

ENVIRONMENTAL QUALIFICATION  
OF  
MECHANICAL EQUIPMENT  
FOR  
THE COMMONWEALTH EDISON COMPANY  
BYRON STATION UNITS 1 AND 2  
BRAIDWOOD STATION UNITS 1 AND 2  
COMPONENT:  
PACIFIC SAFETY INJECTION PUMP  
MODEL 3" 10 STAGE JHF  
REV. 0  
5/23/83

PREPARED BY: M.J. ZEGAR

APPROVED BY: L.I. WALKER

*M J Zegar 4/20/83*

*L I Walker 6/20/83*

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## Introduction

This report documents the Environmental Qualification of the Pacific Safety Injection Pump Model 3" 10 Stage JHF for Commonwealth Edison Byron Station Units 1 and 2 and Braidwood Station Units 1 and 2. The evaluation documented herein verifies that the component, as originally designed, is adequate to meet the plant environmental conditions as specified in the Equipment Specification.

1.0 Equipment/Document Identification

1.1 Specification No.: 677474 Rev. 0 (motor), 678815 Rev. 2 (pump)

1.2 Vendor: Pacific Pumps

1.3 Model/Type: 3 inch 10 stage JHF Safety Injection Pump

1.4 Spin No/Serial No.

SIAPSI - CAE/CBE/CCE/CDE

CAE - 49758/59, CBE - 49760/61

CCE - 49762/63, CDE - 49764/65

1.5 Reference Drawings: AXS-49754 Rev. 7, MB-7187 Rev. E, H-SP-1786-5 Rev. E, JLF-49754 Rev. 2, 5-162-06-018-003 Rev. 2, FC-49758 Rev. 6, VBC-49754 Rev. 4, PLO-49758 Rev. 2.

1.6 Equipment Function(s): To provide emergency core cooling in the event of a break in either the reactor coolant or steam system. Injects water from the refueling water storage tank (RWST) and recirculates water from the containment sump after the RWST supply is exhausted.

1.7 Seismic Qualification Report:

K363, K386 Rev. 3 (pump)

S.O. 75F32374 (motor)



## 2.0 ENVIRONMENTAL SUMMARY

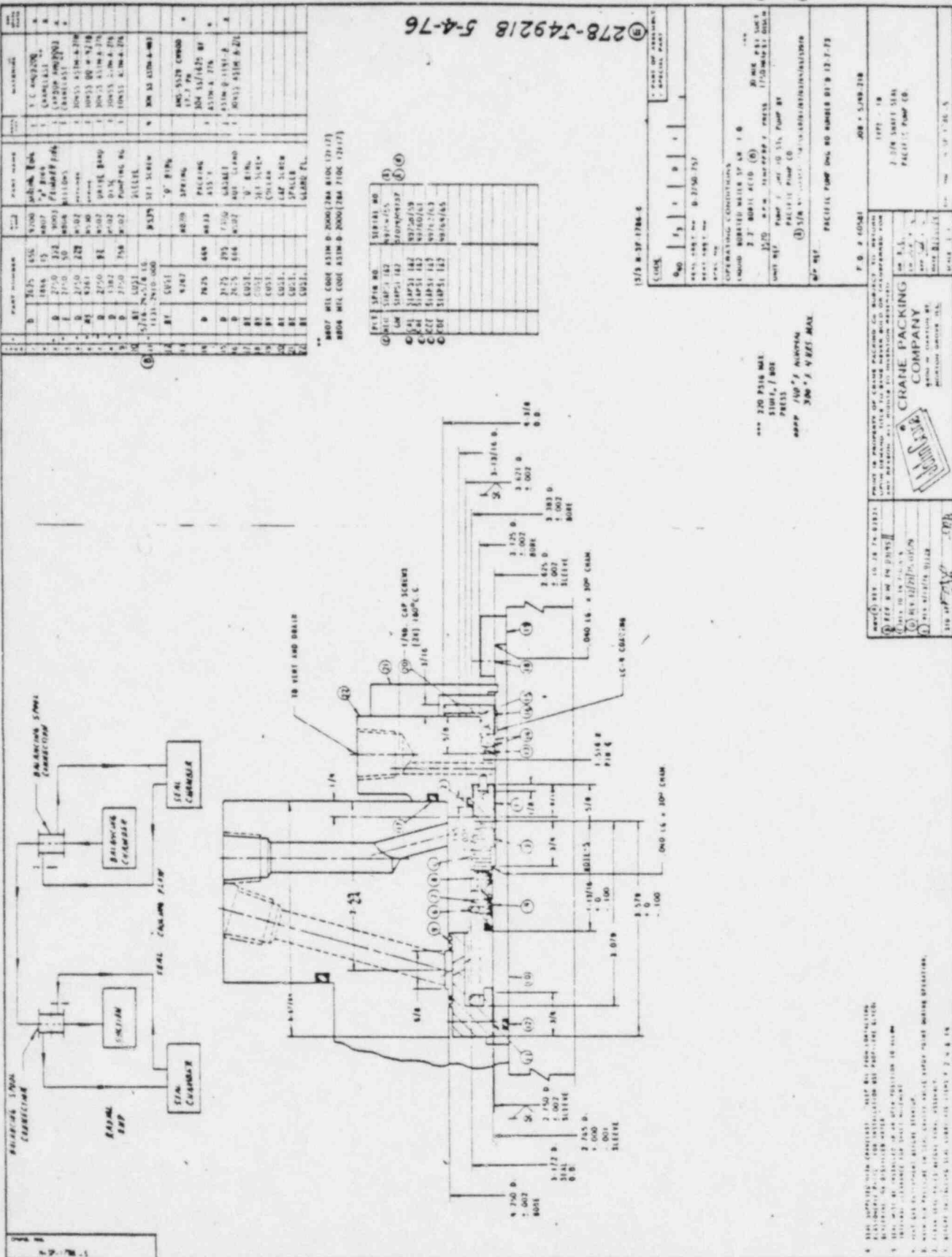
ENVIRONMENTAL PARAMETER	POSTULATED	SPECIFICATION
MAXIMUM TEMPERATURE (°F)	120	130
MAXIMUM PRESSURE	ATM	ATM
MAXIMUM RELATIVE HUMIDITY	95%	NOT SPECIFIED
CONTAINMENT SPRAY	NOT APPLICABLE	NOT APPLICABLE
40 YEAR NORMAL RADIATION DOSE (RADS)	$1.2 \times 10^7$	$1 \times 10^7$
ACCIDENT RADIATION DOSE RADS	INCLUDED IN NORMAL DOSE	INCLUDED IN NORMAL DOSE
TOTAL RADIATION DOSE (RADS)	$1.2 \times 10^7$	$1 \times 10^7$
SUBMERGENCE (YES/NO)	NO	NO

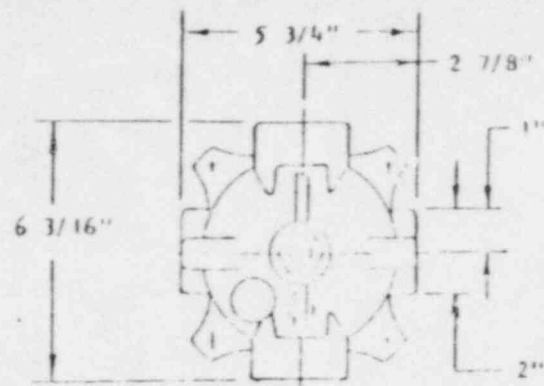
FLUID CONDITONS: DILUTED BORATED WATER: 40° TO 100°F,  
0 - 35 FT.  
ACCIDENT SUMP WATER: 40 TO 300°F, 0 TO 350 FT.

### 3.0 PUMP OUTLINE DRAWINGS









MANUFACTURER - AMF CUNO DIVISION

TYPE *EG AUTO-KLEAN*

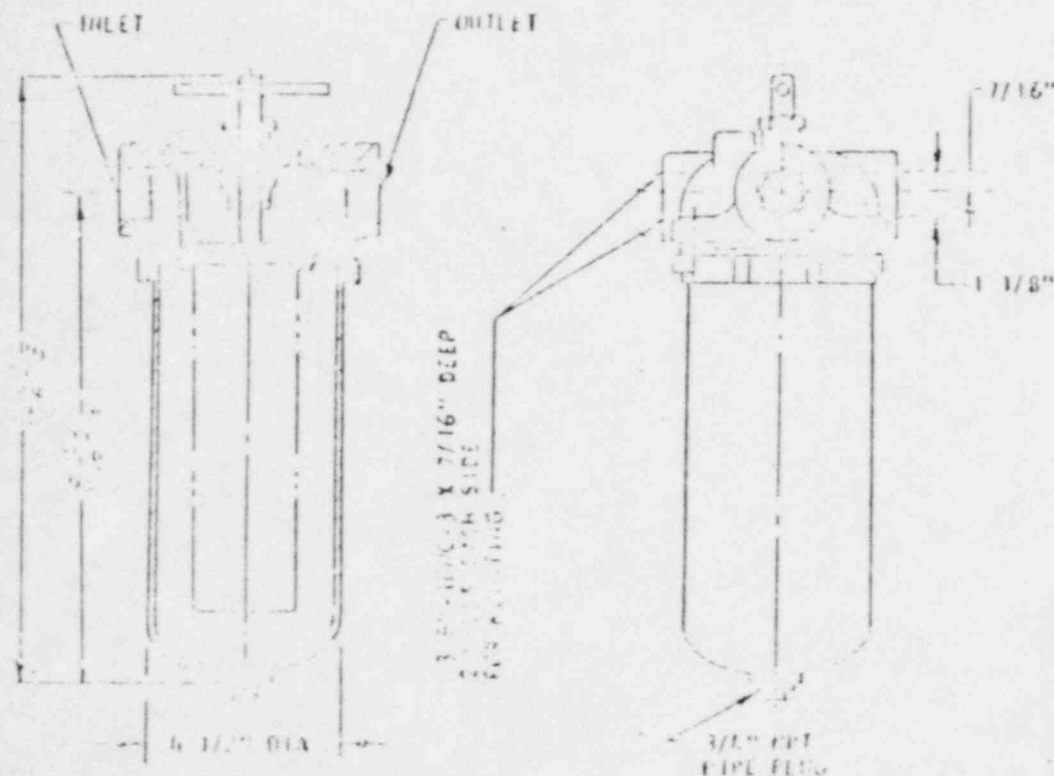
FURNISHED WITH BUILT-IN DIFFERENTIAL PRESSURE BY-PASS VALVE, SET AT 10 PSI DIFFERENTIAL.

INLET / OUTLET SIZE *1" NOT*

CARTRIDGE: SIZE *5"* MATERIAL *BRONZE*

SPACING *0.005"*

MATERIAL OF HEAD *CAST IRON* SUMP *STEEL*



WESTINGHOUSE HES

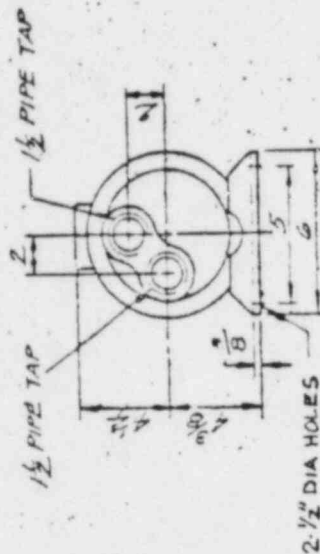
3" - JHP - 10 STAGES

SAFETY INJECTION PUMPS

PP SHOP ORDER NO.	PUMP SERIAL NO.	PLANT AND SPIN NO.
JA 4975	49754/55	GBE-SIAPSI-1/2
JB 4976	49757	GBE-SIAPSI-2
JC 4977	49755/59	GBE-SIAPSI-1/2
JD 4978	49760/61	GBE-SIAPSI-1/2
JE 4979	49762/63	GBE-SIAPSI-1/2
JF 4980	49757/65	GBE-SIAPSI-1/2
JG 49218	52079	GBE-SIAPSI-1

2	5-4-75 R.W. / 8-23-76 P.O. GBE SIAPSI-1/2 PUMP SERIAL NUMBER HAS BEEN REVISED	
1	1-10-75 R.W. / 1-10-75 P.O. REVISED PLANT & SPIN NO'S ORDERING NO JLF 49758 SUBMITTED BY THIS REV	INSURANCE (C) PACIFIC PUMPS
	PL 1-10-75	49754



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AMERICAN SIOILY TRADING DIV.

AMERICAN STD H&A TRADING DIV.  
10000 CLEVELAND PARKWAY

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**Figure 1**


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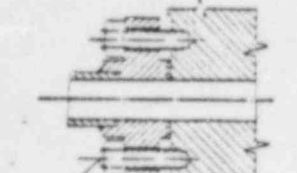
*Journal of Management Education* 30(6)



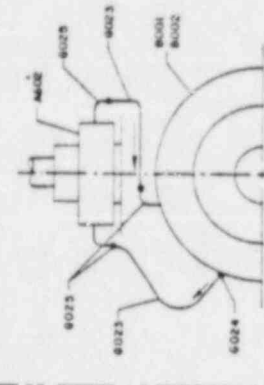
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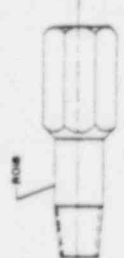
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P0	AG02	SEI	AW	AG01	AW	AG02	AW
P0	AG04	*	*	*	*	*	*
P0	AG02	*	*	*	*	*	*
P0	AG03	1.00000	-3.8	215	100	74	554
P0	AG04	1.11100	-3.516	14	-276	1174	545

ADZ1	PIN TAPER	ADZ1	PACKET	COMMENSED ADDRESS	MAZ131
PR	A721	CASE	STUD	SA 183 00 07	
PR	A722	CASE	STUD	SA 183 00 07	
PR	A723	CASE	STUD	SA 183 00 07	
PR	A731	CASE	CAP MUT	SA 184 00 07	
PR	A732	CASE	CAP MUT	SA 184 00 07	
A771	STUD				
A772	STUD				
A780	MAZ116				
A781	MAZ116				



PRELIMINARY

FOR COOL WATER CONNECTIONS AT LINE OR COOLER

FOR DETAILS SEE	-N
PLANT FOUNDATION PL. 101	-AMERICAN S.D. Dwg.
NEU. FC. 49254	D 2 162 06 000 003
G.O.I. FC. 49254	D 2 162 06 000 003
FC. 49259	D 2 162 06 000 003

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## PACIFIC PUMPS, INC.

