

COMBUSTION ENGINEERING

NUCLEAR POWER SYSTEMS
COMBUSTION ENGINEERING, INC.
COMPONENT SERVICES
Chattanooga, Tennessee

5-0000

WATERFORD UNIT NO. 3
STEAM GENERATOR MANWAYS

CENC-1805

CALCULATION NO.

74270

CONTRACT NO.

LOUISIANA POWER AND LIGHT
CUSTOMER

STEAM GENERATOR
COMPONENT

J. E. Roberts Structural Engineer 3-14-88
PREPARED TITLE DATE

R. B. Hale Structural Engineer 3/15/88
VERIFIED TITLE DATE

P. L. Anderson Supervisor 3/15/88
REVIEWED TITLE DATE

W. J. Hale Manager 3/15/88
APPROVED TITLE DATE

Short PREPARED Date 3-15-88 CHECKED TITLE	3-15-88 DATE 3-15-88 DATE	COMBUSTION ENGINEERING COMPONENT ENGINEERING	SHEET 1 OF 6 CALC. NO. CENC-1805 CONTRACT NO. 74270								
WATERFORD UNIT No. 3 STEAM GENERATOR MANWAYS											
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3-7-88

DATE

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3-15-88

DATE

TITLE

CONSTRUCTION ENGINEERING

COMPONENT ENGINEERING

SHEET 5 OF 6
CALL NO. CENC-1805
CONTRACT NO. 74270

WATERFORD UNIT No. 3 STEAM
GENERATOR MANWAYS

4. ANALYSIS

B. PRIMARY MANWAY REINFORCEMENT (REF. 1 SHEET A-42)

$A_1 = 89.769 \text{ in}^2$

$A_2 = 8.387 \text{ in}^2$

$A_{\text{BOLT}} = 3.125 (1.625) = 5.078 \text{ in}^2$

$A_{\text{AVAILABLE}} = A_1 + A_2 - A_{\text{BOLT}} = 43.013 \text{ in}^2 > A_{\text{REQD}} = 31.906 \text{ in}^2$

THE REINFORCEMENT REQUIREMENT IS MET BASED ON
USING A 1 1/2" HELICOIL INSERT.

C. SECONDARY MANWAY REINFORCEMENT (REF. 1 SHEET A-48)

$A_1 = 1.596 \text{ in}^2$

$A_2 = 10.370 \text{ in}^2$

$A_3 = 4.396 \text{ in}^2$

$A_4 = (4.24)(\text{CH} - \text{TRN}) - 1.625(3) - (1.032 - \text{CH}) (.375)$
 $A_2 = (4.24)(8.561) - 4.875 - (1.630)(.375) = 31.187 \text{ in}^2$

$\text{AVAILABLE } A = A_1 + A_2 + A_3 + A_4 = 47.499 \text{ in}^2 > A_{\text{REQD}} = 43.452 \text{ in}^2$

THE REINFORCEMENT REQUIREMENT IS MET BASED ON
USING A 1 1/2" HELICOIL INSERT.

