5330	-111	1058	1427	11.35	49.859
0.30	0	3012	-5419	-79	1058	1428	10.10	49.896
0.32	0	3072	-5511	-68	1058	1429	8.90	49.910
0.35	0	3179	-5607	-59	1058	1429	7.73	49.922
0.37	0	3269	-5706	-50	1058	1429	6.59	49.934
0.40	0	3362	-5808	-42	1058	1430	5.48	49.945
0.42	0	3459	-5913	-34	1058	1430	4.41	49.955
0.45	0	3558	-6021	-26	1058	1430	3.37	49.966
0.47	0	3564	-6131	-18	2154	1431	2.37	49.976
0.50	0	2147	-6210	-12	2644	1431	1.68	49.983
0.52	0	1903	-6275	-8	2948	1431	1.13	49.988
0.55	0	1730	-6332	-5	3175	1431	0.65	49.993
0.57	0	1597	-6384	-1	3356	1431	0.23	49.997

\* Agrees with Pages 53 and E-10

DETERMINATION OF CLOSING TIME

VALVE SIZE 18" VALVE CLASS 150

ACTUATOR M. T. Y. 26072-SR80

AMOUNT OF VALVE OPENING 65°

DIRECTION OF FLOW Non preferred

ACTUATOR TORQUES (IN. LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420

ACTUATOR VOLUME 2.87 SCF ACTUATOR YOKE RADIUS 2.63 IN.

ACTUATOR PRESSURE 80 PSIG SOLENOID VALVE C<sub>v</sub> 9.48

MEDIA Air VALVE C<sub>v</sub> 11053

HYDRODYNAMIC TORQUE @ 90 3245 SHUT OFF PRESSURE DROP 50 PSI

PACKING TORQUE 1058 IN. LBS. SEAL TORQUE 2396 IN. LBS.

STEM DIA. 1.75 IN. GAGE DIA. 16.454 IN.

BUILDING PRESSURE 50 PSIG dt 1.05 SEC.

<u>DEG.</u>	<u>AERO TORQUE (IN. LBS.)</u>	<u>PRESSURE DROP (PSI)</u>
10	<u>2410</u>	<u>49.9</u>
20	<u>2146</u>	<u>49.6</u>
30	<u>399</u>	<u>48.8</u>
40	<u>- 1145</u>	<u>46.4</u>
50	<u>- 4763</u>	<u>41.1</u>
60	<u>- 9453</u>	<u>30.6</u>
70	<u>- 13350</u>	<u>14.5</u>
80	<u>- 9100</u>	<u>9.00</u>
90	<u>6879</u>	<u>5.19</u>

NOTE: The AERO TORQUE is the calculated LOCA torque minus the packing and bearing torques.

# DETERMINATION OF CLOSING TIME 18 - 150 CLASS VALVE WITH A 26072-SK80 ACTUATOR

THE VALVE OPENING IS RESTRICTED TO 65 DEGS.

THE VALVE IS IN THE NON-PREFERRED DIRECTION

Tspring begin= 10400	Tspring ending= 6420	TEMPERATURE= 300	
ACT. VOL.= 2.84	ACT. YOKE RADIUS= 2.63	ACT. PRESS= 80	SOL. VALVE Cv= 8.48
MEDIA= AIR	VALVE Cv= 11053	HYDRO. TORQUE @ 90= 3245	SHUT-OFF PRES. DROP= 50
PACKING TORQUE= 1058	SEAL TORQUE= 2396	Dstem= 1.75	Dqage= 16.654
BUILDING PRESSURE= 50			

DEG.	10	20	30	40	50	60	70	80	90
AERO TORQUE	2410	2146	339	-1145	-4763	-9452	-13350	-9100	6879
PRES DROP	49.90	49.60	48.80	46.40	41.10	30.60	14.50	9.00	5.19

LOCA CLOSES THE VALVE TO 32 DEGREES WITH THE ACTUATOR STILL ACTUATED

MAXIMUM LOCA TORQUE AS VALVE CLOSES IS -1.401 IN.LBS. @ 65 DEGREES

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P sec
0.00	0	4503	-4628	-2399	1058	1383	32.00	48.320
0.02	0	4058	-4659	-1857	1058	1400	28.55	48.915
0.05	0	3588	-4709	-1344	1058	1407	25.67	49.146
0.07	0	3213	-4768	-915	1058	1412	23.26	49.338
0.10	0	2908	-4831	-552	1058	1417	21.23	49.501
0.12	0	2736	-4890	-319	1058	1420	19.48	49.615
0.15	0	2760	-4961	-279	1058	1422	17.92	49.662
0.17	0	2791	-5032	-240	1058	1423	16.42	49.707
0.20	0	2829	-5107	-204	1058	1424	14.97	49.750
0.22	0	2871	-5187	-168	1058	1425	13.59	49.792
0.25	0	2919	-5270	-134	1058	1427	12.25	49.832
0.27	0	2972	-5367	-101	1058	1428	10.97	49.870
0.30	0	3035	-5467	-74	1058	1429	9.73	49.902
0.32	0	3118	-5540	-65	1058	1429	8.53	49.914
0.35	0	3206	-5637	-56	1058	1429	7.37	49.926
0.37	0	3297	-5737	-48	1058	1430	6.25	49.937
0.40	0	3391	-5840	-39	1058	1430	5.15	49.948
0.42	0	3489	-5946	-31	1058	1430	4.09	49.959
0.45	0	3589	-6054	-23	1058	1430	3.06	49.969
0.47	0	2357	-6165	-15	2392	1431	2.06	49.979
0.50	0	2036	-6237	-11	2781	1431	1.44	49.985
0.52	0	1927	-6299	-7	3047	1431	0.93	49.990
0.55	0	1672	-6354	-3	3253	1431	0.48	49.995
0.57	0	1551	-6404	-0	3421	1431	0.08	49.999

APPENDIX D

Comparison of Actual to Calculated  
Closing Times

NOTE: Calculated positive torques are  
tending to open the valve, negative  
torques are tending to close the  
valve.

DETERMINATION OF CLOSING TIMEVALVE SIZE 18 VALVE CLASS 150ACTUATOR Matryx 2002-SR80AMOUNT OF VALVE OPENING 90°DIRECTION OF FLOW No Flow

ACTUATOR TORQUES (IN.LBS)

SPRING BEGINNING 10400 SPRING ENDING 6420ACTUATOR VOLUME 2.84 SCFACTUATOR YOKE RADIUS 2.03 IN.ACTUATOR PRESSURE 80 PSIGSOLENOID VALVE  $C_v$  8.48

MEDIA \_\_\_\_\_

VALVE  $C_v$  \_\_\_\_\_

HYDRODYNAMIC TORQUE @ 90 \_\_\_\_\_

SHUT OFF PRESSURE DROP \_\_\_\_\_ PSI

PACKING TORQUE 1058 IN.LBS.SEAL TORQUE 2396 IN.LBS.STEM DIA. 1.75 IN.GAGE DIA. 10.654 IN.BUILDING PRESSURE 0 PSIGdt .05 SEC.

<u>DEG.</u>	<u>DENSITY (LBS/FT<sup>3</sup>)</u>	<u>VELOCITY (FT/SEC)</u>	<u>PRESSURE DROP (PSI)</u>
10	_____	_____	_____
20	_____	_____	_____
30	_____	_____	_____
40	_____	_____	_____
50	_____	_____	_____
60	_____	_____	_____
70	_____	_____	_____
80	_____	_____	_____
90	_____	_____	_____

DETERMINATION OF CLOSING CAP  
18 - 150 CLASS VALVE WITH A 26072-SR80 ACTUATOR

THERE IS NO FLOW

Testing begin= 10400  
ACT. VOL.= 2.84  
PACKING TORQUE= 1058  
BUILDING PRESSURE= 0

Testing ending= 8420  
ACT. YOKE RADIUS= 2.63  
SEAL TORQUE= 2396

TEMPERATURE= 70  
ACT. PRESS= 80  
Dstem= 1.75

SOL. VALVE Cv= 8.48  
Dgage= 16.654

TIME sec	TORQUE tend to open	TORQUE air	TORQUE spring	TORQUE flow	TORQUE packing & seal	TORQUE bearing	ANGLE degrees	DELTA P psi
0.00	6903	16805	-10390	-0	1058	1431	90.00	49.99
0.05	7029	14931	-10390	-0	1058	1431	90.00	49.99
0.10	5550	13451	-10390	-0	1058	1431	90.00	49.99
0.15	4349	12250	-10390	-0	1058	1431	90.00	49.99
0.20	3353	11254	-10390	-0	1058	1431	90.00	49.99
0.25	2512	10414	-10390	-0	1058	1431	90.00	49.99
0.30	1793	9694	-10390	-0	1058	1431	90.00	49.99
0.35	1169	9070	-10390	-0	1058	1431	90.00	49.99
0.40	623	8524	-10390	-0	1058	1431	90.00	49.99
0.45	141	8042	-10390	-0	1058	1431	90.00	49.99
0.50	0	7613	-10390	-0	1058	1431	90.00	49.99
0.55	0	6382	-8871	-0	1058	1431	85.41	49.99
0.60	0	5166	-7655	-0	1058	1431	80.36	49.99
0.65	0	4252	-6741	-0	1058	1431	75.13	49.99
0.70	0	3577	-6067	-0	1058	1431	69.83	49.99
0.75	0	3090	-5580	-0	1058	1431	64.56	49.99
0.80	0	2748	-5237	-0	1058	1431	59.41	49.99
0.85	0	2519	-5008	-0	1058	1431	54.40	49.99
0.90	0	2378	-4868	-0	1058	1431	49.57	49.99
0.95	0	2309	-4799	-0	1058	1431	44.90	49.99
1.00	0	2197	-4687	-0	1058	1431	40.39	49.99
1.05	0	2144	-4634	-0	1058	1431	36.13	49.99
1.10	0	2138	-4628	-0	1058	1431	32.09	49.99
1.15	0	2173	-4603	-0	1058	1431	28.25	49.99
1.20	0	2243	-4733	-0	1058	1431	24.60	49.99
1.25	0	2346	-4835	-0	1058	1431	21.11	49.99
1.30	0	2477	-4967	-0	1058	1431	17.79	49.99
1.35	0	2636	-5126	-0	1058	1431	14.63	49.99
1.40	0	2821	-5311	-0	1058	1431	11.63	49.99
1.45	0	3030	-5520	-0	1058	1431	8.79	49.99
1.50	0	3260	-5749	-0	1058	1431	6.11	49.99
1.55	0	3508	-5998	-0	1058	1431	3.59	49.99
1.60	0	1937	-6262	-0	2893	1431	1.23	49.99
1.65	0	1553	-6403	-0	3418	1431	0.08	49.99

NUCLEAR

## VALVE ASSEMBLY CYCLE TEST REPORT

NUCLEAR

PSI VALVE SERIAL NO.		TRAVELER NO.	
14758-2-A		78-14758-02-0800	
CUSTOMER	PURCHASE ORDER NO.	ITEM	TAG NO.
Texas Utilities	CP-0086		CPI-VAD PBC - 09 (1-HV-5549)
OPERATOR TYPE	MANUFACTURER	SERIAL NO.	
# 26072 SR-80	Matryx	21927-1-2	

ACCESSORIES: As Listed per Spec. Sheet

CYCLE TEST REQUIREMENTS PER SPECIFICATION: 14758 TP-4, Rev. -

## TEST CONDITIONS

1. Each valve shall be cycled four (4) times with its operator to insure that no binding or unusual operating characteristics exist. During this test, the closing time shall be five (5) seconds or less. No pressure on the valve is required during this test.
2. Each valve shall be cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure shall be dumped to atmosphere. The valves shall be cycled using the specified actuators.
3. For all cycle testing, the air pressure to pneumatic actuators shall be 80 PSIG and electric actuators shall use available standard electrical service.

## TEST RESULTS

1. First Closing: 1.5 Seconds  
Second Closing: 1.5 Seconds  
Third Closing: 1.5 Seconds  
Fourth Closing: 1.5 Seconds
2. Each valve and its actuator were cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure was dumped to atmosphere.
3. The cycle test was performed and completed satisfactorily to the requirements per PSI Specification No. 14758 TP-4, -.

TESTED BY	DATE	INSPECTED BY	DATE
R. Giles	7-31-78	F. Rose	7-31-78
WITNESSED BY	DATE	AUTHORIZED INSPECTOR	DATE



CLEAR

## VALVE ASSEMBLY CYCLE TEST REPORT

NUCLEAR

PSI VALVE SERIAL NO.		TRAVELER NO.	
14758-2-B		78-14758-02-0800	
CUSTOMER	PURCHASE ORDER NO.	ITEM	TAG NO.
Texas Utilities	CP-0086		CPI-VADPC-10 (1-HV-5548)
OPERATOR TYPE	MANUFACTURER	SERIAL NO.	
# 26072 SR-80	Matryx	21927-1-1	

ACCESSORIES: As Listed per Spec. Sheet

CYCLE TEST REQUIREMENTS PER SPECIFICATION: 14758 TP-4, Rev. -

## TEST CONDITIONS

1. Each valve shall be cycled four (4) times with its operator to insure that no binding or unusual operating characteristics exist. During this test, the closing time shall be five (5) seconds or less. No pressure on the valve is required during this test.
2. Each valve shall be cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure shall be dumped to atmosphere. The valves shall be cycled using the specified actuators.
3. For all cycle testing, the air pressure to pneumatic actuators shall be 80 PSIG and electric actuators shall use available standard electrical service.

## TEST RESULTS

1. First Closing:   /   Seconds  
Second Closing:   /   Seconds  
Third Closing:   /   Seconds  
Fourth Closing:   /   Seconds
2. Each valve and its actuator were cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure was dumped to atmosphere.
3. The cycle test was performed and completed satisfactorily to the requirements per PSI Specification No. 14758 TP-4, -.

TESTED BY	DATE	INSPECTED BY	DATE
E. GEARS	7/31/78	D. G. [Signature]	7/31/78
WITNESSED BY	DATE	AUTHORIZED INSPECTOR	DATE



NUCLEAR

## VALVE ASSEMBLY CYCLE TEST REPORT

NUCLEAR

PSI VALVE SERIAL NO.

TRAVELER NO.

14759-2-A

79-1459-02-0800

CUSTOMER

PURCHASE ORDER NO.

ITEM

TAG NO.

Texas Utilities

CP-0086

CP2-VADPC-

OPERATOR TYPE

MANUFACTURER

SERIAL NO.

#26072 SR80

MATRYX

25377-2

ACCESSORIES: As Listed per Spec. Sheet

CYCLE TEST REQUIREMENTS PER SPECIFICATION: 14758 TP-4, -

## TEST CONDITIONS

1. Each valve shall be cycled four (4) times with its operator to insure that no binding or unusual operating characteristics exist. During this test, the closing time shall be five (5) seconds or less. No pressure on the valve is required during this test.
2. Each valve shall be cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure shall be dumped to atmosphere. The valves shall be cycled using the specified actuators.
3. For all cycle testing, the air pressure to pneumatic actuators shall be 80 PSIG and electric actuators shall use available standard electrical service.

## TEST RESULTS

1. Failure Mode: Close
2. First Closing: 2.5 Seconds  
Second Closing: 2.5 Seconds  
Third Closing: 2.5 Seconds  
Fourth Closing: 2.5 Seconds
3. Each valve and its actuator were cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure was dumped to atmosphere.
4. The cycle test was performed and completed satisfactorily to the requirements per PSI Specification No. 14758 TP-4, -.

TESTED BY

DATE

INSPECTED BY

DATE

W. Debigare

1-11-80

7. Brown

1-11-80

WITNESSED BY

DATE

AUTHORIZED INSPECTOR

DATE

NUCLEAR

## VALVE ASSEMBLY CYCLE TEST REPORT

NUCLEAR

PSI VALVE SERIAL NO.

TRAVELER NO.

14759-2-B

79-1459-02-0800

CUSTOMER

PURCHASE ORDER NO.

ITEM

TAG NO.

Texas Utilities

CP-0086

CP2-VADPC-

OPERATOR TYPE

MANUFACTURER

SERIAL NO.

#26072 SR80

MATRYX

25377 1

ACCESSORIES: As Listed per Spec. Sheet

CYCLE TEST REQUIREMENTS PER SPECIFICATION: 14758 TP-4, -

## TEST CONDITIONS

1. Each valve shall be cycled four (4) times with its operator to insure that no binding or unusual operating characteristics exist. During this test, the closing time shall be five (5) seconds or less. No pressure on the valve is required during this test.
2. Each valve shall be cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure shall be dumped to atmosphere. The valves shall be cycled using the specified actuators.
3. For all cycle testing, the air pressure to pneumatic actuators shall be 80 PSIG and electric actuators shall use available standard electrical service.

## TEST RESULTS

1. Failure Mode: Close
2. First Closing: 2.5 Seconds  
Second Closing: 2.5 Seconds  
Third Closing: 2.5 Seconds  
Fourth Closing: 2.5 Seconds
3. Each valve and its actuator were cycled once with a 50 PSIG pressure applied to one (1) side of the disc. Upon opening, the pressure was dumped to atmosphere.
4. The cycle test was performed and completed satisfactorily to the requirements per PSI Specification No. 14758 TP-4, -.

TESTED BY

DATE

INSPECTED BY

DATE

W. Debigare

1/25/80

D. H. Lina

1/25/80

WITNESSED BY

DATE

AUTHORIZED INSPECTOR

DATE

APPENDIX E

Seismic and LOCA Stress Analyses

# NUCLEAR LOCA & SEISMIC ANALYSIS

VALVE SIZE: 18" Valve V-10  
 VALVE CLASS: ISO  
 ACTUATOR: Matryx 26072-SR80  
 CUSTOMER: Texas Utilities  
 P.O. NO.: CPF-12296-S  
 SPEC. NO.: 2323-SS-20  
 REFERENCE NO.: 34977 (14758 & 1)  
 ITEM NO.: 1 (2)

## REFERENCE DWGS.

A. ASS'Y DWG. NO.	<u>14758-1</u>	REV.	<u>D</u>
B. BODY DWG. NO.	<u>9144-180-3</u>	REV.	<u>-</u>
C. DISC DWG. NO.	<u>1144-180-9</u>	REV.	<u>B</u>
D. STEM DWG. NO.	<u>1304-362</u>	REV.	<u>-</u>
E. PIN DWG. NO.	<u>1306-14</u>	REV.	<u>A</u>
F. BRACKET DWG. NO.	<u>12300</u>	REV.	<u>-</u>

## ALLOWABLE STRESSES

A. BODY	<u>SASIS GR 70</u>	<u>26250</u>	PSI
B. STEM	<u>SA 564 GR 630 Cond H 1100</u>	<u>52500</u>	PSI
C. PIN	<u>SA 564 GR 630 Cond H 1100</u>	<u>35000</u>	PSI
D. BRACKET	<u>A 500</u>	<u>41400</u>	PSI
E. BOLTING	<u>A 193 GR B7</u>	<u>94500</u>	PSI
F. DISC	<u>SA 351 GR CF8M</u>	<u>24750</u>	PSI

## DESIGN CONDITIONS

PRESSURE:	<u>50</u>	PSI
TEMPERATURE:	<u>300</u>	DEG. F.
FLOW PRESS. DROP:	<u>0.17</u>	PSI

## VALVE TORQUES

LOCA TORQUE:	<u>20609</u>	IN-LBS	<u>Case 3B</u>
MEDIA:	<u>Air</u>		<u>90°</u>
FLOW DIRECTION:	<u>Preferred</u>		<u>Page B-51</u>

## G-LOADINGS

TRANSVERSE:	<u>3% Damping</u>	<u>2.6 x 1.5 = 3.9</u>
VERTICAL:	<u>1</u>	<u>(3.2+1) x 1.5 = 6.3</u>
LONGITUDINAL:	<u>SSB</u>	<u>2.6 x 1.5 = 3.9</u>

## OCCASIONAL LOAD

TRANSVERSE:	<u>0</u>	LBS
VERTICAL:	<u>500</u>	LBS

# DIMENSIONAL DATA

SIZE 18

CLASS 130

ACTUATOR Matryx 26072-SR80

## DESCRIPTION OF VARIABLE

INPUT COMPUTER  
NAME NAME

TRANSVERSE DIST. ACTUATOR C.G. TO $\phi$ VALVE	3.32	X1	X1
LONGITUDINAL DIST. ACTUATOR C.G. TO $\phi$ VALVE	0.63	X2	X2
VERTICAL DIST. ACTUATOR C.G. TO BRACKET	6.48	X3	X3
HEIGHT BRACKET	4	X4	X4
HEIGHT VALVE NECK	3	X5	X5
ACTUATOR WEIGHT	491	W1	W4
BRACKET WEIGHT	11	W2	W5
DISC WEIGHT	123	W4	W3
THICKNESS OF BRACKET LOWER PLATE	.5	T1	T1
WIDTH OF BRACKET	4.5	T2	T2
WIDTH OF VALVE NECK	7	T3	T3
THICKNESS OF VALVE NECK	3.688	T4	T4
THICKNESS OF BRACKET SIDE PLATES	1.5	T9	T9
THICKNESS OF BRACKET TOP PLATE	.5	T0	T0
VALVE NECK O.D.	0	d3	Do
PACKING BORE I.D.	2.5	D1	E5
STEM DIA.	1.75	D1	D1
GAGE DIA. OF DISC	16.654	D	D
WIDTH SMALL DIA. BACK OF DISC	2.438	E1	E1
WIDTH LARGE DIA. OF DISC	18/3	E2	E2
DISC SPHERICAL INDENTATION DIA.	12.25	D2	D2
DIST. BETWEEN T-WASHER FLATS ON DISC	16.34	L3	L3
STEM ENGAGEMENT IN DISC	2.625	L4	L4
DEPTH OF SPHERICAL INDENTATION IN DISC	.8/3	E4	E4
LARGE DIA. OF CONICAL SECTION OF DISC	15.75	D3	D3
SMALL DIA. OF CONICAL SECTION OF DISC	3.375	D4	D4
THRUST WASHER THICKNESS	.348	L2	L2
DIST. $\phi$ STEM TO FRONT OF DISC	1.8/2	Y2	Y2
NO. OF ACTUATOR BOLTS	4	N1	N1
TENSILE STRESS AREA OF ACTUATOR BOLTS	1.1416	A1	A1
DBC OF ACTUATOR BOLTS	4.5	X6	X6
NO. OF BRACKET/VALVE BOLTS	4	N2	N2
TENSILE STRESS AREA OF BRACKET/VALVE BOLTS	.334	A2	A2
TRANSVERSE DIST. BETWEEN BRACKET BOLTS	5	X7	X7
LONGITUDINAL DIST. BETWEEN BRACKET BOLTS	2	X8	X8
LENGTH OF BRACKET	8	T5	T5
DISTANCE BETWEEN VALVE BODY BOLT HOLES	0	R5	R5
DIA. OF VALVE BODY BOLT HOLES	0	R6	R6
% TORQUE ON DISC PIN	6.5	%	T7
DISC PIN DIA.	1.496	dp	D5
VALVE BODY O.D.	2.5	d1	D8
VALVE BODY WATERWAY DIA.	17	R7	R7
ADJACENT PIPING O.D.	18	R8	R8
ADJACENT PIPING I.D.	16.876	R9	R9
LENGTH ACROSS GUSSETS	0	L	L
THICKNESS OF GUSSETS	0	T	T
DIA. OF NECK FLANGE	0	D6	D6
THICKNESS OF NECK FLANGE	0	T6	T6
MODULUS OF ELASTICITY	30,000,000	E	E
MAXIMUM PIPING BENDING MOMENT	N/A	M4	M4
NO. OF BODY BOLTS	1	N3	N3
DEC OF BODY BOLTS	1	X0	X0
ROOT AREA OF BODY BOLTS	1	E0	E0

B

# NUCLEAR SEISMIC ANALYSIS

18 CLASS 150 VALVE ASSEMBLY  
WITH 26072-SR80 ACTUATOR

CUSTOMER: TEXAS UTILITIES  
P.O. NO. CPF-12296-S  
SPEC. NO.: 3323-SS-20  
REF. NO.: 34977  
ITEM NO.: 1

REFERENCE DWGS.	ALLOWABLE STRESSES (PSI)	DESIGN & FLOW CONDITIONS
A. ASS'Y DWG. NO.: 14758-1 REV. D	A. BODY: 26250	
B. BODY DWG. NO.: 9144-180-3 REV. -	B. STEM: 52500	SHUT-OFF PRESS. (PSI) = 50
C. DISC DWG. NO.: 1144-180-9 REV. G	C. PIN: 35000	TEMPERATURE (°F) = 300
D. STEM DWG. NO.: 1304-362 REV. -	D. BRACKET: 41400	FLOW PRESS. DROP (PSI) = 6.17
E. PIN DWG. NO.: 1306-14 REV. A	E. BOLTING: 94500	
F. BRACKET DWG. NO. 12300 REV. -	F. DISC: 24750	

LOCA TORQUE = 20509	G LOADINGS	OCCASSIONAL LOADS
MEDIA = AIR	TRANSVERSE = 1.7	TRANSVERSE = 2
FLOW DIRECTION = PREFERRED	VERTICAL = 6.3	VERTICAL = 500
	LONGITUDINAL = 3.2	

DIMENSIONAL DATA					
X1 = 3.32	X2 = .63	X3 = 6.48	X4 = 4	X5 = 3	W1 = 481
W4 = 131	T1 = .5	T2 = 4.5	T3 = 7	T4 = 3.788	T9 = .5
d3 = 0	D1 = 2.5	D1 = 1.75	D = 16.654	E1 = 2.438	E2 = .813
D2 = 12.25	L3 = 16.34	L4 = 2.625	E4 = .813	D3 = 15.75	D4 = 3.375
L2 = .348	Y2 = 1.812	N1 = 4	A1 = .1416	X6 = 4.5	N2 = 4
A2 = .334	X7 = 5	X8 = 2	T5 = 8	R5 = 0	R6 = 0
Z = 65	dp = .496	d1 = 25	R7 = 17	R8 = 18	R9 = 16.876
L = 0	T = 0	Da = 0	T6 = 0	E = 30000000	

NATURAL FREQUENCIES (HZ.)	ACTUATOR BOLT STRESSES
LONGITUDINAL (Z) ACT./VALVE = 17 VS. 33HZ.	SHEAR = 23841 PSI
TRANSVERSE (Y) ACT./VALVE = 40 VS. 33HZ.	TENSILE = 28898 PSI
LATERAL DISC/STEM = 411 VS. 33HZ.	COMBINED = 42328 PSI VS. ALLOW. = 94500

BRACKET BOLT STRESSES	BRACKET STRESSES
SHEAR = 8629 PSI	SHEAR = 9856 PSI
TENSILE = 18044 PSI	TENSILE = 19317 PSI
COMBINED = 21523 PSI VS. ALLOW. = 94500	COMBINED = 23458 PSI VS. ALLOW. = 41400

VALVE NECK STRESSES	STEM STRESSES
SHEAR = 1358 PSI	SHEAR = 19584 PSI
TENSILE = 2292 PSI	TENSILE = 4608 PSI
COMBINED = 2923 PSI VS. ALLOW. = 26250	COMBINED = 22023 PSI VS. ALLOW. = 52500

DISC PIN STRESS	SECTION MODULUS
SHEAR = 35790 PSI VS. ALLOW. = 21000	VALVE = 1205.99 IN <sup>3</sup>
	PIPING = 130.16 IN <sup>3</sup>

ACTUATOR DEFLECTIONS	BODY BOLTING
LONGITUDINAL = .242729 INCHES	NOT APPLICABLE
TRANSVERSE = 4.37000000E-02 INCHES	

WITH FLOW	DISC SHUT
BENDING STRESS, Sx = 1989	BENDING STRESS, Sx = 2032
BENDING STRESS, Sy = 1910	BENDING STRESS, Sy = 1509
COMBINE STRESS = 3376 PSI VS. ALLOW. = 24750	COMBINE STRESS = 4856 PSI VS. ALLOW. = 24750

SIGNED.....*John B. Long*.....DATED...4/15/85...



# NUCLEAR LOCA & SEISMIC ANALYSIS

VALVE SIZE: 18 Valve - V10  
 VALVE CLASS: 150  
 ACTUATOR: Malloy 26072-SR80  
 CUSTOMER: Texas Utilities  
 P.O. NO.: CPF-12296-S  
 SPEC. NO.: 2323-SS-20  
 REFERENCE NO.: 34977 (14758 & 9)  
 ITEM NO.: 1 (2)

## REFERENCE DWGS.

A. ASS'Y DWG. NO.	<u>14758-1</u>	REV.	<u>D</u>
B. BODY DWG. NO.	<u>9144-180-3</u>	REV.	<u>-</u>
C. DISC DWG. NO.	<u>1144-180-9</u>	REV.	<u>B</u>
D. STEM DWG. NO.	<u>1304-362</u>	REV.	<u>-</u>
E. PIN DWG. NO.	<u>1306-14</u>	REV.	<u>A</u>
F. BRACKET DWG. NO.	<u>12300</u>	REV.	<u>-</u>

## ALLOWABLE STRESSES

A. BODY	<u>SASIS GR 70</u>	<u>26250</u>	PSI
B. STEM	<u>SA 564 GR 630 Cond H 1100</u>	<u>52500</u>	PSI
C. PIN	<u>SA 564 GR 630 Cond H 1100</u>	<u>35000</u>	PSI
D. BRACKET	<u>A 500</u>	<u>41400</u>	PSI
E. BOLTING	<u>A 193 GR B7</u>	<u>94500</u>	PSI
F. DISC	<u>SA 351 GR CF8M</u>	<u>24750</u>	PSI

## DESIGN CONDITIONS

PRESSURE:	<u>50</u>	PSI
TEMPERATURE:	<u>300</u>	DEG. F.
FLOW PRESS. DROP:	<u>19.03</u>	PSI

## VALVE TORQUES

LOCA TORQUE:	<u>12198</u>	IN-LBS	<u>Case 2 B</u>
MEDIA:	<u>Air</u>		<u>70°</u>
FLOW DIRECTION:	<u>Pressure</u>		<u>B-33</u>

## G-LOADINGS

TRANSVERSE:	<u>3% Damping</u>	<u>2.6 x 1.5 = 3.9</u>
VERTICAL:		<u>(2.2+1) x 1.5 = 6.3</u>
LONGITUDINAL:	<u>5SE</u>	<u>2.6 x 1.5 = 3.9</u>

## OCCASIONAL LOAD

TRANSVERSE:	<u>0</u>	LBS
VERTICAL:	<u>500</u>	LBS

DIMENSIONAL DATA

SIZE 18

CLASS 150

ACTUATOR Matryx 2C072-5P80

DESCRIPTION OF VARIABLE

INPUT COMPUTER  
NAME NAME

TRANSVERSE DIST. ACTUATOR C.G. TO $\phi$ VALVE	3.32	X1	X1
LONGITUDINAL DIST. ACTUATOR C.G. TO $\phi$ VALVE	0.63	X2	X2
VERTICAL DIST. ACTUATOR C.G. TO BRACKET	6.48	X3	X3
HEIGHT BRACKET	4	X4	X4
HEIGHT VALVE NECK	3	X5	X5
ACTUATOR WEIGHT	491	W1	W4
BRACKET WEIGHT	11	W2	W5
DISC WEIGHT	123	W4	W3
THICKNESS OF BRACKET LOWER PLATE	.5	T1	T1
WIDTH OF BRACKET	4.5	T2	T2
WIDTH OF VALVE NECK	7	T3	T3
THICKNESS OF VALVE NECK	3.688	T4	T4
THICKNESS OF BRACKET SIDE PLATES	.15	T9	T9
THICKNESS OF BRACKET TOP PLATE	.15	T0	T0
VALVE NECK O.D.	0	d3	D0
PACKING BORE I.D.	2.5	D1	B5
STEM DIA.	1.75	D1	D1
GAGE DIA. OF DISC	16.654	D	D
WIDTH SMALL DIA. BACK OF DISC	2.438	E1	E1
WIDTH LARGE DIA. OF DISC	.813	E2	E2
DISC SPHERICAL INDENTATION DIA.	12.25	D2	D2
DIST. BETWEEN T-WASHER FLATS ON DISC	16.34	L3	L3
STEM ENGAGEMENT IN DISC	2.625	L4	L4
DEPTH OF SPHERICAL INDENTATION IN DISC	.813	E4	E4
LARGE DIA. OF CONICAL SECTION OF DISC	15.75	D3	D3
SMALL DIA. OF CONICAL SECTION OF DISC	3.375	D4	D4
THRUST WASHER THICKNESS	.348	L2	L2
DIST. $\phi$ STEM TO FRONT OF DISC	1.812	Y2	Y2
NO. OF ACTUATOR BOLTS	4	N1	N1
TENSILE STRESS AREA OF ACTUATOR BOLTS	1.416	A1	A1
DBC OF ACTUATOR BOLTS	4.5	X6	X6
NO. OF BRACKET/VALVE BOLTS	4	N2	N2
TENSILE STRESS AREA OF BRACKET/VALVE BOLTS	.334	A2	A2
TRANSVERSE DIST. BETWEEN BRACKET BOLTS	5	X7	X7
LONGITUDINAL DIST. BETWEEN BRACKET BOLTS	2	X8	X8
LENGTH OF BRACKET	8	T5	T5
DISTANCE BETWEEN VALVE BODY BOLT HOLES	0	R5	R5
DIA. OF VALVE BODY BOLT HOLES	0	R6	R6
% TORQUE ON DISC PIN	.65	%	T7
DISC PIN DIA.	1.496	dp	D5
VALVE BODY O.D.	2.5	d1	D8
VALVE BODY WATERWAY DIA.	17	R7	R7
ADJACENT PIPING O.D.	18	R8	R8
ADJACENT PIPING I.D.	11.876	R9	R9
LENGTH ACROSS GUSSETS	0	L	L
THICKNESS OF GUSSETS	0	T	T
DIA. OF NECK FLANGE	0	D6	D6
THICKNESS OF NECK FLANGE	0	T6	T6
MODULUS OF ELASTICITY	30,000,000	E	E
MAXIMUM PIPING BENDING MOMENT	N/A	M4	M4
NO. OF BODY BOLTS	1	N3	N3
DBC OF BODY BOLTS	1	X0	X0
ROOT AREA OF BODY BOLTS	1	E0	E0



# POSI-SEAL INTERNATIONAL, INC. NUCLEAR SEISMIC ANALYSIS

18 CLASS 150 VALVE ASSEMBLY  
WITH 26072-SR80 ACTUATOR

CUSTOMER: TEXAS UTILITIES  
P.O. NO. CPF-12296-S  
SPEC. NO. 12323-SS-20  
REF. NO. 14977  
ITEM NO. 1

REFERENCE DWGS.  
A. ASS'Y DWG. NO. 14758-1 REV. D  
B. BODY DWG. NO. 1144-180-3 REV. -  
C. DISC DWG. NO. 1144-180-9 REV. 0  
D. STEM DWG. NO. 1304-362 REV. -  
E. PIN DWG. NO. 1308-14 REV. A  
F. BRACKET DWG. NO. 12300 REV. -

ALLOWABLE STRESSES (PSI)  
A. BODY: 26250  
B. STEM: 52500  
C. PIN: 35000  
D. BRACKET: 41400  
E. BOLTING: 94500  
F. DISC: 24750

DESIGN & FLOW CONDITIONS  
SHUT-OFF PRESS. (PSI) = 50  
TEMPERATURE (°F) = 300  
FLOW PRESS. DROP (PSI) = 19.03

LOCA TORQUE = 12198  
MEDIA = AIR  
FLOW DIRECTION = PREFERRED

C LOADINGS  
TRANSVERSE = 3.7  
VERTICAL = 5.3  
LONGITUDINAL = 3.7

SECCASSIONAL LOADS  
TRANSVERSE = 0  
VERTICAL = 500

## DIMENSIONAL DATA

X1 = 3.32	X2 = .63	X3 = 6.40	X4 = 4	X5 = 3	W1 = .81	W2 = .11
W4 = 133	T1 = .5	T2 = 4.5	T3 = 7	T4 = 3.688	T9 = .5	T0 = .5
d3 = 0	D1 = 2.5	D1 = 1.75	D = 16.654	E1 = 2.438	E2 = .813	
D2 = 12.25	L3 = 16.34	L4 = 2.625	E4 = .813	D3 = 15.75	D4 = 3.375	
L2 = .348	Y2 = 1.812	N1 = 4	A1 = .1416	X6 = 4.5	N2 = 4	
A2 = .334	X7 = 5	X8 = 2	T5 = 8	R5 = 0	T6 = 0	
Z = 65	dp = .496	d1 = 25	R7 = 17	R8 = 18	R9 = 16.876	
L = 0	T = 0	D6 = 0	T6 = 0	E = 30000000		

NATURAL FREQUENCIES (HZ.)  
LONGITUDINAL (Z) ACT./VALVE = 17 VS. 33HZ.  
TRANSVERSE (Y) ACT./VALVE = 40 VS. 33HZ.  
LATERAL DISC/STEM = 411 VS. 334Z.

ACTUATOR BOLT STRESSES  
SHEAR = 17274 PSI  
TENSILE = 28898 PSI  
COMBINED = 36970 PSI VS. ALLOW. = 94500

## BRACKET BOLT STRESSES

SHEAR = 8404 PSI  
TENSILE = 18064 PSI  
COMBINED = 20104 PSI VS. ALLOW. = 94500

## BRACKET STRESSES

SHEAR = 6557 PSI  
TENSILE = 16825 PSI  
COMBINED = 19079 PSI VS. ALLOW. = 41400

## VALVE NECK STRESSES

SHEAR = 976 PSI  
TENSILE = 2292 PSI  
COMBINED = 2651 PSI VS. ALLOW. = 26250

## STEM STRESSES

SHEAR = 11591 PSI  
TENSILE = 4608 PSI  
COMBINED = 14122 PSI VS. ALLOW. = 52500

## DISC PIN STRESS

SHEAR = 21183 PSI VS. ALLOW. = 21000

## SECTION MODULUS

VALVE = 1205.99 IN<sup>3</sup>  
PIPING = 130.16 IN<sup>3</sup>

## ACTUATOR DEFLECTIONS

LONGITUDINAL = .242729 INCHES  
TRANSVERSE = 4.37000000E-02 INCHES

## BODY BOLTING

NOT APPLICABLE

## DISC STRESSES

WITH FLOW  
BENDING STRESS, Sx = 2978  
BENDING STRESS, Sy = 2903  
COMBINE STRESS = 5094 PSI VS. ALLOW. = 24750

## DISC SHUT

BENDING STRESS, Sx = 2032  
BENDING STRESS, Sy = 3509  
COMBINE STRESS = 4056 PSI VS. ALLOW. = 24750

SIGNED... *[Signature]* ..... DATED... 10/15/85...

# NUCLEAR LOCA & SEISMIC ANALYSIS

VALVE SIZE: 18" Valve - V10  
 VALVE CLASS: 150  
 ACTUATOR: Matrox 26072-SR80  
 CUSTOMER: Texas Utilities  
 P.O. NO.: CPF-12296-S  
 SPEC. NO.: 2323-SS-20  
 REFERENCE NO.: 34977 (1475819)  
 ITEM NO.: 1 (2)

## REFERENCE DWGS.

A. ASS'Y DWG. NO.	<u>14758-1</u>	REV.	<u>D</u>
B. BODY DWG. NO.	<u>9144-180-3</u>	REV.	<u>-</u>
C. DISC DWG. NO.	<u>1144-180-9</u>	REV.	<u>B</u>
D. STEM DWG. NO.	<u>1304-362</u>	REV.	<u>-</u>
E. PIN DWG. NO.	<u>1306-14</u>	REV.	<u>A</u>
F. BRACKET DWG. NO.	<u>12300</u>	REV.	<u>-</u>

## ALLOWABLE STRESSES

A. BODY	<u>SASIS GR 70</u>	<u>26250</u>	PSI	
B. STEM	<u>SA 564 GR 630 Cond 41100</u>	<u>52500</u>	PSI	
C. PIN	<u>SA 564 GR 630 Cond 41100</u>	<u>35000</u>	PSI	B
D. BRACKET	<u>A 500</u>	<u>41400</u>	PSI	
E. BOLTING	<u>A 193 GR B7</u>	<u>94500</u>	PSI	
F. DISC	<u>SA 351 GR CF8M</u>	<u>24750</u>	PSI	B

## DESIGN CONDITIONS

PRESSURE:	<u>50</u>	PSI
TEMPERATURE:	<u>300</u>	DEG. F.
FLOW PRESS. DROP:	<u>2.6</u>	PSI

## VALVE TORQUES

LOCA TORQUE:	<u>7677</u>	IN-LBS	<u>Case 2 B</u>
MEDIA:	<u>Air</u>		<u>650</u>
FLOW DIRECTION:	<u>Preferred</u>		<u>C-12</u>

## G-LOADINGS

TRANSVERSE:	<u>3% Damping</u>	<u>2.6 x 1.5 = 3.9</u>
VERTICAL:	<u>1</u>	<u>(2.2+1) x 1.5 = 6.3</u>
LONGITUDINAL:	<u>SSS</u>	<u>2.6 x 1.5 = 3.9</u>

## OCCASIONAL LOAD

TRANSVERSE:	<u>0</u>	LBS
VERTICAL:	<u>500</u>	LBS

DIMENSIONAL DATA

SIZE 18" CLASS 150  
 ACTUATOR Matryx 26072-SR80

DESCRIPTION OF VARIABLE	INPUT COMPUTER	
	NAME	NAME
TRANSVERSE DIST. ACTUATOR C.G. TO & VALVE	3.32	X1
LONGITUDINAL DIST. ACTUATOR C.G. TO & VALVE	0.63	X2
VERTICAL DIST. ACTUATOR C.G. TO BRACKET	6.48	X3
HEIGHT BRACKET	4	X4
HEIGHT VALVE NECK	3	X5
ACTUATOR WEIGHT	491	W1
BRACKET WEIGHT	11	W2
DISC WEIGHT	133	W4
THICKNESS OF BRACKET LOWER PLATE	.5	T1
WIDTH OF BRACKET	4.5	T2
WIDTH OF VALVE NECK	7	T3
THICKNESS OF VALVE NECK	3.688	T4
THICKNESS OF BRACKET SIDE PLATES	.15	T9
THICKNESS OF BRACKET TOP PLATE	.15	T0
VALVE NECK O.D.	0	d3
PACKING BORE I.D.	2.5	D1
STEM DIA.	1.75	D1
GAGE DIA. OF DISC	16.654	D
WIDTH SMALL DIA. BACK OF DISC	2.438	E1
WIDTH LARGE DIA. OF DISC	.813	E2
DISC SPHERICAL INDENTATION DIA.	12.25	D2
DIST. BETWEEN T-WASHER FLATS ON DISC	16.34	L3
STEM ENGAGEMENT IN DISC	2.625	L4
DEPTH OF SPHERICAL INDENTATION IN DISC	.813	E4
LARGE DIA. OF CONICAL SECTION OF DISC	15.75	D3
SMALL DIA. OF CONICAL SECTION OF DISC	3.375	D4
THRUST WASHER THICKNESS	.348	L2
DIST. & STEM TO FRONT OF DISC	1.812	Y2
NO. OF ACTUATOR BOLTS	4	N1
TENSILE STRESS AREA OF ACTUATOR BOLTS	1.1416	A1
DBC OF ACTUATOR BOLTS	4.5	X6
NO. OF BRACKET/VALVE BOLTS	4	N2
TENSILE STRESS AREA OF BRACKET/VALVE BOLTS	.334	A2
TRANSVERSE DIST. BETWEEN BRACKET BOLTS	.5	X7
LONGITUDINAL DIST. BETWEEN BRACKET BOLTS	2	X8
LENGTH OF BRACKET	8	T5
DISTANCE BETWEEN VALVE BODY BOLT HOLES	0	R5
DIA. OF VALVE BODY BOLT HOLES	0	R6
% TORQUE ON DISC PIN	65	T7
DISC PIN DIA.	1.496	dp
VALVE BODY O.D.	25	d1
VALVE BODY WATERWAY DIA.	17	R7
ADJACENT PIPING O.D.	18	R8
ADJACENT PIPING I.D.	16.876	R9
LENGTH ACROSS GUSSETS	0	L
THICKNESS OF GUSSETS	0	T
DIA. OF NECK FLANGE	0	D6
THICKNESS OF NECK FLANGE	0	T6
MODULUS OF ELASTICITY	30,000,000	E
MAXIMUM PIPING BENDING MOMENT	1/P.F.	M4
NO. OF BODY BOLTS	1	N3
DBC OF BODY BOLTS		X0
ROOT AREA OF BODY BOLTS		E0

B

# NUCLEAR SEISMIC ANALYSIS

18 CLASS 150 VALVE ASSEMBLY  
WITH 26072-SR80 ACTUATOR

CUSTOMER: XAS UTILITIES  
P.O. NO. CPF-12296-S  
SPEC. NO.: 2323-SS-20  
REF. NO.: 34977  
ITEM NO.: 1

REFERENCE DWGS.  
A. ASS'Y DWG. NO.: 14758-1 REV. D  
B. BODY DWG. NO.: 9144-180-3 REV. -  
C. DISC DWG. NO.: 1144-180-9 REV. 0  
D. STEM DWG. NO.: 1304-362 REV. -  
E. PIN DWG. NO.: 1306-14 REV. A  
F. BRACKET DWG. NO.: 12300 REV. -

ALLOWABLE STRESSES (PSI)  
A. BODY: 26250  
B. STEM: 52500  
C. PIN: 35000  
D. BRACKET: 41400  
E. BOLTING: 94500  
F. DISC: 24750

DESIGN & FLOW CONDITIONS  
SHUT-OFF PRESS. (PSI) = 50  
TEMPERATURE (°F) = 300  
FLOW PRESS. DROP (PSI) = 26.6

LOCK TORQUE = 7877  
MEDIA = AIR  
FLOW DIRECTION = PREFERRED

C LOADINGS  
TRANSVERSE = 3.7  
VERTICAL = 6.3  
LONGITUDINAL = 3.7

OCCASIONAL LOADS  
TRANSVERSE = 1  
VERTICAL = 500

DIMENSIONAL DATA									
X1 = 3.32	X2 = .63	X3 = 6.48	X4 = 4	X5 = 3	W1 = 481	W2 = 11			
W4 = 133	Y1 = .5	Y2 = 4.5	Y3 = 7	Y4 = 3.588	Y5 = .5	Y6 = .5			
d3 = 0	D1 = 2.5	D1 = 1.75	D = 16.654	E1 = 2.438	E2 = .813				
D2 = 12.25	L3 = 16.34	L4 = 2.625	E4 = .813	D3 = 15.75	D4 = 3.375				
L2 = .348	Y2 = 1.812	N1 = 4	A1 = .1416	X6 = 4.5	N2 = 4				
A2 = .334	X7 = 5	X8 = 2	T5 = 8	R5 = 0	R6 = 0				
X = 65	dp = .496	d1 = 25	R7 = 17	R8 = 18	R9 = 16.876				
L = 0	T = 0	D6 = 0	T6 = 0	E = 30000000					

NATURAL FREQUENCIES (HZ.)  
LONGITUDINAL (Z) ACT./VALVE = 17 VS. 33HZ.  
TRANSVERSE (Y) ACT./VALVE = 40 VS. 33HZ.  
LATERAL DISC/STEM = 411 VS. 33HZ.

ACTUATOR BOLT STRESSES  
SHEAR = 13756 PSI  
TENSILE = 28898 PSI  
COMBINED = 34400 PSI VS. ALLOW. = 94500

BRACKET BOLT STRESSES  
SHEAR = 5248 PSI  
TENSILE = 18064 PSI  
COMBINED = 19478 PSI VS. ALLOW. = 94500

BRACKET STRESSES  
SHEAR = 4784 PSI  
TENSILE = 15485 PSI  
COMBINED = 16844 PSI VS. ALLOW. = 41400

VALVE NECK STRESSES  
SHEAR = 771 PSI  
TENSILE = 2292 PSI  
COMBINED = 2527 PSI VS. ALLOW. = 26250

STEM STRESSES  
SHEAR = 7295 PSI  
TENSILE = 4608 PSI  
COMBINED = 9954 PSI VS. ALLOW. = 52500

DISC PIN STRESS  
SHEAR = 13332 PSI VS. ALLOW. = 21000

SECTION MODULUS  
VALVE = 1205.99 IN<sup>3</sup>  
PIPING = 130.16 IN<sup>3</sup>

ACTUATOR DEFLECTIONS  
LONGITUDINAL = .242727 INCHES  
TRANSVERSE = 4.37000000E-02 INCHES

NOT APPLICABLE BODY BOLTING

WITH FLOW DISC STRESSES  
BENDING STRESS, S<sub>x</sub> = 3584  
BENDING STRESS, S<sub>y</sub> = 3507  
COMBINE STRESS = 6141 PSI VS. ALLOW. = 24750

DISC SHUT  
BENDING STRESS, S<sub>x</sub> = 2032  
BENDING STRESS, S<sub>y</sub> = 3509  
COMBINE STRESS = 4856 PSI VS. ALLOW. = 24750

SIGNED... *[Signature]* ... DATED... 10/15/85...

# NUCLEAR LOCA & SEISMIC ANALYSIS

VALVE SIZE: 18" Valve - V9  
 VALVE CLASS: ISO  
 ACTUATOR: Matryx 26072-SR80  
 CUSTOMER: Texas Utilities  
 P.O. NO.: CPF-12296-S  
 SPEC. NO.: 2323-SS-20  
 REFERENCE NO.: 34977 (14758 #1)  
 ITEM NO.: 1 (2)

## REFERENCE DWGS.

A.	ASS'Y DWG. NO.	<u>14758-1</u>	REV.	<u>D</u>
B.	BODY DWG. NO.	<u>9144-180-3</u>	REV.	<u>-</u>
C.	DISC DWG. NO.	<u>1144-180-9</u>	REV.	<u>B</u>
D.	STEM DWG. NO.	<u>1304-362</u>	REV.	<u>-</u>
E.	PIN DWG. NO.	<u>1306-14</u>	REV.	<u>A</u>
F.	BRACKET DWG. NO.	<u>12300</u>	REV.	<u>-</u>

## ALLOWABLE STRESSES

A.	BODY	<u>SA515 GR 70</u>	<u>26250</u>	PSI	
B.	STEM	<u>SA564 GR 630 Cond H1100</u>	<u>52500</u>	PSI	
C.	PIN	<u>SA564 GR 630 Cond H1100</u>	<u>35000</u>	PSI	B
D.	BRACKET	<u>A500</u>	<u>41400</u>	PSI	
E.	BOLTING	<u>A193 GR B7</u>	<u>94500</u>	PSI	
F.	DISC	<u>SA351 GR CF8M</u>	<u>24250</u>	PSI	B

## DESIGN CONDITIONS

PRESSURE:	<u>50</u>	PSI	
TEMPERATURE:	<u>300</u>	DEG. F.	
FLOW PRESS. DROP:	<u>14.46</u>	PSI	B

## VALVE TORQUES

LOCA TORQUE:	<u>13350</u>	IN-LBS	<u>Case 2A</u>
MEDIA:	<u>Air</u>		<u>70°</u>
FLOW DIRECTION:	<u>Non preferred</u>		<u>Page 53</u>

## G-LOADINGS

TRANSVERSE:	<u>3% Damping</u>	<u>(2.641) x 1.5 = 3.96</u>
VERTICAL:		<u>3.2 x 1.5 = 4.8</u>
LONGITUDINAL:	<u>5SE</u>	<u>2.6 x 1.5 = 3.9</u>

## OCCASIONAL LOAD

TRANSVERSE:	<u>500</u>	LBS	
VERTICAL:	<u>0</u>	LBS	

# DIMENSIONAL DATA

SIZE 18" CLASS 150  
 ACTUATOR Matryx 26072-SR80

DESCRIPTION OF VARIABLE	INPUT COMPUTER		
	NAME	NAME	
TRANSVERSE DIST. ACTUATOR C.G. TO $\phi$ VALVE	3.32	X1	X1
LONGITUDINAL DIST. ACTUATOR C.G. TO $\phi$ VALVE	0.63	X2	X2
VERTICAL DIST. ACTUATOR C.G. TO BRACKET	6.48	X3	X3
HEIGHT BRACKET	4	X4	X4
HEIGHT VALVE NECK	3	X5	X5
ACTUATOR WEIGHT	491	W1	W4
BRACKET WEIGHT	11	W2	W5
DISC WEIGHT	173	W4	W3
THICKNESS OF BRACKET LOWER PLATE	.5	T1	T1
WIDTH OF BRACKET	4.5	T2	T2
WIDTH OF VALVE NECK	7	T3	T3
THICKNESS OF VALVE NECK	7.688	T4	T4
THICKNESS OF BRACKET SIDE PLATES	.5	T9	T9
THICKNESS OF BRACKET TOP PLATE	.5	T0	T0
VALVE NECK O.D.	0	d3	Do
PACKING SCORE I.D.	2.5	D1	B5
STEM DIA.	1.75	D1	D1
GAGE DIA. OF DISC	16.654	D	D
WIDTH SMALL DIA. BACK OF DISC	2.438	E1	E1
WIDTH LARGE DIA. OF DISC	.813	E2	E2
DISC SPHERICAL INDENTATION DIA.	12.35	D2	D2
DIST. BETWEEN T-WASHER FLATS ON DISC	16.34	L3	L3
STEM ENGAGEMENT IN DISC	2.625	L4	L4
DEPTH OF SPHERICAL INDENTATION IN DISC	.813	E4	E4
LARGE DIA. OF CONICAL SECTION OF DISC	15.75	D3	D3
SMALL DIA. OF CONICAL SECTION OF DISC	3.375	D4	D4
THRUST WASHER THICKNESS	.348	L2	L2
DIST. $\phi$ STEM TO FRONT OF DISC	1.812	Y2	Y2
NO. OF ACTUATOR BOLTS	4	N1	N1
TENSILE STRESS AREA OF ACTUATOR BOLTS	1.1416	A1	A1
DBC OF ACTUATOR BOLTS	4.5	X6	X6
NO. OF BRACKET/VALVE BOLTS	4	N2	N2
TENSILE STRESS AREA OF BRACKET/VALVE BOLTS	.334	A2	A2
TRANSVERSE DIST. BETWEEN BRACKET BOLTS	5	X7	X7
LONGITUDINAL DIST. BETWEEN BRACKET BOLTS	2	X8	X8
LENGTH OF BRACKET	6	T5	T5
DISTANCE BETWEEN VALVE BODY BOLT HOLES	0	R5	R5
DIA. OF VALVE BODY BOLT HOLES	0	R6	R6
% TORQUE ON DISC PIN	.65	%	T7
DISC PIN DIA.	1.496	dp	D5
VALVE BODY O.D.	2.5	d1	D8
VALVE BODY WATERWAY DIA.	17	R7	R7
ADJACENT PIPING O.D.	18	R8	R8
ADJACENT PIPING I.D.	16.876	R9	R9
LENGTH ACROSS GUSSETS	0	L	L
THICKNESS OF GUSSETS	0	T	T
DIA. OF NECK FLANGE	0	D6	D6
THICKNESS OF NECK FLANGE	0	T6	T6
MODULUS OF ELASTICITY	30,000,000	E	E
MAXIMUM PIPING BENDING MOMENT	N/A	M4	M4
NO. OF BODY BOLTS	1	N3	N3
DBC OF BODY BOLTS	1	X0	X0
ROOT AREA OF BODY BOLTS	1	E0	E0



# NUCLEAR SEISMIC ANALYSIS

18 CLASS 150 VALVE ASSEMBLY  
WITH 26072-SR80 ACTUATOR

CUSTOMER: TEXAS UTILITIES  
P.O. NO. CPF-12296-S  
SPEC. NO.: 2323-SS-20  
REF. NO.: 34977  
ITEM NO.: 1

REFERENCE DWGS.	ALLOWABLE STRESSES (PSI)	DESIGN & FLOW CONDITIONS
A. ASS'Y DWG. NO.: 14758-1 REV. D	A. BODY: 26250	SHUT-OFF PRESS. (PSI) = 50 TEMPERATURE (°F) = 300 FLOW PRESS. DROP (PSI) = 14.46
B. BODY DWG. NO.: 9144-190-3 REV. -	B. STEM: 52500	
C. DISC DWG. NO.: 1144-190-9 REV. D	C. PIN: 35000	
D. STEM DWG. NO.: 1304-362 REV. -	D. BRACKET: 41400	
E. PIN DWG. NO.: 1306-14 REV. A	E. BOLTING: 94500	
F. BRACKET DWG. NO.: 12300 REV. -	F. DISC: 24750	

LOCAL TORQUE = 13350 MEDIA = AIR FLOW DIRECTION = NONPREFERRED	G. LOADINGS TRANSVERSE = 5.4 VERTICAL = 4.8 LONGITUDINAL = 3.2	OCCASIONAL LOADS TRANSVERSE = 500 VERTICAL = 0
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DIMENSIONAL DATA					
X1 = 3.32	X2 = .63	X3 = 6.48	X4 = 4	X5 = 3	W1 = 481
W4 = 133	T1 = .5	T2 = 4.5	T3 = 7	T4 = 3.598	W2 = 11
d3 = 0	D1 = 2.5	D1 = 1.75	D = 16.654	E1 = 2.438	T9 = .5
D2 = 12.25	L3 = 16.34	L4 = 2.625	E4 = .813	O3 = 15.75	T0 = .5
L2 = .348	Y2 = 1.812	N1 = 4	A1 = .1416	X6 = 4.5	E2 = .813
A2 = .334	X7 = 5	X8 = 2	T5 = 8	R5 = 0	O4 = 3.375
Z = .65	dp = .496	d1 = 25	R7 = 17	R8 = 18	N2 = 4
L = 0	T = 0	D6 = 0	T6 = 0	E = 30000000	R6 = 0
					R9 = 16.876

NATURAL FREQUENCIES (HZ.)	ACTUATOR BOLT STRESSES
LONGITUDINAL (Z) ACT./VALVE = 17 VS. 33HZ.	SHEAR = 19891 PSI
TRANSVERSE (Y) ACT./VALVE = 40 VS. 33HZ.	TENSILE = 29658 PSI
LATERAL DISC/STEM = 411 VS. 33HZ.	COMBINED = 39640 PSI VS. ALLOW. = 94500

BRACKET BOLT STRESSES	BRACKET STRESSES
SHEAR = 7486 PSI	SHEAR = 7705 PSI
TENSILE = 18761 PSI	TENSILE = 22920 PSI
COMBINED = 21382 PSI VS. ALLOW. = 94500	COMBINED = 25270 PSI VS. ALLOW. = 41400

VALVE NECK STRESSES	STEM STRESSES
SHEAR = 1072 PSI	SHEAR = 12686 PSI
TENSILE = 2504 PSI	TENSILE = 4668 PSI
COMBINED = 2918 PSI VS. ALLOW. = 26250	COMBINED = 15233 PSI VS. ALLOW. = 52500

DISC PIN STRESS	SECTION MODULUS
SHEAR = 33184 PSI VS. ALLOW. = 21000	VALVE = 1205.99 IN <sup>3</sup>
	PIPING = 130.16 IN <sup>3</sup>

ACTUATOR DEFLECTIONS	BODY BOLTING
LONGITUDINAL = .202605 INCHES	NOT APPLICABLE
TRANSVERSE = 4.26000000E-02 INCHES	

DISC STRESSES	DISC SHUT
WITH FLOW	
BENDING STRESS, Sx = 2554	BENDING STRESS, Sx = 2042
BENDING STRESS, Sy = 2544	BENDING STRESS, Sy = 3583
COMBINE STRESS = 4415 PSI VS. ALLOW. = 24750	COMBINE STRESS = 4948 PSI VS. ALLOW. = 24750

SIGNED.....*John R. Rodey*.....DATED.....10/25/85.....

# NUCLEAR LOCA & SEISMIC ANALYSIS

VALVE SIZE: 18"  
 VALVE CLASS: 150  
 ACTUATOR: Matco 26072-SR80  
 CUSTOMER: Texas Utilities  
 P.O. NO.: CIF-12296-5  
 SPEC. NO.: 2323-SS-20  
 REFERENCE NO.: 34927 (14758 & 9)  
 ITEM NO.: 1 (2)

## REFERENCE DWGS.

A. ASS'Y DWG. NO.	<u>14758-1</u>	REV.	<u>D</u>
B. BODY DWG. NO.	<u>9144-180-3</u>	REV.	<u>-</u>
C. DISC DWG. NO.	<u>1144-180-9</u>	REV.	<u>B</u>
D. STEM DWG. NO.	<u>1304-362</u>	REV.	<u>-</u>
E. PIN DWG. NO.	<u>1306-14</u>	REV.	<u>A</u>
F. BRACKET DWG. NO.	<u>12300</u>	REV.	<u>-</u>

## ALLOWABLE STRESSES

A. BODY	<u>SA515 GR 70</u>	<u>26250</u>	PSI
B. STEM	<u>SA564 GR 630 Cond H 1100</u>	<u>52500</u>	PSI
C. PIN	<u>SA564 GR 630 Cond H 1100</u>	<u>35000</u>	PSI
D. BRACKET	<u>A500</u>	<u>41400</u>	PSI
E. BOLTING	<u>A193 GR B7</u>	<u>94500</u>	PSI
F. DISC	<u>SA351 GR CFBM</u>	<u>24750</u>	PSI

## DESIGN CONDITIONS

PRESSURE:	<u>50</u>	PSI
TEMPERATURE:	<u>300</u>	DEG. F.
FLOW PRESS. DROP:	<u>22.6</u>	PSI

## VALVE TORQUES

LOCA TORQUE:	<u>11401</u>	IN-LBS	Case 2A
MEDIA:	<u>Air</u>		55°
FLOW DIRECTION:	<u>Non preferred</u>		Page C-12

## G-LOADINGS

TRANSVERSE:	<u>3.2 Damping (2.6+1) x 1.5 = 5.4</u>
VERTICAL:	<u>2.2 x 1.5 = 3.3</u>
LONGITUDINAL:	<u>2.6 x 1.5 = 3.9</u>

## OCCASIONAL LOAD

TRANSVERSE:	<u>500</u>	LBS
VERTICAL:	<u>0</u>	LBS



# DIMENSIONAL DATA

SIZE 18" CLASS 150

ACTUATOR Matryx 2C072-SR80

## DESCRIPTION OF VARIABLE

INPUT COMPUTER  
NAME NAME

TRANSVERSE DIST. ACTUATOR C.G. TO $\phi$ VALVE	3.32	X1	X1
LONGITUDINAL DIST. ACTUATOR C.G. TO $\phi$ VALVE	0.63	X2	X2
VERTICAL DIST. ACTUATOR C.G. TO BRACKET	6.48	X3	X3
HEIGHT BRACKET	4	X4	X4
HEIGHT VALVE NECK	3	X5	X5
ACTUATOR WEIGHT	491	W1	W4
BRACKET WEIGHT	11	W2	W5
DISC WEIGHT	133	W4	W3
THICKNESS OF BRACKET LOWER PLATE	5	T1	T1
WIDTH OF BRACKET	4.5	T2	T2
WIDTH OF VALVE NECK	7	T3	T3
THICKNESS OF VALVE NECK	7.698	T4	T4
THICKNESS OF BRACKET SIDE PLATES	15	T9	T9
THICKNESS OF BRACKET TOP PLATE	15	T0	T0
VALVE NECK O.D.	0	d3	Do
PACKING BORE I.D.	2.5	D1	B5
STEM DIA.	1.75	D1	D1
CAGE DIA. OF DISC	16.654	D	D
WIDTH SMALL DIA. BACK OF DISC	2.435	E1	E1
WIDTH LARGE DIA. OF DISC	18.13	E2	E2
DISC SPHERICAL INDENTATION DIA.	12.35	D2	D2
DIST. BETWEEN T-WASHER FLATS ON DISC	16.34	L3	L3
STEM ENGAGEMENT IN DISC	2.625	L4	L4
DEPTH OF SPHERICAL INDENTATION IN DISC	18.13	E4	E4
LARGE DIA. OF CONICAL SECTION OF DISC	15.75	D3	D3
SMALL DIA. OF CONICAL SECTION OF DISC	3.375	D4	D4
THRUST WASHER THICKNESS	1.348	L2	L2
DIST. $\phi$ STEM TO FRONT OF DISC	1.812	Y2	Y2
NO. OF ACTUATOR BOLTS	4	N1	N1
TENSILE STRESS AREA OF ACTUATOR BOLTS	1.416	A1	A1
DBC OF ACTUATOR BOLTS	4.5	X6	X6
NO. OF BRACKET/VALVE BOLTS	4	N2	N2
TENSILE STRESS AREA OF BRACKET/VALVE BOLTS	1.334	A2	A2
TRANSVERSE DIST. BETWEEN BRACKET BOLTS	5	X7	X7
LONGITUDINAL DIST. BETWEEN BRACKET BOLTS	2	X8	X8
LENGTH OF BRACKET	8	T5	T5
DISTANCE BETWEEN VALVE BODY BOLT HOLES	0	R5	R5
DIA. OF VALVE BODY BOLT HOLES	0	R6	R6
% TORQUE ON DISC PIN	65	Z	T7
DISC PIN DIA.	1.496	dp	D5
VALVE BODY O.D.	25	d1	D8
VALVE BODY WATERWAY DIA.	17	R7	R7
ADJACENT PIPING O.D.	18	R8	R8
ADJACENT PIPING I.D.	11.876	R9	R9
LENGTH ACROSS GUSSETS	0	L	L
THICKNESS OF GUSSETS	0	T	T
DIA. OF NECK FLANGE	0	D6	D6
THICKNESS OF NECK FLANGE	0	T6	T6
MODULUS OF ELASTICITY	30,000,000	E	E
MAXIMUM PIPING BENDING MOMENT	1/2	M4	M4
NO. OF BODY BOLTS	1	N3	N3
DBC OF BODY BOLTS	1	X0	X0
ROOT AREA OF BODY BOLTS	1	E0	E0

# NUCLEAR SEISMIC ANALYSIS

18 CLASS 150 VALVE ASSEMBLY  
WITH 26072-SR80 ACTUATOR

CUSTOMER: TEXAS UTILITIES  
P.O. NO. CPF-12298-S  
SPEC. NO.: 2323-S8-20  
REF. NO.: 34977  
ITEM NO.: 1

REFERENCE DWGS.			ALLOWABLE STRESSES (PSI)			DESIGN & FLOW CONDITIONS		
A. ASS'Y DWG. NO.: 14750-1 REV. 0			A. BODY: 26250			SHUT-OFF PRESS. (PSI) = 50 TEMPERATURE (°F) = 300 FLOW PRESS. DROP (PSI) = 22.6		
B. BODY DWG. NO.: 9144-180-3 REV. -			B. STEM: 52500					
C. DISC DWG. NO.: 1144-180-9 REV. 0			C. PIN: 35000					
D. STEM DWG. NO.: 1304-362 REV. -			D. BRACKET: 41400					
E. PIN DWG. NO.: 1306-14 REV. A			E. BOLTING: 94500					
F. BRACKET DWG. NO.: 12300 REV. -			F. DISC: 24750					
			G. LOADINGS			OCCASIONAL LOADS		
LOCA TORQUE = 11401			TRANSVERSE = 5.4			TRANSVERSE = 500		
MEDIA = AIR			VERTICAL = 4.8			VERTICAL = 0		
FLOW DIRECTION = NONPREFERRED			LONGITUDINAL = 3.7					
DIMENSIONAL DATA								
X1 = 3.32	X2 = .63	X3 = 6.48	X4 = 4	X5 = 3	W1 = 481	W2 = 11		
Q4 = 133	Y1 = .5	Y2 = 4.5	T3 = 7	T4 = 3.688	T7 = .5	T8 = .5		
d3 = 0	D1 = 2.5	D1 = 1.75	D = 16.654	E1 = 2.438	E2 = .813			
D2 = 12.25	L3 = 16.34	L4 = 2.625	E4 = .813	D3 = 15.75	D4 = 3.375			
L2 = .348	Y2 = 1.812	N1 = 4	A1 = .1416	X6 = 4.5	N2 = 4			
A2 = .134	X7 = 5	X8 = 2	T5 = 8	R5 = 0	R6 = 0			
Z = .65	dP = .496	d1 = 25	R7 = 17	R8 = 18	R9 = 16.876			
L = 0	T = 0	D5 = 0	T6 = 0	E = 30000000				
NATURAL FREQUENCIES (HZ.)				ACTUATOR BOLT STRESSES				
LONGITUDINAL (Z) ACT./VALVE = 17 VS. 33HZ.				SHEAR = 18362 PSI				
TRANSVERSE (Y) ACT./VALVE = 40 VS. 33HZ.				TENSILE = 29658 PSI				
LATERAL STRENGTH/STEM = 411 VS. 33HZ.				COMBINED = 38431 PSI VS. ALLOW. = 94500				
BRACKET BOLT STRESSES				BRACKET STRESSES				
SHEAR = 4941 PSI				SHEAR = 6941 PSI				
TENSILE = 13761 PSI				TENSILE = 22343 PSI				
COMBINED = 22064 PSI VS. ALLOW. = 94500				COMBINED = 24324 PSI VS. ALLOW. = 41400				
VALVE NECK STRESSES				STEM STRESSES				
SHEAR = 983 PSI				SHEAR = 10834 PSI				
TENSILE = 2594 PSI				TENSILE = 4468 PSI				
COMBINED = 2483 PSI VS. ALLOW. = 26250				COMBINED = 13417 PSI VS. ALLOW. = 52500				
DISC PIN STRESS				SECTION MODULUS				
SHEAR = 19799 PSI VS. ALLOW. = 21000				VALVE = 1205.79 IN³				
				PIPING = 130.16 IN³				
ACTUATOR DEFLECTIONS				BODY BOLTING				
LONGITUDINAL = .202605 INCHES				NOT APPLICABLE				
TRANSVERSE = 4.26000000E-02 INCHES								
DISC STRESSES				DISC SHUT				
WITH FLOW				BENDING STRESS, Sx = 2062				
BENDING STRESS, Sx = 3368				BENDING STRESS, Sy = 3583				
BENDING STRESS, Sy = 3320				COMBINE STRESS = 4948 PSI VS. ALLOW. = 24750				
COMBINE STRESS = 5792 PSI VS. ALLOW. = 24750								

SIGNED.....*[Signature]*.....DATED...10/15/85...

APPENDIX F

Determination of Pressure Drops  
for Normal Flow Conditions

VALVE SIZE= 18 IN. CLASS=150

Valve Angle 90°

D= 120000 CV= 10053 PI= 184.7 Fp= 1  
FR= 1 Xts= .33 GR= 1 'F= 300 Z= 1

DELTA P= 0.00039649 PSI = 0.000167 IN OF WATER

VALVE SIZE= 18 IN. CLASS=150

Valve Angle 90°

D= 120000 CV= 8753 PI= 187.7 Fp= 1  
FR= 1 Xts= .42 GR= 1 'F= 300 Z= 1

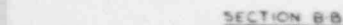
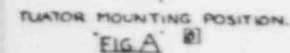
DELTA P= 0.00074208 PSI = 0.000355 IN OF WATER

VALVE SIZE= 18 IN. CLASS=150

Valve Angle 60°

D= 120000 CV= 4974 PI= 184.7 Fp= 1  
FR= 1 Xts= .46 GR= 1 'F= 300 Z= 1

DELTA P= 0.00145424 PSI = 0.000202 IN OF WATER



TEXAS UTILITIES GENERATING COMPANY  
% BROWN & ROOT INC  
COMANCHE PEAK STEAM ELECTRIC STATION UNIT #1  
GLEN ROSE, TEXAS

☒ TAG N#:  
 CPI-VADPBC-09(IHV-5549)  
 CPI-VADPBC-10(IHV-5549)

THIS VALVE ASS'Y IS DESIGNED & MANUFACTURED IN  
ACCORDANCE WITH THE ASME B1PV CODE SECT. III,  
SUBSECTION NC CLASS 2, 1974 EDITION INCLUDING  
WINTER 1975 ADDENDA, INCLUDING CODE CASE  
1621-1.

PO. NR. CP. 0000  
SERVICE TOWNSHIP VENTILATION ISOLATION VALVES  
SIES OF VALVE INLET & OUTLET 6 3/4", SEAT BORE 6 3/4"  
MATERIAL: NODI-C, CAST IRON RETAINER IN C  
RATED FLOW 2,000 CFM, PRESSURE DROP 0.5 IN WG.  
SEISMIC CATEGORY I



28	1	KEY 1/8 SQ x 2 1/2 L	KEY STOCK
27	4	LOCK WASHER 1/2 SIZE	C5
26	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
25	4	LOCK WASHER 1/2 SIZE	C5
24	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
23	4	LOCK WASHER 1/2 SIZE	C5
22	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
21	4	LOCK WASHER 1/2 SIZE	C5
20	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
19	4	LOCK WASHER 1/2 SIZE	C5
18	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
17	4	LOCK WASHER 1/2 SIZE	C5
16	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
15	4	LOCK WASHER 1/2 SIZE	C5
14	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
13	4	LOCK WASHER 1/2 SIZE	C5
12	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
11	4	LOCK WASHER 1/2 SIZE	C5
10	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
9	4	LOCK WASHER 1/2 SIZE	C5
8	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
7	4	LOCK WASHER 1/2 SIZE	C5
6	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
5	4	LOCK WASHER 1/2 SIZE	C5
4	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
3	4	LOCK WASHER 1/2 SIZE	C5
2	4	HEX HD BOLT 1/2 DIA 2 1/2 IN	C5
1	4	LOCK WASHER 1/2 SIZE	C5

DIMENSIONS CERTIFIED CORRECT		PD 207 12/27/72		POSI SEAL INTERNATIONAL IN NORTH STONINGTON, CT. 06355	
BY: 12/27/72	DATE: 12/27/72	CHECKER: KS 12/27/72			
CUSTOMER P.O. # CD-00000					
POSI SEAL REF. # 1255					
TAG NO. SEE ABOVE					
DIMENSIONS ARE IN INCHES		18" CLASS ISO SINGLE P.L.G. WELD VLV ASSY (MATRIX) 7072 SR			
		VLN TAGS 3-5-17 D 1000		14712-2	

NO LIFE

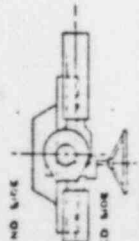
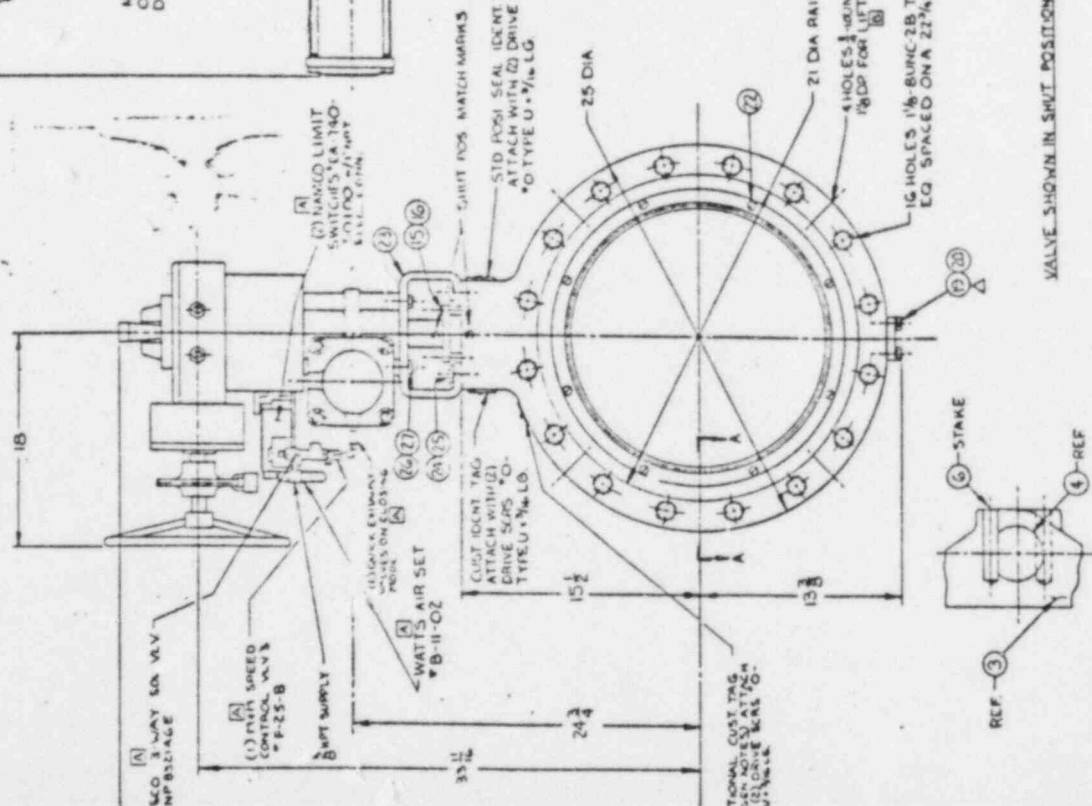
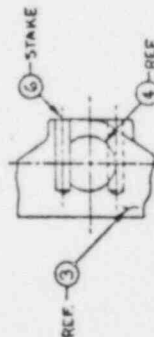


FIGURE A

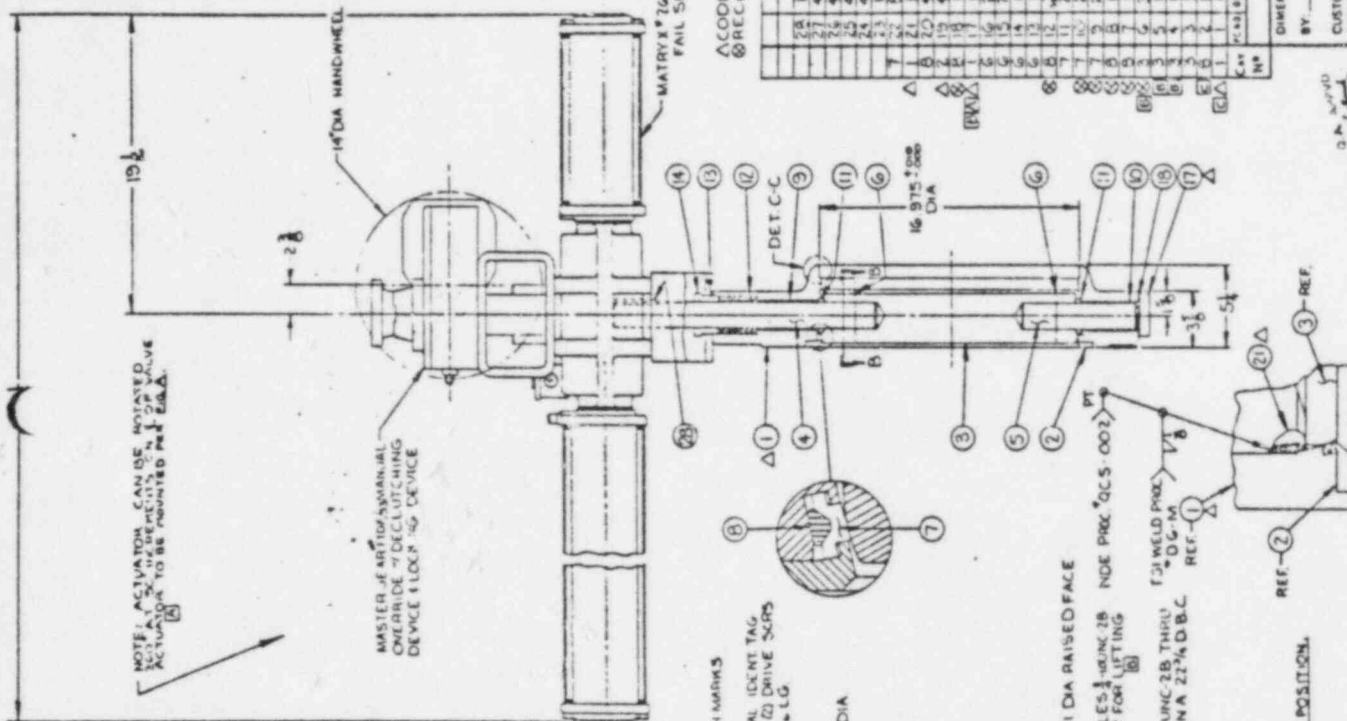


SECTION A-A

VALVE SHOWN IN SHUT POSITION.



SECTION B-B



SECTION B-B

NOTE: ACTIVATOR CAN BE MOUNTED IN EITHER POSITION TO BE MOUNTED IN EITHER POSITION.

18 HOLES 1/8\"/>

21 DIA RAISED FACE

4 HOLES 1/8\"/>

16 HOLES 1/8\"/>

25 DIA

16.915 DIA

DET C.C.

14 DIA HANDWHEEL

19.2

17.000 DIA

16.915 DIA

16.915 DIA

# GENERAL NOTES:

TEXAS UTILITIES GENERATING COMPANY  
% BROWN & ROOT INC  
COMANCHE PEAK STEAM ELECTRIC STATION UNIT #2  
GLEN ROSE, TEXAS

CUST. PO. NO. CP-00066

TAG NO. CP2-VADPBC-03 (HV-5549)  
CP2-VADPBC-10 (HV-5540)

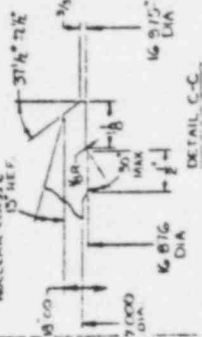
DESIGN PRESS. 50 PSIG

MEDIA: AIR

THIS VALVE ASSY IS DESIGNED & MANUFACTURED IN ACCORDANCE WITH THE ASME B14.1 CODE SECT. III, SUBSECTION NC CLASS 2, PART 4 EDITION INCLUDING WINTER 1955 ADDENDUM, INCLUDING CODE CASE 1621-1.

ADDITIONAL TAGGING (USE BACK OF 5814)

100 PSI & 0.001\"/>



## ACODE PARTS & RECOMMENDED SPARE PARTS

QTY	PART NO.	PART DESCRIPTION	UNIT
1	100	KEY 3/8" x 1/2"	
1	101	LOCKWASHER 1/2"	
1	102	HEX HD BOLT 1/2" x 1 1/2"	
1	103	LOCKWASHER 1/2"	
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1	411	LOCKWASHER 1/2"	
1	412	HEX HD BOLT 1/2" x 1 1	





## **I. INTRODUCTION**

This technical bulletin is intended to assist in the selection of Posi-Seal trunnion valves to control a given set of flow conditions.