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## PARTS LIST

WESTINGHOUSE ELECTRIC CORP.

OUR ORDER NO. 359218

QTY	TOTAL REQD	PART NO.	DESCRIPTION	PATTERN NO.	QUANTITY	LINE NO.	COLUMN NO.	MATERIAL	WEIGHT (LBS)	UNIT
1	12	A001	FLW 10 STG FINAL 3 JHP CASE		D15145			193		
1	12	A053	BUSH FINAL SP 13 PRESS REDUCING		E17677	1		132	25	SP
1	12	A538	WHEEL PR SP 20 1/2X7/8LG					23		SP
2	24	A521	PIN TAPER P/F 10X6LG		JT1000	6		14		
1	12	A551	GASKET 1/64THK					147		
1	1	A551	CASE PARTING FLG GASKET 57X22X1/64THKB					147		
1	2	A502	SP4 BY PASS ASSY	C	D16468			192		SP
2	24	A612	FLEXITALLIC CG 6E GSKT					244		
8	96	A613	SP20 5/8 11NCX4 3/4LG STUD TFL					259		SP
8	96	A614	SP20 5/8 11NC NUT HEX					183		SP
32	384	A751	CASE CAP NUT SP 6 2		B20277			183		SP
6	72	A752	CASE CAP NUT SP 6 1 1/2		B20279			183		SP
32	128	A771	SLUG P/F 2		A62221	9		14		
6	24	A772	SLUG P/F 1 1/2		A62221	5		14		
1	12	R001	FINAL RAD SP 8 3 SEAL HSG CCM		D15269					SP
1	12	R002	FINAL THM SP 8 3 SEAL HSG CCM		D15270					SP
2	24	R010	GASKET SEAL HSG ARP 568 369					174		
2	2	R010	SEAL HOUSING GASKET ARP 568 369	B				174		
12	144	R711	SCW SOC CAP SP20 3/4 10NCX3 1/4LG					652		SP
10	120	R005	SP 20 3 SHAFT SPLIT RING		A61893			42	21	SP
10	120	R007	ROTH END RD SP 20 KEY IMP		A20207	12	1	23		SP
1	12	R010	FINAL CLR.010.012 SPI2 3 SLEEVE PR		B20198	2		22	60	SP
1	12	R013	ROTH ENDS RD SP20 KEY SLV PR		A20207	12	1	23		SP
1	12	R014	KEY FLEX CPLG ONE END RD ONE END SQ		A20216	6	1	14		
1	12	R040	SP 13 LOCKNUT IMP SUCT END		A61892			42	21	SP
1	12	R042	SP 13 LOCKNUT SLV PR		A61891			42	21	SP
1	12	R044	SP 13 SLV IMP SPACER 1ST STG		A61897	9		42	21	SP
1	12	R045	SP 13 SLV IMP SPACER 2ND STG		A61897	8		42	21	SP
1	12	R046	SP 13 SLV IMP SPACER 3RD STG		A61897	7		42	21	SP
1	12	R047	SP 13 SLV IMP SPACER 4TH STG		A61897	6		42	21	SP
1	12	R048	SP 13 SLV IMP SPACER 5TH STG		A61897	5		42	21	SP
1	12	R049	SP 13 SLV IMP SPACER 6TH STG		A61897	4		42	21	SP
1	12	R050	SP 13 SLV IMP SPACER 7TH STG		A61897	3		42	21	SP
1	12	R051	SP 13 SLV IMP SPACER 8TH STG		A61897	2		42	21	SP
1	12	R052	SP 13 SLV IMP SPACER 9TH STG		A61897	1		42	21	SP
1	12	R068	SP 13 WRG RING SUCT SPACER		A61894			132	25	SP
3	36	R069	INTERM CVR WRG SP 20 LOCKING KEY		A22985	2		23		SP
1	12	R084	SP20 3 DISCH DIFF SPGR RG CLR.010.020		A61898	1		23		SP
9	108	R086	SP 13 WRG RING CVR INTERM		A61895			132	25	SP
27	324	R087	SP 20 LOCKING KEY WRG RING		A22985	2		23		SP
9	108	R089	SP 13 BUSH CVR INTERM		A61896			132	25	SP
27	324	R090	SP 20 LOCKING KEY BUSH CVR INT		A60504	6		23		SP
10	120	R095	INTERM SUCT SPACER 0 RING CVR		B12574	23		174		
10	10	R095	COVER 0 RING INTERM & SUCT SPACER	B	B12574	23		174		
2	24	R591	SCW SOC CUP PT SP 20 1/4 20NCX3/8LG					23		SP
10	120	R597	WHEEL CIGSUCT SPACER SP20 1/4X3/4LG					23		SP
9	108	R511	WHEEL SLV SPACER SP20 3/32X3/8LG					23		SP
1	12	F001	HSG RAD BRG U/H & KBJRY	H5739	D15148	1		2		
1	12	F001	HOUSING RAD BRG L/H	H5758	D15148	1		2		
1	12	F020	BEARING RAD SLV		C10602	3		277		

## PACIFIC PUMPS, INC.

Page 2 of 3

## PARTS LIST

WESTINGHOUSE ELECTRIC CORP.

OUR ORDER NO.		PART NO.		DESCRIPTION		PATTERN NO.		MATERIAL NO.		LINE NO.		QUANTITY		WEIGHT		REMARKS	
149218																	
1	1	F020		RADIAL BRG SLEEVE	B		E10602			3			277				
1	12	F024		BAFFLE STAT OIL 2 HLVS OUTER			B12426			1	3		11				
1	12	F025		BAFFLE STAT OIL 2 HLVS INNER			B12426			1	3		11				
1	12	F029		DEFLECTOR RING			A17097			3			11				
1	12	F001		HOUSING THR BRG L/H		M6283	B15149			1			2				
1	12	F001		HOUSING THR BRG & KBURY U/H		M6282	B15149			1			2				
1	12	F017		BEARING THR SLV			E10519			3			277				
1	1	F017		THRUST BRG SLEEVE	B		E10519			3			277				
1	12	F020		BAFFLE STA OIL 2 HLVS INNER			B12426			1	3		11				
1	12	F022		DEFLECTOR RING			A17097			3			11				
1	12	F025		COVER END CCW		M6284	B07637						2				
1	12	F030		BEARING THR LESS COLLAR & JHJ KBURY			E00100			6							
12	12	F030		THRUST BRG SHOE & JHJ KBURY	B												
1	12	F037		COLLAR THR & KBURY			B11343			19			15				
1	1	F037		COLLAR THR & KBURY	B		B11343			19			15				
1	12	F038		KEY THR COLLAR BOTH ENDS ROUND			A20210			2	1		14				
1	12	F039		SPACER THR COLLAR			B10100			6	16		185				
1	12	F040		NUT THR CCW ROT LH THREAD			A25047						68			16	
1	12	F042		PLATE RETAINER INNER			A23355						14				
1	12	F043		PLATE RETAINER OUTER			A23398						11				
1	12	F045		HOUSING GEAR PUMP		M5747	B06151			2			11				
1	12	F046		COVER GEAR PUMP			A19302						11				
1	12	F054		WORM PUMP SET			A19630			1			89				
1	12	F055		KEY WORM			A20207			23	1		14				
1	12	F056		FA LOCKNUT & WASHER SKF N W 06			LH0100				6						
1	12	F058		SHAFT GEAR PUMP			A19316						185				
1	12	F059		COLLAR THR WORM G			A19315			1			185				
1	12	F060		GEAR WORM			A19631						106				
1	12	F061		KEY GEAR WORM			A20207			17	2		14				
1	12	F062		FA LOCKNUT & WASHER SKF N W 04			LH0100				4						
1	12	F065		GEAR PUMP			B10744			2			89				
1	12	F066		KEY GEAR PUMP			A20207			21	1		14				
1	12	F067		SHAFT IDLER GEAR			A19317						185				
1	12	F068		GEAR IDLER			B10744			1			89				
2	24	G002		J CR 18 W/PR AUX PACK BS 1 SEAL													
2	2	G002		MECH SEAL ITEMS 1 THRU 16 BSI	B												
2	24	G003		SP 7 PLATE SEAL FINAL			C17814										SP
2	24	G006		GASKET O RING ARP 5/8 246									374				
2	2	G006		SEAL PLATE O RING ARP 5/8 246	B								374				
8	96	G007		SCR CAP SOCK HD SP20 1/2 13NCX1 3/4LG									452				SP
2	4	G008		PLATE RETAINER	B		A61970			1			21				
2	24	G009		SCR CAP SOCK HD 3/8 16NCX3/4LG			HR0061			60	12		23				
8	96	G011		SCR SOC HD CAP 1/4 20NCX1/2LG			HR0042				8		23				
2	24	G012		SP 14 SLEEVE SEAR SHAFT			B20274						364			21	SP
2	24	G013		SL RG SFT SLV ARP 5/8 228									374				
2	24	G014		PIN DRIVE 1/8X5/16LG			JR0004				10		23				
2	24	G015		SEAL COLLAR SHAFT SLV			A61969						21				
12	144	G016		SCR SOC SET CUP PT 3/8 16NCX3/8LG			HR0061			60	6		23				
6	72	G017		SCR SOC SET HF DOG PT1/4 20NCX1/4LG			HD0042				4		23				
2	24	G018		SP 20 KEY SFT SLV			A22822			1			23				SP

PACIFIC PUMPS, INC.

Page 3 of 3

## PARTS LIST

FOR WESTINGHOUSE ELECTRIC CORP.

OUR ORDER NO. J59218

PCB UNIT	TOTAL REQ'D	PART NO.	DESCRIPTION	PATTERN NO.	DRAWING NO.	LINE NO.	COL LINE NO.	MATERIAL NO.	WGT TOLAT	PROG
2	24	6019	SCR FLAT HD MACH SP 20 4 40NCX3/BLG	B				23		P
	1	2	B55 W/REV D CPLG ZURN 102 1/2	B						
	1	2	B54 CPLG ZURN 102 1/2 BALANCED	B						
1	2	2	B5 3 CPLG ZURN 102 1/2 BALANCED	D						
	1	47	BASE	B	150J84	92	18	78		
		366	HOLDER UNION	A	B18856	99	1			
3	6	695	NIPPLE TBE 1/2X2LG SCH80	A				308		
3	6	806	WSHR STOP COLL GRIZZLY 3/16X1/2X1/4	A	EXX007	10		229		
3	6	981	SPRING CENTURY STOCK 384	A	A31528	1		342		
3	6	992	FERRULE 3/16 IMPERIAL NO 60 F B	A	EXX007	14		210		
6	12	993	FLAT WSHR 10 32 BOLT SIZE	A	EXX007	12		210		
	6	1020	O RG ARPS68008	A	L05680		8	211		
3	6	1214	HD4 TERMINAL 5 C15666 FBG2	A						
1	2	1248	ELEMENT SHEATH 5LG	A						
2	4	1248	ELEMENT SHEATH 8 LG	A						

#### 4.0 Methodology

- 4.1 The evaluation/verification was performed by first reviewing the complete pump assembly inclusive of the bill of materials (Reference 1). Each part was then evaluated as to its specific function with respect to the function of the overall assembly. The parts were then categorized as critical or noncritical.
- 4.2 Critical parts are defined as those parts whose integrity, when exposed to the postulated environments, are critical to the overall component operation and whose failure would preclude the equipment from performing its intended safety function. The critical parts are identified in Table 2.
- 4.3 Subsequent to the above, the critical parts were then individually evaluated with respect to their ability to withstand the postulated environmental/operating conditions.

#### 5.0 Acceptance Criteria

- 5.1 The acceptance criteria is defined as follows:

The critical parts shall be technically shown to be able to withstand the postulated environmental conditions without exhibiting common mode failure effects.

TABLE 2  
CRITICAL PARTS LIST

<u>Part No.</u>	<u>Quantity</u>	<u>Description</u>	<u>Material</u>
A551	13	Gaskets/Parting Flange Gasket	Compressed Asbestos Sheet
A612	24	Flexitallic Gasket	Stainless Steel and Asbestos
B010	26	Seal Housing Gaskets	EPDM
D095	130	Suction Spacer O-Ring and Suction Cover O-Ring	EPDM
G006	26	Gasket O-Ring and Seal Plate O-Ring	EPDM
G013	24	Seal Ring-Shaft Sleeve	EPDM
G002	26	Mechanical Seal	EPR Graphite
F060	12	Worm Gear	Canvas Micarta
(Heat Exchanger Items 4 & 5)	2	Gaskets	Compressed Asbestos



## 6.0 Critical Parts Material Discussion

### 6.1 Ethylene Propylene Rubber (EPR)

(Cranelast) and graphite safety bushings

- 6.1.1 EPR and graphite are specifically used for the bellows and bushings respectively in the pump seals Part G002. While previous industry tests were performed on these materials under static conditions. Additional tests were required under dynamic conditions. Tests were performed on the complete seal assembly (Reference 2 ) which simulated the seal environment and operating conditions. The testing encompassed the following conditions inclusive of exposure to boric acid moderator solutions:

Pressure - 0.2 - 400 psig

Temperature - 140 - 300°F

Test Duration - 50 - 500 hrs.

Shaft Speeds - 1800 and 3600 RPM

Radiation -  $1.1 \times 10^8$  rads gamma

- 6.1.2 The tests concluded that the seal assembly which included the organic parts successfully performed their function under both normal and adverse conditions.

- 6.1.3 Graphite is considered not to be effected by thermal aging. It's radiation threshold, however, is  $1 \times 10^{10}$  rads gamma.



## 6.2 Asbestos/Compressed Asbestos

- 6.2.1 This material is used in parts A551 and A612 for gasketing applications. Based on common industry data, asbestos is acceptable for use in elevated temperature applications up to 150°C. This material is not considered as organic and, therefore, not subject to heat aging. Asbestos retains its properties and is usable for radiation doses in excess of  $10^{10}$  Rads. When this material is used in contact with coolant fluid and/or stainless steels, the chloride/flouride content is controlled to not exceed 200 PPM. This eliminates any possible stress corrosion effects.

## 6.3 Ethylene Propylene Diene Momer Rubber (EPDM)

- 6.3.1 EPDM is used in various parts of the pump assembly in gasketing and/or seal applications. (Parts B010, D095, G006, G013). This material has excellent thermal aging and radiation restance at temperature in excess of 300°F and  $2 \times 10^8$  rads gamma respectively.
- 6.3.2 Additional full sequence test data has been made recently available through the W Environmental Qualification of ASCO Solenoid Valves (Ref. 3). This additional data further reinforces the adequacy of EPDM for the above listed parts.

#### 6.4 Canvas Micarta

- 6.4.1 This material is specifically used for the gear pump worm gear Part F060. The property of importance is flexural strength. Using standard acceptance criteria of 50% retention, the data indicates acceptable performance up to 230°F.
- 6.4.2 The most radiation sensitive properties are elongation and impact strength which are not considered to be primary properties for this application. However, these properties, used for comparison purposes, are only reduced by 25% in the dose range of  $8 \times 10^6$  Rads.

## 7.0 Pump Motor

- 7.1 The W motor used on this pump application has been qualified by separate W testing. The test specifics are reported in Reference 4.

## 8.0 Lubricants

- 8.1 Various lubricants are acceptable for use under the operating conditions specified in the Pump Instruction and Maintenance Manual. It is the responsibility of the user to provide documentation for the lubricant used and maintain the equipment as recommended in the Instruction and Maintenance Manual. (Reference 1)

## 9.0 Maintenance

- 9.1 Maintenance shall be considered to be the responsibility of the user. The user shall establish a maintenance program to maintain the equipment as described in Reference 1. Periodic maintenance to maintain qualification is required on critical parts per the qualified life as stated in Table 3.

## 10.0 Conclusion

- 10.1 As stated in Section 5.0, the environments postulated for the critical parts were compared with the specific material capabilities. Table 3 summarized this comparison. Based on this evaluation and comparison, the Pacific Model 3" 10 stage JHF safety injection pump is considered environmentally qualified to perform its intended safety function of Para. 1.6 without common mode failure effects.
- 10.2 The qualified life of this equipment is 40 years provided the recommended maintenance is performed per Reference 1 and the critical parts identified on Table 3 are replaced at the end of their individual qualified lives.

TABLE 3  
CRITICAL PARTS - ENVIRONMENTAL QUALIFICATION SUMMARY

Material	Postulated Environmental Conditions (Worst Case)	Material Capabilities	Property Evaluated	Activation Energy (eV)	Qualified Life (Yrs)
Asbestos	120°F Normal 300°F Accident $1.2 \times 10^7$ Rads Gamma (1)	300°F $10^{10}$ Rads Gamma	(2)	N/A	> 40
EPR (Seal)	120°F Normal 300°F Accident $1.2 \times 10^7$ Rads Gamma (1)	300°F $1.1 \times 10^8$ Rads Gamma	60% Elongation Retention	0.93	1 (4)
Graphite	120°F Normal 300°F Accident $1.2 \times 10^7$ Rads Gamma (1)	> 300°F $1 \times 10^{10}$ Rads	Wear (2)	N/A	1 (4)
Canvas Micarta	140°F (3) $4 \times 10^6$ Rads Gamma (1)	230°F $> 8 \times 10^6$ Rads Gamma	50% Retention of Flexural Strength	2.10	> 8
EPDM (5)	120°F Normal 300°F Accident $1.2 \times 10^7$ Rads Gamma (1)	> 300°F $2 \times 10^8$ Rads Gamma	Retention of Initial Elongation	0.94	8

Notes: 1. Includes normal plus accident dose (accident duration is one (1) year).

2. Considered to be unaffected by thermal aging.

3. Environment is governed by maximum allowable lubricant temperature.

4. Based on continuous operation with maintenance.

5. Can also be used as an alternative seal material in place of EPR.

REFERENCES

1. Pacific Pump Instruction and Maintenance Manual for Commonwealth Edison 3" 10 Stage JHF Safety Model Injection Pumps.
2. "Seal Performance Testing for Nuclear Power Plant Safety Injection Systems," by Crane Packing Company. Bulletin #3472 (W Proprietary).
3. Cesarski, W.V. "Equipment Qualification Test Report ASCO Solenoid Valves" WCAP 8687, Supplement 2 - H02A (Proprietary) June 1981.
4. Anderson, A.A., "Equipment Qualification Test Report for the Westinghouse Large Pump Motor," WCAP-8687, Supplement 2-AE-2A (Proprietary).
5. W Corporate Thermal-Radiation Materials Application Data Manual.

ATTACHMENT B2

AIR OPERATED VALVE 8028  
(PVORT/NSSS)

Responses to the audit findings are as follows:

Open Item 1 - During the audit the final static deflection test report prepared by the vendor for this valve had not been approved by Westinghouse. As noted in Attachment A20, Item 3, this report has been reviewed.

Open Item 2 - As discussed in the FSAR, operability testing is done on a representative number of valves. Each valve tested covers a series of valves and is selected based on size, type, weight, vendor and other pertinent valve characteristics. The valve selected represents the worst set of parameters for the particular series of valves being grouped together. As noted in the approval of the test report for this valve (see Attachment A20, Item 1), a number of valves were covered by the one report. Additionally, the enclosed letter from the vendor (ITT Grinnell) documents the applicability of the test report to this valve.

Open Item 3 - This item refers to the air supply used to actuate valve 8028. During operability testing a pressure of 72 psig was applied to the air operator. The specified pressure range of the valve is 63 to 85 psig. The 72 psig used during the testing is adequate to demonstrate operability of the valve for the following reasons:

- The valve is designed to fail closed (safe position) in the event of low or a complete loss of air pressure. Hence, if the valve did not remain open at the lower end of the actuator design pressure, no safety problem would be encountered.
- The air pressure applied to the operator is used to hold the valve in the open position. Thus, if the valve functioned at 72 psig, it would also remain open at higher pressures in its design range.



- The air line to the valve is equipped with regulators to limit the actuator pressure to 85 psig.
- The valve functional specifications require closure within 10 seconds. During the operability tests the valve closed in the 2-3 second range at 72 psig. An increase in the actuator air pressure by 20% (i.e., to 85 psig) would have very little effect on the closing time and the closing forces on the valve.

Based on the above discussion, operability testing with the valve actuator at 72 psig adequately demonstrates the functional capability of this valve and testing at the maximum and minimum actuator air pressure limits is not considered necessary.

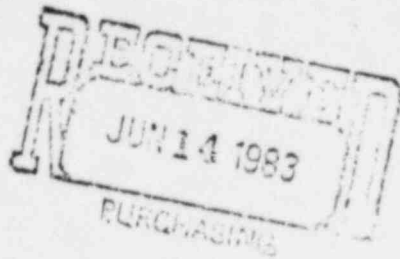
Open Item 4 - A static analysis using a finite element model was performed by the valve vendor (ITT Grinnell) to calculate the natural frequency of the valve and extended structure. In performing tests on similar valves for other organizations, Grinnell determined that the horizontal natural frequencies of this valve were below 33 Hz. Hence, the natural frequency analysis reviewed by the audit team has been invalidated. Grinnell has performed additional static fragility testing on this valve and determined that the valve's operability is not impaired for accelerations as high as 10g's. In light of this information Westinghouse has reanalyzed piping systems containing similar valves in the Byron plant using flexible valve models. The results of this reanalysis indicate that both the valve and piping system loads are acceptable. The information described above is currently in draft form with final reports in the process of being completed. Upon issuance of these reports and the finalization of piping analyses SQRT and PVORT Forms will be revised and submitted to the NRC.

Open Item 5 - The impact of dynamic fluid loads on operability testing is described in Attachment 38 Item 6. For the valve in question fluid dynamic effects are negligible.



Open Item 6 - The approach implemented by Commonwealth Edison to address the environmental qualification of mechanical equipment is discussed in response to draft SER Item 3. In order to demonstrate the acceptability of this approach Westinghouse has prepared information on the critical soft parts of the safety injection pump as agreed to with the Staff at a May 13, 1983 meeting. Please refer to Attachment B1, Item 4, and Attachment B8 Item 3.

ITT



*ITT Grinnell Corporation*

*DIA-FLO Division*

*33 Centerville Road*

*Lancaster, Pennsylvania 17603*

*(717) 291-1901*

*Telex: 84-8420*

June 8, 1983

Westinghouse Purchase Order 191000  
Valve Operability Test Certification

This is to certify that ITT Grinnell Operability Test Report W-210  
qualifies valve I.D.'s as listed below:

3/8DA42R, 3/8DA92R, 3/4DA32R, 3/4DA42R, 3/4DA62R, 3/4DA92R, 1DA32R,  
1DA42R, 1DA42RZ, 1DA92R, 2DA42D, 2DA92D, 2DA32R, 2DA42R, 2DA62RZ,  
2DA92R, 3DA42D, 3DA32R, 3DA42R, 3DA62RZ, 3DA92R, 4DA42D, 4DA32R,  
4DA42R, 4DA92R, 3/4X32D, 3/4X42D, 3/4X92D, 1X32D, 1X42D, 1X92D,  
2X32D, 2X42D, 2X92D, 3X32D, 3X42D, 3X62D, 3X92D, 4X32D, 4X42D, 4X92D.

Michael J. Panciera  
Sen. Prod. Eng.

MJP/twa

Attachment B3  
Gate Valve - Main Steam Isolation  
(PVORT-BOP)

Response to the audit findings are as follows:

- a) The maximum pressure in the accumulators will be 5000 psi under accident conditions. As a result the design pressure used was 5000 psi rather than the 3750 psi normal pressure. The accumulator was supplied as part of the MSIV operator and as such, was not subject to a separate test program by Anchor/Darling or Commonwealth Edison. However, the accumulator was purchased as an ASME Section VIII component. As a result, the vendor (Parker-Hannafin, holder of ASME "U" Stamp) certifies that the accumulator met or exceeded code testing requirements. This requirement is a test pressure of 1.5 x design pressure or, in this case, a minimum of 7500 psi. Documentation of subcomponent design and testing is retained by the valve supplier, Anchor/Darling.

REFERENCE: F/L 2756L MS-Isolation Valve Specification

- b) The original external environmental temperatures and pressures specified were overly conservative. FSAR Section C3.6 summarizes the calculation of the valve room and steam tunnel pressurization. The calculated peak pressure is less than 20 psig. A calculation to extend the transient shows that the pressure in the valve rooms is very close to ambient within five seconds. The transient is very rapid with the peak occurring less than 1.0 second after the break. The sharp pressure rise will force open doors and ventilation areas in the valve house and rapidly vent the steam to the environment. This ventilation also serves to rapidly reduce the temperature in the valve house and steam tunnel. The temperatures used in the environmental condition specification were very conservatively calculated. A more recent calculation shows that the temperature will drop to below 150°F in less than 15 minutes after an exterior door opens.

The concern about valve operability under elevated pressures results from a design limitation of the actuator. The air supply must be 59 psig above the environmental pressure. The supplied pressure at Byron will be in a range of 80-100 psig. Since the calculated maximum pressure is actually less than 20 psig, the required minimum supply pressure would be less than 79 psig. Therefore, a small margin is available even if a design basis double ended break occurs while the supplied air pressure is at its minimum value, and the closure signal is received at the valve during the short (1-2 seconds) time that the environmental pressure is elevated.

Gate Valve - Main Steam Isolation  
(PVORT-BOP)

- c) Assurance of the ability of the valve to function under full flow conditions is achieved by a combination of analysis and test. The vendor provides assurance that the closure time requirements are met by calculating the force necessary to close the valve against a full flow load and then testing the actuator to insure that adequate closing force was available. Documentation of this testing is retained by the valve manufacturer. Because it is not practical to fully stroke these valves during normal operation, the valves will be part stroke exercised every three months and fully stroked during shutdown. This is documented on Page 21 of the Byron Inservice Testing Program Plan for Valves. A description of this program was submitted to the NRC in the November 4, 1982 letter from T.R. Tramm to H.R. Denton.
- d) The actuator used for seismic qualification was identical to the actuators installed at Byron. The actuator, Model 64324-C, was qualified by analysis. Wyle Lab report number 43847-2 lists this model number which is identical to the model number of the installed actuator.
- e) The actuator was vibrationally tested (Wyle Lab report 43847-2) and this report is available. The valve itself was qualified by analysis and, as a result no vibration testing data is available. The analysis establishes that the valve is not sensitive to vibration predicted in seismic events. The Byron vibration monitoring program detects any unusually high amplitude vibration during pre-operational testing.

Aging of the Class 1E electrical equipment is considered in the qualification program for 1E equipment in a harsh environment. For tested equipment, the sequential test requirements of IEEE 323-1974 have been complied with. Aging of mechanical components is addressed by a program of maintenance and surveillance which will detect age related degradation of mechanical components. This program is further discussed in SQRT Draft SER Generic Item #6.

- f) A loss of air in the accumulators would lead to inability to close the valve. However, the loss of air would have to occur in both accumulators to actually cause the valve to fail to close. The accumulators are independent, and are subject to surveillance. As noted previously, the accumulators are ASME code designed and inspected components. Failure because of accumulator leakage would require an initiating event plus two independent failures in a safety grade system. This scenario is beyond the required plant design basis.

Attachment B4  
Main Steam Safety Valve (1MS013A)  
(PVORT-BOP)

Responses to the open audit items are as follows:

- a) The leakage rate of the Main Steam Relief Valves, while an important consideration for plant operating efficiency, is not a safety concern. The secondary side of the steam supply system is not normally radioactive. Leakage will cause an offsite release only if the primary to secondary side boundary is breached. The analysis in FSAR Section 15.6.3 for steam generator tube rupture assumes a loss of offsite power and control of steam generator pressure by opening of steam generator relief valves. This clearly conservatively bounds the effect of a leaking valve.
- b) The Byron Preservice and Inservice Testing Program Plan for Valves includes valve 1MS013A (page 21 of the Program Plan). Prior to startup and at each refueling outage, the valve setpoint will be verified in accordance with IWV-3510 of ASME Section XI.
- c) The test report for the Main Steam Safety Valve 1MS013A (EMD 003901) is divided into three parts. Phase 1 is a determination of the resonant frequency. Phase 2 is a determination of the effects of vibration and resonance on the valve operation. Phase 3 is the actual test of valve operability under vibrational loading. Phases 1 and 2 can be characterized as developmental testing.

The valve has a lowest horizontal natural frequency of 37 Hz. When tested with a high amplitude 30 Hz input some operability problems were noted in the Phase 2 testing. Some modifications were made to the valve but the object was not to completely eliminate the resonance problems associated with the 30 Hz input.

Phase 3 is the actual qualification testing of the valve. The prototype as tested in Phase 3 is the same as the production valve installed by Byron with the exception of the differences noted in Section 4.2.1 of the report.

The results of the testing indicate that the valve will perform as required.

- d) Establishment of a qualified life as a result of aging is not required for these valves. These valves are all metal and do not have any critical non-metallic parts subject to aging. A maintenance and surveillance program has been established to schedule repair or replacement of the valve or valve parts based on manufacturers recommendations and operating experience. The surveillance program for these valves is described in the response to finding (b).



Attachment B5  
Essential Service Water Pump (1SX01PA)  
(PVORT-BOP)

Responses to the open audit items are as follows:

- a) The minimum critical speed has been determined by the pump manufacturer to be 2611 rpm. This is approximately three times the normal operating speed of 880 rpm. Therefore, the critical speed has no effect on the operation of the pump.
- b) A revised PVORT from has been prepared and is attached.
- c) Attached is a section of the preoperational test procedure for the subject pump. Section 9.20 is the procedure for the Essential Service Water Pump 1A Performance Test. In addition, this pump is included in the Byron Preservice Inspection Testing Program Plan for Pumps. (Page 2 of Program Plan for Pumps, T.R. Tramm letter to H.R. Denton dated November 4, 1982) This program has been developed in accordance with ASME Sec. XI requirements and includes provisions for monitoring pump vibration, flowrate, discharge pressure, and bearing temperature. Although final preoperational test results are not available, these test plans show that adequate pump performance will be demonstrated prior to plant operation.
- d) The essential service water pumps are normally operating components located in a mild environment. As a result, the aging is only due to normal operation. This is addressed by the maintenance and surveillance programs which have been established at Byron. The maintenance program is described in the response to SQR draft SER generic item 6. The surveillance program is described in the response to item (c) above.
- e) The pump is qualified by analysis (EMD-013374). In the dynamic model, the bed plate is assumed to be supported only at the foundation bolts. This is true for upward forces but for downward forces the bed plate channel flanges are continuously supported. This conservative assumption was made so that the idealized model would be more flexible than the actual assembly, thus leading to conservative stress and deflection calculations.

In the qualification document, a seismic analysis for the coupling (see 6.2 of McDonald Engineering Analysis Company Report No. ME-523) is presented for functional capability of the pump.

PUMP AND VALVE  
OPERABILITY ASSURANCE REVIEW

I. PLANT INFORMATION

1. Name: Byron/Braidwood Unit No. 1&2 2. Docket No.: 50-454, 50-455  
50-456, 50-457
3. Utility: Commonwealth Edison Company
4. NSSS: Westinghouse Electric ☒ PWR ☐ BWR
5. A/E: Sargent & Lundy

II. GENERAL COMPONENT\* INFORMATION

1. Supplier: ☐ NSSS ☒ BOP
2. Location: a. Building/Room Auxiliary  
b. Elevation 330' (15-N)  
c. System Essential Service Water

3. Component number on in-house drawings: ISX01PA

4. If component is a ☒ Pump complete II.5.

If component is a ☐ Valve complete II.6.

5. General Pump Data

	a. Pump	b. Prime-mover
Name	<u>ISX01PA</u>	<u>ISX01-PA-M</u>
Mfg.	<u>Bingham-Willamette</u>	<u>Westinghouse</u>
R   Model	<u>Suction Double Volute</u>	<u>HHS-DPO</u>
S/N	<u>16210001</u>	<u>1013BA-01</u>
Type	<u>HSA</u>	<u>Frame 8009S55</u>

\* The component, whether pump or valve, is considered to be an assembly composed of the body, internals, prime-mover (or actuator) and functional accessories.

a. Pump (continued)

Size 24 x 30 x 30

Weight 14,200 lbs.

Mounting Method Bolted

Required B.H.P. 1247

Parameter	Design	Operating
Press	<u>125</u>	<u>100</u>
Temp	<u>100</u>	<u>100</u>
Flow	<u>24,000</u>	<u>24,000</u>
Head	<u>180</u>	<u>180</u>

Required NPSH at maximum

flow 46'

Available NPSH 40'

Operating Speed 800 rpm

R | Critical Speed 2611 rpm

List functional accessories:\*

b Prime-mover (continued)

Size 1250 HP

Weight 12,000 lbs.

Mounting Method Bolted

H.P. 1250

Power requirements: (include normal, maximum and minimum).

Electrical 4000 VAC

60 Hz, 3 phase

1250 Hp

Other 80% Minimum starting voltage

If MOTOR list:

Duty cycle Continuous

Stall current 520% full load

Class of insulation B

Control switch @ 1PM06J and 1PL04J.

Lube oil pump and motor space heater.

List control signal inputs: Interlocks with 1SX027A, 1PS-SX139,  
lube oil pressure, 1SX016A, 1SX001A, 1PSL-SX023 ESW pump low suction  
pressure.

\* Functional accessories are those sub-components not supplied by the manufacturer that are required to make the pump assembly operational, (e.g., coupling, lubricating oil system, etc.)



# 6. General Valve Data

## a. Valve

Name \_\_\_\_\_

Mfg. \_\_\_\_\_

Model \_\_\_\_\_

S/N \_\_\_\_\_

Type \_\_\_\_\_

Size \_\_\_\_\_

Weight \_\_\_\_\_

Mounting  
Method \_\_\_\_\_

Required  
Torque \_\_\_\_\_

Parameter	Design	Operating
Press	_____	_____
Temp	_____	_____
Flow	_____	_____
Max $\Delta P$ across valve	_____	_____
Closing time @ max $\Delta P$	_____	_____
Opening time @ max $\Delta P$	_____	_____
Power requirements for functional accessories, (if any)	_____	

List control signal inputs: \_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

## b. Actuator (if not an integral unit)

Name \_\_\_\_\_

Mfg. \_\_\_\_\_

Model \_\_\_\_\_

S/N \_\_\_\_\_

Type \_\_\_\_\_

Size \_\_\_\_\_

Weight \_\_\_\_\_

Mounting  
Method \_\_\_\_\_

Torque \_\_\_\_\_

Power requirements: (include normal, maximum and minimum):

Electrical \_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

Other: ☐ Pneumatic ☐ Hydraulic

\_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

\_\_\_\_\_

List functional accessories: \* \_\_\_\_\_

### III. FUNCTION

1. Briefly describe components normal and safety functions: \_\_\_\_\_  
Transfers heat loads generated by safety-related equipment, or equipment essential to safe shutdown of the reactor to the ultimate heat sink.

2. The components normal state is: ☒ Operating ☐ Standby

3. Safety function:

a. ☒ Emergency reactor shutdown

b. ☒ Containment heat removal

c. ☐ Containment isolation

d. ☐ Reactor heat removal

e. ☐ Reactor core cooling

f. ☐ Prevent significant release of radioactive material to environment

g. ☐ Does the component function to mitigate the consequences of one or more of the following events? ☒ Yes ☐ No  
If "Yes", identify.

☒ LOCA

☐ HELB

☐ MSLB

☐ Other \_\_\_\_\_

4. Safety requirements:

☐ Intermittent Operation

☐ During postulated event

☒ Continuous Operation

☐ Following postulated event

If component operation is required following an event, give approximate length of time component must remain operational.

R | \_\_\_\_\_ throughout duration of accident (e.g., hours, days, etc.)

\* Functional accessories are those sub-components not supplied by the manufacturer that are required to make the valve assembly operational, (e.g., limit switches).

5. For VALVES:

does the component ☐ Fail open ☐ Fail closed ☐ Fail as is

Is this the fail safe position? ☐ Yes ☐ No

Is the valve used for throttling purposes? ☐ Yes ☐ No

Is the valve part of the reactor coolant pressure boundary?  
☐ Yes ☐ No

Does the valve have a specific limit for leakage? ☐ Yes ☐ No

If "Yes" give limit: \_\_\_\_\_

IV. QUALIFICATION

1. Reference by specific number those applicable sections of the design codes and standards applicable to the component: \_\_\_\_\_

ASME Section III 1974 Ed. through 5 '76 ADD.

2. Reference those qualification standards, used as a guide to qualify the component: IEEE 323 - 1974 (Motor only)

IEEE 344 - 1975

3. Identify those parts of the above qualification standards deleted or modified in the qualification program.

Deleted:

Modified:

4. Have acceptance criterias been established and documented in the test plan(s) for the component? ☒ Yes ☐ No

5. What is the expected failure mode that would keep the pump or valve assembly from performing its safety function? Pump trip,  
loss of power to motor, loss of pump suction.

6. Are the margins\* identified in the qualification documentation?  
☒ Yes ☐ No

d. Margin is the difference between design basis parameters and the test parameters used for equipment qualification.

If component is a PUMP, complete IV.7.

If component is a VALVE, complete IV.8.

7. Pump operability has been demonstrated by: ☐ Analysis

☒ Test ☐ Combination

Identify PUMP tests performed:

- |   |  |
|---|--|
| a. <input checked="" type="checkbox"/> Shell hydrostatic<br>(ASME Section III)            | b. <input type="checkbox"/> Bearing temperature<br>evaluations |
| c. <input type="checkbox"/> Seismic loading   | d. <input type="checkbox"/> Vibration levels                   |
| e. <input checked="" type="checkbox"/> Exploratory vibration<br>(Fundamental freq. _____) | f. <input type="checkbox"/> Seal leakage @ hydro press         |

- |   |   |
|---|---|
| g. <input checked="" type="checkbox"/> Aging: <input checked="" type="checkbox"/> Thermal<br>(Motor only) <input type="checkbox"/> Mechanical | h. <input checked="" type="checkbox"/> Flow performance<br>Are curves provided <input checked="" type="checkbox"/> Yes<br><input type="checkbox"/> No |
|---|---|

- |  |  |
|--|--|
| i. <input checked="" type="checkbox"/> Pipe reaction end<br>loads (nozzle loads) | j. <input type="checkbox"/> Others _____ |
|--|--|

- k. ☒ Extreme environment:

Motor only ☒ Humidity

☐ Chemical

☒ Radiation

8. Valve operability has been demonstrated by: ☐ Analysis

☐ Test ☐ Combination

Identify VALVE tests performed:

- |  |   |
|--|---|
| a. <input type="checkbox"/> Shell hydrostatic<br>(ASME Section III)            | b. <input type="checkbox"/> Cold cyclic List times:<br>Open _____<br>Closed _____ |
| c. <input type="checkbox"/> Seismic loading                                    | d. <input type="checkbox"/> Hot cyclic List times:<br>Open _____<br>Closed _____  |
| e. <input type="checkbox"/> Exploratory vibration<br>(Fundamental freq. _____) | f. <input type="checkbox"/> Main seat leakage                                     |

- g. ☐ Aging: ☐ Thermal ☐ Back seat leakage  
☐ Mechanical
- i. ☐ Pipe reaction end j. ☐ Disc hydrostatic  
loading
- k. ☐ Extreme environment l. ☐ Flow interruption capability  
☐ Humidity  
☐ Chemical  
☐ Radiation
- m. ☐ Flow characteristics n. ☐ Others \_\_\_\_\_  
Are curves provided? \_\_\_\_\_  
☐ Yes ☐ No \_\_\_\_\_

9. As a result of any of the tests (or analysis), were any deviations from design requirements identified? ☐ Yes ☒ No  
If "Yes", briefly describe any changes made in tests (or analysis) or to the component to correct the deviation.  
\_\_\_\_\_  
\_\_\_\_\_  
\_\_\_\_\_

- R. 10. Was the test component precisely identical (as to model, size, etc.) to the in-plant component? ☒ Yes\* ☐ No If "No", is installed component ☐ oversized or ☐ undersized?  
\*for pump tests only, motor qualified by generic test
11. If type test was used to qualify the component, does the type test meet the requirements of IEEE 323-1974, Section 5.? ☒ Yes ☐ No Motor only
12. Is component orientation sensitive? ☐ Yes ☐ No ☒ Unknown  
If "Yes", does installed orientation coincide with test orientation? ☐ Yes ☐ No However, installed orientation concides with test orientation
13. Is the component mounted in the same manner in-plant as it was during testing (i.e., welded, same number and size bolts, etc.) ☒ Yes ☐ No ☐ Unknown  
Motor only.

R

14. Were the qualification tests performed in sequence and on only one component? ☒ Yes ☐ No Motor only

If "Yes" identify sequence, (e.g., radiation, seismic, cyclic, thermal, etc.): thermal, mechanical cycling, seismic

15. If "aging"\* was performed, identify the significant aging mechanisms: \_\_\_\_\_

R

16. Identify loads imposed (assumed) on the component for the qualification tests (analysis) performed:

- a. ☐ Plants (shutdown loads)      b. ☐ Extreme environment  
c. ☒ Seismic load                      d. ☐ Others \_\_\_\_\_

R

17. Have component design specifications been reviewed in-house to assure they envelope all expected operating, transient, and accident conditions? ☒ Yes ☐ No

R

18. Does the component utilize any unique or special materials? (Examples are special gaskets or packing, limitations on nonferrous materials, or special coatings or surfaces.)  
☐ Yes ☒ No

If "Yes", identify: \_\_\_\_\_

R

19. Does component require any special maintenance procedures or practices, (including shorter periods between maintenance).  
☐ Yes ☐ No

If "Yes", identify: The standard plant maintenance program will address all required maintenance considerations for this component.

R

20. Is the qualified life for the component less than 40 years?  
☐ Yes ☒ No If "Yes", what is the qualified life? \_\_\_\_\_

\* As outlined in Section 4.4.1 of IEEE-627 1990.



21. Information Concerning Qualification Documents for the Component1SX01PA

Report Number	Report Title	Date	Company/Organization Preparing Report	Company/Organization Reviewing Report
FL-2758A	Miscellaneous Pumps (Safety Category I)	5-4-77	S&L	S&L/CECo
FD-162 10001/ 2 & 5/6	Essential Service Water Pump	4-12-78	Bingham Pump Co	S&L/CECo
NPV-1	Manufacturer's Data Report	8-31-78	Bingham - Willamette	S&L/CECo
16210001	Performance Test	4-13-78	Bingham - Willamette	S&L/CECo
1013EA	Seismic Analysis for SX Pump Motors	7-31-78	Westinghouse	S&L/CECo
ME-523	Seismic - Stress Analysis of Horizontal Pumps	3-23-78	Bingham - Willamette	S&L/CECo
2.7.6. 10	Essential Service Water Pre-Operational Test		CECo	S&L/CECo

#### 4.0 - ACCEPTANCE CRITERIA

- 4.1 Each Essential Service Water pump shall have a minimum capacity of 24,000 gal./min. at a discharge head of  $180 \pm 10\%$  feet.
- 4.2 Each Essential Service Water Diesel Make-up Pump shall have a minimum capacity of 1500 gal./min. at a discharge head of  $360 \pm 10\%$  feet.
- 4.3 Each Essential Service Water pump shall provide a minimum supply of 5320 gal./min. to the RCFC units.
- 4.4 Each Essential Service Water Pump shall provide a minimum supply of 16,000 gal./min. to either the Unit 1 or Unit 0 Component Cooling Heat Exchanger.
- 4.5 Each Essential Service Water Pump shall provide a minimum supply of 105 gal./min. to each ESW cubicle cooler (1A and 1B).

## 9.0 TEST PROCEDURE (Continued)

### 9.20 Essential Service Water Pump 1A Performance Test

NOTE The following section will test the performance of the 1A SX Pump. The OA SX cooling tower basin must be filled to a normal level. The strainer must be clean.

- 9.20.1 Verify OPEN suction valve 1SX001A using handswitch  
1HS-SX017 on 1PM06J. \_\_\_\_\_/\_\_\_\_\_
- 9.20.2 Verify OPEN RCFC inlet valve 1SX016A using handswitch  
1HS-SX106 on 1PM06J. \_\_\_\_\_/\_\_\_\_\_
- 9.20.3 Verify OPEN RCFC outlet valve 1SX027A using handswitch  
1HS-SX107 on 1PM06J. \_\_\_\_\_/\_\_\_\_\_
- 9.20.4 Verify alarm window 12D01 "ESS SW PUMP SUCTION  
PRESS LOW" is CLEAR Pump 1A. \_\_\_\_\_/\_\_\_\_\_
- 9.20.5 Verify pump casing is filled and vented. \_\_\_\_\_/\_\_\_\_\_
- 9.20.6 Verify normal oil levels in sight glasses. \_\_\_\_\_/\_\_\_\_\_
- 9.20.7 Verify cooling water is lined up. \_\_\_\_\_/\_\_\_\_\_
- 9.20.8 Position discharge valve 1SX143A to approximately  
10% open. \_\_\_\_\_/\_\_\_\_\_
- 9.20.9 Verify Racked in the breaker for SX Pump 1A. 4160 V  
Bus 141 CUB #2 (1SX01PA). \_\_\_\_\_/\_\_\_\_\_
- 9.20.10 Verify the breaker for Aux. Lube Oil Pump 1A  
is CLOSED.  
MCC 131 X1 Compt. J2 \_\_\_\_\_/\_\_\_\_\_
- 9.20.11 START the Aux. Lube Oil Pump 1A from the local  
handswitch. \_\_\_\_\_/\_\_\_\_\_

9.0 - TEST PROCEDURE (Continued)

9.20.12 START SX Pump 1A by placing handswitch LHS-SX001 to the CLOSE position until the main pump starts. As soon as the pump discharge pressure begins to rise, slowly throttle the discharge valve, 1SX143A, open approximately 40%. \_\_\_\_\_/\_\_\_\_\_

9.20.13 STOP the Aux. Lube Oil Pump 1A. \_\_\_\_\_/\_\_\_\_\_

NOTE: Allow the pump to reach normal operating temperatures.

9.20.14 Take pump data on Data Sheet 11.3. \_\_\_\_\_/\_\_\_\_\_

9.20.15 Take vibration signatures on the pump and motor. \_\_\_\_\_/\_\_\_\_\_

9.20.16 Using RCFC outlet valves 1SX021A, 1SX025A, 1SX021C and 1SX025C, balance the flow to the individual coils. Each individual flow must fall within  $\pm 25$  gpm of the average of the four flows. Record data at item 11.5.1. \_\_\_\_\_/\_\_\_\_\_

9.20.17 Verify ESW Diesel Makeup Pump OA is in PULL TO LOCK. \_\_\_\_\_/\_\_\_\_\_

9.20.18 Using blowdown, bring the Tower OA level down to the minimum operating level. Record tower level and record any vortexing observed near the suction duct. Monitor OLS-SX096. Record the level at which the switch closes.

Level \_\_\_\_\_

Observed vortexing \_\_\_\_\_

OLS-SX096 \_\_\_\_\_ (level)

9.0 - TEST PROCEDURE (Continued)

9.20.19 Increase the tower level using available makeup.

Establish a maximum operating level of approximately  
874'2" or 6'5" from the Basin Bottom. Record tower level  
and any observed vortexing.

Level \_\_\_\_\_

Observed Vortexing \_\_\_\_\_

\_\_\_\_\_.      / \_\_\_\_\_

# DATA SHEET 11.3

## PUMP DATA FOR SX PUMP 1A

DATA POINT	SUCTION PRESSURE (PSIG) 1PI-SX148	DISCHARGE PRESSURE (PSIG) 1PI-SX008	FLOW RATE (GPM) FLOW METER
1			
2			
3			
4			
5			
6*			

NAME

DATE

\*Data Point 6 will prove the capacity of the SX Pump per the Acceptance Criteria.



Attachment B6  
Containment Spray Pump (1CS01PA)  
(PVORT-BOP)

The responses to the open audit items are as follows:

Open Item 1 - The qualification documentation has been corrected to be consistent with the serial number of the installed "A" pump. The documentation has been approved and signed off.

Open Item 2 - Attached is a section of the Containment Spray System preoperational test procedure results for the subject pump and the corresponding acceptance criteria. The test results meet the acceptance criteria and demonstrate that the pump performs adequately within the system. In addition, this pump is included in the Byron Preservice/Inservice Testing Program Plan for Pumps (page 1 of Program Plan for Pumps, T.R. Tramm letter to H.R. Denton dated November 4, 1982). This program has been developed in accordance with ASME Sec. XI requirements and includes provisions for monitoring pump vibration, flowrate, and discharge pressure. The preservice and periodic inservice testing described above will demonstrate the operability of the pump for the life of the plant.

Open Item 3 - The containment spray pump and pump motor are located in a cubicle. They are exposed to a harsh environment only because of the radiation levels associated with the location adjacent to the containment and the accident mode of operation (handling radioactive containment sump fluid). The containment spray pumps are operated only during system tests and following a LOCA or Main Steam Line Break inside containment.

The containment spray pump motor has been environmentally and seismically qualified (Westinghouse Report WCAP-8754 Rev. 1 and Shop Order 77F14089) and has a qualified life of 40 years under Byron specific conditions. The pump has been seismically qualified (Ingersoll-Rand Report EAS-TR-7801-IR). Although a specific program for environmental qualification of mechanical equipment is not required, a review of the design of this pump has been performed to insure that it has been properly designed to function in the specified high radiation environment. A summary of this review is attached.

Containment Spray Pump (1CS01PA)  
(PVORT-BOP)

Open Item 4 - Seismic qualification of the containment spray pump was accomplished by analysis (S&L Report EMD-025038). The analysis was done using the faulted condition. The results of the analysis indicated that the lowest natural frequency was greater than 33 Hz (page 4, EMD-025038 attached) hence the equipment is considered rigid and qualified by static analysis. The containment spray pump is designated as seismic Cat. I, active equipment which means the pump must meet performance requirements during and following a safe shutdown earthquake. In the static analysis, the seismic forces on the pump assembly are obtained by concentrating the entire mass at the center of gravity of the assembly and multiplying it by the appropriate seismic accelerations. In this case the seismic accelerations for safe shutdown conditions are as follows:

Horizontal = 0.25g (in each of two orthogonal directions)

Vertical = 0.85g (page 3, EMD-025038 attached)

These acceleration values are used to produce the equivalent static forces for the analysis.

# APPROVED FOR TESTING

## 3.0 TEST OBJECTIVES

- 3.1 To demonstrate the ability of the Unit One components of the Containment Spray System to meet their design and operational requirements.
- 3.2 To insure proper operation of all equipment, controls, alarms and interlocks associated with the Containment Spray System.
- 3.3 To insure that the Containment Spray System will be capable of delivering a fluid solution at proper flows and pressures to the Containment Spray headers.
- 3.4 To verify the spray nozzles are unobstructed.

## ACCEPTANCE CRITERIA

- 4.1 Verify that the capacity of containment spray pump 1CS01PA is a minimum of 3545 gpm at a head of 450 feet.
- 4.2 Verify that the capacity of containment spray pump 1CS01PB is a minimum of 4055 gpm at a head of 450 feet.
- 4.3 Verify the ability to educt a minimum of 55 gpm indicated of fluid from the spray addition tank into the spray discharge flow using the installed flow instrumentation.
- 4.4 Verify all spray nozzles are unobstructed.

# APPROVED FOR TESTING

## TEST PROCEDURE (Continued)

### 9.19 Containment Spray Pump LCS01PA Integrated Testing

9.19.1 Verify the Blank Flange is installed at LCS04MA and the temporary jumper to the Fuel Handling Building rainwater header is installed. MMR 112-15-82

9.19.2 Verify that CS pump 1A and all loop A piping has been filled and vented. MMR 112-15-82

9.19.3 Vent the pump seal cavity by opening the seal vent valve LCS049A. MMR 112-15-82

9.19.4 Verify that the CS pump 1A motor oil reservoir has been filled to the proper level. MMR 112-15-82

9.19.5 Verify a minimum level of greater than 20% for the RWST. MMR 112-15-82

23  
22  
~~9.19.6 Install an ultrasonic flowmeter on line LCS12AA 3".~~

9.19.7 Verify a normal operating line-up for Containment Spray Pump 1A and Containment Spray Loop 1A from the RWST as per M-46 and M-61-4. MMR 112-15-82

9.19.8 Open test line discharge isolation valve LSI001A. MMR 112-15-82

9.19.9 Rack-in the breaker for CS pump 1A at Bus 141, Cub. 9. MMR 112-15-82

9.19.10 Close valve LCS040A. MMR 112-15-82

9.19.11 "OPEN" LCS010A by placing its handswitch at LPM06J in the "Test" position. MMR 112-15-82

9.19.12 Place the CS pump 1A Test Transfer Switch on MCB LPM06J in the "Test" position. MMR 112-15-82

9.19.13 Start CS pump 1A by going to close on MCB LPM06J. MMR 112-15-82

# APPROVED FOR TESTING

## 9.0 - TEST PROCEDURE (Continued)

9.19.14 Verify CS pump 1A operating parameters on  
Data Sheet 11-19, throttle valves 1CS007A  
and 1SI001A to obtain these flow rates.

MA R / 12-15-82

9.19.15 Verify the SAT is filled to nominally 55% with  
Grade "A" water.

MA R / 12-15-82

9.19.16 Open valve 1CS040A.

MA R / 12-15-82

9.19.17 Open valve 1MOV-CS019A at MCB 1PM06J.

MA R / 12-15-82

9.19.18 With CS pump 1A in operation at a flow rate  
of nominally 3545 gpm, adjust valve 1CS021A to yield  
approximately 58 gpm as indicated on 1F1-CS015.

MA R / 12-15-82

9.19.19 Record the following:

CS pump 1A Disch Flow Rate (1F1-CS011) 3500 gpm

(minimum = 3105 gpm) MA R / 12-15-82

CS eductor 1A Flow Rate (1F1-CS013) 135 gpm

(10% of 1F1-CS011 reading  $\pm 12$  gpm) MA R / 12-15-82

Spray Additive Flow Rate (1F1-CS015) 58 gpm

(58 gpm, -0, +2.25 gpm) MA R / 12-15-82

~~Sonic Flowmeter Spray Additive Flow Rate \_\_\_\_\_ gpm~~

~~(58 gpm, -0, +2.25 gpm)~~

Spray Additive Tank level 47 1/2 % MA R / 12-15-82  
(20-55%)

9.19.20 Lock valve 1CS021A in its present position.

MA R / 12-15-82

9.19.21 Close valve 1CS019A.

MA R / 12-15-82

9.19.22 Close valve 1CS007A.

MA R / 12-15-82

TCR #2  
MA R  
9-20-82

TCR #23  
MA R  
11-9-82

APPROVED  
FOR TESTING

TEST PROCEDURE (Continued)

9.19.23 Open valve 1CS019A and verify the CS pump 1A  
*mark 8-13-82*  
operating parameters on Data Sheet 11-19.

mark / 12-28-82

9.19.24 Close valve 1CS019A.

mark / 12-28-82

9.19.25 Verify CS pump 1A operating parameters on  
Data Sheet 11-19.

mark / 12-28-82

9.19.26 Trip CS pump 1A.

mark / 12-28-82

9.19.27 Close valve 1CS040A.

mark / 12-28-82

9.19.28 Close valve 1S1001A.

mark / 12-28-82



## Item #      Noun Name or Equivalent

## 11.19.14 Flow Rate 2000 gpm \*

- c Suction Pressure (1PI-CS003) ~~ft~~ psi *max*  
 c Discharge Pressure (1PI-CS004) ~~ft~~ psi *12-15-82*  
 c CS Pump 1A Disch Flow Rate (1FI-CS011) gpm  
 c CS Eductor 1A Flow Rate (1FI-CS013) gpm  
 Running Current (amps) (maximum = 75 amps)

2000  
 34 1/2  
 266  
 2000  
 145  
 52  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82

## Flow Rate 2800 gpm \*

- c Suction Pressure (1PI-CS003) ~~ft~~ psi *max*  
 c Discharge Pressure (1PI-CS004) ~~ft~~ psi *12-15-82*  
 c CS Pump 1A Disch Flow Rate (1FI-CS011) gpm  
 c CS Eductor 1A Flow Rate (1FI-CS013) gpm  
 Running Current (amps) (maximum = 75 amps)

2800  
 34  
 257  
 2800  
 140  
 59  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82

## Flow Rate 3545 gpm \*

- c Suction Pressure (1PI-CS003) ~~ft~~ psi *max*  
 c Discharge Pressure (1PI-CS004) ~~ft~~ psi *12-15-82*  
 c CS Pump 1A Disch Flow Rate (1FI-CS011) gpm  
 c CS Eductor 1A Flow Rate (1FI-CS013) gpm

## Vibration Data Taken

- Running Current (amps) (maximum = 75 amps)  
 TO 900 VS CS 1A MOT INBD BRGT (maximum = 19.5°F)  
 TO 901 VS CS 1A MOT OUTBD BRGT (maximum = 19.5°F)

3550  
 33 1/2  
 242  
 3550  
 135  
 405  
 64  
 152  
 62  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82  
*max* / 12-15-82

*max* / 12-15-82

APPROVED  
FOR TESTING

DATA SHEET 11-19

Item #	Nameplate/ Noun Name or Equivalent	TESTED	INITIALS/DATE
11.19.24	¢ Suction Press. (1PI-CS003) ft. psi <i>mag 12-15-82</i>	<del>retest</del> <i>mag 12-28-82</i>	
	¢ Discharge Press. (1PI-CS004) ft. psi <i>12-15-82</i>	<del>43.5</del> 40.5	<i>mag 12-28-82</i>
	¢ CS Pump 1A Disch Flow (1FI-CS011) gpm	<del>290</del> 28.5	<i>mag 12-28-82</i>
<i>TCR #29</i> <i>mag 12-11-82</i>	¢ Test Line Flow (1FI-SI006) gpm (or ultrasonic)	400	<i>mag 12-28-82</i>
	¢ Spray Additive Flow (1FI-CS015) gpm	42.5	<i>mag 12-28-82</i>
	¢ CS Eductor 1A Flow (1FI-CS013) gpm	5.5	<i>mag 12-28-82</i>
<i>TCR #23</i> <i>mag 11-4-82</i>	<del>¢ Sonic Flowmeter gpm on Spray Additive Line</del>	<del>14.5</del>	<del><i>mag 12-28-82</i></del>
	<i>mag 4-4-82</i>		
11.19.26	¢ Suction Press. (1PI-CS003) ft. psi <i>mag 12-15-82</i>	40.5	<i>mag 12-28-82</i>
	¢ Discharge Press. (1PI-CS004) ft. psi <i>12-15-82</i>	28.5	<i>mag 12-28-82</i>
	¢ CS Pump 1A Disch Flow (1FI-CS011) gpm	400	<i>mag 12-28-82</i>
<i>TCR #28</i> <i>mag 12-11-82</i>	¢ Test Line Flow (1FI-SI006) gpm (or ultrasonic)	42.5	<i>mag 12-28-82</i>
	¢ CS Eductor 1A Flow (1FI-CS013) gpm	14.5	<i>mag 12-28-82</i>
	Vibration Data Taken	yes	<i>mag 12-28-82</i>

*mag 12-28-82*

APPROVED  
FOR TESTING

**SARGENT LUNDY****ENGINEERS**  
CHICAGOCalcs. For ENVIRONMENTAL QUALIFICATION  
OF CONTAINMENT SPRAY PUMPS☒

Safety-Related

Non-Safety-Related

Calc. No. COD- 006151

Rev. 00 Date 2-25-83

Page ES of F15

Client COMMONWEALTH EDISON COMPANY

Project BYRON / BRAIDWOOD

Proj. No. 4391/92-00

4683/84-00

Equip. No. 1,2CS01PA,PB

Prepared by J. Chan

Date 2-25-83

Reviewed by J. Chan

Date 2-25-83

Approved by J. Chan

Date 2/25/83

Rev. 00  
Proj. No. 4391/92/4683/84-00  
Page CS of C15Purpose:

To justify the environmental qualification of the containment spray pumps for Byron/Braidwood Nuclear Power Station by Analysis of the Pumps' materials.

Spec.: F/L 2758-B

Location: Aux. Building, elevation: -346' for PA  
-343' for PB

Design: ASME Sec. III, class 2

Seismic class: Category 1

Environmental zone: A13C

Size & Model: Ingersoll-Rand Co's 8x23 WDF,  
Vertical Centrifugal

**SARGENT & LUNDY**ENGINEERS  
CHICAGO

Calcs. For Environmental Qualification of

Containment Spray Pump

☒ Safety-Related☐ Non-Safety-Related

Calc. No CQD-006151

Rev. 00 Date 2-25-83

Page C6 of C15

Client Commonwealth Edison Company

Project Byron/Braidwood Units 1 &amp; 2

Proj. No 4391, 2/4683, 4-00 Equip. No. 1, 2CS01PA, PE

Prepared by

Date

Reviewed by

Date

Approved by

Date

## REFERENCES: in Tab G

Calc. No. CQD 006151  
Rev. 00 Date 2-25-83  
Proj. No. 4391/92/4683/84-00  
Page C6 of C15

- 1 - Ingersoll-Rand Co.'s Drawings: C-8x23 WDF 86x7B; P.L. 8x23 WDF 500x6, pg. 1&2  
C-8x23 WDF 500x6A; L.N. 8x23 WDF 86x7,  
Sheet 1, 2, 3 of 5.
- 2 - Durametallic Co.'s Drawing: #2D-157002-RI dated 7-7-77.
- 3 - Telephone conversation of M. Khan (S&L) with Stan Samuelson (Ingersoll-Rand)  
dated 11-19-82.
- 4 - Telephone conversation of M. Khan (S&L) with F.B. Heakhcote and Henry Schelter  
of Ingersoll-Rand, dated 11-18-82.
- 5 - Lyon's "Valve Designer's Handbook".
- 6 - Wyle Laboratory's Qualification Plan #17491-5.
- 7 - Byron & Braidwood FSAR, Table 3.11-2 for Zone A13C.
- 8 - Ingersoll-Rand and Durametallic maintenance and Instruction Manuals.
- 9 - K. J. Green's memo to T. Thorsell, dated 7-21-82.
- 10 - Ingersoll-Rand Co.'s Certificate of Conformance for the design of the pumps.
- 11 - ASME See XI-Div. 1, Article IWP-1000.
- 12 - NRC's memorandum from Z. R. Rosztoczy to W. V. Johnston, dated 6-23-82.
- 13 - IEEE 627-1980, "Standard for Design Qualification of Safety Systems equipment  
used in Nuclear Power Generating Stations.
- 14 - P&ID Drawings: M-46 and M-61 Sheet 4 of 6.

Client

Prepared by

Date

Project

Reviewed by

Date

Proj. No.

Equip. No.

Approved by

Date

Procedure:

Calc. No: CQD- 006151  
 Rev: 00 Date: 2-25-83  
 Proj. No: 4391/92/4683/84-00  
 Page C7 Of C15

The procedure to be followed will be in accordance with the guidelines of IEEE 627-1980 Standard (ref. 13) & ref. 12. There is no previous environmental qualification available and as such we will qualify these Pumps by Analysis and any information or data available. References will be provided for any information used. The Report applies only to the environmental qualification of the pumps. The dynamic qualification is covered by our report EMD # 025038.

Analysis/Justification:

The pumps safety related function is to pump the containment water (with a mixture of boric acid and NaOH) to the Spray Nozzles in the Containment. For details refer to S&L Spec. F/L-2758 B.

The pumps will be utilized only in the event of a loss of coolant accident and will be automatically started. A conservative estimate of operation time is 200 hours during the plant life. (ref. 9)

A comparison of the Procurement and Design Specifications and the environment conditions of the zone A13c of these pumps is shown in the Environmental Qualification Parameters Comparison table in this Tab.

Per IEEE-627-1980, only significant aging mechanisms have to be analyzed and considered for aging program. The standard lists four criteria to be met for significant aging mechanism.

The pumps' analysis show that their aging mechanisms do not meet the above mentioned criteria and hence are not significant for the purpose of an aging program.

For these pumps, leakage due to the wearing of Seals and O-Rings can be the potential failure due to



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aging. This can be detected by the visual inspection of the pumps during required maintenance and inspection and since the equipment is assessable for normal inservice maintenance, the third criteria is not satisfied and the mechanism is insignificant for aging purpose.

Also, Vendor's Supplied Certificate of Conformance (ref. 10) verifies the design of these pumps per S&L Specification.

With no significant aging mechanism, the pumps are deemed qualified for 40 years design life provided the maintenance and inspection is carried out in accordance with ASME Sec. XI-Div. I, Article IWP-1000 (ref. 11) and the Manufacturer's recommended manuals (ref. 2) besides the maintenance and inspection laid out in Byron/Braidwood FSAR.

The aging mechanism of these pumps has been determined insignificant and as such no aging program has to be developed.

In addition, Analysis of the pumps and their components show that there is no component which could be susceptible to any aging condition and cause a common mode failure.

Following is the analysis of the pumps and justification for qualification:

The Material list in reference 1 Drawings, P.L. 8x23 WDF 500x6 shows that the only non-metallic component is O-Ring of Ethylene Propylene Terpolymer.

The mechanical seal used is Durametallic's Type HPTO. This seal is verified in the manufacturer's Proposal Technical Data, Amd. 1, dated 6-3-77 found in Spec. F/L2752B. Reference 2, drawing # 2D-157002 lists the materials of this seal. The only non-metallics identified are #5 Carbon and Ethylene Propylene Terpolymer.

Telephone Conversation with Henry Scheller of Durametallic (ref. 4)



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CARBON GRAPHITE

Higher temperature resistant, i.e. can operate at  
temp.  $> 1300^{\circ}\text{F}$  (ref 5, P. 391)  
§ 325

High Radiation threshold, i.e:  $1.5 \times 10^9 \text{ Rads}$  (ref. 5, P. 395)

The high ratings of Carbon graphite as compared to  
the plant's environment justify its qualification for  
thermal and Radiation aging.

Humidity : - Effectively used for water jacketed Stuffing boxes (ref 5, P. 385)  
- Good sealant for hot water at 700 Psi (ref. 5, P. 386)

Corrosion : No Corrosion at 100% NaOH & Boric Acid.  
(ref. 5, P. 390)  
Does not Soften at any temp. (ref. 5, P. 391)

Pressure : Has been successfully used for  
sealing hot water at pressure of 700 Psi  
(ref. 5, P. 386)

Note that an adequate margin per IEEE 327-1974  
(greater than 10%) has been shown.

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ETHYLENE PROPYLENE TERPOLYMER (EPT)

Temperature : Range :  $-65^{\circ}$  to  $300^{\circ}\text{F}$  (ref. 5, P: 416)

E.P. rubber good for at least 1000 hrs.  
at max temp. of  $250^{\circ}\text{F}$ . Can  
stand up to  $400^{\circ}\text{F}$  varying with  
environment. (ref. 5 P: 427)

E.P. used in O-Rings for Seals  
rated for :  $120^{\circ}\text{C}$  (ref. 6, P: 23)

Radiation : - Threshold :  $5 \times 10^7$  Rads (ref. 6, P: 23)

Corrosion : Excellent for NaOH & Boric acid  
at a range of  $-70$  to  $250/400^{\circ}\text{F}$  (ref. 5, P: 432 &  
446)

Humidity : Recommended for water & Steam ( $\leq 400^{\circ}\text{F}$ ), Dilute  
Acids & Alkalies. (for details, see ref. 5, P: 416)

The material's ratings are much higher  
than the plant's environment and hence will be  
exempted from Aging Qualification.

This also provides an adequate margin ( $> 10\%$ ) per  
IEEE 323-1974.

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verifies that #5 Carbon used is Carbon graphite.

Together, in the pump and the seal assembly, the only nonmetallics identified are Carbon graphite and EPT. EPT is Ethylene Propylene rubber with a third additional monomer, Diene. (ref. 5, Page: 427)

To justify the aging qualification of the pumps by analysis, we have to look at these two materials and see if they can withstand operating and ambient environment of zone A13C.

Plant's environment : (Zone-A13C) (Ref. 7)

Ambient Temperature : max. 115°F for 40 years

Water " : max. 290°F for total operation of 200 hours.

Radiation :  $1 \times 10^7$  rads (T.I.D.)

Additional radiation

due to the recirculation (Spec F/L2758B, P: 2-2)

of Radiated water : 0.275 rads/hour for 24 hours of recirculation + 2.93 rads (T.I.D.) → insignificant as compared to  $10^7$  rads.

Pressure : -0.5 in H<sub>2</sub>O

Humidity : 0 - 90 % R.H.

NOTE: The Radiation requirement for these pumps as shown in E/S FSAR, Amd. 39, Sept. 1982, Sec. 3.3 does not agree with the S&L Spec. F/L-2758B. This has been brought to NSLD's (S&L) notice (CQD#006155). However, this does not affect the status of our qualification.

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From the above analysis of the nonmetallics used in the pump and seal, it is determined that the pumps do not need any Thermal or Radiation aging and are qualified with an adequate margin per IEEE 323-1974.

### Pressure

The pumps are designed for 350 Psig pressure. (ref. 1)  
The design is verified per Manufacturer's Certificate of Conformance.

The plant's <sup>environmental</sup> pressure of -0.5 to 0.0 in w.g. is insignificant as compared to the design pressure.

### Humidity

The pumps' components are designed for water at 290°F which means all the parts are designed for a higher humidity condition than the ambient of 0-90% R.H.

This design has been verified and the parts have been checked for their Integrity in the operating condition by method of Analysis.

Thus, the pumps components will not be effected by the ambient relative Humidity of 0-90%.

This has also been shown in the materials Analysis in the previous pages.



