

PRESSURE LOCKING EVALUATION FOR REQUIRED PULLOUT FORCE

= INPUTS

MOV

1-FCV-68-333

Valve type =

Flex Wedge

VALVE DESIGN DATA

VALVE DWG

88405-2

MARK No.

8000B

$\mu = 0.4$
 $a = 1.2810$ inches
 $b = 0.406$ inches

theta = 5 deg
 $ro = 0.406$ inches
 $v = 0.3$

PRESSURE and TEMPERATURE DATA

Pbon =
 Phigh =
 Flow =
 DP =
 Normal Temp =
 Max Temp =
 Delta Temp =
 Pbonnet = Pbon + (Delta Temp*33psi/F)

2250 psi
 0 psi
 0 psi
 0 psid
 0 F
 0 F
 0 F
 2250 psi

EPRI MOV PPP INTERNAL DESIGN INFO, Ref. _____

D1 E1 M1
 2.937 2.187 0.912

 $a = ((D1 + E1) / 2) / 2 = 1.281$ $b = M1 / 2 = 0.406 = ro$ $v = \text{Poisson's Ratio} = 0.3$ $\mu = \text{friction coefficient}$

based on Calc method. & EPRI results

theta = 1/2 total valve
 disk angle

ROARK FORMULAS

$C2 = (1/4)\{1 - (b/a)^2[1 + 2\ln(a/b)]\}$
 $C3 = (b/4a)\{[(b/a)^2 + 1]\ln(a/b) + (b/a)^2 - 1\}$
 $C8 = (1/2)[1 + v + (1 - v)(b/a)^2]$
 $C9 = (b/a)\{[(1 + v)/2]\ln(a/b) + [(1 - v)/4][1 - (b/a)^2]\}$
 $L11 = (1/64)\{1 + 4(ro/a)^2 - 5(ro/a)^4 - 4(ro/a)^2[2 + (ro/a)^2]\ln(a/ro)\}$
 $L17 = (1/4)\{1 - [(1 - v)/4][1 - (ro/a)^4] - (ro/a)^2[1 + (1 + v)\ln(a/ro)]\}$

0.167176
 0.028914
 0.685158
 0.286608
 0.005962
 0.144066

Load Constant = $(C2L17 - C8L11) / (C2C9 - C3C8)$

0.711633

SEAT REACTION LOAD (BONNET psi)Reaction at Hub Perimeter = $QH(\text{Bonnet}) = P_{\text{bonnet}}(a)(\text{Load Constant})$ Reaction at Seat: $QS(\text{Bonnet}) = QH(\text{Bonnet})(b/a) - (P_{\text{bonnet}}/2a)(a^2 - b^2)$

2051.1 lb/in
 -646.3 lb/in

SEAT REACTION LOAD (Disk Area DP)Reaction at Hub perimeter = $QH(\text{DiskDP}) = (DP)(a)(\text{Load Constant})$ Reaction at Seat = $QS(\text{DiskDP}) = QH(\text{DiskDP})(b/a) - ((DP)/2a)(a^2 - b^2)$

0 lb/in
 0 lb/in

SEAT REACTION LOAD (Hub Area DP)Force on Hub due to DP = $W_{\text{hub}} = QH(DP_{\text{disk}}) + (DP)PI(b^2)/2PI(r$ Reaction at Seat = $QS(\text{HubDP}) = (-W_{\text{hub}})(b/a)$

0 lb/in
 0 lb/in

Sum of Seat Reaction Loads = $2 \cdot QS(\text{Bonnet}) - QS(\text{DiskDP}) + QS(\text{HubDP})$ $VF = \mu / (\cos A + \mu \sin A)$ Thrust Load = $VF \cdot (\text{Sum Seat Reaction}) \cdot (\text{Seat Circumference})$

Unwedging Load (Static test)

-1292.57 lb/in
 0.3879
 -4035.97 lbs
 -3666 lbs

TOTAL THRUST = Thrust Load + Unwedging Load =

NOTE: THE NEGATIVE SIGN INDICATES THAT THE STEM IS IN TENSION

-7702 lbs

7608130257 740806
 PDR ADOCK 05000327
 P PDR

Rev.

Prepared:

Checked:

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VALVE DWG

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MARK No.