## **II. FLOW COEFFICIENT - $C_v$**

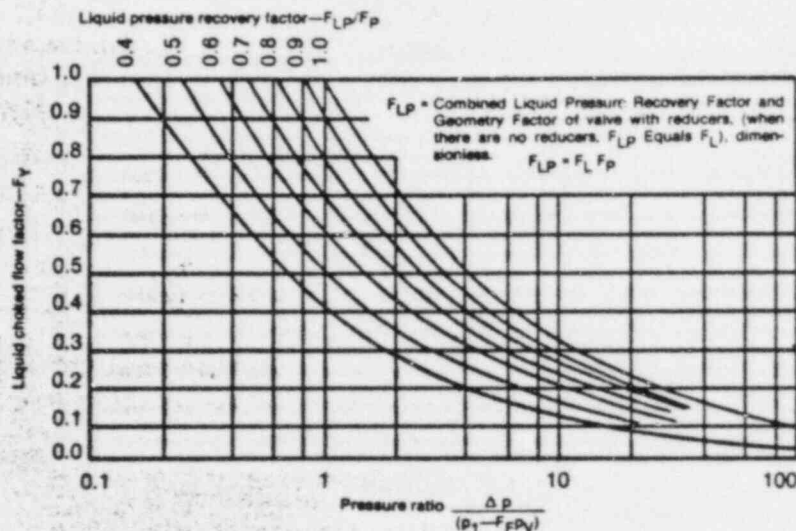
The flow coefficient or  $C_v$  of a valve is used to describe its inherent flow capacity. This value is defined as the number of U.S. gallons of water per minute at standard conditions (60°F and 14.7PSIA) that will flow thru a valve at a constant 1.0PSI pressure drop. Accordingly, a  $C_v$  value based on extensive flow testing of valves at these conditions has been assigned to each Posi-Seal trunnion valve. Using this  $C_v$  value, the capacity of each valve with regard to other fluids under various conditions can be related to this basic  $C_v$  value.

## **III. VALVE SIZING AND SELECTION**

Proper valve sizing and selection of Posi-Seal trunnion valves are to be based on the following criteria.

1. Throttling control valves should be sized between the 15° and 80° disc open position.
2. To prevent actuator/valve instability resulting from a hydrodynamic torque reversal when flowing liquids for throttling service, valves should be installed with the retaining ring side of the valve downstream. Complete information on this torque reversal phenomenon can be found in Posi-Seal Technical Bulletin No. 1A.
3. The maximum recommended operating differential pressures and pipeline velocities noted in Posi-Seal Technical Bulletin No. 6 are to be used in valve selection.
4. Valve materials of construction are governed by media and operating conditions.
5. Liquid, gas and steam gas flow limitations are governed by the parameters noted in this technical bulletin.

FIGURE 1



## PIPING EFFECTS

For valves that are installed in piping where the connecting pipe diameter is greater or less than the nominal valve diameter, the factor  $F_p$  is utilized in the sizing equations to account for additional friction losses due to piping reducers or expanders directly adjacent to the valve.

$$F_p = \sqrt{\frac{C_{vp}^2}{C_v^2 + C_{vp}^2}}$$

$$C_{vp} = \frac{29.8D^2}{\sqrt{\left(1 - \frac{D^2}{D_2^2}\right)^2 + 5\left(1 - \frac{D^2}{D_1^2}\right)}}$$

WHERE:

$C_{vp}$  = flow coefficient of pipe enlargement and contraction combined.

$C_v$  = valve flow coefficient

$D$  = ID of pipe equal to valve size, inches

$D_1$  = ID of upstream pipe, inches

$D_2$  = ID of downstream pipe, inches

FOR INSTALLATIONS WHERE  $D_1$  EQUALS  $D_2$ :

WHERE:

$$C_{vp} = KD^2$$

$D$  = ID of pipe equal to valve size, inches

$K$  = Refer to Fig. 2

FIGURE 2

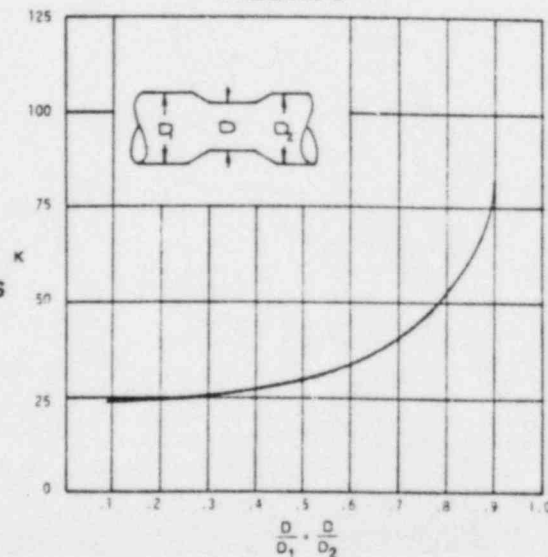




FIGURE 4

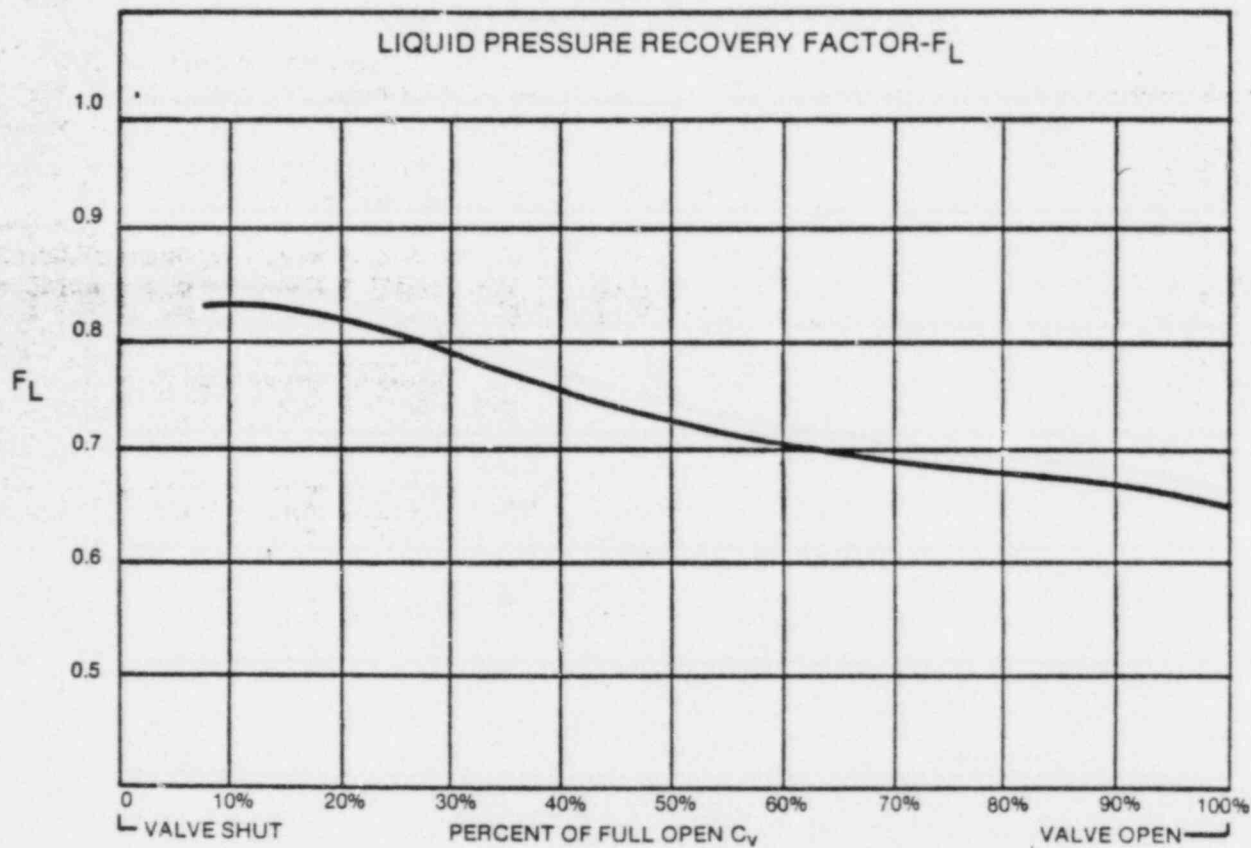
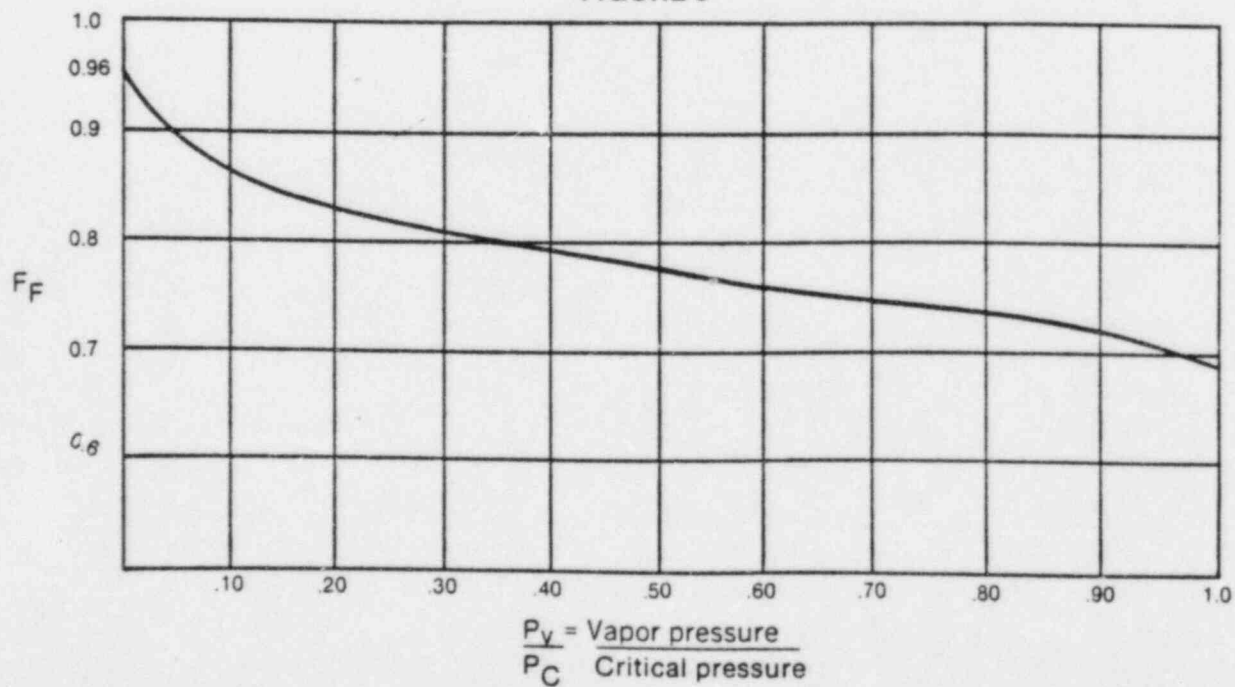


FIGURE 5



## CRITICAL FLOW

$$Q_{MAX} = 907.12 F_P C_V P_1 \sqrt{\frac{F_K X_T}{G T_1 Z}}$$

OR

$$C_V MIN = \frac{Q}{907.12 F_P P_1 \sqrt{\frac{F_K X_T}{G T_1 Z}}}$$

$$\Delta P_C = F_K X_T P_1$$

$Q_{MAX}$  = Max Flow that can pass through valve at the stated conditions.

$C_V MIN$  = Minimum Required  $C_V$  in order to pass flow at the stated conditions.

$\Delta P_C$  = Max usable differential pressure drop above which no increase in flow will occur.

$F_L$  = Rated Liquid Pressure Recovery Factor (See Figure 4).

FIGURE 6  
 $X_T$  VS. % FLOW

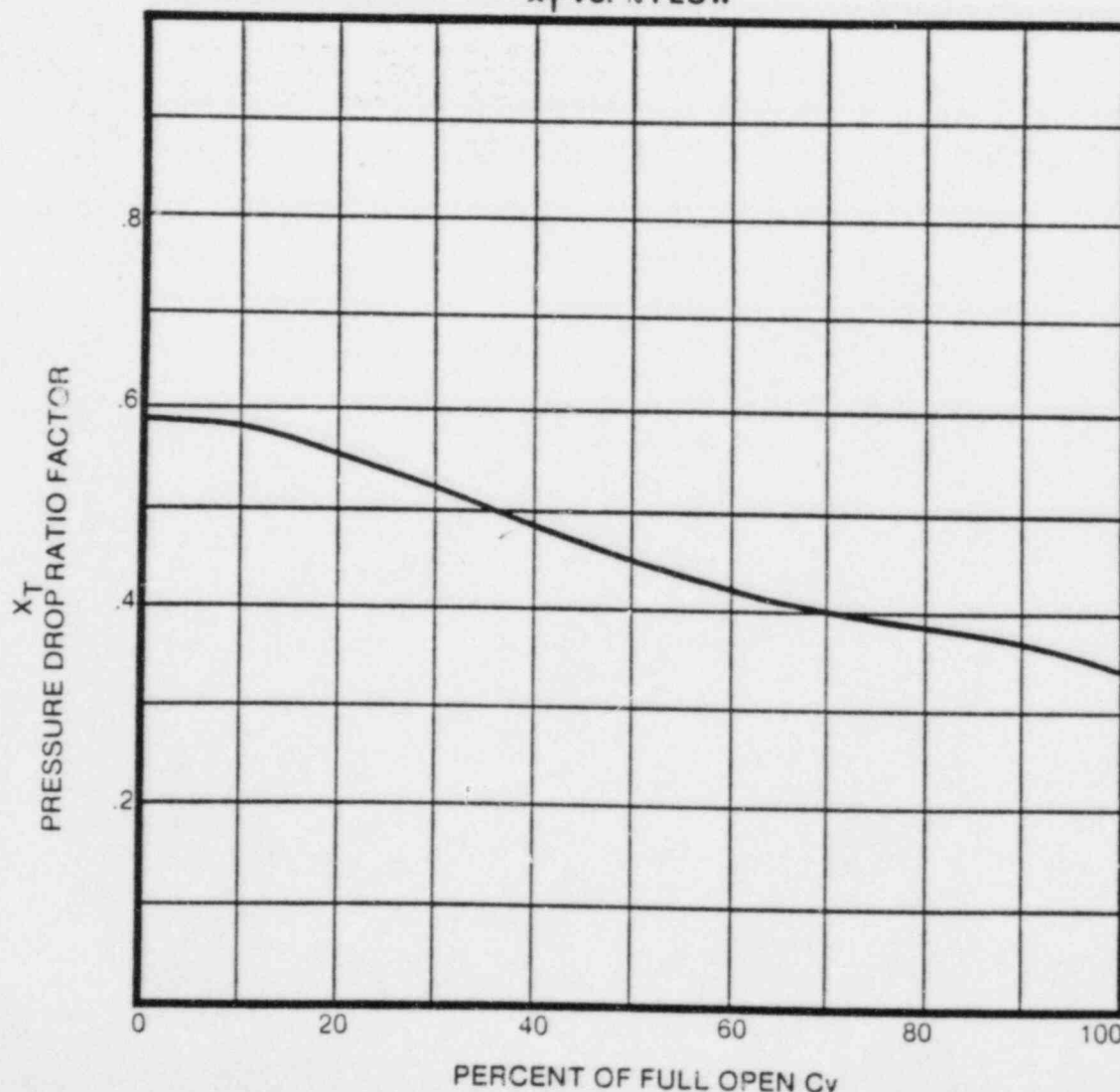


TABLE 1

PHYSICAL CONSTANTS OF VARIOUS FLUIDS								
FLUID	FORMULA OR SYMBOL	MOLECULAR WEIGHT	BOILING POINT (°F @ 14.7 PSIA)	VAPOR PRESSURE @ 70°F (PSIG)	CRITICAL TEMPERATURE (°F)	CRITICAL PRESSURE (PSIA)	SPECIFIC GRAVITY	
							LIQUID-G 60°F/60°F	GAS-G 60°F AIR=1
Acetic Acid	$\text{HO}_2\text{H}_3\text{O}_2$	60.05	245		612	841	1.05	
Acetic Anhydride	$(\text{CH}_3\text{CO})_2\text{O}$	102.09	283			676		1.08
Acetone	$\text{C}_3\text{H}_6\text{O}$	58.08	133		455	691	0.79	2.01
Acetylene	$\text{C}_2\text{H}_2$		-119		97	911	0.62	0.91
Air	$\text{N}_2\text{O}_2$	28.97	-317		-221	547	0.86	1.0
Alcohol, Ethyl	$\text{C}_2\text{H}_5\text{O}$	46.07	173	2.3 <sup>1</sup>	470	925	0.794	1.59
Alcohol, Methyl	$\text{CH}_4\text{O}$	32.04	148	4.63 <sup>1</sup>	463	1174	0.796	1.11
Ammonia	$\text{NH}_3$	17.03	-28	114	270	1636	0.62	0.59
Aniline	$\text{C}_6\text{H}_7\text{N}$	93.12	365		798	770	1.02	
Argon	A	39.94	-302		-188	705	1.65	1.38
Benzene	$\text{C}_6\text{H}_6$	78.11	176	3.22 <sup>1</sup>	552	710	0.88	2.69
Bromine	$\text{Br}_2$	159.84	138		575	1485	2.93	5.52
Butadiene	$\text{CH}_2\text{CHCH:CH}_2$	54.09	24			627		0.63
n-Butane	$\text{C}_4\text{H}_{10}$	58.12	31	51.6 <sup>1</sup>	305	550	0.58	2.0
Butyl Alcohol	$\text{C}_4\text{H}_9\text{CH}_2\text{CH}_2\text{OH}$	74.12	242			711		0.81
Carbon Dioxide	$\text{CO}_2$	44.01	-109	839	88	1072	0.80	1.52
Carbon Monoxide	CO	28.01	-314		-220	507	0.80	0.97
Carbon Tetrachloride	$\text{CCl}_4$	153.84	170		542	661	1.59	5.31
Chlorine	$\text{Cl}_2$	70.91	-30	85	291	1118	1.42	2.45
Dowtherm A						454	0.85 <sup>2</sup>	
Dowtherm E						585	1.12 <sup>2</sup>	
Ethane	$\text{C}_2\text{H}_6$	30.07	-127		90	709	0.38	1.04
Ethyl Chloride	$\text{C}_2\text{H}_5\text{Cl}$	64.52	55			2750	0.90	2.22
Ethylene	$\text{C}_2\text{H}_4$	28.05	-154		48	742		0.97
Ethyl Ether	$\text{C}_4\text{H}_{10}\text{O}$				383	522		0.46
Fluorine	$\text{F}_2$	38.00	-305	300	-200	809	1.11	1.31
Helium	He	4.003	-454		-450	33	0.18	0.14
Hydrochloric Acid	HCL	36.47	-115				1.64	
Hydrogen	$\text{H}_2$	2.016	-422		-400	188	0.07	0.07
Hydrogen Chloride	HCL	36.47	-115	613	125	1198	0.86	1.26
Hydrogen Sulfide	$\text{H}_2\text{S}$	34.07	-76	252	213	1308	0.79	1.17
Isobutane	$\text{C}_4\text{H}_{10}$	58.12	11	72.2	274	529	0.56	2.01
Isopropyl Alcohol	$\text{C}_3\text{H}_8\text{O}$	60.09	180		455	779	0.78	2.08
Methane	$\text{CH}_4$	16.04	-258		-116	673	0.30	0.55
Methyl Chloride	$\text{CH}_3\text{Cl}$	50.49	-11	59	290	967	0.99	1.74
Naphthalene	$\text{C}_{10}\text{H}_8$	128.16	424				1.14	4.43
Nitric Acid	$\text{HNO}_3$	63.02	187				1.5	
Nitrogen	$\text{N}_2$	28.02	-320		-233	492	0.81	0.97
n-Octane	$\text{C}_8\text{H}_{18}$	114.23	258	0.54 <sup>1</sup>	564	362	0.71	3.94
Oxygen	$\text{O}_2$	32.00	-297		-181	730	1.14	1.105
n-Pentane	$\text{C}_5\text{H}_{12}$	72.14	96	15	386	485	0.63	2.49
Phenol	$\text{C}_6\text{H}_5\text{OH}$	94.11	358		786	389	1.07	
Phosphoric Acid	$\text{H}_3\text{PO}_4$	98.00	415				1.83	
Propane	$\text{C}_3\text{H}_8$	44.10	-43	190 <sup>1</sup>	206	617	0.51	1.52
Propylene	$\text{CH}_2\text{CH:CH}_2$	42.08	-54		198	661		0.61
Propyl Alcohol	$\text{CH}_3\text{CH}_2\text{CH}_2\text{OH}$	60.09	208			735		0.80
Propyl Chloride	$\text{CH}_3\text{CH}_2\text{CH}_2\text{Cl}$	78.54	115			664		0.89
Refrigerant 11	$\text{CCl}_3\text{F}$	137.38	75	13.4	388	635		5.04
Refrigerant 12	$\text{CCl}_2\text{F}_2$	120.93	-22	70.2	234	597		4.2
Refrigerant 21	$\text{CHCl}_2\text{F}$	102.93	48	9.4	353	750		3.82
Refrigerant 22	$\text{CHClF}_2$	86.48	-41	122.5	205	713		
Styrene	$\text{C}_8\text{H}_8$	104.15	293	0.24 <sup>1</sup>	706	580	0.91	3.59
Sulfur Dioxide	$\text{SO}_2$	64.6	14	34.4	316	1142	1.39	2.21
Toluene	$\text{C}_7\text{H}_8$	92.14	231	1.03 <sup>1</sup>	605	611	0.87	3.18
Water	$\text{H}_2\text{O}$	18.016	212	0.95	706	3206.2	1.00	0.62

1. Vapor Pressure in PSIA @ 100°F

2. Specific Gravity @ 494.8°F.

TABLE 3  
COMPRESSIBILITY FACTORS FOR GAS

Gas	Pressure		-100 F	0 F	200 F	1000 F	2000 F
	atm	psia					
Argon	1	14.7	0.997	0.999	1.000	1.000	1.000
	10	147	0.970	0.987	0.999	1.003	1.002
	40	588	0.877	0.952	0.995	1.011	1.009
	100	1470	0.690	0.887	0.995	1.029	1.022
Carbon monoxide	1	14.7	0.997	0.999	1.000	1.000	1.000
	10	147	0.973	0.991	1.001	1.004	1.003
	40	588		0.967	1.007	1.017	1.012
	100	1470			1.027	1.044	1.031
Carbon dioxide	1	14.7		0.991	0.997	1.000	1.000
	10	147		0.910	0.974	1.001	1.003
	40	588			0.894	1.006	1.010
	100	1470			0.721	1.018	1.026
Hydrogen	1	14.7	1.001	1.001	1.001		
	10	147	1.007	1.006	1.005		
	40	588	1.028	1.026	1.021		
	100	1470	1.076	1.067	1.052		

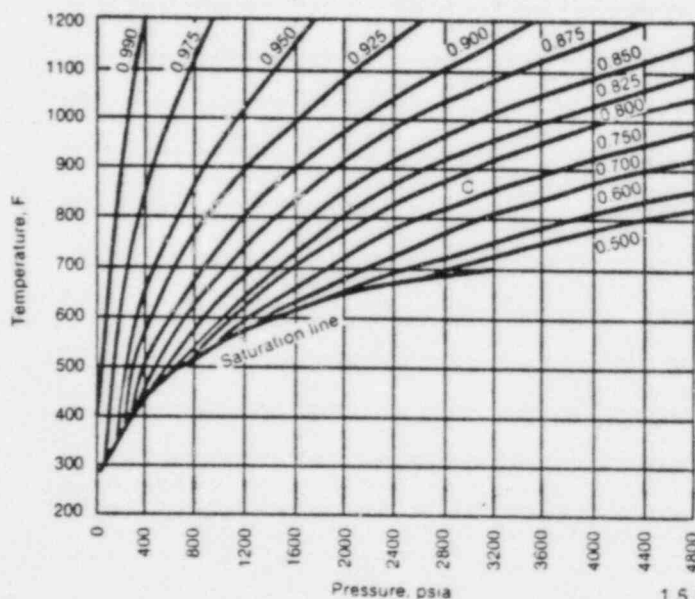


FIGURE 10  
COMPRESSIBILITY FACTORS  
FOR SUPERHEATED STEAM

FIGURE 11  
COMPRESSIBILITY FACTORS  
FOR NITROGEN

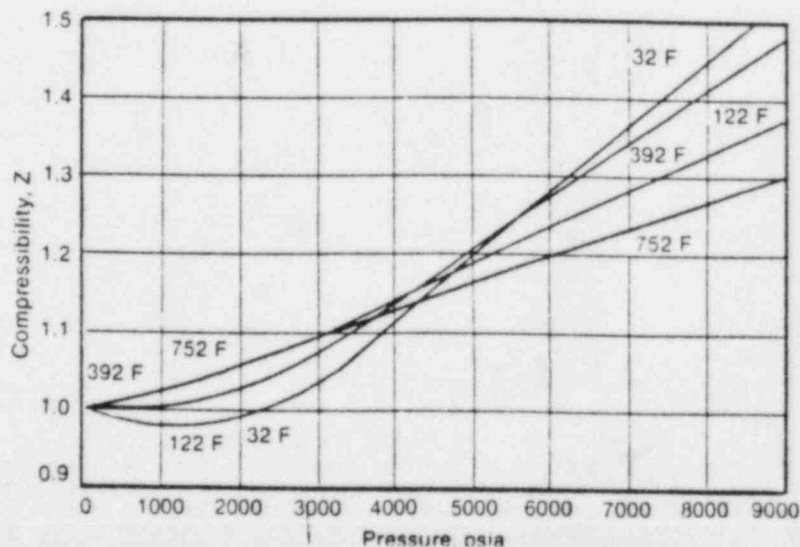


TABLE 6  
SUPERHEATED VAPOR

Pressure (psia)		Temperature (°F)													
		200	250	300	350	400	450	500	600	700	800	900	1000	1100	1200
10	v	38.85	42.56	45.00	48.63	51.04	54.05	57.05	63.04	69.01	74.98	80.95	86.92	92.88	98.84
	h	1146.6	1175.1	1193.9	1221.9	1240.6	1268.7	1287.5	1335.1	1383.4	1432.5	1482.4	1533.2	1585.0	1637.6
	s	1.7927	1.8341	1.8595	1.8950	1.9172	1.9488	1.9689	2.0160	2.0596	2.1002	2.1383	2.1744	2.2086	2.2413
20	v		21.11	22.36	24.21	25.43	27.25	28.46	31.47	34.47	37.46	40.45	43.44	46.42	49.41
	h		1172.2	1191.6	1220.3	1239.2	1267.6	1286.6	1334.4	1382.9	1432.1	1482.1	1533.0	1584.7	1637.4
	s		1.7545	1.7808	1.8170	1.8396	1.8716	1.8918	1.9392	1.9829	2.0235	2.0618	2.0978	2.1321	2.1648
50	v			8.773	9.557	10.065	10.815	11.309	12.532	13.744	14.950	16.152	17.352	18.550	19.747
	h			1184.3	1215.2	1235.1	1264.5	1283.9	1332.5	1381.4	1430.9	1481.1	1532.1	1584.0	1636.8
	s			1.6721	1.7112	1.7349	1.7680	1.7887	1.8368	1.8809	1.9219	1.9602	1.9964	2.0308	2.0636
100	v				4.663	4.937	5.333	5.589	6.218	6.835	7.446	8.052	8.656	9.259	9.860
	h				1205.7	1227.6	1258.8	1279.1	1329.1	1378.9	1428.9	1479.5	1530.8	1582.9	1635.7
	s				1.6258	1.6518	1.6869	1.7085	1.7581	1.8029	1.8443	1.8829	1.9193	1.9538	1.9867
150	v				3.023	3.223	3.502	3.081	4.113	4.532	4.944	5.352	5.758	6.162	6.564
	h				1195.1	1219.4	1252.9	1274.1	1325.7	1376.3	1426.9	1477.8	1529.4	1581.7	1634.7
	s				1.5706	1.5995	1.6372	1.6599	1.7109	1.7566	1.7984	1.8374	1.8740	1.9086	1.9416
200	v					2.361	2.585	2.726	3.060	3.380	3.693	4.002	4.309	4.613	4.917
	h					1210.3	1246.5	126.9	1322.1	1373.6	1424.8	1476.2	1528.0	1580.5	1633.7
	s					1.5594	1.6001	1.6240	1.6767	1.7232	1.7655	1.8048	1.8415	1.8763	1.9094
300	v						1.6638	1.7675	2.005	2.227	2.442	2.652	2.859	3.065	3.269
	h						1232.5	1257.6	1314.7	1368.3	1420.6	1472.8	1525.2	1578.1	1631.7
	s						1.5434	1.5701	1.6268	1.6751	1.7184	1.7582	1.7954	1.8305	1.8638
500	v							0.9927	1.1591	1.3044	1.4405	1.5715	1.6996	1.8256	1.9504
	h							1231.3	1298.6	1357.0	1412.1	1466.0	1519.6	1573.4	1627.6
	s							1.4919	1.5588	1.6115	1.6571	1.6982	1.7363	1.7719	1.8056
700	v								0.7934	0.9077	1.0108	1.1082	1.2024		1.3853
	h								1280.6	1345.0	1403.2	1459.0	1513.9		1623.5
	s								1.5084	1.5665	1.6147	1.6573	1.6963		1.7666
1000	v								0.5140	0.6084	0.6878	0.7604	0.8294	0.8962	0.9615
	h								1248.8	1325.3	1389.2	1448.2	1505.1	1561.3	1617.3
	s								1.4450	1.5141	1.5670	1.6121	1.6525	1.6897	1.7245
2000	v									0.2489	0.3074	0.3532	0.3935	0.4311	0.4668
	h									1240.0	1335.5	1409.2	1474.5	1536.2	1596.1
	s									1.3783	1.4576	1.5139	1.5603	1.6012	1.6384
3000	v									0.0984	0.1760	0.2159	0.2476	0.2757	0.3018
	h									1060.7	1267.2	1365.0	1441.8	1510.0	1574.3
	s									1.1966	1.3690	1.4439	1.4984	1.5437	1.5837



TABLE 9

VALVE FLOW COEFFICIENTS  $C_v$   
CLASS 300 STD. RATING

Valve Size	DEGREES OF DISC OPENING								
	10°	20°	30°	40°	50°	60°	70°	80°	90°
1 1/2"	1	3	6	10	15	22	29	34	38
2"	2	4	9	15	23	34	43	51	57
3"	6	14	29	50	77	111	143	167	188
4"	12	30	63	107	165	238	307	359	404
6"	32	81	167	285	441	635	818	957	1076
8"	40	100	206	352	545	783	1010	1183	1329
10"	71	178	367	628	971	1398	1800	2108	2369
12"	110	276	570	975	1509	2172	2797	3276	3681
14"	136	341	704	1204	1863	2681	3454	4045	4545
16"	169	422	873	1492	2309	3323	4280	5012	5632
18"	247	617	1276	2181	3374	4856	6255	7325	8230
20"	286	714	1476	2524	3906	5620	7240	8478	9526
24"	375	938	1939	3315	5129	7381	9508	11135	12511
30"	715	1788	3696	6319	9776	14068	18121	21221	23844
36"	1104	2760	5704	9752	15087	21711	27967	32751	36799
42"	1711	4279	8843	15118	23390	33659	43358	50774	57050
48"	1867	4667	9645	16490	25513	36713	47292	55381	62226

Table 10

FLOW COEFFICIENTS  $C_v$   
CLASS 600 STD. RATING

Valve Size	DEGREES OF DISC OPENING								
	10°	20°	30°	40°	50°	60°	70°	80°	90°
3"	5	16	31	51	84	122	151	169	182
4"	8	23	43	70	116	169	209	234	252
6"	26	78	147	242	397	579	717	803	864
8"	35	104	197	324	532	775	960	1076	1157
10"	62	185	350	576	947	1379	1709	1915	2059
12"	85	255	481	793	1302	1897	2350	2633	2831
14"	104	312	589	971	1595	2323	2878	3225	3468
16"	128	383	723	1192	1958	2851	3532	3958	4256
18"	152	456	862	1420	2332	3397	4208	4715	5070
20"	175	524	990	1630	2678	3900	4831	5413	5821
24"	349	1046	1977	3256	5349	7791	9651	10814	11628

HYDRODYNAMIC TORQUE  
OF  
HIGH PERFORMANCE TRUNNION VALVES

The increased use of High Performance Trunnion (Offset Butterfly) Valves has caused increased use in high flow applications and therefore, a need for a more accurate prediction of the Hydrodynamic Torque behavior of this type of valve. Improper actuator sizing, structural damage and control instability are all possible consequences of using inaccurate Hydrodynamic Torque data.

The Trunnion Valve is essentially a modified butterfly valve with the rotary stem offset from the disc sealing surface. See Figure 1. Normally, this style of valve exhibits high opening and closing torques with operating torques depreciating as the valve disc rotates 90° to the fully open position. However, on occasions where a considerable quantity of fluid is being pumped through the valve, the Hydrodynamic Torque may exceed the opening torque or be of sufficient magnitude that the actuator will be unable to further open the valve.

Realizing the importance of having accurate Hydrodynamic Torque data, Posi-Seal International, Inc. in North Stonington, Conn. launched an extensive R&D program in order to obtain this data. At Posi-Seal's new Hydraulics Laboratory, (See Figure 2 for schematic of lab), Hydrodynamic Torque data was measured on valve sizes - 1 1/2" through 14" for valve classes 150 through 600 lb. Additional data was taken on a 14" - 900 and a 14" - 1500 lb. valve. Data was recorded for both preferred and non-preferred fluid flow, measured at every 10 degrees of valve rotation. In order to obtain the Hydrodynamic Torque factors, the valve torque was measured while both opening and closing the valve. By averaging the above data, stem packing and bearing friction torques were negated and pure Hydrodynamic Torque was obtained. The above measured torque when divided by the differential pressure across the valve yielded the Hydrodynamic Torque factor for that particular valve at that angular location. This data was statistically analyzed on Posi-Seal's

technical mini computer and equations were developed from which Hydrodynamic Torque factors were calculated for all sizes and classes of high performance Trunnion Valves.

Based upon the flow testing performed at Posi-Seal, the following general observations concerning Hydrodynamic Torque of Trunnion Valves can be made:

- (1) For preferred flow, the Hydrodynamic Torque will always want to close the valve. See Figure 3.
- (2) For non-preferred flow, the Hydrodynamic Torque will, through  $70^{\circ}$  to  $80^{\circ}$  of valve rotation (the exact location varies with valve class), want to close the valve. Beyond this point, the Hydrodynamic Torque will want to open the valve. See Figure 4.
- (3) Except for the  $90^{\circ}$  valve location where the Hydrodynamic Torque factors are of equal magnitude for both preferred and non-preferred flow but of opposite sign, the non-preferred Hydrodynamic Torque factors are considerably less in magnitude than those for preferred flow. See Figures 3 and 4.
- (4) All of the Hydrodynamic Torque factors (for both preferred and non-preferred flow) decrease with increase in valve class. ex: Hydrodynamic Torque factors for a 600 lb. valve are significantly less than those for a 150 lb. valve. See Figures 3 and 4.
- (5) The Hydrodynamic Torque factors increase approximately proportional to the cube of the valve size. (The exact amount varies depending upon valve class).

The total Trunnion Valve Torque at some angular position (other than opening or closing) is in actuality a summation of three separate torques, stem packing torque, bearing friction torque and the Hydrodynamic Torque discussed above. Except for small size valves (6" and less), the stem packing torque is a small percentage of the total valve torque. For large sized valves, above approximately  $45^{\circ}$ , the Hydrodynamic Torque is the major component of the total valve torque, below  $45^{\circ}$  bearing friction torque becomes the major contributor. All three components must be considered in order to accurately evaluate a valve's performance.

In order to better understand the reason for Hydrodynamic Torque occurring, one must visualize the valve's disc as an airfoil. The Hydrodynamic Torque generated at some angle is a composite of fluid flow lift and drag forces acting over the disc's surface. The general formulas for lift and drag being:



$$F_D = C_D P \frac{AV^2}{2} \quad (\text{Drag})$$

$$\text{and } F_L = C_L P \frac{AV^2}{2} \quad (\text{Lift})$$

Where:

$C_D, C_L$  - Drag and lift coefficients which are related to the geometry of flow obstruction. (Disc)

P - density of the fluid medium

A - projected or surface area of flow obstruction (Disc)

V.- velocity of fluid medium

Both the lift and drag forces are dependent upon the shape of the valve disc, its orientation to the flow stream and the direction of fluid flow. As the valve angle is decreased, or flow is reversed, both the magnitude and location of these forces shift causing a change in the resultant Torque. As the valve angle is further decreased the drag forces will increase while the lift forces will deteriorate due to increased turbulence and a breakdown of the flow stream along the downstream side of the valve disc. See Figures 5 and 6.

FIGURE No. 1  
SCHEMATIC  
OF  
HIGH PERFORMANCE TRUNION VALVE

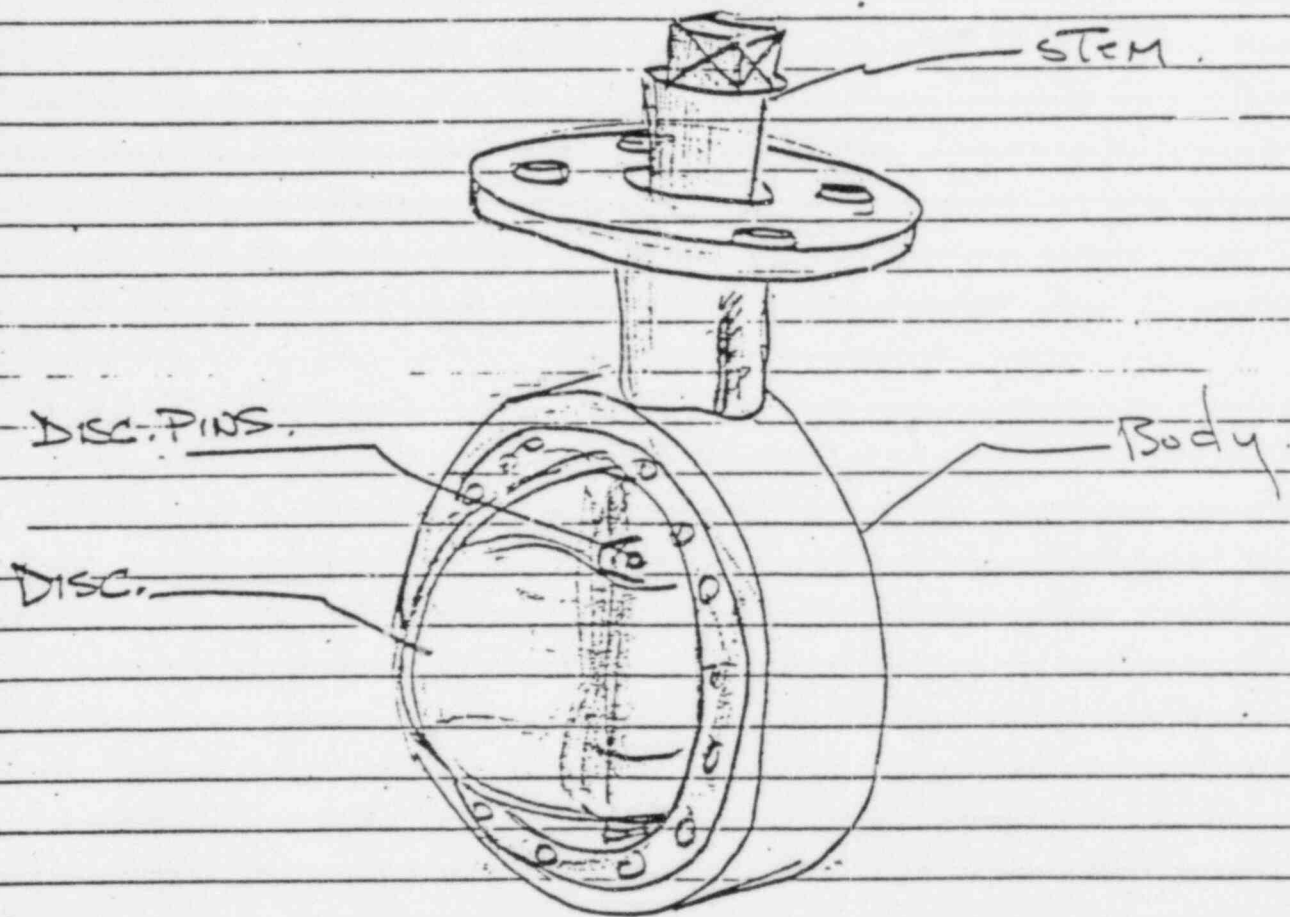
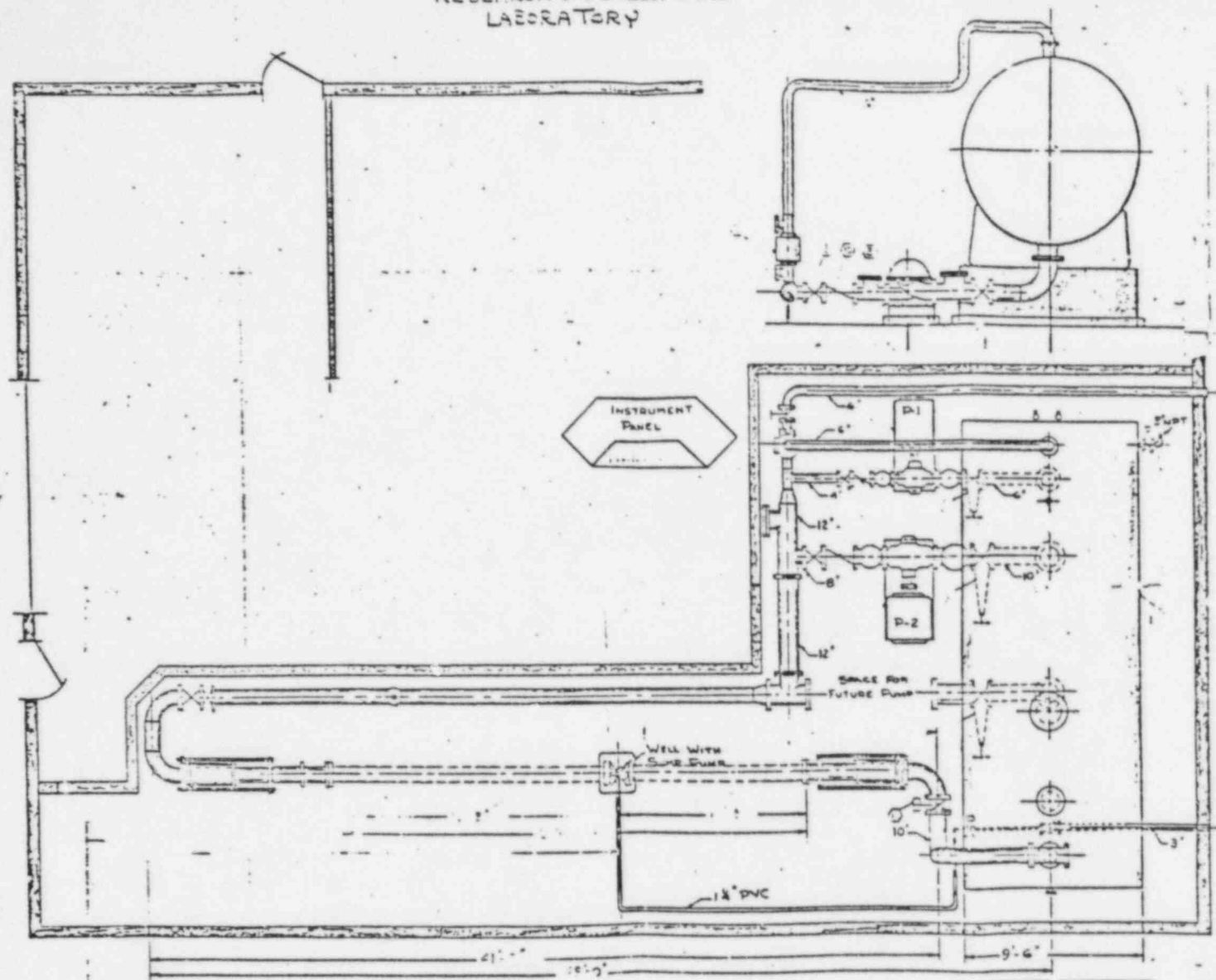
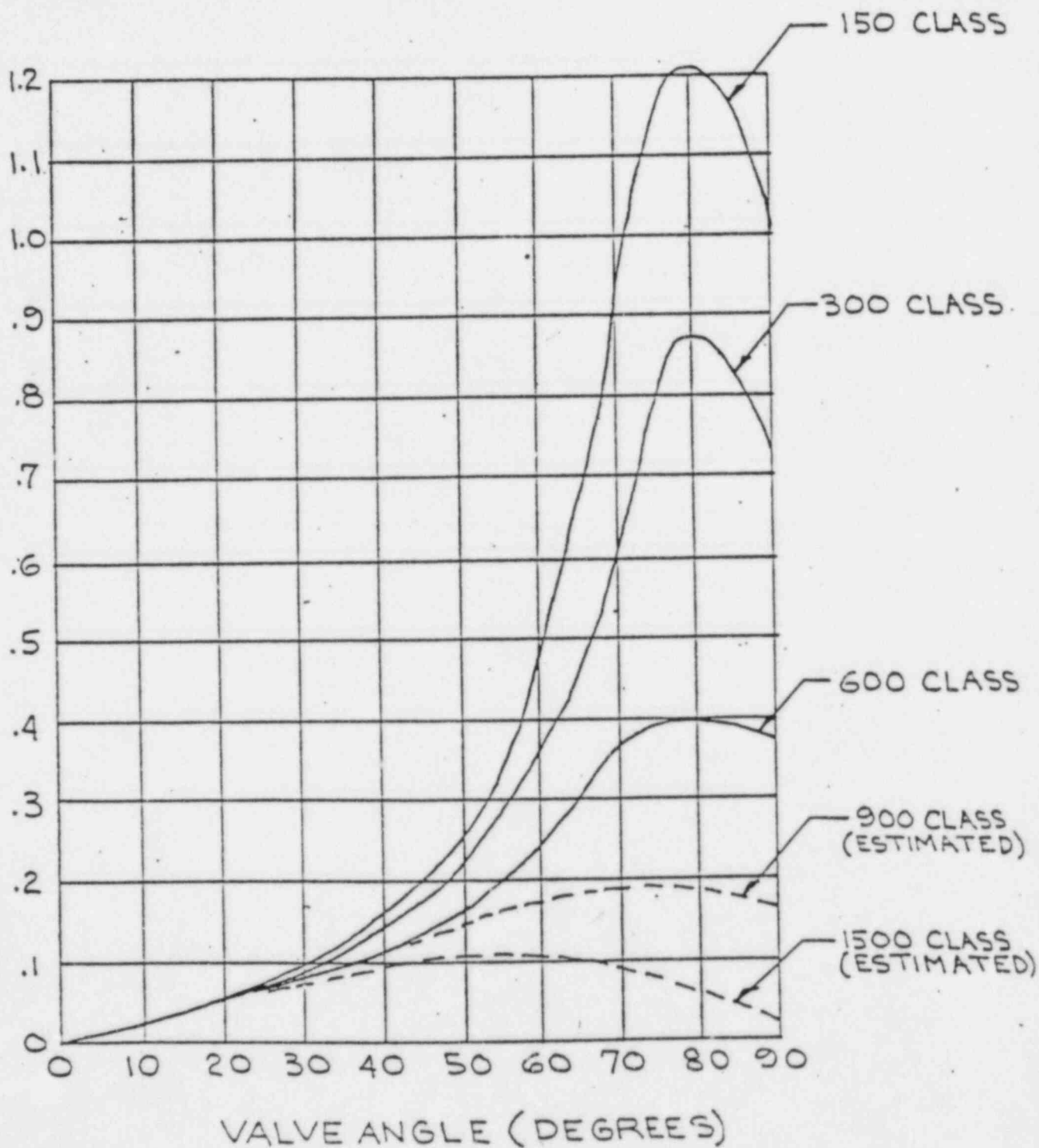


FIGURE NO. 5  
SCHEMATIC OF POST-SEAL  
RESEARCH & DEVELOPMENT  
LABORATORY



VS  
VALVE ANGLE

PERCENT OF HYDRODYNAMIC TORQUE AT 90°  
(OF 150 LB. CLASS. VALVE)



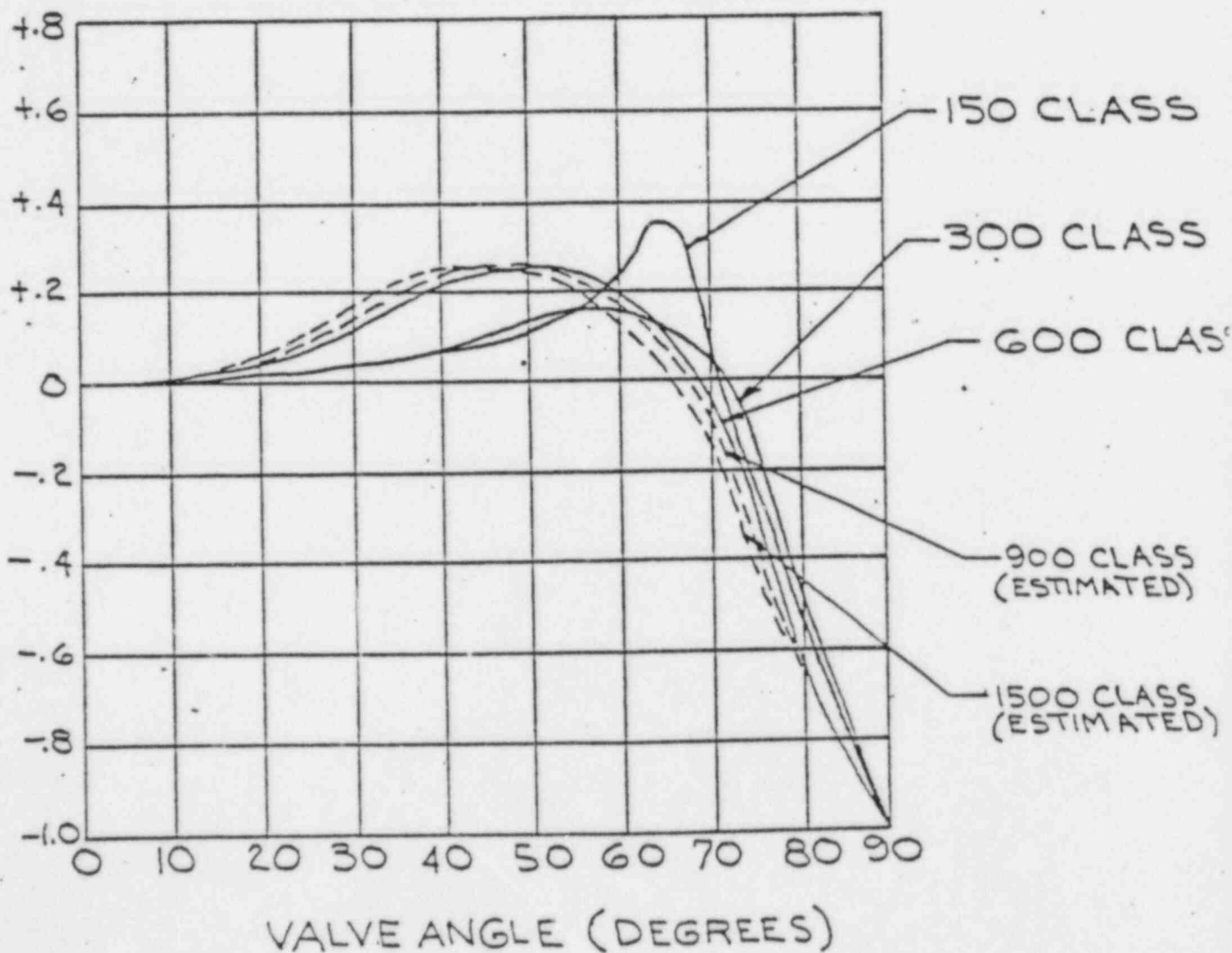
PREFERRED  
DIRECTION

# HYDRODYNAMIC TORQUE

VS

VALVE ANGLE

PERCENT OF HYDRODYNAMIC TORQUE AT 90°



NON PREFERRED  
DIRECTION

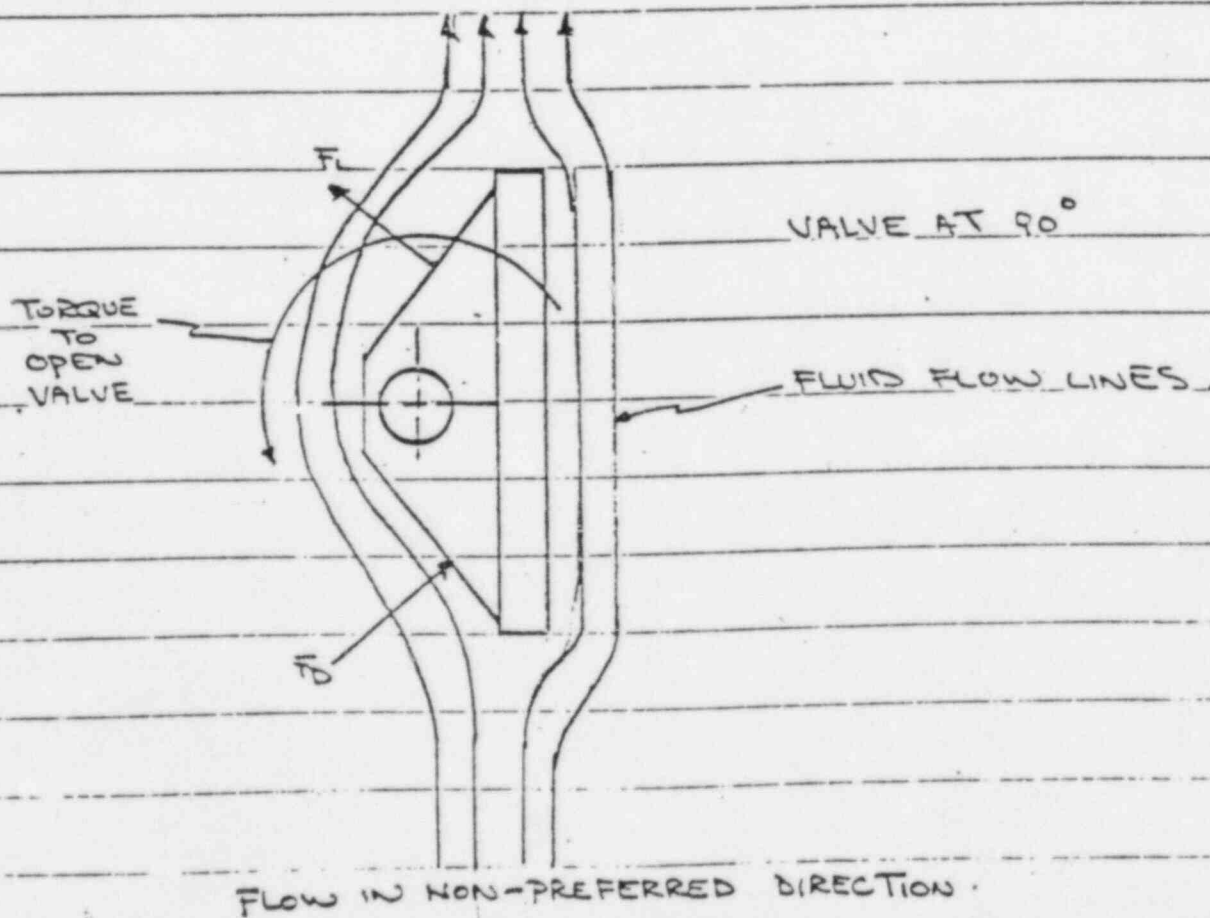
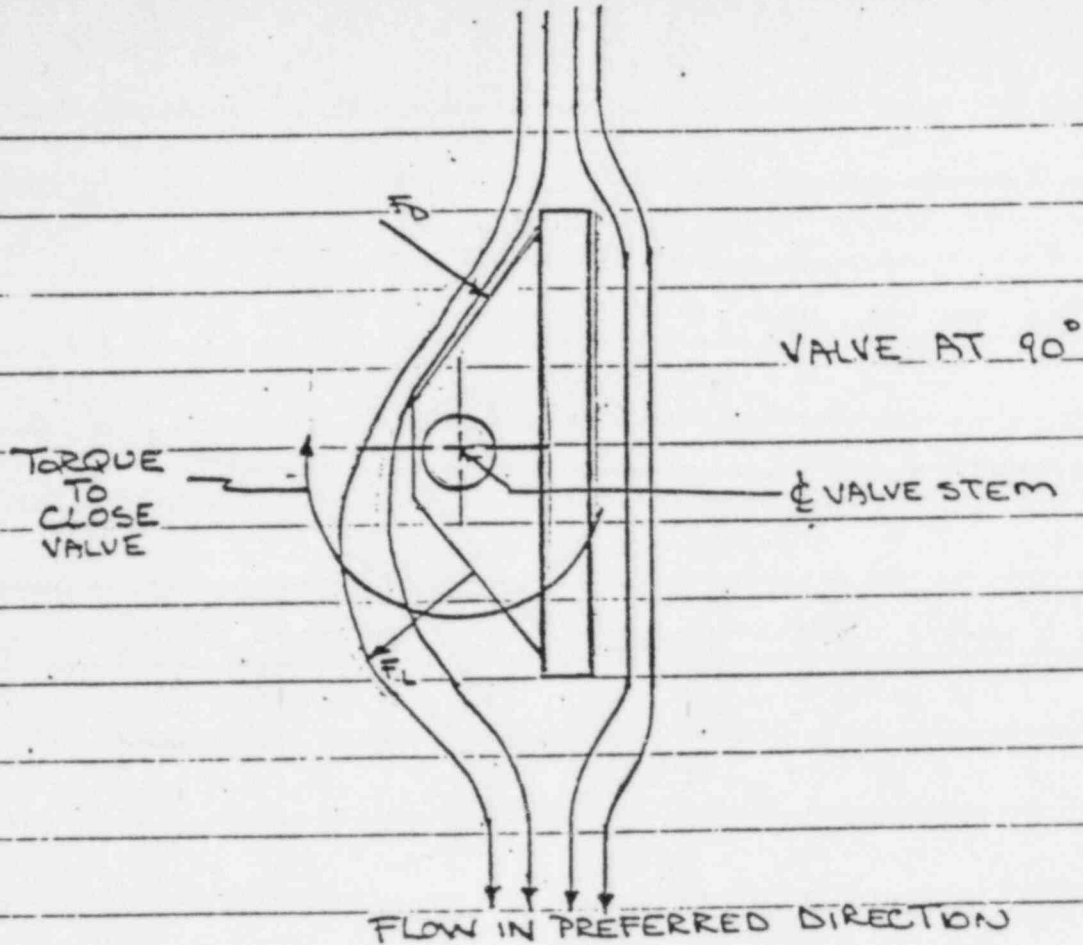


FIGURE 5

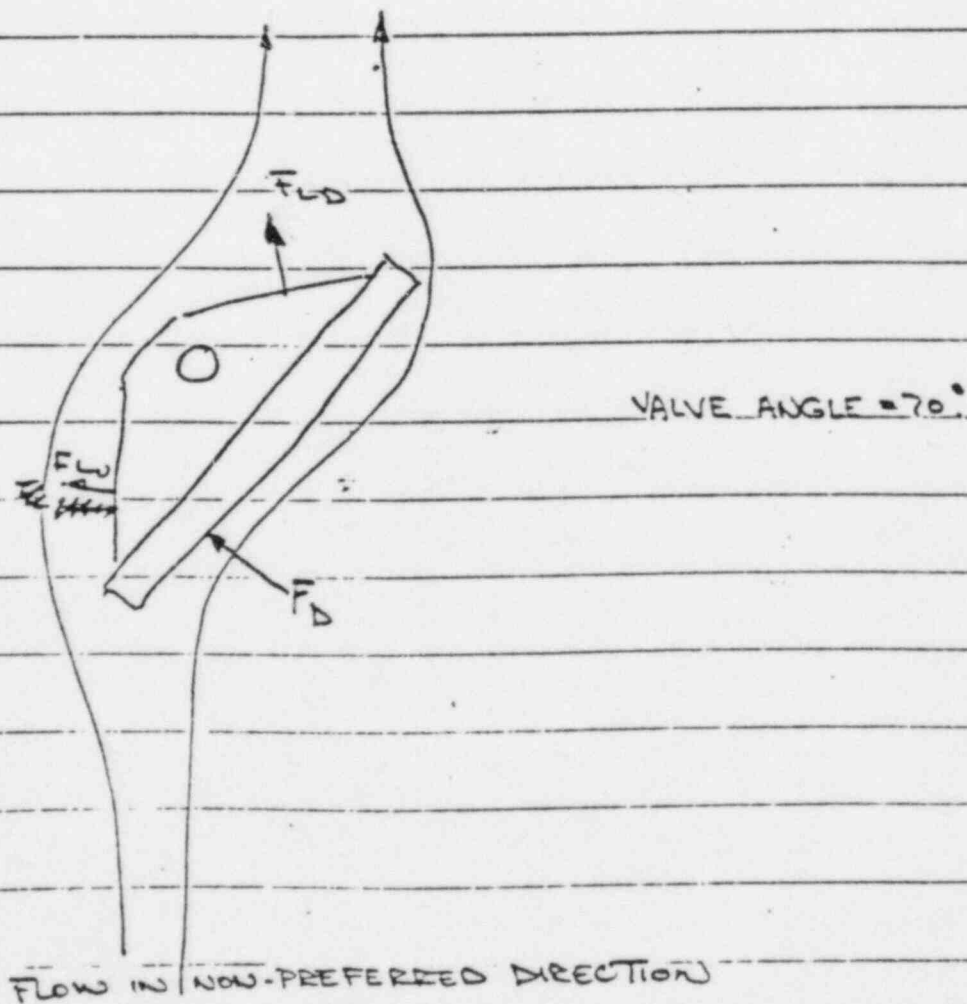
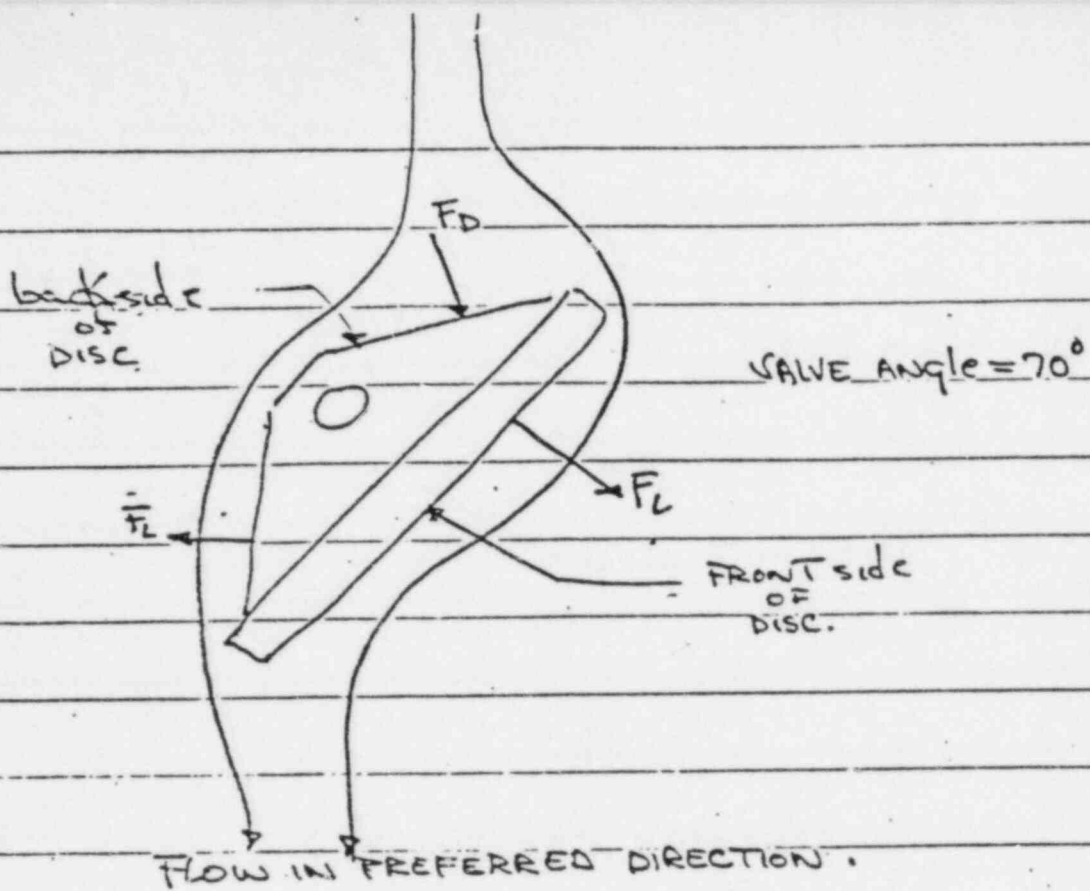
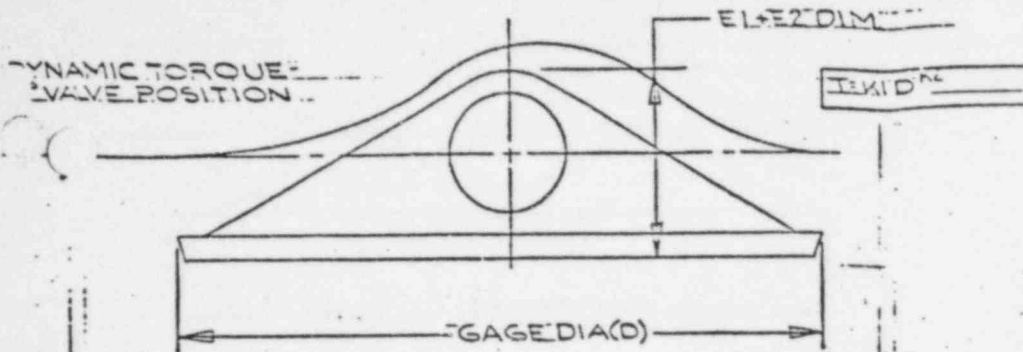


FIGURE 6



# SHAPE FACTOR VS. POWER FACTOR



T = HYDRODYNAMIC TORQUE (IN.)  
D = DISC. GAGE DIA. (IN.)  
K1 = SHAPE FACTOR  

$$K1 = \frac{E1 + E2}{D}$$

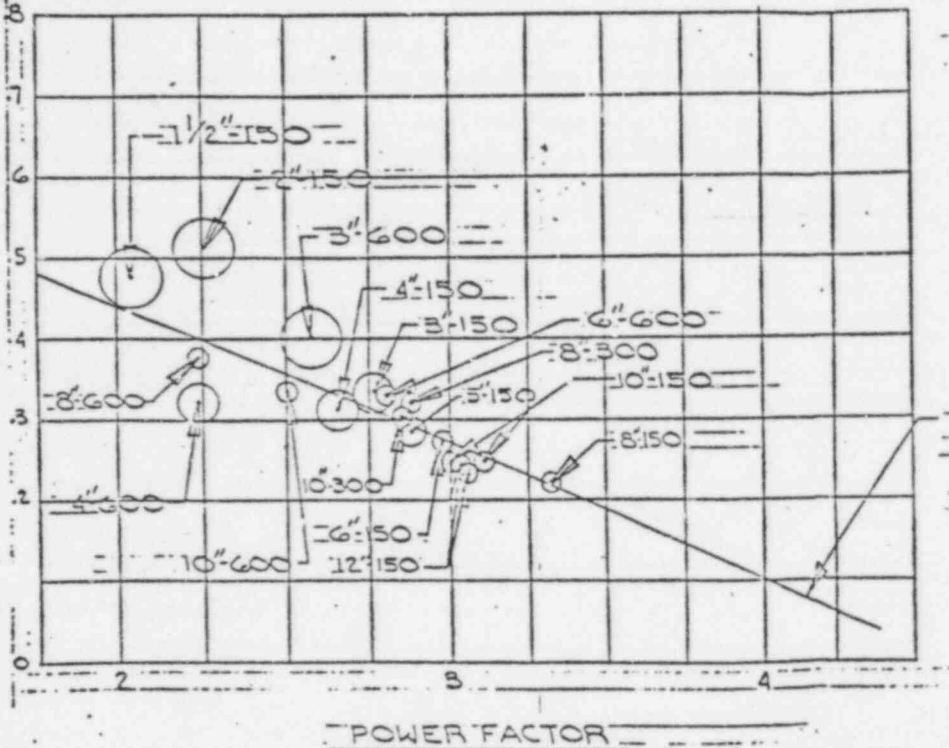
K2 = POWER FACTOR  

$$K2 = B0 + B1 \cdot K1$$

B0, B1 = COEFFICIENTS DETERMINED BY LINEAR REGRESSION ANALYSIS OF EMPIRICAL DATA

OR:  

$$K2 = \frac{\ln[TLK1]}{\ln D}$$



B0 = +4.6038 [PER LEAST SQUARE REGRESSION ANALYSIS]  
B1 = -5.9150

$$K2 = 4.6038 - 5.915 K1$$

FIGURE 7

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION

Page       

By Jansen DATE: 12/12/77 Checked By       

VALVE SIZE: 3" ISO

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON / Buna

WATER TEMPERATURE (TANK): 68 °F

DISC TYPE & DWG:       

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	( $\frac{Q}{P}$ ) PSI
0					
10		42		+45	1.1
20		41		+55	1.3
30		39		+60	1.5
40		36		+75	2.1
50		33		+85	2.6
60		31.5		+100	3.2
70		27	27	+140	5.2
80		22		+155	7.0
90		19		+140	7.4
90		19		+120	6.7
80		22		+140	6.4
70		25	27	+145	4.1
60		34		+100	2.9
50		42.5		+80	1.9
40		40.5		+60	1.5
30		42		+30	0.7
20		45		+10	0.2
10		46		0	0
0					

PACKING TORQUE: +10 IN-LB OPENING

: +5 IN-LB CLOSING

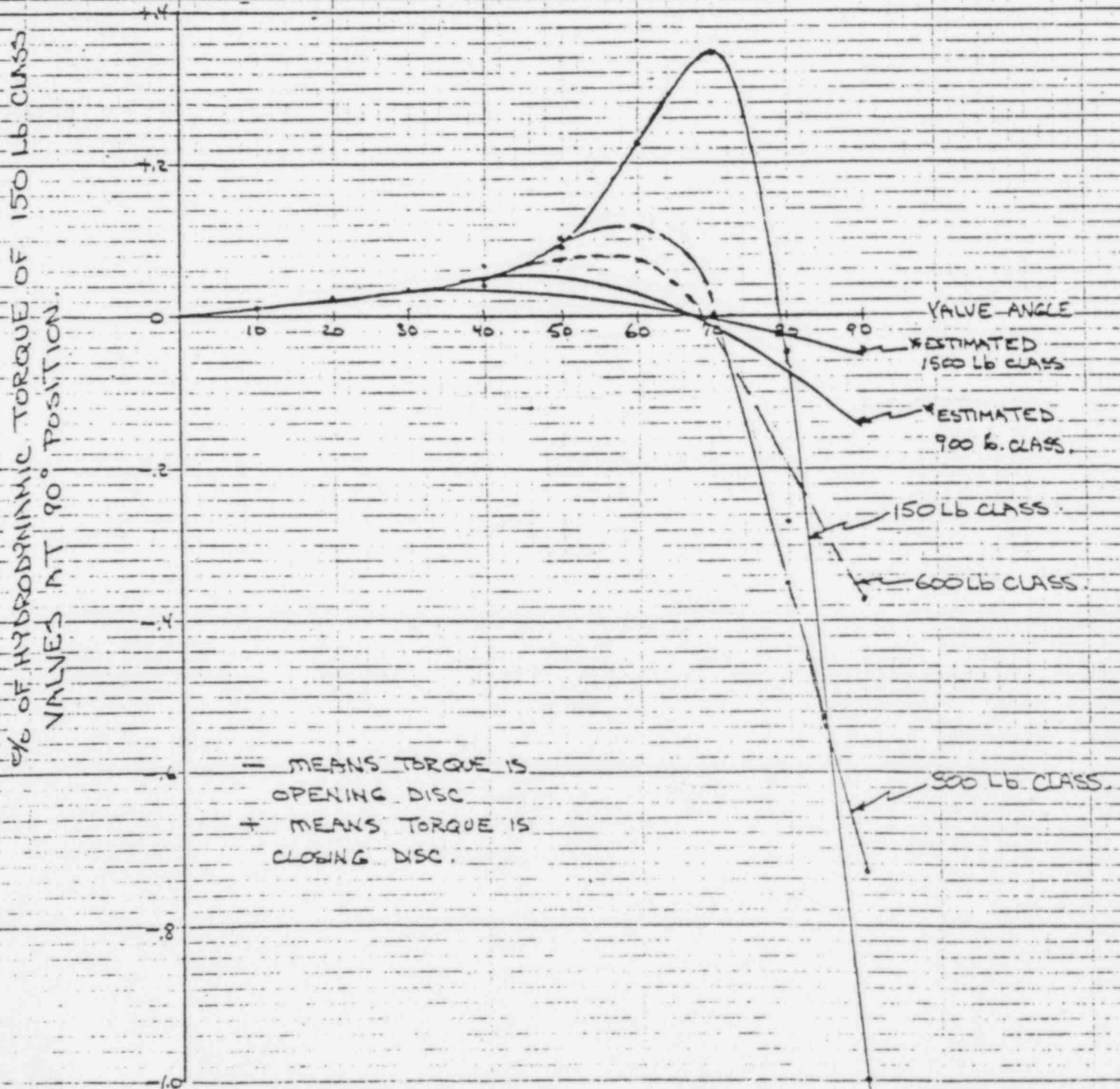
Title NON-PREFERRED DIRECTION

Page       

Calc. By       

Checked By       

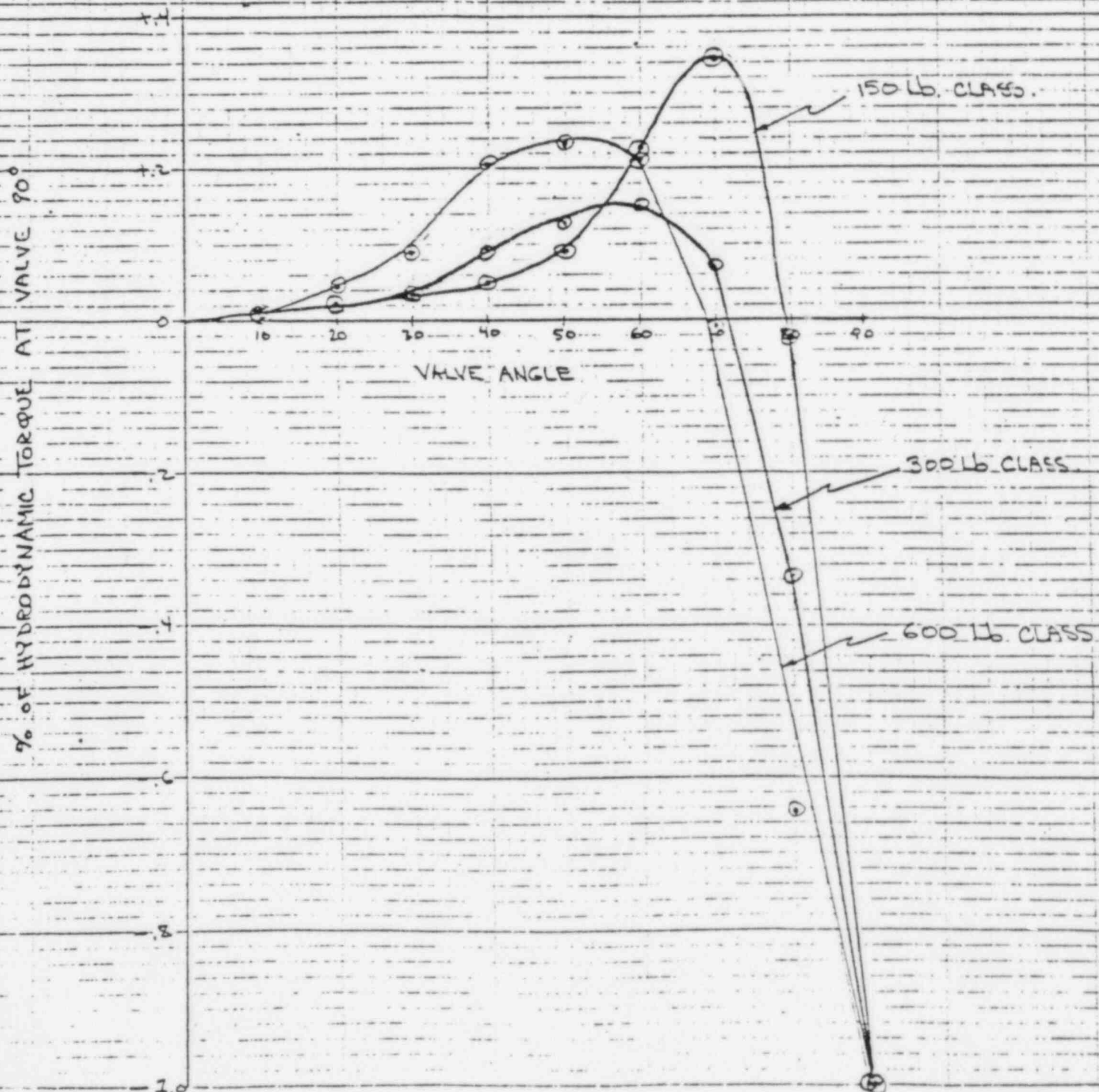
HYDRODYNAMIC TORQUE  
VS.  
VALVE ANGLE.



title NON PREFERRED DIRECTION

Page

Colg By \_\_\_\_\_ Checked By \_\_\_\_\_

HYDRODYNAMIC TORQUE  
VS.  
VALVE ANGLE



# ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUES

Page

C-1c. By NON-PREFERRED DIRECTION

Checked By QANSEN

8/15/78

ANGLE

VALVE SIZE - % TORQUE @ 90°

	10"=150	8"=150	12"=150	AVG.
90	(-378.5) -1.0	(-178) -1.0	(-468.5) -1.0	-1.0
80	(-717) -1.309	(+38.7) +1.217	(+205) +0.44	+0.16
70	(+112) +1.296	(+63.35) +1.356	(+1805) +1.315	+1.346
60	(+94) +1.248	(+32.55) +1.83	(+118.5) +1.253	+1.228
50	(+47.5) +1.115	(+15.05) +1.085	(+32) +1.079	+1.083
40	(+15.5) +1.041	(+8.35) +1.047	(+29) +1.062	+1.05
30	(+14) +1.037	(+4.775) +1.027	(+21.5) +1.046	+1.037
20	(+2) +1.005	(+3.125) +1.018	(+24) +1.051	+1.025
10	(0) +0	(+2.165) +1.012	(+2) +1.004	+1.005

## ENGINEERING CALCULATIONS

## HYDRODYNAMIC TORQUES

Title

Page

Calc. By NON-PREFERRED DIRECTION

Checked By

DANSEN 8/8/78

ANGLE

VALVE SIZE - % OF TORQUE @ 90°

AVG.

90

-40.95  
(-2.0)-65.1  
(-2.0)-107.3  
(-1.0)-217  
(-1.0)

-2.0

80

0  
(0)28.9  
(-.444)-31.25  
(-.291)-133  
(-.613)

-337

70

+4.335  
(+.106)+1.3  
(+.02)+20.85  
(+.194)-55  
(-.025)

+.074

60

+3.91  
(+.095)+6.55  
(+.101)+21.4  
(+.199)+44.5  
(+.205)

+.15

50

+1.375  
(+.034)+4.1  
(+.063)+23.8  
(+.222)+40.5  
(+.187)

+.127

40

-.27  
(-.007)+3.7  
(+.057)+11.85  
(+.110)+15.5  
(+.071)

+.093

30

-.585  
(-.014)+3.25  
(+.05)+6.25  
(+.058)+1  
(+.005)

+.025

20

-.45  
(-.011)-.5  
(-.008)+5.35  
(+.050)+1  
(+.005)

+.009

10

-.645  
(-.016)+4.6  
(+.077)+1.65  
(+.015)+2  
(+.009)

+.020

# ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUES

Page       

Calc. By NON-PREFERRED DIRECTION

Checked By       

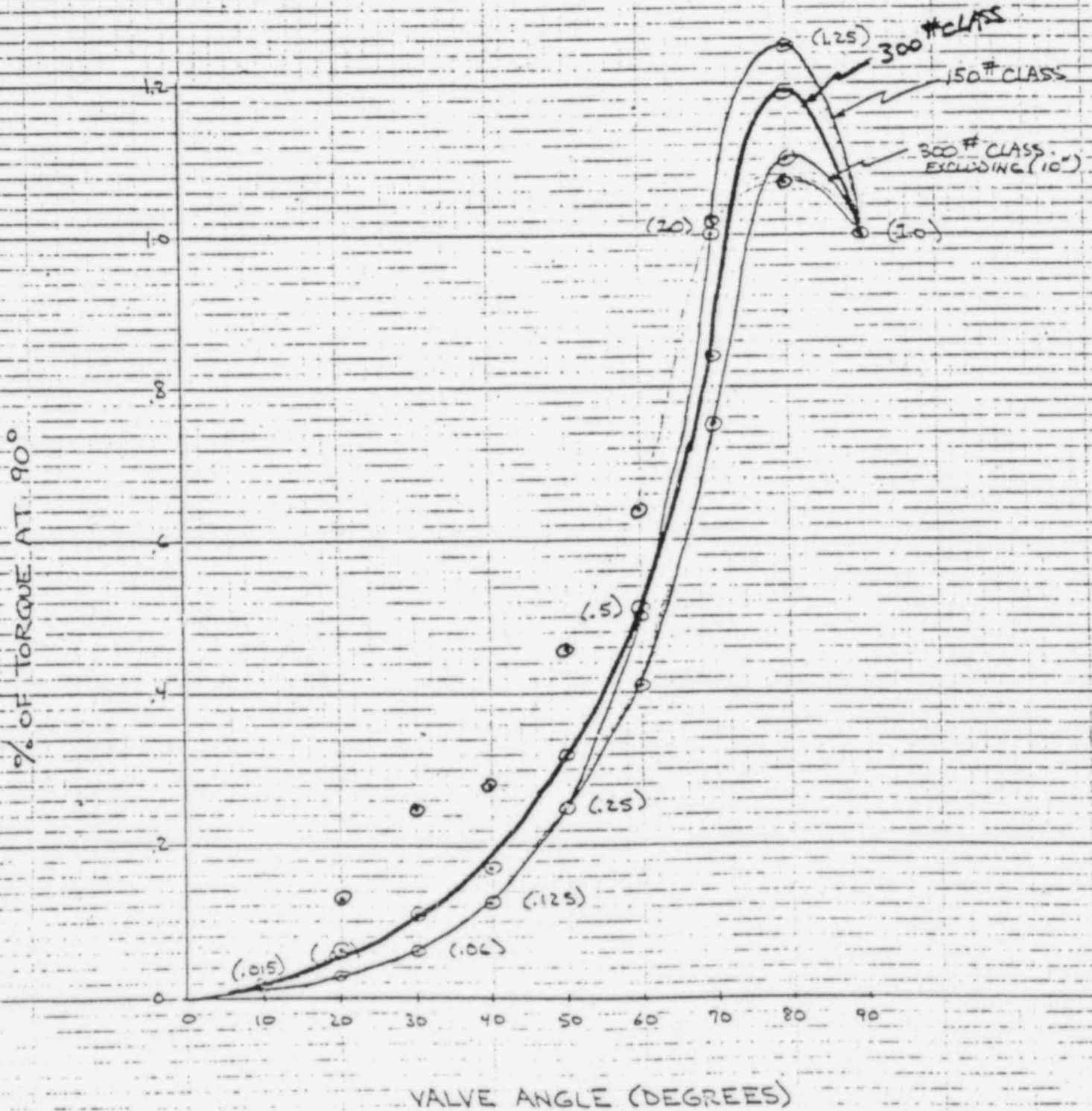
QANSEN 8/16/78

ANGLE	VALVE SIZE - % OF TORQUE AT 90°			
	6"-600	8"-600	10"-600	AVG.
90	<del>-46.775</del> <del>-7.25</del> (-1.0)	-75 (-1.0)	-129.45 (-1.0)	(-1.0)
80	<del>-20.57</del> <del>-3.75</del> (-0.5)	-52.5 (-.7)	-101.2 (-.782)	-6.41
70	<del>+2.73</del> <del>+0.68</del> (+0.63)	-82.5 (-1.1)	+8.75 (+0.68)	+0.07
60	<del>+11.93</del> <del>+2.55</del> (+0.245)	+13.1 (+1.75)	+28.75 (+2.22)	+2.17
50	<del>+12.435</del> <del>+2.55</del> (+0.5)	+17.1 (+2.28)	+28.855 (+2.23)	+2.39
40	8 (+1.71)	+18.5 (+2.41)	+25.9 (+2)	+2.04
30	+1.91 (+0.04)	+14.65 (+1.95)	+9.1 (+0.76)	+0.9
20	-3.1095 (-.066)	+13.9 (+1.85)	+2.6 (+.02)	+0.46
10	-2.212 (-.047)	+7.2 (+.095)	+1.15 (+.009)	+0.21



Title \_\_\_\_\_ Page \_\_\_\_\_  
By \_\_\_\_\_ Preferred Direction \_\_\_\_\_ Checked By \_\_\_\_\_ Hansen 8/8/78

HYDRODYNAMIC TORQUE  
VS.  
VALVE ANGLE



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

HYDRODYNAMIC TORQUES

Page

PREFERRED DIRECTION

Checked By

CANSEN

8/8/78

ANGLE

VALVE SIZE - 9" TORQUE @ 90°

	6"-600	8"-600	10"-600	AVG.
90	(47.3) 1.0	(75-)* 1.0	(128-)* 1.0	1.0
80	(46.2) .98	(91.7) 1.22	(138.15) 1.02	1.07
70	(33.7) .71	(101-) 1.35	(134.05) .99	1.02
60	(16.8) .35	(61.45) .53	(100.2) .74	.64
50	(9.6) .20	(46.4) .62	(76.9) .57	.46
40	(6.4) .13	(25.6) .34	(52.1) .38	.28
30	(5.6) .12	(28-) .37	(36.8) .27	.25
20	(3.6) .08	(9.4) .13	(24.2) .18	.13
10	(.75) -.02	(.4) .01	(4.65) .03	.01

?

\* BASED ON NON PREFERRED DIRECTION.

Title HYDRODYNAMIC TORQUES Page 8/8/78  
 Calc By PREFERRED DIRECTION Checked By QANSEN

ANGLE	VALVE SIZE - % TORQUE @ 90°				AVG (ALL)	AVG (EX. 10° & 20°)
	5" - 150	6" - 150	8" - 300	10" - 300		
90	(28) 1.0	(54.62) 1.0	(95.45) 1.0	(180) 1.0	1.0	1.0 [4.0]
80	(30) 1.07	(60.4) 1.11	(103.6) 1.08	(271.5) 1.5	1.19	1.09 [2.7]
70	(18) .64	(41.15) .75	(77) .81	(205.5) 1.14	.84	.73 [1.5]
60	(9) .32	(26.4) .51	(39.2) .41	(141) .78	.505	.41 [1.1]
50	(5.2) .19	( <del>45</del> ) X 45°	(30.3) .32	(79) .44	.32	.26 [1.25]
40	(3.35) .12	(8.54) .16	(18.1) .19	(39) .22	.17	.16 [1.15]
30	(2.05) .07	(6.32) .12	(9.6) .10	(22.5) .13	.11	.10 [1.1]
20	(1.115) .04	(4.35) .08	(4.35) .05	(13.5) .08	.06	.06 [1.05]
10	(.465) .02	(.88) .02	(3.5) .04	(4.5) .03	.02	.01 [1.01]

\* 5" & 6" - 150# ARE EQUIVALENT TO 300# CLASS VALVES



# HYDRODYNAMIC TORQUES

Page

PREFERRED DIRECTION

Checked By

QANSEN

8/7/78

ANGLE	VALVE SIZE - % TORQUE @ 90°						AVG.	
	10"-150 <sup>(2)</sup>	12"-150 <sup>(2)</sup>	8"-150	10"-150 <sup>(1)</sup>	12"-150 <sup>(1)</sup>			
90	(245) 1.0	(467)(353) 1.0	(24) 1.0	(37) 1.0	(350) 1.0	1.0	1.0	[1.0]
80	(32) 1.27	(425) 2.5, 1.26	(13) .65	(324) 1.03	(434) 1.29	1.25	1.2	[1.2]
70	(227) .93	(315) 1.04	(83) .39	(234) .74	(322) .92	.91	.9	[.9]
60	(122) .50	(186) .56	(-)	(118) .37	(183) .52	.49	.5	[.5]
50	(59) .24	(104) .31	(-)	(59) .18	(77) .22	.24	.25	[.25]
40	(30) .12	(87) .26	(-)	(32) .10	(34) .10	.15	.15	[.15]
30	(16) .07	(33) .10	(-)	(25) .08	(24) .07	.08	.08	[.08]
20	(9) .03	(26) .08	(-)	(14) .04	(18) .05	.05	.05	[.05]
10	(3) .01	(11) .03	(-)	(8) .02	(-) .03	.02	.02	[.02]

(?)

DELETE  
NOT GOOD  
DATA

Title HYDRODYNAMIC TORQUE DETERMINATION

Page \_\_\_\_\_

(by JamesDATE: 12/16/77

Checked By \_\_\_\_\_

VALVE SIZE: 150VALVE DIRECTION: ~~STEM UPSTREAM~~ STEM DOWNSTREAMVALVE SEAL TYPE: TEFLON/BUNAWATER TEMPERATURE (TANK): 66 °FDISC TYPE & DWG:

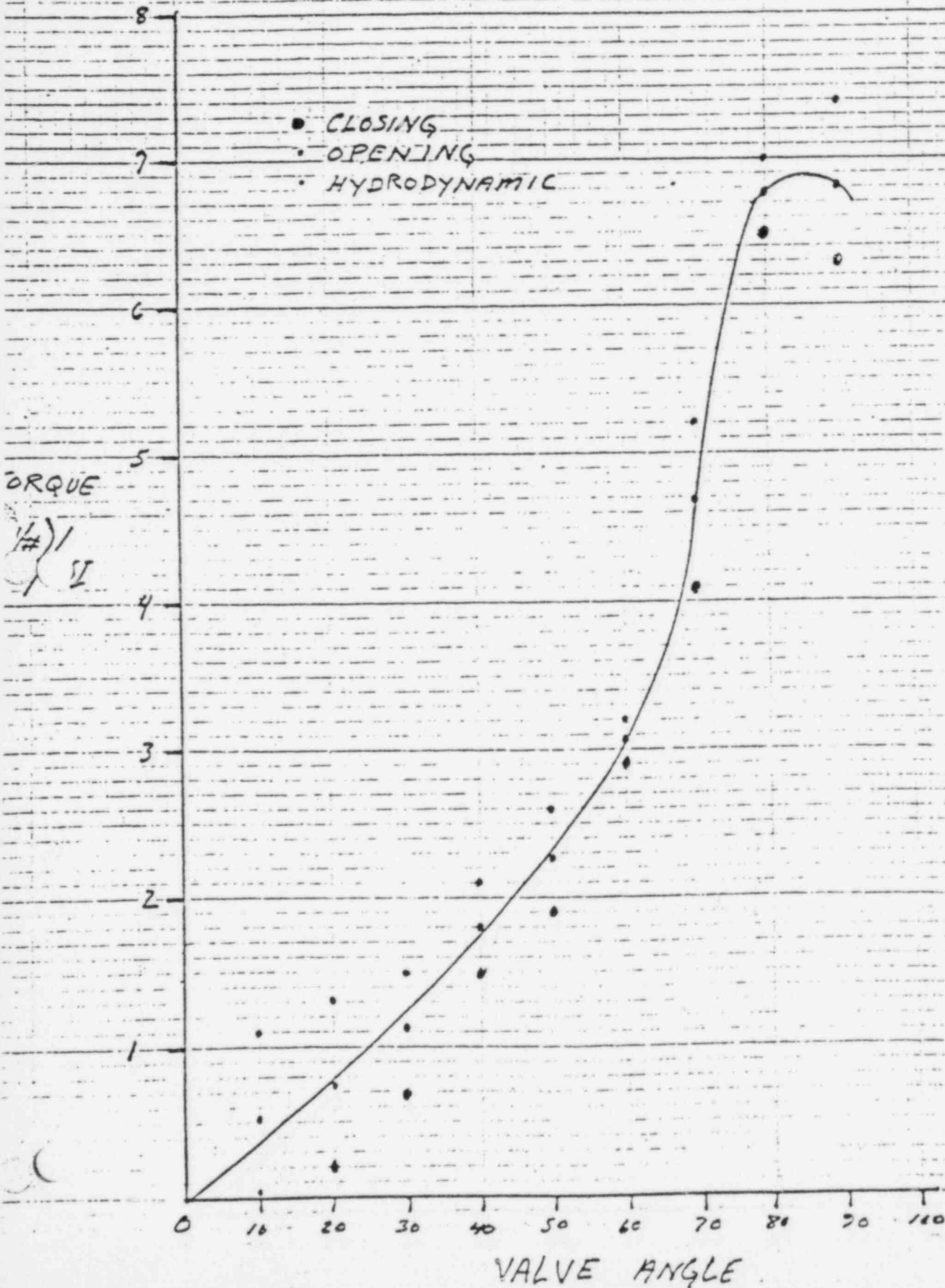
	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	(IN-LB) / PSI
OPENING	0					
	10		43		+45	1.0
	20		41		+45	1.1
	30		38.5		+40	1.0
	40		36		+45	1.3
	50		32		+55	1.7
	60		35		+60	1.7
	70		25		0	0
	80		19		-75	-3.9
	90		23		-90	-3.9
CLOSING	90		23		-90	-3.9
	80		19.5		-75	-4.5
	70		21		-40	-1.9
	60		27.5		-10	-0.4
	50		36		+10	0.3
	40		28.5		0	0.0
	30		30.5		-15	0.5
	20		32		-15	0.5
	10		32		-15	0.5
	0					

PACKING TORQUE: 20 IN-LB OPENING: 0 IN-LB CLOSING

Title 3-150LB HYDRODYNAMIC TORQUE CURVE

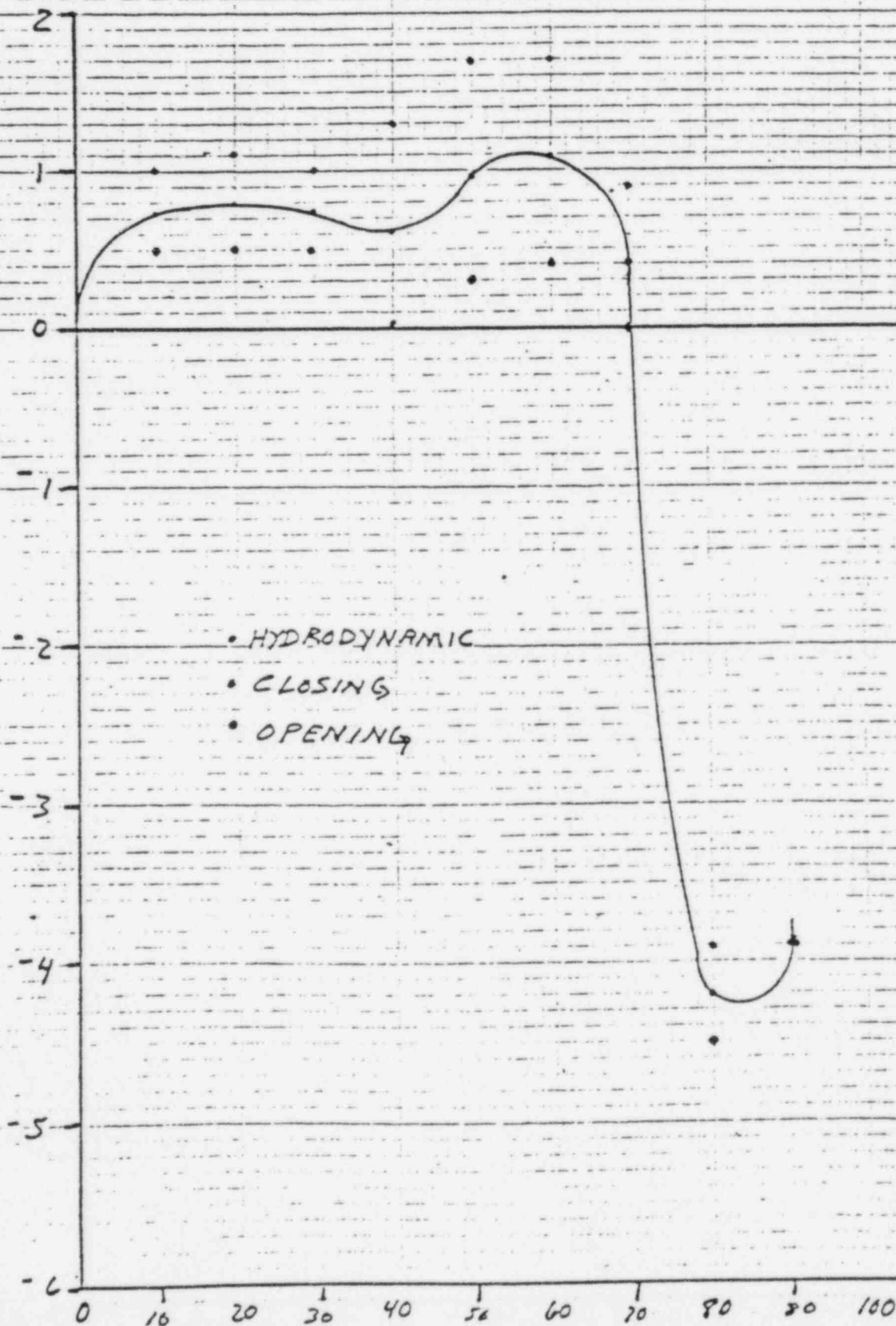
Page     

Calc By C. L. WARD (STEM UP STREAM) Checked By     



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 3" 150LB - HYDRODYNAMIC TORQUE CURVE Page         
By C. Livors (STEM DOWN STREAM) Checked By       





## ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION

Page \_\_\_\_\_

Calc. By JanDATE: 12/5/77

Checked By \_\_\_\_\_

VALVE SIZE: 4"-150VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAMVALVE SEAL TYPE: TEFLON / BUNAWATER TEMPERATURE (TANK): 65 °FDISC TYPE & DWG:

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
0		X		X	
10		46.5		+50	1.07
20		42.5		+75	1.76
30		37		+75	2.03
40		31		+75	2.42
50		32		+100	3.12
60		25.5		+125	4.90
70		17.5		+140	8.00
80		11.5		+150	13.04
90		8.7		+125	14.37
90		X		X	
90		8.5		+75	8.82
80		11		+125	11.36
70		16.5		+125	7.57
60		27		+125	4.63
50		31.5		+85	2.70
40		35		+50	1.43
30		36.3		+40	1.10
20		37.8		0	0
10		38.8		-60	-1.55
0	7	X	0	X	

OPENING

CLOSING

PACKING TORQUE: 5 IN-LB OPENING: 5 IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATION Page 3Calc. By RAY MARSHALL DATE: 11-17-77 Checked By \_\_\_\_\_VALVE SIZE: 4"-150VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAMVALVE SEAL TYPE: TEFLON/BUNAWATER TEMPERATURE (TANK): °FDISC TYPE & DWG:

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
0		X		X	
10		30.5		+35+40	1.31
20		32		+40	1.25
30		30.5		+45	1.47
40		30.5		+60	1.97
50		35		+80	2.28
60		27		+50	1.85
70		18		+49	2.72
80		11		-10	-.91
90		8.5		-55	-6.47
		X		X	
90		8.7		-80	-9.2
80		11		-30	-2.73
70		17.5		0	0
60		27		+15	.56
50		35		+45	1.29
40		43.5		+25	.57
30		29.5		+10	.34
20		29		-5	.17
10		28		0	0
0		X		X	

PACKING TORQUE: 10 IN-LB OPENING5 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

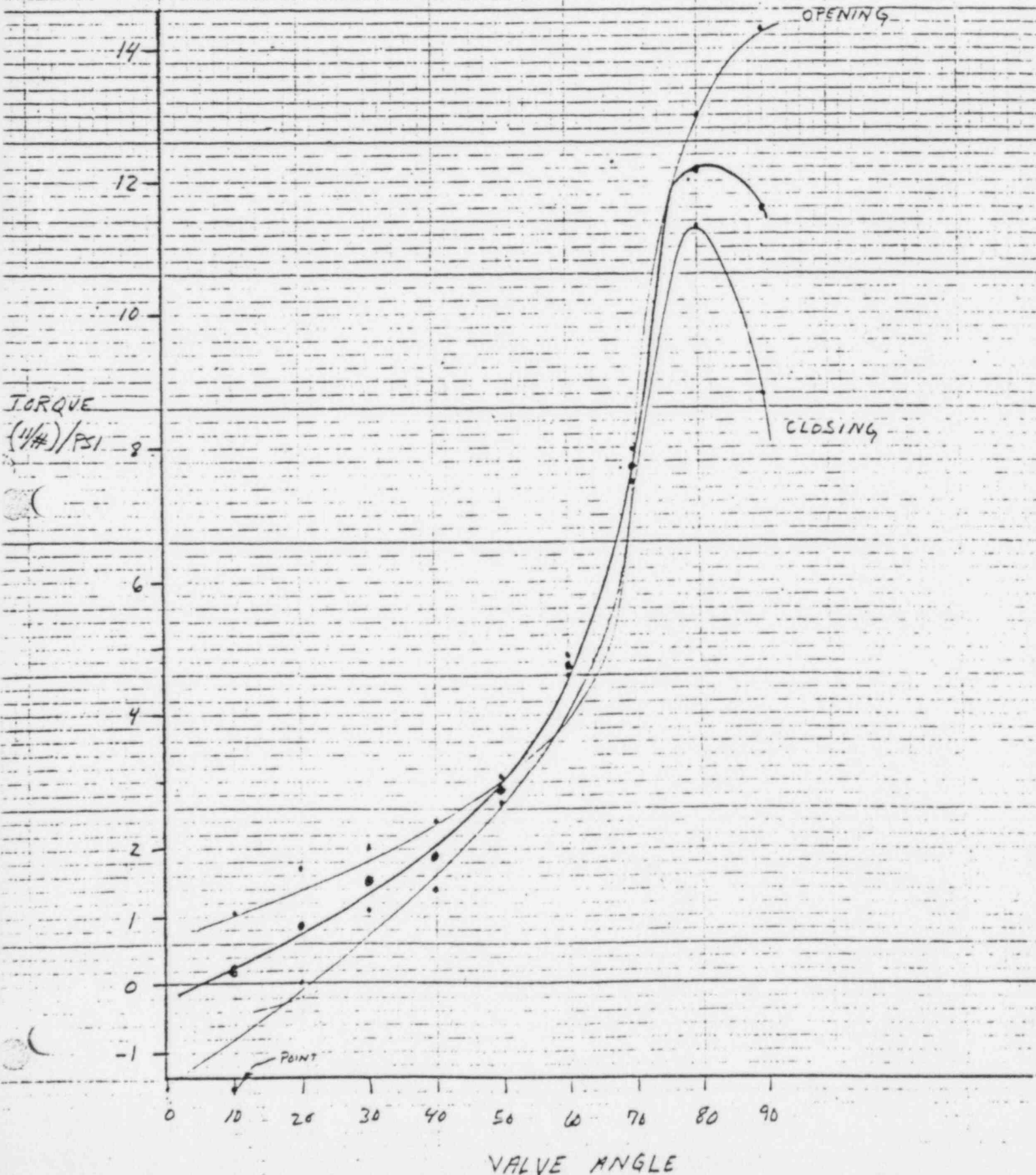
Title 4" 150LB HYDRODYNAMIC TORQUE CURVE

Page     

By C. LIVORSI

(STEM UPSTREAM)

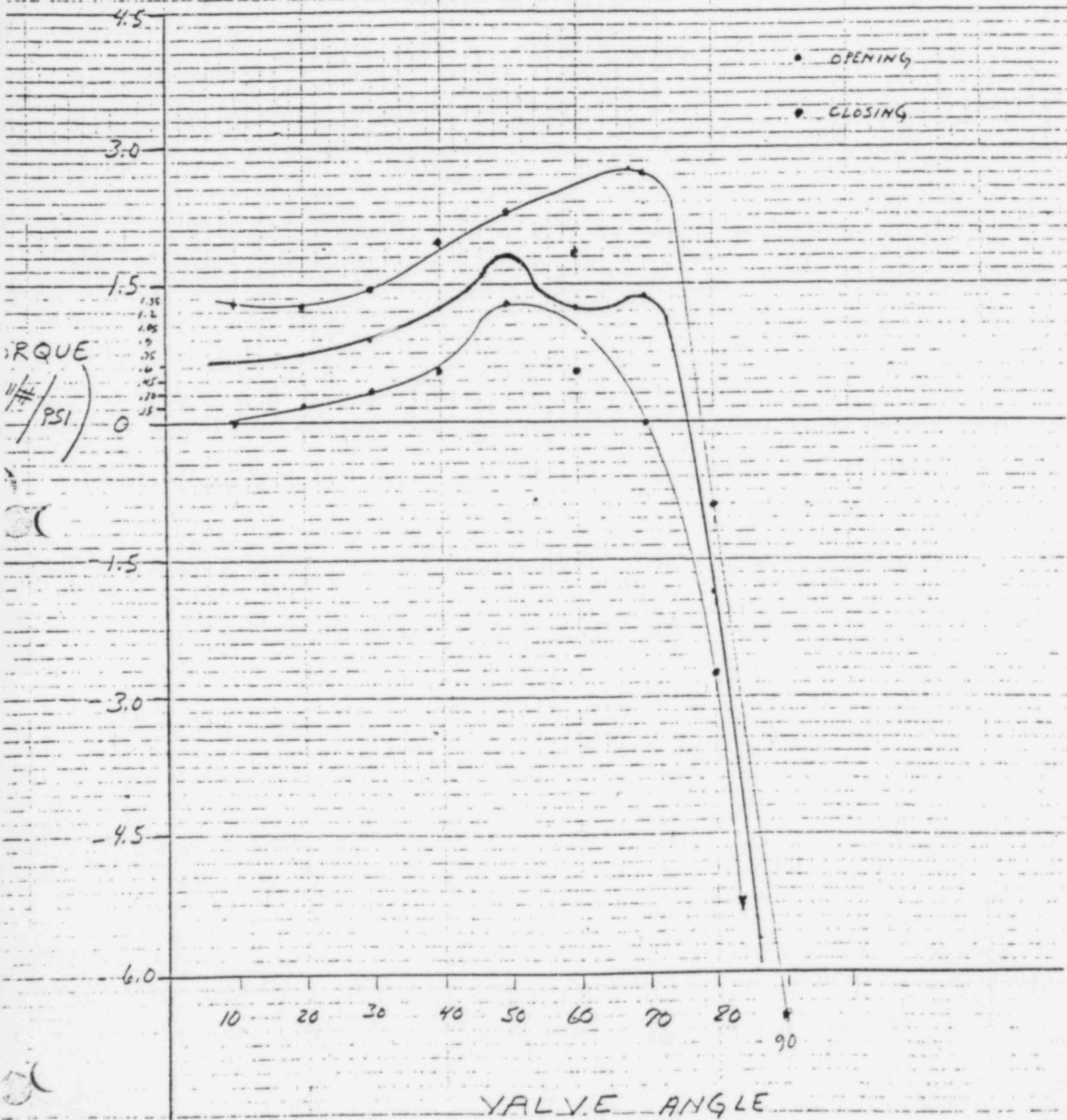
Checked By     





POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 4" 150LB HYDRODYNAMIC TORQUE CURVE Page 2 of 2  
By C. Livorsi (STEM DOWN STREAM) Checked By \_\_\_\_\_



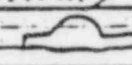
Title HYDRODYNAMIC TORQUE DETERMINATION

Page \_\_\_\_\_

Calc. By T. CORY

DATE: \_\_\_\_\_

Checked By \_\_\_\_\_

CONST. UPSTREAM PRESS.VALVE SIZE: 6" - 850 - 300VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAMVALVE SEAL TYPE: TEF / BUNAWATER TEMPERATURE (TANK): 37 °FDISC TYPE & DWG: 2 

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
0					
10	18.1	12		-50	-4.17
20	18.6	14		0	0
30	18.6	13.5		-50	3.7
40	18.1	13.8		70	5.07
45	18.7	15		100	6.67
60	18.7	14		300	21.43
70	18.5	12.5		500	40
80	18.3	11.5		700	60.87
90	18.4	11		500	54.55
90	18.6	12.8		700	54.69
80	18.4	12.5		750	60
70	18.4	13.2		550-700	42.3
60	18.4	13		440	33.85
45	18.4	15		300	20
40	18.4	12.5		150	12
30	18.4	14		125	8.93
20	18.3	11.5		100	8.7
10	18.1	13.5		80	5.93
0					

PACKING TORQUE:IN-LB OPENINGIN-LB CLOSING

TORQUE FACTOR [IN-LB/PSI]

100

80

60

40

20

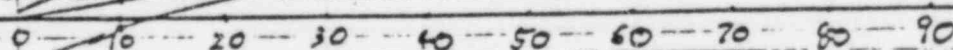
-20

-40

0 10 20 30 40 50 60 70 80 90

VALVE ANGLE

bad point





# ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION Page       

Calc. By        DATE:        Checked By       

VALVE SIZE: 6" 150 - 300

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON / BUCA

WATER TEMPERATURE (TANK): 86 °F

DISC TYPE & DWG: Z


	VALVE ANGLE DEGREES	P, UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
	0					
	10		38.5		180	
	20		1.9		150	
OPENING	30		6.2		80	
	40		7.8		110	
	50		8.2		160	19.5
	60		6.5 13		206 - 540	38.5 21.5
	70		8.75 15		500 - 650	45.7 49
	80		8.5 11.2		530 - 650	62.3 58
	90		9.1 8.5		500 - 500	54.9 59
	90		9.1 8.5		510 - 500	
	80		11.5 11		700 - 600	10.8 54
	70		16.7 16.2		730 - 750	43.7 46
	60		14.5 22.6		350 - 520	24.1 23
	50		14.5		150	2.4
CLOSING	40		12.5		60	
	30		15		60	
	20		34		90	
	10		50.5		25	
	0					

PACKING TORQUE: IN-LB OPENING

IN-LB CLOSING



## ENGINEERING CALCULATIONS

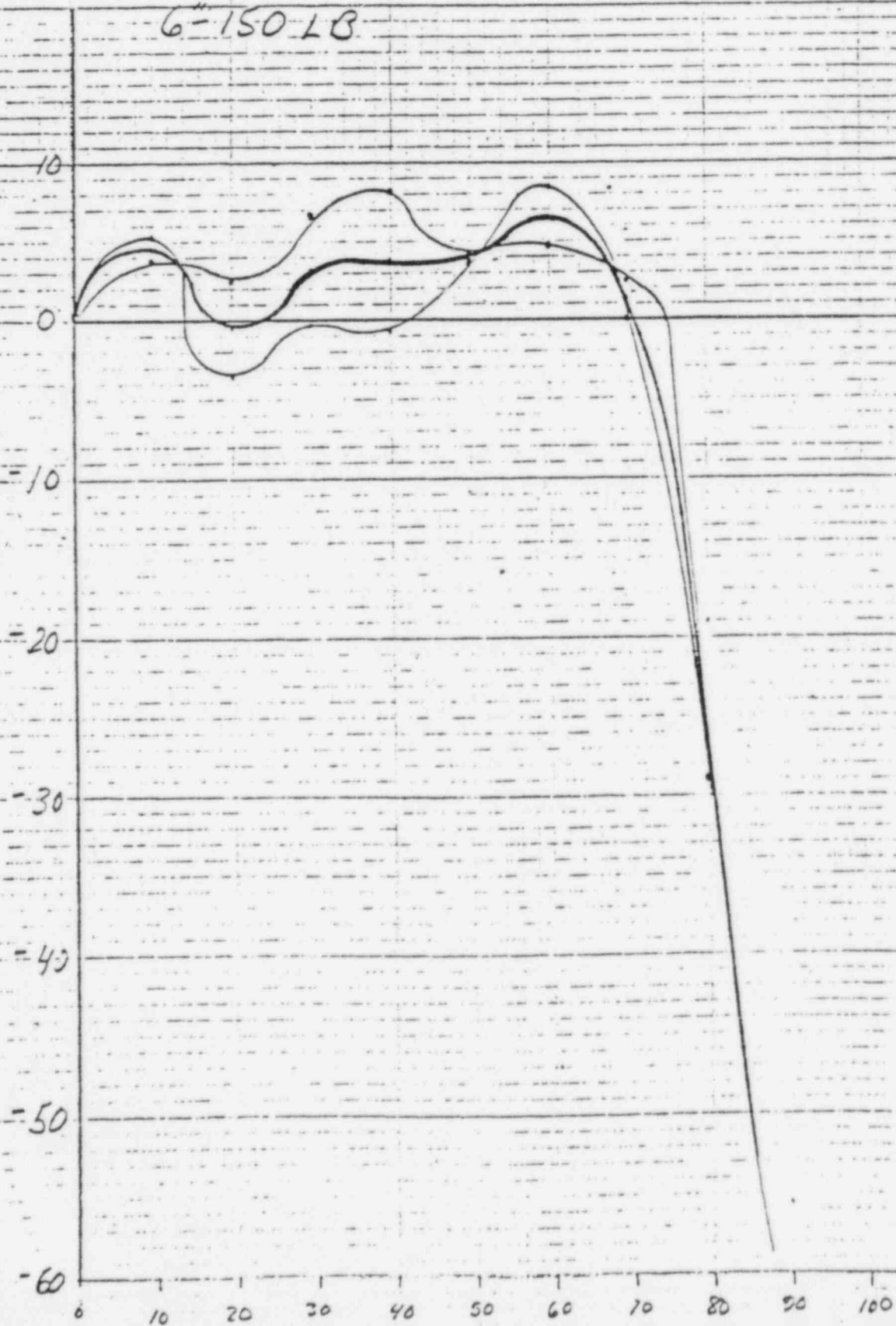
Title HYDRODYNAMIC TORQUE DETERMINATION Page \_\_\_\_\_Calc. By R. OANSEN DATE: 9/20/77 Checked By \_\_\_\_\_VALVE SIZE: 6" - 150/300VALVE DIRECTION: ~~STEM UPSTREAM~~ / STEM DOWNSTREAMVALVE SEAL TYPE: TEF/BUNA  TYPE 2.WATER TEMPERATURE (TANK): 63° FDISC TYPE & DWG:

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	( $\frac{1}{2}$ in) / PSI
	0		<del>XXXX</del>		<del>XXXX</del>	
	10		44.5		+175	3.9
	20		30.0		+75	2.5
OPENING	30		11		+75	6.8
	40		11		+90	8.2
	50		10.2		+45	4.4
	60		9.7		+45	4.6
	70		5.7		+15	2.6
	80		9.2		-275	28.0
	90		8.4		-550	65.5
			<del>XXXX</del>		<del>XXXX</del>	
	90		8.5		-550	64.7
	80		9.0		-260	28.9
	70		23.5		+200	0.11
	60		23.5		+200	8.5
	50		22.5		+85	3.8
CLOSING	40		18.5		-15	0.8
	30		19.0		-5	0.3
	20		31.5		-110	3.5
	10		28.5		-150	5.3
	0		<del>XXXX</del>		<del>XXXX</del>	

PACKING TORQUE: 10 IN-LB OPENING: 10 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE CURVE Page         
By C. LIVORSI (STEM DOWN STA) Checked By       



Title HYDRODYNAMIC TORQUE DETERMINATION Page 5  
By C. LIVORSI DATE: \_\_\_\_\_ Checked By \_\_\_\_\_

VALVE SIZE: 8" ISOLB

VALVE DIRECTION: STEM UPSTREAM

VALVE SEAL TYPE:

WATER TEMPERATURE (TANK): 71 °F

DISC TYPE & DWG:

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	( $\frac{1}{2}$ ) / 95
OPENING	0	69.5	65.		400	
	10		59.5		100	1.68
	20		56		425	7.6
	30		51.5		550	10.7
	40		43		725	16.2
	50		31		775	25
	60		18		740	41.1
	70		8.5		700	92.3
	80		4.5		650	144.4
	90		2.5		600	240
CLOSING	90		2.0		375	187.5
	80		3.0		400	133.1
	70		6.0		500	93.3
	60		12.5		550	44
	50		26.5		575	21.7
	40		38.5	37	400	10.6
	30		37.5	47	300	7.7
	20		41.5	54	300	7.3
	10		54.5		100	1.8
	0				375	

PACKING TORQUE: 160 IN-LB OPENING

: 80 IN-LB CLOSING



Title HYDRODYNAMIC TORQUE DETERMINATION

Page 6

By C. LIVERSI

DATE: \_\_\_\_\_

Checked By: \_\_\_\_\_

1st pump

VALVE SIZE: 8" ISO

VALVE DIRECTION: ~~UPSTREAM~~ / STEM DOWNSTREAM

VALVE SEAL TYPE: \_\_\_\_\_

WATER TEMPERATURE (TANK): 70 °F

DISC TYPE & DWA: \_\_\_\_\_

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
0				900 $\frac{1}{16}$	
10		58		350	6.03
20		56		350	6.25
30		54		375	7.35
40		40		415	10.4
50		28.5		450	15.8
60		15		460	30.7
70		7.5	AP @ 75° = 8.5 T @ 75° = 600	450	60
80		5.3	AP @ 85° = 4.5 T @ 85° = 0	410	72.1
90		2.6		450	175
90		3.0		550	183
80		3.0	AP @ 85° = 2.5 T @ 85° = 360	0	0
70		7.5	AP @ 75° = 8.5 T @ 75° = 375	500	66.7
60		16		550	34.4
50		28		400	14.5
40		39.5		250	6.3
30		46		100	2.2
20		54.5		0	0
10		58		100	1.7
0				900 $\frac{1}{16}$	

PACKING TORQUE: 150 IN-LB OPENING

: 100 IN-LB CLOSING

# POSI-SEAL INTERNATIONAL, INC. ENGINEERING CALCULATIONS

Title 8" 150 LB  
HYDRODYNAMIC TORQUE (PREFERRED) Page         
By C. LIVORSI Checked By       

300

200

100

0

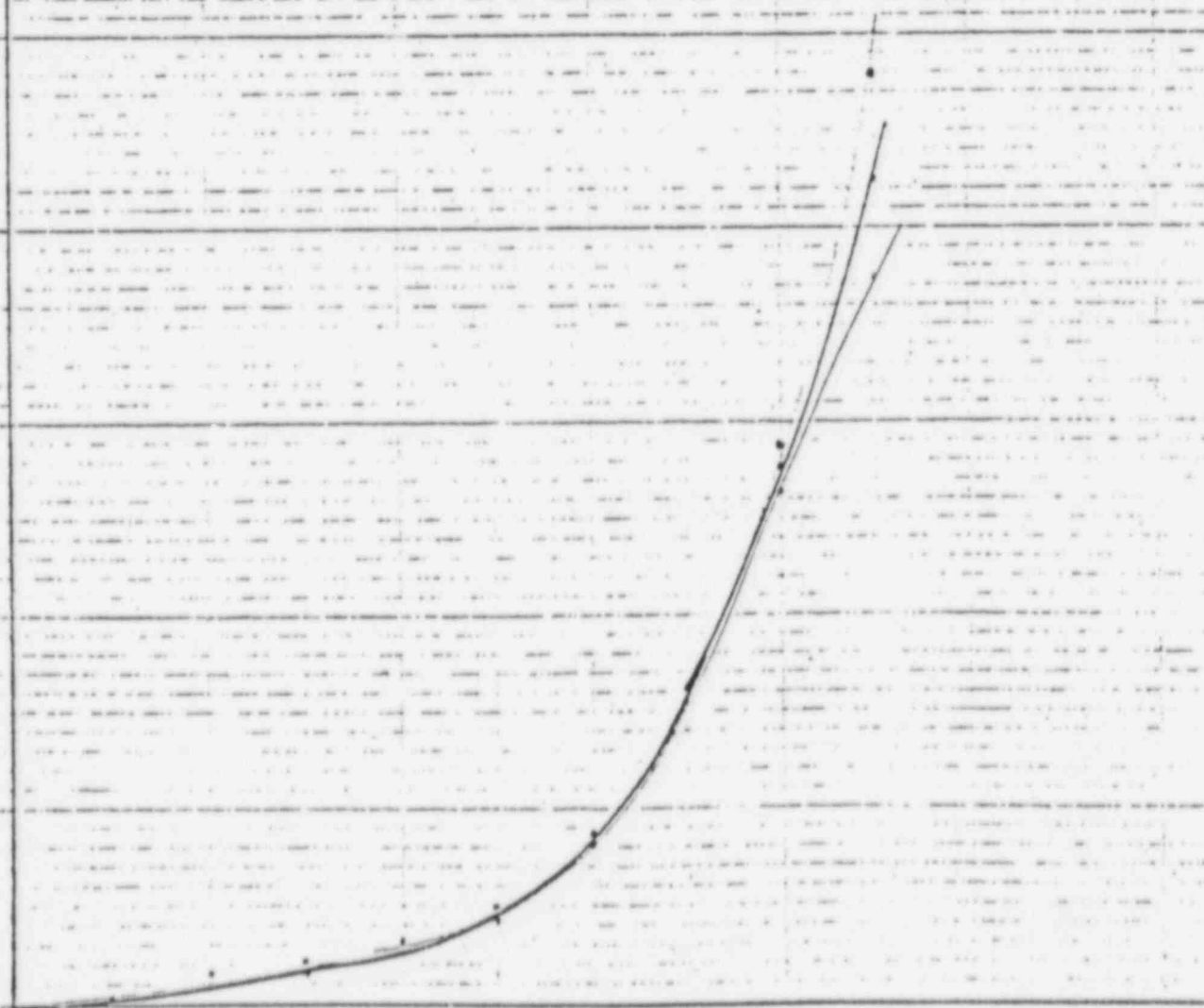
20

40

60

80

100



1/4" #1  
CPSI

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

8"-150

Title HYDRODYNAMIC TORQUE CURVE--(UNPREFERED) Page \_\_\_\_\_  
By C. LIVORSI Checked By \_\_\_\_\_

150

100

50

0

50

100

150

200

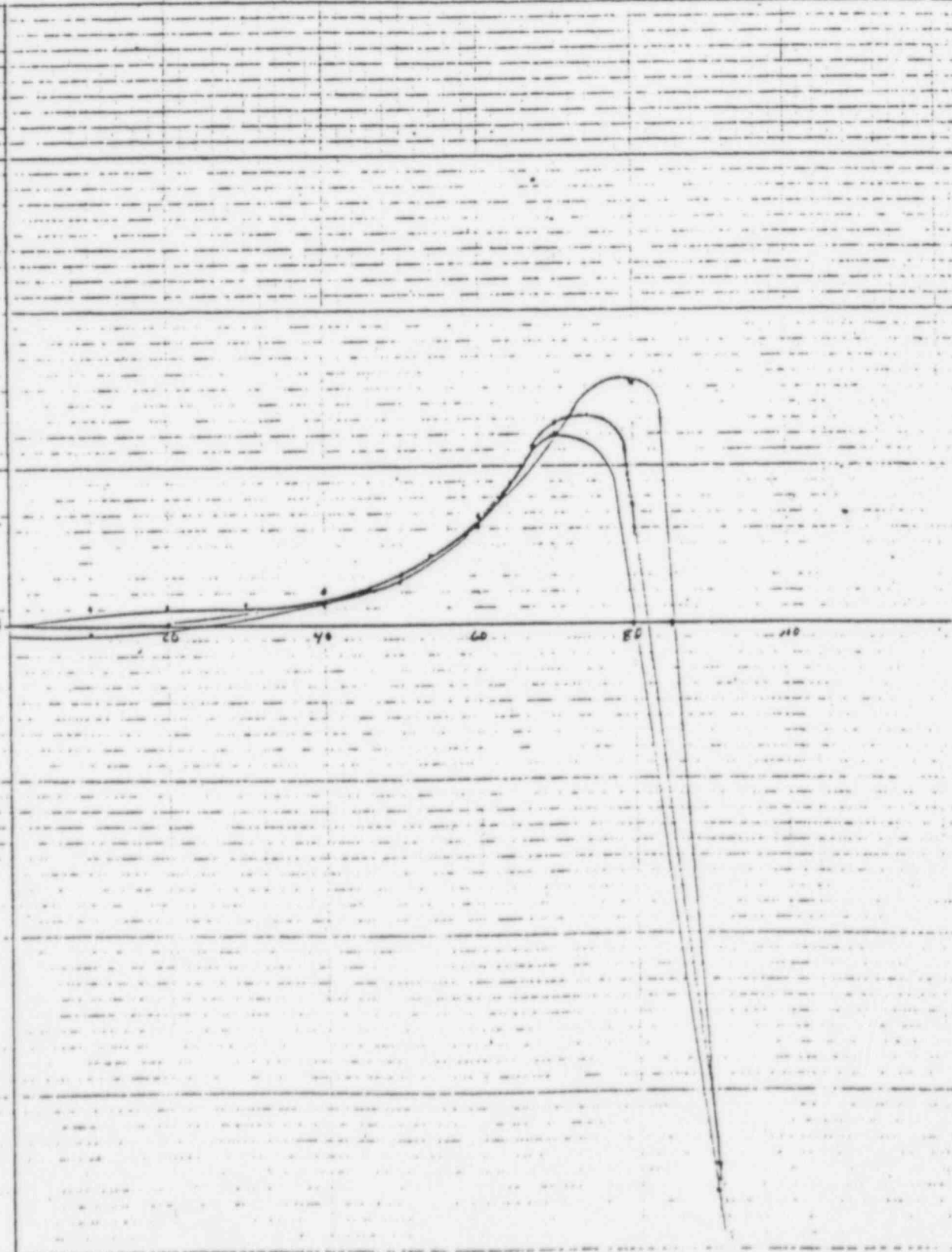
20

40

60

80


100



Title HYDRODYNAMIC TORQUE DETERMINATION (FI) Page         
By RO DATE: 6/17/97 Checked By       

VALVE SIZE: 10" - 150

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON BUNA / DISC TYPE: 2 

WATER TEMPERATURE (TANK): 98° °F

DISC TYPE & DWG:

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	$\frac{10^{-5} T}{\Delta P S}$
OPENING	0	54.9	<del>226.51</del>	0	41250 / 1150 / 1200	
	10	54.9	50.5	395	850	16.8
	20	30	25.5	895	550	21.6
	30	16	11.5	1030	350	30.4
	40	8.5	3.6	1100	150	41.7
	50	25	15	3560	<del>250</del> 900	60
	60	18.5	6.3	3590	850	134.9
	70	16.0	3.2	3590	800	250
	80	15.0	2.2	3580	800	363.67
	90	41	<del>75.101</del>	2685	600	600
CLOSING	90	<del>46</del>	<del>1.5</del>	<del>2623</del>	390	<del>1000</del>
	90	40	1.5	2623	150	75
	80	40	2.0	2863	250	125
	70	41	3.0	2720	275	94.7
	60	41	4.0	2620	300	75
	50	42	7.0	2538	<del>250</del> 350-400	57.2
	40	45	17.0	2260	375	22
	30	51	32.0	1800	225	10
	20	<del>51</del> 54	51	1195	0	
	10	61	56	360	0	
	0					

PACKING TORQUE: 50 IN-LB OPENING


: 50 IN-LB CLOSING



Title HYDRODYNAMIC TORQUE DETERMINATION (2) Page 2  
By NC DATE: 6/17/77 Checked By

VALVE SIZE: 10"-150

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON / BUNA DISC TYPE: 2 

WATER TEMPERATURE (TANK): 80 °F

DISC TYPE & DWG:

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	IN-LB ΔPSE
OPENING	0	35.8	16.5	218	329	
	10	<del>44.5</del>	<del>19.4</del>	<del>250</del>	<del>521</del>	
	20	<del>51.5</del>	<del>10.5</del>	<del>1670</del>	<del>578</del>	
	30	42.8	3.2	1740	750	23.4
	40	22.2	1.5	2217	900	26.7
	50	16.6	7.5	2727	450	60
	60	20.2	6.0	3587	750	125
	70	17.6	3.5	3594	700	200
	80	16.6	2.5	3596	700	280
	90	16.0	1.75	3595	450	259.1
CLOSING	90	↓	↓	↓	300	171.4
	80	16.3	2.0	3583	650	325
	70	17.3	3.0	3597	700	233
	60	19.8	6.0	3599	725	120.8
	50	25.5	13.5	3572	800	
	40	49.5	14.0	2140	450	
	30	57.2	11.0	1180	300	
	20	62.1	14.0	732	300	
	10	65.9	15.0	311	200	
	0					

PACKING TORQUE: 50 IN-LB OPENING

: 50 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS


Title HYDRODYNAMIC TORQUE DETERMINATION #3. Page         
By        DATE: 6/20/77 Checked By       

VALVE SIZE: 10"-150

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON/BUNA

WATER TEMPERATURE (TANK): 78 °F

DISC TYPE & DWG: 2 

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	IN-LB P.S.
OPENING	0	5.5/31.5				
	10	45.7/42.3	33/23.5	300/232	850/750	1100/1175
	20	17.5/5.2	17.5		700	40
	30	8.5			450	53
	40	17.5			700	51
	50	7.0			575	82
	60	5.0			675	135
	70	2.5			625	250
	80	2.0			725	363
	90	1.5			550	367
CLOSING	90		1.5		400	267
	80		2.0		575	281
	70		3.0		650	277
	60		6.0		600	100
	50		13.0		450	35
	40		26.0/15		300/150	12/8
	30		23		-100	-4
	20		23		-275	-12
	10		33.5		-600	-18
	0					

PACKING TORQUE: 50 X IN-LB OPENING

: 50 X IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATION #4 Page       

By        DATE: 6/20/77 Checked By       

CAVITATION measurements - control valve wide open.

VALVE SIZE: 10"=150

VALVE DIRECTION: STEM UPSTREAM / ~~STEM DOWNSTREAM~~

VALVE SEAL TYPE: TEFLON / BUNA

WATER TEMPERATURE (TANK): 77 °F

DISC TYPE & DWG: 2 

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
	0		X		X	
OPENING	10		51		1200	24
	20		50.5		1400	25
	30		43.5		1575	36
	40		28.5		1450	51
	50		13.5		1100	81
	60		6.25		925	142
	70		3.0		825	275
	80		2.25		800	356
	90		2.0		550	295
	90		1.75		375	214
CLOSING	80		2.25		600	267
	70		3.5		625	170
	60		6.75		650	96
	50		14.25		525	37
	40		29.5		275	9
	30		44.5		200	4
	20		51.5		550	11
	10		58.5		1100	19
	0					

PACKING TORQUE: 50 X IN-LB OPENING

: 50 X IN-LB CLOSING




Title HYDRODYNAMIC TORQUE DETERMINATION Page         
By          DATE: 6/21/77 Checked By         

VALVE SIZE: 10" - 150

VALVE DIRECTION: ~~STEM UPSTREAM~~ STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON/ISMA

WATER TEMPERATURE (TANK): 76 °F

DISC TYPE & DWA: 2 

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔF PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
	0					
	10		42.5		-800	19
	20		28		-650	23
OPENING	30		11.5		-425	27
	40		13.0		-525	30
	50		10.5		-700	67
	60		6.5		-775	119 T=0
	70		3.5		-550	75 2.4
	80		1.7		-175	103 47
	90		1.7		-625	268
	90		1.8		-700	239
	80		1.8		-50	25
	70		3.0		+200	67
	60		6.5		+450	69
	50		13.5		+275	20
CLOSING	40		16.8/8.5		-75	9
	30		16.5/19/30.5		-50 / -50 / -27	5.3 /
	20		17.5		-325	19
	10		18.5		-350	19
	0					

PACKING TORQUE: 50 IN-LB OPENING

: 50 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION Page         
By Chuan DATE: 6/2/77 Checked By:       


Full open - control valve.

VALVE SIZE: 10" - 150

VALVE DIRECTION: OPEN UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON / Buna.

WATER TEMPERATURE (TAH): 80 °F

DISC TYPE & DWG: 2 

	VALVE ANGLE DEGREES	P, UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
OPENING	0				
	10		57.5		+1050
	20		47.7		+850
	30		39.5		+750
	40		26.5		+800
	50		14.5		+925
	60		7.0		+800
	70				
	80				
	90				
CLOSING	90				
	80				
	70				
	60		6.5		+350
	50		13.5		+200
	40		25		0
	30		39.5		-450
	20		51.0		-100
	10		58.5		-800
	0				

PACKING TORQUE: 50 IN-LB OPENING

: 50 IN-LB CLOSING

# POST-SEAL INTERNATIONAL, INC. ENGINEERING CALCULATIONS

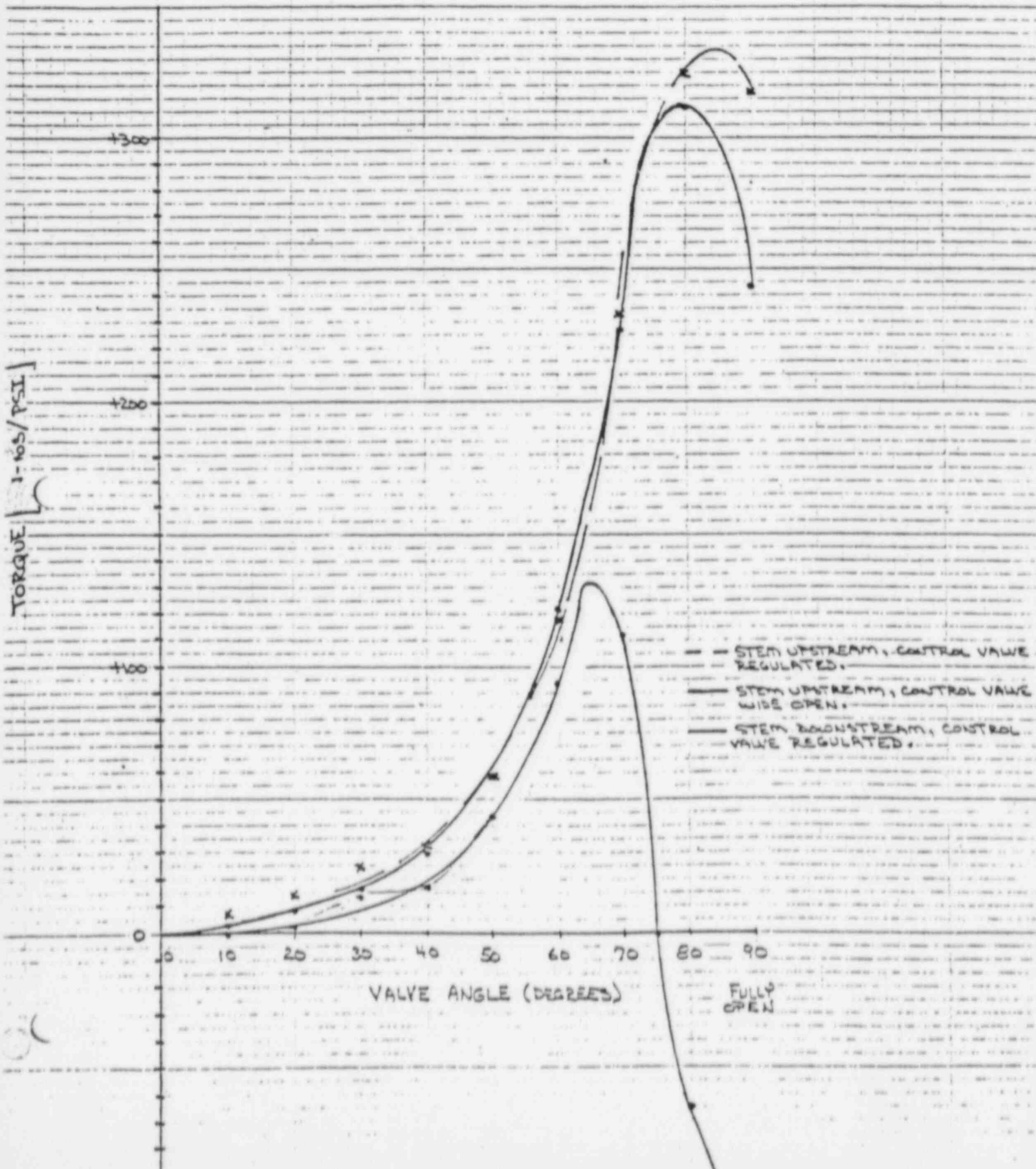
Title TORQUE VS. VALVE ANGLE

10'-150

Page

By R. CHANSEN

Checked By





POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION Page       

3y C. LIVORSI DATE:        Checked By       

VALVE SIZE: 12"-150

VALVE DIRECTION: ☒ STEM UPSTREAM / ☐ STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON

WATER TEMPERATURE (TANK):        °F

DISC TYPE & DWG:       

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0				
10	50	-1400	-28	
20	36.5	-1300	-36	
30	20.5	-850	-41	
40	20.5	-1000	-49	
50	8.5	850	100	
60	3.5	825	236	
70	2.3	750	326	
80	1.5	760	467	
90	1.0	500	500	
90	1.0	200	200	
80	1.0	400	400	
70	1.5	475	317	
60	3.5	750	129	
50	8.0	425	53	
40	17.5	325	19	
30	36.5	225	6	
20	48.5	0	0	
10	55	550	-10	
0				

PACKING TORQUE: 150 IN-LB OPENING

: 200 IN-LB CLOSING



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION Page 2  
By C. Livorsi DATE: \_\_\_\_\_ Checked By \_\_\_\_\_

VALVE SIZE: 12" 150

☒ VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM ☐

VALVE SEAL TYPE: TEFLON

WATER TEMPERATURE (TANK): \_\_\_\_\_ °F

DISC TYPE & DWA: \_\_\_\_\_

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	62 55	1800 1000		
10	41.5	950	23	
20	25	900	36	
30	12	550	46	
40	2.5	350	140	
50	3	400	133	
60	3.5	650	186	
70	1.8	600	333	
80	1.2	600	500	
90	1.2	300	333	
90	1.9	10	0	
80	1	350	350	
70	1.4	500	357	
60	2.7	500	185	
50	2.4	550	74	
40	16.5	550	33	
30	18.5	350	19	
20	30.6	450	15	
10	24.5	50	2	
0	62	1210		

PACKING TORQUE: 150 IN-LB OPENING

: 150 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION

Page       

By C. LIVORSI DATE: 7-7-78 Checked By       

VALVE SIZE: 12" - 150

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON

WATER TEMPERATURE (TANK): 67 °F

DISC TYPE & DWG:

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	62	1000		
10	21	550	-26	
20	5	300	60	
30	10	450	45	
40	7.7	450	58	
50	5.5	350	63	
60	4.0	650	162	
70	1.9	450	236	
80	1.2	200	166	87° = 0
90	.8	300	375	
90	.8	450	562	
80	.8	100	125	
70	1.6	200	125	
60	4.0	300	75	
50	9.5	100	11	
40	16	0	0	
30	30	-50	-2	
20	44	-550	-13	
10	53	-1150	-22	
0	63	100	-2	

PACKING TORQUE: 125 IN-LB OPENING

: 150 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

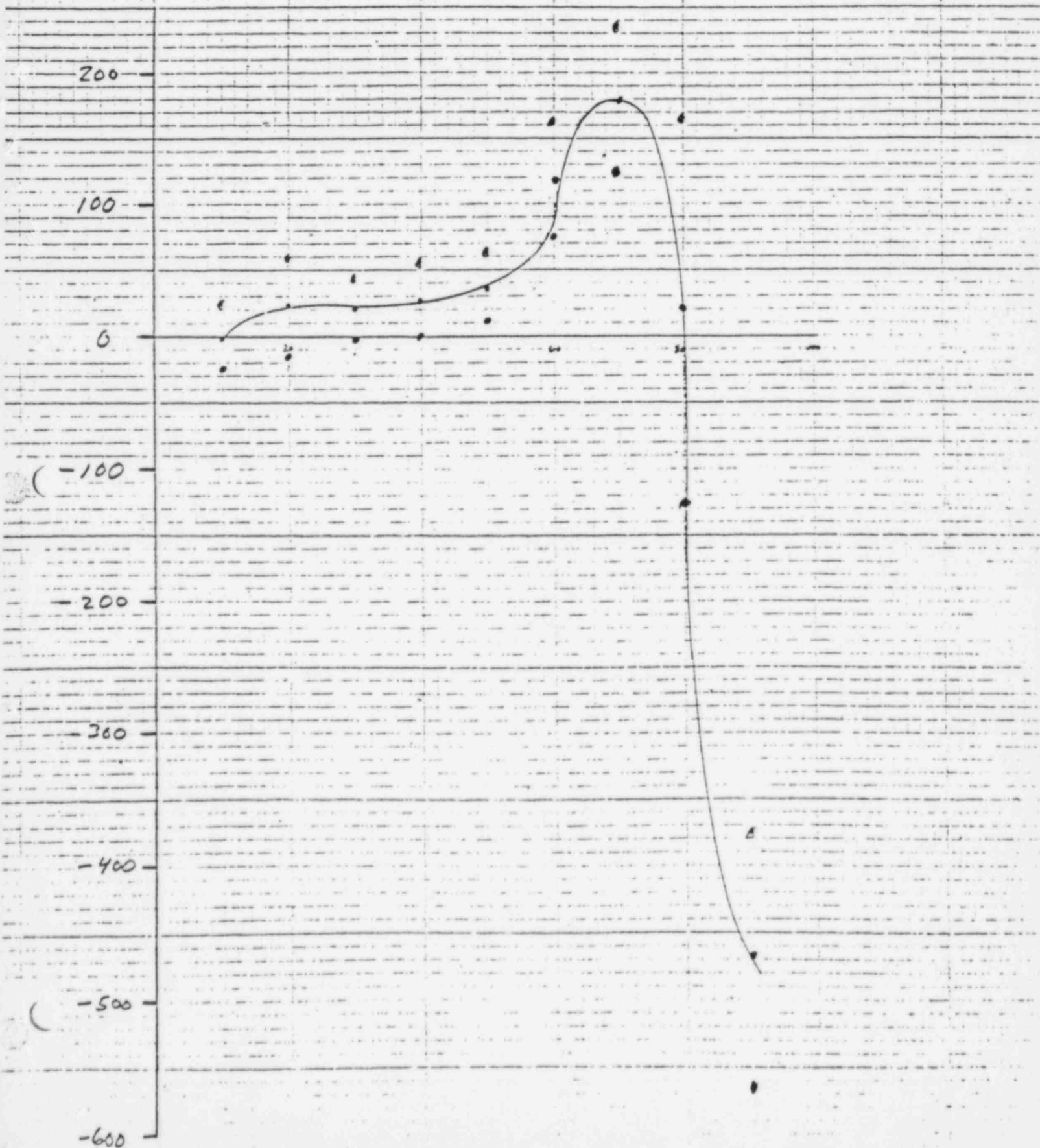
Title HYDRODYNAMIC TORQUE CURVE 12<sup>N</sup> 150

Page 1

By C. LIVORSI

Checked By                     

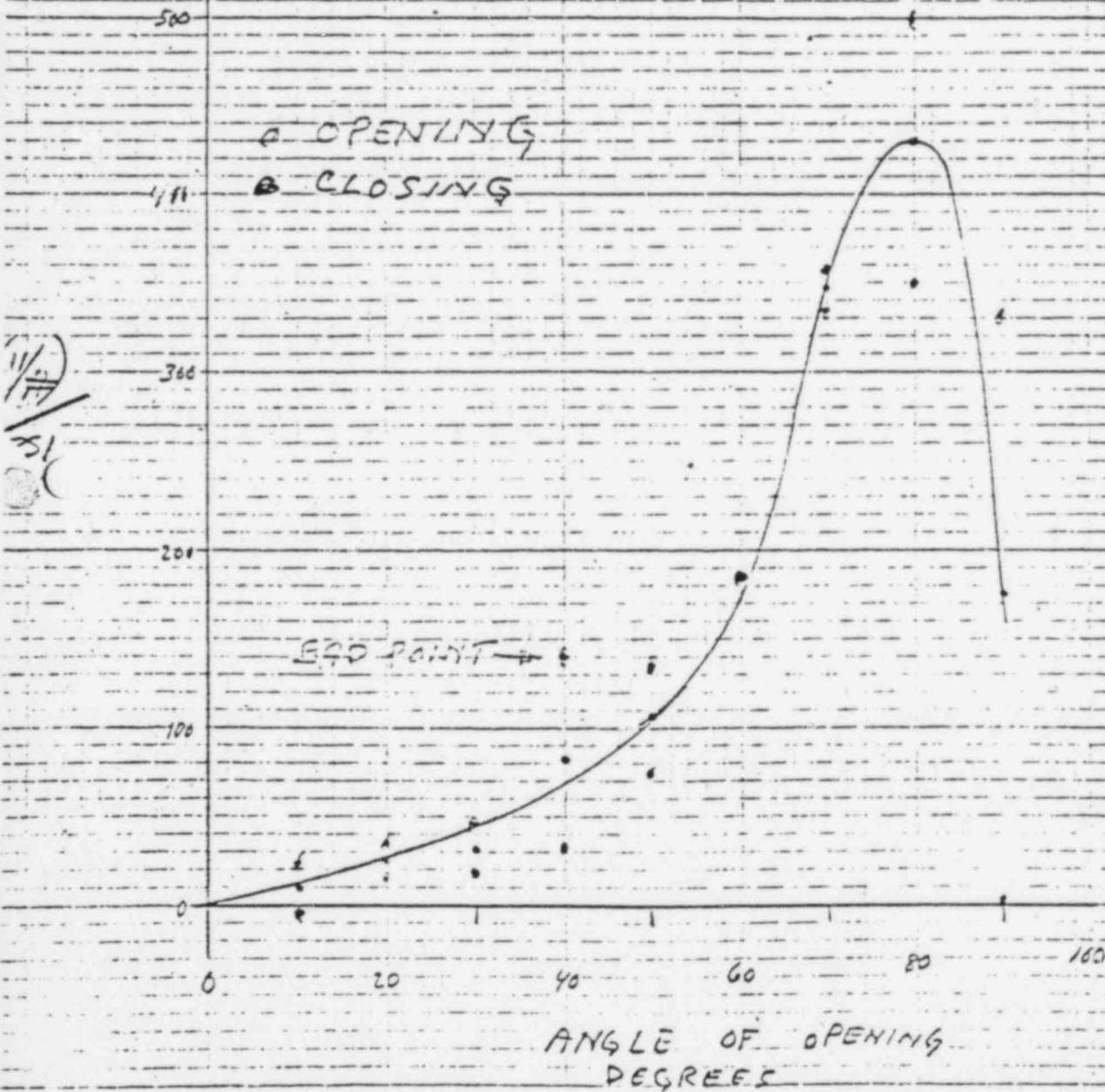
STEM DOWNSTREAM





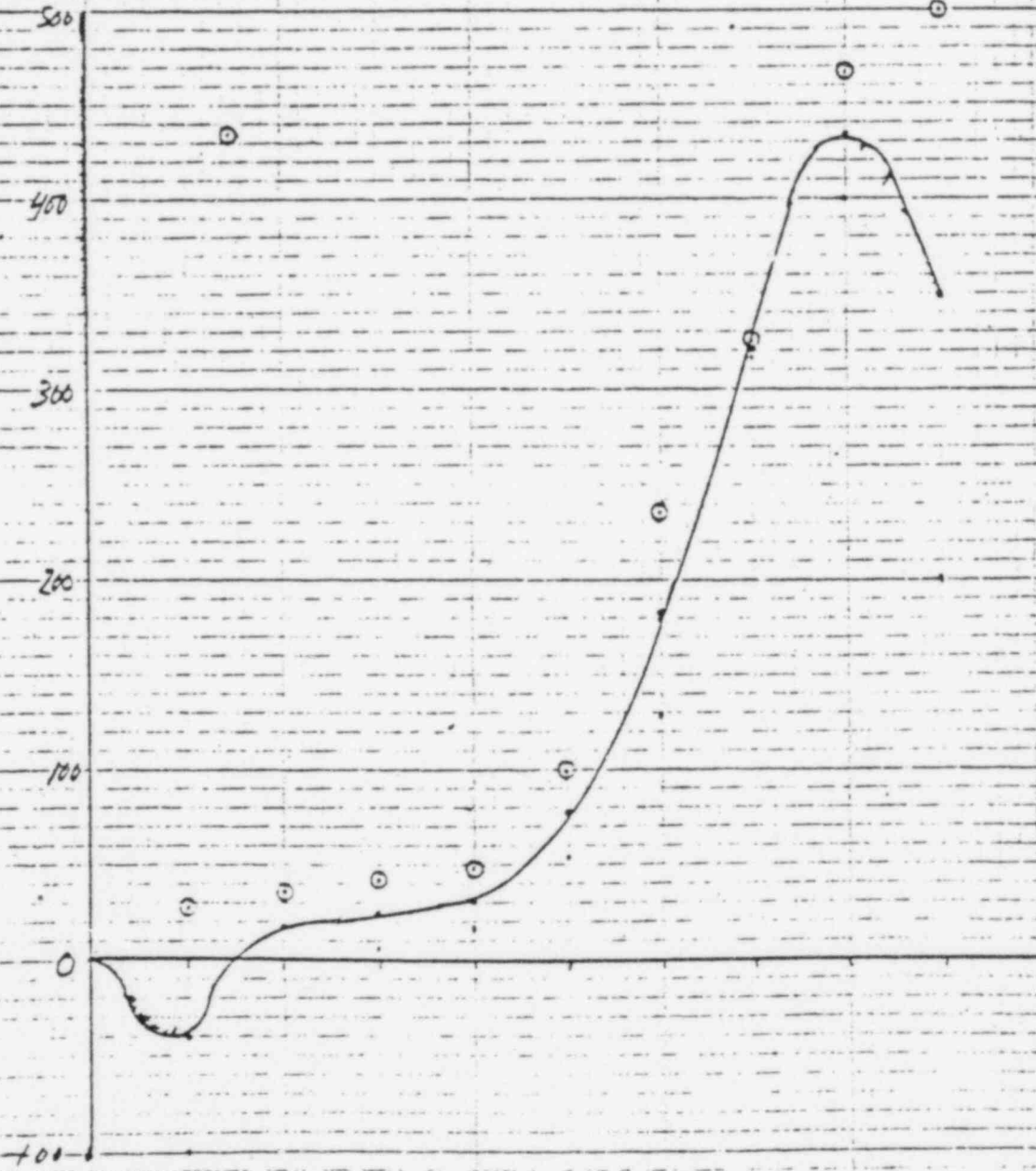
POST-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 12-150 STEM UP STREAM HYDRO DYNAMIC TORQUE CURVE Page         
By C. LIVORSI Checked By       



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 12" ISO (PREFERED) HYDRODYNAMIC TORQUE CURVE Page         
( By C. LIFORSI Checked By       



Title HYDRODYNAMIC TORQUE DETERMINATION Page \_\_\_\_\_

By \_\_\_\_\_ DATE: \_\_\_\_\_ Checked By \_\_\_\_\_

VALVE SIZE: 14"-150

VALVE DIRECTION: ☐ STEM UPSTREAM / ☒ STEM DOWNSTREAM

VALVE SEAL TYPE:

WATER TEMPERATURE (TANK): 65 °F

DISC TYPE & DWG:

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS) <del>FT-LBS</del>	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0				
10	49.5	525	11	21.5 275
20	41	700	17	7 400
30	26.5	900	34	20.5 800
40	16	750	47	
50	7.2	750	104	
60	3	700	233	
70	1.5	600	400	
80	1	300	300	
90	.6	-275	-458	
90	.6	-275	-458	
80	.7	-50	-71	
70	1.3	300	231	
60	3	500	167	
50	7.2	450	62.5	
40	15.6	200	13	
30	26.6	150	6	
20	23.6	408	17	
10	42.5	-300	-7 + 9	22 - 200
0				

PACKING TORQUE: 5 FT-LB ~~HOLD~~ OPENING  
: 5 FT-LB ~~HOLD~~ CLOSING

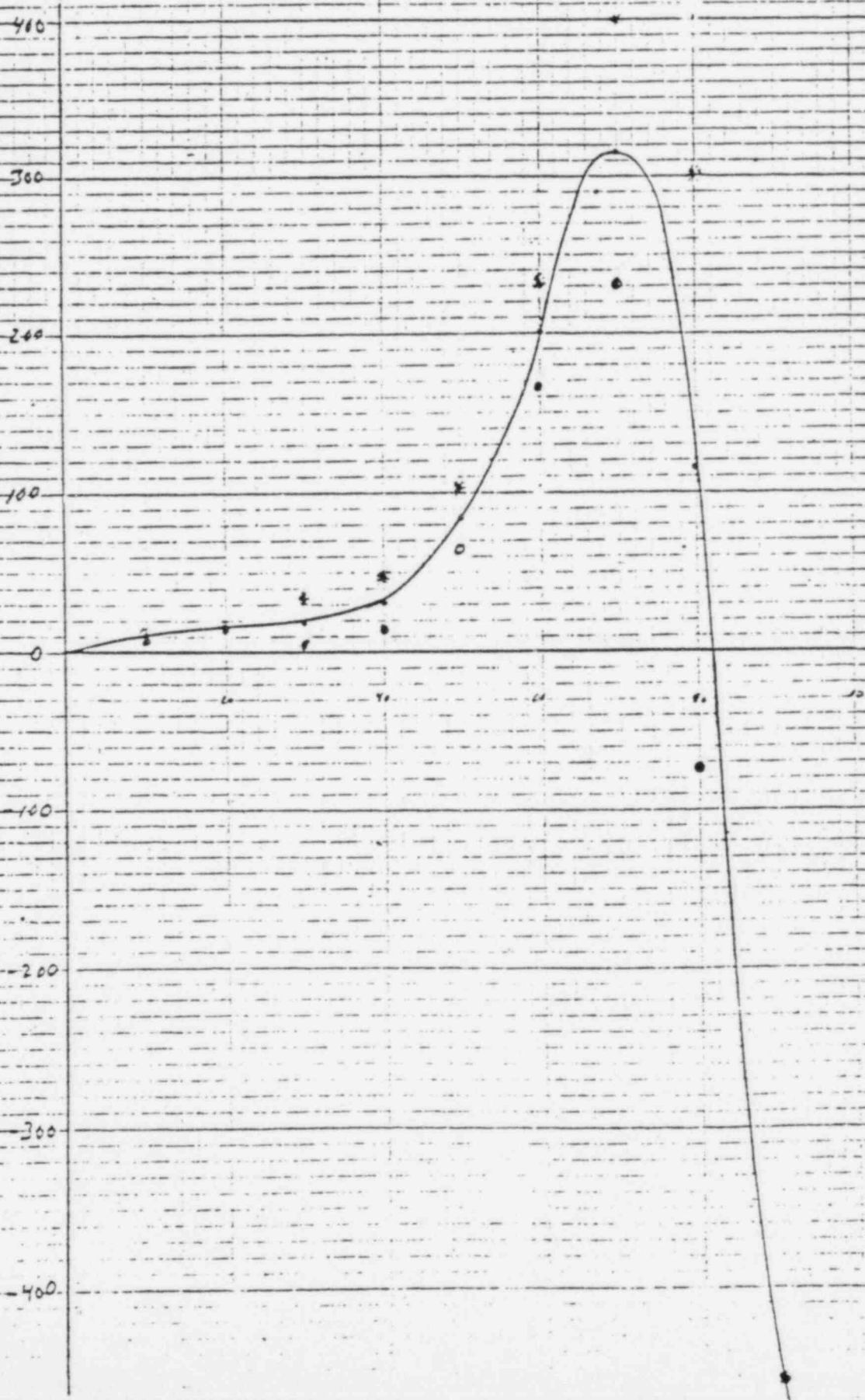
POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 14" 150 STEAM DOWN STREAM

Page       

By C. LIVORSI

Checked By       





POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title TORQUE VS. VALVE ANGLE

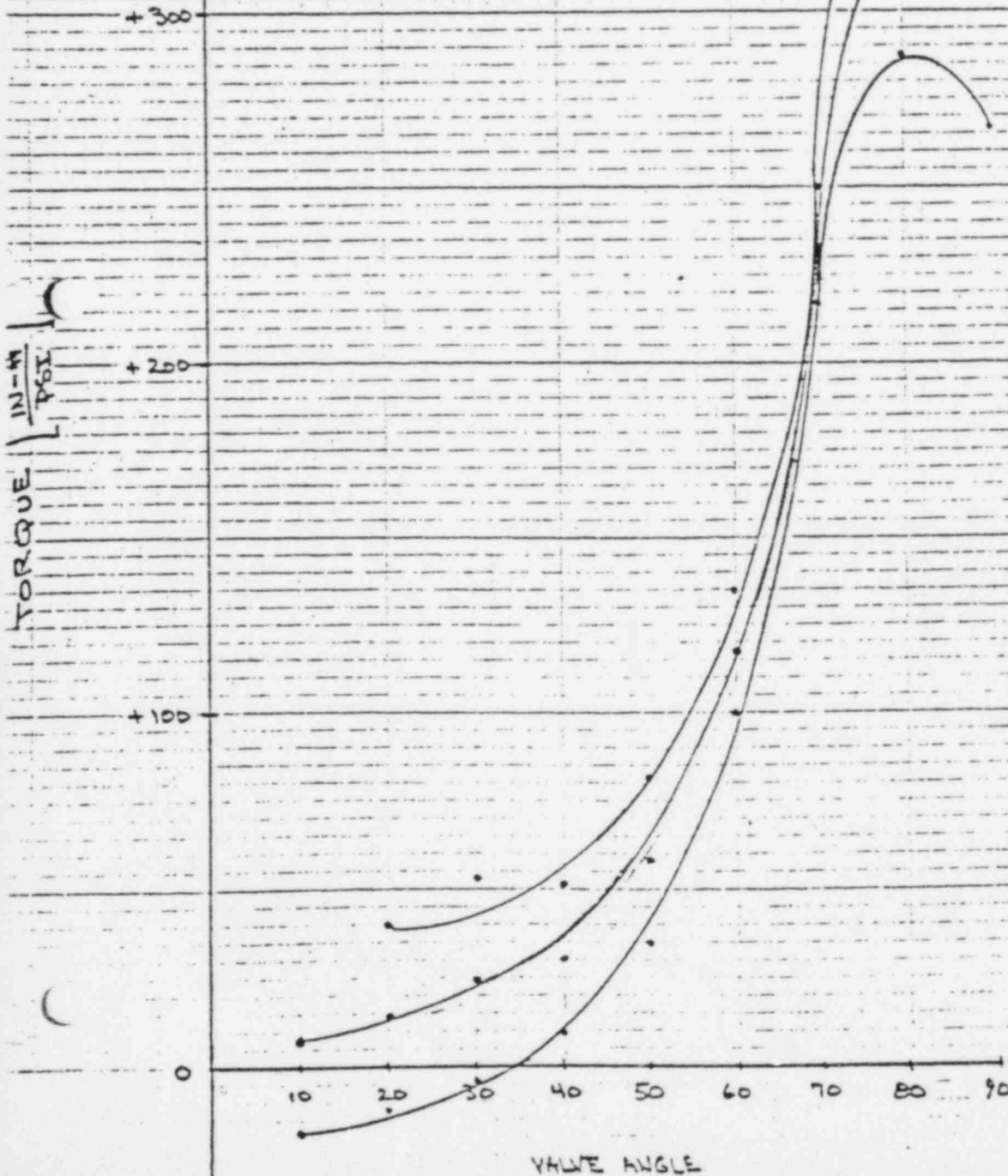
Page \_\_\_\_\_

By R. DANSEN

Checked By \_\_\_\_\_

10°-160°

STEM UPSTREAM  
CONTROL VALVE REGULATED.



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title VALVE TORQUE = 10" - 150

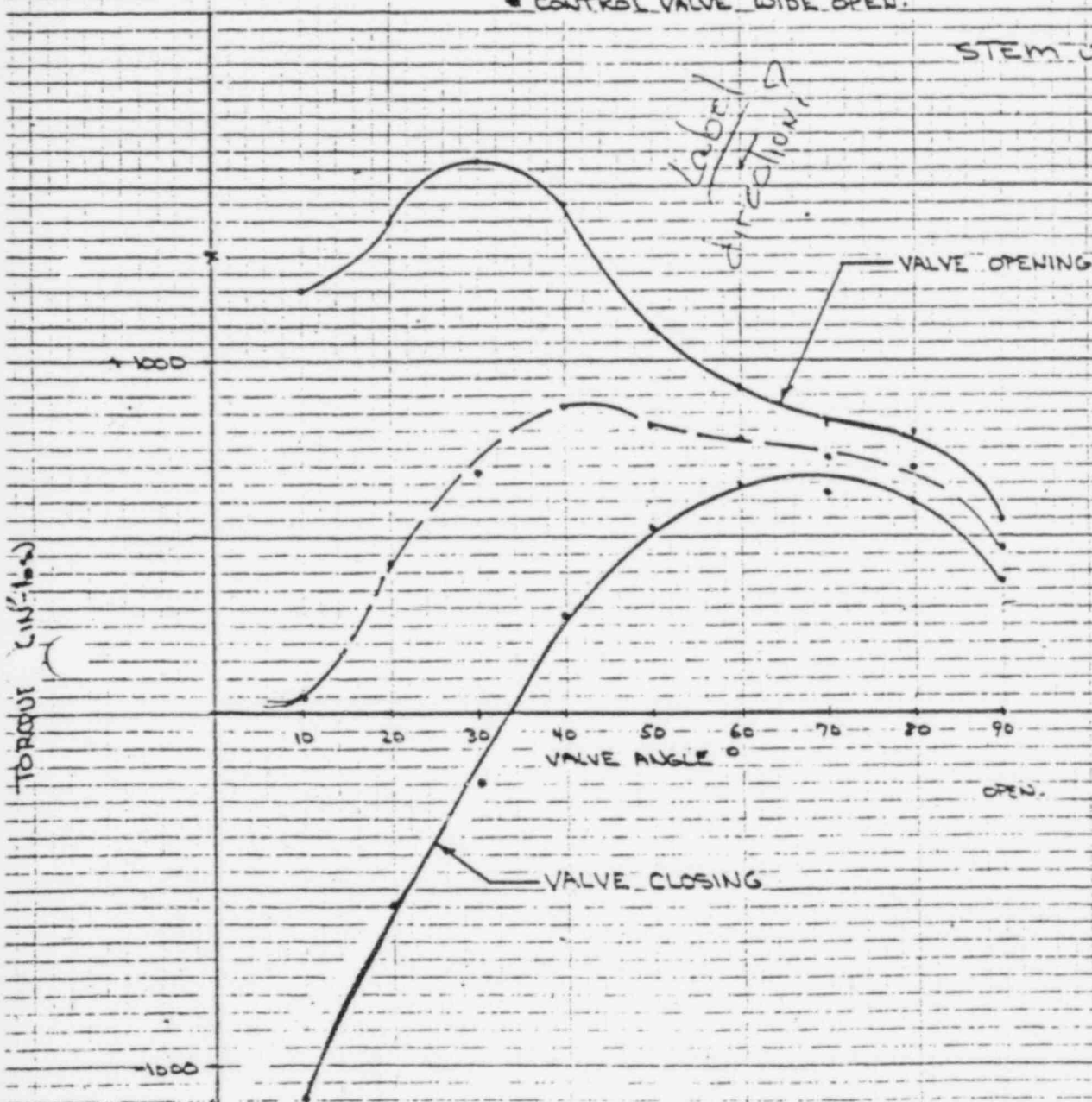
Page

By

Checked By

\* CONTROL VALVE WIDE OPEN.