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Cycling

A conservative estimate of operation time has been determined as 200 hours/plant life. (ref. 9)

The number of cycles in the operation time of 200 hours will be insignificant to consider for cyclic aging.

Also, the nonmetallic components of the pumps are static and will not be subjected to operational stress.

Operation time of 200 hours was determined in K. J. Green's memo to T. Thorsell, dated 7-21-1982 and includes the periodic test times which are very short intervals. (Ref. Tab G.9)

Lubricant

A telephone conversation with Mr. Stan Samuelson of Gingersoll-Rand (ref. 3) verifies the fact that the pumps are product lubricated. The product is water and hence there is no lubricant for qualification.

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CONCLUSION

Based on the preceeding Analysis/Justification, the c.s. pumps are deemed qualified for Byron/Braidwood Nuclear Power Station's environment of zone A B C. A qualified life of 40 years can be justified if proper maintenance and inspection is carried out in accordance with vendor's recommended manuals (ref. 9) and ASME, Sec. XI-Div. 1, Article IWP-1000 (ref. 12) to meet the basic requirements of 10 CFR 50 (Q.A. Criteria for N.P. Generating stations). Tab E also gives the maintenance & Surveillance imposed for the qualification of pumps for 40 years.





Calcs. For Environmental Qualification of	
Ingersoll-Rand's Centrifugal Pumps, 8x23WDF	
X	Safety-Related
	Non-Safety-Related

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Client Commonwealth Edison Company
Project Byron/Braidwood, Units 1 & 2
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ENVIRONMENTAL QUALIFICATION PARAMETERS COMPARISON

Equipment Name: Containment Water Spray Pumps Manufacturer: Ingersoll-Rand Co.

Location: Aux. Building, Elev. 346' for PA Model (Catalog) No.: 8x23WDF,  
& 343' for PB Centrifugal

Function: To pump reactor grade water to the spray nozzles.  
(see S&L spec. F/L-2758B and Proposal Technical Data in Spec.)

PARAMETERS	PROCUREMENT SPECIFICATION	DESIGN SPECIFICATION	CONTROLLED ENVIRONMENT CONDITIONS
Temperature	290°F	290°F	115°F*
Pressure	-	350 Psig	-0.5-0 in H <sub>2</sub> O
Humidity	-	-	0-90%
Radiation	-	-	1x10 <sup>7</sup> Rads
Fluid	Reactor grade water	Reactor grade water	Reactor grade water
Voltage	-	-	-
Current	-	-	-
Frequency	-	-	-
Seismic Class	Category I	Category I	Verified in EMD-02 50.33
Power	-	-	-
Qualified Life	40 years	40 years	-
OTHERS			
	*During operation, the water in the pump is at 93° - 290°F		

### 3.0 DESIGN LOADING CONDITIONS

Seismic - The DBE accelerations used in the static analysis for rigid, active equipment are as follows:

Horizontal .25g (in each of two orthogonal directions)

Vertical .85g

Multiplying these accelerations by the weight of the pump equipment produces the equivalent static forces for the analysis.

Nozzle - The faulted loads are listed in Appendix E. These loads are considered in the analysis of the casing, foot and anchor bolting.

Torque - The torque created by the rated motor horsepower calculates to be 21,245 in.-lb.

Steady-State - Design conditions for the pump are:

Pressure: 350 psi

Temperature: 290°F

Transient - The thermal and pressure transients are as follows:

<u>Temperature Differential</u>	<u>Cycles</u>	<u>Mode</u>
290°F	1	Accident
<u>Pressure Differential</u>	<u>Cycles</u>	<u>Mode</u>
275 psi	481	Testing & Accident

The analysis considers the combined loading which creates the maximum stresses and deflections.

#### 4.0 SUMMARY OF RESULTS

##### 4.1 Natural Frequency

First Five (5) calculates frequencies

1. 58.4 HZ
2. 58.5
3. 108.7
4. 108.8
5. 110.1

##### 4.2 Structural Integrity (Faulted Condition)

<u>Component</u>	<u>Calculated, psi</u>	<u>Allowable, psi</u>
Casing Foot Attachment	1,745	23,250
Casing Disch. Noz. Attachment	4,708	23,250
Casing Suct. Noz. Attachment	3,625	23,250
→ Main Flange Bolting	16,767	37,500
Foot	12,229	29,830
→ Foot Weld	12,320	29,880
Anchor Bolting	Tension Shear	40,000 15,390
Support Head	174	18,900
Motor Attachment Bolting	3,043	37,500

##### 4.3 Operability

<u>Description</u>	<u>Calculated</u>	<u>Allowable</u>
Rotor/Stator Deflection (Motor Air Gap)	.0003	.051 in.
Impeller/Ring Deflection	.001	.0115
Shaft/Cover Deflection at Mechanical Seal	.0004	.010

Attachment B7  
Essential Service Water Butterfly Valve (1SX027A)  
(PVORT-BOP)

Responses to the open audit items are as follows:

- a) A revised PVORT form is included.
  - b) The subject valve was procured under Specification F/L-2884. This specification contained detailed information on four valve sizes (12, 24, 36 and 48") but not on 16" valves such as 1SX027A. This valve was procured via Purchase Order 803068 (attached) which references Data Sheet D5004, Rev. 1 (attached) and Specification F/L-2884. This data sheet provides all size specific information required and with Specification F/L-2884, fully defines the valve requirements. This method of equipment procurement is a common procedure and is fully documented and traceable. As a result no revision to the specification is required.
  - c) The subject valves are marked with an arrow to indicate the preferred installation direction. This installation direction is independent of flow direction; the valve will close against flow in either direction. The arrow indicates the preferred direction for sealing against flow. A field check has been made to verify that in the installed position the valve will seal against flow from the containment when used to isolate the containment.
  - d) The seismic qualification report for this valve has been received, reviewed and approved (Jamesbury Corporation Report JCS 82-02, Rev. 2). The valve was qualified by analysis. The Limitorque operator has been qualified by test. (Limitorque generic qualification report).
- These valves are included in the Byron preservice and inservice test program. The valve will be fully stroked every three months and a position indication test will be performed at refueling outages. Refer to page 43 of the Byron Preservice/Inservice Testing Program Plan for Valves submitted to the NRC in the November 4, 1982 letter from T.R. Tramm to H.R. Denton.
- e) The operating torque of 1180 ft-lbs., as reported in the vendor report VHA-76-71, was used to review the valve design and was compared with the available operator torque to verify that the valve would function as required. The operator torque was obtained from Limitorque and found to be 1250 ft./lb. (memo of telecon attached)
  - f) The valve operator has been environmentally qualified in the generic Limitorque qualification program. Environmental qualification of the valve itself is not required. The only non-metallic parts in the valve are the valve seat (EPT-Ethylene Propylene Terpolymer) and the valve packing (John Crane 187-I). This valve is included in the inservice testing program and containment isolation valve leak rate testing program. Any degradation of the valve which could affect its ability to isolate the containment will be detected by testing and surveillance.



a Pump (continued)

b Prime-mover (continued)

Size \_\_\_\_\_

Size \_\_\_\_\_

Weight \_\_\_\_\_

Weight \_\_\_\_\_

Mounting Method \_\_\_\_\_

Mounting Method \_\_\_\_\_

Required B.H.P. \_\_\_\_\_

H.P. \_\_\_\_\_

Parameter    Design    Operating

Power requirements: (include normal, maximum and minimum).

Press \_\_\_\_\_

Electrical \_\_\_\_\_

Temp \_\_\_\_\_

\_\_\_\_\_

Flow \_\_\_\_\_

\_\_\_\_\_

Head \_\_\_\_\_

Other \_\_\_\_\_

Required NPSH at maximum flow \_\_\_\_\_

If MOTOR list:

Available NPSH \_\_\_\_\_

Duty cycle \_\_\_\_\_

Operating Speed \_\_\_\_\_

Stall current \_\_\_\_\_

Critical Speed \_\_\_\_\_

Class of insulation \_\_\_\_\_

List functional accessories:\*

\_\_\_\_\_  
\_\_\_\_\_

List control signal inputs: \_\_\_\_\_

\_\_\_\_\_  
\_\_\_\_\_

\* Functional accessories are those sub-components not supplied by the manufacturer that are required to make the pump assembly operational, (e.g., coupling, lubricating oil system, etc.)



6. General Valve Data

a. Valve

Name 1SX027A  
Mfg. Jamesbury  
Model 16-8026 Ex. "A"  
S/N NC-48856 - 03C  
Type Wafersphere  
Size 16"  
Weight 190  
Mounting Method Bolted  
Required Torque 1180 ft.-lbf

Parameter	Design	Operating
Press	<u>240</u>	<u>50</u>
Temp	<u>190</u>	<u>189</u>
Flow	<u>6,000</u>	<u>6,000</u>
Max $\Delta P$ across valve	<u>100 psi</u>	
Closing time @ max $\Delta P$	<u>15 sec</u>	
Opening time @ max $\Delta P$	<u>15 sec</u>	
Power requirements for functional accessories, (if any)	<u>120AC</u>	

List control signal inputs: 4106V breaker feeding 1SX01PA; 480V breaker feeding 1VP01CA and 1VP01CC

b. Actuator (if not an integral unit)

Name 1SX027A  
Mfg. Limatorque  
Model SMB - 00/7.5 H1BC  
S/N 264768  
Type H1BC  
Size SMB-00  
Weight 335 lbs.  
Mounting Method Bolted  
Torque 1320 ft.-lbf.

Power requirements: (include normal, maximum and minimum).

Electrical 33 Hp  
460 V - 3 phase  
60 Hz  
Class B Insulation

Other: ☐ Pneumatic ☐ Hydraulic

List functional accessories: \* OT2 Control Switch @ IPM06J

### III. FUNCTION

1. Briefly describe components normal and safety functions: Normally open for normal and post accident essential service water system (sx) in the event of a breach of the essential service water system in containment.

2. The components normal state is: ☒ Operating ☐ Standby

3. Safety function:

a. ☐ Emergency reactor shutdown

b. ☐ Containment heat removal

c. ☒ Containment isolation\*  
\*for breach of sx system only

d. ☐ Reactor heat removal

e. ☐ Reactor core cooling

f. ☐ Prevent significant release of radio-active material to environment

g. ☐ Does the component function to mitigate the consequences of one or more of the following events? ☒ Yes\* ☐ No  
If "Yes", identify. \*Confirmatory open signal

☒ LOCA

☐ HELB

☒ MSLB

☒ Other Breach of the sx system inside containment

4. Safety requirements:

☐ Intermittent Operation

☐ During postulated event

☒ Continuous Operation

☒ Following postulated event

If component operation is required following an event, give approximate length of time component must remain operational.  
single operation required immediately after accident

(e.g., hours, days, etc.)

\* Functional accessories are those sub-components not supplied by the manufacturer that are required to make the valve assembly operational, (e.g., limit switches).

5. For VALVES:

does the component ☐ Fail open ☐ Fail closed ☒ Fail as is

Is this the fail safe position? ☒ Yes\*

\*except for breach of sx system

Is the valve used for throttling purposes? ☐ Yes ☒ No

Is the valve part of the reactor coolant pressure boundary?

☐ Yes ☒ No

Does the valve have a specific limit for leakage? ☐ Yes ☒ No

If "Yes" give limit: \_\_\_\_\_

IV. QUALIFICATION

1. Reference by specific number those applicable sections of the design codes and standards applicable to the component: \_\_\_\_\_

ASME III 1974 ed. thru S'75

2. Reference those qualification standards, used as a guide to qualify the component: IEEE 323 - 1974 (actuator only)

IEEE 344 - 1975

3. Identify those parts of the above qualification standards deleted or modified in the qualification program.

Deleted:

Modified:

4. Have acceptance criterias been established and documented in the test plan(s) for the component? ☒ Yes ☐ No

5. What is the expected failure mode that would keep the pump or valve assembly from performing its safety function? fail open

after breach of sx system

6. Are the margins\* identified in the qualification documentation? ☒ Yes ☐ No

d. Margin is the difference between design basis parameters and the test parameters used for equipment qualification.

If component is a PUMP, complete IV.7.

If component is a VALVE, complete IV.8.

7. Pump operability has been demonstrated by: ☐ Analysis  
☐ Test ☐ Combination

Identify PUMP tests performed:

- |  |   |
|--|---|
| a. <input type="checkbox"/> Shell hydrostatic<br>(ASME Section III)  | b. <input type="checkbox"/> Bearing temperature<br>evaluations  |
| c. <input type="checkbox"/> Seismic loading  | d. <input type="checkbox"/> Vibration levels  |
| e. <input type="checkbox"/> Exploratory vibration<br>(Fundamental freq. _____)   | f. <input type="checkbox"/> Seal leakage @ hydro press  |
| g. <input type="checkbox"/> Aging: <input type="checkbox"/> Thermal<br><input type="checkbox"/> Mechanical   | h. <input type="checkbox"/> Flow performance<br>Are curves provided <input type="checkbox"/> Yes<br><input type="checkbox"/> No |
| i. <input type="checkbox"/> Pipe reaction end<br>loads (nozzle loads)  | j. <input type="checkbox"/> Others _____<br>_____<br>_____  |
| k. <input type="checkbox"/> Extreme environment:<br><input type="checkbox"/> Humidity<br><input type="checkbox"/> Chemical<br><input type="checkbox"/> Radiation | _____<br>_____<br>_____<br>_____  |

8. Valve operability has been demonstrated by: ☐ Analysis  
☐ Test ☐ Combination

Identify VALVE tests performed:

- |  |   |
|--|---|
| a. <input checked="" type="checkbox"/> Shell hydrostatic<br>(ASME Section III)             | b. <input type="checkbox"/> Cold cyclic List times:<br>Open _____<br>Closed _____ |
| c. <input checked="" type="checkbox"/> Seismic loading                                     | d. <input type="checkbox"/> Hot cyclic List times:<br>Open _____<br>Closed _____  |
| e. <input checked="" type="checkbox"/> Exploratory vibration*<br>(Fundamental freq. _____) | f. <input checked="" type="checkbox"/> Main seat leakage                          |

\*Resonant frequency analytically determined.

- g. ☒ Aging\* ☒ Thermal ☐ Back seat leakage  
 \*Actuator only ☒ Mechanical
- i. ☐ Pipe reaction end loading j. ☐ Disc hydrostatic
- k. ☒ Extreme environment l. ☐ Flow interruption capability  
 Actuator Only ☐ Humidity  
☐ Chemical  
☒ Radiation
- m. ☐ Flow characteristics n. ☒ Others pneumatic test  
 Are curves provided? \_\_\_\_\_  
☐ Yes ☐ No \_\_\_\_\_

9. As a result of any of the tests (or analysis), were any deviations from design requirements identified? ☐ Yes ☒ No  
 If "Yes", briefly describe any changes made in tests (or analysis) or to the component to correct the deviation.  
 \_\_\_\_\_  
 \_\_\_\_\_  
 \_\_\_\_\_

10. Was the test component precisely identical (as to model, size, etc.) to the in-plant component? ☒ Yes ☐ No If "No", is installed component ☐ oversized or ☐ undersized?
11. If type test was used to qualify the component, does the type test meet the requirements of IEEE 323-1974, Section 5.? ☒ Yes ☐ No \*for motor operator only
12. Is component orientation sensitive? ☐ Yes ☐ No ☒ Unknown\*  
 If "Yes", does installed orientation coincide with test orientation? ☐ Yes ☐ No
13. Is the component mounted in the same manner in-plant as it was during testing (i.e., welded, same number and size bolts, etc.) ☒ Yes ☐ No ☐ Unknown

\*However, installed orientation coincides with test orientation.

14. Were the qualification tests performed in sequence and on only one component? ☒ Yes ☐ No A tuator only.

If "Yes" identify sequence. (e.g., radiation, seismic, cyclic, thermal, etc.): \_\_\_\_\_

thermal aging, mechanical aging, radiation, seismic

15. If "aging"\* was performed, identify the significant aging mechanisms: \_\_\_\_\_

16. Identify loads imposed (assumed) on the component for the qualification tests (analysis) performed:

- a. ☐ Plants (shutdown loads)      b. ☐ Extreme environment  
c. ☒ Seismic load                      d. ☐ Others \_\_\_\_\_

17. Have component design specifications been reviewed in-house to assure they envelope all expected operating, transient, and accident conditions? ☒ Yes ☐ No

18. Does the component utilize any unique or special materials? (Examples are special gaskets or packing, limitations on nonferrous materials, or special coatings or surfaces.)  
☐ Yes ☒ No

If "Yes", identify: \_\_\_\_\_

19. Does component require any special maintenance procedures or practices, (including shorter periods between maintenance).  
☐ Yes ☐ No

If "Yes", identify: The standard plant maintenance programs will  
address all required maintenance considerations for this component.

20. Is the qualified life for the component less than 40 years?  
☐ Yes ☒ No If "Yes", what is the qualified life? \_\_\_\_\_

\* As outlined in Section 4.4.1 of IEEE-627 1980.





FORM ME-5.6.1 (4/75) Approved By

Dept. Mgr.

SERVICE

BODY

OPERATOR

ACCESS

1	SPEC NO. F/Lr	2884	6	DESCRIPTION & TAG NO	
2	PIPE CLASS:	E SEISMIC CLASS: A		1SX016A	2SX016A
3	P & ID: M-	242-05		1SX016B	2SX016B
4	PIPE DESIGN TABLE:	105 BB		1SX027A	2SX027A
5				1SX027B	2SX027B
7	FLUID: <input type="checkbox"/> RAW WATER <input type="checkbox"/> OIL <input type="checkbox"/> STEAM <input type="checkbox"/> DEMIN. WATER <input type="checkbox"/> AIR <input checked="" type="checkbox"/> OTHER SEAFREE WATER				
8	INLET PRESSURE (PSIG)		MAXIMUM UNDER NORMAL CONDITIONS		DESIGN
9	TEMPERATURE SX016 VALVES; SX027 VALVES (°F)		100		125
10	MAXIMUM Δ P VALVE MUST OPEN AGAINST (PSIG):		100; 189		100; 190
11	MAXIMUM Δ P VALVE MUST CLOSE AGAINST (PSIG):		100		
12	MAXIMUM DESIGN RADIOACTIVITY LEVEL: 1.0 X 10 <sup>4</sup> RADS				
13	FLUID VELOCITY @ VALVE INLET 11 (FT/SEC)				
14	Δ P WHEN VALVE FULL OPEN 3 MAX. (PSIG)		0 6000 <input type="checkbox"/> LB/HR <input checked="" type="checkbox"/> GPM <input type="checkbox"/> SCFM		
15	BODY SIZE (INCHES): 16				
16	BODY TYPE: <input type="checkbox"/> GATE <input type="checkbox"/> GLOBE <input type="checkbox"/> DIAPHRAGM <input checked="" type="checkbox"/> BUTTERFLY <input type="checkbox"/> OTHER				
17	BODY MATERIAL: <input checked="" type="checkbox"/> CARBON STEEL: ASME SA 515 Gr. 70 <input type="checkbox"/> 2 CR-MOLY				
	<input type="checkbox"/> CAST IRON: ASTM <input type="checkbox"/> BRONZE: ASTM				
	<input type="checkbox"/> 304 SS: ASTM <input type="checkbox"/> 316 SS: ASTM <input type="checkbox"/> OTHER				
18	TYPE END CONNECTION		INLET		OUTLET
			<input type="checkbox"/> BUTT WELD <input type="checkbox"/> FLNGD. <input type="checkbox"/> BUTT WELD <input type="checkbox"/> FLNGD.		
			<input type="checkbox"/> SOC. WELD <input type="checkbox"/> SCRD. <input type="checkbox"/> SOC. WELD <input type="checkbox"/> SCRD.		
			<input checked="" type="checkbox"/> FLANGELESS (WAFER) <input checked="" type="checkbox"/> FLANGELESS (WAFER)		
19	CONNECTING PIPE SIZE/SCHEDULE 16 1 3/2" NPS				
20	WELD END STANDARD (BUTT WELD ONLY)		<input type="checkbox"/> MF-270.8. <input type="checkbox"/> OTHER		<input type="checkbox"/> MF-270.8. <input type="checkbox"/> OTHER
21	FLOW ACTION: <input checked="" type="checkbox"/> OPEN <input type="checkbox"/> CLOSE <input type="checkbox"/> BI-DIRECTIONAL <input type="checkbox"/> NEUTRAL				
22	STEM LEAKOFF REQUIRED: <input type="checkbox"/> YES <input checked="" type="checkbox"/> NO				
23	MOTOR RATING: <input checked="" type="checkbox"/> AC <input type="checkbox"/> DC				
24	MOTOR VOLTAGE: <input checked="" type="checkbox"/> PER PROJECT SPEC. OR <input type="checkbox"/> VOLTS				
25	CLASS IE: <input type="checkbox"/> NO <input checked="" type="checkbox"/> YES				
26	VALVE CLOSING TIME: <input type="checkbox"/> AS REQUIRED FOR SPECIFIED STEM SPEED				
	<input checked="" type="checkbox"/> OTHER, SEC. 15 OR LESS				
27	VALVE OPENING TIME: <input type="checkbox"/> AS REQUIRED FOR SPECIFIED STEM SPEED				
	<input checked="" type="checkbox"/> OTHER, SEC. 15 OR LESS				
28	FREQUENCY OF OPERATION: 12 CYCLES/YEAR				
29	POSITION TRANSMITTER: <input checked="" type="checkbox"/> NO <input type="checkbox"/> YES				
30	ZONE: <input type="checkbox"/> CONTAINMENT <input checked="" type="checkbox"/> AUXILIARY <input type="checkbox"/> TURBINE <input type="checkbox"/> OTHER				
31	SAFETY FUNCTION CLASSIFICATION: <input checked="" type="checkbox"/> ACTIVE <input type="checkbox"/> PASSIVE <input type="checkbox"/> NOT APPLICABLE				

- 1. DISC MATERIAL: 316 SS SA 351 CF8M
- 2. SEAT MATERIAL: EPT
- 3. ADJ. PIPE FLANGES: 150# ANSI

\* REV. 1 ADDED LINES 30 & 31  
 REVISED LINES 12 & 14

MOTOR-OPERATED VALVE DATA SHEET	
PROJECT BYRON/BRAIDWOOD	
CLIENT COMMONWEALTH EDISON	JOB NO. 4391/4683
<div style="border: 2px solid black; padding: 5px; display: inline-block;"> <b>CANNON &amp; LUNNEY</b>  <small>ENGINEERS</small> </div>	STANDARD
	DATA SHEET NO. D 5004 REV. 1

DESIGN BY	J.C. Raul	DATE	4-25-77
CHECKED BY	J.C. Raul	DATE	4-25-77
REVISED BY	J.C. Raul	DATE	4-25-77

# SARGENT & LUNDY

## MEMORANDUM TELEPHONE CONVERSATION

Date: June 29, 1983

Time: 11:15 a.m.

Person Called: Dave Montgomery of Limitorque Corporation  
(Name) (Company)

Person Calling: H. G. L. McCullough of Sargent & Lundy  
(Name) (Company)

Project: Byron/Braidwood Project No. 4391-00

Subject Discussed: \_\_\_\_\_  
Motor operator for valves 1SX027A, B, for Butterfly Valves  
Specification F/1 2884.

### Summary of Discussion, Decisions and Commitments:

Per Drawing #9030 - 2 Sheets

Question: Does Motor Operator SMB00/7.5-HIBC develop 1180 ft-lbs  
or more of torque?

Answer: This model has a motor torque normal rating of 7.5 ft-lbs.  
HI BC refers to worm gear reducer, with a maximum rating  
of 1300 ft-lbs. For a 15 second valve closure time, the  
motor operator gear ratio is 24:1. When operated on  
100% voltage required, the developed nominal torque is  
1250 ft-lbs.

The operator is adequate for the valve operational  
requirements.

cc

CC. K. J. Green ✓

File:

H. G. L. McCullough  
Signature

Attachment B6

Responses to PVORT Draft  
SER Generic Items

PORVT Draft SER  
Generic Item 1

The status of completion of qualification documentation is as shown on the attached updated SQRT summary sheets. The status of completion of equipment installation can be inferred from these summary sheets. Complete (C) means equipment is in place and mounted. However, incomplete construction details on individual pieces of equipment are not tracked by the SQRT summary sheets to ensure completion.

There are provisions presently incorporated in the construction process and preoperational test program to identify and track incomplete construction details (such as temporary pipe supports, unconnected drain lines, etc.) In the construction process, contractors to Commonwealth Edison use procedures for installing equipment that have been reviewed and approved by Commonwealth Edison. These procedures include QC/QA signoffs to ensure installation is complete and in conformance with design drawings. The contractors' QC department inspects equipment to verify it is completely and properly installed. This review and inspection function is periodically audited by the contractors' QA department and the Commonwealth Edison QA department, also.

In the preoperational test program, all systems are walked down and inspected by the construction department and the system test engineer before performing the preoperational test on the system. If any incomplete installation details are identified during these walkdowns, a turnover deficiency is written to document the problem at the time of turnover for test, or thereafter. All turnover deficiencies and test deficiencies for each system are evaluated at the time the preoperational test results are reviewed and a schedule is established for resolving all deficiencies.

In addition, all Category 1 piping systems are given an as-built walkdown for piping analysis reconciliation. Any incomplete installation details with respect to pipe supports or hangers would be noted at that time.

Based on the documentation and inspection programs described above, all Category 1 equipment will be completely installed by fuel load.



PVORT Draft SER  
Generic Item 2

Qualification documentation for pump and valve operability primarily consists of QC Documents (Test Reports and Procedures) and Engineering Documents (Environmental Qualification Reports, Seismic and Stress Reports, Performance Curves, Vendor Calculations). The documentation required for safety-related pumps and valves includes various material test reports such as radiographs, liquid penetrant tests and magnetic particle tests. Documentation which assures pump and valve operability may include material test reports, welding reports, hydrostatic and seat leakage test reports, rotor balancing test reports, performance test reports, seismic and IE electrical qualification reports. Specific requirements for pumps and valves vary depending on type and required function.

QC documentation has consistently been reviewed, approved and transferred to the Byron Site QA Office, however, an actual listing of required documents was not made until recently. Currently, Master Documentation Lists are being compiled for each S&L procurement specification on which all required QC documents and Engineering Technical Data Documents are listed for each piece of equipment (by S&L Tag Number). As of 06-01-83, all lists for safety-related specifications have been transferred to Byron Station. The lists show which items are on file at the Site QA Office and items which are still required for submittal by vendors for review. S&L Monthly Engineering Status Reports on Environmental Qualification, Seismic and Stress are used to correlate the status of items on these Master Lists. Engineering Reports (Seismic, Stress and EQ) are in the process of being transferred to both stations. All of the QC Documentation for the BOP pump and valve audit items has been transferred to the site. Engineering documents are on file at S&L. Verification that all items on file at the site correspond with all items listed on the Master Documentation List is being performed by the site QA Department.

To demonstrate the acceptability and completeness of the operability qualification for the Byron plant, the Westinghouse files were reviewed. Enclosed is a summary report of this review which identifies the qualification documentation associated with each active pump and valve. It was agreed with the Staff at the 05-13-83 meeting that such a review and summary report would be adequate to demonstrate completion of the operability qualification for Byron.



Attachment A1  
Electrical Penetration Assemblies  
(BOP/1)

Responses to the open audit items are as follows:

Open Item 1 - The report describing the environmental aging and qualification testing (Conax IPS-369, Rev. C) has been reviewed and approved and is available for SQRT review. This report was audited by the NRC during the Environmental Qualification Audit on June 21, and 22, 1983.

Open Item 2 - The nitrogen supply to the Byron Unit 1 penetrations was originally designed to meet the requirement of the Amphenol containment penetrations. Subsequently, Conax penetrations were installed in place of the Amphenol penetrations. The Conax penetrations are qualified to operate with air rather than nitrogen between the seals. The nitrogen has been retained on Byron Unit 1 as an additional design feature beyond qualification requirements. The nitrogen lines are provided with a Category I isolation valve to allow for isolation in the event that one of the seals does leak. The nitrogen system is not Category I because its failure will not adversely affect the penetration integrity.