D. SHEAR STRENGTH OF HELICOIL / STUD - PRIMARY MANWAY
HELICOIL SPECIFICATIONS

MATERIAL: SA-479 TYPE 304 REF. 4

S_m : 16.2 KSI @ 650 REF. 2

SIZE: 1 1/2" - BN x 2 1/2" REF. 3

STRENGTH OF THE STUD

TENSILE STRESS AREA OF STUD BASED ON SHANK DIAMETER d

$A = \frac{\pi d^2}{4} = \frac{\pi (1.31)^2}{4} = 1.348 \text{ in}^2$

MAXIMUM DESIGN STUD LOAD AT 650°F

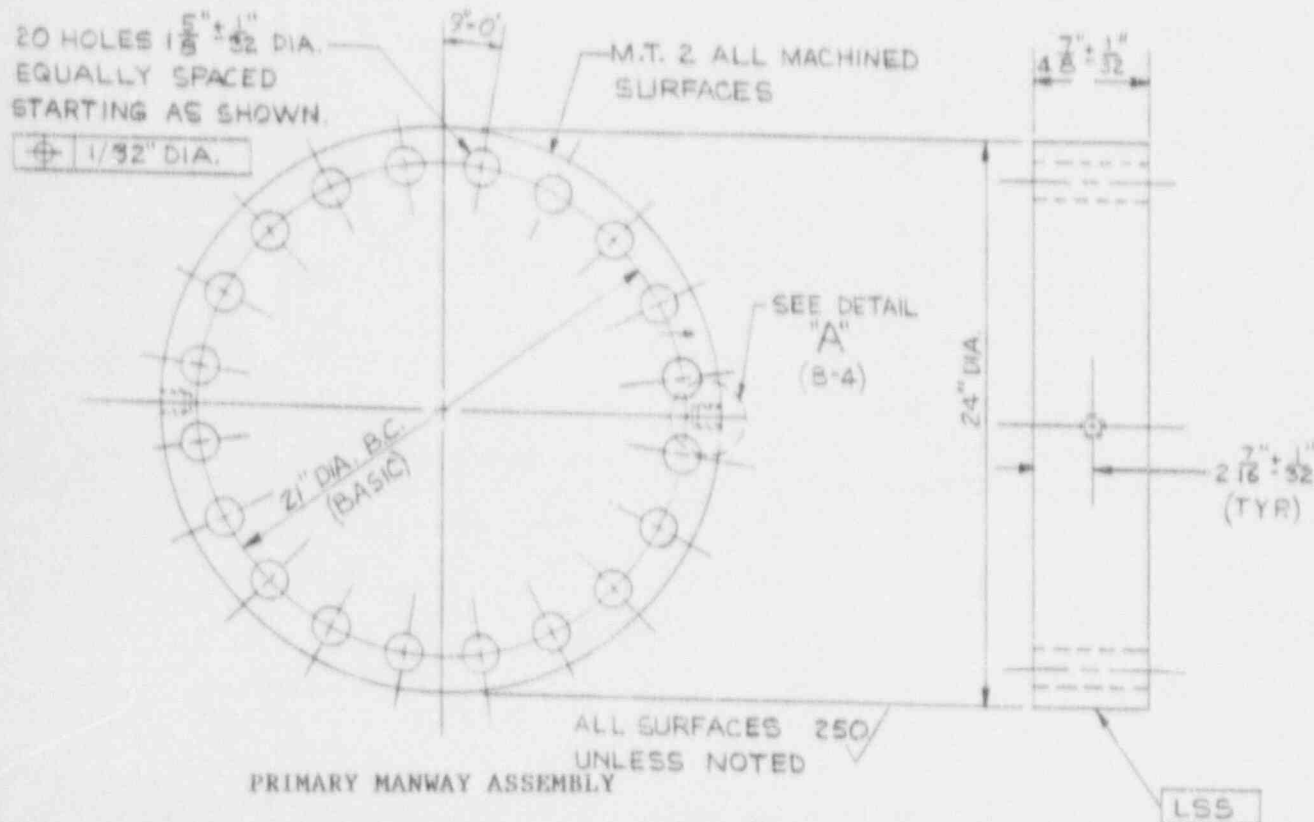
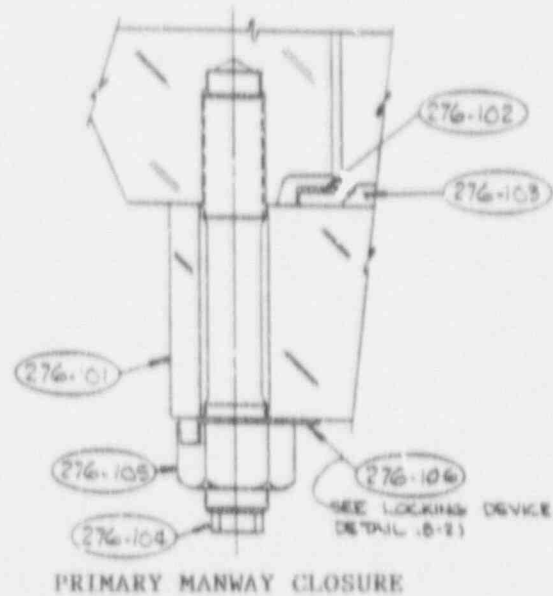
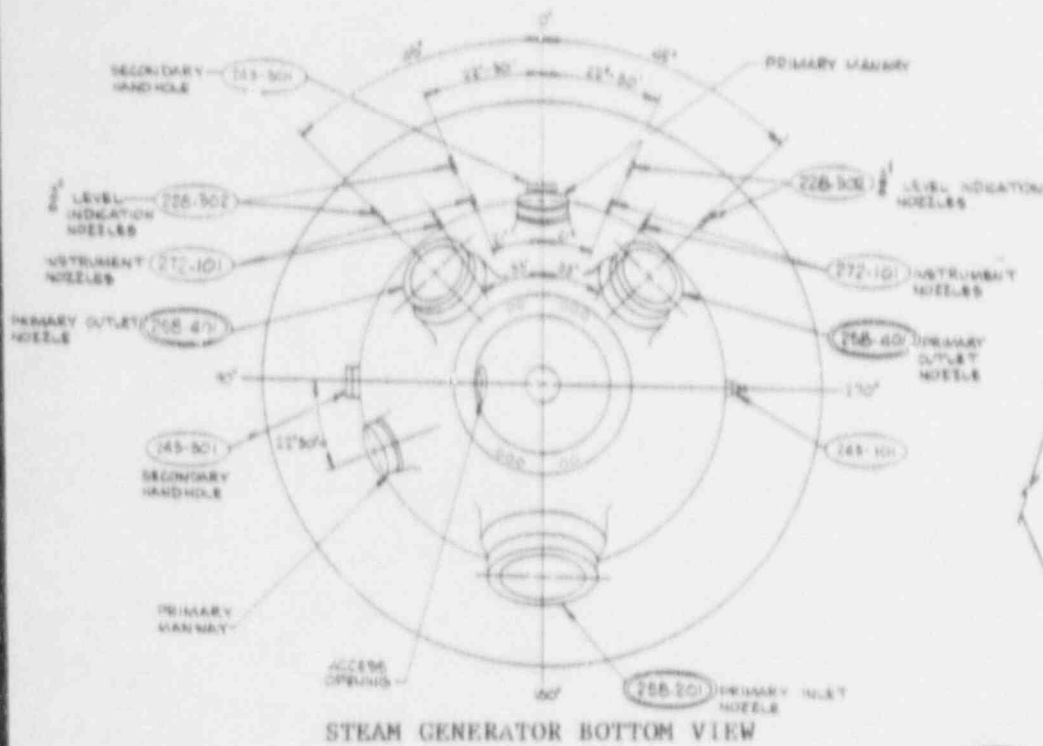
$F = A S_m = 1.348 (34.8) = 46.91 \text{ KIPS}$

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B-4007 5 10 6
 CALS. NO. CENC-1805
 CONTRACT NO. 74370

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STEAM GENERATOR PRIMARY MANWAY CLOSURE DETAILS



Waterford 3
Steam Generator Design Data

Application:	Steam Generator Primary Manway Closure, 20 Studs per Closure
Oper. Pressure/Temperature:	2250 psia/611°F (hot leg)/553°F (cold leg)
Design Pressure/Temperature:	2500 psia/650°F
Stud Size:	1-1/2" - 8N - 2A
Stud Material:	SA-540, Grade B24, Class 3 (Sy) min. yield 130 ksi
Manway Material:	SA-533, Class 2 (Sy) min. yield 50 ksi
Helical Material:	SA-479, Type 304 stainless (Sy) min. yield 150 ksi (after cold working)
Max. Stud Service Stress Intensity, Sm: (from preload, pressure, and operating transients)	36.7 ksi @ 611°F operating temperature 36.7 ksi is less than 71.4 ksi NOTE: ASME Section III allowable at 611°F is 71.4 ksi from $2 \times S_m = 71.4 \text{ ksi}$ (S_m Design Value = 35.7)

Sy = Yield Stress
Sm = Stress intensity
KSI = Kips per in²