STEM UPSTREAM.



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION

Page       

( 3y C. LIVORSI

DATE: 2/78

Checked By       

VALVE SIZE: 8" 300 LB

VALVE DIRECTION: ☒ STEM UPSTREAM / ☐ STEM DOWNSTREAM

VALVE SEAL TYPE: STO TEFELON

WATER TEMPERATURE (TANK): 64 °F

DISC TYPE & DWA:       

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	62	650		
10	58.5	350	6.0	
20	51.5	450	8.7	
30	45	600	13.3	
40	32.5	700	21.5	
50	20	700	35	
60	18	650	36.1	
70	9.0	750	83.3	
80	6.5	750	115.4	
90	5.5	700	127.3	
90	5.5	350	63.6	
80	6.0	550	91.7	
70	8.5	600	70.6	
60	13.0	550	42.3	
50	19.5	500	25.6	
40	30.5	450	14.7	
30	43	250	5.8	
20	52	0	0	
10	59	150	2.5	
0	62	550		

PACKING TORQUE: 200 IN-LB OPENING

: 200 IN-LB CLOSING



POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION

Page \_\_\_\_\_

By C. LIVORSI

DATE: 2/78

Checked By \_\_\_\_\_

VALVE SIZE: 8" 300 LB

VALVE DIRECTION: ☐ STEM UPSTREAM / ☒ STEM DOWNSTREAM

VALVE SEAL TYPE: STD. TEFLON

WATER TEMPERATURE (TANK): 64 °F

DISC TYPE & DWG:

OPENING

CLOSING

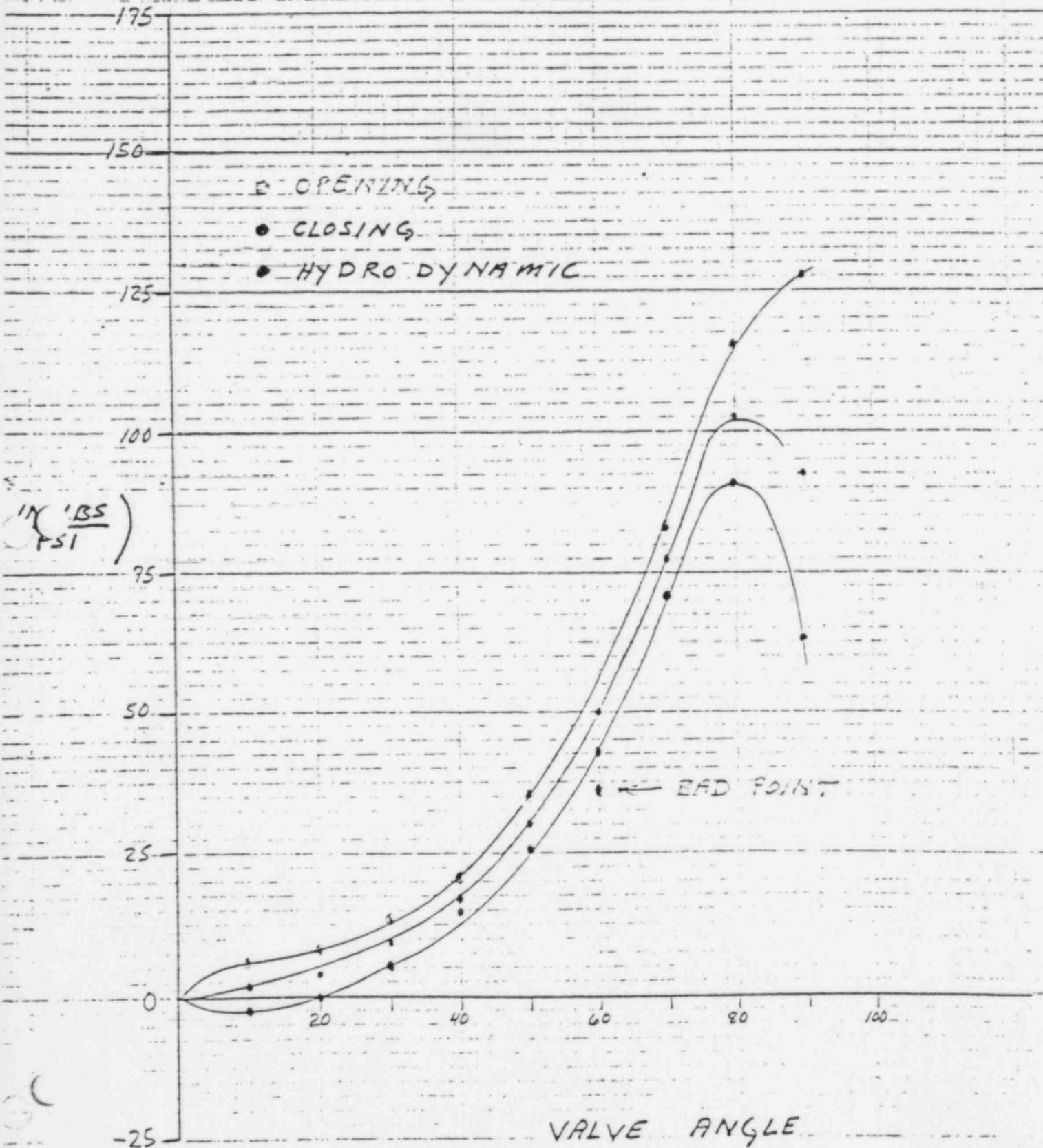
VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	6.3	600		
10	60.5	450	7.4	
20	51.5	550	10.7	
30	36	450	12.5	
40	22.5	350	15.5	
50	10.5	350	33.3	
60	7.0	300	42.8	
70	7.0	250	50	
80	5.0	0	0	0-TORQUE @ 78°
90	5.0	500	100	
90	4.8	550	114.6	
80	4.0	250	62.5	
70	6.0	50	8.3	
60	8.5	0	0	0-TORQUE @ 65°
50	17.5	250	14.3	
40	30.5	250	8.2	
30	44	0	0	
20	50.5	0	0	
10	60.5	250	4.1	
0	65	600		

PACKING TORQUE: 100 IN-LB OPENING

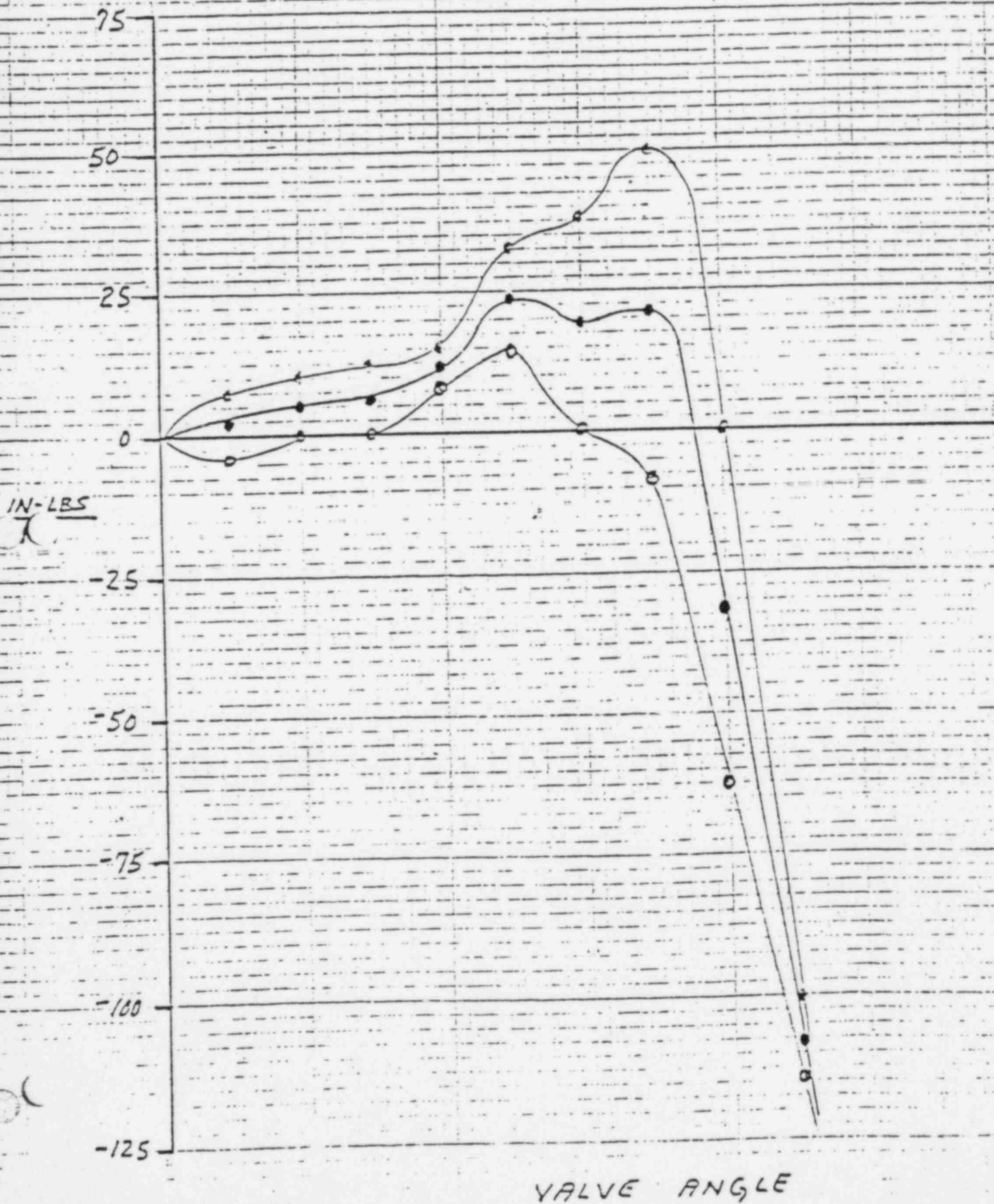
: 200 IN-LB CLOSING

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 8" 300 LB HYDRO DYNAMIC TORQUE CURVE Page         
By C. LIVORSI Checked By       



Title 8-300 HYDRODYNAMIC TORQUE CURVE Page         
By C. LIVORSI (UNPREFERRED) Checked By       










Title HYDRODYNAMIC TORQUE DETERMINATION Page       Calc. By ØANSEN DATE: 8/1/77 Checked By       VALVE SIZE: 10" 300VALVE DIRECTION: ~~STEM UPSTREAM~~ / STEM DOWNSTREAMVALVE SEAL TYPE: TEFLON/BUNAWATER TEMPERATURE (TANK): 83 °FDISC TYPE & DWG: 2 

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
0				
10		58.5		+700
20		47.5		+450
30		37.5		+300
40		25		+600
50		14		725
60		7.5		575
70		4.5		200
80		2.5		-250
90		3		-500
90		3		-825
80		3		-575
70		4.5		-275
60		7		0
50		12		+175
40		23		50
30		37.5		-300
20		51		-300
10		57.5		-600
0				

PACKING TORQUE: 175 IN-LB OPENING  
250 IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATION

Page

Calc. By CRANSENDATE: 7/20/77

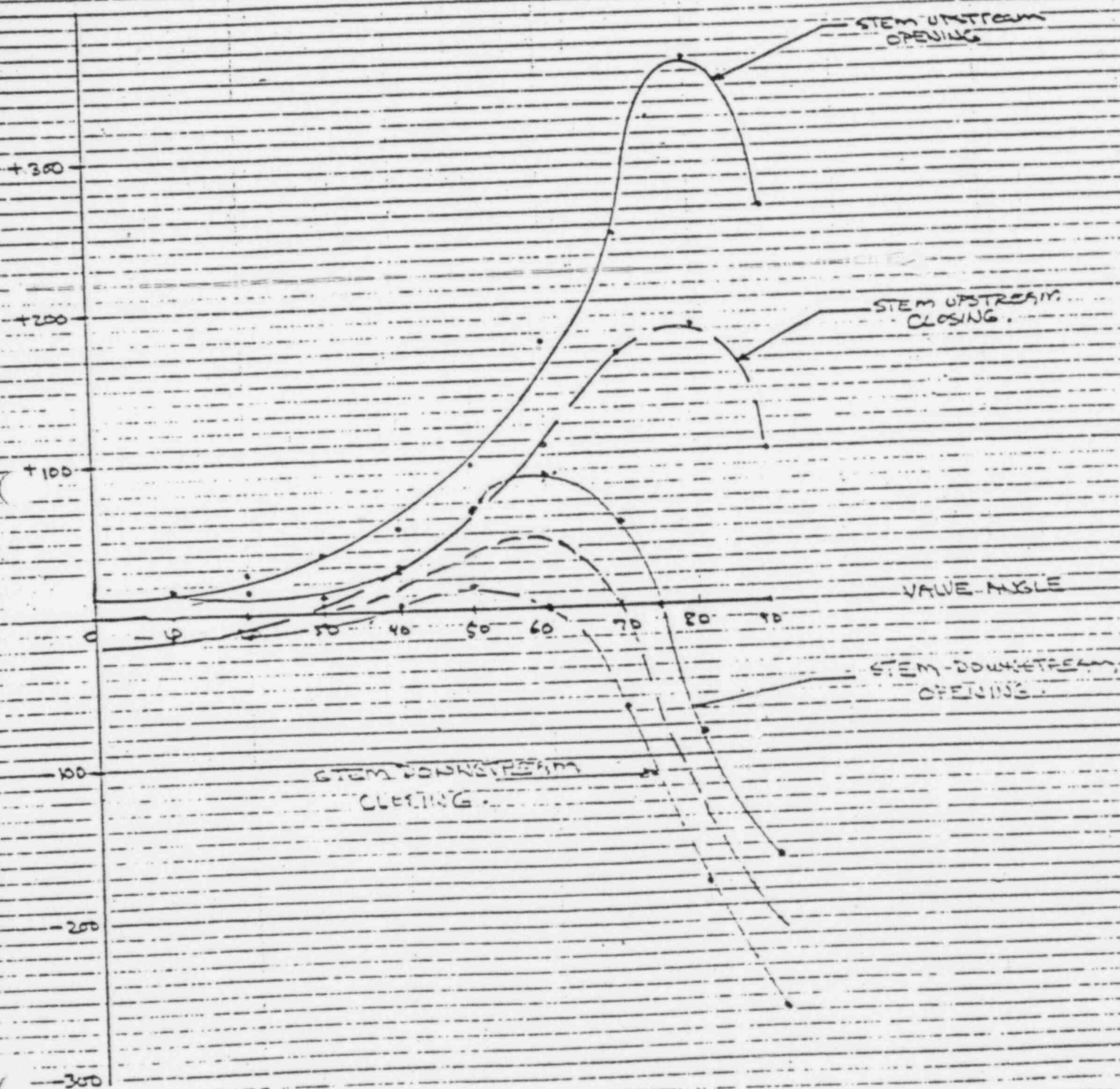
Checked By

VALVE SIZE: 10" 300VALVE DIRECTION: STEM UPSTREAM / STEADYVALVE SEAL TYPE: TEFLON / BUNAWATER TEMPERATURE (TANK): 87 °FDISC TYPE & DWA: 2

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
OPENING	0		<del>5</del>		<del>600</del>
	10		56		600
	20		26		700
	30		35		7350
	40		22.5		1200
	50		11		1050
	60		6		1050
	70		4.5		1100
	80		2.5		900
	90		2.5		650
CLOSING	90		2.5		250
	80		3		550
	70		4.5		750
	60		7.5		800
	50		12		750
	40		26		650
	30		12.5		75
	20		26		0
	10		55-20		-450 -300
	0		<del>5</del>		<del>600</del>

PACKING TORQUE: 200 IN-LB OPENING: 250 IN-LB CLOSING

# TORQUE VS. VALVE ANGLE





Title HYDRODYNAMIC TORQUE DETERMINATION

Page       

lc. By CHANDEN

DATE: 3/21/77 Checked By       

VALVE SIZE: 3" 600

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: Teflon/Buna

WATER TEMPERATURE (TANK): 67 °F

DISC TYPE & DWA:

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
0				
10		42		+60
20		38.5		+30
30		36.5		+30
40		34		+25
50		30.5		+40
60		39		+30
70		31.5		0
80		25.5		-5
90		22.5		-40
90		22.5		-60
80		25		-45
70		31.5		-25
60		39		-5
50		28		0
40		30.5		0
30		41.5		-5
20		48.5		-125
10		47.5		
0				

PACKING TORQUE: 10 IN-LB OPENING

: 5 IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATION

Page     

Calc. By Pham

DATE: 3/21/77

Checked By:     

VALVE SIZE: 3"-600

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TETAN / BUNA

WATER TEMPERATURE (TANK): 68 °F

DISC TYPE & DWG:      

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
0				
10		54		+75
20		52		+60
30		43.5		+75
40		40.5		+75
50		43.5		+115
60		37.5		+120
70		31.5		+145
80		26		+160
90		23		+145
90		23		+140
80		26.5		+125
70		31		+115
60		36		+100
50		43.5		+75
40		38.5		+45
30		41.5		0
20		43		0
10		44		-55
0				

PACKING TORQUE: 10 IN-LB OPENING

5 IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATIONPage     Calc. By SanzenDATE: 11/14/77Checked By     VALVE SIZE: 4"-600VALVE DIRECTION: (STEM UPSTREAM) / STEM DOWNSTREAMVALVE SEAL TYPE: TEFLON / EPRWATER TEMPERATURE (TANK):      °FDISC TYPE & DWG:     

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
0		<del>    </del>		<del>    </del>
10		49		+275
20		35		+275
30		34		+250
40		23.5		+175
50		26.5		+175
60		26		+250
70		22		+275
80		17		+250
90		16.5		+150
90		<del>    </del>		<del>    </del>
80		16.5		+75
80		17		+150
70		21.5		+125
60		27.5		+50
50		35.5		-10
40		25		-50
30		39		-100
20		48.28		-250 -200
10		26.5		-200
0	✓	<del>    </del>	✓	<del>    </del>

PACKING TORQUE: 15 IN-LB OPENING5 IN-LB CLOSING



# ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE DETERMINATION

Page     

By DAWSEN

DATE: 11/3/77

Checked By     

VALVE SIZE: 4"-600

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON / EPR

WATER TEMPERATURE (TANK): 68 °F

DISC TYPE & DWG:      

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
OPENING	0		X		X
	10		43		+75
	20		40.5		+25
	30		37		+50
	40		41.5		+50
	50		39		+50
	60		31		+25
	70		23.5		-25
	80		19		-75
	90		17.5		-125
CLOSING	90		X		X
	80		17.5		-150
	70		18.5		-100
	60		23.5		-50
	50		30.5		-25
	40		39.5		-25
	30		47		0
	20		50		-25
	10		49.5		-75
	0		50		-150
	0		X		X

PACKING TORQUE: 250 IN-LB OPENING

: 250 IN-LB CLOSING

Title: HYDRODYNAMIC TORQUE DETERMINATION

Page

By: [Signature] DATE: 8/1/77 Checked By: \_\_\_\_\_

VALVE SIZE: 6"-600

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: FECTION / BUNA

WATER TEMPERATURE (TANK): 83 °F

DISC TYPE & DWG: \_\_\_\_\_

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
OPENING	0		X		X	
	10		44		+200	+4.5
	20		33		+150	+4.5
	30		29.5		+250	+8.7
	40		25		+400	+16
	50		19.5		+450	+23.0
	60		20.5		+450 +24.95	
	70		15		+250 +16.67	0.0
	80		10.5 / 8.5		0 -25.0	-13
	90		15.5		-650	-11.5
CLOSING	90		X		X	
	90		15.5		-800	-51
	80		19.0		-525	-27
	70		18.5		-200	-10.7
	60		26		+50	+1.95
	50		28		+50	+1.73
	40		33		0	0
	30		34		-275	-8.05
	20		32.5		-350	-10.5
	10		39		-350	-11.9
	0	41	X	41	X	

PACKING TORQUE: 175 IN-LB OPENING

225 IN-LB CLOSING



Title HYDRODYNAMIC TORQUE DETERMINATION

Page 1

By Janice DATE: 8/19/77 Checked By

VALVE SIZE: 6" 600

VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSTREAM

VALVE SEAL TYPE: TEFLON / BUNA

WATER TEMPERATURE (TANK): 85 °F

DISC TYPE & DWA: WIDE OPEN ROQ WIDE OPEN

OPENING

CLOSING

VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB	
0		X	X	X	X
10		30.5	59.5	+250 6.6	+325
20		33	58.5	+300 9.1	+50
30		24	55.5	+300 12.5	+45
40		25	49.5	+300 12	+55
50		26.5	44.5	+400 15.1	+625 + 2
60		20	32.5	+450 22.5	+700 + 5
70		19.5	24.5	+650 33.3	+8
80		19.2	19.0	+1025 53.4	+9
90		14.8	14.8	+750 (50.7)	+5
90		X	X	X	X
90		14.8	14.8	+650 (43.9)	+5
80		19.3	18.8	+750 38.4	+6
70		24.2	24.4	+825 34.1	+7
60		20.5	31.5	+225 11	+5
50		24.5	43.5	+100 4.1	+2
40		34.5	50.5	+25 1.7	+1
30		34.5	55.5	-50 1.4	+1
20		38.5	58	-75 1.9	-
10		34.0	60.5	-275 8.1	-2
0		X	X	X	X

PACKING TORQUE: 175 IN-LB OPENING

: 25 IN-LB CLOSING

# HYDRODYNAMIC TORQUE V.S. ANGLE

Title

Page

BY

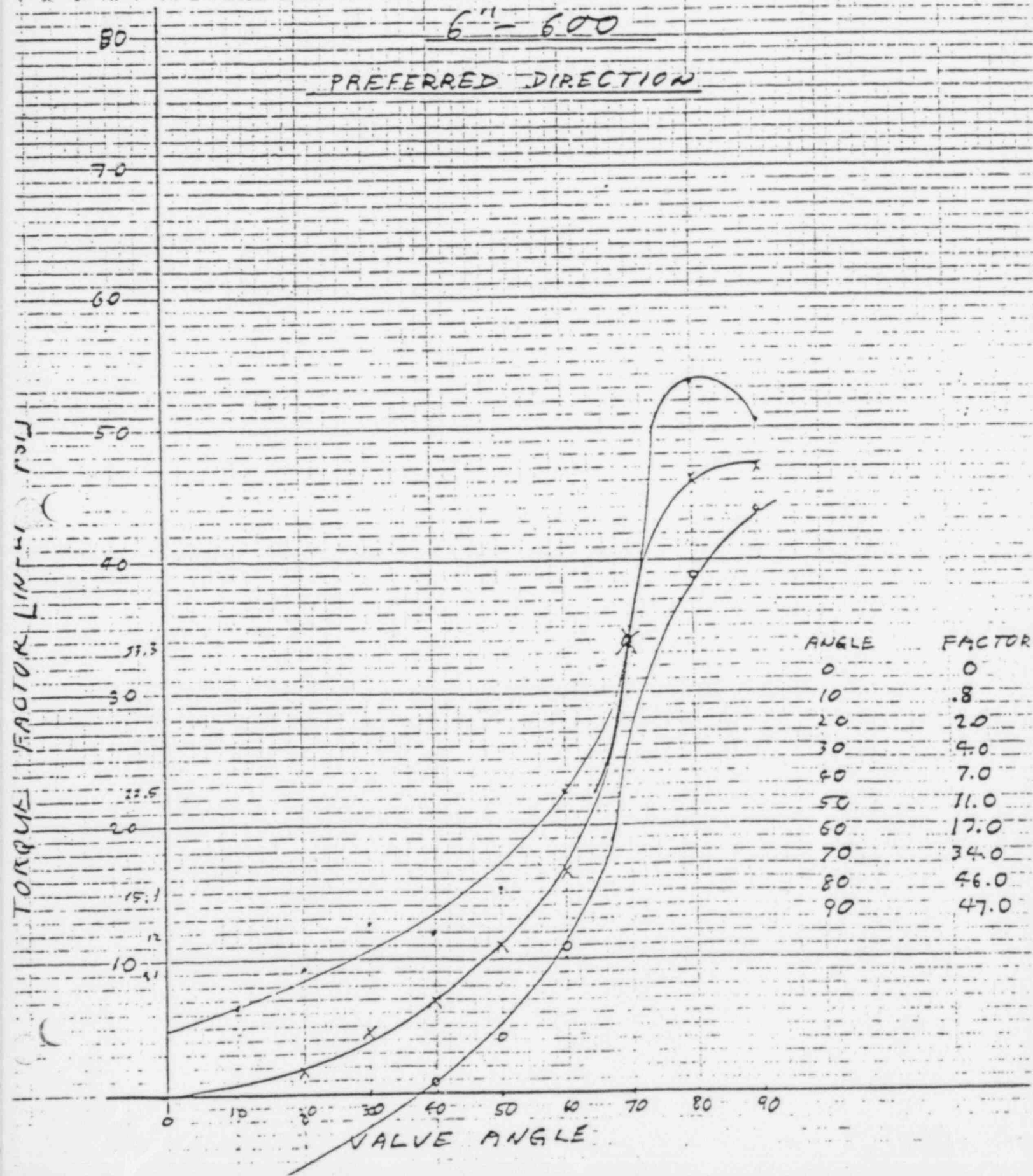
J. CORY

8/23/77

Checked By

6" - 600

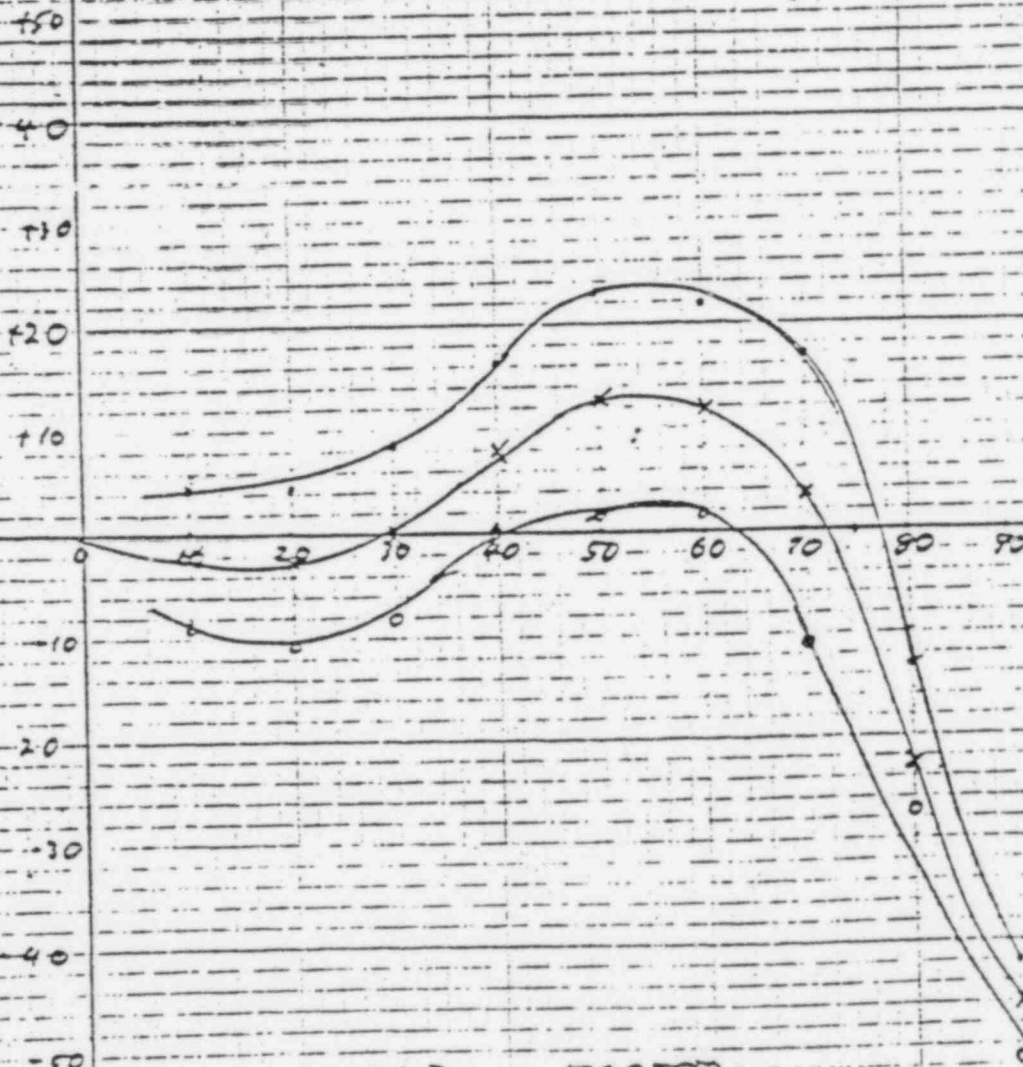
PREFERRED DIRECTION



Title HYDRODYNAMIC TORQUE V.S. ANGLE Page         
( 3y J CORY 8/23/72 Checked By       

6" - 600

NON PREFERRED DIRECTION



ANGLE	FACTOR
0	0
10	-3
20	-3
30	0
40	7
50	12
60	12
70	4
80	-22
90	-25



Title HYDRODYNAMIC TORQUE DETERMINATIONPage 1Calc. By C. LivorsiDATE:           Checked By           VALVE SIZE: 8" 600#VALVE DIRECTION: ☐ STEM UPSTREAM ☒ STEM DOWNSTVALVE SEAL TYPE: URATHANEWATER TEMPERATURE (TANK): 69 °FDISC TYPE & DWG:           

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	0	1600		
10	5.0	500	-10	
20	11	250	-22.7	
30	11.5	300	-26.1	
40	11.0	350	-31.8	
50	9.5	325	-34.2	
60	5.7	200	-35.4	
70	6.5	125	19.2	72° TORQUE WENT
80	5.0	125	-25	NEG
90	6.0	300	-50	
90	6.0	600	-100	
80	5.0	400	-80	
70	7.0	250	-35.7	
60	11.2	100	-8.9	
50	18.5	0	0	
40	23.5	700	4.3	
30	31.0	100	3.2	
20	39	200	5.1	
10	5.7	250	4.4	
0	0	1100		

PACKING TORQUE: 150 IN-LB OPENING: 150 IN-LB CLOSING



Title HYDRODYNAMIC TORQUE DETERMINATIONCalc. By C. LIVORSI

DATE:

Checked By

VALVE SIZE: 8" 600#VALVE DIRECTION: ☐ STEM UPSTREAM / ☒ STEM DOWNSVALVE SEAL TYPE: URATHANEWATER TEMPERATURE (TANK): 69 °F

DISC TYPE &amp; DWG:

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	0	1600		
10	5.0	500	-10	
20	11	250	-22.7	
30	11.5	300	-26.1	
40	11.0	350	-31.8	
50	9.5	325	-34.2	
60	5.7	200	-35.4	
70	6.5	-125	19.2	72° TORQUE WENT
80	5.0	-125	-25	NEG
90	6.0	-300	-50	
90	6.0	-600	-100	
80	5.0	-400	-80	
70	7.0	-250	-35.7	
60	11.2	-100	-8.9	
50	18.5	0	0	
40	23.5	700	4.3	
30	31.0	100	3.2	
20	39	-200	5.1	
10	5.7	-250	-4.4	
0	0	-1100		

PACKING TORQUE: 150 IN-LB OPENING: 150 IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATIONPage     Calc. By C. LIVOCCIDATE:     Checked By     VALVE SIZE: 8" 600VALVE DIRECTION: ☒ STEM UPSTREAM ☐ STEM DOWNSVALVE SEAL TYPE: URATHANEWATER TEMPERATURE (TANK): 71 °FDISC TYPE & DWG:     

OPENING

CLOSING

VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0	67	7800		
10	50.5	400	7.9	
20	16	300	19.8	
30	5	250	50	
40	16	400	40	
50	9	600	66.7	
60	6.0	500	83.7	
70	4.2	600	142.6	
80	3.0	400	133.3	
90	4.6	300	65.2	
90	5.0	0	0	
80	4.0	200	50	
70	5.5	325	59.1	
60	8.0	225	40.6	
50	11.5	300	26.1	
40	9.0	100	14.1	
30	19.0	115	6.1	
20	19.0	0	0	
10	49.0	350	7.1	
0	67	1800		

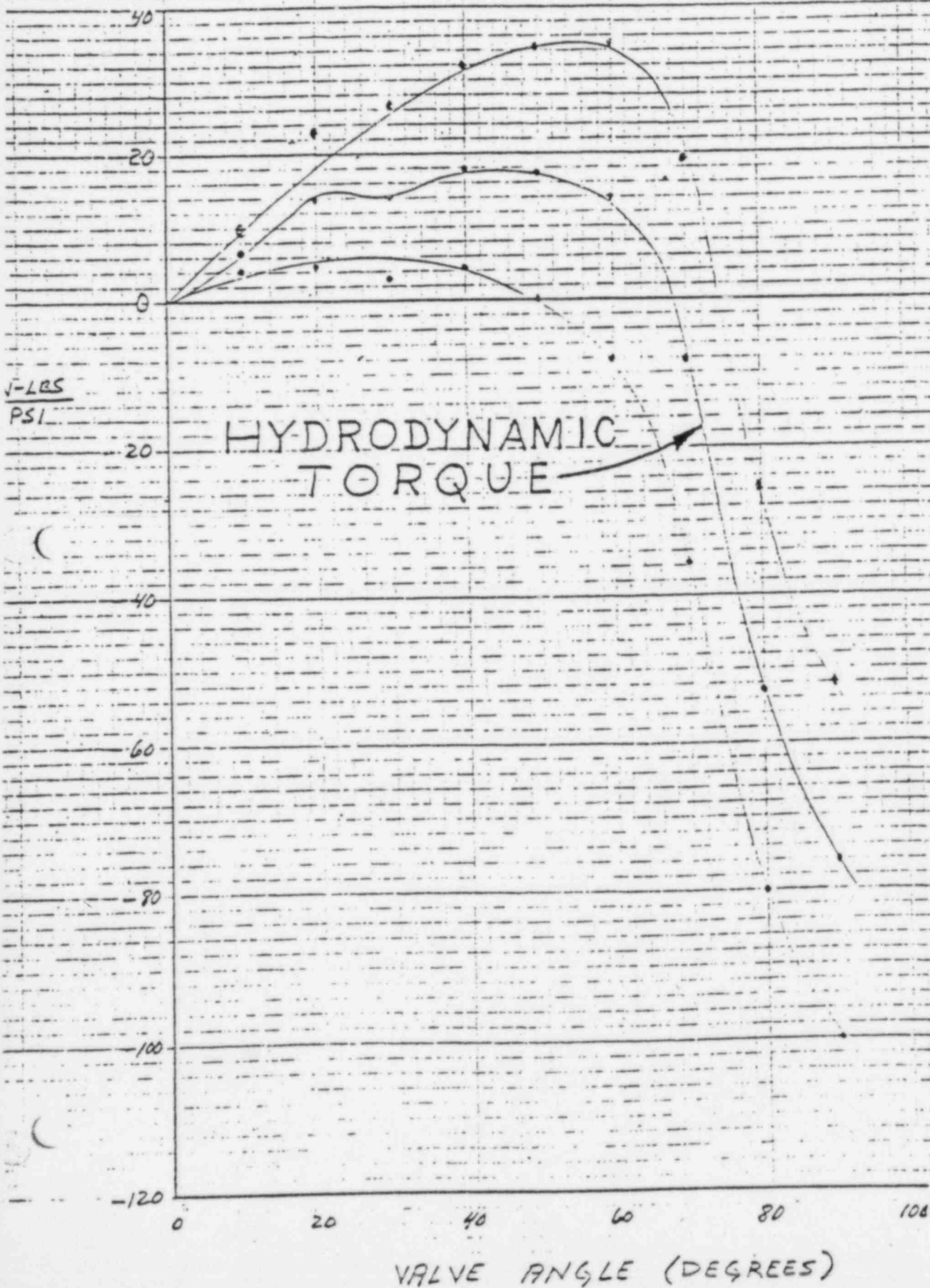
PACKING TORQUE: 150 IN-LB OPENING: 100 IN-LB CLOSING

8"-600LB

Title HYDRODYNAMIC TORQUE CURVE (NON-PREF. DIRECTION) Page

By C. LIVORSI

Checked By





8"-600LB

## ENGINEERING CALCULATIONS

Title HYDRODYNAMIC TORQUE CURVE (PERF. DIRECTION) Page 1Calc By C. LivorsiChecked By 

140

120

100

80

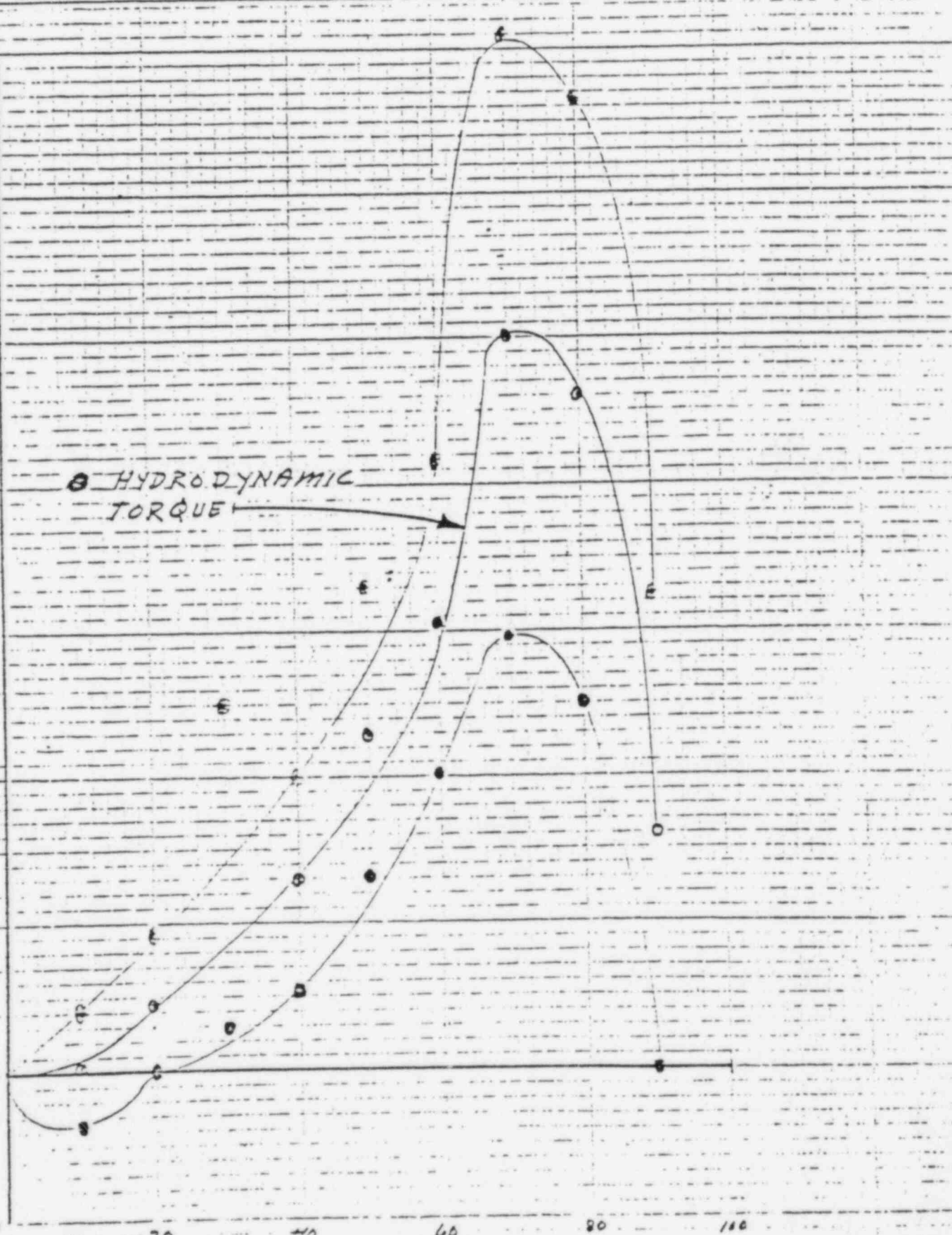
60


40

20

0

-20

HYDRODYNAMIC  
TORQUEVALVE ANGLE  
(DEGREES)

Title HYDRODYNAMIC TORQUE DETERMINATION Page       Calc. By KAUSEN DATE: 8/3/77 Checked By       Control valve - regulatedVALVE SIZE: 10" - 600VALVE DIRECTION: STEM UPSTREAM / STEM DOWNVALVE SEAL TYPE: TEFLON / BUNAWATER TEMPERATURE (TANK): 84 °FDISC TYPE & DWG: 2 

	VALVE ANGLE DEGREES	P, UPSTREAM PSIG	$\Delta P$ PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN - LB
OPENING	0		<del>2</del>		
	10		47.385		+900 ✓
	20		38.25.9		+1200 ✓
	30		28.3-25		+1550 - 1500
	40		19.7-24		+1500 - 1750
	50		14.9+14.4		+1500 - 1550
	60		10.5-9.8		+1450 - 1350
	70		7.3		+1400
	80		5.2		+1200
	90		4.4		+800
	95-85		4.4		0
CLOSING	80		4.4		+200
	70		5.9		+450
	60		8.2		+550
	50		13.0		+600
	40		20.8		+650
	30		33		+450
	20		40.4		+50
	10		39		-550
	0		<del>X</del>	<del>0</del>	

PACKING TORQUE: 350 IN-LB OPENING400 IN-LB CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATIONPage     Calc. By ChenDATE: 8/5/77Checked By     Control valve regulatedVALVE SIZE: 10" 600VALVE DIRECTION: STEAM UPSTREAM / STEM DOWNSVALVE SEAL TYPE: TEFLON / BUNAWATER TEMPERATURE (TANK): 88° FDISC TYPE & DWG: 2 

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
OPENING	0		X		X
	10		46		+100
	20		31.5		+900
	30		27.5		+900
	40		24.5		+1500
	50		14.0		+1100
	60		9.1		+950 +800
	70		5.8		+600 +400
	80		4.3 4.2		-321 +100 -100
	90		4.1		-250
CLOSING	90		X		X
	90		4.7		-950
	80		4.2		-750
	70		6.4		-550
	60		9.6		-450
	50		14.4		-300
	40		26.5		-250
	30		27.5		-400
	20		32		-750
	10		37		-800
	0		X		X

PACKING TORQUE:

590 IN-LB OPENING400 IN-LB CLOSING



Title HYDRODYNAMIC TORQUE DETERMINATION

Page

Calc. By DanDATE: 8/5/77

Checked By

Control valve wide open

VALVE SIZE: 10" - 600VALVE DIRECTION: STEM UPSTREAM / STEM DOWNSVALVE SEAL TYPE: TEFLON / BUNAWATER TEMPERATURE (TANK): 88 °FDISC TYPE & DWG: 2

	VALVE ANGLE DEGREES	P <sub>1</sub> UPSTREAM PSIG	ΔP PSI	Q GPM	TORQUE + TENDING TO CLOSE - TENDING TO OPEN IN-LB
	0	1	X		X
	10		57		+60-1500
	20		48.5		+950
OPENING	30		37		+825
	40		25.5		+850
	50		14.5		+900
	60		9.4		+900
	70		6.5		+250
	80		4.3		0-25
	90		4.6		-300
			X		X
	90		4.7		-1350
	80		4.2		-800
	70		6.4		-550
	60		9.3		-500
CLOSING	50		14.0		-325
	40		24		-325
	30		37		-575
	20		48		-950
	10		57.0		-1025
	0	1	X	0	X

PACKING TORQUE: 350-450 IN-LB OPENING400-450 IN-LB CLOSING

# HYDRODYNAMIC TORQUE DETERMINATION

Calc. By C. Livorsi DATE: 9-1-78 Checked By \_\_\_\_\_

VALVE SIZE: 14" 1500 LB

VALVE DIRECTION: ☒ STEM UPSTREAM / ☐ STEM DOWNSTREAM

VALVE SEAL TYPE: METAL

WATER TEMPERATURE (TANK): 77 °F

DISC TYPE & DWG: \_\_\_\_\_

VALVE ANGLE DEGREES	ΔP (PSIG)	TORQUE (IN-LBS) (44-185)	PSI	COMMENTS
0	59.5	190	3.2	
10	54.5	195	3.6	
20	40.5	205	5.1	
30	24.5	205	8.4	
40	17.5	190	10.9	
50	12.0	175	14.6	
60	9.5	165	17.4	
70	8.0	160	20.0	
80	7.5	145	19.3	
90	8.0	-35	-4.4	
90	7.5	-20	-2.7	
70	8.0	-5	-0.6	
60	8.5	-5	-0.6	
50	11.5	-5	-0.4	
40	17.5	-10	-0.6	
30	31.0	-50	-1.6	
20	44.5	-105	-2.4	
10	52.5	-210	-4.0	
0				

CLOSING

OPENING

PACKING TORQUE: 70 ft-LBS IN-LBS  
70 ft-LBS IN-LBS CLOSING

Title HYDRODYNAMIC TORQUE DETERMINATIONPage     Calc. By C. LIVORSIDATE: 9-1-78Checked By     VALVE SIZE: 14" 1500 LBVALVE DIRECTION: STEM UPSTREAM / STEM DOWNS ☐ ☒VALVE SEAL TYPE: METALWATER TEMPERATURE (TANK): 77 °FDISC TYPE & DWG:      

OPENING

CLOSING

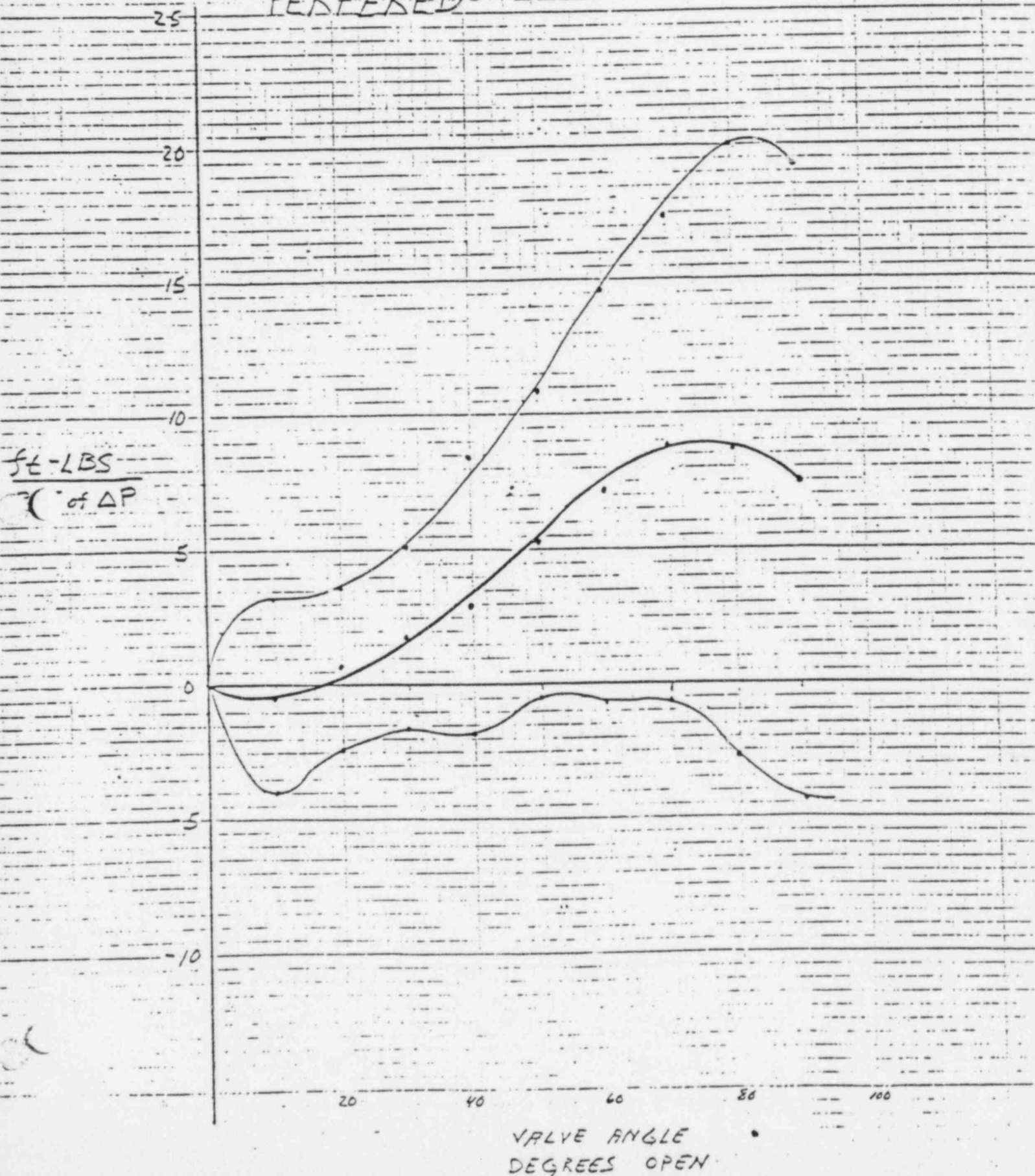
VALVE ANGLE DEGREES	$\Delta P$ (PSIG)	TORQUE (IN-LBS)	IN-LBS PSI	COMMENTS + TENDING TO CLOSE - TENDING TO OPEN
0				
10	57.5	150	2.6	
20	48.5	190	3.9	
30	39.0	200	5.1	
40	28.5	170	6.7	
50	16.5	130	7.9	
60	12.5	90	7.2	
70	9.0	50	5.5	
80	7.5	25	3.3	
90	6.8	15	2.2	
90	7.0	-115	-16.4	
80	7.0	-115	-16.4	
70	7.2	-95	-13.2	
60	8.0	-90	-11.3	
50	11.5	-70	-6.1	
40	19.5	-65	-3.3	
30	32.0	-70	-2.2	
20	45.0	-100	-2.2	
10	54.0	-125	-2.3	
0				

PACKING TORQUE: 70 ft-LBS IN-LBS OPENING: 70 ft-LBS IN-LBS CLOSING



14" 1500 LB HYDRODYNAMIC TORQUE CURVE Page  
 C. LIVORSI Checked By

PERFERED



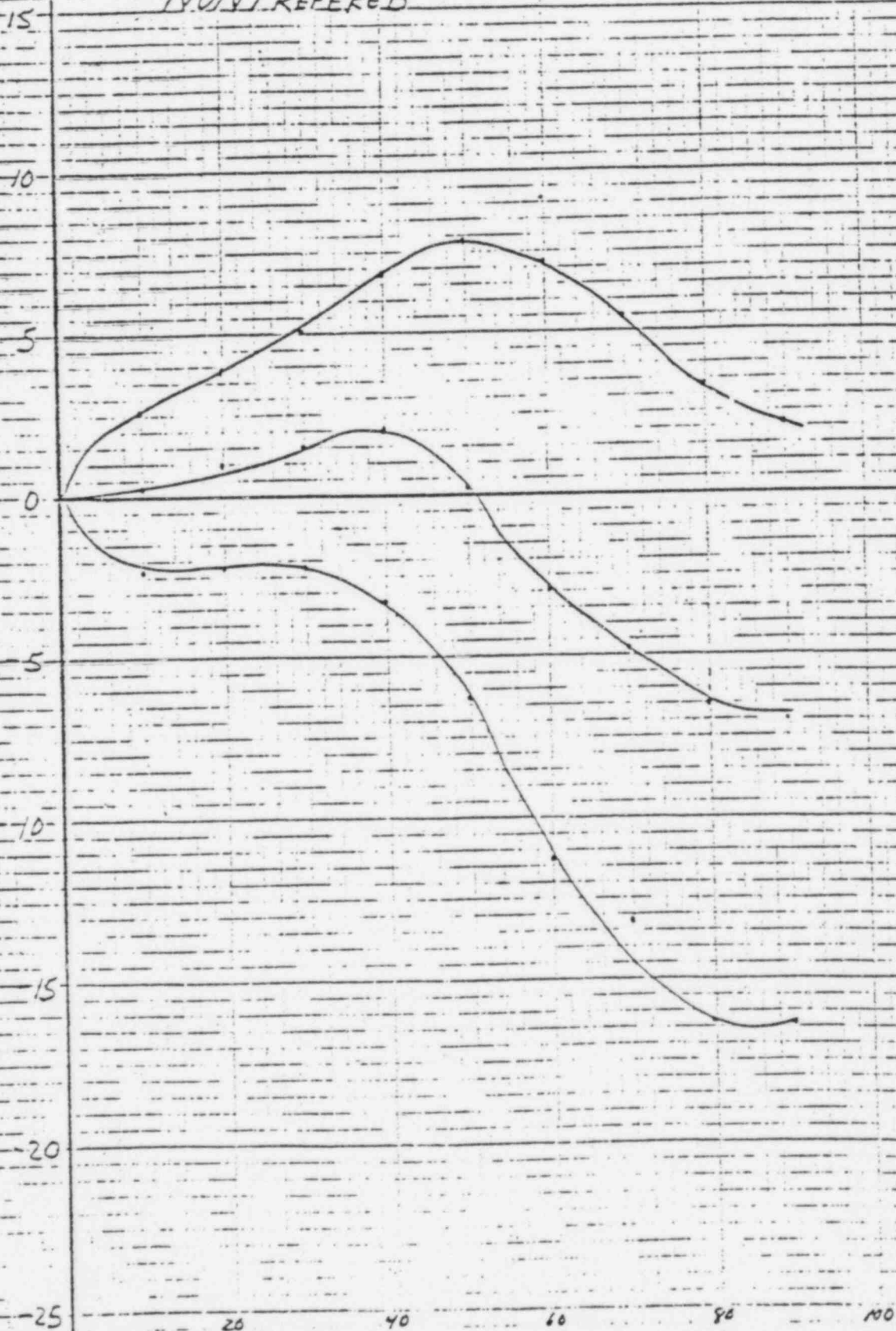
le 14" 1500 LB HYDRODYNAMIC TORQUE CURVE

Page

C. LIVORSI

Checked By

NON PREFERED

VALVE ANGLE  
DEGREES OPEN

**TORQUE VALUES TO OPEN AND SHUT**

The Posi-Seal Trunnion Valve is a low torque valve which normally reaches its highest torque when opening. This torque value will vary with the seat material and stem packing selected in addition to the maximum operating differential pressure ( $\Delta P$ ) across the valve.

Torque values to open and shut for standard Posi-Seal Trunnion Valves are listed in Tables 1, 2 and 3, and are denoted as  $T_1$ ,  $T_2$ , and  $T_3$ .

$T_1$  and  $T_2$  torque values are due to the stem packing and seat material selected and are added together to determine preload torque.

The static pressure torque factor  $T_3$  is the static pressure torque per  $\text{PSI}\Delta P$ . This factor ( $T_3$ ) is multiplied by the maximum operating  $\Delta P$  or maximum line pressure to obtain the torque value due to pressure.

To obtain the maximum torque to open or shut the  $T_1$ ,  $T_2$ , and  $T_3 \Delta P$  values are simply added together.

**EXAMPLE:**

A. Requirement — What is the maximum opening torque of a 12" Class 150 valve with teflon chevron packing, a teflon seal ring with rubber back-up ring and a maximum  $\Delta P$  of 200 PSI.

B. Solution — From Table 1

$$T_1 = (B) = 70 \text{ in. lbs.}$$

$$T_2 = (G) = 986 \text{ in. lbs.}$$

$$T_3 = 10.10 \text{ in. lbs. per PSI}\Delta P = 10.10 \times 200 = 2020 \text{ in. lbs.}$$

$$\text{Total opening torque} = T_1 + T_2 + (T_3 \times \text{PSI}\Delta P) = 70 + 986 + (10.10 \times 200) = 3076 \text{ in. lbs.}$$

Although the valve opening torque is normally the highest operating torque used in actuator sizing, it is often necessary with flowing liquids to check for total hydrodynamic torque. Refer to Section II for data on calculating total hydrodynamic torque.



TABLE 2  
CLASS 300 STD. RATING

VALVE SIZE	PRELOAD TORQUE = $T_1 + T_2$								STATIC PRESSURE TORQUE PER PSI $T_3$	
	$T_1$			$T_2$					316 M.S.	REXNORD
	A	B	C	E	F	G	H	J		
3"	302	28	227	122	73	61	244	280	0.24	.16
4"	338	31	254	218	131	109	436	501	0.50	.33
6"	454	42	340	510	306	255	1020	1173	1.56	1.04
8"	756	70	567	750	450	375	1500	1725	3.84	2.56
10"	907	84	680	1242	745	621	2484	2857	7.64	5.09
12"	1058	98	794	1722	1063	886	3544	4076	12.70	8.47
14"	1210	112	907	2160	1296	1080	4321	4968	17.71	11.81
16"	1361	126	1021	3164	1898	1582	6328	7277	29.16	19.44
18"	1512	140	1134	3602	2161	1801	7204	8285	36.90	24.60
20"	1814	168	1361	4534	2720	2267	9068	10428	55.74	37.16
24"	2117	196	1588	7564	4538	3782	15128	17397	108.46	72.31
30"	2722	252	2041	10764	6458	5382	21528	24757	198.45	132.30
36"	3024	280	2268	16278	9767	8139	32556	37439	333.46	222.31
42"	3175	294	2381	23402	14041	11701	46804	53825	503.30	.....
48"	4234	392	3175	30383	18230	15192	60766	69881	871.40	.....

Valve torque (opening) =  $T_1 + T_2 + (T_3 \times \text{PSI} \Delta P)$

Valve torque (closing, on-off service) =  $T_1 + T_2 + (.5T_3 \times \text{PSI} \Delta P)$

Valve torque (closing, modulating service) =  $T_1 + T_2 + (T_3 \times \text{PSI} \Delta P)$

When the operating  $\Delta P$  used for actuator selection is less than the maximum line pressure, contact the factory for sizing torque.

#### NOTES:

$T_1$  (A) Asbestos jam packing  
(B) Teflon Chevron packing  
(C) Graphite jam packing

$T_2$  (E) Urethane seal ring with rubber back-up ring,  
Metal seal ring with and without rubber back-up ring, Kel-F seal ring without rubber back-up ring  
(F) Teflon seal ring with Teflon back-up ring,  
Tefzel seal ring with rubber back-up ring  
(G) Teflon seal ring with rubber back-up ring  
(H) Metal seal ring with Teflon insert and rubber back-up ring  
(J) Metal seal ring with urethane, Tefzel, or Kel-F insert and rubber back-up ring

## II. TOTAL VALVE OPERATING HYDRODYNAMIC TORQUE

As previously stated the valve opening torque is normally the highest operating torque used for actuator sizing and selection. However, when flowing liquids, it is often necessary to calculate the total valve operating hydrodynamic torque.

The location of the maximum valve operating torque (total hydrodynamic torque) is a result of the overall system operating parameters in addition to the particular valve disc hydrodynamic torque characteristics.

When the ratio of thru valve  $\Delta P$  to total system  $\Delta P$  is high, generally above 25 percent, the maximum valve operating hydrodynamic torque will occur at or about the 70° to 80° disc open position. As this ratio decreases, the maximum valve operating hydrodynamic torque will shift towards the 0° shut position.

To allow for system operating variables it is recommended that the maximum valve operating hydrodynamic torque ( $TH_T$ ) be calculated at both the 20° and 80° disc open position.

The total valve operating hydrodynamic torque ( $TH_T$ ) is the summation of three torque components. These components are: stem packing torque ( $T_1$ ), stem bearing friction torque (static pressure torque per  $PSI\Delta P$ ) and the disc hydrodynamic lift and drag torque. Torque values  $T_1$  and  $T_3$  can be found in Tables 1 thru 3.

Extensive flow testing has shown that the disc hydrodynamic lift and drag torque values are dependent upon the direction of flow entering the valve. With liquid flow entering the valve from the stem side, with the seal retaining downstream, the disc hydrodynamic lift and drag torque value ( $T_4$ ) is positive to the full open position acting to return the disc to the shut position. Liquid flow entering the valve from the opposite direction, seal retaining ring up stream, results in a torque value that remains positive, acting to return the disc to the shut position, until about the 70° to 80° open position. At this point the torque value becomes negative acting to move the disc to the full open position. With liquid flow in this direction the disc hydrodynamic lift and drag torque values are designated as  $T_5$ . Also, test results have shown that flowing liquids in this direction results in a positive  $T_5$  value that is lower than the corresponding  $T_4$  value with flow in opposite direction.

TABLE 4

## CLASS 150 STD RATING

Seal Retaining Ring Downstream  
Disc Hydrodynamic Lift & Drag Torque —  $T_d$

VALVE SIZE	$T_d$ VS. OPEN POSITION								
	10°	20°	30°	40°	50°	60°	70°	80°	90°
3"	0	0	0	0	1	2	5	6	5
4"	0	0	1	1	3	6	12	16	13
6"	1	2	4	8	13	27	51	68	56
8"	4	10	16	30	49	100	185	247	204
10"	6	16	26	49	79	161	300	399	330
12"	15	38	60	114	182	372	692	920	761
14"	23	59	94	177	283	579	1075	1430	1182
16"	41	102	164	307	492	1005	1876	2482	2052
18"	64	162	259	486	778	1590	2953	3927	3245
20"	99	248	397	746	1193	2437	4526	6019	4974
24"	214	536	858	1610	2576	5259	9768	12988	10734
30"	462	1156	1850	3469	5551	11334	21049	27988	23131
36"	1884	4710	7537	14132	22611	46164	85734	113998	94213
42"	2281	5704	9127	17114	27383	55908	103829	138058	114098
48"	3265	8164	13063	24494	39191	80016	148602	197592	163299
54"	7388	18471	29554	55414	88662	181020	336180	447008	369428
60"	11165	27913	44661	83740	133984	273552	508026	675506	558270
66"	14942	37355	59768	112066	179306	366084	679972	904005	747112
72"	22995	57488	91981	172465	275945	563388	1046293	1391225	1149773
CLASS 150 150 PSI RATING									
24"	255	637	1020	1913	3061	6251	11609	15437	12758
30"	857	2144	3431	6433	10293	21016	39030	51897	42890
36"	1792	4480	7168	13441	21505	43907	81542	108424	89607
42"	3202	8007	12811	24021	38434	78469	145729	193772	160142
48"	5412	13530	21648	40591	64946	132598	246254	327437	270609
54"	7336	18341	29346	55025	88040	179749	333820	443871	366835
60"	14428	36072	57715	108216	173146	353506	656512	872945	721442
66"	14942	37355	59768	112066	179306	366084	679872	904005	747112
72"	30124	75310	120496	225931	361490	738044	1370653	1822516	1506212

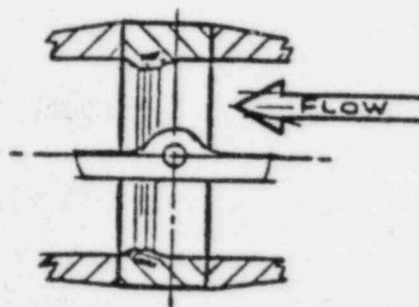
1.  $T_d$  values = in. lbs. per PSI  $\Delta P$ .

2. All  $T_d$  values are positive acting to shut valve.

3. 0  $T_d$  values  $\cong < 1$ .

TABLE 6

## CLASS 300 STD RATING



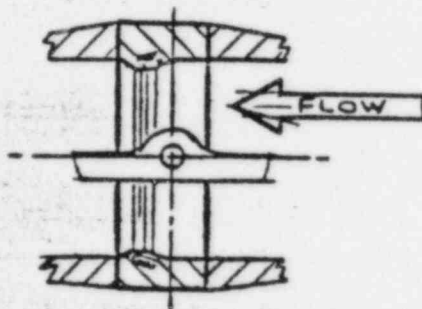
Seal Retaining Ring Downstream  
Disc Hydrodynamic Lift & Drag Torque —  $T_d$

VALVE SIZE	$T_d$ VS. DISC OPEN POSITION								
	10°	20°	30°	40°	50°	60°	70°	80°	90°
.....	.....	.....	.....	.....	.....	.....	.....	.....	.....
.....	.....	.....	.....	.....	.....	.....	.....	.....	.....
3"	0	0	0	0	1	2	5	6	5
4"	0	0	1	1	3	6	12	16	13
6"	1	2	4	8	13	27	51	68	56
8"	1	4	7	11	21	34	57	81	68
10"	3	9	17	27	51	81	136	192	162
12"	7	21	39	60	113	179	298	422	355
14"	9	29	54	84	158	250	416	590	495
16"	12	37	68	105	198	313	520	737	620
18"	28	86	158	244	460	726	1208	1711	1438
20"	30	92	169	261	493	778	1294	1833	1540
24"	41	125	229	355	668	1054	1754	2485	2088
30"	143	429	788	1218	2292	3618	6019	8526	7165
36"	287	861	1579	2441	4596	7253	12065	17092	14363
42"	686	2058	3773	5381	10976	17321	28812	40817	34300
48"	429	1287	2359	3646	6864	10832	18018	25526	21451

1.  $T_d$  values = in. lbs. per  $\text{PSI}\Delta P$ .
2. All  $T_d$  values are positive acting to shut valve.
3. 0  $T_d$  values  $\cong < 1$ .

TABLE 8

CLASS 600 STD RATING



Seal Retaining Ring Downstream  
Disc Hydrodynamic Lift & Drag Torque —  $T_d$

VALVE SIZE	$T_d$ VS. DISC OPEN POSITION								
	10°	20°	30°	40°	50°	60°	70°	80°	90°
3"	0	0	0	0	0	0	0	0	0
4"	0	0	0	0	0	0	0	0	0
6"	0	1	3	3	6	8	13	14	13
8"	0	5	10	12	19	27	44	46	43
10"	1	13	26	29	48	67	107	112	105
12"	1	20	38	43	71	99	158	165	154
14"	2	27	52	58	96	133	213	223	208
16"	2	36	69	78	128	178	284	298	279
18"	3	43	83	93	153	214	341	358	334
20"	3	41	79	88	146	203	324	340	317
24"	12	166	319	357	587	818	1303	1367	1278

1.  $T_d$  values = in. lbs. per PSI  $\Delta P$ .
2. All  $T_d$  values are positive acting to shut valve.
3. 0  $T_d$  values  $\cong < 1$ .



COMPARISON OF CALCULATED AERODYNAMIC TORQUES  
WITH TEST DATA

The purpose of this Enclosure is to compare aerodynamic torques calculated in the manner outlined in the report and compare them to measured aerodynamic torques.

Since at this time Posi-Seal does not have actual aerodynamic torque test data, use will be made of data given in ISA transaction 68-923, Ref. (a). Presented in Ref. (a) is a plot of both aerodynamic and hydrodynamic torque versus pressure drop. This plot is based on data obtained from a 4" butterfly valve with a disc rotation of 60 degrees and an upstream pressure of 214.4 psia.

Shown in Figure 1 on Page 3 of this Enclosure is a recreation of this plot. Superimposed on this plot are plots of calculated hydrodynamic torques and aerodynamic torques. The calculation of the hydrodynamic torque is nothing more than multiplying the hydrodynamic torque coefficient times the pressure drop. The hydrodynamic torque coefficient for a 4" 150 Class valve at 60 degrees is 6 per Ref. (b).

As for the aerodynamic torques, shown on Pages 4 thru 9 are computer printouts of aerodynamic torques which uses the same equations as the LOCA analysis.

As can be seen in Figure 1 relatively good correlation is obtained for both the calculated aerodynamic torque and the hydrodynamic torque.



It should also be noted that the method of calculating the aerodynamic torque presented in the ISA transaction is equivalent to the method developed by Posi-Seal as demonstrated below.

Per the ISA transaction the aerodynamic torque is calculated as follows:

$$T_D = K_1 D^3 \Delta P_e \quad \text{Equation (25)}$$

$$\text{where } \Delta P_e = P_1 \frac{C_1 C_2}{59.64}^2 \sin^2 \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right] \text{ rad}$$

Equations (16) & (26)

Posi-Seal's aerodynamic equation for air is:

$$T_{\text{air}} = .0436 T_{4w} \frac{T_1}{P_1} \left[ \frac{Q}{283.8 C_v} \right]^2$$

Since the expression  $K_1 D^3$  is equivalent to Posi-Seal's hydrodynamic torque coefficient  $T_{4w}$ ,

$$\Delta P \text{ should equal } .0436 \frac{T_1}{P_1} \left[ \frac{Q}{283.8 C_v} \right]^2$$

To prove this, the expression for Q given in the ISA transaction, Equation (14), is substitute into the above equation.

$$Q = \sqrt{\frac{520}{GT}} P_1 C_1 C_2 C_v \sin \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right] \text{ rad}$$

$$Q^2 = \frac{520}{GT} P_1^2 C_1^2 C_2^2 C_v^2 \sin^2 \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right]$$

$$\Delta P_e = \frac{.0436 \frac{T_1}{P_1}}{283.8^2 C_v^2} \left[ \frac{520}{GT} P_1^2 C_1^2 C_2^2 C_v^2 \sin^2 \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right] \right]$$

$$\Delta P_e = .0002815 P_1 C_1^2 C_2^2 \sin^2 \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right]$$

$$P_c = P_1 \left[ \frac{C_1 C_2}{59.60} \right]^2 \sin^2 \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right]$$

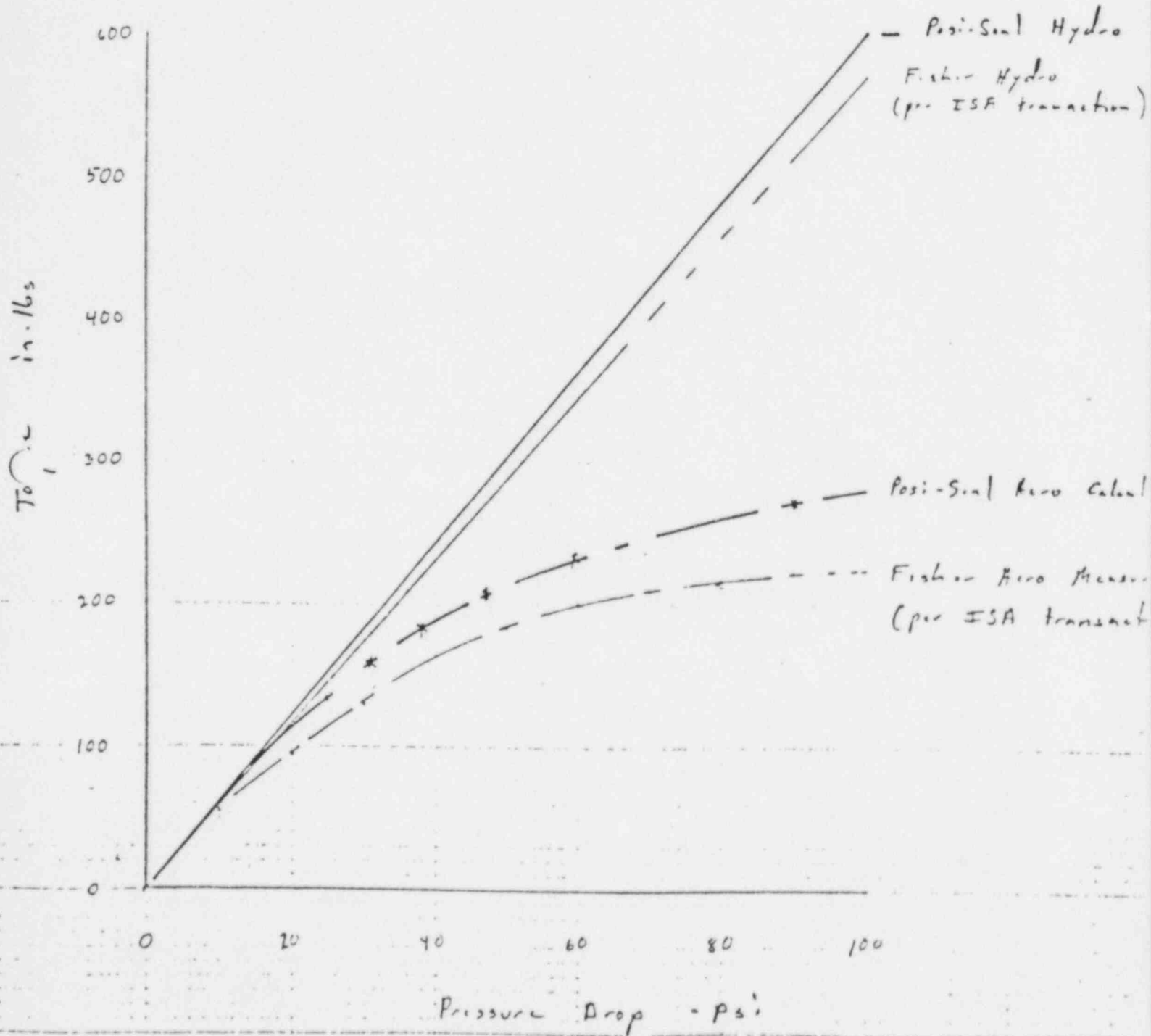
$$\approx P_1 \left[ \frac{C_1 C_2}{59.64} \right]^2 \sin^2 \left[ \frac{59.64}{C_1 C_2} \sqrt{\frac{\Delta P}{P_1}} \right]$$

REFERENCE (a) ISA Transaction 6 8-923 "Effect of Fluid Compressibility on Torque in Butterfly Valves", Floyd P. Harthun

REFERENCE (b) Posi-Seal Technical Bulletin 1A, June 1982

POSI-SEAL INTERNATIONAL, INC.  
ENGINEERING CALCULATIONS

Title 4" R.H. Fl. Valve - Flow induced torque Page 3  
Calc. By J. [signature] 1/7/85 Checked By \_\_\_\_\_



(1)  $P_1 = 214.4 \text{ psia}$   
Valve Angle =  $60^\circ$

CONTROL SYSTEM ANALYSIS - CAS

VALVE SIZE= 4

VALVE CLASS=100

AKO 1

INPUT DATA				
FLOW(SCFH) AT 90 DEG. 999999	SHUT-OFF PRESSURE 214.4	UPSTREAM PRESSURE 214.4	DOWNSTREAM PRESSURE 14.7	TEMPERATURE 70
Sg 1	HF @ 90 deg. 13	GV @ 90 deg. 404	Z 1	Fv 1
STEM DIA. .552	GAGE DIA. 3.375	PACKING 1	SEAL 1	BEARING 1

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 241.23 IN LBS      PI= 214.40 PSIA

CONDITIONS AT THE VARIOUS VALVE ANGLES

ANGLE	FLOW SCFH	PI	DP ACROSS VALVE	Topening	Tclosing	Tzero
90	1,000,000.0	214.40	8.73	141	-87	105
80	990,077.3	214.40	11.17	192	-110	155
70	973,065.9	214.40	15.23	104	-106	145
60	935,058.4	214.40	25.01	173	-90	175
50	851,315.3	214.40	50.55	200	-89	145
40	670,364.2	214.40	113.83	199	-22	110 CRITICAL
30	303,105.3	214.40	100.08	157	13	71 CRITICAL
20	207,939.4	214.40	124.35	135	51	40 CRITICAL
10	51,784.8	214.40	124.35	109	77	14 CRITICAL

CONTROL SYSTEM ANALYSIS - GAS

VALVE SIZE= 4

VALVE CLASS=300

AERO 1

FLOW(SCFH)	SHUT-OFF	INPUT DATA	DOWNSTREAM	TEMPERATURE
AT 90 DEG.	PRESSURE	UPSTREAM	PRESSURE	
1099000	214.4	214.4	14.7	70

Sg	HF @ 90 deg.	Cv @ 90 deg	Z	Fk
1	13	404	1	1

STEM DIA.	GAGE DIA.	PACKING	SEAL	BEARING
.562	3.375	1	1	1

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE SHUT  
VALVE TORQUE= 24173 IN LBS      PI= 214.40 PSIA

CONDITIONS AT THE VARIOUS VALVE ANGLES

ANGLE	FLOW SCFH	P1	DP ACROSS VALVE	Topening	Tclosing	Tzero
90	1,100,000.0	214.40	11.04	164	-40	127
80	1,086,834.6	214.40	13.77	226	-149	187
70	1,064,447.4	214.40	19.77	215	-133	174
60	1,016,266.2	214.40	35.98	206	-111	158
50	910,500.0	214.40	84.76	230	-102	166
40	870,356.2	214.40	113.63	199	-22	110 CRITICAL
30	393,105.3	214.40	120.06	157	12	75 CRITICAL
20	207,739.4	214.40	124.35	138	51	42 CRITICAL
10	51,234.0	214.40	124.35	108	79	14 CRITICAL

CONTROL SYSTEM ANALYSIS - CAS

VALVE SIZE= 4 VALVE CLASS=100 AERU 1

FLOW(SCFH) SHUT-OFF INPUT DATA  
AT 90 DEG. PRESSURE PRESSURE PRESSURE PRESSURE  
1200000 214.4 214.4 14.7 70

Sg HF @ 90 deg. Cv @ 90 deg. Z Fk  
1 11 404 1 1

STEM DIA. GAGE DIA. PACKING SEAL BEARING  
.552 3.876 1 1 1

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 241.23 IN LBS P1= 214.40 PSIA

CONDITIONS AT THE VARIOUS VALVE ANGLES

ANGLE	FLOW SCFH	P1	DP ACROSS VALVE	Topening	Tclosing	Taero
90	1,200,000.0	214.40	13.45	170	-114	152
80	1,192,765.8	214.40	16.76	230	-182	222
70	1,154,262.1	214.40	22.90	247	-161	204
60	1,092,070.0	214.40	33.07	231	-132	183
50	964,320.5	214.40	93.03	255	-109	167
40	670,364.2	214.40	113.63	197	-22	110 CRITICAL
30	303,106.3	214.40	120.05	167	15	75 CRITICAL
20	207,939.4	214.40	124.35	136	51	40 CRITICAL
10	51,794.8	214.40	124.35	108	79	14 CRITICAL

# CONTROL SYSTEM ANALYSIS - GAS

VALVE SIZE= 4

VALVE CLASS=300

AERO 1

FLOW(SCFH)	SHUT-OFF PRESSURE	INPUT DATA UPSTREAM PRESSURE	DOWNSTREAM PRESSURE	TEMPERATURE
AT 90 DEG. 1300000	214.4	214.4	14.7	70

Sg	HF @ 90 deg.	Cv @ 90 deg	Z	Fk
1	13	404	1	1

STEM DIA.	GAGE DIA.	PACKING	SEAL	BEARING
.352	3.795	1	1	1

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 24173 IN LBS P1= 214.40 PSIA

CONDITIONS AT THE VARIOUS VALVE ANGLES

ANGLE	FLOW SCFH	P1	DP ACROSS VALVE	Topening	Tclosing	Taero
90	1,300,000.0	214.40	16.25	210	-137	170
80	1,272,421.5	214.40	20.20	301	-218	259
70	1,242,112.1	214.40	27.67	292	-192	237
60	1,155,072.1	214.40	48.91	283	-153	200
50	959,715.8	214.40	102.91	272	-105	189 CRITICAL
40	670,264.2	214.40	113.63	199	-32	110 CRITICAL
30	333,105.2	214.40	120.08	167	18	75 CRITICAL
20	207,239.4	214.40	124.35	136	51	42 CRITICAL
10	51,784.8	214.40	124.35	108	77	14 CRITICAL



CONTROL SYSTEM ANALYSIS - GAS

VALVE SIZE= 4

VALVE CLASS=100

AERO 1

INPUT DATA

FLOW(SCFH) AT 90 DEG.	SHUT-OFF PRESSURE	UPSTREAM PRESSURE	DOWNSTREAM PRESSURE	TEMPERATURE
13999.9	214.4	214.4	14.7	70
Sq	HF @ 90 deg.	Cv @ 90 deg.	Z	Fk
1	13	404	1	1
STEM DIA.	GAGE DIA.	PACKING	SEAL	BEARING
.552	3.075	1	1	1

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE SHUT

VALVE TORQUE= 241.23 IN LBS      PI= 214.40 PSIA

CONDITIONS AT THE VARIOUS VALVE ANGLES

ANGLE	FLOW SCFH	PI	DP ACROSS VALVE	Topening	Tclosing	Taero
90	1,400,000.3	214.40	19.45	248	-185	207
80	1,373,155.3	214.40	24.18	343	-255	299
70	1,328,817.3	214.40	33.34	319	-303	271
60	1,235,513.3	214.40	53.93	295	-173	234
50	969,476.3	214.40	102.71	272	-105	188 CRITICAL
40	670,264.3	214.40	113.83	199	-22	110 CRITICAL
30	393,105.3	214.40	120.05	187	15	75 CRITICAL
20	207,939.4	214.40	124.35	136	51	42 CRITICAL
10	51,984.3	214.40	124.35	108	79	14 CRITICAL

# CONTROL SYSTEM ANALYSIS - CAS

VALVE SIZE- 4

VALVE CLASS-300

ACRO 1

FLOW(SCFH)	SHUT-OFF	INPUT DATA	DOWNSTREAM	TEMPERATURE
AT 90 DEG.	PRESSURE	UPSTREAM	PRESSURE	
1499999	214.4	214.4	14.7	70

Sq	HF @ 90 deg.	Cv @ 90 deg	Z	Fk
1	13	404	1	1

STEM DIA.	GAGE DIA.	PACKING	SEAL	BEARING
.532	1.376	1	1	1

FLOW IN PREFERRED DIRECTION

CONDITIONS WITH VALVE SHUT

VALVE TORQUE- 241.23 IN LBS PIF 214.40 PSIA

CONDITIONS AT THE VARIOUS VALVE ANGLES

ANGLE	FLOW SCFH	P1	DP ACROSS VALVE	Topening	Tclosing	Taero
90	1,500,000.0	214.40	33.27	231	174	237
80	1,467,121.2	214.40	29.95	387	295	341
70	1,413,408.4	214.40	40.23	359	255	307
60	1,304,977.3	214.40	90.64	313	183	246 CRITICAL
50	969,476.8	214.40	100.91	272	105	184 CRITICAL
40	670,864.2	214.40	113.63	199	22	110 CRITICAL
30	393,105.3	214.40	120.68	167	15	75 CRITICAL
20	207,939.4	214.40	124.35	136	51	42 CRITICAL
10	51,984.8	214.40	124.35	108	79	14 CRITICAL