8000B

VALVE DESIGN DATA

$\mu = 0.4$
 $a = 1.2810$ inches
 $b = 0.406$ inches

theta = 5 deg
 $r_o = 0.406$ inches
 $v = 0.3$

EPRI MOV PPP INTERNAL DESIGN INFO, Ref. _____

D1 E1 M1
 2.937 2.187 0.812

 $a = ((D1 + E1) / 2) / 2 = 1.281$ $b = M1 / 2 = 0.406 = r_o$ $v = \text{Poisson's Ratio} = 0.3$ $\mu = \text{friction coefficient}$

based on Calc method. & EPRI results

theta = 1/2 total valve
 disk angle

PRESSURE and TEMPERATURE DATA

Pbon =
 Phigh =
 Plow =
 DP =
 Normal Temp =
 Max Temp =
 Delta Temp =
 $P_{bonnet} = P_{bon} + (\text{Delta Temp} \times 33 \text{ psi/F})$

2235 psi
 300 psi
 0 psi
 300 psid
 0 F
 0 F
 0 F
 2235 psi

ROARK FORMULAS

$C2 = (1/4) \{ 1 - (b/a)^2 [1 + 2 \ln(a/b)] \}$
 $C3 = (b/4a) \{ [(b/a)^2 + 1] \ln(a/b) + (b/a)^2 - 1 \}$
 $C8 = (1/2) [1 + v + (1-v)(b/a)^2]$
 $C9 = (b/a) \{ [(1+v)/2] \ln(a/b) + [(1-v)/4] [1 - (b/a)^2] \}$
 $L11 = (1/64) \{ 1 + 4(ro/a)^2 - 5(ro/a)^4 - 4(ro/a)^2 [2 + (ro/a)^2] \ln(a/ro) \}$
 $L17 = (1/4) \{ 1 - [(1-v)/4] [1 - (ro/a)^4] - (ro/a)^2 [1 + (1+v) \ln(a/ro)] \}$

0.167176
 0.028914
 0.685158
 0.286608
 0.005962
 0.144066

Load Constant = $(C2L17 - C8L11) / (C2C9 - C3C8)$

0.711633

SEAT REACTION LOAD (BONNET psi)

Reaction at Hub Perimeter = $QH(\text{Bonnet}) = P_{bonnet}(a)(\text{Load Constant})$
 Reaction at Seat: $QS(\text{Bonnet}) = QH(\text{Bonnet})(b/a) - (P_{bonnet}/2a)(a^2 - b^2)$

2037.4 lb/in
 -642.0 lb/in

SEAT REACTION LOAD (Disk Area DP)

Reaction at Hub perimeter = $QH(\text{DiskDP}) = (DP)(a)(\text{Load Constant})$
 Reaction at Seat = $QS(\text{DiskDP}) = QH(\text{DiskDP})(b/a) - ((DP)/2a)(a^2 - b^2)$

273.4806 lb/in
 -86.1714 lb/in

SEAT REACTION LOAD (Hub Area DP)

Force on Hub due to DP = $W_{hub} = QH(DP_{disk}) + (DP)PI(b^2)/2PI(b)$
 Reaction at Seat = $QS(\text{HubDP}) = (-W_{hub})(b/a)$

334.3806 lb/in
 -105.979 lb/in

Sum of Seat Reaction Loads = $2 \times QS(\text{Bonnet}) - QS(\text{DiskDP}) + QS(\text{HubDP})$ VF = $\mu / (\cos A + \mu \sin A)$ Thrust Load = $VF \times (\text{Sum Seat Reaction}) \times (\text{Seat Circumference})$

Unwedging Load (Static test)

-1303.76 lb/in
 0.387951
 -4070.91 lbs
 -3668 lbs

TOTAL THRUST = Thrust Load + Unwedging Load =

-7737 lbs

NOTE: THE NEGATIVE SIGN INDICATES THAT THE STEM IS IN TENSION

PRESSURE LOCKING REQUIREMENT =

7737 lbs

ACTUATOR/VALVE CAPABILITY EVALUATION

CALCULATION:

EPM-RJP-031191

Actuator Model

SMB-00

Motor Start Torque (MT)

25 flbs

Overall Gear Ratio (OGR)

49

Pullout Efficiency (PE)

40 %

Application Factor (AF)

0.9 or 1.0

0.9

Valve Factor (VF)

1.0 or (VF/460)**2

1.0000

424.4VAC

Stem Factor (SF)

0.0156

Max Temperature (Temp)

327 F

Loss Percentage (elevated temp)

23.2 %

Note 1

Torque Loss Factor = $1 - \text{Loss\%} \cdot (\text{Temp} - 104) / (356 - 77)$

0.8146

Actuator Torque Capability = $(\text{MT} \cdot \text{OGR} \cdot \text{PE} \cdot \text{AF} \cdot \text{TLF} \cdot \text{VF})$

359.22 flbs

SF =

Actuator Thrust Capability = Torque/Stem Factor

23027 lbs

0.0156

Actuator Thrust /Torque Rating

19600 lbs

250.00 flbs

Valve Weak Link Thrust/Torque Rating

14650 lbs

SF =

228.54 flbs

0.0156

Seismic Thrust/Torque Rating

21500 lbs

SF =

335.40 flbs

0.0156

Note 1: Reference: Limitorque Technical Update 93-03

RESULTS:

PRESSURE LOCKING REQUIREMENT =

7737 lbs

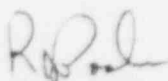
ACTUATOR / VALVE LIMITING CAPABILITY =

14650 lbs

COMMENTS:

Rev.

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Checked:

