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<b>CHANGE CONTROL RECORDS:</b>	France:	N	
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## REVISIONS

REVISION	DATE	EXPLANATORY NOTES
7.0	See 1 <sup>st</sup> page release date	Section 2.6.9 <ul style="list-style-type: none"> <li>Corrected typographical error.</li> </ul>
		Section 3.2.2 <p>Corrected typographical error of three occurrences where the Fahrenheit temperatures do not have the degree symbol (°)</p>
		Section 4.4 <ul style="list-style-type: none"> <li>Section header changed typographical error.</li> </ul>
		Section 6 <ul style="list-style-type: none"> <li>Updated Table 6-3</li> <li>Updated Table 6-59</li> <li>Additional changes were made on pages 402, 414, 415, 424, 461, and 469 to support the changes in Table 6-3 and Table 6-59</li> <li>Page 506, format change to Reference 20</li> </ul> <p>These changes were made in response to a NRC RAI</p>
		Section 8.2.2 paragraph was revised to provide a description of the qualification and acceptance tests for the ATRIUM 11 fuel rod and end caps in response to a NRC RAI.
6.0	8/31/2017	Section 1 <ul style="list-style-type: none"> <li>Section 1. Line four in first paragraph change page number for Glossary of Terms and Acronyms from "15" to "23"</li> <li>Section 1.2 Line two in first paragraph changed "real" to "distinct" and in first line of fifth paragraph added "are"</li> <li>Section 1.2.3. Deleted "primarily as" and "and U<sub>3</sub>O<sub>8</sub>" in fourth sentence of first paragraph.</li> <li>Table 1-3 Corrected footnote 1</li> </ul>
		Section 2 <ul style="list-style-type: none"> <li>Corrected Figure 2-13 number to Figure 2-12 and renumbered subsequent figure numbers.</li> <li>Section 2.7.4.1. Corrected typographic error</li> </ul>
		Section 3 <ul style="list-style-type: none"> <li>Section 3.4.2. Corrected number to "1.1145" in first sentence</li> <li>Section 3.5.2. Correct typographic error in third paragraph</li> <li>Section 3.5.3.2. Changed "perfect " to "ideal" in third sentence of first paragraph and changed "psia" to "psi" in second equation on page 167</li> <li>Table 3-5 Updated table and Note</li> </ul>




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REVISION	DATE	EXPLANATORY NOTES
6.0 continued		<p>Section 4</p> <ul style="list-style-type: none"> <li>Section 4.1.1. Update to section</li> <li>Section 4.2.2. Numbers changed</li> <li>Section 4.4. Numbers changed and added sentence to fourth paragraph</li> </ul> <p>Section 6</p> <ul style="list-style-type: none"> <li>Table 6-3 Updated "Fuel Assembly Package Array HAC" information and added new footnote a and renumbered existing footnote to b</li> <li>Table 6-59 Updated "Fuel Assembly Package Array HAC" information and added new footnote a and renumbered existing footnote to b</li> <li>Section 6.12.3.1.1. added "(single package)" to bullet item eight and added bullet item nine. Updated last two paragraphs in this section on page 402</li> <li>Figure 6-52 added "5 wt%" to captions for first two figures and added "5 wt% (NCT, HAC single package)" caption to third figure and added fourth figure with caption "3.3 wt% (HAC package array)"</li> <li>Section 6.12.3.2. Updated second and fourth paragraph</li> <li>Table 6-60 Added footnotes a and b</li> <li>Updated bullet five on page 414</li> <li>Section 6.12.3.4.1. Updated number in last paragraph</li> <li>Section 6.12.3.5.11. Added last paragraph on page 426</li> <li>Table 6-79 Added last row to table and footnote a</li> <li>Section 6.12.4.1. Added new last paragraph on page 460</li> <li>Section 6.12.4.2. Added new last paragraph on page 461</li> <li>Section 6.12.5.1. Added "with axial uniform enrichment" in second paragraph and added last paragraph on page 466</li> <li>Section 6.12.6.1. Added "for the uniform axial enrichment" to last bullet item, added "with uniform axial enrichment" to second, and fourth paragraph; fifth paragraph deleted "worst" from the second and third sentence and fourth sentence; and added new last paragraph (all changes on page 468)</li> <li>Section 6.12.10.1. added reference to Section 6.12.3.2. and Table 6-60</li> <li>Section 6.12.10.2. added reference to Section 6.12.3.2. and Table 6-60</li> </ul>
5.0	2/15/2017	<p>Section 1.2.3.4.7. "Fuel Rods in a Protective Case" – changed "polyethylene" to "polyurethane" to reflect actual configuration.</p> <p>Corrections to Tables 6-1, 6-56, and 6-61.</p> <p>Sections 7.1.2.4. and 7.1.2.6. paragraph 3 added "11x11 or".</p>
4.0	11/17/2016	<p>Section 1</p> <ul style="list-style-type: none"> <li>Section 1.2.3 "Contents" Added 11x11 arrays.</li> <li>Table 1-2 Updated to include Type 11x11. Table reformatted to improve readability.</li> <li>Table 1-3 Updated to reflect the higher uranium dioxide weight of a 11x11 fuel assembly.</li> </ul>

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REVISION	DATE	EXPLANATORY NOTES
4.0 continued		<p>Section 1 - continued</p> <ul style="list-style-type: none"> <li>Table 1-4 Updated to reflect an 11x11 fuel assembly.</li> <li>Section 1.2.3.4.7 "Fuel Rods in a Protective Case" – changed "polyurethane" to "polyethylene" to reflect actual configuration.</li> <li>Table 1-5 Deleted as information is available in other tables and sections.</li> <li>Table 1-5 Added "and Zirconium alloy" to the list of structural material for nuts to reflect current fuel assembly designs.</li> <li>Section 1.41. The pages for the drawings on these pages were changed to a 11x17 size in order to improve readability. The drawings are unchanged from revision 3.</li> </ul> <p>Section 3</p> <ul style="list-style-type: none"> <li>Table 3-5 was updated to include the ATRIUM 11 and updated other designs to the current limiting design requirements.</li> </ul> <p>Section 6</p> <p>Added Appendix B, Section 6.12, for the criticality analysis of 11x11 fuel assemblies and rods</p> <p>Other Changes Include:</p> <ul style="list-style-type: none"> <li>Table 6-1, replaced theoretical density limit with a gram density, added 11x11 fuel assembly information</li> <li>Table 6-2, replaced theoretical density limit with a gram density, added 11x11 fuel rod information</li> <li>Section 6.1.1.1, added an option for using 9 pcf foam for the FANP 10x10 and 11x11 assemblies</li> <li>Table 6-3, added the 11x11 fuel assembly and rod results</li> <li>Table 6-4, added the 11x11 fuel assembly information</li> <li>Section 6.1.3, added headers for clarification</li> <li>Table 6-41, corrected typographical error</li> <li>Table 6-44, corrected typographical error</li> <li>Table 6-48, corrected typographical error</li> <li>Table 6-51, corrected typographical error</li> <li>Table 6-54, corrected typographical error</li> </ul>
3.0	4/2/2014	Revisions in Sections 1.2.3.4.6 and 1.2.3.4.7 have been changed in response to a NRC RAI.
		Revisions in Section 2.5.1 have been changed in response to a NRC RAI.
		Correction in Table 6-8 have been changed in response to a NRC RAI.
		Revisions in Section 8.1.2 have been changed in response to a NRC RAI.


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REVISION	DATE	EXPLANATORY NOTES
2.0	3/14/2014	Added 3 figures on pages 49, 50, and 51p per NRC RAI.
1.0	1/28/2014	New document

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## Glossary of Terms and Acronyms

**ASME** – American Society of Mechanical Engineers

**ASME B&PVC** – ASME Boiler and Pressure Vessel Code

**ASNT** – American Society for Non-destructive Testing

**CG** – Center of Gravity

**CTU** – Certification Test Unit

**BWR** – Boiling Water Reactor

**HAC** – Hypothetical Accident Condition

**IC** – Inner Container

**IC Inner Thermal Insulator (Aluminum Silicate)** – The Alumina Silicate thermal insulation between the inner and outer walls of IC container to provide added margin to criteria set forth for HAC fire condition in 10 CFR 71.73(c)(4)

**IC Lid** – The lid of the inner container

**IC Body** – The body of the inner container consisting of the outer wall the thermal insulation, the inner wall, the polyethylene liner and the shock absorbing system along with the fuel securement system

**JIS** – Japanese Industrial Standards

**JSNDI** – Japanese Society for Non-destructive Inspection

**LDPE** – Low Density Polyethylene

**NCT** – Normal Conditions of Transport

**NDIS** – Non-destructive Inspection Society

**OC** – Outer Container

**OC Body** – The assembly consisting of the OC lower wall, and the internal shock absorbing material

**OC Lid** – The lid for the outer container.

**Packaging** – The assembly of components necessary to ensure compliance with packaging requirements as defined in 10 CFR 71.4. Within this SAR, the packaging is denoted as the TN-B1 packaging

**Package** – The packaging with its radioactive contents, as presented for transportation as defined in 10 CFR 71.4. Within this SAR, the package is denoted as the TN-B1 package.

**Payload** – Unirradiated fuel assemblies and fuel rods.

**RAM** – Radioactive Material

**SAR** – Safety Analysis Report (this document)

**TI** – Transport Index

**USL** – Upper Safety Limit

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## 1. GENERAL INFORMATION

This chapter of the Safety Analysis Report (SAR) presents a general introduction and description of the TN-B1 package. The major components comprising the TN-B1 package are presented in Figure 1-1 through Figure 1-4. Detailed drawings presenting the TN-B1 packaging design are included in Appendix 1.4.1. Terminology and acronyms used throughout this document are presented in the Glossary of Terms and Acronyms on page 23. This package is intended to be used to transport Boiling Water Reactor (BWR) fuel assemblies containing both Type A and Type B fissile material.

### 1.1. INTRODUCTION

The model TN-B1 package is derived from the RAJ-II package (NRC CoC 9309) and has the same structural design. The only distinct difference between the TN-B1 and the RAJ-II will be the allowed contents.

The model TN-B1 package has been developed to transport unirradiated fuel for Boiling Water Reactors. The cladding of the fuel provides the primary containment for the radioactive material. The inner and outer containers provide both thermal protection as well as mechanical protection from drops or accident conditions.

The integrity of the fuel is maintained by the protective outer package, the insulated inner package and the fuel rod cladding through both Normal Conditions of Transport (NCT) and Hypothetical Accident Conditions (HAC) deformations. A variety of full-scale engineering development tests were included as part of the certification process. Ultimately, two full-scale Certification Test Units (CTUs) were subjected to a series of free drops and puncture drops.


The payload within each TN-B1 package consists of a maximum of two unirradiated Boiling Water Reactor (BWR) fuel assemblies or individual rods (BWR, Uranium Carbide, or generic Pressurized Water Reactor (PWR)) contained in a cylinder, protective case or bundled together and positioned in one or both sides of the inner container. See Table 6-1 TN-B1 Fuel Assembly Loading Criteria. See Table 6-2 TN-B1 Fuel Rod Loading Criteria. The containment is provided by the leak tested cladding making up the fuel rods.

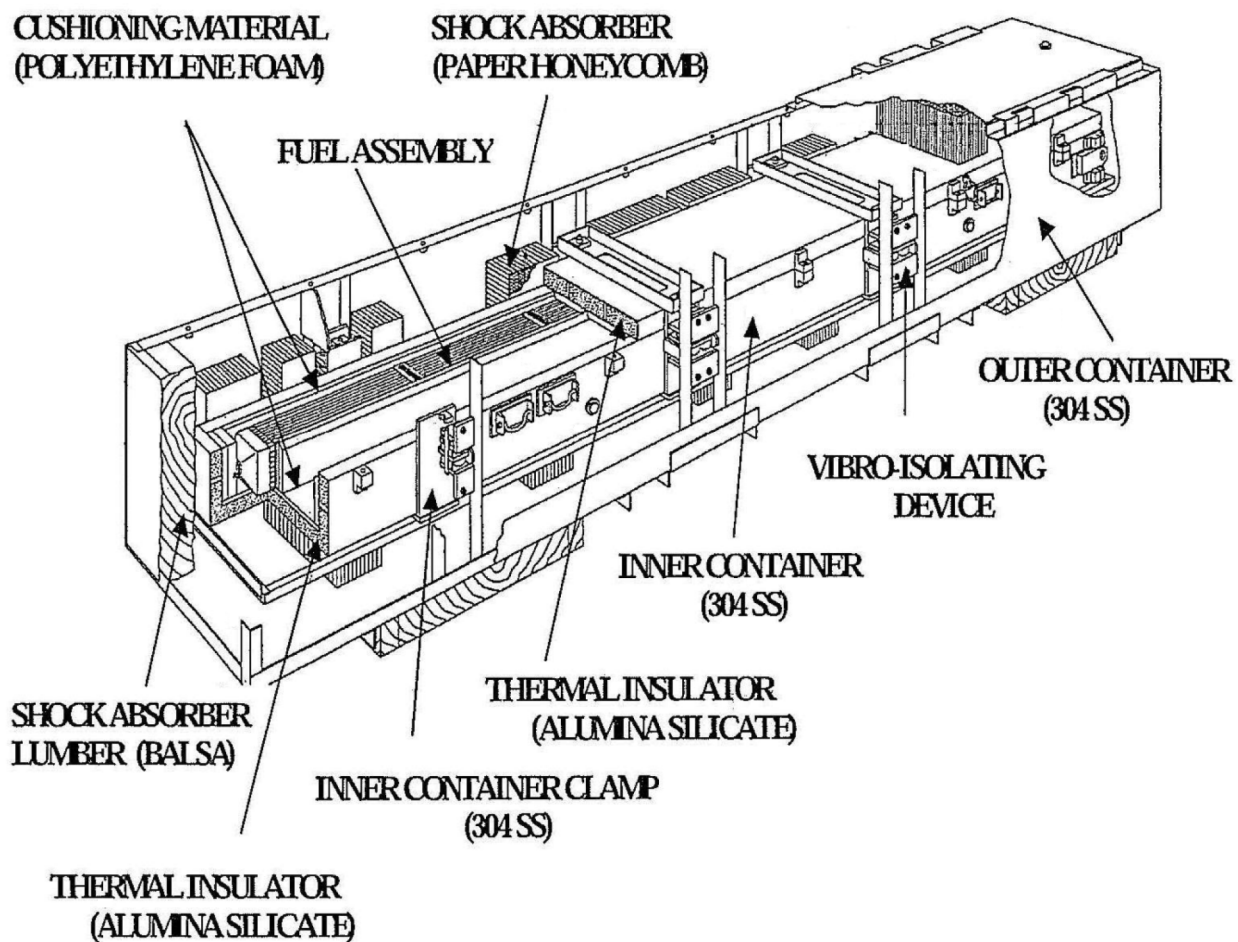
The shielding and criticality assessments are provided in Chapter 5.0 and Chapter 6.0. The Criticality Safety Index (CSI) for the TN-B1 package is defined in Chapter 6.0.

The TN-B1 package is designed for shipment by truck, ship, or rail as either a Type B(U) fissile material or Type A fissile material package per the definition in 10 CFR 71.4 and 49 CFR 173.403.


Dimensions of the packaging identified in the text, tables, figures, etc. of this SAR, are intended to be nominal. The drawings provided in Appendix 1.4.1 contain the dimensions and the tolerances.

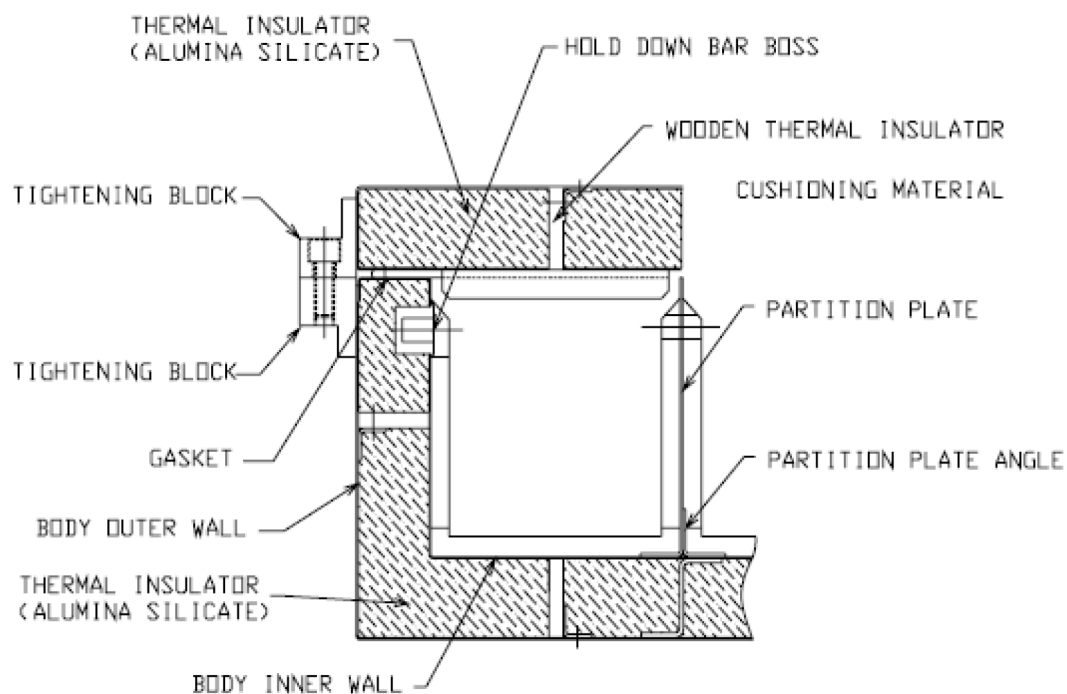


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**Figure 1-1 TN-B1 PACKAGE ASSEMBLY**

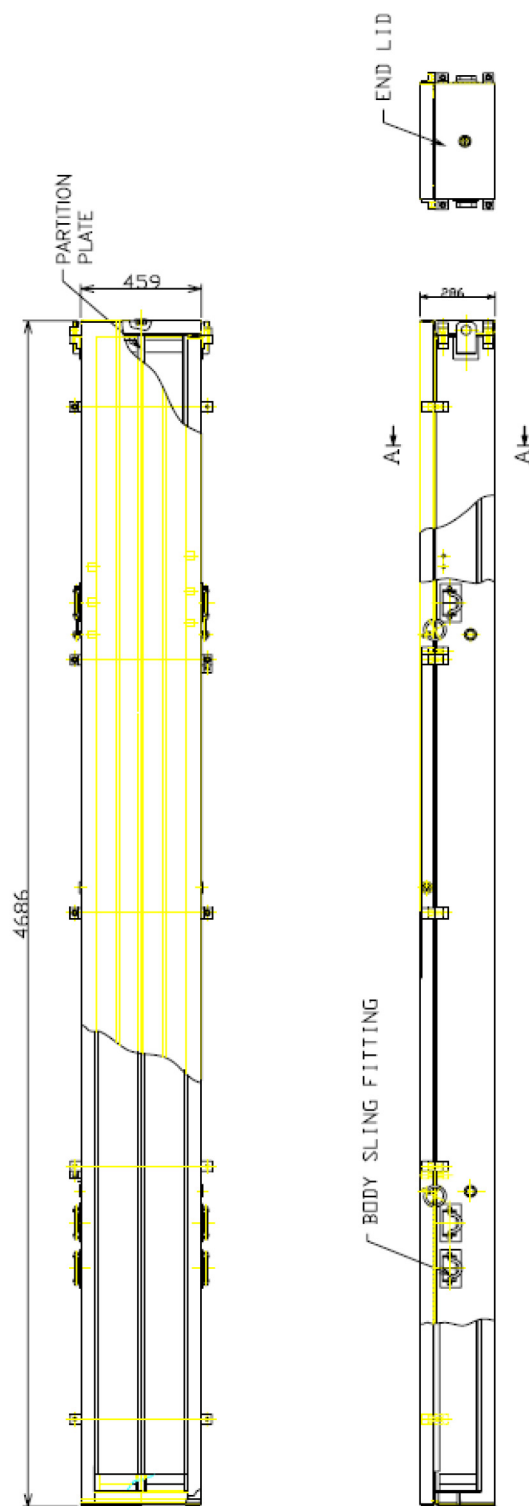
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**Figure 1-2 Cross-Section of Inner Container**

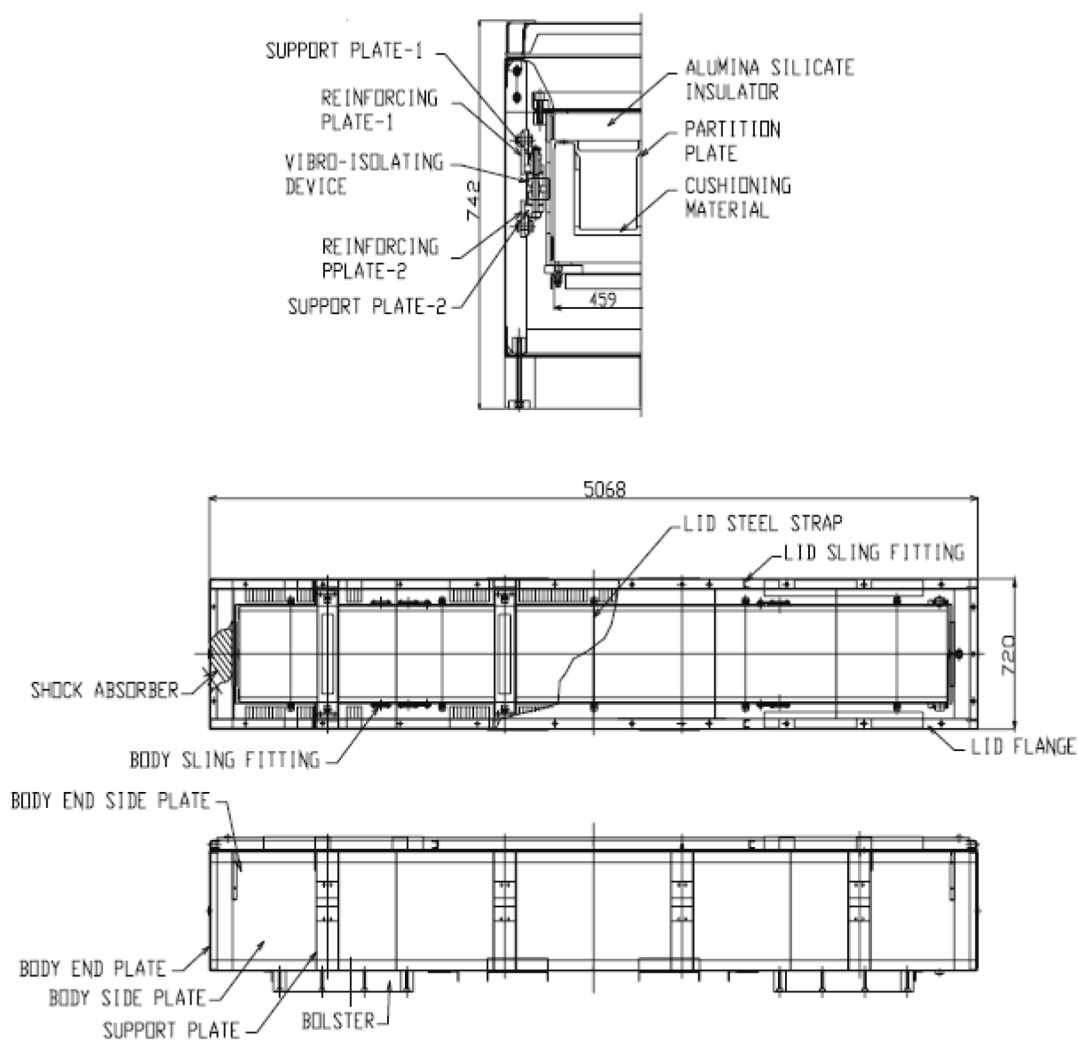


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**Figure 1-3 Inner Container**

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**Figure 1-4 Inner and Outer Container**

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## 1.2. **PACKAGE DESCRIPTION**

This section presents a basic description of the model TN-B1 package. General arrangement drawings of the TN-B1 package are presented in Appendix 1.4.1. The Transport Index (TI) for this package is based on shielding and criticality assessments provided in Chapter 5.0 and Chapter 6.0.

### 1.2.1. ***Packaging***


The packaging is comprised of one inner container and one outer container both made of stainless steel. The inner container is comprised of a double-wall stainless steel sheet structure with alumina silicate thermal insulator filling the gap between the two walls to reduce the flow of heat into the contents in the event of a fire. Foam polyethylene cushioning material is placed on the inside of the inner container for protection of the fuel assembly. The outer container is comprised of a stainless steel angular framework covered with stainless steel plates. Inner container clamps are installed inside the outer container with a vibro-isolating device between to alleviate vibration occurring during transportation. Additionally, wood and a honeycomb resin impregnated kraft paper (hereinafter called "paper honeycomb") are placed as shock absorbers to reduce shock due to a drop of the package. In addition to the packaging described above, the fuel rod clad and ceramic nature of the fuel pellets provide primary containment of the radioactive material.

The design details and overall arrangement of the TN-B1 packaging are shown in Appendix 1.4.1 TN-B1 General Arrangement Drawings.

#### 1.2.1.1. **Inner Container (IC)**

The structure of the inner container is shown in Figure 1-2 and Figure 1-3. The inner container is comprised of three parts: an inner container body, an inner container end lid (removable), and an inner container top lid (removable). These components are fastened together by bolts made of stainless steel through tightening blocks. The inner container body is fitted with six sling fittings and the inner container lid is fitted with four sling fittings as shown in Figure 2-2 Inner Container Sling Locations. The inner container body has a double wall structure made of stainless steel. Its main components are an outer wall, inner wall and alumina silicate thermal insulator.

The outer wall is made of a 1.5 mm (0.0591 in) thick stainless steel sheet formed to a U-shape that constitutes the bottom and sides of the inner container body. A total of 14 stainless steel tightening blocks are attached on the sides of the outer wall, seven per side, to fasten the inner container lid and the inner container end lid by bolts. Additionally, six stainless steel sling fittings are attached on the sides (three on each side) for handling.

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The inner wall of the inner packaging is formed into U-shape with 1.0 mm (0.0391 in) thick stainless steel sheet. The inner packaging is partitioned down the center with 2.0 mm (0.0787 in) thick stainless steel sheet welded to the bottom of the packaging. Foam polyethylene is placed on the inner surface of the inner wall where the fuel assemblies are seated. The void space between the outer and inner steel sheeting is filled with an alumina silicate thermal insulation 48 mm (1.89 in) thick.

#### 1.2.1.2. **Outer Container (OC)**

The structure of the outer container is shown in Figure 1-4. The outer container is comprised of three parts: a container body, a container lid and inner container hold clamps made of stainless steel and fastened together using stainless steel bolts.

Two tamper-indicating device attachment locations are provided, one on each end, of the outer container.

##### 1.2.1.2.1. **Outer Container Body**

The outer container is made from a series of stainless steel angles (50mm x 50mm x 4mm)(1.97 inch x 1.97 inch x 0.157 inches) that make the framework. Welded to the framework are a bottom plate and side plates made of 2 mm (0.079 inch) thick stainless steel.

Sling holding angles for handling with a crane and protective plates for handling with a forklift are welded on the outside of the container body.


A total of eight sets of support plates are welded on the inside of the outer container body for installing the inner container hold clamps. Additionally, shock absorbers made of 146 mm (5.75 in) wood are attached to each end and paper honeycomb shock absorbers are attached to the bottom and sides for absorbing shock due to a drop. The geometry of the shock absorber is shown in Figure 1-5. The shock absorbers are 157 mm (6.18 in) thick and 108 mm (4.25 in) thick.

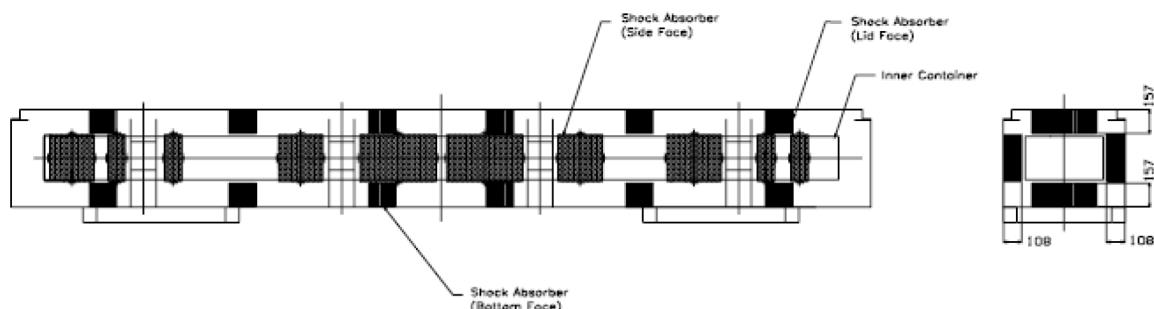
##### 1.2.1.2.2. **Outer Container Lid**

The outer container lid is comprised of a lid flange and a lid plate made of stainless steel.

Stainless steel lid sling fittings are welded four places on the top surface of the outer container lid. A paper honeycomb shock absorber, 157 mm (6.18 in) thick by 160 mm (6.30 in) wide and 380 mm (14.96 in) long is attached to the bottom side of the lid similar to the attachment at the bottom of the container.

The outer container lid has holes for bolts in its flange so that it can be fastened to the outer container body by the stainless steel bolts.

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**Figure 1-5 Shock Absorber Geometry**

#### 1.2.1.2.3. ***Inner Container Hold Clamp (Located on Outer Container)***

The inner container hold clamp consists of an inner container receptacle and a vibro-isolating device.


The inner container receptacle consists of an inner container support plate, a support frame, a bracket and an inner container hold clamp fastener made of stainless steel. The receptacle guides the inner container to the correct position. The inner container receptacle is fitted with the vibro- isolating device through the gusset attached to the bracket.

The vibro-isolating material is attached on the upper and lower side of the gusset. Shock mount fastening bolts go through the center of each piece of vibro-isolating rubber. The bolts at both ends are tightened so that the vibro-isolating rubber pieces press the gusset.

There are four sets (eight pieces) of the vibro-isolating devices mounted on the outer container. Finally, a variety of stainless steel fasteners are used as specified in Appendix 1.4.1.

#### 1.2.1.3. **Gross Weight and Dimensions**

The maximum gross shipping weight of a TN-B1 package is 1,614 kg (3,558 pounds) maximum. A summary of the major component weights and dimensions are given in Table 1-1. A summary of overall component weights is delineated in Table 2-1.

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**Table 1-1 Maximum Weights and Outer Dimensions of the Packaging**

Item	Weight and outer dimensions
Maximum weight of inner	308 kg (679 lb)
Maximum weight of outer	622 kg (1,371 lb)
Maximum weight of packaging	930 kg (2,050 lb)
Dimensions of inner container	Length: 4,686 mm (184.49 in) Width: 459 mm (18.07 in) Height: 286 mm (11.26 in)
Dimensions of outer container	Length: 5,068 mm (199.53 in) Width: 720 mm (28.35 in) Height 742 mm (29.21 in) (including bolsters)

#### 1.2.1.4. **Materials and Component Dimensions**

##### 1.2.1.4.1. ***Inner Container***

The materials and component dimensions of the inner container are shown in Appendix 1.4.1.

##### 1.2.1.4.2. ***Outer Container***

The materials and component dimensions of the outer container are shown in Appendix 1.4.1.

#### 1.2.1.5. **Criticality Control Features**

The TN-B1 package does not require specific design features to provide neutron moderation and absorption for criticality control. The contents of the package rely on gadolinia loading for criticality control based on enrichment. Gadolinia loading requirements are provided in Table 6-1 TN-B1 Fuel Assembly Loading Criteria. There are no spacers required for criticality control. Fissile materials in the payload are limited to an amount that ensures safely sub-critical packages for both NCT and HAC. Further discussion of criticality control features is provided in Chapter 6.0.

#### 1.2.1.6. **Heat Transfer Features**

The unirradiated fuel has negligible decay heat, therefore, the TN-B1 package is not designed for dissipating heat. The packaging is designed to protect the fuel and its containment by

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providing containment during the Hypothetical Accident Conditions (HAC). A more detailed discussion of the package thermal characteristics is provided in Chapter 3.0

#### 1.2.1.7. **Coolants**

Due to the passive design of the TN-B1 package with regard to heat transfer, there are no coolants utilized within the TN-B1 package.

#### 1.2.1.8. **Protrusions**

The only significant protrusions on the TN-B1 packaging exterior are those associated with the lifting features on the outer container exterior. These are the sling holding angles and the bolsters at the bottom of the packaging. The bolsters protrude the furthest at 80 mm (3.15 in).

The only significant protrusions on the inner container exterior are the lifting sling fittings and the tightening blocks that are used for securing the lid. There are lifting sling fittings on the body and the main lid. Each of the sling fittings fold down so they protrude only the thickness of the lifting rod or bail.

#### 1.2.1.9. **Lifting and Tie-down Devices**

The lifting devices for the TN-B1 consist of the sling holding angles on the outer container which keep the slings from moving when used to sling the container during handling. The loaded container is designed to use four slings that form basket hitches under the container. The empty container is handled with two slings. The package may also be handled by the use of a forklift. The sling hold angles are designed so that even if they failed it would not affect the performance of the package.

The inner container is handled by the use of a series of lifting sling fittings. They are attached in a manner that even if they fail it will not compromise the performance of the inner container. On both the inner and outer containers, the lid lifting devices are marked to ensure proper use. A detailed discussion of lifting and tie-down designs, with corresponding structural analyses, is provided in Section 2.4.1 and 2.4.2.

#### 1.2.1.10. **Shielding**

Due to the nature of the unirradiated fuel payload, no biological shielding is necessary or provided by the TN-B1 packaging.

#### 1.2.1.11. **Packaging Markings**

The packaging will be marked with its model number, serial number, gross weight and also with the package identification number assigned by the NRC.

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### 1.2.2. *Containment System*

The containment system components are identified above in Section 1.2.1 and accompanying figures. The primary containment boundary of this package is the fuel rod cladding as shown in example Figure 1-6 Example Fuel Rod (Primary Containment). The fuel rod is completed by loading the uranium dioxide pellets into a zirconium alloy cladding tube. The tubes are pressurized with helium and zirconium end plugs are welded to the tube which effectively seals and contains the radioactive material. Welds of the fuel rods are verified for integrity by such means as X-ray inspection, ultrasonic testing, or process control. A representative nominal internal pressure of fuel rods at room temperature conditions is 1.1 MPa (160 psia) (absolute pressure). The TN-B1 package cannot be opened unintentionally. Both the OC and IC lids are attached to their respective bodies with socket-headed cap screws. There are twenty-four bolts holding the outer lid in place. There are no other openings in the outer container. The inner container has ten bolts holding the main lid in place and four bolts holding the end closure in place. Thus, the requirements of 10 CFR 71.43(c) are satisfied.



**Figure 1-6 Example Fuel Rod (Primary Containment)**


#### 1.2.2.1. **Pressure Relief System**

There are no pressure relief systems included in the TN-B1 package design to relieve pressure from within either the inner or outer containers or the fuel rod. Fire-consumable fusible plugs are used on the exterior surface of both the outer and inner containers to prevent pressure build up from the insulating and shock absorbing material during a fire event. These fusible plugs may be made of plastic. Two plugs are installed in the outer container body and two in the outer container lid. Four are installed in the inner container body, one in the end lid and two in its main lid.

### 1.2.3. *Contents*

A maximum of two fuel assemblies are placed in each packaging, see Table 6-1. The packaging is designed and analyzed to ship fuel configured either in 8x8, 9x9, 10x10 or 11x11 arrays or as loose rods contained in a cylinder, protective case or positioned in one or both sides of the inner container, see Table 6-2. Fuel assemblies may also be shipped in the BWR fuel channel. The nuclear fuel pellets located in rods and contained in the packaging are uranium oxide (UO<sub>2</sub>). The fuel assembly average enrichment is less than or equal to 5.0%



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U-235 (the fuel rod maximum enrichment is less than or equal to 5.0% U-235). In addition to the shipment of fuel assemblies, Section 1.2.3.4.5, Section 1.2.3.4.6 and Section 1.2.3.4.7 describe contents configurations for shipping individual fuel rods not contained in a fuel assembly.

Where fuel rods are referenced as being loaded with uranium dioxide mixed with gadolinium oxide (hereinafter gadolinia) the pellets in the gadolinia fuel rods contain a minimum of 2.0% gadolinium.

#### 1.2.3.1. **Type A contents**

Where the contents of the packaging is commercial grade uranium or other uranium materials where the A2 value is not exceeded, the packaging may be considered to contain Type A quantities.

#### 1.2.3.2. **Type B contents**


Where the contents of the packaging is enriched reprocessed uranium or other origin uranium not exceeding the values in Table 1-3, the packaging is considered to contain Type B quantities.

#### 1.2.3.3. **Quantity of Radioactive Materials of Main Nuclides**

Where the content of the packaging consists of Type B quantities of material, the main nuclides are treated as shown in Tables 1-2 through 1-4 to calculate total activity, activity fractions and A2 for the mixture.

**Table 1-2 Quantity of Radioactive Materials (Type A and Type B)**

Characteristic	Fuel Assembly Materials			
	Type 8×8	Type 9×9	Type 10×10	Type 11×11
Main nuclides	Low enriched uranium less than or equal to 5% U-235			
State of uranium	Uranium oxide ceramic pellet (Solid)			
Fuel assembly average enrichment	5.0% maximum			
Fuel rod maximum enrichment	5.0% maximum			
Number of fuel rods containing gadolinia	See Table 6-1			
Weight of uranium dioxide pellets (per fuel assembly)	235 kg	240 kg	275 kg	281 kg

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**Table 1-3 Type B Quantity of Radioactive Material**

<b>Isotope</b>	<b>Maximum content<sup>1</sup></b>	<b>Maximum mass, g</b>	<b>Specific Activity<sup>2</sup>, TBq/g</b>	<b>Total Activity, TBq</b>	<b>Total Activity, Ci</b>
U-232	2.00E-09 g/gU	9.92E-04	0.83	8.23E-04	2.23E-02
U-234	2.00E-03 g/gU	9.92E+02	2.30E-04	2.28E-01	6.17E+00
U-235	5.00E-02 g/gU	2.48E+04	8.00E-08	1.98E-03	5.36E-02
U-236	2.50E-02 g/gU	1.24E+04	2.40E-06	2.98E-02	8.04E-01
U-238	9.23E-01 g/gU	4.58E+05	1.20E-08	5.49E-03	1.48E-01
NP-237	1.66E-06 g/gU	8.23E-01	2.60E-05	2.14E-05	5.79E-04
PU-238	6.20E-11 g/gU	3.08E-05	6.30E-01	1.94E-05	5.24E-04
PU-239	3.04E-09 g/gU	1.51E-03	2.30E-03	3.47E-06	9.37E-05
PU-240	3.04E-09 g/gU	1.51E-03	8.40E-03	1.27E-05	3.42E-04
Gamma Emitters <sup>3</sup>	5.18E+05 MeV-Bq/kgU	N/A	N/A	2.57E-02	6.94E-01
			<b>Total</b>	<b>2.92E-01</b>	<b>7.89E+00</b>

1. Based on a maximum payload of 281 kg UO<sub>2</sub> per assembly, 248 kg U (562 kg UO<sub>2</sub>, 496 Kg U total).
2. 10CFR71, Appendix A
3. Assuming gamma energy of 0.01 MeV to maximize total content.

**Table 1-4 Isotopes and A2 Fractions**

Isotope	Maximum Radioactivity content (Ci)	10CFR71 A <sub>2</sub> per isotope (Ci)	Activity Fraction	A <sub>2</sub> Fraction
U-232	2.23E-02	0.0270	2.75E-03	1.02E-01
U-234	6.17E+00	0.1600	7.63E-01	4.77E+00
U-235	5.36E-02	Unlimited		
U-236	8.04E-01	0.1600	9.94E-02	6.22E-01
U-238	1.48E-01	Unlimited		
Np-237	5.79E-04	0.0540	7.18E-05	1.33E-03
Pu-238	5.24E-04	0.0270	6.50E-05	2.41E-03
Pu-239	9.37E-05	0.0270	1.16E-05	4.29E-04
Pu-240	3.42E-04	0.0270	4.23E-05	1.57E-03
Gamma Emitters	6.94E-01	0.5400	8.59E-02	1.59E-01
<b>Total</b>	<b>7.89E+00</b>		<b>Sum of A<sub>2</sub> fractions</b>	<b>5.65E+00</b>
<b>Mixture A<sub>2</sub></b>				<b>0.18 Ci</b>

#### 1.2.3.4. Physical Configuration

##### 1.2.3.4.1. Fuel Assembly

The configuration of typical fuel assemblies is shown in Figure 1-8 Fuel Assembly with Optional Packing Materials. The fuel assemblies may be of various model and type as long as they meet the requirements listed. The dimensions of the main components in the fuel assemblies are listed in Table 1-5. The maximum weight of contents including fuel and packing material is 684 kg (1,508 lb).

##### 1.2.3.4.2. Chemical Properties

Example of structural materials of the fuel assembly is shown in Table 1-6. Zirconium alloy, stainless steel and Ni-Cr-Fe alloy are chemically stable materials, and they are excellent in heat resistance and corrosion resistance.

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#### 1.2.3.4.3. ***Density of Materials***

The density for the fuel assembly materials is presented in Table 1-7.

#### 1.2.3.4.4. ***Packing Materials***

A number of packing materials may be used to guard the fuel assembly (e.g., cluster separators, and polyethylene bags). An example of the packing materials and their use is shown in Figure1-8.

#### 1.2.3.4.5. ***Bundled Fuel Rods***

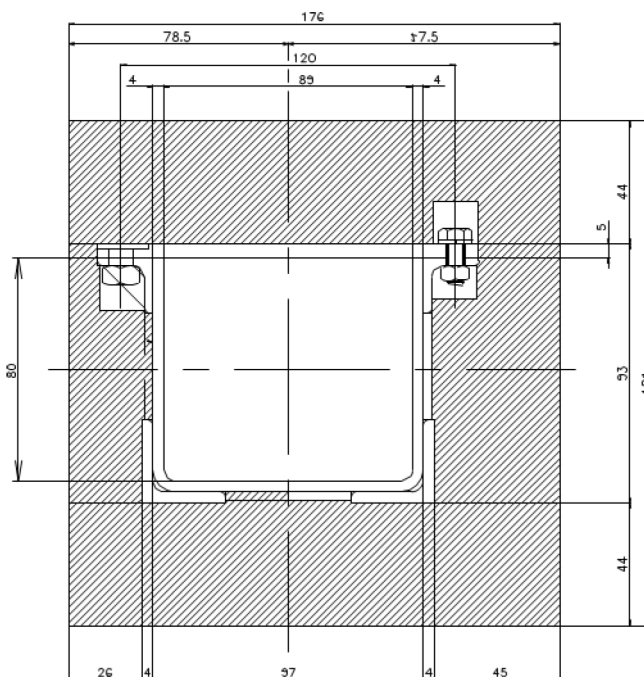
In addition to the fuel assembly configuration described above, fuel rods may be shipped bundled together in groups of rods up to 25 total rods. Fuel rods are fixed together using ring clamps. The criticality safety case for loose rods that shows that as many as 25 fuel rods per side can be arranged in any configuration within the volume of the inner container. Based on this criticality safety analysis the ring clamps are not relied on or needed for maintaining the configuration of the fuel rods.

#### 1.2.3.4.6. ***Fuel Rods In a 5-Inch Pipe***


Another physical configuration is the use of a 5-inch diameter schedule 40 stainless steel pipe. The physical configuration of the pipe is shown in drawing 0028B98. The number of fuel rods shipped in this configuration is limited by the quantities in Table 6-2. See Section 6.3.1.3.1 and 6.3.1.3.2 for other descriptions of the pipe.

#### 1.2.3.4.7. ***Fuel Rods in a Protective Case***

Figure 1-7 shows the configuration of the protective case. The protective case is a stainless steel box comprised of a body, lid, wood spacer absorber and end plate. In addition to the figure below, detailed drawings of the protective case are provided in Appendix 1.4.1. The protective case is surrounded by polyurethane foam cushioning material, which provides a snug fit within the inner container. Depending on the rod type, the protective case may be used to transport any number of authorized fuel rods up to a maximum of 30 rods. See Table 6-2.



### Figure 1-7 Protective Case

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**Table 1-5 Example of Fuel Structural Materials**

<b>Component parts</b>	<b>Structural materials</b>
Pellets	Uranium dioxide sintered (in some cases uranium dioxide blended with gadolinia)
Cladding tube	Zirconium alloy, metallic zirconium
Internal spring	Stainless steel
Getter	Zirconium alloy and stainless steel
Upper and Lower end plug	Zirconium alloy
Water rod	Zirconium alloy
Upper and Lower tie plate	Stainless steel
Spacer	Zirconium alloy
Finger spring	Ni-Cr-Fe alloy
Expansion spring	Ni-Cr-Fe alloy
Nut	Stainless steel and Zirconium alloy
Locking tab washer	Stainless steel

**Table 1-6 Density of Structural Materials**

<b>Main structural materials</b>	<b>Density</b>
Zirconium alloy metallic zirconium	Approximately 6.5 g/cm <sup>3</sup> (0.235lb/in <sup>3</sup> )
Uranium dioxide pellet	Approximately 10.4 g/cm <sup>3</sup> (0.376 lb/in <sup>3</sup> )
Stainless steel	Approximately 7.8 g/cm <sup>3</sup> (0.282 lb/in <sup>3</sup> )
Ni-Cr-Fe alloy	Approximately 8.5 g/cm <sup>3</sup> (0.307 lb/in <sup>3</sup> )

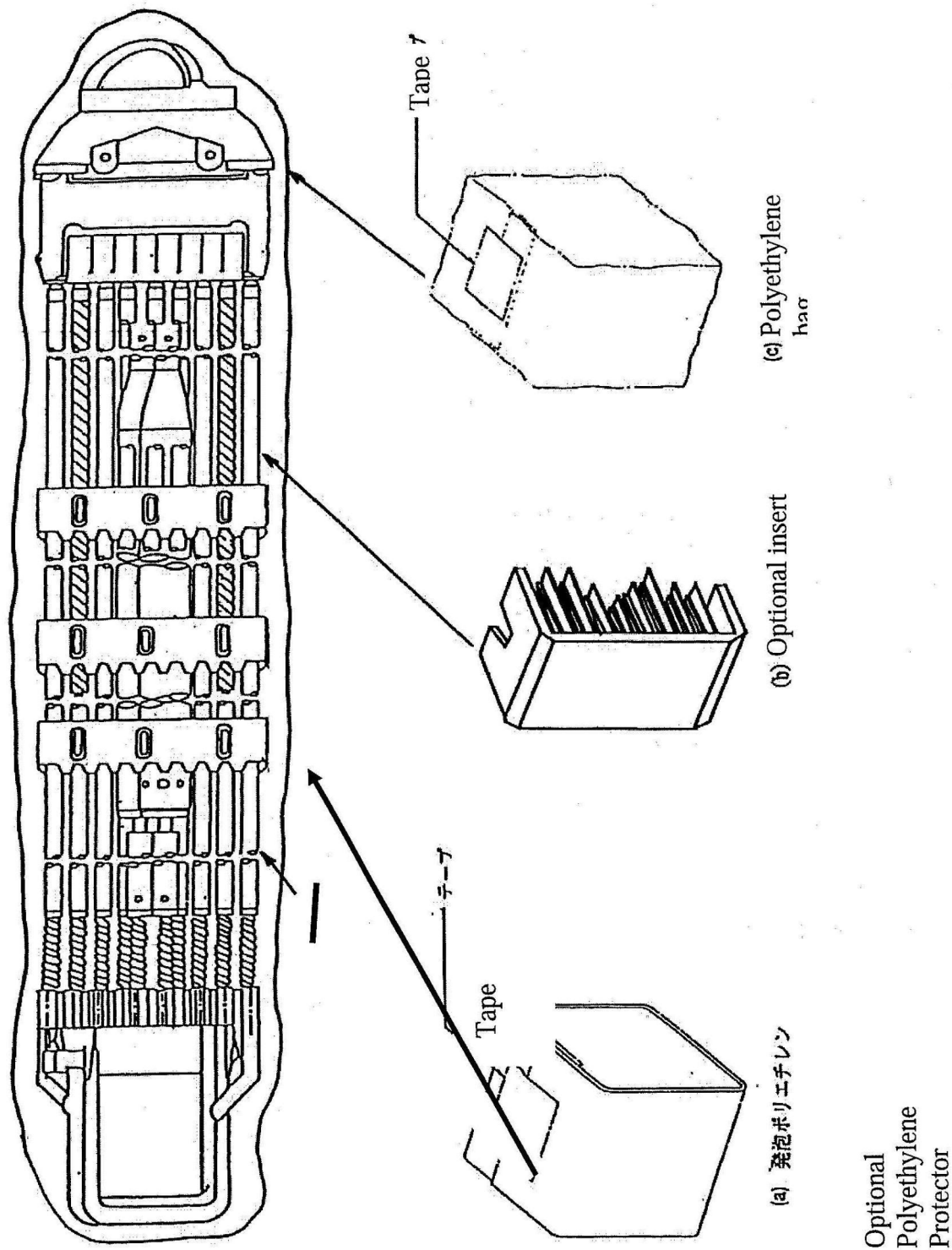



Figure 1-8 Assembly with Optional Packing Materials



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#### **1.2.4. *Operational Features***

The TN-B1 packaging is not considered operationally complex. Operational features are readily apparent from an inspection of the drawings provided in Appendix 1.4.1 and the previous discussions presented in Section 1.2.1. Operational procedures and instructions for loading, unloading, and preparing empty TN-B1 packages for transport are provided in Chapter 7.0

### **1.3. GENERAL REQUIREMENTS FOR ALL PACKAGES**

#### **1.3.1. *Minimum Package Size***

The TN-B1 package is a rectangular box that is 742 mm (29.21 in) high by 720 mm (28.35 in) wide by 5,068 mm (199.53 inches) long. Thus, the requirement of 10 CFR 71.43(a) is satisfied.


#### **1.3.2. *Tamper-Indicating Feature***

Seal pins are provided at each end of the outer container body and lid for the use of tamper indicating seals. A tamper indicating seal is attached at each end of the loaded outer container by inserting the seal through the holes in the body and lid seal pins and securing the seal. The tamper indicating seal is not readily breakable and would provide evidence of tampering or opening by an unauthorized person. Thus, the requirement of 10 CFR 71.43(b) is satisfied.

### **1.4. APPENDIX**

#### **1.4.1. *TN-B1 General Arrangement Drawings***


This section presents the TN-B1 packaging general arrangement drawing consisting of 15 drawings entitled, *TN-B1 SAR Drawing*, see drawing list below. Within the packaging general arrangement drawing, dimensions important to the packaging safety are dimensioned and toleranced. Other dimensions are provided as a reference dimension, and are toleranced in accordance with the JIS (Japan Industrial Std.) B 0405. See 2.1.4.1 and 2.1.4.2.

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#### 1.4.1.1. Drawing List

**Table 1-7 Outer Container Drawings**

<b>Drawing number</b>	<b>Number of Sheets</b>	<b>Revision #</b>	<b>Name</b>
105E3737	1	6	Outer/Inner Container Assembly Licensing Drawings
105E3738	2 (Sheets 1&2)	8	Outer Container Main Body Assembly Licensing Drawings
105E3738	1 (Sheet 3)	7	Outer Container Main Body Assembly Licensing Drawing
105E3739	1	4	Outer Container Fixture Assembly Licensing Drawings
105E3740	1	4	Outer Container Fixture Assembly Installation Licensing Drawings
105E3741	1	1	Outer Container Shock Absorber Assembly Licensing Drawings
105E3742	1	3	Outer Container Bolster Assembly Licensing Drawings
105E3743	1	5	Outer Container Lid Assembly Licensing Drawings
02-9162717	1	1	Outer Container Marking Licensing Drawings

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**Table 1-8 Inner Container Drawings**

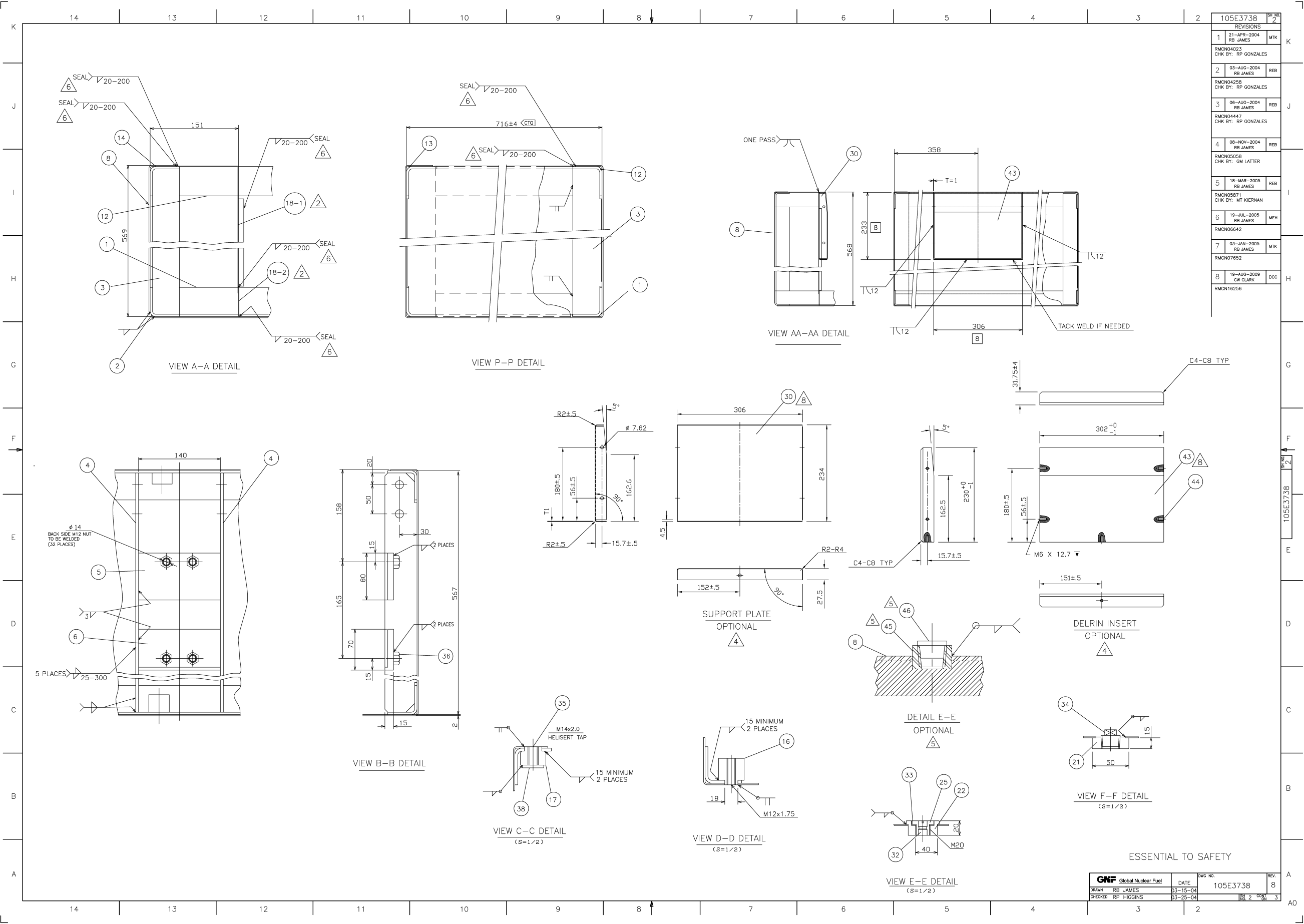
<b>Drawing number</b>	<b>Number of Sheets</b>	<b>Revision #</b>	<b>Name</b>
105E3745	4	8	Inner Container Main Body Assembly Licensing Drawings
105E3746	1	1	Inner Container Parts Assembly Licensing Drawings
105E3747	1	4	Inner Container Lid Assembly Licensing Drawings
105E3748	1	2	Inner Container End Lid Assembly Licensing Drawings
02-9162722	1	1	Inner Container Marking Licensing Drawings

**Table 1-9 Contents Drawings**

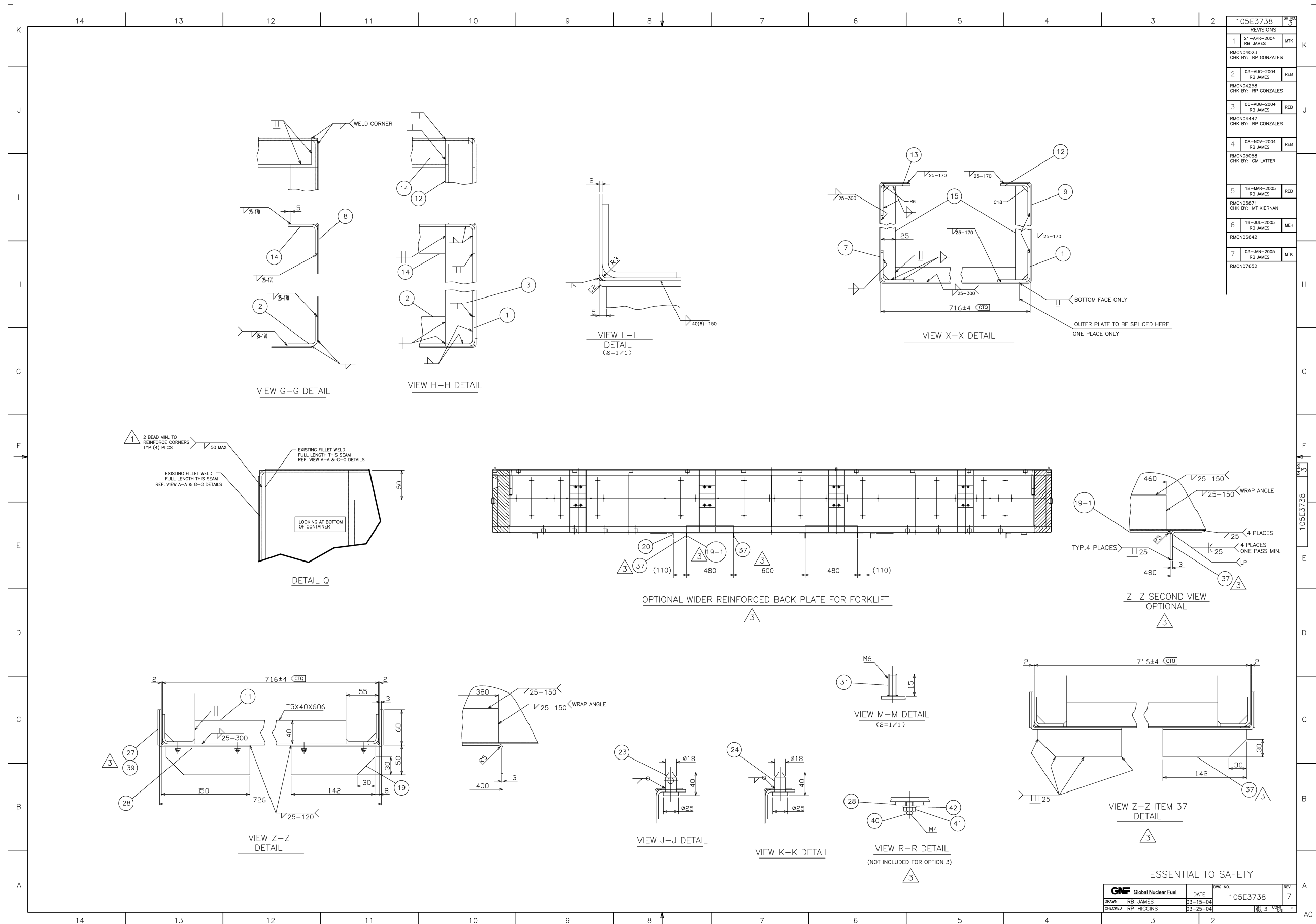
<b>Drawing number</b>	<b>Number of Sheets</b>	<b>Revision #</b>	<b>Name</b>
105E3773	1	1	Protective Case
0028B98	1	1	Shipping Container Loose Fuel Rods





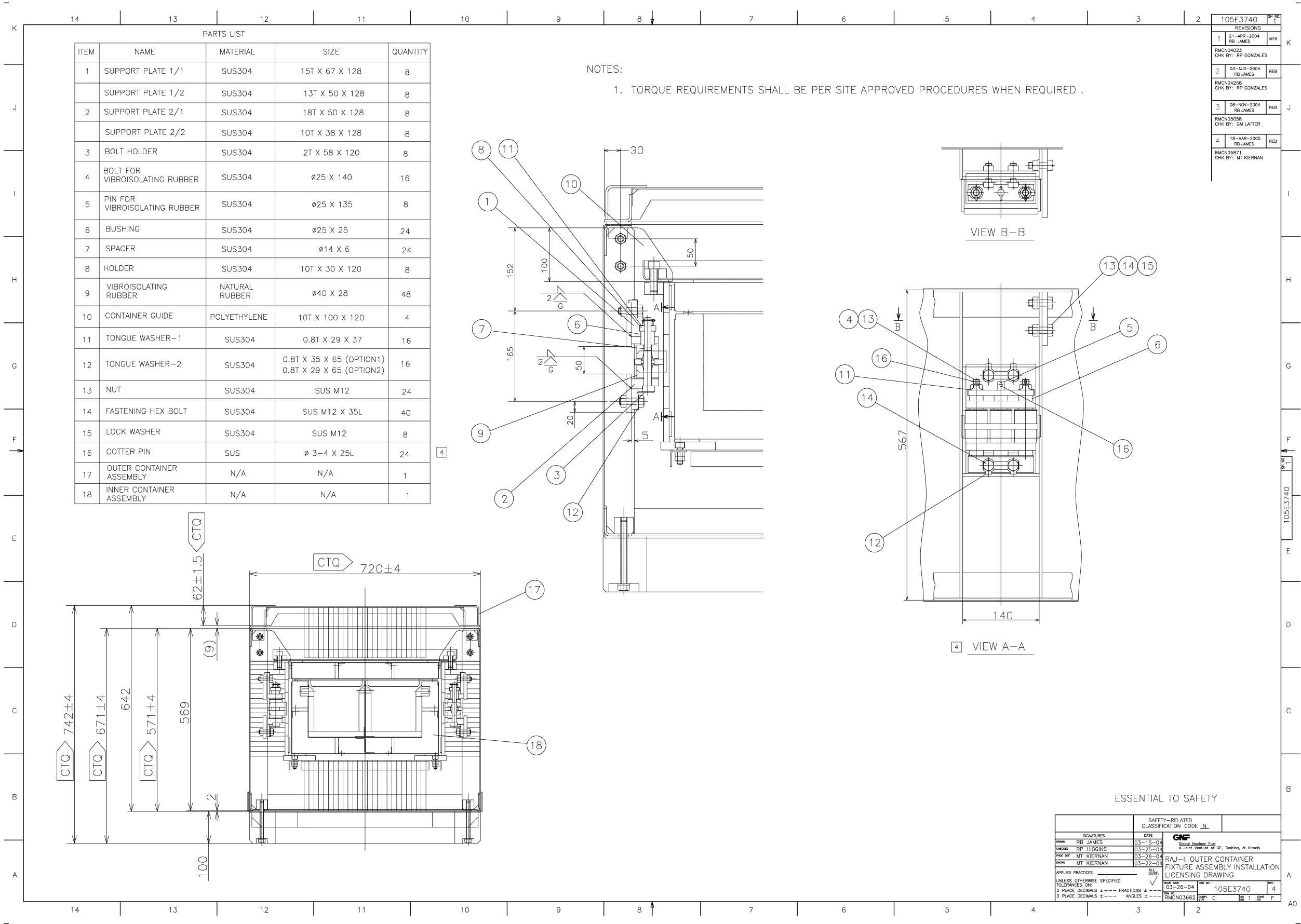


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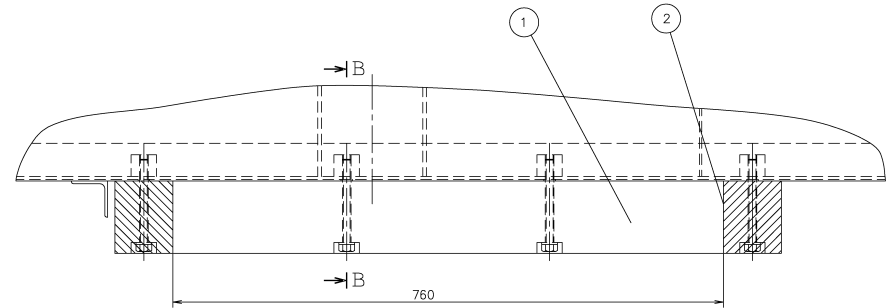
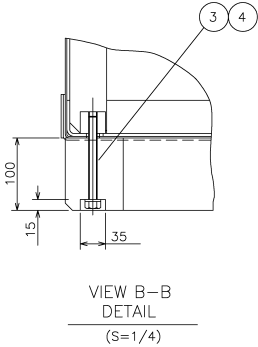




- NOTES:
1. TORQUE REQUIREMENTS SHALL BE PER SITE APPROVED PROCEDURES WHEN REQUIRED.
2. INSTALL BOLSTER FASTENING BOLT (ITEM #3) USING A THREAD LOCK COMPOUND OR A LICENSE HOLDER APPROVED LOCKWASHER.

105E3742		REV NO.
REVISIONS		
1	21-APR-2004 RB JAMES	MTK
RMCN04023 CHK BY: RP GONZALES		
2	17-MAR-2005 RB JAMES	REB
RMCN05871 CHK BY: MT KIERNAN		
3	19-JUL-2005 RB JAMES	MEH
RMCN06642		

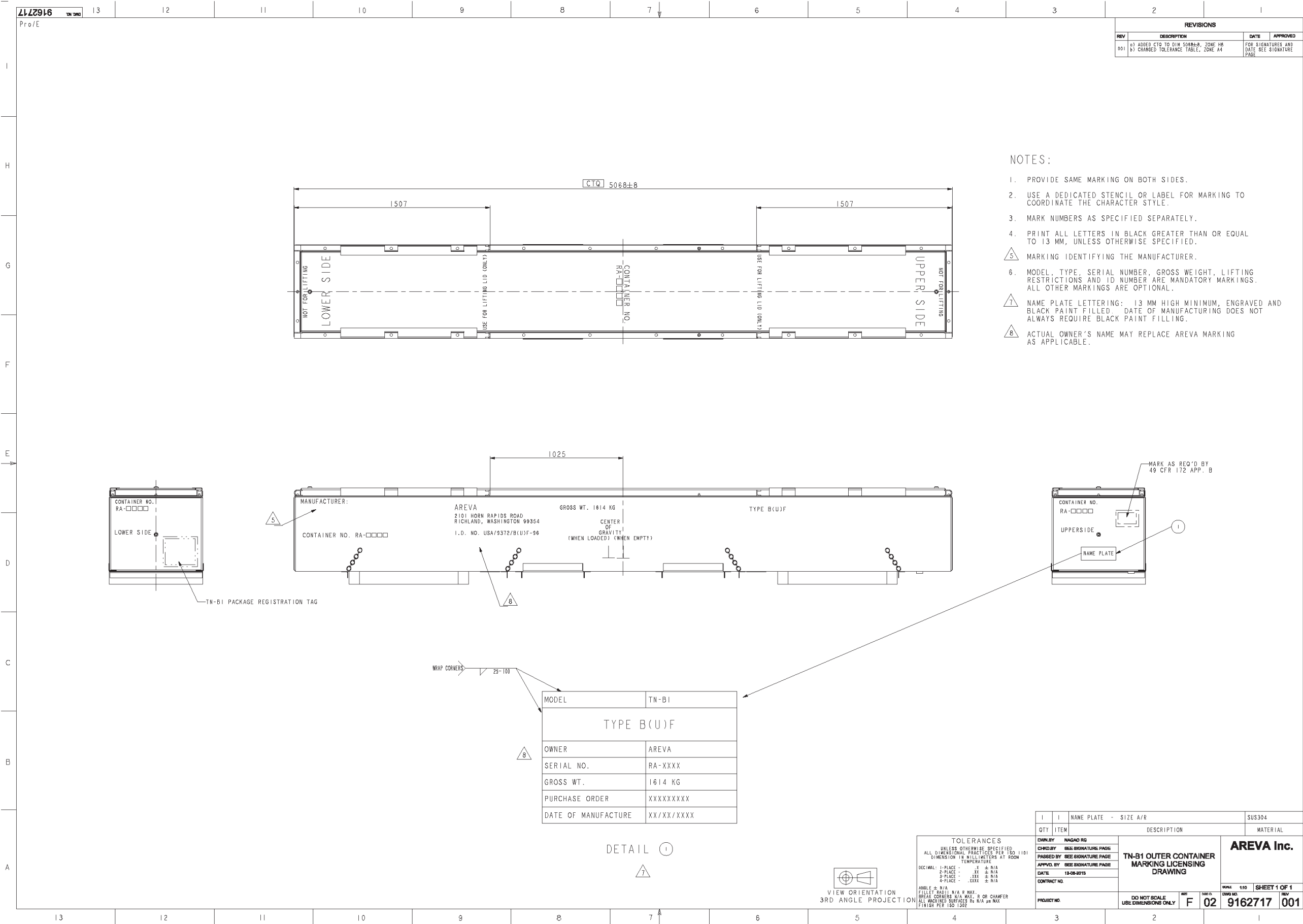
PARTS LIST				
ITEM	NAME	MATERIAL	SIZE	QUANTITY
1	BOLSTER-1	LUMBER (HEMLOCK) OR SUPERIOR STRENGTH WOOD	80t X 100 X 758	4
2	BOLSTER-2	LUMBER (HEMLOCK) OR SUPERIOR STRENGTH WOOD	80t X 100 X 720	4
3	BOLSTER FASTENING HEXAGONAL BOLT	SUS304	M12 X 115L	16
4	WASHER	SUS304	SUS M12	16



ESSENTIAL TO SAFETY

SIGNATURES		DATE	SAFETY-RELATED CLASSIFICATION CODE <u>N</u>	
DRAWN	RB JAMES	03-15-04	<b>GNF</b> Global Nuclear Fuel A Joint Venture of GE, Toshiba, & Hitachi  RAJ-II OUTER CONTAINER BOLSTER ASSEMBLY LICENSING DRAWINGS	
CHECKED	RP HIGGINS	03-25-04		
PROD REV	MT KIERNAN	03-26-04		
DESIGN	MT KIERNAN	03-22-04		
APPLIED PRACTICES		DATE: 03-26-04	REV NO. 105E3742	3
UNLESS OTHERWISE SPECIFIED		DATE: RMCN03662	REV NO. C	F
TOLERANCES ON:		2 PLACE DECIMALS ± --- FRACTIONS ± ---		
		3 PLACE DECIMALS ± --- ANGLES ± ---		

ALL UNIT IN MILLIMETERS		SAFETY-RELATED CLASSIFICATION CODE <u>N</u>		REVIEWED FOR CTQ
		DATE		
SIGNATURES	0.3-15-04		<b>GNF</b> General Nuclear Energy A Joint Venture of GE, Toshiba, & Hitachi	
DRAWN RB JAMES	0.3-25-04			
CHECKED RP HIGGINS	0.3-25-04			
DESIGNED MT KIERNAN	0.3-26-04			
INCHES MT KIERNAN	0.3-26-04			
APPLIED PRACTICES:		RAJ II OUTER CONTAINER LID ASSEMBLY LICENSING DRAWING		
UNLESS OTHERWISE SPECIFIED				
TOLERANCES ON:				
FRACTIONS ± ± ± FRACTIONS ± ± ±				
PLACES DECIMALS ± ± ± ANGLES ± ± ±				
STANDARD		REV. DATE		REV.
ASME Y14.5M-2009		0.3-26-04		5
FIG. NO.		NO. 1053743		
PROJECT NO. 80003662		BY J. SONG		





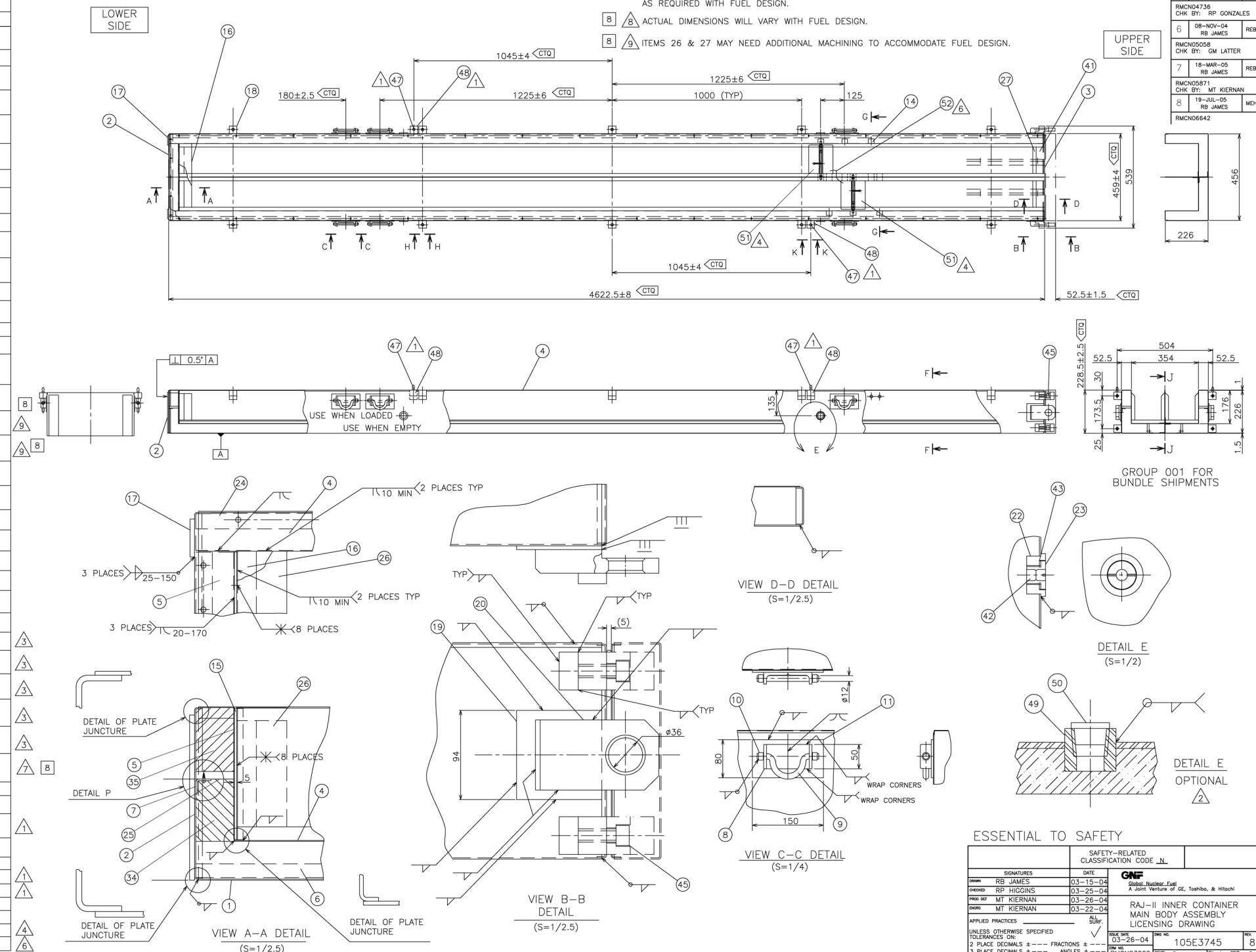
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ITEM	NAME	PARTS LIST		
		MATERIAL	SIZE	QUANTITY
1	BODY OUTER SHELL-1	SUS304	1.5T X 900 X 4612	1
2	BODY OUTER SHELL-2	SUS304	1.5T X 244 X 474	1
3	BODY OUTER SHELL-3	SUS304	1.5T X 244 X 474	1
4	BODY INNER WALL-1	SUS304	1T X 830 X 4612	1
5	BODY INNER WALL-2	SUS304	1T X 231.5 X 374	1
6	SPACER LONG	SUS304	2T X 20 X 20 X 4619	4
7	SPACER SHORT	SUS304	2T X 20 X 20 X 416	1
8	BODY SLING FITTING	SUS304	1.50 X 50 X 6T X 50	12
9	SLING FITTING	SUS304	ø12 X 165	6
10	SLING FITTING COLLAR	SUS304	ø 16 X 10	12
11	BODY SLING FITTING PROTECTING PLATE	SUS304	4T X 80 X 150	6
12	PARTITION	SUS304	2T X 202 X 4567	1
13	PARTITION ANGLE	SUS304	2T X 30 X 30 X 56	7
14	HOLD DOWN BAR BOSS	SUS304	ø32 X 25	6
15	INNER WALL BOTTOM PLATE	SUS304	3T X 174 X 350	1
16	LOWER END FACE LUMBER FASTENING PLATE	SUS304	1T X 70 X 355	1
17	LEG PLATE	SUS304	6T X 50 X 218	2
18	BOSS (BODY TIGHTENING BLOCK)	SUS304	40T X 40 X 50	14
19	VERTICAL SLING FITTING PROTECTIVE PLATE	SUS304	4T X 94 X 94	2
20	VERTICAL SLING FITTING	SUS304	25T X 74 X 130	2
21	SUPPORT PLATE	SUS304	1.5T X 30 X 50 X 30 X 104	7
22	FUSIBLE PLUG SEAT	SUS304	ø45 X 20	4
23	FUSIBLE PLUG	SUS304	ø30 X 20	4
24	THERMAL INSULATOR LUMBER LONG (BODY)	LUMBER (HEMLOCK)	10T X 50 X 4619	4
25	THERMAL INSULATOR LUMBER SHORT (BODY)	LUMBER (HEMLOCK)	10T X 50 X 356	1
26	END FACE LUMBER (BOTTOM)	LUMBER (HEMLOCK)	140 X 140 THICKNESS AS REQD	2
27	END FACE LUMBER (TOP) OPTIONAL	LUMBER (HEMLOCK)	140 X 140 THICKNESS AS REQD	2
28	THERMAL INSULATOR-1	ALUMINA SILICATE	48T X 80 X 4619	2
29	THERMAL INSULATOR-2	ALUMINA SILICATE	48T X 83 X 4619	2
30	THERMAL INSULATOR-3	ALUMINA SILICATE	48T X 81 X 4619	1
31	THERMAL INSULATOR-4	ALUMINA SILICATE	48T X 81 X 4619	1
32	THERMAL INSULATOR-5	ALUMINA SILICATE	48T X 130 X 4619	1
33	THERMAL INSULATOR-6	ALUMINA SILICATE	48T X 130 X 4619	1
34	THERMAL INSULATOR-7	ALUMINA SILICATE	48T X 80 X 351	1
35	THERMAL INSULATOR-8	ALUMINA SILICATE	48T X 82 X 351	1
36	SHOCK ABSORBER-1	POLYETHYLENE FOAM	18T X 145 X 4562	1
37	SHOCK ABSORBER-2	POLYETHYLENE FOAM	18T X 145 X 4562	1
38	SHOCK ABSORBER-3	POLYETHYLENE FOAM	18T X 145 X 4562	1
39	SHOCK ABSORBER-4	POLYETHYLENE FOAM	18T X 145 X 4562	1
40	SHOCK ABSORBER-5	POLYETHYLENE FOAM	18T X 176 X 4562	2
41	SHOCK ABSORBER-6	POLYETHYLENE FOAM	140 X 140 THICKNESS AS REQD	2-8
42	BISMUTH	PURITY 99.99%	SPECIFIC GRAVITY 9.8	4
43	FUSIBLE PLUG GASKET	ASBESTOS	1.5T X ø21 X ø30	4
44	HELISERT	PHOSPHOR BRONZE	M14 X 2.0	16
45	LID CLAMP BOLT	SUS304	SUS M14 X 35L	14
46	ROUND WOOD SCREW	SUS304	SUS ø3.1 X 10L	135
47	ALIGNMENT PIN	SUS304	ø16 X 71.5L (M14)	2
48	ALIGNMENT BLOCK	SUS304	40T X 30 X 50	2
49	HALF COUPLING	304 S/S	1/2 - 14 NPT	4
50	PIPE PLUG	PLASTIC	1/2 NPT	4
51	HOLD DOWN BAR	304 S/S	N/A	4
52	COATED S/S TETHER	304 S/S	AR	1

NOTES:

- ⑤ OPTIONAL ALIGNMENT PIN AND BLOCK (ITEM 47 AND 48) 2 PLACES.
- ⑥ OPTIONAL HALF COUPLING AND PLASTIC PIPE PLUG (ITEMS 49 & 50) IN LIEU OF FUSIBLE PLUG 4 PLACES.
- ⑦ POLYETHYLENE FOAM THICKNESS MAY BE REDUCED TO 13 MILLIMETERS MINIMUM FOR THE INNER CONTAINER BODY TO ACCOMMODATE CHANNEL FUEL SHIPMENT.
- ⑧ OPTIONAL HOLD DOWN BAR (ITEM 51).
5. GLUE CUSHIONING MATERIAL AND GASKET WITH 3M SCOTCH GRIP CONTACT ADHESIVE, 10 NEUTRAL OR EQUIVALENT.
- ⑥ OPTIONAL HOLD DOWN BAR CONNECTING TETHER.
- ⑧ ⑦ OPTIONAL SHOCK ABSORBER 1-6 (POLYETHYLENE FOAM, ITEMS 36-41) AND THE THERMAL INSULATOR LUMBER LONG "BODY" (LUMBER-HEMLOCK, ITEM 24) MAY BE INSTALLED IN MULTIPLE SECTIONS AS REQUIRED WITH FUEL DESIGN.
- ⑧ ⑧ ACTUAL DIMENSIONS WILL VARY WITH FUEL DESIGN.
- ⑧ ⑨ ITEMS 26 & 27 MAY NEED ADDITIONAL MACHINING TO ACCOMMODATE FUEL DESIGN.



105E3745		SR	NO
REVISONS			
1	21-APR-2004 RB JAMES	MTK	
RMCN04023 CHK BY: RP GONZALES			
2	22-APR-2004 RB JAMES	MTK	
RMCN04038 CHK BY: RP GONZALES			
3	03-04U-2004 RB JAMES	REB	
RMCN04258 CHK BY: RP GONZALES			
4	02-SEP-04 RB JAMES	REB	
RMCN04691 CHK BY: RP GONZALES			
5	15-SEP-04 RB JAMES	REB	
RMCN04736 CHK BY: RP GONZALES			
6	08-NOV-04 RB JAMES	REB	
RMCN05058 CHK BY: GM LATTER			
7	18-MAR-05 RB JAMES	REB	
RMCN05871 CHK BY: MT KIERMAN			
8	19-JUL-05 RB JAMES	MEH	
RMCN06642			

UPPER  
SIDE

LOWER  
SIDE

GROUP 001 FOR  
BUNDLE SHIPMENTS

DETAIL E  
(S=1/2)

DETAIL E  
OPTIONAL  
2

ESSENTIAL TO SAFETY

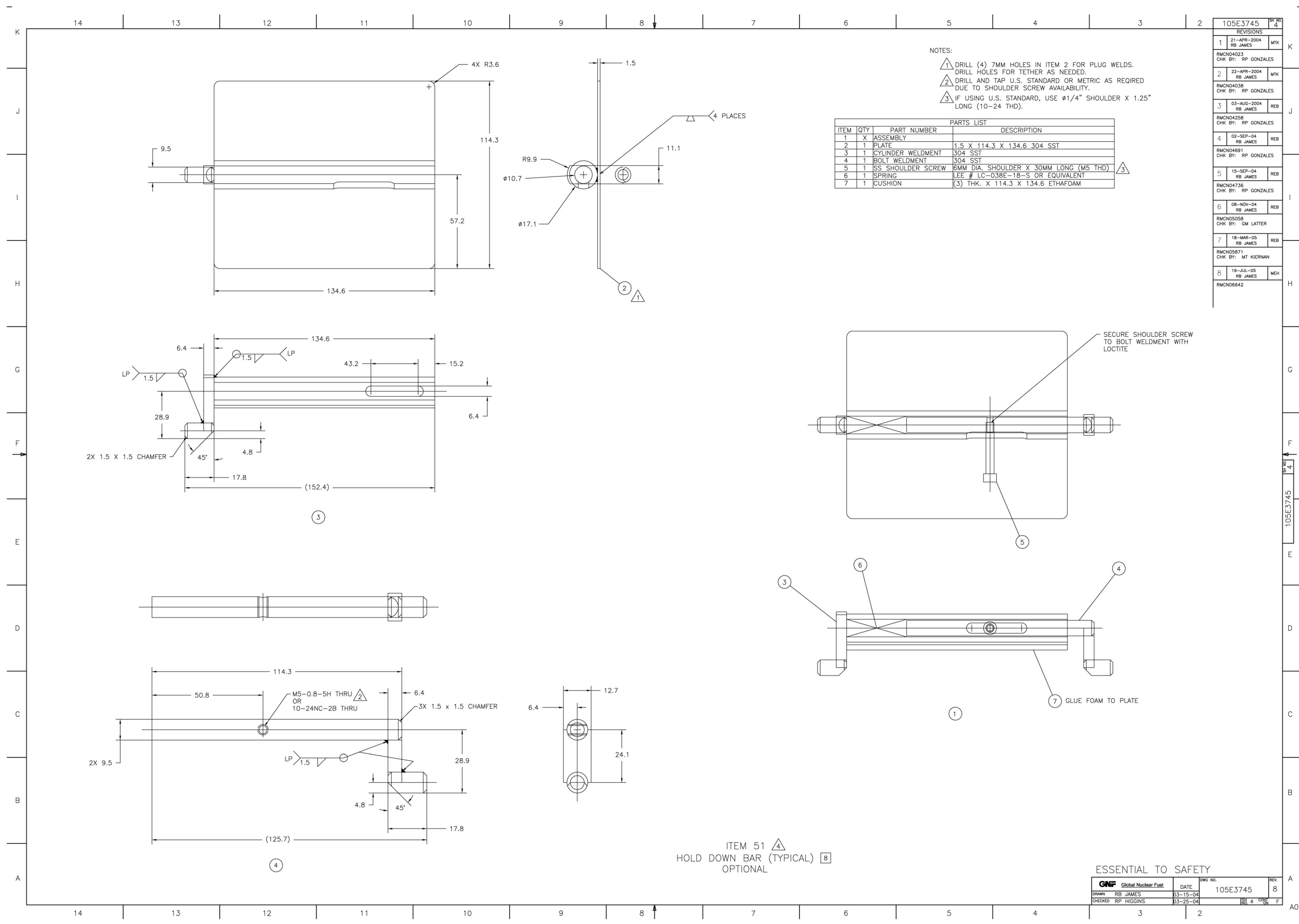
[illegible]







**AREVA TN-B1**  
**Docket No. 71-9372**  
**Safety Analysis Report**



**AREVA TN-B1**  
**Docket No. 71-9372**  
**Safety Analysis Report**



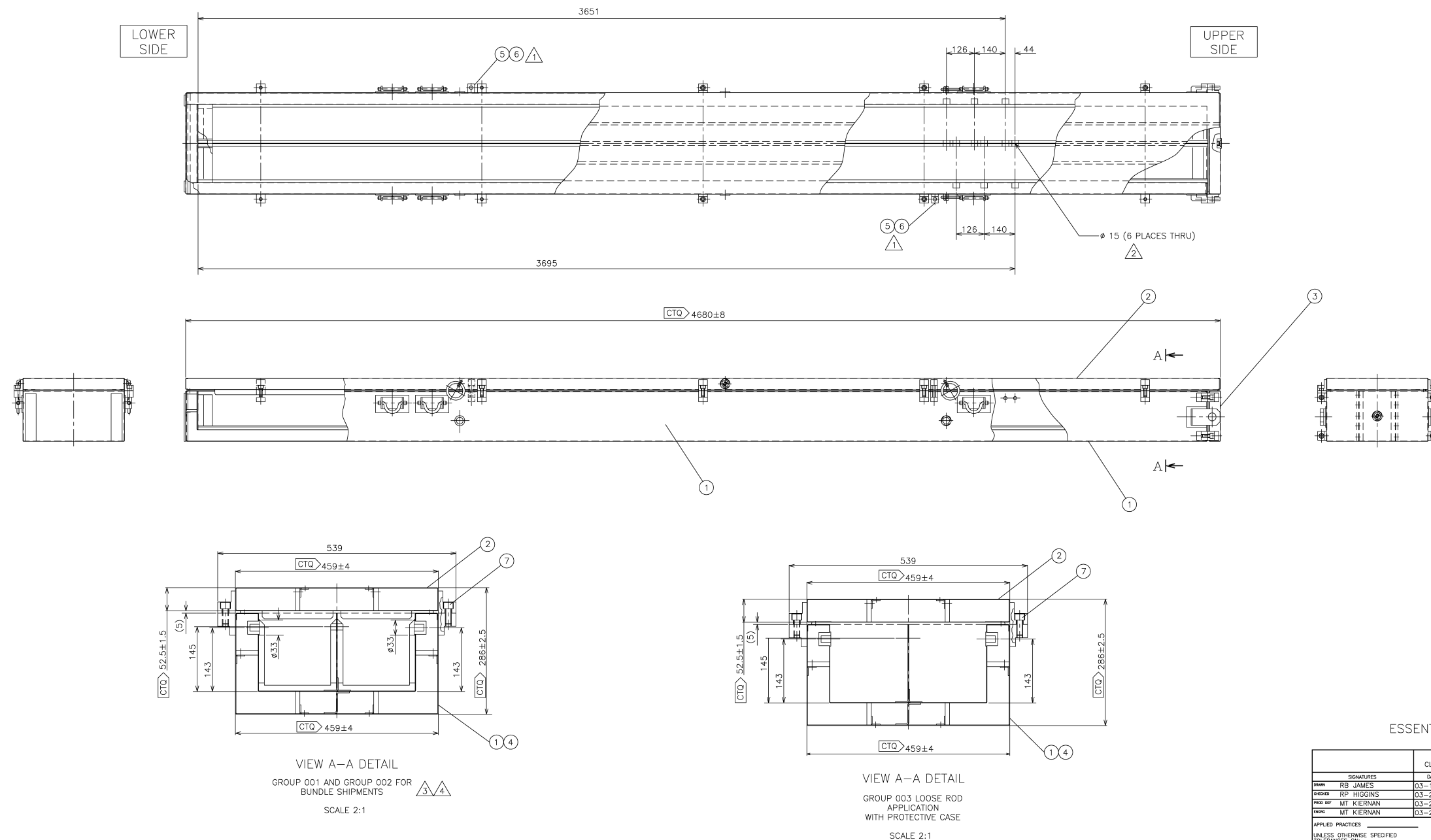
PARTS LIST				
ITEM	NAME	MATERIAL	SIZE	QUANTITY
1	INNER CONTAINER BODY ASSEMBLY 1/2	*	*	1
1	INNER CONTAINER BODY ASSEMBLY 2/2	*	*	1
2	INNER CONTAINER LID ASSEMBLY	*	*	1
3	INNER CONTAINER END LID ASSEMBLY	*	*	1
4	INNER CONTAINER MARKING	*	*	N/A
5	ALIGNMENT PIN	SUS304	ø16 X 71.5	2
6	ALIGNMENT BLOCK	SUS304	40T X 30 X 50	2
7	LID BOLT	SUS304	M14 X 35L	14

\* SEE DETAIL DRAWINGS


NOTES:

1. OPTIONAL ALIGNMENT PIN AND BLOCK (ITEM 5 AND ITEM 6) 2 PLACES.
2. CUT POLYETHYLENE FOAM OUT ABOVE BOSS AND PARTITION HOLES TO ALLOW FULL ENGAGEMENT OF HOLD DOWN BAR.
3. LOOSE RODS MAY BE SHIPPED UTILIZING THIS DESIGN PER LICENSED CONFIGURATIONS.
4. THE ONLY DIFFERENCE BETWEEN G001 AND G002 IS THE POLYETHYLENE FOAM CUTOUT AND THICKNESS.

105E3746		SH NO. 1
REVISIONS		
1	21-APR-2004 RB JAMES	MTK
RMCND4023 CHK BY: RP GONZALES		



ESSENTIAL TO SAFETY 1

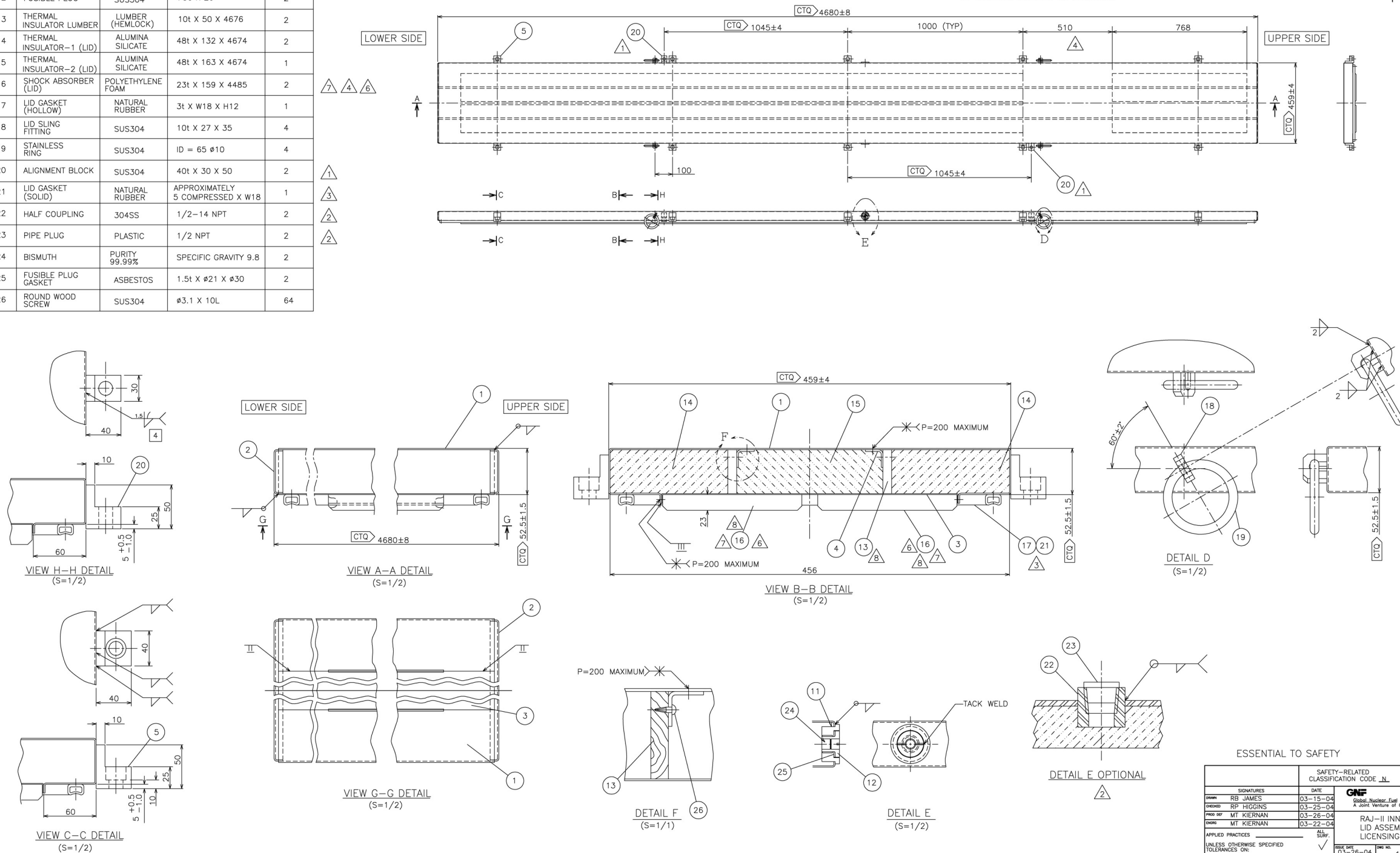
SIGNATURES		DATE		 General Nuclear Fuel A Joint Venture of GE, Toshiba, & Hitachi	
DESIGN	RB JAMES	03-15-04			
ENGINEER	REP HIGGINS	03-25-04			
DESIGN	MT KIERMAN	03-26-04			
DESIGN	MT KIERMAN	03-26-04			
APPLIED PRACTICES		ASME SECTION 5		PARTS ASSEMBLY LICENSING DRAWING	
UNLESS OTHERWISE SPECIFIED TOLERANCES ON:				DATE 03-26-04	DESIG 105E3746
1	2 PLACE DECIMALS ± ± ±	FRACTIONS ± ± ±		PROJ NO 000.003662	DESIG 105E3746
3	PLACE DECIMALS ± ± ±	ANGLES ± ± ±		SHEET NO 1	TOTAL SHEETS 1

ITEM	NAME	MATERIAL	SIZE	QUANTITY
1	LID OUTER SHELL-1	SUS304	1.5t x 690 X 4670	1
2	LID OUTER SHELL-2	SUS304	1.5t x 67.5 X 474	2
3	LID INNER WALL	SUS304	1t x 359 x 4670	1
4	SPACER (LID)	SUS304	2t x 36 x 4676	2
5	TIGHTENING BLOCK (LID)	SUS304	40t x 40 X 50	10
11	FUSIBLE PLUG SEAT	SUS304	ø45 X 20	2
12	FUSIBLE PLUG	SUS304	ø30 X 20	2
13	THERMAL INSULATOR LUMBER	LUMBER (HEMLOCK)	10t x 50 X 4676	2
14	THERMAL INSULATOR-1 (LID)	ALUMINA SILICATE	48t x 132 X 4674	2
15	THERMAL INSULATOR-2 (LID)	ALUMINA SILICATE	48t x 163 x 4674	1
16	SHOCK ABSORBER (LID)	POLYETHYLENE FOAM	23t x 159 x 4485	2
17	LID GASKET (HOLLOW)	NATURAL RUBBER	3t x W18 X H12	1
18	LID SLING FITTING	SUS304	10t x 27 X 35	4
19	STAINLESS RING	SUS304	ID = 65 ø10	4
20	ALIGNMENT BLOCK	SUS304	40t x 30 X 50	2
21	LID GASKET (SOLID)	NATURAL RUBBER	APPROXIMATELY 5 COMPRESSED X W18	1
22	HALF COUPLING	304SS	1/2"-14 NPT	2
23	PIPE PLUG	PLASTIC	1/2 NPT	2
24	BISMUTH	PURITY 99.99%	SPECIFIC GRAVITY 9.8	2
25	FUSIBLE PLUG GASKET	ASBESTOS	1.5t x ø21 x ø30	2
26	ROUND WOOD SCREW	SUS304	ø3.1 x 10L	64

NOTES:

1. OPTIONAL ALIGNMENT BLOCK (ITEM 20) 2 PLACES.
2. OPTIONAL HALF COUPLING AND PLASTIC PIPE PLUG (ITEM 22 & 23) IN LIEU OF FUSIBLE PLUG.
3. OPTIONAL SOLID GASKET MAY BE USED.
4. OPTIONAL CUT OUT OF POLYETHYLENE FOAM FOR HOLD DOWN BAR.
5. GLUE CUSHIONING MATERIAL AND GASKET WITH 3M SCOTCH GRIP CONTACT ADHESIVE, 10 NEUTRAL OR EQUIVALENT.
6. POLYETHYLENE FOAM ITEM 16 NOT REQUIRED FOR G003.
7. OPTIONAL REDUCTION IN THICKNESS BY 5MM FOR G002.
8. OPTIONAL SHOCK ABSORBER (POLYETHYLENE FOAM, ITEM 16) AND THE THERMAL INSULATOR LUMBER (LUMBER-HEMLOCK, ITEM 13) MAY BE INSTALLED IN MULTIPLE SECTIONS AS REQUIRED.

105E3747		SH #1
REVISONS		
1	21-APR-2004 RB JAMES	MTK
RMCND4023 CHK BY: RP GONZALES		
2	03-AUG-2004 RB JAMES	REB
RMCND4258 CHK BY: RP GONZALES		
3	08-NOV-2004 RB JAMES	REB
RMCND5058 CHK BY: GM LATTER		
4	17-MAR-2005 RB JAMES	REB
RMCND5871 CHK BY: MT KIERNAN		





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**Docket No. 71-9372**  
**Safety Analysis Report**

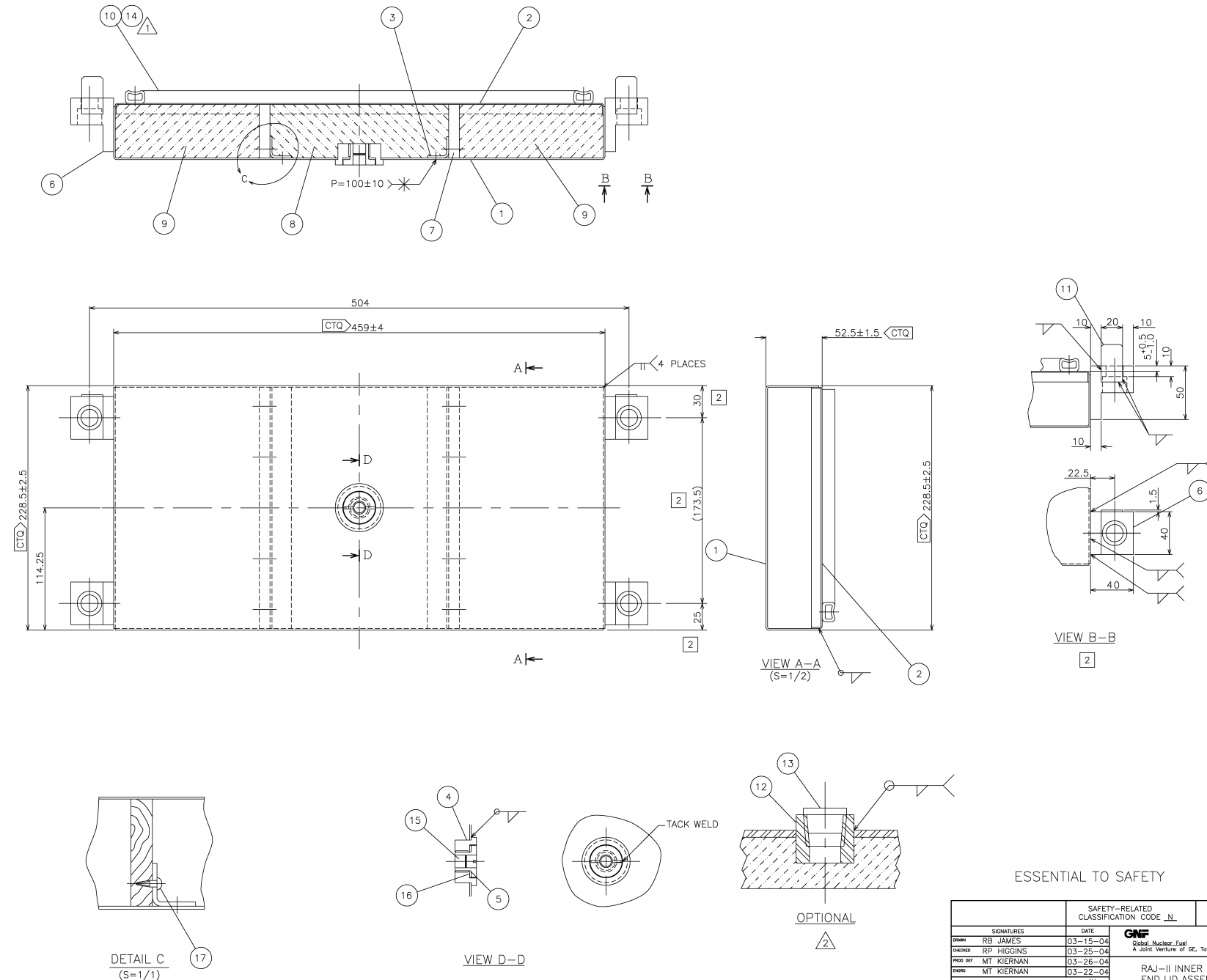


PARTS LIST				
ITEM	NAME	MATERIAL	SIZE	QUANTITY
1	END LID OUTER SHELL	SUS304	1.5t X 321.5 X 552	1
2	END LID INNER WALL	SUS304	1t X 245.5 X 476	1
3	SPACER (END LID)	SUS304	2t X 36 X 222	2
4	FUSIBLE PLUG SEAT	SUS304	ø45 X 20	1
5	FUSIBLE PLUG	SUS304	ø30 X 20	1
6	TIGHTENING BLOCK (END LID)	SUS304	40t X 40 X 50	4
7	THERMAL INSULATOR LUMBER (END LID)	LUMBER (HEMLOCK)	10t X 50 X 222	2
8	THERMAL INSULATOR-1 (END LID)	ALUMINA SILICATE	48t X 163 X 222	1
9	THERMAL INSULATOR-2 (END LID)	ALUMINA SILICATE	48t X 132 X 222	2
10	HOLLOW GASKET (END LID)	NATURAL RUBBER	3t X W18 X H12	1
11	FALL PREVENTION PLATE	SUS304	1.5t X 20 X 35	2
12	HALF COUPLING	SUS304	1/2 – 14 NPT	1
13	PIPE PLUG	PLASTIC	1/2 – 14 NPT	1
14	SOLID GASKET (END LID)	NATURAL RUBBER	APPROXIMATELY 5 COMPRESSED X W18	1
15	BISMUTH	PURITY 99.99%	SPECIFIC GRAVITY 9.8	1
16	FUSIBLE PLUG GASKET	ASBESTOS	1.5t X ø21 X ø30	1
17	ROUND WOOD SCREW	SUS304	ø3.1 X 10L	10


NOTES:

1. OPTIONAL SOLID GASKET MAY BE USED.
2. 2. OPTIONAL HALF COUPLING AND PLASTIC PIPE PLUG (ITEM 12 & 13) IN LIEU OF FUSIBLE PLUG.
3. 3. GLUE CUSHIONING MATERIAL AND GASKET WITH 3M SCOTCH GRIP CONTACT ADHESIVE, 10 NEUTRAL OR EQUIVALENT.

105E3748		SPT NO 1
REVISIONS		
1	21-APR-2004 RB JAMES	MTK
RMCN04023 CHK BY: RP GONZALES		
2	03-AUG-2004 RB JAMES	REB
RMCN04258 CHK BY: RP GONZALES		



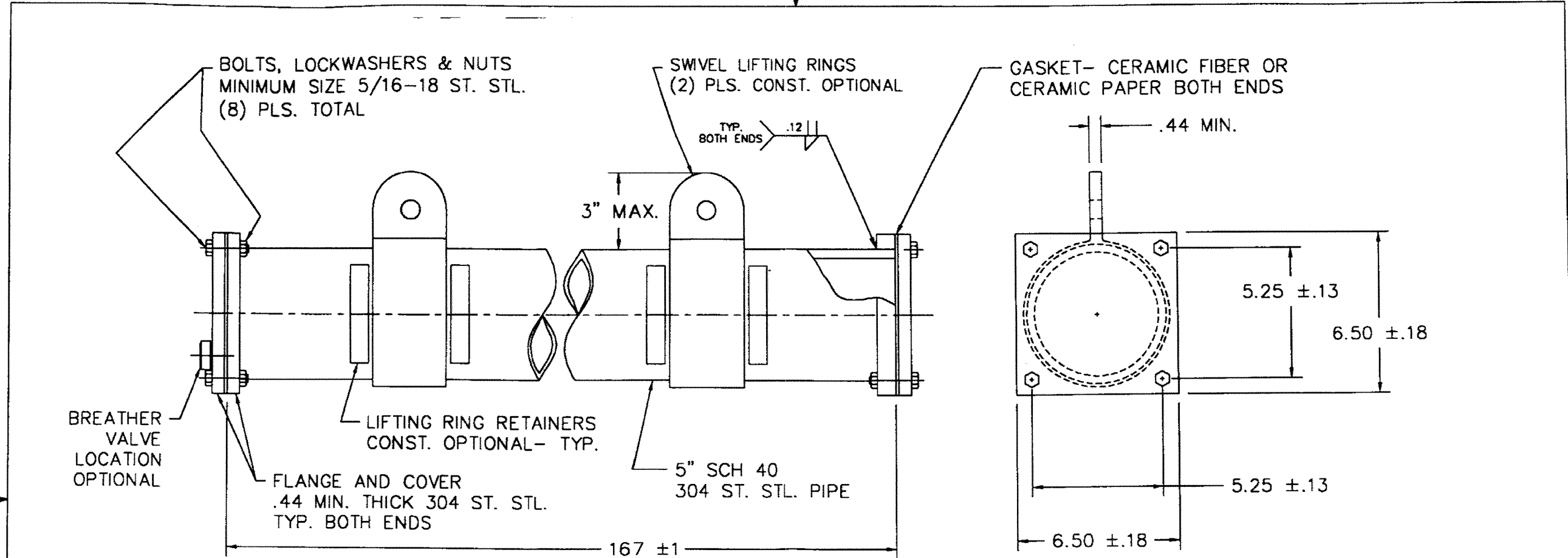
ESSENTIAL TO SAFETY


SAFETY-RELATED CLASSIFICATION CODE <u>N</u>		
SIGNATURES		DATE
DESIGNED	RB JAMES	03-15-04
CHECKED	RPT HIGGINS	03-25-04
PROD DTD	MT KIERMAN	03-26-04
DESIGNED	MT KIERMAN	03-22-04
APPROVED PRACTICES		SUPV.
UNLESS OTHERWISE SPECIFIED TOLERANCES ON:		
2 PLACE DECIMALS ± ——— FRACTIONS	✓ ± ———	
3 PLACE DECIMALS ± ——— ANGLES	✓ ± ———	
 A joint venture of GE, Toshiba, & Hitachi RAJ-I — INNER CONTAINER END LID ASSEMBLY LICENSING DRAWING		
REV. DATE	REV. DATE	REV. DATE
03-15-04	03-26-04	03-26-04
001	002	003
1053748		2
DES. NO. <u>1053748-0</u>		QTY. <u>1</u> QTY. <u>3</u>










1	eDRF 0000-0018-1484	ADDED "ESNTL. TO SAFETY".	DB	MTK	6/13/03
0	FIRST ISSUE FMA-324		BA	<i>LES</i>	6-3-98
REV.	DESCRIPTION		BY	CR#	APPROVAL
REVISIONS					
SIGNATURES		DATE		<div> NEP</div> <div>GENERAL ELECTRIC COMPANY</div> <div>NUCLEAR ENERGY PRODUCTION</div> <div>WILMINGTON, N.C.</div>	
DRAWN	B. AYERS (IDD)	6/3/98			
CHECKED					
ENGRG					
ENGRG					
SCALE 1/4		ALL SURF. ✓		<div>TYPE OF DRAWING</div> <div>SHIPPING CONTAINER</div> <div>LOOSE FUEL RODS</div> <div>FMF NRC LICENSING DRAWING</div>	
UNLESS OTHERWISE SPECIFIED					
TOLERANCES ON :					
2 PLACE DECIMALS ±		FRACTIONS ±			
3 PLACE DECIMALS ±		ANGLES ±		<div><div>ISSUE DATE</div><div>6-3-98</div><div>CR NO.</div><div></div></div> <div><div>DWG NO.</div><div>0028B98</div><div>FILE NAME</div><div>0028B98</div></div> <div><div>REV.</div><div>1</div><div>SH NO.</div><div>1</div><div>CONT ON</div><div>F</div></div>	

EQUIPMENT CLASS CODE P

ESSENTIAL TO SAFETY

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Handling: None	Page 65/517		

## 2. STRUCTURAL EVALUATION

This section presents evaluations demonstrating that the TN-B1 package meets applicable structural criteria. The TN-B1 packaging, consisting of unirradiated fuel assemblies that provide containment, an inner container, and an outer container with paper honeycomb spacers, is evaluated and shown to provide adequate protection for the payload. Normal Conditions of Transport (NCT) and Hypothetical Accident Condition (HAC) evaluations, using analytic and empirical techniques, are performed to address 10 CFR 71 performance requirements.

Numerous tests were successfully performed on the RAJ-II package during its initial qualification in Japan that provided a basis for selecting the certification tests. RAJ-II certification testing involved two full-scale Certification Test Units (CTU) at Oak Ridge, TN. The RAJ-II CTUs were subjected to a series of free drop and puncture drop tests. The RAJ-II CTUs protected the simulated fuel assemblies, allowing them to remain undamaged and leak tight throughout certification testing. Since the RAJ-II and TN-B1 structural designs are identical, the RAJ-II tests are completely applicable to the TN-B1 package. Details of the certification test program are provided in Appendix 2.12.1.

### 2.1. DESCRIPTION OF STRUCTURAL DESIGN

#### 2.1.1. *Discussion*


A comprehensive discussion on the TN-B1 packaging design and configuration is provided in chapter 1.0. Drawings provided in Appendix 1.4.1 show the construction of the TN-B1 and how it protects the fuel assemblies. The containment is provided by the fuel cladding and welded end fittings of the fuel rods. The fuel is protected by an inner container that provides thermal insulations and soft foam that protects the fuel from vibration. The inner container is supported by vibration isolation system inside the outer container that has shock absorbing blocks of balsa and honeycomb made of resin impregnated kraft paper (hereinafter called "paper honeycomb"). Specific discussions relating to the aspects important to demonstrating the structural configuration and performance to design criteria for the TN-B1 packaging are provided in the following sections. Standard fabrication methods are used to fabricate the TN-B1 package.

Detailed drawings showing applicable dimensions and tolerances are provided in Appendix 1.4.1.

Weights for the various components and the assembled packaging are provided in Section 2.1.3.

##### 2.1.1.1. **Containment Structures**

The primary containment for the radioactive material in the TN-B1 is the fuel rod cladding, which is manufactured to high standards for use in nuclear reactors. The fabrication standards for the

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fuel are in excess of what is needed to provide containment for shipping of the fuel. The fuel rod cladding is designed to provide containment throughout the life of the fuel, prior to loading, in transportation, and while used in the reactor where it operates at higher pressures and temperatures, and must contain fission products as well as the fuel itself.

The cladding tubes for the fuel are high quality seamless tubing. The clad fuel is verified leaktight before shipment.

#### **2.1.1.2. Non-Containment Vessel Structures**

The TN-B1 is made up of two non-containment structures, the inner container, and the outer container that are designed to protect the fuel assemblies and clad rods which serve as the containment. The inner container design provides some mechanical protection although its primary function is to provide thermal protection. The outer container consists of a metal wall with shock absorbing devices inside and vibration isolation mounts for the inner container. Section 1.2.1 provides a detailed description of the inner and outer container. Non-containment structures are fabricated in accordance with the drawings in Appendix 1.4.1.

Welds for the non-containment vessel walls are subjected to visual inspection as delineated on the drawings in Appendix 1.4.1.

#### **2.1.2. Design Criteria**

Proof of performance for the TN-B1 package is achieved by a combination of analytic and empirical evaluations. The acceptance criteria for analytic assessments are in accordance with 10 CFR 71 and the applicable regulatory guides. The acceptance criterion for empirical assessments is a demonstration that both the inner and outer container are not damaged in such a way that their performance in protecting the fuel assemblies during the thermal event is not compromised and the fuel itself is not damaged throughout the NCT and HAC certification testing. Additionally, package deformations obtained from certification testing are considered in subsequent thermal, shielding, and criticality evaluations are validated.

##### **2.1.2.1. Analytic Design Criteria (Allowable Stresses)**

The allowable stress values used for analytic assessments of TN-B1 package structural performance come from the regulatory criteria such as yield strength or 1/3 of yield or from the ASME Code for the particular application. Material yield strengths, taken from the ASME Code, used in the analytic acceptance criteria,  $S_y$ , and ultimate strengths,  $S_u$ , are presented in Table 2-2 of Section 2.2.

##### **2.1.2.2. Containment Structures**

The fuel cladding provides the primary containment for the nuclear fuel.

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### 2.1.2.3. **Non-Containment Structures**

For evaluation of lifting devices, the allowable stresses are limited to one-third of the material yield strength, consistent with the requirements of 10 CFR 71.45(a). For evaluation of tie-down devices, the allowable stresses are limited to the material yield strength, consistent with the requirements of 10 CFR 71.45(b).

### 2.1.2.4. **Miscellaneous Structural Failure Modes**

#### 2.1.2.4.1. ***Brittle Fracture***

By avoiding the use of ferritic steels in the TN-B1 packaging, brittle fracture concerns are precluded. Specifically, most primary structural components are fabricated of austenitic stainless steel. Since this material does not undergo a ductile-to-brittle transition in the temperature range of interest (above -40 °F), it is safe from brittle fracture.

The closure bolts used to secure the inner and outer container lids are stainless steel, socket head cap screws ensuring that brittle fracture is not of concern. Other fasteners used in the TN-B1 packaging assembly provide redundancy and are made from stainless steel, again eliminating brittle fracture concerns.

#### 2.1.2.4.2. ***Extreme Total Stress Intensity Range***

Since the response of the TN-B1 package to accident conditions is typically evaluated empirically rather than analytically, the extreme total stress intensity range has not been quantified. Two full- scale certification test units (see Appendix 2.12.1) successfully passed free-drop and puncture testing. The CTUs were also fabricated in accordance with the drawings in Appendix 1.4.1, thus incurring prototypic fabrication induced stresses. Exposure to these conditions has demonstrated leak tight containment of the fuel, geometric configuration stability for criticality safety, and protection for the fuel. Thus the intent of the extreme total stress intensity range requirement has been met.

#### 2.1.2.4.3. ***Buckling Assessment***

Due to the small diameter of the containment boundary (the fuel rod cladding) and the fact that its radial deflection is limited by the internal fuel pellets, radial buckling is not a failure mode of concern for the containment boundary. Axial buckling deflection is also limited by the inner wall of the inner container and lid. The applied axial load to the fuel is also limited by the wood at the end of the packaging. The limited horizontal movement of the fuel during an end drop limits the ability of the fuel to buckle as demonstrated in tests performed on CTU 2 (see Appendix 2.12.1).

It is also noted that 30-foot drop tests performed on full-scale models with the package in various orientations produced no evidence of buckling of any of the fuel (see Appendix 2.12.1). Certification testing does not provide a specific determination of the design margin against

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buckling, but is considered as evidence that buckling will not occur. In addition buckling is a potential concern to ensure adequate geometric configuration control of the post accident package for criticality control. This involves not only the internal configuration of the package but the potential spacing between packages as well. Deformation of the TN-B1 is limited by its redundant structure. The wall of the package acts to stiffen the support plates that carry the load of the inner container via the vibration isolating mechanism. Part of the redundant system to minimize deformation of the fuel is the paper honeycomb that absorbs shocks that would impart side loading to the fuel. The inner container, consisting of an inner wall separated from an outer wall by thermal insulation, is lined with cushioning material that supports the fuel. Regardless of the specific failure mechanism of the support plates, the total deformation is limited by the shock absorbers (paper honeycomb). These blocks immediately share the load. Hence, even if the support plates would buckle allowing the outer wall to plastically deform, the amount of deformation is limited by the shock absorbing material. This has been demonstrated by test to allow only 118 mm (4.7 inches) of deformation of the shock absorbing blocks. The criticality evaluation takes into consideration this deformation. The redundant support system combined with the vibro-isolation and shock absorption system prevents the deformation of the inner container and the fuel.


The axial deformation resulting from an end drop is controlled in a similar manner. The end of the outer container has a wood shock absorber built in that carries the load from the inner container to the outer wall after the vibro-isolation device deflects. This reduces the load carried by the outer wall and support plates. It prevents large loads and deformations that could contribute to buckling of the fuel. The inner container constrains the fuel from large deformations or buckling.

Therefore, the support system prevents buckling of the packaging or fuel that would affect the criticality control or containment.

### **2.1.3. *Weights and Centers of Gravity***

The maximum gross weight of a TN-B1 package, including a maximum payload weight of 684 kg (1,508 pounds) is 1,614 kg (3,558 pounds). The maximum vertical Center of Gravity (CG) is located 421 mm (16.57 inches) above the bottom surface of the package for a fully loaded package. A maximum horizontal shift of the horizontal CG is 92 mm (3.62 inches). This is allowed for in the lifting and tie-down calculations presented in Section 2.5.1. Figure 2-1 shows the locations of the center of gravity for the major components and the location of the center of gravity for the assembled. A detailed breakdown of the TN-B1 package component weights is summarized in Table 2-1.



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#### 2.1.3.1. Effect of CG Offset

The shift of the CG of the package 92 mm (3.6 inches) has very little effect on the performance of the package due to the length of the package, 5,068 mm (199.53 in). This results in a small shift of the weight and forces from one end of the package to the other. The actual total shift is:

$$3.6\% = 1 - \frac{(2)((5068/2) - 92)}{5068}$$

The offset of the CG is taken into account in the lifting and tie down calculations. The effect of this relatively small offset can be neglected.

#### 2.1.4. Identification of Codes and Standards for Package Design


The radioactive isotopic content of the fuel is primarily U-235 with small amounts of other isotopes that make it Type B. Using the isotopic content limits shown in Section 1.2.3 the package would be considered a Category II. As such the applicable codes that would apply are the ASME Boiler and Pressure Vessel Code Section III, Subsection ND for the containment boundary which is the fuel cladding and Section III, Subsection NG for the criticality control Structure and the Section VIII for the non containment components.

The fuel cladding, due to its service in the reactor and need for high integrity, is designed to and fabricated to standards that exceed those required by ASME Section III Subsection ND. The structure used to maintain criticality control is demonstrated by test. The packaging capabilities are verified by test and the codes used in fabrication are called out on the drawings in Appendix 1.4.1. The sheet metal construction of the packaging requires different joint designs and manufacturing techniques that would normally be covered by the above referenced codes.

##### 2.1.4.1. JIS/ASTM Comparison of Materials

The Certification Test Units (CTUs) were manufactured in Japan using material meeting JIS specifications. The fuel cladding and ceramic pellets were manufactured in the US to US specifications. The future manufacturing of TN-B1 packages may be performed using American standards (ASTM or ASME) that are appropriate substitutes for the Japanese standards (JIS) material comprising the CTUs. In order to assure that the packaging manufactured in the future meets the performance requirements demonstrated for the RAJ-II CTUs a detailed review of the differences between the American and Japanese standards was performed. The scope of the study included the: stainless steel products, wood products, rubber, paper honeycomb, and polyethylene foam. The study concluded that American standards material is available and compatible to the JIS standards. Future manufacturing of these packages for domestic use may



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be to American or Japanese specifications meeting the tolerances specified in the general arrangement drawings.

#### 2.1.4.2. JIS/ASME Weld Comparison

Based upon an evaluation, it is concluded that the following standards are equivalent for the purposes of fabrication of the TN-B1 container in the United States:


Japanese Specification	American Specification
JIS Z 3821 Standard qualification procedure for welding technique of stainless steel	ASME Section IX
JIS Z 3140 Method of inspection for spot weld	ASME Section IX
JIS Z 3145 Method of bend test for stud weld	ASME Section IX

#### 2.1.4.3. JIS/JSNDI/ASNT Non-destructive Examination Personnel Qualification and Certification Comparison

The following standards are considered equivalent for Non-destructive Examination Personnel Qualification and Certification. Personnel with these qualifications and certifications are authorized to perform examinations of the fabrication inspection requirements for the TN-B1 container in the United States. Although these documents cover other disciplines, this comparison only applies to Liquid Penetrant Examination.

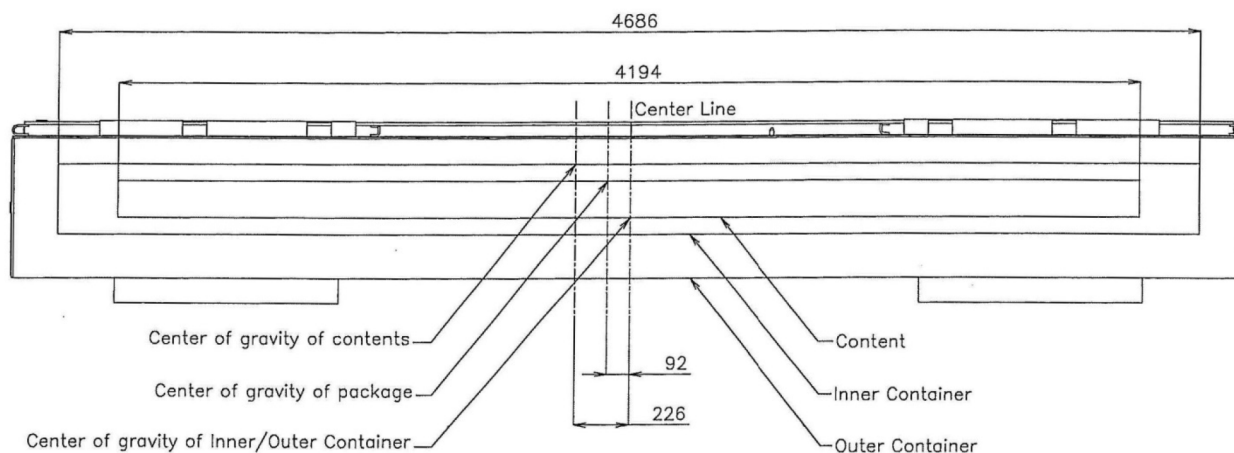
Japanese Specification	American Specification
JIS Z 2305 Qualification and Certification for NDT Personnel	SNT-TC-1A* Recommended Practice
Certification NDIS 0601	SNT-TC-1A Recommended Practice
Certification NDIS J001	SNT-TC-1A Recommended Practice

\*Society of Non-destructive Testing – Technical Council

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**Table 2-1 TN-B1 Weight**

Contents	Number of assemblies per package	Maximum 2 Assemblies
	Number of fuel rods per package	See section 1.2.3
	Total weight	684 kg (1,508 lb)
Inner container	Body	200 kg (441 lb) (including bolts)
	Lid	101 kg (223 lb)
	End lids	7 kg (15.4 lb)
	Total weight	308 kg (679 lb)
Outer container	Body	485 kg (1,069 lb) (including bolts)
	Lid	137 kg (302 lb)
	Total weight	622 kg (1,371 lb)
Total weight of package		1,614 kg (3,558 lb)



(unit: mm)

**Figure 2-1 Center of Gravity of Package Components**

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## 2.2. **MATERIALS**

### 2.2.1. ***Material Properties and Specifications***

The major structural components, i.e., the Outer Container (OC) and Inner Container (IC) walls, supports, and attachment blocks are fabricated from austenitic stainless steel. Other materials performing a structural function are lumber (bolster), balsa (shock absorber), paper honeycomb (shock absorber), alumina silicate (thermal insulator), polyethylene foam (cushioning material), and zirconium alloy (fuel rod cladding). The drawings presented in Appendix 1.4.1 delineate the specific material(s) used for each TN-B1 packaging.

The remainder of this section presents and discusses pertinent mechanical properties for the materials that perform a structural function. Both the materials that are used in the analytics and those whose function in the package is demonstrated by test such as the shock absorbing material are presented. In general the analytics covering the lifting and tie down capabilities of the package and some normal condition events are limited to the stainless steel structure of the packaging.

Table 2-2 presents the bounding mechanical properties for the series 300 stainless steel used in the TN-B1 packaging. Each of the representative mechanical properties is those of Type 304 stainless steel and is taken from Section II, Parts A and D, of the ASME Boiler and Pressure Vessel Code. These properties are applicable to both packages that may have been made in Japan to Japanese specifications, Japanese Industrial Standards (JIS) or using ASME specification material. The density of stainless steel is taken as 0.29 lb/in<sup>3</sup> (8.03E3 kg/m<sup>3</sup>), and Poisson's Ratio is 0.3.


Table 2-3 presents the mechanical properties of the main non-stainless steel components of the package necessary for the structural analysis.

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**Table 2-2 Representative Mechanical Properties of 300 Series  
Stainless Steel Components**

①		②	③	④	⑤
Minimum Elongation (%)	Temperature °C (°F)	Yield Strength, Sy MPa (×10 <sup>3</sup> psi)	Ultimate Strength, Su MPa (×10 <sup>3</sup> psi)	Elastic Modulus, E GPa (×10 <sup>6</sup> psi)	Thermal Expansion Coefficient, α x 10 <sup>-6</sup> mm/mm/°C (×10 <sup>-6</sup> in/in/°F)
35	-29 (-20)	206.8 (30.0)	517.1 (75.0)	-----	-----
40	21 (70)	206.8 (30.0)	517.1 (75.0)	195.1 (28.3)	-----
30	38 (100)	206.8 (30.0)	517.1 (75.0)	-----	15.39 (8.55)
25	93 (200)	172.4 (25.0)	489.5 (71.0)	190.3 (27.6)	15.82 (8.79)
30	149 (300)	155.1 (22.5)	455.1 (66.0)	186.2 (27.0)	16.2 (9.00)
40	204 (400)	142.7 (20.7)	444.0 (64.4)	182.7 (26.5)	16.54 (9.19)
40 <sup>⑥</sup>	23°C <sup>⑥</sup>	205 MPa Min <sup>⑥</sup>	520 MPa Min <sup>⑥</sup>	-----	-----
40 <sup>⑦</sup>	21°C <sup>⑦</sup>	205 MPa Min <sup>⑦</sup>	515 MPa Min <sup>⑦</sup>	-----	-----

- Notes:**
- ① ASME Code, Section II, Part A
  - ② ASME Code, Section II, Part D, Table Y-1
  - ③ ASME Code, Section II, Part D, Table U
  - ④ ASME Code, Section II, Part D, Table TM-1, Material Group G
  - ⑤ ASME Code, Section II, Part D, Table TE-1, 18Cr-8Ni, Coefficient B
  - ⑥ JIS Handbook Ferrous Materials and Metallurgy I, Sections G4303, G4304, G4305 Material Specifications
  - ⑦ ASTM A240, A666 & A276 Material Specifications

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**Table 2-3 Mechanical Properties of Typical Components**

Materials  (Usage)	Yield stress or yield strength	Tensile strength	Compressive strength	Bending strength	Static initial peak stress	Modulus of longitudinal elasticity	Density  (g/cm <sup>3</sup> )
Lumber (bolster)	56.3 MPa  Nominal	–	50.5 MPa  Nominal	72.0 MPa  Nominal	–	7.85 GPa  Nominal	0.53  Nominal
Balsa (shock absorber)	–	–	16 MPa  Nominal	–	–	–	0.18  Nominal
Paper honeycomb (shock absorber)	–	–	–	–	2.35 MPa  Nominal	–	0.06  Nominal
Alumina Silicate (thermal insulator)	–	–	294 kPa  Nominal	314 kPa  Nominal	–	–	0.25  Nominal
Foam polyethylene (cushioning mat'l)	–	–	Approx. 0.2MPa @ 50% strain	–	0.69 MPa  Nominal	–	0.068  Nominal
Zirconium alloy (fuel rods)  ASTM B811	241 MPa  (35,000psi)	413 MPa  (60,000psi)	–	–	–	97.1 GPa  Nominal	6.5  Nominal
300 Series Stainless Socket Headed Cap screw	241 MPa  (35,000psi) (Min)	379 MPa  (75,000psi) (Min)	–	–	–	–	–

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### **2.2.2. Chemical, Galvanic, or Other Reactions**

The major materials of construction of the TN-B1 packaging (i.e., austenitic stainless steel, polyurethane foam, alumina thermal insulator, resin impregnated paper honeycomb, lumber (hemlock and balsa), and natural rubber) will not have significant chemical, galvanic or other reactions in air, inert gas or water environments, thereby satisfying the requirements of 10 CFR 71.43(d). These materials have been previously used, without incident, in radioactive material (RAM) packages for transport of similar payload materials. A successful RAM packaging history combined with successful use of these fabrication materials in similar industrial environments ensures that the integrity of the TN-B1 package will not be compromised by any chemical, galvanic, or other reactions.

The TN-B1 packaging is primarily constructed of series 300 stainless steel. This material is highly corrosion resistant to most environments. The metallic structure of the TN-B1 packaging is composed entirely of this material and compatible 300 series weld material. Since both the base and weld materials are 300 series materials, they have nearly identical electrochemical potential thereby minimizing any galvanic corrosion that could occur.

The stainless steel within the IC cavity between the inner and outer walls is filled with a ceramic alumina silicate thermal insulator. This material is non-reactive with either the wood or the stainless steel, both dry or in water. The alumina silicate is very low in free chlorides to minimize the potential for stress corrosion of the IC structure.

The polyethylene foam that is used in the IC for cushioning material has been used previously and is compatible with stainless steel. The polyethylene foam in is very low in free halogens and chlorides.

Resin impregnated paper honeycomb is used in the TN-B1 packaging as cushioning material. The impregnated paper is resistant to water and break down. It is low in leachable halides.

The natural rubber that is used as a gasket for the lids and in the vibro-isolating system, contains no corrosives that would react adversely affect the TN-B1 packaging. This material is organic in nature and non-corrosive to the stainless steel boundaries of the TN-B1 packaging.

#### **2.2.2.1. Content Interaction with Packaging Materials of Construction**

The materials of construction of the TN-B1 packaging are checked for compatibility with the materials that make up the contents or fuel rods that are to be shipped in the TN-B1. The primary materials of construction of the fuel assembly that could come in contact with the packaging are the stainless steel and the zirconium alloy material that is used for the cladding of the fuel rods. Zirconium alloy (including metal zirconium), stainless steel, and Ni-Cr- Fe alloy, which form a passivated oxide film on the surface under normal atmosphere with slight moisture, are essentially stable. The contact of the above three kinds of metals with polyethylene is



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chemically stable. These materials are compatible with the stainless steel, polyethylene, and natural rubber that could come in contact with the contents.

### **2.2.3. *Effects of Radiation on Materials***

Since this is an unirradiated fuel package, the radiation to the packaging material is insignificant. Also, the primary materials of construction and containment, austenitic stainless steel and the zirconium alloy cladding of the fuel are highly resistant to radiation.

## **2.3. FABRICATION AND EXAMINATION**

### **2.3.1. *Fabrication***

The TN-B1 is fabricated using standard fabrication techniques. This includes cutting, bending and welding the stainless steel sheet metal. As shown on the drawing the welding is done to AWS D1.6 Welding of Stainless Steel. The process may also be controlled by ASME Section IX or other international codes. The containment, the cladding of the fuel rods is fabricated to standards that exceed the required Section VIII of the ASME Boiler and Pressure vessel code due to the service requirements of the fuel in reactors.

### **2.3.2. *Examination***


The primary means of examination to determine compliance of the TN-B1 to the design requirements is visual examination of each component and the assembled units. This includes dimensional verification as well as material and weld examination. The materials will also be certified to the material specifications. Shock absorbing material such as the paper honeycomb will also have verified material properties.

## **2.4. LIFTING AND TIE-DOWN STANDARDS FOR ALL PACKAGES**

For analysis of the lifting and tie-down components of the TN-B1 packaging, material properties from Section 2.2 are taken at a bounding temperature of 75°C (167 °F) per Section 2.6.1.1. This is the maximum temperature that the container reaches when in the sun. The primary structural material is 300 series stainless steel that is used in the Outer Container (OC).

A loaded TN-B1 package can be lifted using either a forklift or by slings. The gross weight of the package is a maximum of 1,614 kg (3,558 lb). Locating/protection plates for the forklift and locating angles for the sling locate the lift points for the package. In both cases the package is lifted from beneath. The failure of these locating/protective features would not cause the package to drop nor compromise its ability to perform its required functions.

The inner container may be lifted empty or filled with the contents using the sling fittings that are attached at the positions shown in Figure 2-2. The details of the sling fittings are as shown in Figure 2-3. Since the center of gravity depends on existence of the contents, the sling fittings

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for the filled container and the empty container are marked respectively as "Use When Loaded" and "Use When Empty" to avoid improper operations. Also, the sling fittings on the lid of inner container to lift the lid only are marked as "Use for Lifting Lid" similar to the outer container.

The sling devices are mechanically designed to be able to handle the package and the inner container filled with the fuel assemblies in safety; they can lift three times the gross weight of the package, or three times the gross weight of the filled inner container respectively, so that they can with stand rapid lifting.

Properties of 300 series stainless steel are summarized below.

**Table 2-4 Properties of 300 Series Stainless Steel**

Material Property	Value	Reference
<b>At 75°C (167 °F)</b>		
Elastic Modulus, E	191.7 GPa  ( $27.8 \times 10^6$ psi)	Table 2-2
Yield Strength, $\sigma_y$	184.7 MPa  (26,788 psi)	
Shear Stress, equal to (0.6) $\sigma_y$	110.8 MPa  (16,073 psi)	

#### **2.4.1. Lifting Devices**

This section demonstrates that the attachments designed to lift the TN-B1 package are designed with a minimum safety factor of three against yielding, per the requirements of 10 CFR71.45 (a).

The lifting devices on the outer container lid are restricted to only lifting the outer container lid, and the lifting devices in the inner lid are restricted to only lifting the inner container lid. Although these lifting devices are designed with a minimum safety factor of three against

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yielding, detailed analyses are not specifically included herein since these lifting devices are not intended for lifting a TN-B1 package.

The outer container can be handled by either forklift or slings in a basket hitch around the package, requiring no structural component whose failure could affect the performance of the package.

#### 2.4.1.1. Lifting of Inner Container

The inner container is lifted when loaded with fuel from the outer container with sling fittings attached to the body of the inner container. Three pairs (six in total) of the sling fittings are attached to the inner container as shown in Figure 2-2. The center of gravity depends upon whether the container is filled or not. Since the six sling fittings are the same, the stress in the sling fittings are evaluated for the case of at the maximum weight condition that occurs when the inner container is filled with fuel assemblies.

The stress on the sling fitting when lifting the inner container filled with contents is evaluated by determining the maximum load acting on any given fitting.

The maximum load,  $P_v$ , (see Figure 2-9) acting on one of the sling fitting vertically when lifting is given by the following equation:

$$P_v = \frac{(W_2 + W_3)}{n} \cdot g$$

Where:


$P_v$ : maximum load acting to sling fitting in vertical direction	N
$W_2$ : mass of inner container	308 kg (679 lb)
$W_3$ : mass of contents	684 kg (1,508 lb)
$n$ : number of sling fittings	4
$g$ : acceleration of gravity	9.81 m/s <sup>2</sup>

Accordingly, the maximum load acting on the sling fitting vertically is calculated as:

$$P_v = \frac{684 + 308}{4} \times 9.81 = 2.433 \times 10^3 \text{ N (546.9 lbf)}$$

The load,  $P$ , acting to the sling fitting when the sling is at a minimum angle of 60° is calculated as:

$$P = \frac{P_v}{\sin \theta} = \frac{2.433 \times 10^3}{\sin 60^\circ} = 2.809 \times 10^3 \text{ N (631 lbf)}$$

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Also, the maximum load,  $P_H$ , acting on the sling fitting horizontally is calculated as:

$$P_H = \frac{P_v}{\tan \theta} = \frac{2.433 \times 10^3}{\tan 60^\circ} = 1.405 \times 10^3 \text{ N (316 lbf)}$$

Each sling fitting is made up of a hooking bar which is a 12mm diameter bent rod and a perforated plate that is made up of two pieces of angle that are welded together. The perforated plate of the sling fitting is welded to a support of that is welded to the body of the inner container.

The shearing stress in the hooking bar (see Figure 2-6) is given by the following equation:

$$\tau_N = \frac{Px\phi}{A}$$

Where

$\tau_N$ : shearing stress on hooking bar of sling fitting	MPa
P: maximum load	$2.809 \times 10^3 \text{ N (631 lbf)}$
A: cross-section of hooking bar of sling fitting	$\pi/4 \times 12^2 = 113 \text{ mm}^2 (0.175 \text{ in}^2)$
$\phi$ : load factor	3

Accordingly, the shearing stress on the hooking bar of the sling fitting at its center is calculated as:

$$\tau_N = \frac{2.809 \times 10^3 \times 3}{113} = 74.58 \text{ MPa (10,820 psi)}$$


The yield stress for stainless steel is 184.7 MPa (26,790 psi) and the shear allowable is  $0.6 \times 184.7 = 110.8 \text{ MPa (16,070 psi)}$  at the maximum normal temperature, hence the margin (MS) is

$$MS = \frac{110.8}{74.58} - 1 = 0.48$$

Therefore, the sling fitting can withstand three times the load without yielding in shear.

The strength of the perforated plate of a sling fitting is evaluated for failure by shearing. The shear stress on a perforated plate (see Figure 2-7) of the sling fitting by the total load is given by the following equation:

$$\tau_N = \frac{Px\phi}{A}$$

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Where:

$\tau_N$ : shearing stress on the perforated plate of a sling fitting      MPa  
P: maximum load       $2.809 \times 10^3 \text{ N (631 lbf)}$   
A: cross-section of the upper part of the perforated plate:       $2x \frac{50-14}{2} x 6 = 216 \text{ mm}^2 (0.33 \text{ in}^2)$   
 $\emptyset$ : load factor      3

Accordingly, the shearing stress,  $\tau_N$ , on the perforated plate of sling fitting is calculated as:

$$\tau_N = \frac{2.809 \times 10^3 \times 3}{216} = 39.01 \text{ MPa (5,658 psi)}$$

The allowable shearing stress for stainless steel is 110.8 MPa (16,073 psi). Then the margin of Safety (MS) is:

$$MS = \frac{110.8}{39.01} - 1 = 1.84$$

Therefore, the shear strength of the plate meets the requirement of not yielding under three times the load.

Next, the strength of welds of the sling fittings is evaluated for the torsional loads applied. Torsional loads are applied to the welds of sling fitting per Figure 2-8.

The moment of inertia of area,  $I_P$ , to the welds of sling fittings is given by the following equation:


$$I_P = I_X + I_Y$$

$$I_X = I_{X2} - I_{X1}$$

$$I_Y = \sum I_{Yi}$$

Where

$I_P$  : moment of inertia of area to welds       $\text{mm}^4$   
 $I_X$  : moment of inertia of area to welds for X-axis       $\text{mm}^4$   
 $I_Y$  : moment of inertia of area to welds for Y-axis       $\text{mm}^4$   
 $I_{X1}$  : moment of inertia of area to inside of weld for X-axis       $\text{mm}^4$   
 $I_{X2}$  : moment of inertia of area to outside of weld for X-axis       $\text{mm}^4$   
 $I_{Y1}$  : moment of inertia of area to each weld for Y-axis       $\text{mm}^4$

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The moment of inertia of area,  $I$ , to a cross-sectional area of width,  $b$ , and height,  $h$ , is given by:

$$I = \frac{1}{12}bh^3$$

Conservatively only the outside welds not including any corner wrap around that attach the sling fitting to the support plate are considered. Thus, the moment of inertia of area,  $I_x$  and  $I_y$  to the welds for X-axis and Y-axis are calculated as:

$$I_x = \left( \frac{1}{12} \times 88 \times 54^3 \right) - \left( \frac{1}{12} \times 88 \times 50^3 \right) = 2.38 \times 10^5 \text{ mm}^4 (0.57 \text{ in}^4)$$

$$I_y = 2I_{y1} = 2 \times \frac{1}{12} \times 2 \times 88^3 = 2.27 \times 10^5 \text{ mm}^4 (0.55 \text{ in}^4)$$

Accordingly, the moment of inertia of area,  $I_p$ , to the welds is calculated as

$$I_p = (2.38 \times 10^5) + (2.27 \times 10^5) = 4.65 \times 10^5 \text{ mm}^4 (1.12 \text{ in}^4)$$

The shearing stress,  $S_d$ , on the weld due to the load acting on the sling fitting is given by the following equation:

$$s_d = \frac{P \cdot \phi}{A}$$

Where:


$S_d$ shearing stress on welds due to the load to sling fitting	MPa
P: maximum load acting to one of sling fitting	$2.809 \times 10^3 \text{ N (631 lbf)}$
A: overall cross-section of welds	$2 \times 88 = 176 \text{ mm}^2 (0.273 \text{ in}^2)$
$\phi$ : load factor	3

Accordingly, the shearing stress on welds due to the load acting to the sling fitting is calculated as:

$$S = \frac{2.809 \times 10^3 \times 3}{176} = 47.9 \text{ MPa (6,950 psi)}$$

The maximum bending moment acting to the sling fitting is given by the following equation from Figure 2-9.

$$M_{\max} = P \cdot l$$

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Where:

$M_{\max}$ : maximum bending moment acting to sling fitting      N · mm  
P: maximum load acting to one of sling fitting       $2.809 \times 10^3$  N (631 lbf)  
l: distance from fulcrum to load point      17 mm (0.67 in)

Therefore, the maximum bending moment acting to the sling fitting is calculated as:

$$\begin{aligned}
 M_{\max} &= 2.809 \times 10^3 \times 17 \\
 &= 4.8 \times 10^4 \text{ N} \cdot \text{mm} (424.8 \text{ in} \cdot \text{lbf})
 \end{aligned}$$

The stress due to this bending moment is given by the following equation:

$$S_m = \frac{M_{\max} \cdot r \cdot \phi}{I_p}$$

Where:

$S_m$ : Stress acting to a point at r from center of gravity due to bending moment MPa  
r: distance from center of gravity to end of welds  $\sqrt{44^2 + 25^2} = 50.6 \text{ mm} (1.99 \text{ in})$   
 $M_{\max}$ : maximum bending moment acting to sling fitting       $4.8 \times 10^4$  N·mm (424.8 in·lbf)  
 $I_p$ : moment of inertia of area to welds       $4.65 \times 10^5 \text{ mm}^4 (1.12 \text{ in}^4)$   
 $\phi$ : load factor      3

From this equation, the maximum bending moment,  $S_m$ , acting to the sling fitting is calculated as:

$$S_m = \frac{4.8 \times 10^4 \times 50.6 \times 3}{4.65 \times 10^5} 15.6 \text{ MPa} (2,260 \text{ psi})$$

In addition, the composite shearing stress, S, on the welds is given by the following equation:

$$S = \sqrt{S_d^2 + S_m^2 + 2S_d S_m \cos \theta}$$

Where

$$\cos \theta = 25/50.6$$



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From this equation, the composite shearing stress, S, is calculated as

$$S = \sqrt{47.9^2 + 15.5^2 + 2 \times 47.9 \times 25/50.6}$$

$$= 57.2 \text{ MPa (8,300 psi)}$$

Meanwhile, the allowable shearing stress for 300 series stainless steel is 110.8 MPa (16,073 psi).

Then the margin (MS) is:

$$MS = \frac{110.8}{57.2} - 1 = 0.94$$

The welds are capable of carrying 3 times the expected load without yielding.

Likewise the welds of the support plates for sling fittings are evaluated in the same manner. Since the welds of the support plates (see Figure 2-10) receive the same load as mentioned above in the case of the welds of the sling fittings, it is evaluated by same analytic method as mentioned above. The symbols used here shall have same meaning.

The moment of inertia of area,  $I_P$ , to the welds of support plate is given by the following equation:

$$I_P = I_X + I_Y$$

Where:

$$I_X = I_{x2} - I_{x1}$$

$$I_Y = I_{y2} - I_{y1}$$


The moment of inertia of areas  $I_X$  and  $I_Y$  to the welds for X-axis and Y-axis are calculated as:

$$I_X = \frac{1}{12} \times 153 \times 83^3 - \frac{1}{12} \times 150 \times 80^3$$

$$= 8.903 \times 10^5 \text{ mm}^4 (2.14 \text{ in}^4)$$

$$I_Y = \frac{1}{12} \times 83 \times 153^3 - \frac{1}{12} \times 80 \times 150^3$$

$$= 2.273 \times 10^6 \text{ mm}^4 (5.46 \text{ in}^4)$$

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Accordingly, the moments of inertia of areas to the welds for the support plates are calculated as:

$$\begin{aligned}
 I_p &= 8.903 \times 10^5 + 2.273 \times 10^6 \\
 &= 3.163 \times 10^6 \text{ mm}^4 (7.60 \text{ in}^4)
 \end{aligned}$$

The overall cross-section, A, of welds of the support plate is:

$$\begin{aligned}
 A &= (153 \times 83) - (150 \times 80) \\
 &= 699 \text{ mm}^2 (1.08 \text{ in}^2)
 \end{aligned}$$

The shearing stress,  $S_d$ , on the welds of the support plate for the sling fitting is calculated by a similar equation as the welds of the sling fitting.

$$S_d = \frac{2.809 \times 10^3 \times 3}{699} = 12.1 \text{ MPa (1,760 psi)}$$

In addition, the stress,  $S_m$ , on the welds of the support plate due to the bending moment is calculated as:

Where:

$$\begin{aligned}
 r &= \sqrt{75^2 + 40^2} = 85 \text{ mm (3.35 in)} \\
 S_m &= \frac{5.9 \times 10^4 \times 85 \times 3}{3.163 \times 10^6} = 4.76 \text{ MPa (690 psi)}
 \end{aligned}$$

Accordingly, the composite shearing stress S on the welds of support plate is calculated as:

$$S = \sqrt{S_d^2 + S_m^2 + 2S_d S_m \cos \theta}$$

Where:

$$\cos \theta = 40/85$$

$$\begin{aligned}
 S &= \sqrt{12.1^2 + 4.76^2 + (2 \times 12.1 \times 4.76 \times (40/85))} \\
 &= 14.9 \text{ MPa (2,160 psi)}
 \end{aligned}$$

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Meanwhile, the allowable shearing stress for 300 series stainless steel is 110.8 MPa (16,073 psi). Then the margin of safety (MS) is:

$$MS = \frac{110.8}{14.9} - 1 = 6.4$$

Therefore, the support plate welds are capable of carrying three times the normal load and not yielding.

As indicated by the margins of safety calculated for each component, the hook bar has the lowest margin; therefore in case of an overload the hook bar will fail prior to any other component. This ensures that, at failure, the rest of the packaging is capable of performing its function of protecting the fuel.

#### 2.4.1.2. Package Lifting Using the Outer Container Lid Lifting Lugs

The outer container lid is lifted by four (4) Ø8-mm (Ø0.315 in.) Type 304 stainless steel bars that are welded to the 50 × 50 × 4 stainless steel lid flange angle. Under a potential excessive loading condition, such as lifting the entire loaded package, these four lifting lugs are required to fail prior to damaging the outer container lid structure.

The outer container lid is also equipped with the four (4) Ø6-mm (Ø0.236 in.) Type 304 stainless steel bar handles, which may be used to manually lift the lid. These bars are welded to the vertical leg of the lid flange angle with single-sided flare-bevel welds for an approximate length of 13 mm, as shown in View G-G on General Arrangement Drawing 105E3743. Since the handles have smaller cross-section (Ø6-mm vs. Ø8-mm), and have smaller and shorter attachment welds, the analysis of the lid lifting bars bounds the handles.

The four lifting bars will be used for this analysis with an assumed lifting angle of 45 degrees. From Table 2-1, the TN-B1 package weighs 1,614 kg [15,827 N] (3,558 lb). For the assumed lifting arrangement, the maximum load on the bar is:

$$F = 1/4 \left[ \frac{15,827}{\sin 45^\circ} \right] = 5,596 \text{ N (1,258 lbs)}$$

Assuming that the lift point is centered above the midpoint of the package (located 1,025 mm longitudinally and 318 mm laterally from lifting bar), the resultant forces on the lifting bar will be:

$$F_{\text{horizontal}} = F_{\text{vertical}} = F \cos 45^\circ = 3,957 \text{ N (890 lbs)}$$

$$F_{//} = F_{\text{horizontal}} \sin \left( \tan^{-1} \left( \frac{1,025}{318} \right) \right) = 3,779 \text{ N (850 lbs)}$$

$$F_{\perp} = F_{\text{horizontal}} \cos \left( \tan^{-1} \left( \frac{1,025}{318} \right) \right) = 1,173 \text{ N (264 lbs)}$$

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where:  $F_{\text{horizontal}}$  = Force in horizontal plane  
 $F_{//}$  = Force parallel to longitudinal axis of package  
 $F_{\perp}$  = Force perpendicular to longitudinal axis of package

These reaction loads will develop both bending and shear stresses in the bar, shear stresses in the attachment welds, and tensile stresses in the flange angle. Each of these stress components will be analyzed separately.

### **Bending of Bar**

The maximum reaction load on the lifting bar will be bending stresses in the bar. Treating the bar as a fixed-fixed beam, the maximum bending stress,  $\sigma_b$ , will be:

$$\sigma_b = \frac{M_{\max}}{Z_{\text{Bar}}}$$

where:  $M_{\max} = 1/8[(F_{\text{vertical}})^2 + (F_{//})^2]^{1/2}(l) = 1/8(5,472)(76) = 51,984 \text{ N-mm (460 lb-in)}$   
 $Z_{\text{bar}} = \pi(d^3)/32 = \pi(8^3)/32 = 50.3 \text{ mm}^3 (0.003 \text{ in}^3)$   
 $l = 2(46-8) = 76 \text{ mm (2.99 in)}$  [assumed equal to bent free length of bar]

Substituting these values results in a maximum bending stress of 1,033 MPa (149,824 psi). The allowable bending stress for the Type 304 material is equal to  $S_y = 184.7 \text{ MPa (26,788 psi)}$ . Therefore, the margin of safety against yielding in bending is:

$$MS = \frac{184.7}{1,033} - 1.0 = -0.8$$

### **Shear of Bar**

The maximum reaction load on the lifting bar will result in shear stresses in the bar. For the shearing the bar, the maximum shear stress will be:

$$\tau_{\text{bar}} = \frac{[(F_{\text{Vertical}})^2 + (F_{//})^2]^{1/2}}{\text{Area}} = \frac{5,472}{(\pi/4)(8)^2} = 108.9 \text{ MPa (15,795 psi)}$$

The allowable shear stress for the Type 304 material is equal to  $0.6S_y = 0.6(184.7) = 110.8 \text{ MPa (16,070 psi)}$ . Therefore, the margin of safety against yielding in shear is:

$$MS = \frac{110.8}{108.9} - 1.0 = 0.02$$

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### **Tension in Bar**

Since the bending stress is well beyond the yield strength, the bar will bend until the reaction load will be reacted as pure tension in the bar. For this condition, the tensile stress,  $\sigma_{t-bar}$ , in the bar will be:

$$\sigma_{t-bar} = \frac{F}{2(Area)} = \frac{5,596}{2 \left[ \left( \frac{\pi}{4} \right) (8^2) \right]} = 55.7 MPa (8,079 psi)$$

The allowable tensile stress for the Type 304 material is equal to the minimum yield strength, 184.7 MPa (26,788 psi). The margin of safety for this condition is then:

$$MS = \frac{184.7}{55.7} - 1.0 = 2.3$$

### **Attachment Welds**

As shown in View F-F on General Arrangement Drawing 105E3743, the lifting bars are welded to the lid flange angle with double-sided flare-bevel welds for an approximate length of 28 mm (1.10 in.) on each leg of the bar. The ends of the bar are welded with a seal fillet weld, which has minimal strength and hence, will be ignored. Since the bar is relatively small, the flare-bevel weld will be treated as an equivalent fillet weld with a 4-mm leg. For this assumption, the maximum primary shear stress,  $\tau_{weld}$ , in the weld will be:

$$\tau_{weld} = \frac{[(F_{vertical})^2 + (F_{//})^2]^{1/2}}{Shear\ area\ of\ welds} = \frac{5,472}{4(4\cos 45^\circ)(28)} = 17.3 MPa (2,509 psi)$$

Due to the off-set, there will also be a secondary (torsion) shear stress,  $\tau'_{weld}$ , component:


$$\tau'_{weld} = \frac{Mr}{J} \frac{Mr}{J}$$

where:

M = applied moment to weld group  
 $= [(F_{vertical})^2 + (F_{//})^2]^{1/2} (\text{distance from centroid} + \text{bend radius} + \frac{1}{2} \text{ bar diameter})$   
 $= 5,472(14 + 8 + 4) = 142,272 \text{ N-mm} (1,259 \text{ lb}_f - \text{in})$

$r_{max}$  = distance from centroid of weld group to farthest point in weld  
 $= [(1/2(46-8))^2 + (14)^2]^{1/2} = 23.6 \text{ mm} (0.929 \text{ in})$

J = second polar moment of inertia of weld group,  $\text{mm}^4$

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Since the four flare-bevel welds are the same size and location, the second polar moment of inertia for the weld group is determined treating the welds as a line\*. For this case, the second polar moment of inertia is:

$$J = 0.707(h) \frac{d(3b^2 + d^2)}{6}$$

where:

h = leg length of weld = 4 mm

d = length of weld = 28 mm

b = distance between weld groups =  $(462 + 462)/2 = 65.1$  mm

Substituting these values results in a secondary polar moment of inertia of  $178,138 \text{ mm}^4$  ( $0.428 \text{ in}^4$ ). The secondary shear stress then becomes:

$$\tau'_{weld} = \frac{(142,272)(23.6)}{178,138} = 18.8 \text{ MPa (2,727 psi)}$$

The total shear stress in the weld is then the square root of the sum of the squares of the primary shear and secondary shear:

$$\tau_{total} = [(\tau_{weld})^2 + (\tau'_{weld})^2]^{1/2} = 25.5 \text{ MPa (3,698 psi)}$$

The allowable shear stress for the Type 304 material is equal to 110.8 MPa (16,070 psi). Therefore, the margin of safety against yielding in shear for the welds is:

$$MS = \frac{110.8}{25.5} - 1.0 = 3.3$$


### **Shear Tearout of Base Metal**

Shear tearout of the 4-mm thick base metal is evaluated by conservatively considering only the area of a section equal to the weld length of the two welds. The 2-mm thick sheet that is attached to the vertical leg of the flange angle is ignored for this calculation. The total tensile area,  $A_t$ , will be:

$$A_{\text{shear}} = 2[4(28)] = 224 \text{ mm}^2 (0.347 \text{ in}^2)$$

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\* Shigley, Joseph E., and Mischke, Charles R., *Mechanical Engineering Design, Fifth Edition*, McGraw-Hill, Inc., 1989.

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For this case, the shear stress of the base metal,  $\tau_{base\ metal}$ , is:

$$\tau_{base\ matal} = \frac{F}{A_{Shear}} = \frac{5,596}{224} = 25.0\text{MPa (3,624 psi)}$$

The allowable shear stress for the Type 304 material is equal to 110.8 MPa (16,070 psi). The margin of safety for this condition is then:

$$MS = \frac{110.8}{25.0} - 1.0 = 3.4$$

### **Summary**


As demonstrated by these calculations, the minimum margin of safety for the outer container lid lifting lugs is -0.8, which results in failure of the bar in bending for lifting the complete loaded package. The largest positive margin of safety (+3.4) occurs in the base metal of the lid flange angle, which demonstrates that the outer container lid structure would not fail in an excessive load condition. All other margins of safety in the load path are positive, but are lower than the base metal. Therefore, potentially lifting the complete package by these lid lifting lugs will fail the lifting bar and have no detrimental effect on the effectiveness of the TN-B1 package.

#### **2.4.2. Tie-Down Devices**

There are no tie-down features that are a structural part of the TN-B1 package. The packages are transported either in container vans or on flatbed trucks. When transported in container vans, blocking and bracing is provided that distributes any loads into the packages. This bracing and blocking is customized to address individual shipping configurations and the specific container van being used. When transported on a flatbed trailer, straps going over the package are used to secure it to the trailer. Therefore, the requirements of 10 CFR 71.45(b) are satisfied since no structural part of the package is used as a tie-down device.

An evaluation is performed on the ability of the package to withstand loadings of 2 g vertical and 5 g laterally when restrained by strapping. The worst case loading situation for the packages is when they are stacked in groups of 9 on a flatbed trailer and secured with a minimum of 3 straps. Although the packages may be shipped in other configurations such as 2x3 the greatest strap loading that would be applied to the package when secured in a 3x3 configuration. Between each adjacent column of packages 2 × 4 wood shoring may be placed where the straps will be applied. The evaluation below is conservatively performed without the 2 × 4 shoring in place.



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As a bounding evaluation, it is assumed that the outside corners of the top outside packages carry all the vertical loads that would result from the vertical acceleration and the vertical load required to resist the over-turning moment from the horizontal acceleration. The corners of all top packages would actually carry the vertical load. See Figure 2-11.

For modeling purposes, the matrix of nine packages is treated as a rigid body. By summing moments, the vertical force required to prevent the over-turning of the stack by the horizontal loads is determined. This load is conservatively applied to one edge of one container

The key dimensions and weights for each package are:

Width	$w = 720 \text{ mm (28.3in)}$
Total Height	$h = 742 \text{ mm (29.2in)}$
CG height	$cgy = 421 \text{ mm (16.6 in)}$
Mass of each package	$m = 1,614 \text{ kg (3,558 lb)}$ Gravitational
acceleration	$g = 9.81 \text{ m/sec}^2$
Vertical acceleration factor	$g_v = 2$
Horizontal acceleration factor	$g_h = 5$

The vertical center of gravity of the 9-package matrix is:

$$CG_y = 3mg(2h + cgy)/9mg + 3mg(h + cgy)/9mg + 3mg(cgy)/9mg = 1.163 \times 10^3 \text{ mm (45.8 in)}$$

Summing the forces in the vertical direction due to the 2 g loading, the strap load applied at the two locations can be determined for this load condition.

$$R_{st} = 9 g_v m g/2 = 1.425 \times 10^5 \text{ N (3.202} \times 10^4 \text{ lb}_f)$$

Summing moments about one of the bottom corners of the stack will determine the strap force required to resist overturning due to the horizontal loading.

$$R_s = \frac{(g_h(CG_y)9mg)}{(3w)} = 3.835 \times 10^5 \text{ N (8.621} \times 10 \text{ lb}_f)$$

Total vertical strap load is:

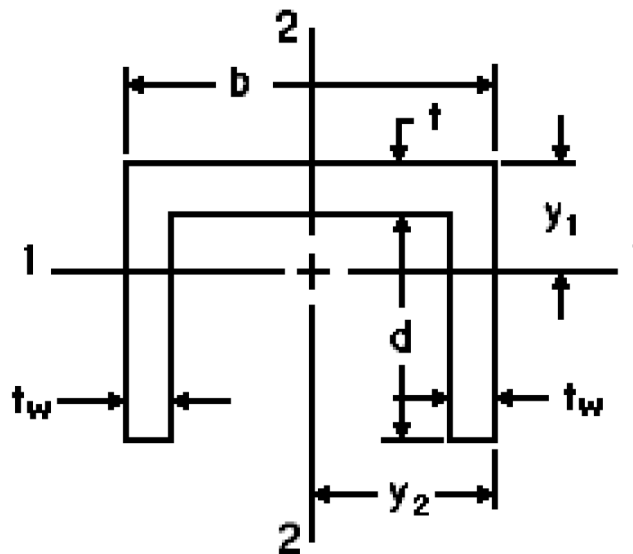
$$R_t = R_{st} + R_s = 5.260 \times 10^5 \text{ N (1.182} \times 10^5 \text{ lb}_f)$$

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Checking the support plate carrying capability:

There are eight (8) 5mm × 55mm support plates in groups of two (2) that carry the vibro-isolation frame inside the outer container. These are skipped welded to the wall, plus have two thick (10 and 15 mm) by 80 mm and 70 mm wide plates welded between them. These plates are in addition to the body straps and the body struts (angles) in corners that provide vertical stiffening to the side panels. On top of the side panel, there are two angles that make up the flange in both the body and the lid that provide load distribution capability to the side wall and the internal structure. In addition these angles are stiffen at the ends by the bolster support angle that further distributes the end strap loads to the end structure of the package reducing load in the sides of the package.

Since the eight support plates are assembled together in groups of two with the reinforcement plates connecting the plates along with the welding to the wall, each two-plate section is considered as a column that is capable of carrying the tie-down loads. Addressing the support plates as a channel section, which is 140 mm wide and 57 mm deep, its properties can be determined.



Channel section

Length of web  $b = 140 \text{ mm (5.5 in)}$

Length of flange  $d = 55 \text{ mm (2.2 in)}$

Web thickness  $t = 2 \text{ mm (0.08 in)}$

Flange thickness  $t_w = 5 \text{ mm (0.2 in)}$

Area  $A = t_b + 2t_w d = 830.3 \text{ mm}^2 (1.287 \text{ in}^2)$

Since there are four of these assemblies to a side the total area is:

$$A_{\text{spt}} = 4A = 3,321 \text{ mm}^2 (5.148 \text{ in}^2)$$

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The compressive stress is

$$\sigma_c = R_t/A_{spt} = 158.4 \text{ MPa (23.0 ksi)}$$

This is less than the yield stress of the Type 304 stainless steel  $S_y = 206.8 \text{ MPa (30.0 ksi)}$ .

The resistance of the plate to buckling is also evaluated. The equation to obtain the moments of inertia of area of the support plate which are subject to buckling is:

$$y_1 = (bt^2 + 2t_w d(2t + d))/2(tb + 2t_w d) = 19.9 \text{ mm (0.783 in)}$$

$$y_2 = b/2 = 70 \text{ mm (2.756 in)}$$

Moments of Inertia

$$I_1 = b(d+t)^3/3 + d^3(b-2t_w)/3 - A(d+t-y_1)^2 = 2.894 \times 10^5 \text{ mm}^4 (0.695 \text{ in}^4)$$

$$I_2 = (d+t)b^3/12 - d(b-2t_w)^3/12 = 2.110 \times 10^7 \text{ mm}^4 (7.122 \text{ in}^4)$$

The radius of gyration can then be calculated for each axis:

$$r_1 = \sqrt{\frac{I_1}{A}} = 18.7 \text{ mm (0.736 in)} \quad r_2 = \sqrt{\frac{I_2}{A}} = 59.7 \text{ mm (2.35 in)}$$

The minimum radius of gyration indicates the weakest orientation for buckling:

$$k = r_1 = 18.7 \text{ mm (0.736 in)}$$

$$\ell: \text{Length of support plate} = 160 \text{ mm (6.3 in)}$$

Also, the slenderness ratio,  $\frac{L}{k}$  is:


$$\frac{L}{k} = \frac{160}{18.7} = 8.6$$

As the ends are fixed, the coefficient “n” becomes 4, so the limit value of the slenderness ratio becomes:

$$85\sqrt{n} = 85\sqrt{4} = 170$$

Because the slenderness ratio of this material is less than the limit value slenderness ratio, Euler's equation is not applicable, and the secant formula for buckling is used. The equation to obtain the support plate's buckling strength is:

$$\frac{P}{A} = \frac{S_y}{1 + \frac{ec}{k^2} \sec \left[ \frac{C\ell}{2k} \sqrt{\frac{P}{AE}} \right]}$$

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Where:

P: Buckling strength (load) of support column N

A: Area of column =  $830.3 \text{ mm}^2$  ( $1.287 \text{ in}^2$ )

Sy: Minimum yield strength of Type 304 stainless steel = 206.8 MPa (30.0 ksi)

C: Coefficient to the long support fixed at both ends = 1.2

E: Elastic modulus of Type 304 stainless steel =  $1.95 \times 10^5 \text{ MPa}$  (Table 2-2 at 40°C)

e: Eccentricity small since the strap load is centered = 5 mm (0.2 in)

ℓ: Unsupported length of the support column = 160 mm (6.3 in)

c: Shortest distance to an outside side edge from the centroid = 19.9 mm (0.783 in)

Substituting these values in the above equation and solving for P iteratively results in a buckling strength of the support plate column of:

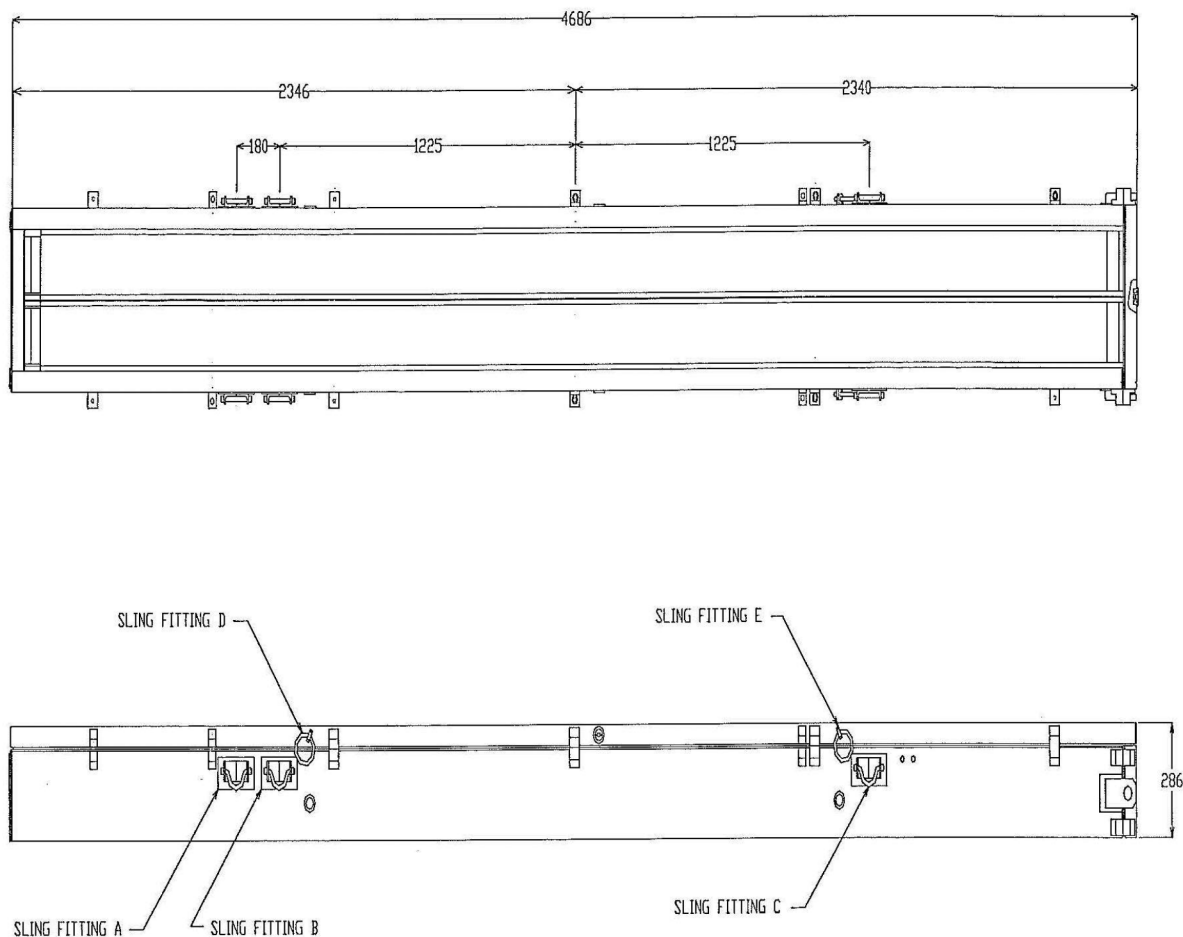
$$P = 1.332 \times 10^5 \text{ N (29,945 lb}_f\text{)}$$

There are four support columns to a side, which results in the sidewall frame having a minimum capacity of:

$$P_t = 4P = 5.328 \times 10^5 \text{ N (119,780 lb}_f\text{)}$$


Since this load capacity is greater than the applied load ( $R_t = 5.259 \times 10^5 \text{ N (1.182} \times 10^5 \text{ lb}_f\text{)}$ ), the supports will not buckle when the worst case tie-down loads are applied to a package. This capacity approaches the force required to yield the columns in compression (i.e.,  $A_{spt}S_y = 6.868 \times 10^5 \text{ N (1.544} \times 10^5 \text{ lb}_f\text{)}$ ).

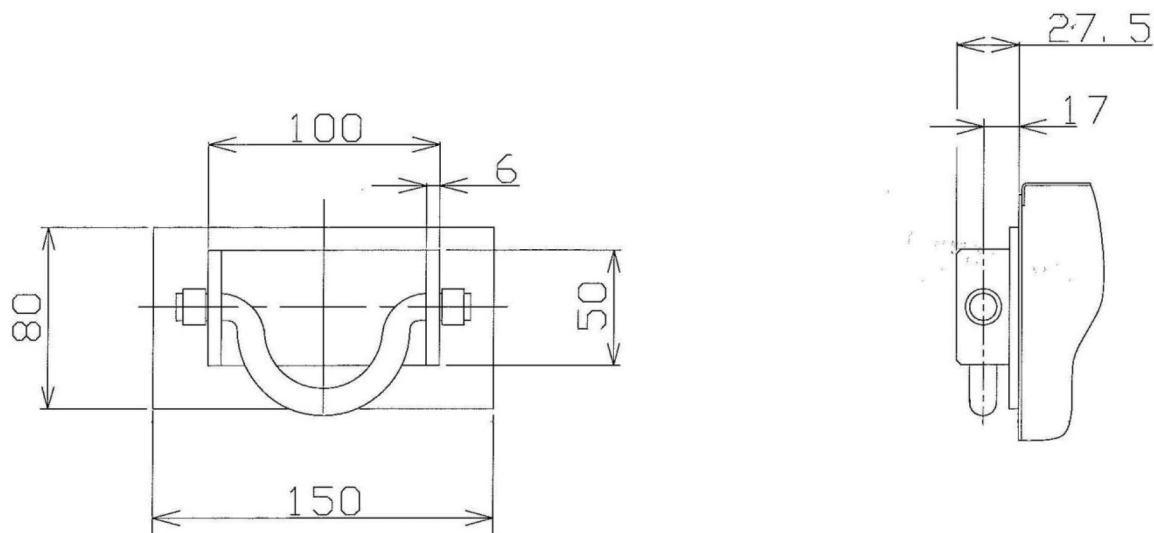
By considering the stiffening of the support plates with the reinforcement plates used to carry the inner support frame, it has been demonstrated that the support plates have sufficient capacity to react the tie-down load if the package experiences a 5 g lateral and a 2 g vertical loading simultaneously. This evaluation does not take into consideration the large carrying capability of the ends of the package where there are corner angles, end plates, and wood overlay plates that further strengthen the package's buckling capability. The use of three or more straps ensures that the load is distributed along the package so that the load can be reacted by the support plates and other internal structure. The stiffness of the OC lid, when the bolster support angles are considered with the reinforced edge of the OC body, ensures that the load is distributed to the internal structure of the package.



Combination of sling fitting	Used for
A and C	Lifting a Loaded Container
B and C	Lifting an Empty Container
D and E	Lifting a Lid

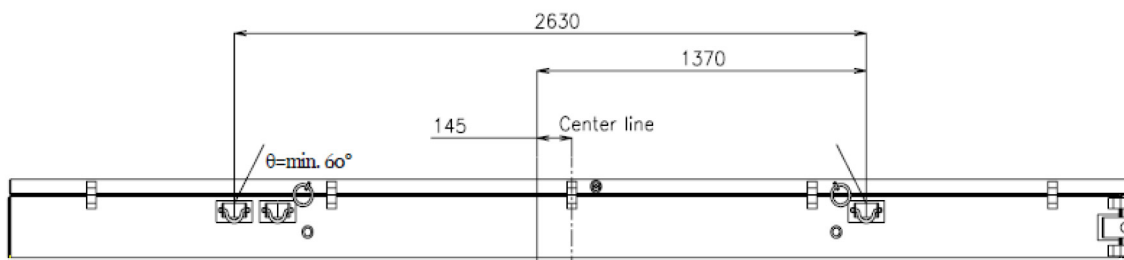
**Figure 2-2 Inner Container Sling Locations**

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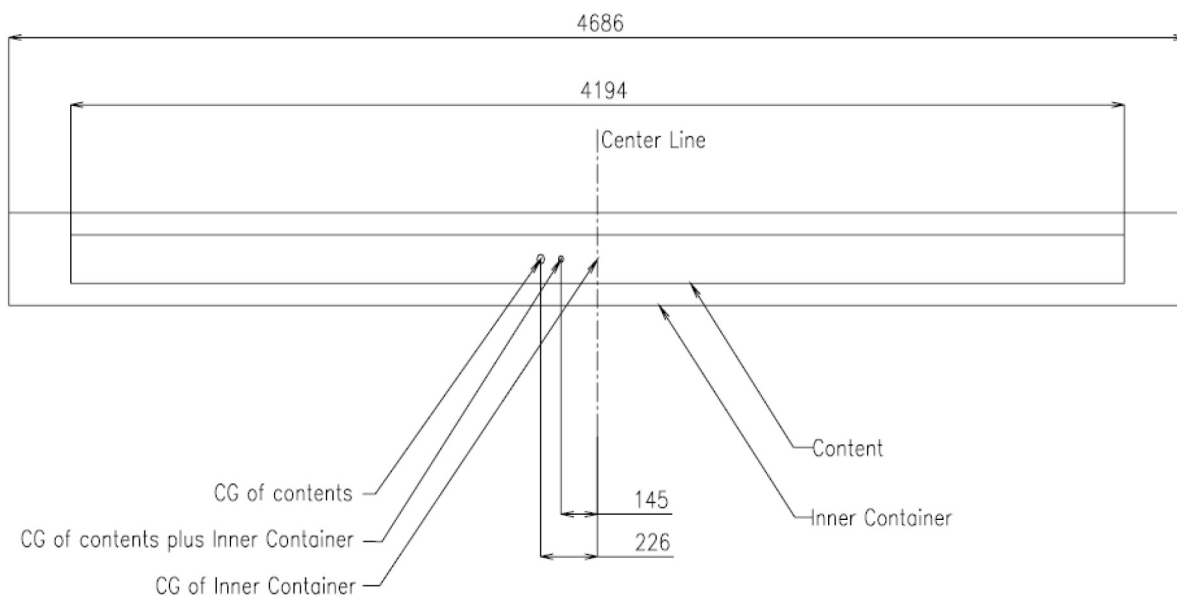


(unit: mm)

**Figure 2-3 Sling Attachment Plate Detail**




**Figure 2-4 Lifting Configuration of Inner Container**

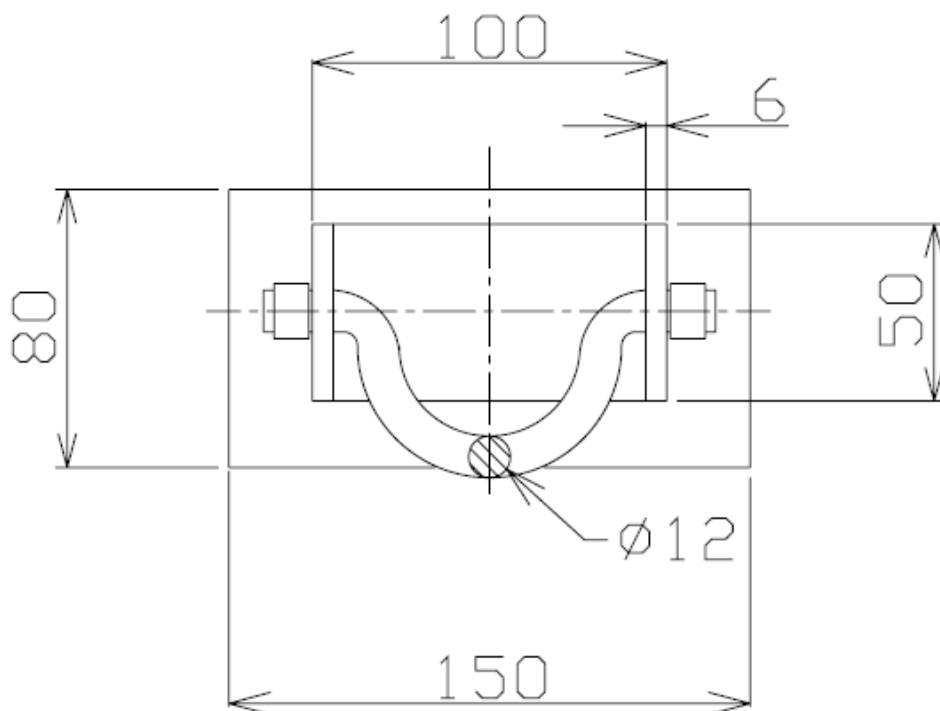


(unit: mm)

**Figure 2-5 Center of Gravity of Loaded Inner Container**




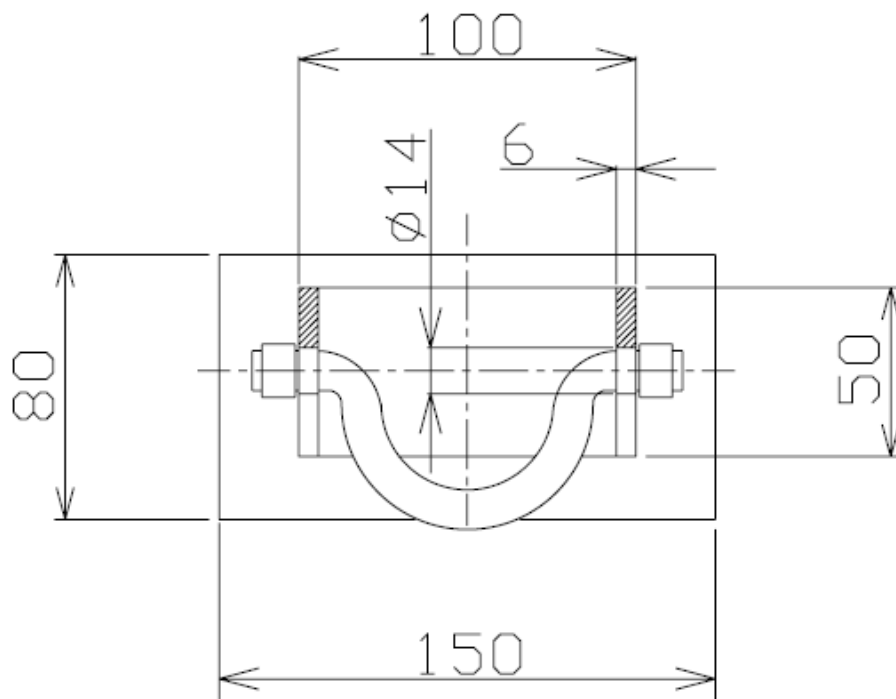
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(unit: mm)

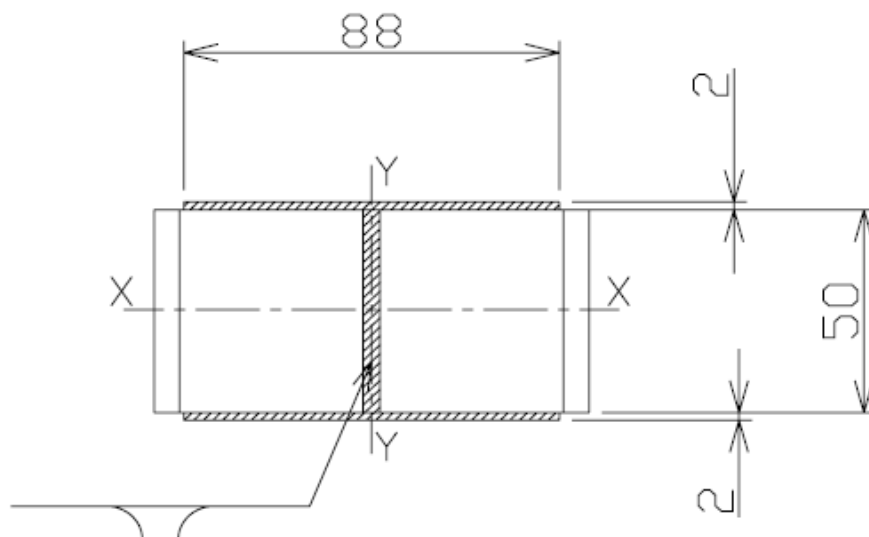
**Figure 2-6 Hooking Bar of Sling Fitting**

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(unit: mm)

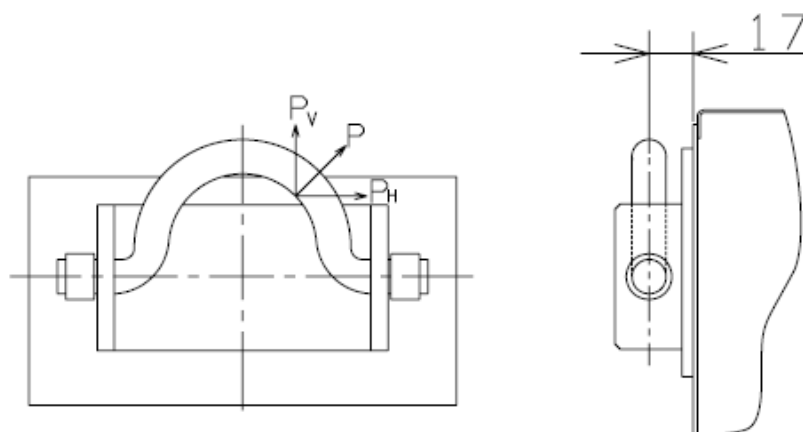
**Figure 2-7 Perforated Plate of Sling Fitting**



(unit: mm)

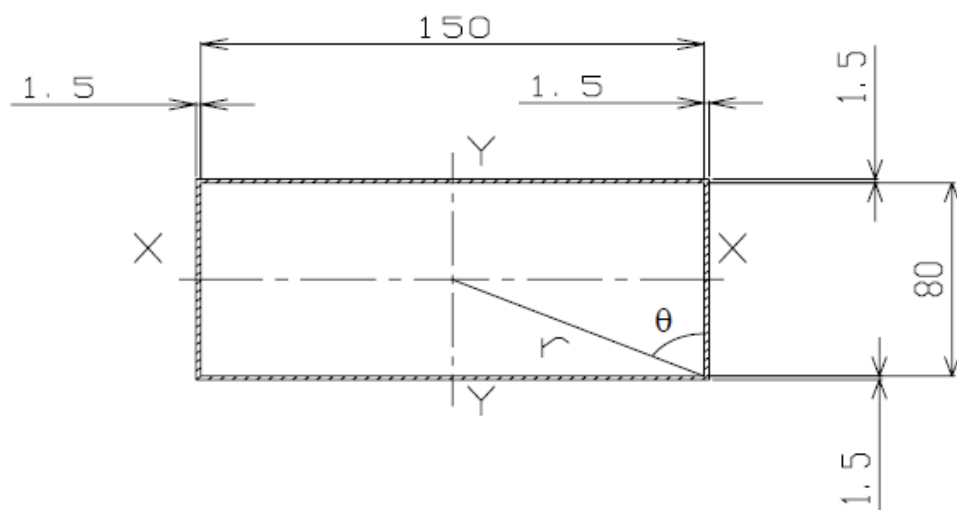
**Figure 2-8 Sling Fitting Weld Geometry for Attachment to Support Plate**

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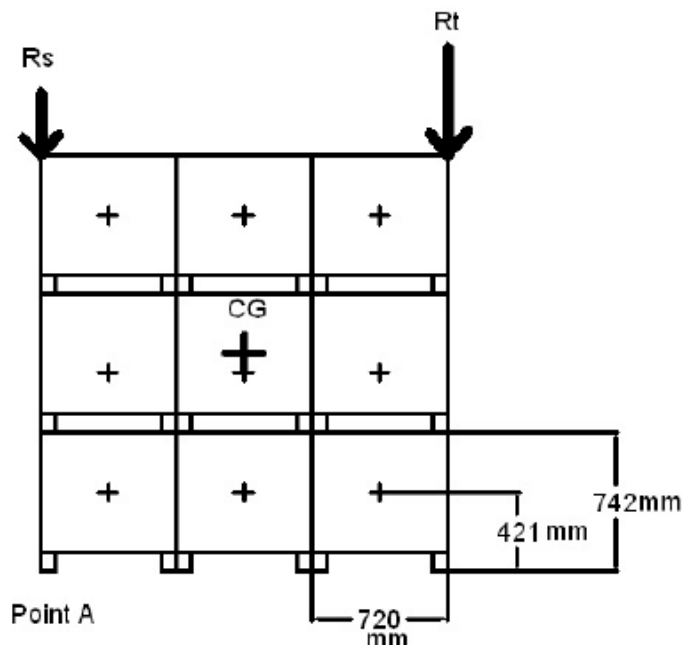
(unit: mm)

**Figure 2-9 Loads on Sling Fitting**



(unit: mm)

**Figure 2-10 Welds for Support Plate Attachment to Body**



**Figure 2-11 Tie-Down Configuration**


## 2.5. GENERAL CONSIDERATIONS

### 2.5.1. *Evaluation by Test*

The primary means of demonstrating that the package meets the regulatory accident conditions was by test. The package was tested full-scale by dropping four full-scale certification test units (CTUs) from 9 meters in different orientations. (Two of the test units were dropped as part of the Japanese certification process.)

Within the GNF-A CTUs, the fuel was mocked up by a metal boxed section that provided the representative weight in one fuel assembly shipping location. The steel section was segmented to prevent the mockup from adding unrealistic stiffness to the package. In the other fuel assembly shipping position a mockup fuel assembly was used. This had the same cross-sectional properties of the actual fuel. The rods were filled with lead to represent the actual fuel. Weights were added along side of the assembly to provide the correct mass for the fuel that may be shipped with channels as well as allowing for the different density between lead and the uranium oxide pellets.

The units tested in Japan had a simulated 8X8 fuel assembly and weights representing the other fuel assembly in each test unit. The weight and dimensions of the mockup fuel approximated the weight of the fuel to be shipped in the container.

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Details of the prototypes in the drop can be found in Section 2.7 and Appendix 2.12.

The damage caused by the test was evaluated in each of the affected sections, Section 3.0, Section 4.0, and Section 6.0. Both the inner and outer lids stayed in place, although damaged. The inner container holding frame deformed but restrained the inner container. Due to the end drop there was some plastic deformation of the fuel but well within the limits of the criticality evaluation. After the testing, the GNF-A fuel rods passed a helium leakage rate test demonstrating containment.

(Note that the TN-B1 and the RAJ-II are structurally identical so that the results from both the GNF-J and GNF-A tests apply to the TN-B1.)

### **2.5.2. *Evaluation by Analysis***

The normal conditions of transport were evaluated by analysis and by comparison to the accident testing. The primary analysis was done for the compression loading. The material properties are taken from Table 2-4, which is based on published ASME properties. A static analysis was performed in Section 2.6.9 Compression.

Since the normal condition pressure and temperatures are well below the design conditions for the fuel cladding no separate analysis was performed.

## **2.6. NORMAL CONDITIONS OF TRANSPORT**

The TN-B1 package, when subjected to the Normal Conditions of Transport (NCT) specified in 10 CFR 71.71, is shown to meet the performance requirements specified in Subpart E of 10 CFR 71. As discussed in the introduction to this chapter, with the exception of the NCT free drop, the primary proof of NCT performance is via analytic methods. Regulatory Guide 7.6 criteria are demonstrated as acceptable for NCT analytic evaluations presented in this section. Specific discussions regarding brittle fracture and fatigue are presented in Sections 2.1.2.4 and 2.6.5 and are shown not to be limiting cases for the TN-B1 package design. The ability of the welded containment fuel rod cladding to remain leak-tight is documented in Section 4.0.

Properties of Type 304 stainless steel as representative of those properties for 300 series stainless steel are summarized below.

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**Table 2-5 Material Properties**

Material Property	Material Property Value (psi)			Reference
	-40 °C (-40 °F)	21°C (70 °F)	75°C (167°F)	
	Type 304 Stainless Steel			
Elastic Modulus, E	198.6GPa (28.8×10 <sup>6</sup> psi)	195.1GPa (28.3×10 <sup>6</sup> psi)	191.7GPa (27.8×10 <sup>6</sup> psi)	Table 2-2
Design Stress Intensity, S <sub>m</sub>	137.9MPa (20,000 psi)	137.9MPa (20,000 psi)	137.9MPa (20,000 psi)	
Yield Strength, S <sub>m</sub>	206.8MPa (30,000psi)	206.8MPa (30,000psi)	184.7MPa (26,788psi)	
Tensile Strength	517.1MPa (75,000psi)	517.1MPa (75,000psi)	498.6MPa (72,300)	


The TN-B1 package's ability to survive HAC, 30-foot free drop, 40-inch puncture drop, and 30-minute thermal event also demonstrated the packages ability to also survive the NCT. Evaluations are performed, when appropriate, to supplement or expand on the available test results. This combination of analytic and test structural evaluations provides an initial configuration for NCT thermal, shielding and criticality performance. In accordance with 10 CFR 71.43(f), the evaluations performed herein successfully demonstrate that under NCT tests the TN-B1 package experiences "no substantial reduction in the effectiveness of the packaging". Summaries of the more significant aspects of the full-scale free drop testing are included in Section 2.6.7, with details presented in Appendix 2.12.1.

### **2.6.1. Heat**

The NCT thermal analyses presented in Section 3.0, consist of exposing the TN-B1 package to direct sunlight and 100 °F still air per the requirements of 10 CFR 71.71(b). Since there is negligible decay heat in the unirradiated fuel, the entire heating came from the solar insolation. The maximum temperature of 77°C (171°F) was located on the lid of the outer container.

#### **2.6.1.1. Summary of Pressures and Temperatures**

The fuel assembly exhibits negligible decay heat. The TN-B1 package and internal components, when loaded with the required 10 CFR 71.71(c) (1) insulation conditions, develop a maximum temperature of 77 °C (171 °F). The resulting pressure at the maximum temperature is 1.33 MPa (192.9 psia).

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### 2.6.1.2. Differential Thermal Expansion

With NCT temperatures throughout the packaging being relatively uniform (i.e. no significant temperature gradients), the concern with differential expansions is limited to regions of the TN-B1 packaging that employ adjacent materials with sufficiently different coefficients of thermal expansion. The IC is a double-walled, composite construction of alumina silicate thermal insulator between inner and outer walls of stainless steel. The alumina silicate thermal insulator is loosely packed between the two walls and does not stress the walls. Differential thermal expansion stresses are negligible in the OC for three reasons: 1) the temperature distribution throughout the entire OC is relatively uniform, 2) the OC is fabricated from only one type of structural material, and 3) the OC is not radially or axially constrained within a tight-fitting structure due to the relatively low temperature differentials and lack of internal restraint within the TN-B1 package.

The cladding of the fuel which serves as containment is not stressed due to differential thermal expansion since a gap remains between the fuel pellet and the cladding at both the cold temperature -40°C and the highest temperature the fuel could see due to the HAC which is 800°C. This is demonstrated as follows:

The nominal fuel pellet and cladding dimensions and the resulting radial gap (0.00335 inches) is shown below based on a temperature of 20°C:

As-Built Dimensions (inches)		
Nominal Clad OD	D <sub>co</sub>	0.3957
Nominal Clad ID	D <sub>ci</sub>	0.348
Nominal Pellet OD	D <sub>fo</sub>	0.3413
Nominal Radial Pellet/Clad	g <sub>n</sub>	0.00335

The strain due to thermal expansion or contraction in the Zr cladding is equal to\*


$$\left(\frac{\Delta D}{D}\right)_{clad} = 7.4 \times 10^{-6} (\Delta T)$$

Where  $\Delta T$  is positive for an increase in temperature and negative for a decrease in temperature.

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\* Framatome ANP MOX Material Properties Manual 51-5010288-03



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The strain due to thermal expansion or contraction in the fuel pellet is equal to\*:

$$\left(\frac{\Delta D}{D}\right)_{clad} = -3.28 \times 10^{-3} + 1.179 \times 10^{-5} T - 2.429 \times 10^{-9} T^2 + 1.219 \times 10^{-12} T^3$$

Where T is the absolute final temperature in degrees Kelvin (K).

The following table summarizes the thermal strain and the thermal growth in the cladding and pellets with a temperature change from 20°C to -40°C ( $\Delta T = -60^\circ\text{C}$ ,  $T = 233\text{K}$ ). All dimensions are expressed in inches.

**Table 2-6 Thermal Contraction at -40°C**

	Strain at -40°C $\left(\frac{\Delta D}{D}\right)$	Thermal Expansion at -40°C $\left(\frac{\Delta D}{D}\right) D$	Dimension at -40°C $D + \left(\frac{\Delta D}{D}\right) D$
Pellet OD	$-6.49 \times 10^{-4}$	$-2.22 \times 10^{-4}$	0.3411
Cladding ID	$-4.44 \times 10^{-4}$	$-1.55 \times 10^{-4}$	0.3478

This results in a radial gap at -40°C of:


$$g_{-40} = \frac{0.3478 - 0.3411}{2} = 0.0034 \text{ in}$$

The following table summarizes the thermal strain and the thermal growth in the cladding and pellets with a temperature change from 20°C to 800°C ( $\Delta T = 780^\circ\text{C}$ ,  $T = 1,073\text{K}$ ). All dimensions are expressed in inches.

**Table 2-7 Thermal Expansion at 800°C**

	Strain at 800°C $\left(\frac{\Delta D}{D}\right)$	Thermal Expansion at 800°C $\left(\frac{\Delta D}{D}\right) D$	Dimension at 800°C $D + \left(\frac{\Delta D}{D}\right) D$
Pellet OD	$8.08 \times 10^{-3}$	$2.76 \times 10^{-3}$	0.3441
Cladding ID	$5.77 \times 10^{-3}$	$2.01 \times 10^{-3}$	0.3500

\* Framatome ANP MOX Material Properties Manual 51-5010288-02

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This results in a radial gap at 800°C of:

$$g_{800} = \frac{0.3500 - 0.3411}{2} = 0.0030 \text{ in}$$

### 2.6.1.3. Stress Calculations

Since the temperatures and pressures generated under normal conditions of transport are well below the design conditions for the boiling water reactor fuel no specific calculations were performed for the fuel containment.

### 2.6.1.4. Comparison with Allowable Stresses

The normal conditions of transport conditions are well below the operating conditions of the fuel no comparison to allowable stresses was performed.

## 2.6.2. Cold

The NCT cold condition consists of exposing the TN-B1 packaging to a steady-state ambient temperature of -40 °F. Insulation and payload internal decay heat are assumed to be zero. These conditions will result in a uniform temperature throughout the package of -40 °F. With no internal heat load (i.e., no contents to produce heat), the net pressure differential will only be reduced from the initial conditions at loading.

For the containment, the principal structural concern due to the NCT cold condition is the effect of the differential expansion of the fuel to the zirconium alloy tube. During the cool-down from 20 °C to -40 °C, the tube could shrink onto the fuel because of difference in the thermal expansion coefficient. However, the clearance between the fuel and the cladding is such that even if the fuel did not shrink, there would still be clearance. Differential thermal expansion stresses are negligible in the package for three reasons: 1) the temperature distribution throughout the entire package is relatively uniform, 2) the package is fabricated from only one type of structural material, and 3) the package is not radially or axially constrained.

Brittle fracture at -40 °F is addressed in Section 2.1.2.4.1.

## 2.6.3. Reduced External Pressure

The effect of a reduced external pressure of 25 kPa (3.5 psia) per 10 CFR 71.71(c)(3) is negligible for the TN-B1 packaging. The TN-B1 package contains no pressure-tight seal and therefore cannot develop differential pressure. Therefore, the reduced external pressure requirement of 3.5 psia delineated in 10 CFR 71.71(c)(3) will have no effect on the package. Compared with the 1.115 MPa (161.7 psia) internal pressure in the fuel rods, a reduced external pressure of 3.5 psia will have a negligible effect on the fuel rods.

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#### **2.6.4. Increased External Pressure**

The TN-B1 package contains no pressure-tight seal and, therefore, cannot develop differential pressure. Therefore, the increased external pressure requirement of 140 kPa (20 psia) delineated in 10 CFR 71.71(c)(4) will have no effect on the package. The pressure-tight cladding of the fuel rods is designed for much higher pressures in its normal service in a reactor and is not affected by the slight increase in external pressure.

The containment is provided by the cladding tubes of the fuel. These tubes, designed for the conditions in an operating reactor, have the capability of withstanding the increased external pressure. The failure mode of radial buckling is not a plausible failure mode since the fuel pellets would prevent any significant deformation due to external pressure.

#### **2.6.5. Vibration**

The TN-B1 packaging contains an internal shock mount system and, therefore, cannot develop significant vibratory stresses for the package's internal structures. Therefore, vibration normally incident to transportation, as delineated in 10 CFR 71.71(c)(5), will have a negligible effect on the package. Due to concerns of possibly damaging the fuel so it cannot be installed in a reactor after transport, extreme care is taken in packaging the fuel using cushioning material and vibration isolation systems. These systems also ensure that the fuel containment boundary also remains uncompromised. The welded structure of the light weight TN-B1 package is unaffected by vibration. However, after each use the packaging is visually examined for any potential damage.

#### **2.6.6. Water Spray**

The materials of construction of the TN-B1 package are such that the water spray test identified in 10 CFR 71.71(c)(6) will have a negligible effect on the package.

#### **2.6.7. Free Drop**

Since the maximum gross weight of the TN-B1 package is 1,614 kg (3,558 lb), a 1.2 m or four-foot free drop is required per 10 CFR 71.71(c)(7). The Hypothetical Accident Condition (HAC), 9 m (30 foot) free drop test required in 10 CFR 71.73(c)(1) is substantially more damaging than the 1.2 m (4 foot) NCT free drop test. Section 2.7.1 demonstrates the TN-B1 package's survivability and bounds the free drop requirements of 10 CFR 71.71(c)(7). Due to the relatively fragile nature of the fuel assembly payload in maintaining its configuration for operational use, any event that would come close to approximating the NCT free drop would cause the package to be removed from service and re-examined prior to continued use.

As part of the effort to obtain package certification in Japan by GNF-J, certification testing of the package, which included both an end drop and a lid-down horizontal drop, was performed. In

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each case a 0.3-meter (1-foot) and a 1.2 meter (4-foot) drop was performed prior to the 9-meter (30-foot) drop. In both cases the test package was slightly damaged but the damage had no significant effect on the performance of the package in relation to either the containment or the ability of the package to meet the requirements of 10 CFR 71. The GNF-J certification testing is discussed in Appendix 2.12.2.

Therefore, the requirements of 10 CFR 71.71(c)(7) are met.

#### **2.6.8. Corner Drop**

This test does not apply, since the package weight is in excess of 100 kg (220 pounds), and the structural materials used in the TN-B1 are not primarily wood or fiberboard, as delineated in 10 CFR 71.71(c)(8).

#### **2.6.9. Compression**

Since the package weighs less than 5,000 kg (11,000 pounds), as delineated in 10 CFR 71.71(c)(9), the package must be able to support five times its weight without damage.

The load to be given as the test condition is the load ( $W_1$ ) times five of the weight of this package or the load ( $W_2$ ) which is obtained through multiplying the package's vertical projected area by 13 kPa, whichever is heavier. In the case of this package, the equations to obtain each load are:

$$W_1 = 5 \times m \times g$$

$$W_2 = 13 \text{ kPa} \times L \times B$$

Where:

m: Mass of package	1,614 kg (3,558 lb)
g: Gravitational acceleration	9.81 m/s <sup>2</sup>
L: Length of package	5,068 mm (199.53 in)
B: Width of package	720 mm (28.35 in)


From this

$$W_1 = 5 \times 1,614 \times 9.81 = 79.16 \text{ kN (17,800 lbf)}$$

$$W_2 = 13 \times 10^{-3} \times 5,068 \times 720 = 47.4 \text{ kN (10,660 lbf)}$$

Therefore, as  $W_1 > W_2$ , the stacking load is assumed as  $W = 79.16 \text{ kN (17,800 lbf)}$ .

The stacking of these packages is as shown in **Figure 2-12**, so the outer container only sustains the stacking load. In this case, it is assumed that loads are carried by a total of eight support plates positioned in the center of the bolster out of sixteen support plates of the outer container

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body positioned at the lowest layer. This assumption makes the load sustaining area smaller, so the evaluation is conservative. The compressive load given to the support plate is the above-mentioned stacking load plus the weight of the outer container's lid.

The equation to obtain the support plate's compressive load is:

$$W_c = W_1 + W_3$$

$W_c$ : Compressive load N

$W_1$ : Stacking load 79.16 kN (17,800 lbf)

$W_3$ : Load by the outer container's lid 1.34 kN (301 lbf)

$m_F$ : Mass of outer container lid 137 kg (302 lb)

$g$ : Gravitational acceleration 9.81m/s<sup>2</sup>

From this, the 80.5 kN (18,100 lbf)

When the fuel assemblies are packed, the gravity center of the outer container is shifted longitudinally, so the load acting on the support plate, which is closer to the gravity center, becomes larger.

Therefore, the equation to obtain the vertical maximum load given to one support plate, which is closer to the gravity center, is:

$$P = \frac{W}{4} \frac{\ell_2}{\ell_0}$$

Where:


$P$ : Maximum load acting on one support plate  
which is nearer to the gravity center N

$W$ : Compressive load given to the support plate 80.5 kN (18,100 lbf)

$\ell_0$ : Longitudinal support plate space 3,510 mm (138.2 in)

$\ell_2$ : Distance from the package's gravity center position  
to the support

$$\frac{3,510}{2} + 92 = 1,847 \text{ mm (73.76 in)}$$

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From this, the maximum load P acted to one support plate, which is nearer to the gravity center, is:

$$P = \frac{80.5 \times 10^3 \times 1,847}{4 \times 3,510} = 10.6 \times 10^3 \text{ N (2,380 lbf)}$$

The resistance of the plate to buckling is also evaluated. The equation to obtain the moment of inertia of area of the support plate which is subject to buckling is:

$$I_z = \frac{1}{12} hb^3$$

Where:

$I_z$ : Moment of inertia of area of support plate mm<sup>4</sup>

b: Thickness of support plate 5 mm (0.2 in)

h: Width of support plate 55 mm (2.2 in)

From this, the moment of inertia of area,  $I_z$ , of the support plate is:

$$I_z = \frac{1}{12} \times 55 \times 5^3 = 572.9 \text{ mm}^4 (1.376 \times 10^{-3} \text{ in}^4)$$

Also, the equation to obtain the radius of gyration of the area of the support plate is:

$$k = \sqrt{\frac{I_z}{A}}$$

Where:

k: Radius of gyration of area of support plate mm

$I_z$ : Moment of inertia of area of support plate 572.9 mm<sup>4</sup> (1.376x10<sup>-3</sup> in<sup>4</sup>)

A: Cross-sectional area of support plate 5 × 55 = 275 mm<sup>2</sup> (0.426 in<sup>2</sup>)

ℓ: Length of support plate 559 mm (22.4 in)

From this, the radius of gyration of area k of the support plate is:

$$k = \sqrt{\frac{572.9}{275}} = 1.44 \text{ mm (0.0568 in)}$$

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Also, the slenderness ratio  $\frac{\ell}{k}$  is:

$$\frac{\ell}{k} = \frac{559}{1.44} = 388$$

As the ends are fixed, the coefficient n becomes 4, so the limit value of the slenderness ratio becomes as below.

$$85\sqrt{n} = 85\sqrt{4} = 170$$

Because the slenderness ratio of this material, 388, exceeds the limit value of slenderness, Euler's equation is used. The equation to obtain the support plate's buckling strength is:

$$P_k = \frac{n\pi^2 E I_z}{\ell^2}$$

Where:

$P_k$ : Buckling strength (load) of support plate	N
n: Coefficient to the long support fixed at both ends	4
E: Longitudinal elasticity modulus of Gr304 stainless steel	$1.94 \times 10^5$ MPa (at 40°C)
$I_z$ : Moment of inertia of area of support plate	$572.9 \text{ mm}^4$ ( $1.376 \times 10^{-3} \text{ in}^4$ )
$\ell$ : Length of the support plate	559 mm (22.4 in)

From this, the buckling strength  $P_k$  of the support plate is:


$$P_k = \frac{4 \times 3.14^2 \times 1.94 \times 10^5 \times 572.9}{559^2} = 14 \times 10^3 \text{ N (3,050 lb)}$$

Therefore,  $P_k > P$ , so the body support plate will not buckle.

#### 2.6.10. **Penetration**

The one-meter (40-inch) drop of a 6 kg (13-pound), hemispherical-headed, 3.2 cm (1.3-inch) diameter, steel cylinder, as delineated in 10 CFR 71.71(c)(10), is of negligible consequence to the TN-B1 package. This is due to the fact that the TN-B1 package is designed to minimize the consequences associated with the much more limiting case of a 40-inch drop of the entire

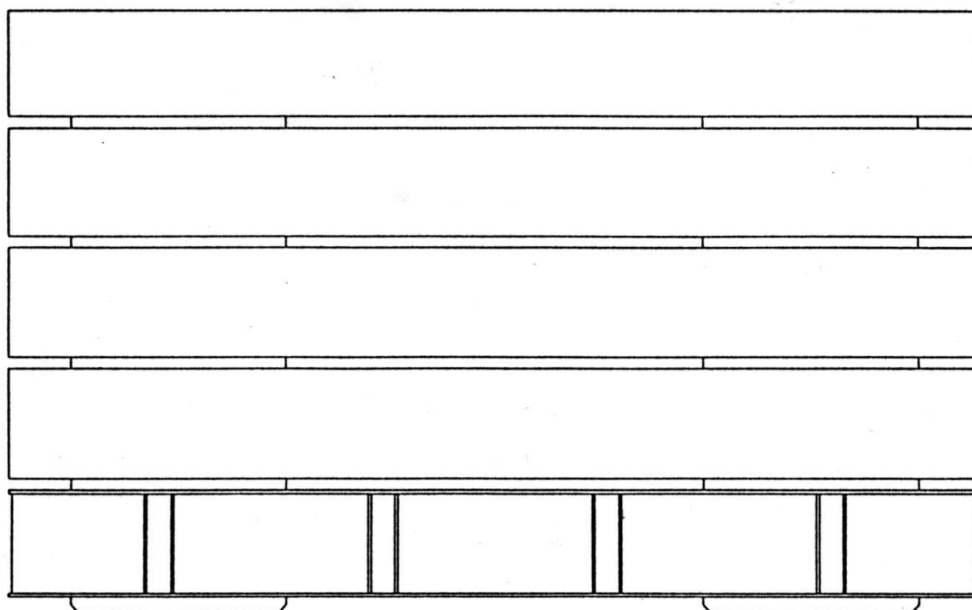


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package onto a puncture bar as discussed in Section 2.7.3. The drop of the 6 kg bar will not damage the outer container.

**Table 2-8 Temperatures**

Location	Maximum temperature
Environment (Open air)	38°C
Package's external surface	77°C
Inner container	<77°C



**Figure 2-12 Stacking Arrangement**

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## 2.7. HYPOTHETICAL ACCIDENT CONDITIONS

The TN-B1 package, when subjected to the sequence of Hypothetical Accident Condition (HAC) tests specified in 10 CFR 71.73 is shown to meet the performance requirements specified in Subpart E of 10 CFR 71. The primary proof of performance for the HAC tests is via the use of full-scale testing. A certification test unit (CTU) was free dropped, and puncture tested to confirm that both the inner and outer containers protected the fuel and allowed containment to be maintained after a worst-case HAC sequence. Another CTU was free dropped from 9 meters on its end with the fuel maintaining containment after the drop. Observations from CTU testing confirm the conservative nature of the deformed geometry assumptions used in the criticality assessment provided in Chapter 6.0. Immersion is addressed by comparison to the design basis for the fuel. Test results are summarized in Section 2.7.8, with details provided in Appendix 2.12.1.

### 2.7.1. *Free Drop*

Subpart F of 10 CFR 71 requires performing a free drop test in accordance with the requirements of 10 CFR 71.73(c)(1). The free drop test involves performing a 30-foot, HAC free drop onto a flat, essentially unyielding, horizontal surface, with the package striking the surface in a position (orientation) for which maximum damage is expected. The ability of the TN-B1 package to adequately withstand this specified free drop condition is demonstrated via testing of four full-scale, certification test units (CTUs).

To properly select a worst-case package orientation for the 30-foot free drop event, items that could potentially compromise containment integrity, shielding integrity, and/or criticality safety of the TN-B1 package must be clearly identified. For the TN-B1 packaging design, there are two primary considerations 1) protect the fuel so that containment is maintained and 2) ensure sufficient structure is around the package to maintain the geometry used in the criticality safety evaluation. Shielding integrity is not a controlling case for the reasons described in Section 5.0. Criticality safety is conservatively evaluated based on measured physical damage to the outer container from certification testing, as described in Section 6.0.

Since the containment is welded closed, the leak-tight capability of the containment may be compromised by two methods: 1) as a result of excessive deformation leading to rupture of the containment boundary, and/or 2) as a result of thermal degradation of the containment material itself in a subsequent fire event and rupture of the weld or the cladding tube by over-pressurization. Importantly, these methods require significant impact damage to the surrounding outer and inner container so that the fuel is either loaded externally or the fuel is directly exposed to the fire.

Additional items for consideration include the possibility of separating the OC lid from the OC body and buckling or deforming of the Outer Container (OC) and/or Inner Container (IC) from an end drop or horizontal drop.

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For the above reasons, testing must include impact orientations that affect the lid and stability of the walls of the containers. In general, the energy absorbing capabilities of the TN-B1 are governed by the deformation of the stainless steel and impregnated paper honeycomb that is not significantly affected by temperature.

Appendices 2.12.1 and 2.12.2 provide a comprehensive report of the certification test process and results. Discussions specific to CTU test orientations for free drop and puncture, including initial test conditions, are also provided.

The TN-B1 package has undergone extensive testing during its development. Testing has included 1.2-meter (4-foot) drops on the end in the vertical orientation and the lid in the horizontal orientation. The package has been also dropped from 9 meters in the same orientation demonstrating that the damage from the 1.2-meter (4-foot) drops has little consequence on the performance of the package in 9-meter (30-foot) drop. Based on these preliminary tests it was determined that the worst case orientation for the 9-meter (30-foot) drop test would be slap-down on the lid. The lid down drop demonstrated that the vibration isolation frame bolts would fail allowing the inner container to come in contact with the paper honeycomb in the lid and partially crush the honeycomb. It was expected that the slap-down orientation would maximize the crush of this material minimizing the separation distance between the fuel assemblies in the post accident condition.

A single “worst-case” 9-meter (30-foot) free drop is required by 10 CFR 71.73(c)(1). Based on the above discussion and experience with other long slender packages similar to the TN-B1, a 15 degree slap-down on the lid was chosen for the 9-meter (30-foot) drop. Following that drop, a 25 degree oblique puncture drop on the damaged lid was performed. See Figure 2-13, Figure 2-14 and Appendix 2.12.1.

Other free drop orientations that were tested include vertical end and bottom corner. These tests demonstrated that the TN-B1 package contains the fuel assemblies without breaching the fuel cladding (containment boundary).

#### 2.7.1.1. End Drop

9-meter (30-foot) end free drops were performed on GNF-J CTU 1J and GNF-A CTU 2. The orientation was selected with the lower end of the fuel down to maximize the damage since the expansion springs in the fuel rods are located in the upper end. This orientation maximized the damage to the energy absorbing wood in the end of the TN-B1 and maximized the axial loading on the fuel assembly. Both tests resulted in deformations of the fuel but were within the limits evaluated in the criticality evaluation in Section 6.0. Following the GNF-A tests, the fuel rods were demonstrated to maintain containment after the free and puncture drops, thus maintaining its containment boundary integrity. Although this orientation caused the most severe damage to the fuel, the damage was well within the structural limits for the fuel and package.

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#### 2.7.1.2. Side Drop

No side drop testing was performed in this certification sequence. A side drop test was done in previous testing of the package. That testing resulted in the inner container holding frame top bolts failing and allowing the inner container to come in contact with the outer lid. The inner package showed little damage and the fuel was not deformed. It was judged that the slapdown and the horizontal drop tests bounded the side drop orientation.

#### 2.7.1.3. Corner Drop

A 9-meter (30-foot) free drop on the OC body bottom corner was performed on GNF-J CTU 1J. The impact point previously sustained damage due to 0.3-meter (1-foot) and 1.2-meter (4-foot) free drops. The resultant cumulative deformation was approximately 163 mm (6 inches). There was no loss of contents or significant structural damage to the OC as a result of this free drop. The maximum recorded impact acceleration was 203g. Refer to Appendix 2.12.2 for complete details of the corner free drop.

#### 2.7.1.4. Oblique Drops

An orientation of 15 degrees from horizontal was tested with GNF-A CTU 1. The IC holding frame was plastically deformed and only a portion of the bolts failed. Neither the fuel nor the IC were not significantly damaged. The damage sustained was bounded by the assumptions utilized in the criticality and thermal evaluations. The fuel was leak tested after the test and was demonstrated to have maintained containment boundary. Refer to Appendix 2.12.1 for complete details of the 15-degree oblique free drop.

#### 2.7.1.5. Horizontal Drop

A 9-meter (30-foot) horizontal free drop on the OC lid was performed on GNF-J CTU 2J. The impact results in a maximum deformation of 19 mm (0.8 inch), which occurred in the OC lid. The side wall of the OC body bulged approximately 19 mm (0.8 inches). Some localized weld failure of OC lid flange/OC lid interface occurred where the bolster angles attach to the lid. None of the OC lid bolts failed as a result of the impact. There was no loss of contents as a result of the free drop. The maximum recorded impact acceleration was 146g. Refer to Appendix 2.12.2 for complete details of the horizontal free drop.

#### 2.7.1.6. Summary of Results

Successful HAC free drop testing of the test units indicates that the various TN-B1 packaging design features are adequately designed to withstand the HAC 30-foot free drop event. The most important result of the testing program was the demonstrated ability of the fuel to remain undamaged and hence maintain its containment capability as defined by ANSI N14.5.

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The TN-B1 also maintained its basic geometry required for nuclear criticality safety. Observed permanent deformations of the TN-B1 packaging were less than those assumed for the criticality evaluation.

The GNF-A mock-up fuel assembly rods were leakage rate tested after the conclusion of the testing and were demonstrated to be leaktight, as defined in ANSI N14.5.

A comprehensive summary of free drop test results are provided in Appendices 2.12.1 and 2.12.2.

### **2.7.2. Crush**

Subpart F of 10 CFR 71 requires performing a dynamic crush test in accordance with the requirements of 10 CFR 71.73(c)(2). Since the TN-B1 package weight exceeds 500 kg (1,100 pounds), the dynamic crush test is not required.

### **2.7.3. Puncture**

Subpart F of 10 CFR 71 requires performing a puncture test in accordance with the requirements of 10 CFR 71.73(c)(3). The puncture test involves a 1-meter (40-inch) free drop of a package onto the upper end of a solid, vertical, cylindrical, mild steel bar mounted on an essentially unyielding, horizontal surface. The bar must be 150 mm (6 inches) in diameter, with the top surface horizontal and its edge rounded to a radius of not more than 6 millimeter (0.25 inch). The package is to be oriented in a position for which maximum damage will occur. The length of the bar used was approximately 1.5 meters (60 inches). The ability of the TN-B1 package to adequately withstand this specified puncture drop condition is demonstrated via testing of the full-scale RAJ-II CTUs.

To properly select a worst-case package orientation for the puncture drop event, items that could potentially compromise containment integrity and/or criticality safety of the TN-B1 package must be clearly identified. For the TN-B1 package design, the foremost item to be addressed is the ability of the containment to remain leak-tight. Shielding integrity is not a controlling case for the reasons described in Chapter 5.0. Criticality safety is conservatively evaluated based on measured physical damage to the outer container walls as described in Section 6.0.

Previous testing has shown that the 1-meter drop onto the puncture bar did not penetrate the outer wall or damage the fuel. Based on this previous testing and other experience, an oblique and horizontal puncture drop orientations centered over the fuel were chosen as the most damaging.

Appendices 2.12.1 and 2.12.2 provide a comprehensive report of the certification test process and results. Discussions specific to the configuration and orientation of the test unit are provided.

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The “worst-case” puncture drop as required by 10 CFR 71.73(c)(3) was performed on the package with the lid down and 25 degrees from horizontal. The angle was chosen based on experience with other packages and the TN-B1. The puncture bar was aimed at the CG of package to maximize the energy imparted to the package.

The puncture pin did not penetrate the outer container. It deformed the lid inward and it contacted the inner container lid and deformed it a small amount. The outer lid total deformation was less than 12 cm (4.7 inches) and the inner container lid deformed less than 5 cm (2.0 inches).

#### **2.7.4. Thermal**

Thermal testing of the GNF-J CTU 2J was performed following the free drop and puncture drop tests (refer to Appendix 2.12.2). Although there was no failure of the containment boundary due to the thermal testing, the thermal evaluation of the TN-B1 package for the HAC heat condition as presented in Section 3.0, demonstrates the regulatory compliance to 10 CFR 71.73(c)(4). Because the TN-B1 package does not contain pressure-tight seals, the HAC pressure for the OC and the IC is zero. The fuel assembly exhibits negligible decay heat.

##### **2.7.4.1. Summary of Pressures and Temperatures**

The maximum predicted HAC temperature for the fuel assembly is 921 K (1,198°F) during the fire event. The fuel rods are designed to withstand a minimum temperature of 1,073 K (1,475°F) without bursting. This has been demonstrated by heating representative fuel rods to this temperature for over 30 minutes. This heating resulted in rupture pressures in the excess of 3.6MPa (520 psi). The pressure due to the accident conditions does not exceed 3.5 MPa (508 psia). Summary of pressures and related stresses are provided in Section 3.0.

##### **2.7.4.2. Differential Thermal Expansion**

The fuel cladding is not restricted by the packaging and hence can not develop any significant differential thermal expansion stresses. The packaging itself is made of the same metal (austenitic stainless steel) eliminating any significant stresses due to differential thermal expansion.

##### **2.7.4.3. Stress Calculations**

Stress calculations for the controlling hoop stress for the fuel cladding that provides containment is provided in Section 3.0.

##### **2.7.4.4. Comparison with Allowable Stresses**

The allowable stress used in the analysis in Section 3.0 is based on empirical data from burst tests performed on fuel rods when heated to 800 °C and above. The allowed fuel cladding

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configurations for the TN-B1 have a positive margin of safety based on stresses required to fail the fuel in the test.

#### **2.7.5. Immersion – Fissile Material**

Subpart F of 10 CFR 71 requires performing an immersion test for fissile material packages in accordance with the requirements of 10 CFR 71.73(c)(5). The criticality evaluation presented in Chapter 6.0 assumes optimum hydrogenous moderation of the contents, thereby conservatively addressing the effects and consequences of water in-leakage.

#### **2.7.6. Immersion – All Packages**

Subpart F of 10 CFR 71 requires performing an immersion test for packages in accordance with the requirements of 10 CFR 71.73(c)(6). Since the TN-B1 package is not sealed against pressure, there will not be any differential pressure with the water immersion loads defined in 10 CFR 71.73(c)(6). The water immersion will have a negligible effect on the container and the payload, consisting of the fuel assemblies that provide the containment. The fuel rods are designed to withstand differential pressures greater than 1,000 psi. Submergence is a normal design condition for the fuel assemblies and the evaluations are performed on that condition.


#### **2.7.7. Deep Water Immersion Test (for Type B Packages Containing More than $10^5 A_2$ )**

Not applicable. The TN-B1 does not contain more than  $10^5 A_2$ .

#### **2.7.8. Summary of Damage**

As discussed in the previous sections, the cumulative damaging effects of the free drops and a puncture drop were satisfactorily withstood by the RAJ-II packaging during certification testing. Subsequent helium leak testing confirmed that containment integrity was maintained throughout the test series. The package was also successfully evaluated for maintaining containment during and after the fire event. The deformation of the package in the worst case HAC did not exceed that which is evaluated for in Chapter 6.0. Therefore, the requirements of 10 CFR 71.73 have been satisfied.



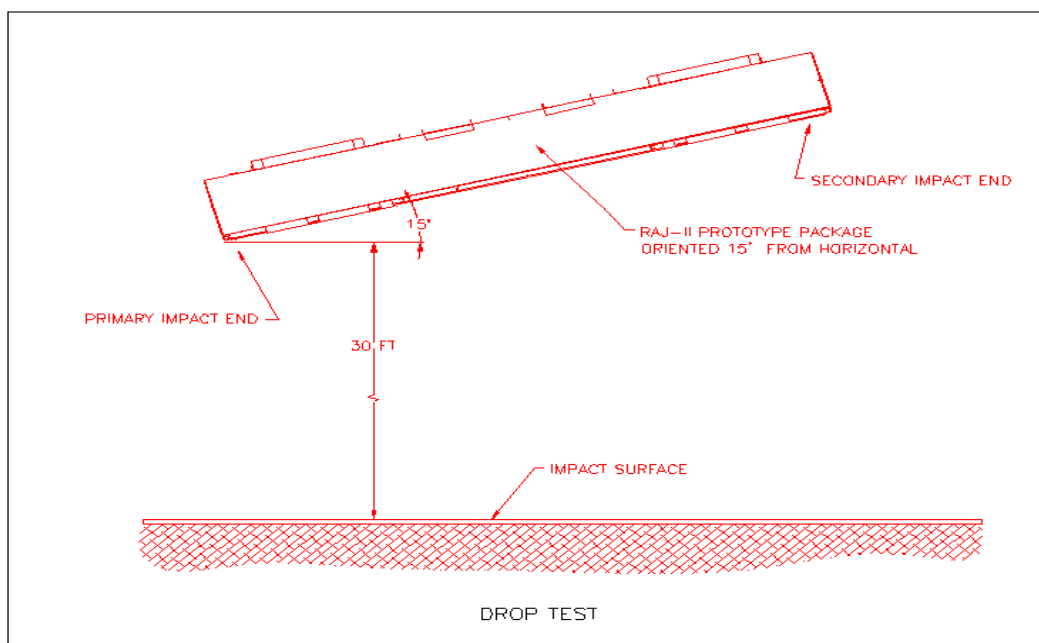
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**Table 2-9 Summary of Tests for RAJ-II**

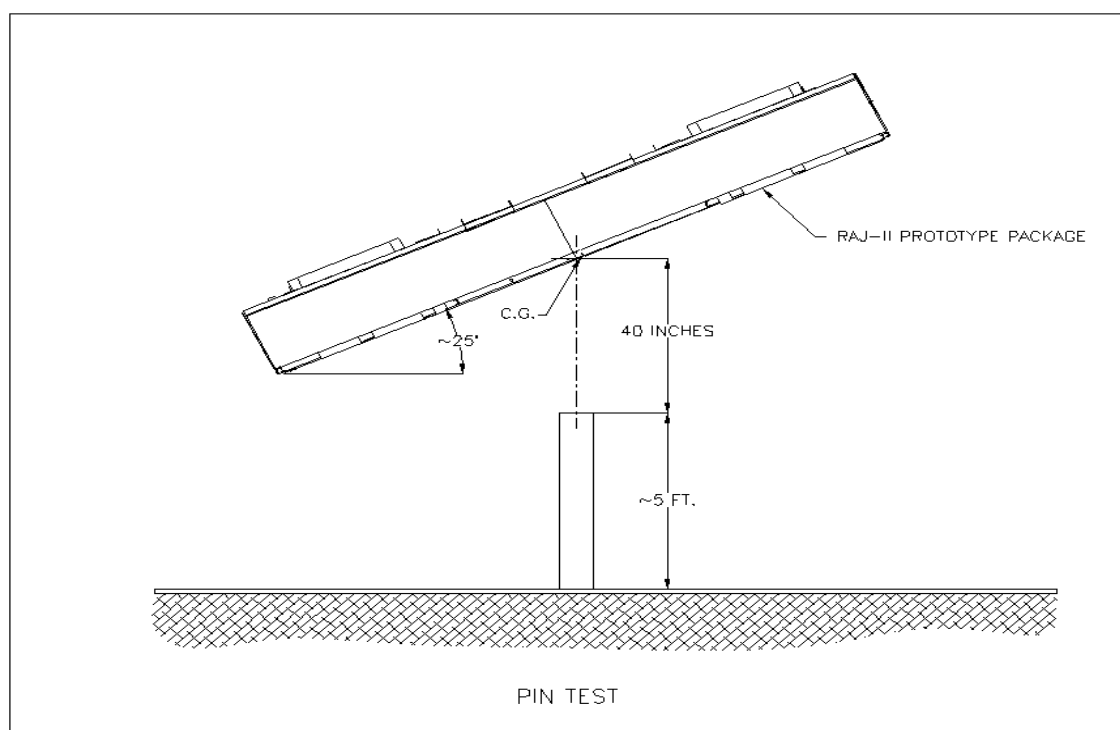
Test No.	Test Description	Test Unit Angular Orientation		CTU Temperature	Remarks
		Axial <sup>1</sup>	Rotational		
1	9 - meter (30-foot) slap down	15°	Lid down	Ambient	Top of package impacted first. Lid crushed over 11 cm (4.3 in).
2	Puncture	25°	Lid down	Ambient	Puncture pin crushed the outer lid down to the inner container lid. It did not rupture the outer lid or significantly deform the inner container lid or fuel.
3	9 - meter (30-foot) end drop	90°	Bottom down	Ambient	Crushed end wood impact absorber. Deformed the fuel assembly but did little damage to the rods

Notes:


<sup>1</sup> Axial angle,  $\theta$ , is relative to horizontal (i.e., side drop orientation)

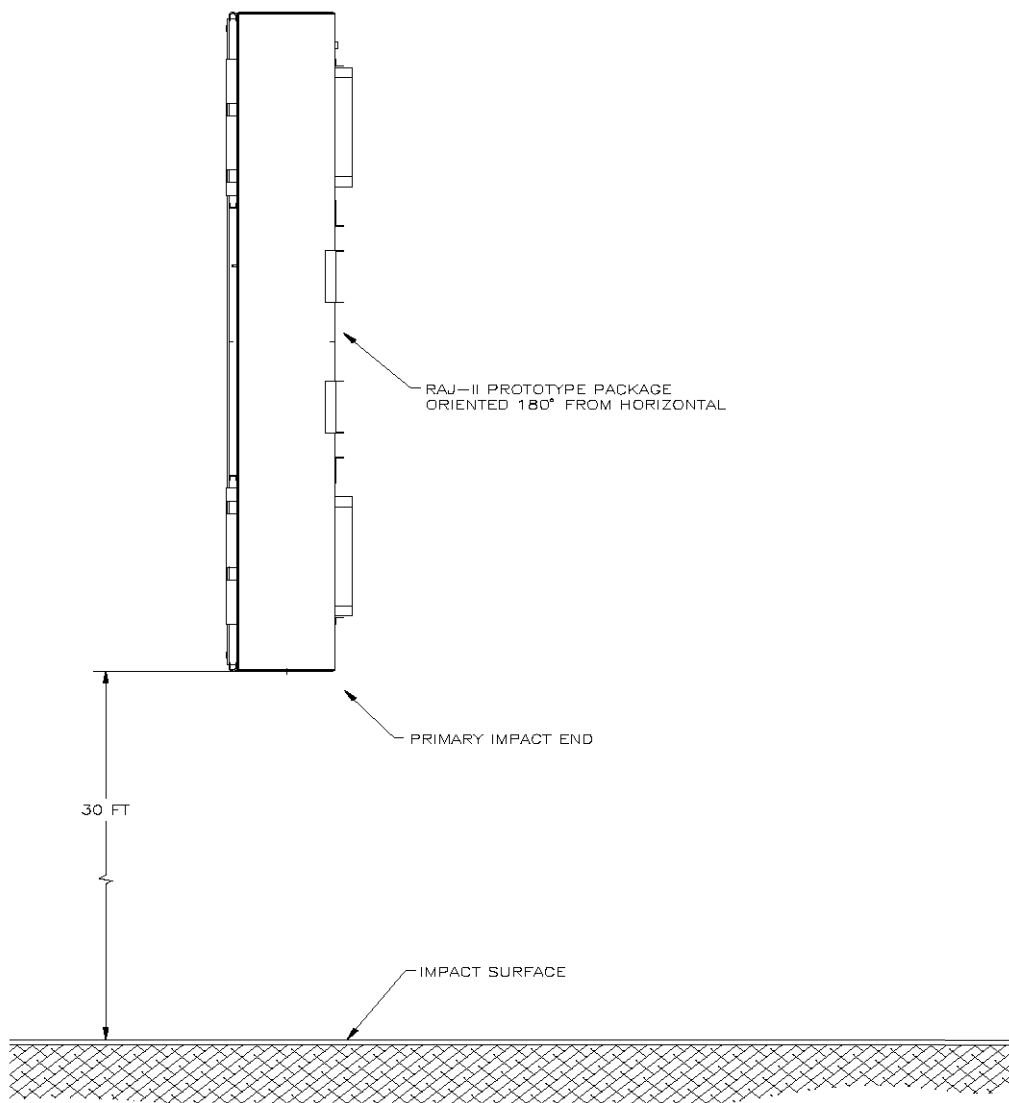


**Figure 2-13 Slap-down Orientation**



**Figure 2-14 Puncture Pin Orientation**

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**Figure 2-15 End Drop Orientation**

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## **2.8. ACCIDENT CONDITIONS FOR AIR TRANSPORT OF PLUTONIUM**

Not Applicable. This package will not be used for the air transport of plutonium.

## **2.9. ACCIDENT CONDITIONS FOR FISSILE MATERIAL PACKAGES FOR AIR TRANSPORT**

Not applicable. This package will not be used for the air transport of fissile material.

## **2.10. SPECIAL FORM**

This section does not apply for the TN-B1 package, since special form is not claimed.

## **2.11. FUEL RODS**

In each event evaluated above either by analysis or by test, the unirradiated fuel rods were protected by the TN-B1 package so that they sustained no significant damage. Fuel rod cladding is considered to provide containment of radioactive material under both normal and accident test conditions. Discussion of this cladding and its ability to maintain sufficient mechanical integrity to provide such containment is described in Section 1.2.3 and Chapter 4.0.

## **2.12. APPENDIX**


### **2.12.1. *Certification Test***

#### **2.12.1.1. Certification Test Unit**

The TN-B1 test packages were fabricated identically to the configuration depicted in the Packaging General Arrangement Drawing found in Appendix 1.4.1. The certification test unit is identical to the production TN-B1 packages except for some minor differences.

1. For ease in documentation/evaluation, tape and marker were used for reference markings during testing.
2. Minor amounts of the internal foam cushioning material were cut out to accommodate added weight in the fuel cavity.
3. Weight was added to the exterior of the package to allow the test units to be at the maximum allowed package weight.

The fuel assemblies were represented by a mock up fuel assembly (an ATRIUM-10 design). Lead rods inside the cladding replaced the fuel pellets. The fuel rods were seal welded using the same techniques used on the production fuel rods. A composite fuel assembly was used to represent the second fuel assembly. Steel tubes represented the ends with added steel for correct weight. The center section was made up of a mock up fuel assembly similar to the full size mock up fuel assembly. The mock up of the fuel approximated the stiffness of the fuel and added no extra strength to the center section of the package that would potentially be damaged

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by the puncture test. See Figure 2-16 through Figure 2-22 for container and mock up fuel preparation. Weight was added to the fuel assembly cavity by placing lead sheeting on the side of the fuel where normally there is foam. The lead weighing 143 pounds represented the weight of the water channels that could be shipped with some fuel assemblies. The lead plate was cut into strips that were not over half the height of the fuel assemblies to ensure that there was no support or protection added to the fuel during any of the tests. The total weight of the CTUs is provided in Table 2-10. The added weight in the contents represents the maximum payload weight including the fuel, fuel assembly fittings and packing material that could be required in the future.

For CTU 1 that was dropped lid down for a 30-foot slap down event and a 1-meter oblique puncture event, the weight was added between the bolster boards at each end. The added weight representing the difference between the actual tare weights of the package and the maximum allowed tare weight consisted of two ½ inch carbon steel plates. For CTU 1, these were held in place by the bolster and brackets attached to the bolster with lag bolts. See Figure 2-23. These plates were taken off CTU 1 and placed on the opposite end of CTU 2 for the end drop. See Figure 2-24.

**Table 2-10 Test Unit Weights**

Property	CTU 1		CTU 2	
As fabricated weight	849 kg	1,872 lb	848 kg	1,869 lb
Max. fabricated weight	930 kg	2,050 lb	930 kg	2,050 lb
Added weight	81.7 kg	180 lb	81.7 kg	180 lb
Content weight	684 kg	1,508 lb	685 kg	1,510 lb
Measured drop weight	1,614 kg	3,558 lb	1,611 kg	3,552 lb
Approximate weight of attaching frame	2.3 kg	5.1 lb	11.3 kg	24.9 lb
Approximate drop weight	1,616 kg	3,562 lb	1,622 kg	3,576 lb

#### 2.12.1.2. Test Orientations

Three certification tests were performed. Two tests were performed on CTU 1, a 9-meter (30-foot) slap-down on the lid and a 1-meter (40-inch) oblique puncture test on the lid. A 9-meter (30-foot) end drop was performed on CTU 2.

The 9-meter (30-foot) drop on the lid was designed to provide maximum acceleration to the end of the fuel as well as maximize the crush of the package for criticality evaluation purposes. The

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top down orientation was chosen since the lid contains the least material. The lid down orientation was also chosen since on previous tests horizontal lid down tests had maximized the crush and had resulted in the failure of the retaining bolts on the frame holding the inner container. As discussed in Section 2.7.1.1, the drop orientation was at 15 degrees with the horizontal. See Figure 2-25.

The 1-meter (40-inch) puncture test was performed on CTU 1 with the lid down after the 9-meter (30-foot) slap-down test. The package was oriented at a 25-degree angle to maximize the possibility of the corner of the puncture bar penetrating the outer container and maximizing the damage to the inner container and fuel. The puncture bar was aligned over the center of gravity of the package. See Figure 2-26 and Figure 2-27.


CTU 2 was dropped 9-meters (30-feet) with its bottom end down. The purpose of this orientation was to maximize the damage to the fuel. The bottom end was chosen since it is the most rigid end of the fuel assembly. The expansion springs inside the cladding tubes are on the upper end. See Figure 2-28.

#### 2.12.1.3. Test Performance

Testing was performed at the National Transportation Research Center in Oak Ridge, Tennessee. The CTUs were shipped to the facility fully assembled. Only the additional tare weight as described in Section 2.12.1.1 was added at the test facility. Tests were performed on the packages prior to them being transported to the Framatome-ANP facility at Lynchburg, Virginia. At Lynchburg the packages were disassembled and examined and the fuel rods were helium leak tested.

The slapdown test at 15 degrees to horizontal demonstrated the ability of the outer package to protect the fuel and the inner container. The energy absorbing capabilities of the package allowed the package to deform and limited the secondary impact to less than the primary impact. See Figure 2-29 and Figure 2-30. This test resulted in deformation inside the package. See Figure 2-36 and Figure 2-37. The crush of the paper honeycomb was limited by the stiffening plates in the lid. See Figure 2-38. The inner container lid was deformed as well. Neither the lid bolts on either container nor the bolts on the inner container clamping device failed. The frame did bend over 3 cm. The fuel rods, although slightly deformed due to the test and the added weight in the fuel cavity, were not damaged. See Figure 2-39. The added weight placed between the bolster timbers caused a slight deformation of the bottom wall of the outer package in the local area of the weights.

The puncture test was performed with the lid down at a 25 degree angle from horizontal. See Figure 2-25. The puncture pin was bolted with three bolts to the drop pad. The puncture pin struck the lid over the CG of the package after the package had undergone the slapdown test. See Figure 2-26. The pin did not penetrate the outer lid. The outer lid was deformed inward

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until it came in contact with the inner container. This was confirmed by a slight mark on the inner container lid. The pin appears to have bounced since there are two indentations very close together which could have been caused by the outer lid bottoming out against the inner container lid. See Figure 2-31 and Figure 2-32. No significant internal package or fuel damage appeared to be attributable to the pin puncture test.

The 9-meter (30-foot) end drop test was performed on CTU 2 with the bottom end down. There was little exterior damage to the outer container. See Figure 2-33, Figure 2-34, and Figure 2-35. Extensive damage occurred to the inside of the inner container as the fuel assemblies and the added weight impacted the interior of the inner container. The rigid end fitting of the assembly crushed the wood located at the end of the package. Although some welds broke, the bottom end of the package remained in place. The fuel rods partially came out of the end fitting. The fuel assemblies bent to the side. See Figure 2-40, Figure 2-41, and, Figure 2-42.

The mock up fuel assemblies from both CTU 1 and CTU 2 were helium leak tested. The Assembly from CTU 1 was found to meet the leak tight requirements of having a leak rate less than  $1 \times 10^{-7}$  atm-cc/s. The assembly from CTU 2 was found to have a He leak rate of  $5.5 \times 10^{-6}$  atm-cc/s. This is within the allowable leakage for the fuel as shown in Section 4.0.

#### 2.12.1.4. Test Summaries

Two 9-meter (30-foot) drops and one oblique puncture pin test were performed on two certification test units. The packages retained the fuel assemblies and protected the fuel. Mockup fuel assemblies from both certification units were leak tested after the drop tests and were determined to have maintained containment. The tests are summarized below.

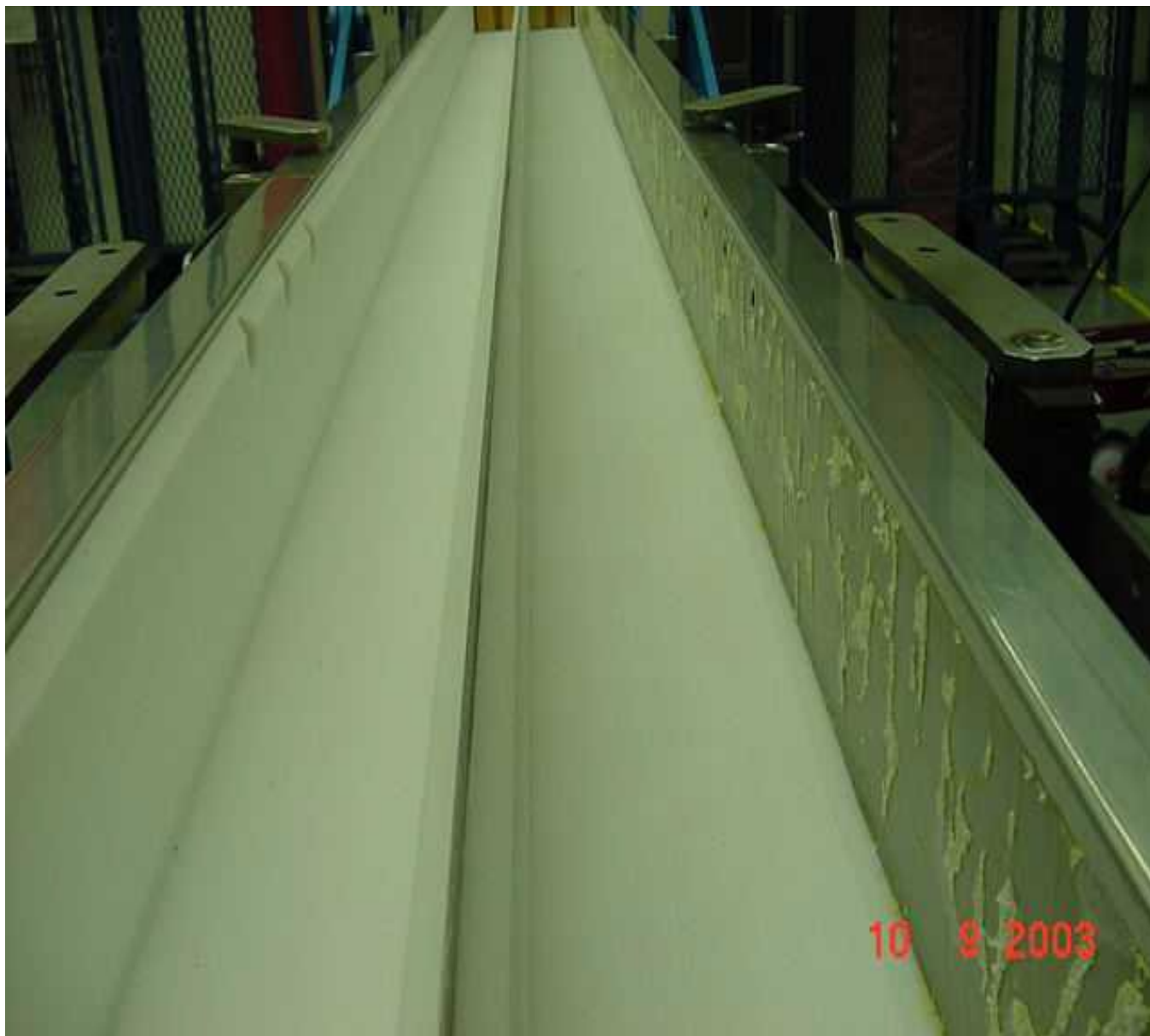
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**Table 2-11 Testing Summary**


Test	CTU	Orientation with horizontal	Exterior damage	Interior damage	Fuel
9-meter (30-foot) lid down	1	15°	Minor deformation on both ends.	No bolts broken on the frame or the lids. Significant deformation to inner container and internal clamp frame. Reduction of spacing between outside of package and fuel to about 4 inches.	Minimal damage to the fuel assemblies. Some twist to the assembly. No real damage to the fuel rods. The fuel was demonstrated to have a leak rate of less than $1 \times 10^{-7}$ atm-cc/s after the testing.
1-meter (40 in) lid down over cg	1	25°	Did not penetrate outer wall	Outer wall contacted inner container. Section 2.12 Figure 2-39 through 2-42 show some damage to the inner container, however, this damage is conservatively modeled in the HAC criticality analyses in Section 6.0 and is not sufficient to allow fuel to leak from the container.	The fuel appeared not to be affected by this test. Passed helium leak test.
9-meter (30-foot) lower end	2	90°	Localized damage on impact end.	Major crushing of the wood at the end of the inner package and breaking of the inner wall of the inner container on the impacted end. The outer wall was damaged but did not fail completely.	Fuel was bent and separated from end fittings. Fuel spacers were damaged. Fuel rods had no significant damage. Fuel bending was influenced by the movement of the weight added to the fuel cavity. Post drop leak test giving a He leak rate of $5.5 \times 10^{-6}$ atm-cc/s demonstrated that containment had been maintained.



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**Figure 2-16 Inner Container Being Prepared to Receive Mockup Fuel and Added Weight**


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**Figure 2-17 Partial Fuel Assemblies in CTU 1**



**Figure 2-18 Top End Fittings on Fuel in CTU 1**

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**Figure 2-19 Contents of CTU 2**



**Figure 2-20 Outer Container without Inner Container**




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**Figure 2-21 Inner Container Secured in Outer Container**



**Figure 2-22 CTU 2 Prior to Testing**

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**Figure 2-23 Addition of Tare Weight to CTU 1**



**Figure 2-24 Addition of Tare Weight to CTU 2**



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**Figure 2-25 CTU 1 Positioned for 15° 9-m (30-foot) Slap-down Drop**



**Figure 2-26 Alignment for Oblique Puncture**


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**Figure 2-27 Position for Puncture Test**



**Figure 2-28 Position for End Drop**

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**Figure 2-29 Primary Impact End Slap-down Damage**



**Figure 2-30 Secondary Impact End Damage**



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**Figure 2-31 Puncture Damage**



**Figure 2-32 Close Up of Puncture Damage**


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**Figure 2-33 End Impact**



**Figure 2-34 Damage from End Impact (Bottom and Side)**

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**Figure 2-35 End Impact Damage (Top and Side)**



**Figure 2-36 Damage Inside Outer Container to CTU 1**




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**Figure 2-37 Internal Damage to Outer Container CTU 1**



**Figure 2-38 Lid Crush on CTU 1**

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**Figure 2-39 Damage to Fuel in CTU 1**



**Figure 2-40 Internal Damage to CTU 2**



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**Figure 2-41 Fuel Damage CTU 2**



**Figure 2-42 Fuel Prior to Leak Testing CTU 2**

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### 2.12.2. ***GNF-J Certification Tests***

Normal conditions of transport (NCT) and hypothetical accident conditions (HAC) certification testing of the RAJ-II package was also performed by GNF-J as part of obtaining a Type AF certificate of compliance\* in Japan. For the U.S. testing, the GNF-J certification tests were utilized to determine the worst-case test orientations for the certification tests identified in Appendix 2.12.1. This appendix summarizes the GNF-J RAJ-II certification tests.

#### 2.12.2.1. **Certification Test Units**

Two certification test units (CTUs) were utilized for the GNF-J RAJ-II tests. Each CTU was fabricated in accordance with the Packaging General Arrangement Drawings found in Appendix 1.4.1, with the following exceptions:

1. The lateral wood bolsters on each end were not installed. Elimination of these wood bolsters is conservative for the free drops.
2. Maximum content weight was 560 kg (1,235 lbs), which results in a maximum package weight of 1,490 kg (3,285 lbs). This weight reduction is less than 8% lower than the maximum gross weight of the RAJ-II package, and will result in higher impact forces. The small difference in weight will have an insignificant effect on the free drop response of the package and/or fuel assembly.

One simulated fuel assembly and one dummy weight were utilized in each CTU to simulate the payload contents. Accelerometers were installed on the CTUs to measure and record each free drop impact. No accelerometers were used for the puncture drop tests.

#### 2.12.2.2. **Test Orientations**

Since the RAJ-II package relies on the fuel cladding as the containment boundary, free drop and puncture drop orientations that could damage the fuel cladding and potentially breach the containment boundary should be included in the test series. In addition, orientations that could damage the package and/or the fuel assemblies such that an unsafe criticality geometry would exist should be included in the test series.

Free drop orientations that could result in this type of damage include:

1. Vertical drop on the package end – maximizes axial impact acceleration to a fuel assembly, potentially buckling and failing the fuel cladding (containment boundary).
2. Horizontal drop of the package – maximizes lateral impact acceleration on a fuel assembly, potentially bending and failing the fuel cladding (containment boundary).
3. CG-over-corner of the package – maximizes deformation of outer container (OC).

\* Global Nuclear Fuel - Japan (fka Japan Nuclear Fuel Co., Ltd), Application for Approval of Packaging, Type RAJ-II, STO-M00-034, dated September 26, 2000.



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All of these orientations were included in the free drop test series of the package. Puncture drop orientations that could potentially breach the containment boundary (cladding) include:

1. Horizontal puncture drop on the center of the package – maximizes puncture impact onto fuel pins and potentially shearing and failure of the fuel cladding (containment boundary).
2. Vertical puncture drop on the end of the package – maximizes puncture impact onto the fuel assembly

Because of the end internal structure and wood dunnage in the outer container, the puncture drop on the end will not result in any significant deformation of the fuel assembly or the inner container. Therefore, this puncture drop orientation is bounded by the horizontal puncture drop on the center of the package.

The free drop tests included NCT drops of 0.3 meters (1 foot) and 1.2 meters (4 feet) prior to performing the 9-meter (30-foot) HAC free drop on each CTU. The horizontal puncture drop test was only performed on CTU 2J.

Two certification test series were performed. Three free drop tests were performed on CTU 1J, and three free drop and one puncture drop tests were performed on CTU 2J. The test series for each CTU is summarized in Table 2-10. All drop tests were performed at ambient temperature.

#### 2.12.2.3. Test Performance

Free drop and puncture testing was performed at two test facilities in Japan. At one facility, the drop pad consisted of a 32-mm (1.26-inch) thick steel plate that was embedded in a 1-meter (40-inch) thick concrete and steel support structure, with an overall length of 8 meters (26 feet). The other drop pad consisted of a 50-mm (1.97-inch) thick × 5-meter (16.4-feet) × 5-meter (16.4-feet) steel plate that was embedded in a 450-mm (12-inch) thick × 8.5-meter (27.9-feet) wide concrete and steel structure. The mass of each drop pad constituted an essentially unyielding surface for the CTUs, which weighed approximately 1,490 kg (3,285 lb).

##### 2.12.2.3.1. **CTU 1J**

CTU 1J was tested for a total of six free drop tests at heights of 0.3 meters (1 foot), 1.2 meters (4 feet), and 9 meters (30 feet). Figures 2-43 through 2-48 sequentially photo-document the CTU 1J tests.

The maximum resultant accumulated deformation, ~163 mm (~6 inches) occurred in the OC body corner. This orientation resulted in the maximum impact acceleration of 203g. No failure of the cladding (containment boundary) occurred from this test series.

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#### 2.12.2.3.2. **CTU 2J**

The testing of CTU 2J focused on free drop orientations not addressed by the CTU 1J tests. In addition, a HAC puncture drop test and HAC thermal test were performed. A total of three free drop tests at heights of 0.3 meters (1 foot), 1.2 meters (4 feet), and 9 meters (30 feet) were performed. Figures 2-49 and 2-50 sequentially photo-document the CTU 2J tests. The maximum resultant accumulated deformation, ~163 mm (~6 inches) occurred in the OC body corner. This orientation resulted in the maximum impact acceleration of 146g. No failure of the cladding (containment boundary) occurred from this test series.

#### 2.12.2.4. **Test Summaries**

Two 0.3-meter (1-foot), four 1.2-meter (4-foot), three 9-meter (30-foot) free drops, one 1-meter (40-inch) puncture drop, and one HAC thermal test were performed on two CTUs. The packages retained the fuel assemblies and protected the fuel. There was no visual damage or loss of fuel pellets from the simulated fuel assemblies from both CTUs. A summary of the test results is provided in Table 2-11.

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**Table 2-12 GNF-J CTU Test Series Summary**

CTU	Drop Height, m (ft)	Test Description	Purpose
1J	0.3 (1)	Free drop, CG-over-bottom end lower corner	Normal operation impact on OC body corner.
	1.2 (4)	NCT free drop, CG-over-bottom end lower corner	Impart initial deformation in same orientation as subsequent HAC free drop
		NCT free drop, horizontal on OC lid	Impart initial deformation in same orientation as planned HAC free drop
		NCT free drop, vertical, bottom end	Impart initial deformation in same orientation as subsequent HAC free drop
	9 (30)	HAC free drop, CG-over-bottom end lower corner	Maximize OC body deformation; potentially fail fuel rod and breach cladding.
		HAC free drop, vertical, bottom end	Maximize axial impact loads on fuel assemblies, potentially buckle fuel rod and
2J	0.3 (1)	Free drop, CG-over-lid corner	Normal operation impact on OC lid/body corner
	1.2 (4)	NCT free drop, horizontal on lid	Impart initial deformation in same orientation as subsequent HAC free drop
	9 (30)	HAC free drop, horizontal on lid	Maximize lateral impact loads on fuel assemblies, potentially breaching cladding.
	1 (3.3)	HAC puncture drop, horizontal on OC lid	Impact directly on HAC free drop damage; attempt to rupture fuel cladding.
	N/A	HAC thermal test	Demonstrate thermal performance of package.

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**Table 2-13 GNF-J CTU Test Series Results**

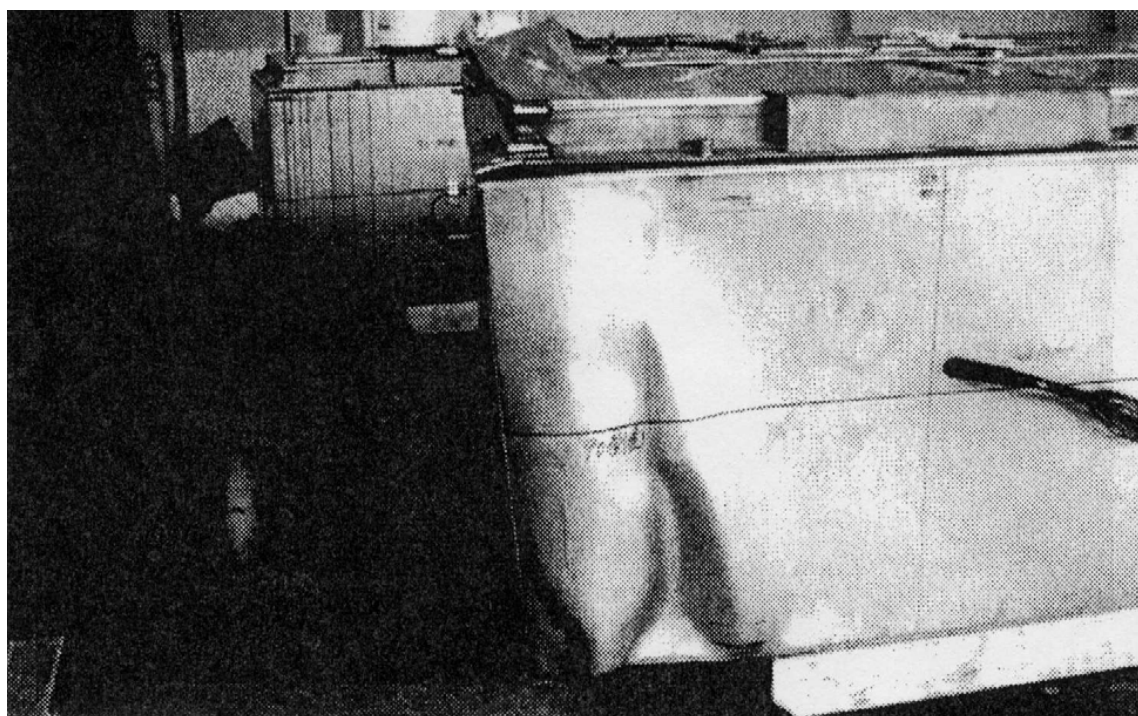
CTU	Drop Height, m (ft)	Test Description	Result
1J	0.3 (1)	Free drop, CG-over-bottom end lower corner	Combined deformation of ~40 mm (~1.6 inches) of bottom corner.
	1.2 (4)	NCT free drop, CG-over-bottom end lower corner	
		NCT free drop, horizontal on OC lid	No significant deformation.
		NCT free drop, vertical, bottom end	Impacted end deformed ~3.9 mm (~0.2 inches)
	9 (30)	HAC free drop, CG-over-bottom end lower corner	Impacted OC bottom corner deformed ~163 mm (~6 inches), OC lid corner ~101 mm (~4 inches). Maximum acceleration of 203g.
		HAC free drop, vertical, bottom end	IC body/lid deformed ~2 - 81 mm (~0.08 - 3 inches) in length, U-shaped lifting bar on fuel assembly bent due to contact with wood end dunnage. Maximum acceleration of 58g.
2J	0.3 (1)	Free drop, CG-over-lid corner	Combined deformation of ~2.9 mm (~0.1 inches) of lid corner.
	1.2 (4)	NCT free drop, horizontal on lid	
	9 (30)	HAC free drop, horizontal on lid	Impacted side deformed ~2 - 19 mm (~0.08 - 0.8 inches), localized weld failure of OC lid flange/OC lid sheet interface, no failure of OC lid bolts. Maximum acceleration of 146g.
	1 (3.3)	HAC puncture drop, horizontal on OC lid	~100 mm deep × ~2,000 mm (~4 inches × ~79 inches) wide indentation in OC lid, no breach of OC lid sheet.
	N/A	HAC thermal test	No failure of simulated fuel assembly cladding.



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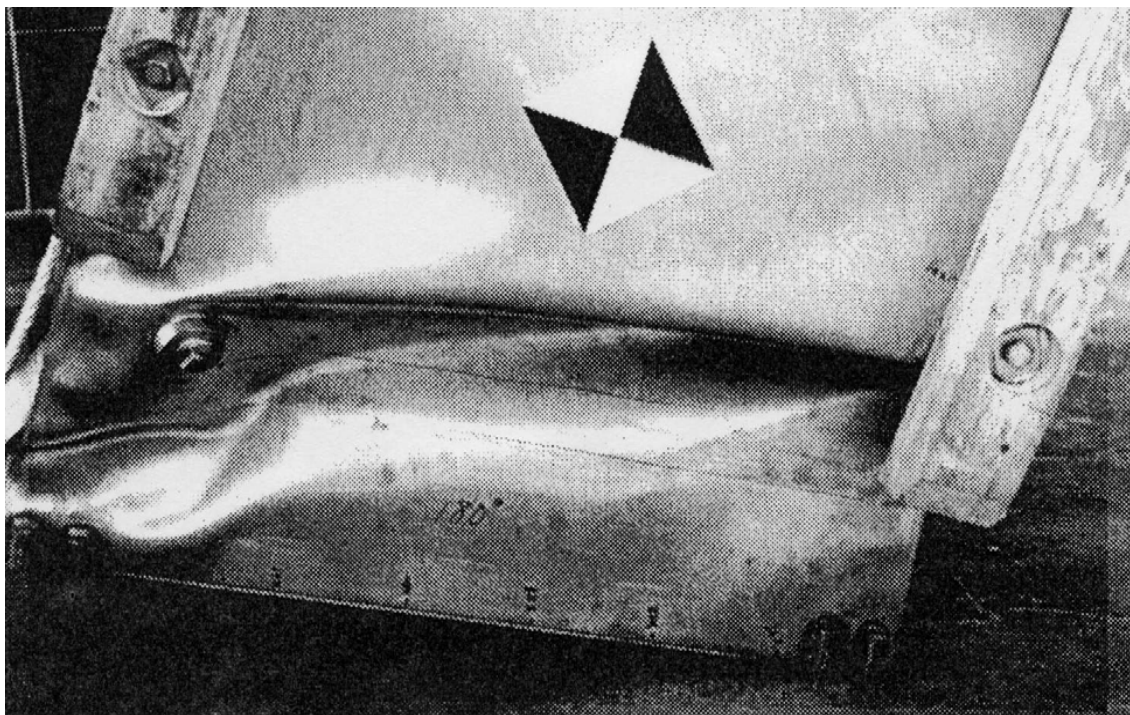
**Figure 2-43 CTU 1J 9 m CG-Over-Bottom Corner Free Drop: View of Impacted Corner**



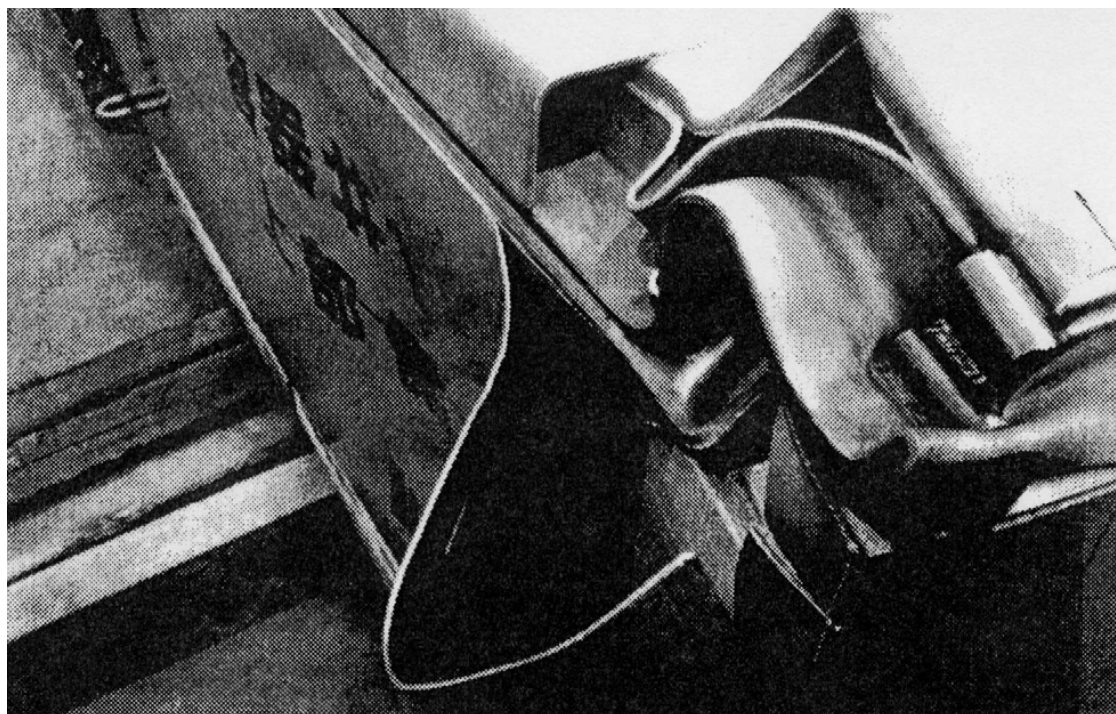
**Figure 2-44 CTU 1J 9 m CG-Over-Bottom Corner Free Drop: View of Opposite Corner**



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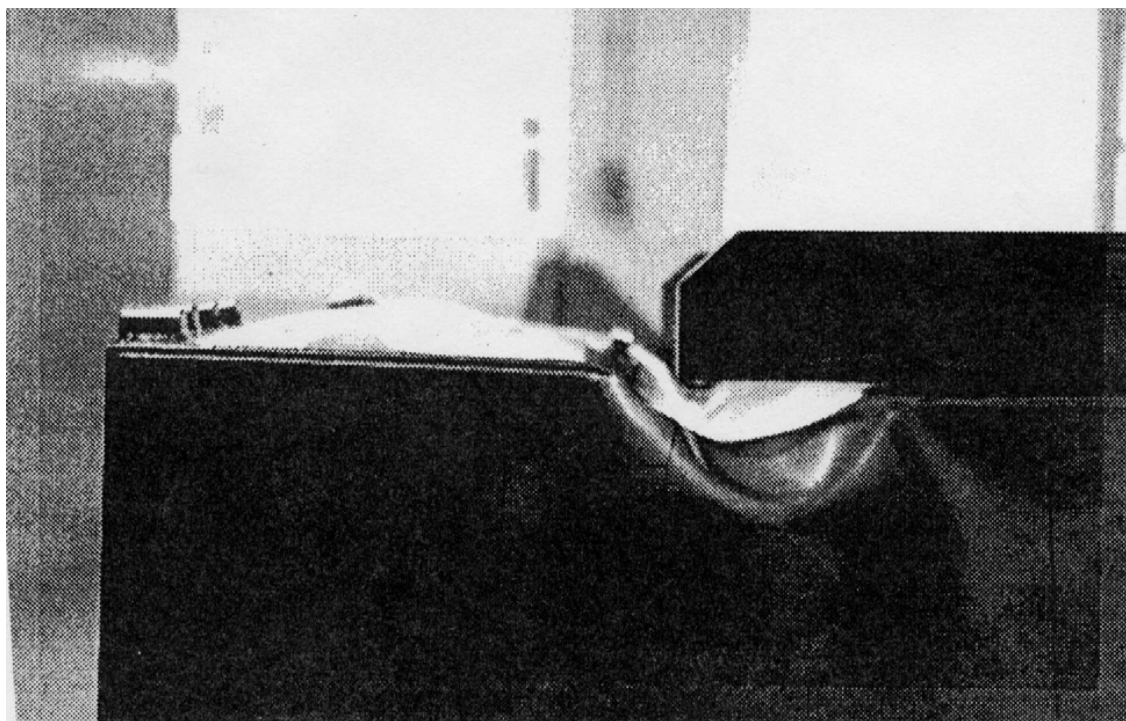
**Figure 2-45 CTU 1J 9 m CG-Over-Bottom Corner Free Drop: View of Bottom**



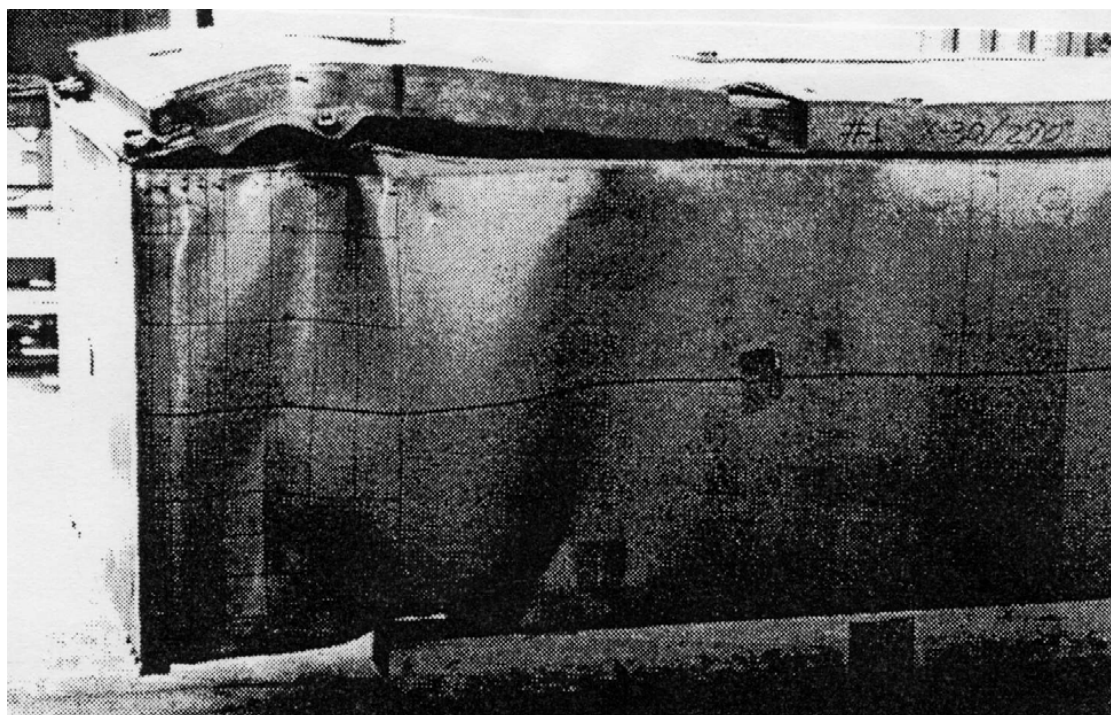
**Figure 2-46 CTU 1J 9 m CG-Over-Bottom Corner Free Drop: Close-up View of Top Corner**



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**Figure 2-47 CTU 1J 9-m Vertical End Drop: Close-up Side View of Bottom Damage**



**Figure 2-48 CTU 1J 9-m Vertical End Drop: Overall View of Damage**



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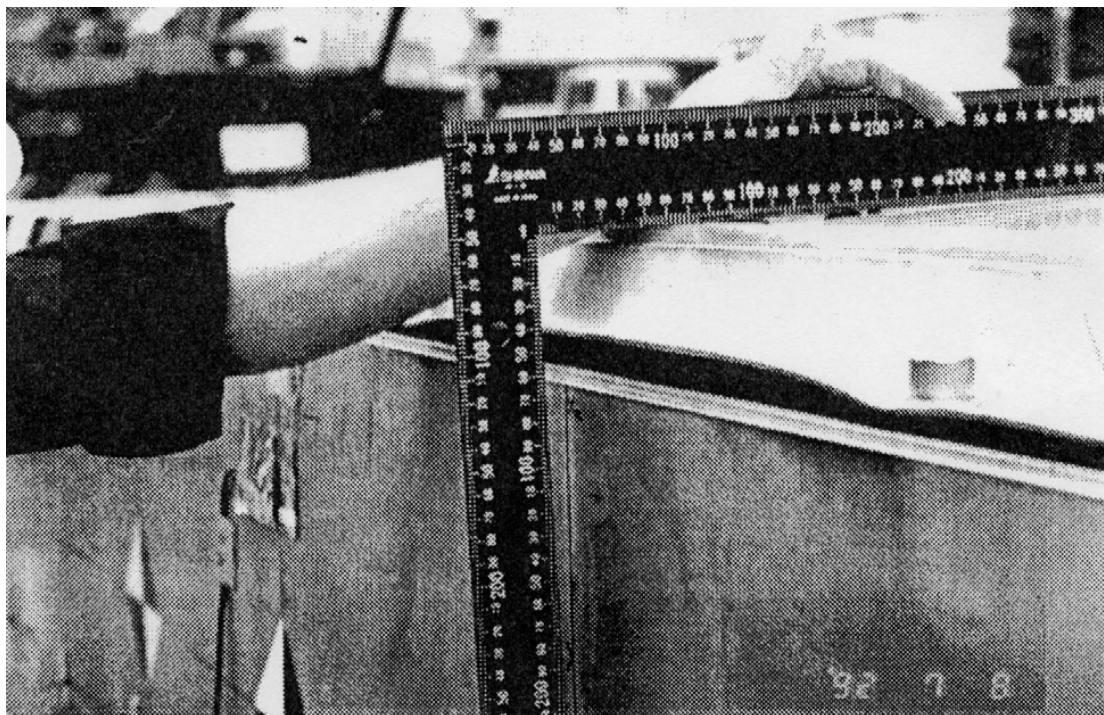


Figure 2-49 CTU 2J 9-m Horizontal Free Drop: Close-up Side View of Damage

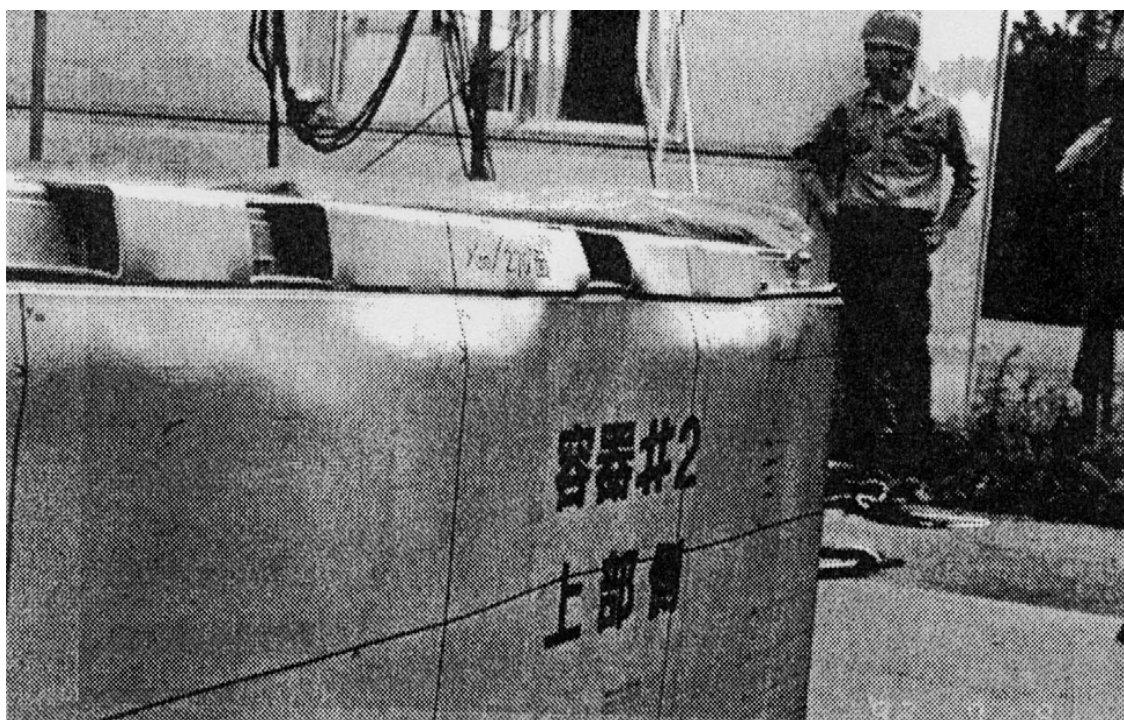


Figure 2-50 CTU 2J 9-m Horizontal Free Drop: Overall Side View of Damage

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### 2.12.3. Outer Container Gasket Sealing Capability

The outer container for the TN-B1 packaging utilizes a 5 mm thick × 40 mm wide × 11,360 mm long, 50 shore durometer, solid natural rubber gasket. As shown in Appendix 1.4.1, Packaging General Arrangement Drawings, the gasket is attached to the flange of the outer container lid. The outer container lid is secured to the outer container body by twenty-four (24) M14 × 2, Type 304 stainless steel bolts, which are tightened to “wrench tight or as defined in user procedures”. Since a specific tightening torque is not specified, the maximum bolt tension will be based on the minimum yield strength of the stainless steel.

The maximum force,  $F_b$ , in each lid bolt will be:

$$F_b = S_y (A_t)$$

where:  $S_y$  = Minimum yield strength = 206.8 MPa (30.0 ksi) (Ref. Table 2-2)

$A_t$  = Tensile area for M14 × 2 bolt = 115 mm<sup>2</sup> (0.1783 in<sup>2</sup>)

Substituting these values into the above equation yields a bolt force of 23,782 N (5,349 lb<sub>f</sub>).

The total compressive force applied to the gasket,  $F_{\text{gasket}}$ , is then:

$$F_{\text{gasket}} = (24)F_b = (24)(23,782) = 570,768 \text{ N (128,376 lb}_f\text{)}$$

For the applied bolt force, the gasket compressive area,  $A_{\text{gasket}}$ , is  $40 \times 11,360 = 454,400 \text{ mm}^2$  (704.3 in<sup>2</sup>). Conservatively neglecting any deflection of the 4-mm thick lid flange between the lid bolts, the resultant compressive stress on the gasket is then:

$$\sigma_{\text{gasket}} = \frac{570,768}{454,400} = 1.256 \text{ MPa (182 psi)}$$

The shape factor,  $s$ , for the 5 × 40 gasket is:

$$s = \frac{\text{One Load Area}}{\text{Total Free Area}} = \frac{\text{Width}}{2(\text{Thickness})} = \frac{40}{10} = 4.0$$

From Figure 5-12 of Handbook of Molded and Extruded Rubber,\* the percent compressive deflection of the 50-durometer gasket with  $s = 4.0$  at 182 psi compressive stress is approximately 3%, or 0.15 mm (0.006 in), which is minimal.

\* *Handbook of Molded and Extruded Rubber, Third Edition*, Goodyear Tire & Rubber Company.



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To determine whether the gasket is compressed with the applied bolt force, the compression modulus and the linear spring rate for the gasket is computed. Equation 3-7 of Handbook of Molded and Extruded Rubber, the linear spring rate,  $K_L$ , for the rubber gasket is:

$$K_L = \frac{E_c (A)}{h}$$

where:  $E_c$  = Compression modulus

$A$  = Compression area of gasket = 454,400 mm<sup>2</sup> (704.3 in<sup>2</sup>)

$h$  = height of gasket = 5 mm (0.197 in)

The compression modulus is extracted from Figure 5-20 of the Handbook of Molded and Extruded Rubber for a shape factor “s” of 4.0 and an approximate compression of 3% for the 50 durometer gasket. From this figure, the compression modulus is interpolated to be 6,912 psi (47.7 MPa). The linear spring rate of the gasket is then:

$$K_L = \frac{6,912(704)}{0.197} = 24.7 \times 10^6 \text{ lb}_f / \text{in} \quad (4.33 \times 10^6 \text{ Nmm})$$

To compress the gasket 0.15 mm (0.006 in), the required force in the bolts is:

$$24F_{\text{bolt}} = K_L \Delta = 24.7 \times 10^6 (0.006) = 148,200 \text{ lb}_f \quad (659,266 \text{ N})$$

$$\Rightarrow F_{\text{bolt}} = 6,175 \text{ lb}_f \quad (27,648 \text{ N})$$

Since the resultant bolt force required to compress the gasket 3% is greater than the yield strength of the lid bolts, the gasket will not be compressed to the estimated 3% compression. To determine the estimated gasket compression with the maximum lid bolt force at yield strength (23,782 N [5,349 lb<sub>f</sub>]), the linear spring rate will be computed for zero compression and then compared to the applied maximum force. From Figure 5-20 of the Handbook of Molded and Extruded Rubber for a shape factor “s” of 4.0, the compression modulus at zero compression will be:


$$E_c = 9,000(0.75) = 6,750 \text{ psi} \quad (46.5 \text{ MPa})$$

For zero compression and this compression modulus, the linear spring rate is:

$$K_L = \frac{6,750(704)}{0.197} = 24.1 \times 10^6 \text{ lb}_f / \text{in} \quad (4.23 \times 10^6 \text{ Nmm})$$

The resultant deformation of the gasket for this spring rate with the maximum bolt force is:

$$\Delta_{\text{gasket}} = \frac{24(F_{\text{bolt}})}{K_L} = \frac{24(23,782)}{4.23 \times 10^6} = 0.135 \text{ mm} \quad (0.005 \text{ in})$$

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This deformation is approximately 2.7% compression of the gasket. Prototypic seal testing in support of the TRUPACT-II package\* has demonstrated that a pressure seal requires a minimum of 10% – 12% compression. Section 3.6, *Squeeze*, of the Parker O-ring Handbook<sup>†</sup> states that “*The minimum squeeze for all seals, regardless of cross-section should be about 0.2 mm (0.007 inches). The reason is that with a very light squeeze almost all elastomers quickly take 100% compression set.*” Based on these test results and the recommendations of Parker, the outer lid gasket will not form a pressure retaining seal.

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\* U. S. Department of Energy (DOE), *Safety Analysis Report for the TRUPACT-II Shipping Package*, USNRC Certificate of Compliance 71-9218, U.S Department of Energy, Carlsbad Field Office, Carlsbad, New Mexico.

† ORD 5700A/US, *Parker O-ring Handbook*, 2001, Parker Hannifin Corporation, Lexington, KY.

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### 3. THERMAL EVALUATION

Provides an evaluation of the package to protect the fuel during varying thermal conditions.

#### 3.1. DESCRIPTION OF THERMAL DESIGN

The TN-B1 package is designed to provide thermal protection as described in Subpart F of 10 CFR 71 for transport of two BWR fuel assemblies with negligible decay heat. Compliance is demonstrated with 10 CFR 71 subpart F in the following subsections. The TN-B1 protects the fuel through the use of an inner and outer container that restricts the exposure of the fuel to external heat loads. The insulated inner container further restricts the heat input to the fuel through its insulation. The fuel requires very little thermal protection since similar fuel has been tested to the 800°C temperature without rupture.

Given negligible decay heat, the thermal loads on the package come solely from the environment in the form of solar radiation for Normal Conditions of Transport (NCT), as described in Section 3.4 or a half-hour, 800°C (1,475°F) fire for Hypothetical Accident Conditions (HAC), described in Section 3.5.

Specific ambient temperatures and solar heat loads are considered in the package thermal evaluations. Ambient temperatures ranging from -40°C to 38°C (-40°F to 100°F) are considered for NCT. The HAC fire event considers an ambient temperature of 38°C (100 F), with solar heat loading (insulation) before and after the HAC half-hour fire event.


Details and assumptions used in the analytical thermal models are described with the thermal evaluations.

##### 3.1.1. *Design Features*

The primary features that affect the thermal performance of the package are 1) the materials of construction, 2) the inner and outer containers and 3) the thermal insulation of the inner container. The stainless sheet metal construction of the structural components of the inner and outer containers influences the maximum temperatures under normal conditions. The material also ensures structural stability under the hypothetical accident conditions as well as provides some protection to the fuel. Likewise the zirconium alloy cladding has also been proven to be stable at the high temperatures potentially seen during the Hypothetical Accident Conditions (HAC).

The multi walled construction of the single walled outer container and the double walled inner container reduces the heat transfer as well as provides additional stability. The multi walled construction also reduces the opportunity for the fire in the accident conditions to impinge directly on the fuel.



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The thermal insulation also greatly reduces the heat transfer to the fuel from external sources. The insulation consists of alumina silicate around most of the package plus the use of wood on the ends that both provide some insulation as well as shock absorbing capabilities.

### **3.1.2. *Content's Decay Heat***

Since the contents are unirradiated fuel, the decay heat is insignificant.


### **3.1.3. *Summary Tables of Temperatures***

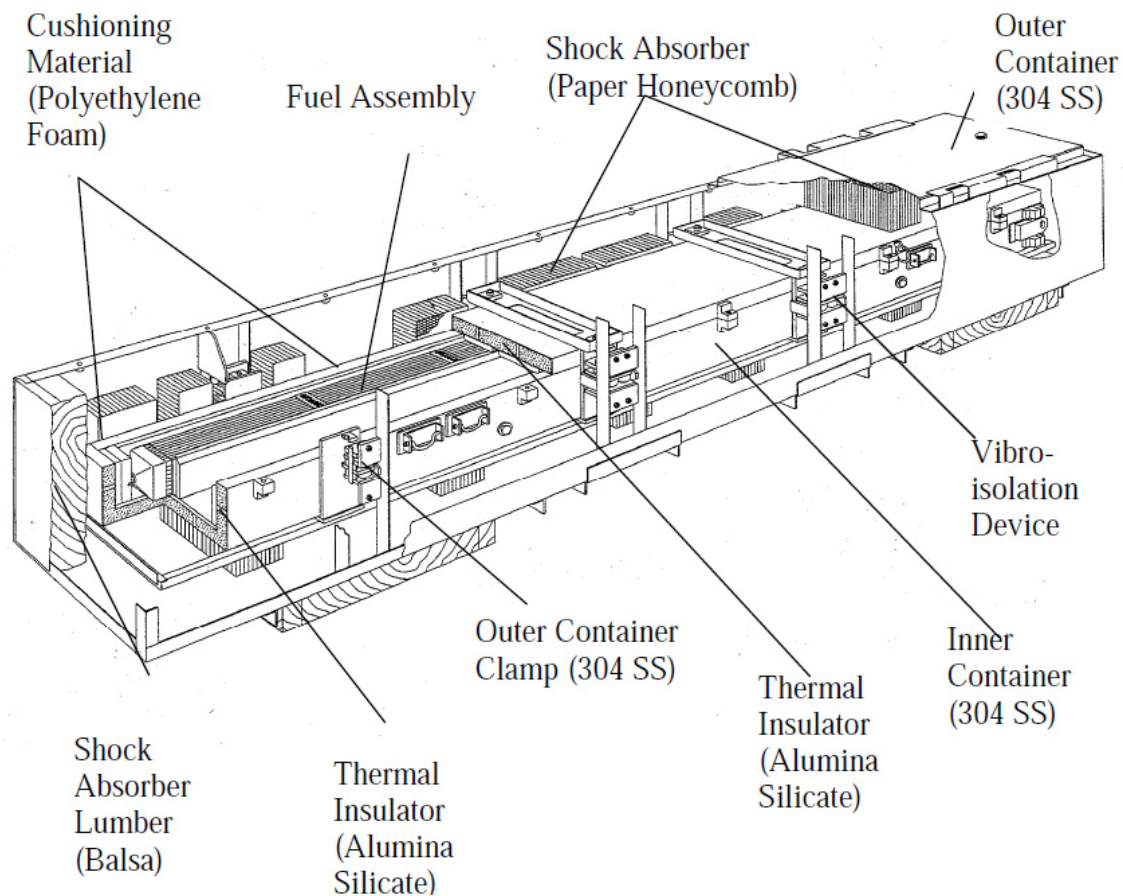
Since the decay heat load is negligible, the maximum NCT temperature of 171°F (77°C, 350 K) occurs on the package exterior, and the maximum HAC temperature of 1198°F (648°C, 921 K) occurs at the inner surface of the inner container at the end of the fire. These analyses demonstrate that the TN-B1 package provides adequate thermal protection for the fuel assembly and will maintain the maximum fuel rod temperature well below the fuel rod rupture temperature of 800+°C under all transportation conditions.

### **3.1.4. *Summary Tables of Maximum Pressures***


The maximum pressure within the containment, the fuel rods during normal conditions of transport is 1.33 MPa (192.9 psia).

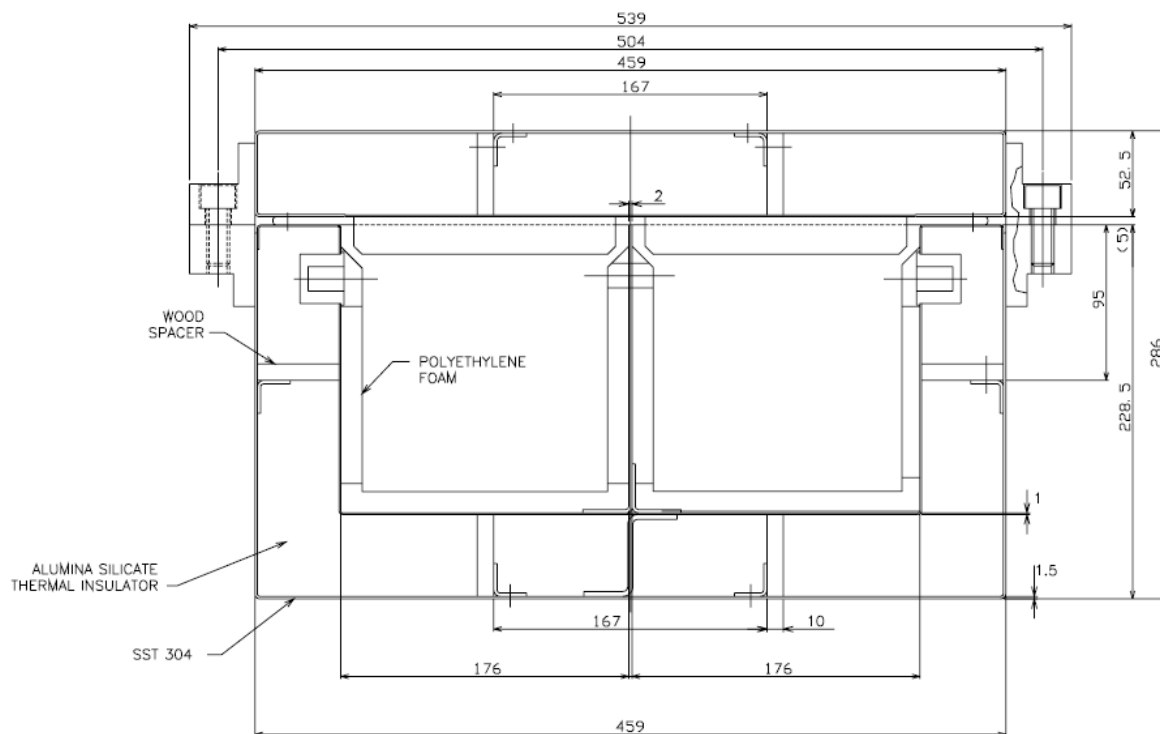
The maximum pressure during the hypothetical accident conditions is 3.50 MPa (508 psia).

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**Figure 3-1 Overall View of TN-B1 Package**

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**Figure 3-2 Transverse Cross-Sectional View of the Inner Container**


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## 3.2. **MATERIAL PROPERTIES AND COMPONENT SPECIFICATIONS**

### 3.2.1. ***Material Properties***

The TN-B1 inner container is constructed primarily of Series 300 stainless steel, wood, and alumina silicate insulation. The void spaces within the inner container are filled with air at atmospheric pressure. The outer container is constructed of series 300 stainless steel, wood, and resin impregnated paper honeycomb. The thermal properties of the principal materials used in the thermal evaluations are presented in Table 3-1 and Table 3-2. Where necessary, the properties are presented as functions of temperature. Note that only properties for materials that constitute a significant heat transfer path are defined. A general view of the package is depicted in Figure 3-1. A sketch of the inner container transversal cross-section with the dimensions used in the calculation is presented in Figure 3-2.

For the Alumina Silicate, maximum values are specified because the maximum conductivity is the controlling parameter. This is because there is no decay heat in the payload and the only consideration is the material's ability to block of heat transfer to the fuel during the fire event.


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**Table 3-1 Material Properties for Principal Structural/Thermal Components**

Material	Temperature, K	Thermal Conductivity, W/m-K	Specific Heat, J/kg-K	Density, kg/m <sup>3</sup>	Notes
Wood	300	0.240	2,800	500	<b>1</b>
Series 300 Stainless Steel	300	15	477	7,900	<b>2</b>
	400	17	515		
	500	18	539		
	600	20	557		
	800	23	582		
	1,000	25	611		
Alumina Silicate Insulation	673	≤0.105	1,046 (Nominal)	250 (Nominal)	<b>3</b>
	873	≤0.151			
	1,073	≤0.198			<b>4</b>
	1,273	≤0.267			<b>4</b>

Notes:

- 1** The material specified for the wood spacers. The properties have been placed with typical values for generic softwood.
- 2** [Reference. 3.6.1.2. p.809, 811, 812, and 820]
- 3** The values shown are based on published data for Unifrax Duraboard LD [Reference 3.6.1.11] and include compensation for the possible variation in test data (see discussion in Section 3.2.1).
- 4** Values at higher temperatures than 1,000 K are linearly extrapolated.

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**Table 3-2 Material Properties for Air**

Temperature (K)	Thermal Conductivity (W/m·K)	Density (kg/m <sup>3</sup> )	Specific Heat (J/kg·K)	Coefficient of Kinematic Viscosity $\nu$ (m <sup>2</sup> /s)	Prandtl Pr
300	0.0267	1.177	1005	15.66 E-06	0.69
310	0.0274	1.141	1005	16.54 E-06	0.69
320	0.0281	1.106	1006	17.44 E-06	0.69
330	0.0287	1.073	1006	18.37 E-06	0.69
340	0.0294	1.042	1007	19.32 E-06	0.69
350	0.030	1.012	1007	20.30 E-06	0.69
360	0.0306	0.983	1007	21.30 E-06	0.69
370	0.0313	0.956	1008	22.32 E-06	0.69
380	0.0319	0.931	1008	23.36 E-06	0.69
390	0.0325	0.906	1009	24.42 E-06	0.69
400	0.0331	0.883	1009	25.50 E-06	0.69
500	0.0389	0.706	1017	37.30 E-06	0.69
600	0.0447	0.589	1038	50.50 E-06	0.69
700	0.0503	0.507	1065	65.15 E-06	0.70
800	0.0559	0.442	1089	81.20 E-06	0.70
900	0.0616	0.392	1111	98.60 E-06	0.70
1000	0.0672	0.354	1130	117.3 E-06	0.70

Source: Reference 3.6.1.2, p.824



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### 3.2.2. *Component Specifications*

None of the materials used in the construction of TN-B1 package, such as series 300 stainless steel and alumina silicate insulation, are sensitive to temperatures within the range of -40°C to 800°C (-40°F to 1,475°F) that spans the NCT and HAC environment. Stainless steel has a melting point above 1,400°C (2,550°F), and maximum service temperature of 427°C (800°F). Similarly, the ceramic fiber insulation has a maximum operating temperature of 1,300°C (2,372°F). Wood is used as dunnage and as part of the inner package wall in the TN-B1 package. Before being consumed in the HAC fire, the wood would insulate portions of the inner container from exposure to the flames. However, the HAC transient thermal analyses presented herein conservatively neglects the wood's insulating effect, and assumes that all of the wood is consumed in the fire generating heat for all of its total mass.

The temperature limit for the fuel assembly's rods is greater than 800°C (1,472°F), based on the pressure evaluation provided in Section 3.5.3.2.

## 3.3. GENERAL CONSIDERATIONS

### 3.3.1. *Evaluation by Analysis*

The normal conditions of transport thermal conditions are evaluated by closed form calculations. The details of this analysis and supporting assumptions are found in that evaluation. The evaluation finds the maximum temperature for the outside of the package due to the insulation and uses that temperature for the contents of the package.

The transient hypothetical accident conditions are evaluated using an ANSYS finite element model. The model does not take credit for the outer container or the wood used in the inner container. Details of the model and the supporting assumptions maybe found in Section 3.5.

### 3.3.2. *Evaluation by Test*

Thermal testing was performed on fuel rods to determine the ability of the cladding (primary containment) to withstand temperatures greater than 800°C. The testing was performed for a range of fuel rods of different diameters, clad thickness and internal pressure. Since some of the current fuel designs for use in the TN-B1 are outside the range of parameters tested, additional thermal analyses have been performed to demonstrate the fuel rod's ability to withstand the HAC fire. In these tests, the fuel rods were heated to various temperatures from 700°C to 900°C for periods over one hour to determine the rupture temperature and pressure of the fuel. It was found that the fuel cladding did not fail at 800°C the temperature of the hypothetical accident conditions. This temperature associated pressure and resulting stress were used to provide the allowable conditions of the fuel which is used for containment.

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### 3.3.3. *Margins of Safety*

For the normal condition evaluation the margins of safety are qualitative, based on comparisons to the much higher temperatures the fuel is designed for when it is in service in the reactors. There is no thermal deterioration of the packaging components at normal condition temperatures therefore no margins for the package components are calculated.

The margins of safety for the accident conditions are evaluated in Section 3.5 and are based on the testing discussed in Section 3.3.2.

## 3.4. THERMAL EVALUATION UNDER NORMAL CONDITIONS OF TRANSPORT

This section presents the results of thermal analysis of the TN-B1 package for the Normal Conditions of Transport (NCT) specified in 10 CFR 71.71. The maximum temperature for the normal conditions of transport is used as input (initial conditions) in the Hypothetical Accident Condition (fire event) analysis.

### 3.4.1. *Heat and Cold*

Per 10 CFR 71.71(c)(1), the maximum environmental temperature is 100°F (311 K), and per 10 CFR 71.71(c)(2), the minimum environmental temperature is -40°F (233 K).

Given the negligible decay heat of the fuel assembly, the thermal loads on the TN-B1 package come solely from the environment in the form of solar radiation for NCT as prescribed by 10 CFR 71.71(c)(1). As such, the solar heat input into the package is 800 g·cal/cm<sup>2</sup> for horizontal surfaces and 200 g·cal/cm<sup>2</sup> for vertical surfaces for a varying insolation over a 24-hour period).

#### 3.4.1.1. **Maximum Temperatures**

For the analysis, the applied insolation is modeled transiently as sinusoidal over a 24-hour period, except when the sine function is negative (the insolation level is set to zero). The timing of the sine wave is set to achieve its peak at 12:00 PM and peak value of the curve is adjusted to ensure that the total energy delivered matched the regulatory values (800 g·cal/cm<sup>2</sup> for horizontal surfaces, 200 g·cal/cm<sup>2</sup> for vertical surfaces). As such, the total energy delivered in one day by the sine wave model is given by:

$$\int_{6-hr}^{18-hr} Q_{peak} \cdot \sin\left(\frac{\pi t}{12 \cdot hr} - \frac{\pi}{2}\right) dt = \left(\frac{24 \cdot hr}{\pi}\right) \times Q_{peak}$$

Using the expression above for the peak rate of insolation, the peak rates for top and side insolation may be calculated as follows:

Based on these inputs, the maximum NCT temperature on the inside surface of the inner container, as calculated in Appendix 3.6.3, is 350 K (77°C, 171°F).

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Given negligible decay heat, the maximum accessible surface temperature of the TN-B1 package in the shade is the maximum environment temperature of 38°C (100°F), which is less than the 50°C (122°F) limit established in 10 CFR 71.43(g) for a non-exclusive use shipment.

#### 3.4.1.2. Minimum Temperatures

The minimum environmental temperature that the TN-B1 package will be subjected to is -40°F, per 10 CFR 71.71(c)(2). Given the negligible decay heat load, the minimum temperature of the TN-B1 package is -40°F.

#### 3.4.2. Maximum Normal Operating Pressure

The fuel rods are pressurized with helium to a maximum pressure of 1.1145 MPa (absolute pressure (161.7 psia) helium at ambient temperature prior to sealing. Hence, the Maximum Normal Operating Pressure (MNOP) at the maximum normal temperature is:

$$MNOP = (P_1 \frac{T_{max}}{T_{ambient}} = 1.1145 \times \frac{350}{293} = 1.33 MPa = 192.9 psia$$

Since there is no significant decay heat and the fuel composition is stable, MNOP calculated above would not be expected to change over a one year time period.

#### 3.4.3. Maximum Thermal Stresses

Due to the construction of the TN-B1, light sheet metal constructed primarily of the same material, 304 SS, there are no significant thermal stresses. The package is constructed so that there is no significant constraint on any component as it heats up and cools down. The fuel cladding which provides containment is likewise designed for thermal transients, greater than what is found in the normal conditions of transport. The fuel rod is allowed to expand in the package. The fuel within the cladding is also designed to expand without interfering with the cladding.

### 3.5. THERMAL EVALUATION UNDER HYPOTHETICAL ACCIDENT CONDITIONS

This section presents the results of the thermal analysis of the TN-B1 package for the Hypothetical Accident Condition (HAC) specified in 10 CFR 71.73(c) (4).

For the purposes of the Hypothetical Accident Conditions fire analysis, the outer container of the TN-B1 package is conservatively assumed to be not present during the fire. This allows the outer surface of the inner container to be fully exposed to the fire event. The wood used in the inner container is conservatively assumed to combust completely. By ignoring the outer container and applying the fire environment directly to the inner container, the predicted temperature of the fuel rods is bounded. To provide a conservative estimate of the worst-case fuel rod temperature, the fuel assembly and its corresponding thermal mass are not explicitly

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modeled as well as the polyethylene foam shock absorber. The maximum fuel rod temperature is conservatively derived from the maximum temperature of the inside surface of the inner stainless steel wall. The analysis considering the insulation and multi-layers of packaging is very conservative because as discussed in Section 3.3.2 the bare fuel has been demonstrated to maintain integrity when exposed to temperatures that equal those found in the hypothetical accident conditions.

Thermal performance of the TN-B1 package is evaluated analytically using a 2-D model that represents a transversal cross-section of the inner container (Figure 3-2) in the region containing the metallic and wood spacers. The 2-D inner container finite element model was developed using the ANSYS computer code [Reference 3.6.1.3]. ANSYS is a comprehensive thermal, structural and fluid flow analysis package. It is a finite element analysis code capable of solving steady state and transient thermal analysis problems in one, two or three dimensions. Heat transfer via a combination of conduction, radiation and convection can be modeled.

The solid entities were modeled in the present analysis with PLANE55 two-dimensional elements and the radiation was modeled using the AUX12 Radiation Matrix method. The developed ANSYS input file is included as Appendix 3.6.2.

The initial temperature distribution in the inner container prior to the HAC fire event is a uniform 375 K conservatively corresponding to the outer surface temperature of the inner container per the normal condition calculations presented in Appendix 3.6.3.

### **3.5.1. Initial Conditions**

The environmental conditions preceding and succeeding the fire consist of an ambient temperature of 38 °C (311 K) and insulation per the normal condition thermal analysis. The solar absorptivity coefficient of the outer surface has been increased for the post-fire period to 1 to include changes due to charring of the surfaces during the fire event.

### **3.5.2. Fire Test Conditions**

The Hypothetical Accident Condition fire event is specified per 10 CFR 71.73(c) (4) as a half-hour, 800°C (1,073 K) fire with forced convection. For the purpose of calculation, the value of the package surface absorptivity coefficient (0.8) is selected as the highest value between the actual value of the surface (0.42) and a value of 0.8 as specified in 10 CFR 71.73(c) (4).

A value of 1.0 for the emissivity of the flame for the fire condition is used in the calculation. The rationale for this is that 1.0 maximizes the heating of the package. This value exceeds the minimum value of 0.9 specified in 10 CFR 71.73(c) (4). The Hypothetical Accident Condition (HAC) fire event is specified per 10 CFR 71.73(c)(3) as a half-hour, 800°C (1,475°F) fire with forced convection and an emissivity of 0.9. The environmental conditions preceding and

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succeeding the fire consist of an ambient temperature of 100 °F and insulation per the NCT thermal analyses.

To model the combustion of the wood, the wood elements of the model are given a heat generation rate based on the high heat value of Western Hemlock of 3630 Btu/lb ( $8.442 \times 10^6$  J/kg) from Reference 3.6.1.8, Section 7, Table 9. It is conservatively assumed that the entire mass of the wood will burn. Moreover, the wood will burn across its thinnest section from opposite faces. Using data burn rate data for redwood which has approximately the same density as hemlock [3.6.1.8], each face will burn 5 mm at a minimum rate of 0.543 mm/min [Reference 3.6.1.10] resulting in a 9.2 minute time of combustion. This conservatively results in the longest burn time for the hemlock, and the greatest effect on temperature. The resulting heat generation rate in the wood spacers is equal to:

$$\dot{Q} = (8.42 \times 10^6) \times (500 \text{ kg} / \text{m}^3) / (9.2 \text{ s} \times 60) = 7.63 \times 10^6 \text{ W/m}^3/\text{s}$$

#### 3.5.2.1. Heat Transfer Coefficient during the Fire Event

During a HAC hydrocarbon fire, the heating gases surrounding the package will achieve velocities sufficient to induce forced convection on the surface of the package. Peak velocities measured in the vicinity of the surfaces were under 10 m/s [Reference 3.6.1.4].

The heat transfer coefficient takes the form [Reference 3.6.1.4, p. 369]:

$$h = k/D \cdot C \cdot (u \cdot D/u)^m \cdot \text{Pr}^{1/3}$$

Where:

D: average width of the cross-section of the inner container (0.373 m)

k: thermal conductivity of the fluid

u: kinematic viscosity of the fluid

u: free stream velocity

C, m: constants that depend on the Reynolds number ( $\text{Re} = u \cdot D/u$ )

Pr: Prandtl number for the fluid

The property values of k, u and Pr are evaluated at the film temperature, which is defined as the mean of the wall and free stream fluid temperatures. At the start of the fire the wall temperature is 375 K (101.7°C, 215°F) and the stream fluid temperature is 1,073 K (1,475°F). The film temperature is therefore 710.5 K, and the property values for air at this temperature (interpolated from Table 3-2) are  $k = 0.0509 \text{ W/m} \cdot \text{K}$ ,  $u = 66.84 \text{E-}06 \text{ m}^2/\text{s}$  and  $\text{Pr} = 0.70$ . Assuming a maximum

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stream velocity of 10 m/s this yields a Reynolds number of 55.8E03. At this value of Re, the constants C and n are 0.102 and 0.675 respectively [Reference 3.6.1.4, Table 7.3].

$$h = \frac{0.0509 \cdot 0.102 \cdot \left(10 \cdot \frac{0.373}{66.84 \times 10^{-6}}\right)^{0.675} \cdot (0.70)^{1/3}}{0.373}$$

$$h = 19.8 \text{ W/m}^2 \text{ K}$$

A value of 19.8 W/m<sup>2</sup>·K was conservatively used in the analysis of the regulatory fire.

### 3.5.2.2. Heat Transfer Coefficient during Post-Fire Period

During the post-fire period of the HAC, it is conservatively assumed that there is negligible wind and that heat is transferred from the inner container to the environment via natural convection. Natural heat transfer coefficients from the outer surface of the square inner container are calculated as follows.

Reference 3.6.1.4 recommends the following correlations for the Nusselt number (Nu) describing natural convection heat transfer to air from heated vertical and horizontal surfaces:

Vertical heated surfaces [Reference 3.6.1.4, p. 493]:

$$Nu = \left(0.825 + \frac{0.387 \times (Gr \times Pr)^{\frac{1}{6}}}{\left(1 + \left(0.492/P_r\right)^{9/16}\right)^{\frac{8}{27}}}\right)^2$$

For entire range of Ra=Gr x Pr (9)

Where:

Nu: Nusselt number

Gr: Grashof number

Pr: Prandtl number

Horizontal heated surfaces facing upward [Reference 3.6.1.4, p.498]:

$$Nu = 0.54 \times (Gr \times Pr)^{1/4} \text{ for } (10^4 < Gr \times Pr < 10^7) \quad (10)$$

$$Nu = 0.15 \times (Gr \times Pr)^{1/3} \text{ for } (10^7 < Gr \times Pr < 10^{11}) \quad (11)$$

and, for horizontal heated surfaces facing downward:

$$Nu = 0.27 \times (Gr \times Pr)^{1/4} \text{ for } (10^5 < Gr \cdot Pr < 10^{10}) \quad (12)$$

The correlations for the horizontal surfaces are calculated using a characteristic length defined by the relation  $L=A/P$ , where A is the horizontal surface area and P is the perimeter [Reference 3.6.1.4, p. 498]. The calculated characteristic length for the horizontal surfaces of the inner container is  $L=0.209 \text{ m}$  ( $A=2.14812 \text{ m}^2$  and  $P=10.278 \text{ m}$ ).



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The following convective heat transfer coefficients (Table 3-1) have been calculated using Eq. (5), (6), (9), (10), (11) and (12). The corresponding characteristic length used in calculating the Nusselt number for each surface is also used in Eq. 5 for calculating the heat transfer coefficient. The thermal properties of air have been evaluated at the mean film temperature  $(=(T_s+T_{\text{ambient}})/2)$ .

The effects of solar radiation are included during the post-fire period by specifying the equivalent heat flow for each node of the surfaces exposed to fire for an additional 3.5 hours, i.e. the fire starts at the time of the peak temperature in the inner container (8 hours after sunrise) and is 0.5 hours in duration. This results in an additional 3.5 hours of solar insolation. Using the peak rates calculated in section 3.4.1.1, the nodal heat flows at 2:30 PM are equal to:

$$q'_{top} = \frac{1,218 \frac{W}{m^2} \left( \sin \left( \frac{\pi \times (6 + 8.5)}{12} - \frac{\pi}{2} \right) \right) (0.459 \text{ m})}{(155 - 1)} = 2.88 \text{ W/m}$$

$$q'_{side} = \frac{305 \frac{W}{m^2} \left( \sin \left( \frac{\pi \times 14.5}{12} - \frac{\pi}{2} \right) \right) (0.281 \text{ m})}{(99 - 1)} = 0.69 \text{ W/m}$$

Where 0.459 m is the width of the inner container, 0.281 m is its height, and the model is 155 nodes in width by 99 nodes in height. For the remaining 3.5 hours of solar insolation, these heat fluxes are conservatively applied as bounding constant values rather than varying with time.

The solar absorptivity coefficient of the outer surface is conservatively assumed to be 1. The duration of the post-fire period has been extended to 12.5 hr to investigate the cool-down of the inner container.

### 3.5.3. *Maximum Temperatures and Pressure*

#### 3.5.3.1. **Maximum Temperatures**

The peak fuel rod temperature, which is conservatively assumed to be the same as the inner wall temperature of the package, response over the course of the HAC fire scenario is illustrated in Figure 3-3. The temperature reaches its maximum point of 921 K or 648°C (1198°F) at the end of the fire or 1,800 seconds after the start of the fire. This peak temperature occurs at top corners of the inner wall.

The maximum temperature even when applied to the fuel directly is well below the maximum temperature the fuel can withstand. Similar fuel with no thermal protection has been tested in fire conditions at over 800°C (1,475°F) for more than 60 minutes without failures.

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### 3.5.3.2. Maximum Internal Pressure

The maximum pressure for the fuel can be determined by considering that the fuel is pressurized initially with helium. As the fuel is heated, the internal pressure in the cladding increases. By applying the ideal gas law the pressure can be determined and the resulting stresses in the cladding can be determined. Since the temperatures can be well above the normal operating range of the fuel the cladding performance can best be determined by comparison to test data.

Similar fuel with similar initial pressures has been heated in an oven to over 800°C for over an hour without failures (Reference 3.6.1.6). The fuel that was tested in the oven was pressurized with 10 atmospheres of helium. When heated to the 800°C it had an equivalent pressure of:


$$P_{max} = (P_1) \frac{T_{max}}{T_{ambient}} = 1.1145 MPa \times \frac{1,073}{293} = 4.08 MPa = 592 psia$$

This results in an applied load to the cladding of 3.98 MPa or 577.3 psig. The fuel that was tested had an outer diameter of 0.4054 inch (10.30 mm). Since the fuel when tested to 850°C had some ruptures but did not rupture at 800°C when held at those temperatures for 1 hour, the stresses at 800°C are used as the conservative allowable stress. Both the tested fuel and the fuels to be shipped in the TN-B1 have similar zirconium cladding. The stress generated in the cladding of the test fuel is:

$$\rho = \frac{pr}{t} = \frac{3.98 MPa \times 4.56 mm}{0.584 mm} = 31.1 MPa = 4,510 psi$$

Recognizing that the properties of the fuel cladding degrade as the temperature increases the above calculated stress is conservatively used as the allowable stress for the fuel cladding for the various fuels to be shipped. The fuel is evaluated at the maximum temperature the inner wall of the inner container sees during the Hypothetical Accident Condition thermal event evaluated above. Table 3-5 shows the maximum pressure for each type of fuel and the resulting stress and margin. The limiting design properties of the fuel, maximum cladding internal diameter, minimum cladding wall thickness and initial pressurization for each type of fuel are considered in determining the margin of safety. Positive margins are conservatively determined for each type of fuel demonstrating that containment would be maintained during the Hypothetical Accident events. The minimum cladding thickness does not include the thickness of the liner if used.

The results of the transient analysis are summarized in Table 3-4. The temperature evolution during the transient in three representative locations on the inner wall and one on the outer wall is included. The maximum temperature on the inner wall is 921 K (648°C, 1198°F) and is reached at the upper inner corners of the container, 1,800 seconds after the beginning of the

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fire. The graphic evolution of the temperatures listed in Table 3-4 is represented in Figure 3-3. Representative plots of the isotherms at various points in time are depicted in Figure 3-4 through Figure 3-7.

The temperatures and resulting pressures are within the capabilities of the fuel cladding as shown by test. Therefore the fuel cladding and closure welds maintain containment during the Hypothetical Accident Conditions.

The temperatures and resulting pressures are within the capabilities of the fuel cladding as shown by test. Therefore the fuel cladding and closure welds maintain containment during the Hypothetical Accident Conditions.


#### **3.5.4. *Accident Conditions for Fissile Material Packages for Air Transport***

Approval for air transport is not requested for the TN-B1.

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**Table 3-3 Convection Coefficients for Post-fire Analysis**

<b>T<sub>s</sub> (surface temperature)</b>		<b>T<sub>ambient</sub></b>		<b>H (vertical surface)</b>	<b>h (horizontal surface facing upward)</b>	<b>h (horizontal surface facing downward)</b>
°F	K	°F	K	(W/m <sup>2</sup> ·K)	(W/m <sup>2</sup> ·K)	(W/m <sup>2</sup> ·K)
150	338.71	100	311	4.68	5.19	2.34
200	366.48	100	311	5.61	6.34	2.74
250	394.26	100	311	6.18	7.05	2.99
300	422.04	100	311	6.60	7.55	3.17
350	449.82	100	311	6.90	7.92	3.30
400	477.59	100	311	7.13	8.18	3.41
600	588.71	100	311	7.64	8.74	3.67
900	755.37	100	311	8.00	9.07	3.89
1,375	1,019.26	100	311	8.25	9.17	4.09

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**Table 3-4 Calculated Temperatures for Different Positions on the Walls of the Inner Container Walls**

Time (s)	Inner Wall Temperature (top right corner) (K)	Inner Wall Temperature (bottom) (K)	Inner Wall Temperature (top) (K)	Outer Wall Temperature (K)
0.1	375	375	375	377
911	750	667	546	1,062
1,800	921	821	696	1,067
1,900	918	823	710	807
2,000	905	817	723	686
2,200	868	797	742	583
2,600	803	761	760	509
3,268	723	715	758	463
4,280	639	662	727	437
27,973	354	335	369	378
45,000	349	324	358	377

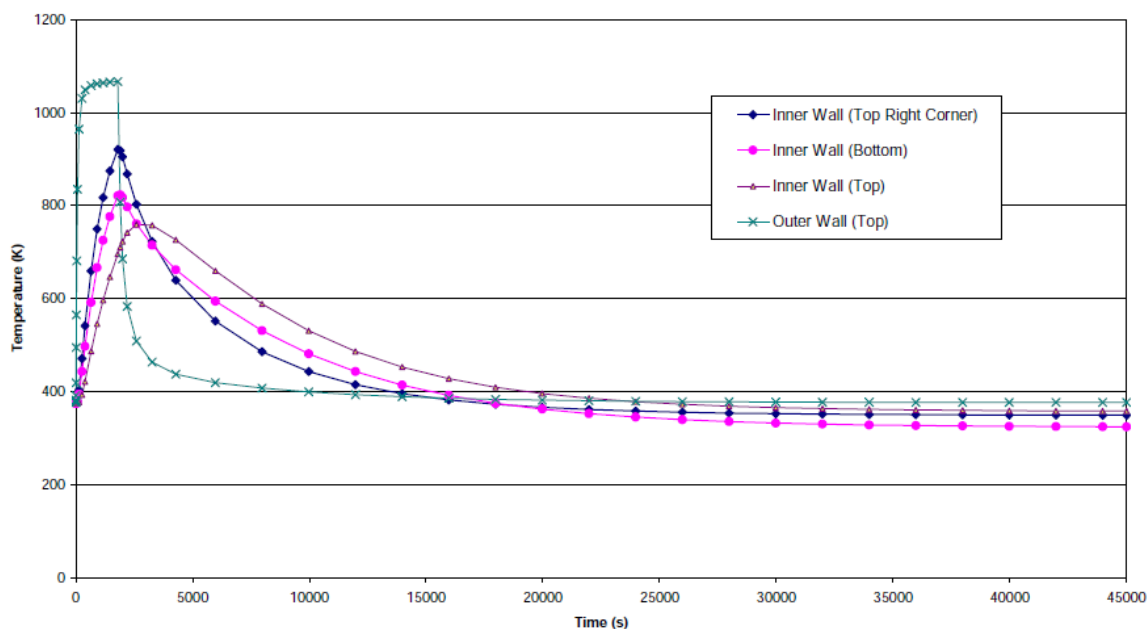
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**Table 3-5 Maximum Pressure**

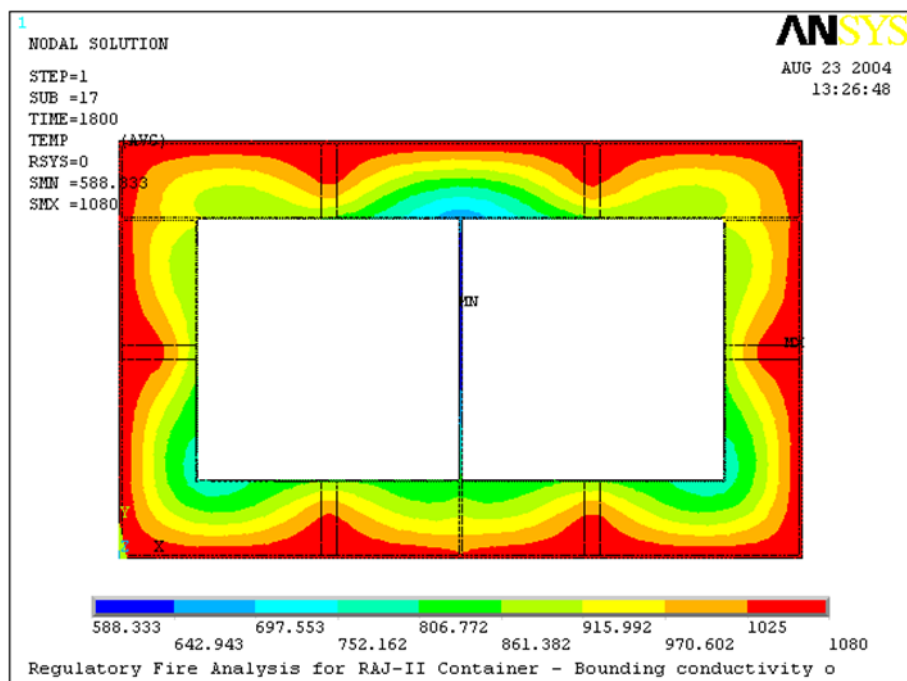
Parameter	Units	8 x 8 Fuel	9 x 9 Fuel	10 X 10 Fuel	11 X 11 Fuel
Initial Pressure	MPa absolute	0.608	1.1145	1.1145	1.1145
Fill temperature	°C	20	20	20	20
Temperature during HAC	°C	648	648	648	648
Outside Diameter Maximum	mm	12.5	11.46	10.52	10.14
	inches	0.492	.4512	.4142	0.399
Minimum Allowable Cladding Thickness	mm	0.68	0.570	0.520	0.500
	inches	0.0268	0.0224	0.0205	0.0197
Cladding Inside Diameter Maximum	mm	11.14	10.32	9.48	9.14
	inches	0.439	.406	.373	0.360
Pressure @ HAC	MPa absolute	1.91	3.50	3.50	3.50
	psia	277	508	508	508
Applied Pressure @ HAC	MPa	1.81	3.40	3.40	3.40
	psig	262	493	493	493
Stress Pr/t	MPa	14.82	30.8	31.0	31.1
	psi	2149	4,467	4,498	4510
Margin, (allowed stress / actual stress) - 1	None	1.10	0.01	0.003	0.000
Max Allowed Cladding Inside Radius / Thickness	None	16.75	9.14	9.14	9.14

Note: Table values for cladding thickness and diameters bound current fuel designs and are for example purposes only. However, all fuel to be shipped must have a maximum pre-pressure times the maximum Inside Radius/Thickness product of  $9.14 \times 1.1145 \text{ MPa} = 10.18653 \text{ MPa}$  or less. The thickness of the liner in liner cladding shall be excluded when determining radius and thickness. Thus, all products must meet the maximum product of allowed pressure multiplied by Inside Radius/Thickness of  $10.18653 \text{ MPa}$ .

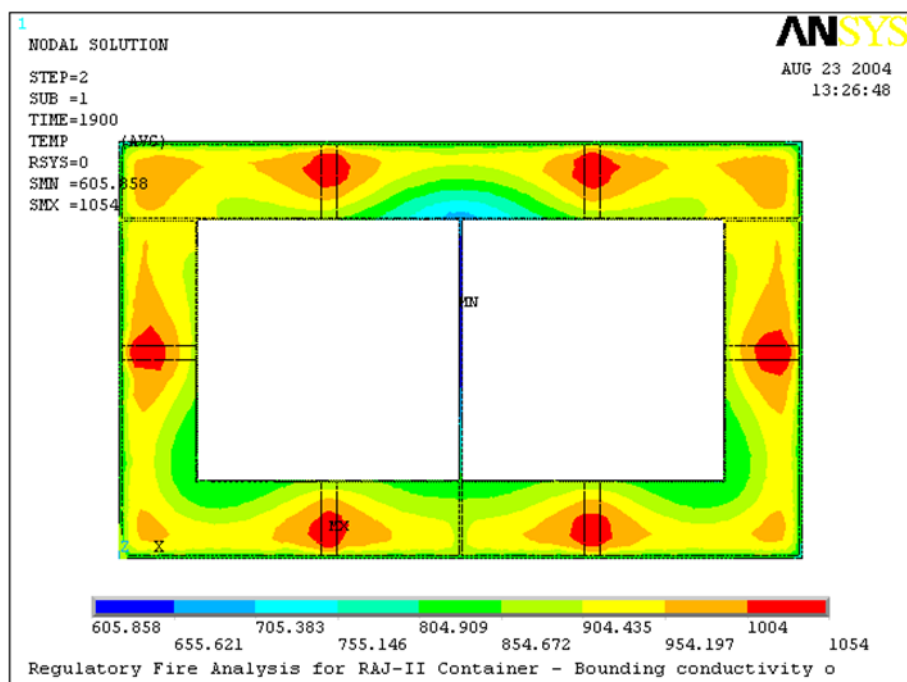




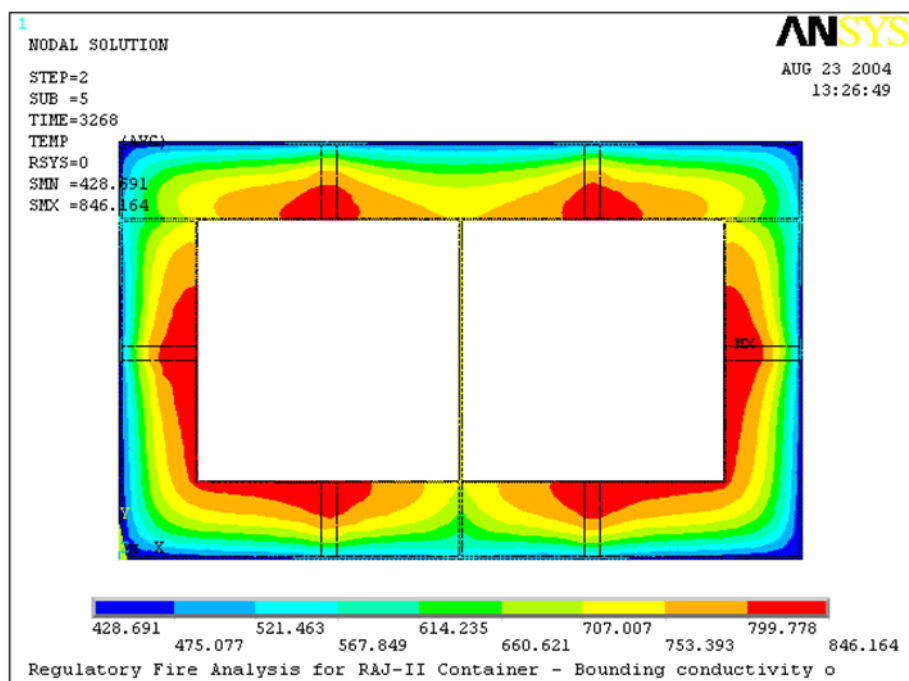
**Figure 3-3 Calculated Temperature Evolution During Transient**



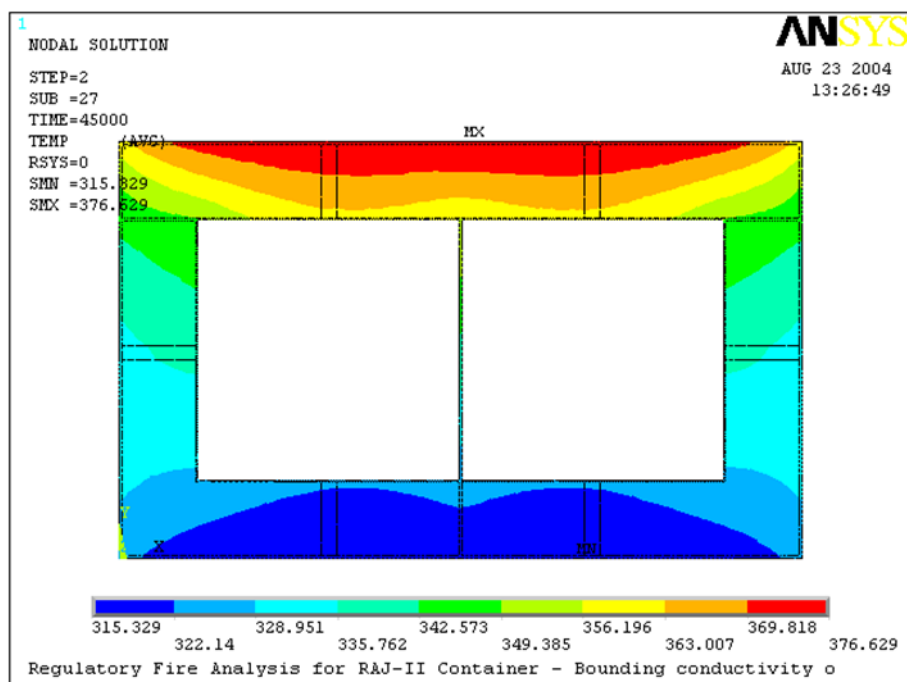
**Figure 3-4 Calculated Isotherms at the End of Fire Phase (1,800 s)**




**Figure 3-5 Calculated Isotherms at 100s After the End of Fire**



**Figure 3-6 Calculated Isotherms at 1,468 s After the End of Fire**



**Figure 3-7 Calculated Isotherms at 12 hr After the End of Fire**

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### 3.6. **APPENDIX**

#### 3.6.1. ***References***

- 3.6.1.1. 10 CFR 71, Packaging and Transportation of Radioactive Material
- 3.6.1.2. Mills, A.F., Heat Transfer, Irwin, Inc., Homewood, Illinois, 1992
- 3.6.1.3. ANSYS Finite Element Computer Code, Version 5.6, ANSYS, Inc., 2000
- 3.6.1.4. McCaffery, B.J., Purely Buoyant Diffusion Flames – Some Experimental Results, Report PB80-112113, U.S. National Bureau of Standards, Washington, D.C., 1979
- 3.6.1.5. Incropera, F.P., Dewitt, D.P., Fundamentals of Heat and Mass Transfer, John Wiley and Sons, Inc., New York, New York, 1996
- 3.6.1.6. GNF-2 Fuel Rod Response to An Abnormal Transportation Event (proprietary)(30 Minute Fire)
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- 3.6.1.9. Thermal Properties of Paper, PTN149, Charles Green, Webster New York, 2002 (<http://www.frontiernet.net/~charmar/>).
- 3.6.1.10. Tran, H.C., and White, R. H., Burning Rate of Solid Wood Measured in a Heat Release Calrimeter, Fire and Materials, Vol. 16, pp 197-206, 1992.
- 3.6.1.11. "Pactec Specification: Regarding Global Nuclear Fuel Specification for Alumina Silicate for use in the RAJ-II Shipping container," Unifrax Corporation, 6/3/04.

### 3.6.2. *ANSYS Input File Listing*

#### Listing of the ANSYS input file (file: model\_fl\_heat.inp)

fini	K,4,0.0015,0.0015,0,
/clear	K,5,0.136,0.0015,0,
/filnam,model_fl_heat,	K,6,0.146,0.0015,0,
/outp,model_fl_heatout,out	K,7,0.2285,0.0015,0,
/PREP7	K,8,0.2305,0.0015,0,
/TITLE, Regulatory Fire Analysis for RAJ-II Container - Bounding conductivity of Alumina	K,9,0.313,0.0015,0,
/UNITS,SI	K,10,0.323,0.0015,0,
/SHOW,JPEG	K,11,0.4575,0.0015,0,
!* !*set element types	K,12,0.459,0.0015,0,
!* ET,1,PLANE55,1	K,13,0.0015,0.0515,0,
ET,2,LINK32	K,14,0.0515,0.0515,0,
ET,3,MATRIX50,1	K,15,0.136,0.0515,0,
!* !* define keypoints	K,16,0.146,0.0515,0,
!* K,1,0,0,0,	K,17,0.2285,0.0515,0,
K,2,0.459,0,0,	K,18,0.2305,0.0515,0,
K,3,0,0.0015,0,	K,19,0.313,0.0515,0,
	K,20,0.323,0.0515,0,
	K,21,0.4075,0.0515,0,
	K,22,0.4575,0.0515,0,
	K,23,0.0515,0.0525,0,
	K,24,0.0525,0.0525,0,






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K,73,0.2285,0.2285,0,	SAVE
K,74,0.2305,0.2285,0,	!*
K,75,0.2355,0.2285,0,	!* define material properties
K,76,0.399,0.2285,0,	!*
K,77,0.4065,0.2285,0,	!*
K,78,0.459,0.2285,0,	!* STAINLESS STEEL (SS304)
K,79,0.,0.2295,0,	!*
K,80,0.0015,0.2295,0,	MP,DENS,1,7900
K,81,0.136,0.2295,0,	MPTEMP,1,300,400,500,600,800,1000
K,82,0.146,0.2295,0,	MPDATA,kxx,1,1,15,17,18,20,23,25
K,83,0.313,0.2295,0,	MPDATA,c,1,1,477,515,539,557,582,611
K,84,0.323,0.2295,0,	!*
K,85,0.4575,0.2295,0,	!* THERMAL INSULATOR
K,86,0.459,0.2295,0,	!*
K,87,0.,0.2795,0,	MP,DENS,2,260
K,88,0.0015,0.2795,0,	MP,C,2,1046
K,89,0.136,0.2795,0,	MPTEMP
K,90,0.146,0.2795,0,	MPTEMP,1,673,873,1073,1273
K,91,0.313,0.2795,0,	MPDATA,KXX,2,1,0.105,0.151,0.198,0.267 !MAX
K,92,0.323,0.2795,0,	VALUES
K,93,0.4575,0.2795,0,	!*
K,94,0.459,0.2795,0,	!*
K,95,0.,0.281,0,	!* WOOD (generic softwood)
K,96,0.459,0.281,0,	!*

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UIMP,3,NUXY, , , ,	FITEM,2,12
UIMP,3,ALPX, , , ,	FITEM,2,11
UIMP,3,REFT, , , ,	FITEM,2,10
UIMP,3,MU, , , ,	FITEM,2,9
UIMP,3,DAMP, , , ,	FITEM,2,8
UIMP,3,DENS, , ,500,	FITEM,2,7
UIMP,3,KXX, , ,0.24,	FITEM,2,6
UIMP,3,C, , ,2800,	FITEM,2,5
UIMP,3,ENTH, , , ,	FITEM,2,4
UIMP,3,HF, , , ,	FITEM,2,3
UIMP,3,EMIS, , , ,	A,P51X
UIMP,3,QRATE, , , ,	FLST,2,7,3
UIMP,3,VISC, , , ,	FITEM,2,3
UIMP,3,SONC, , , ,	FITEM,2,4
UIMP,3,MURX, , , ,	FITEM,2,13
UIMP,3,MGXX, , , ,	FITEM,2,37
UIMP,3,RSVX, , , ,	FITEM,2,41
UIMP,3,PERX, , , ,	FITEM,2,62
!* !* define areas	FITEM,2,61
!* FLST,2,12,3	A,P51X
FITEM,2,1	FLST,2,5,3
	FITEM,2,4
	FITEM,2,5

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FITEM,2,14

FITEM,2,13

A,P51X

FLST,2,4,3

FITEM,2,5

FITEM,2,6

FITEM,2,16

FITEM,2,15

A,P51X

FLST,2,4,3

FITEM,2,6

FITEM,2,7

FITEM,2,17

FITEM,2,16

A,P51X

FLST,2,4,3

FITEM,2,7

FITEM,2,8

FITEM,2,18

FITEM,2,17

A,P51X

FLST,2,4,3

FITEM,2,8

FITEM,2,19

FITEM,2,18

A,P51X

FLST,2,4,3

FITEM,2,9

FITEM,2,10

FITEM,2,20

FITEM,2,19

A,P51X

FLST,2,5,3

FITEM,2,10

FITEM,2,11

FITEM,2,22

FITEM,2,21

FITEM,2,20

A,P51X

FLST,2,7,3

FITEM,2,11


FITEM,2,12

FITEM,2,68


FITEM,2,67

FITEM,2,44


FITEM,2,40

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A,P51X	FITEM,2,19
FLST,2,5,3	FITEM,2,20
FITEM,2,13	FITEM,2,21
FITEM,2,14	FITEM,2,28
FITEM,2,23	FITEM,2,27
FITEM,2,38	FITEM,2,26
FITEM,2,37	FITEM,2,25
A,P51X	FITEM,2,24
FLST,2,8,3	FITEM,2,23
FITEM,2,23	A,P51X
FITEM,2,24	FLST,2,8,3
FITEM,2,29	FITEM,2,25
FITEM,2,49	FITEM,2,26
FITEM,2,64	FITEM,2,33
FITEM,2,63	FITEM,2,55
FITEM,2,42	FITEM,2,74
FITEM,2,38	FITEM,2,73
A,P51X	FITEM,2,54
FLST,2,14,3	FITEM,2,32
FITEM,2,14	A,P51X
FITEM,2,15	FLST,2,8,3
FITEM,2,16	FITEM,2,27
FITEM,2,17	FITEM,2,28


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FITEM,2,43	A,P51X
FITEM,2,66	FLST,2,4,3
FITEM,2,65	FITEM,2,41
FITEM,2,60	FITEM,2,42
FITEM,2,36	FITEM,2,63
A,P51X	FITEM,2,62
FLST,2,5,3	A,P51X
FITEM,2,21	FLST,2,4,3
FITEM,2,22	FITEM,2,43
FITEM,2,40	FITEM,2,44
FITEM,2,39	FITEM,2,67
FITEM,2,28	FITEM,2,66
A,P51X	A,P51X
FLST,2,4,3	SAVE
FITEM,2,37	FLST,2,6,3
FITEM,2,38	FITEM,2,61
FITEM,2,42	FITEM,2,62
FITEM,2,41	FITEM,2,63
A,P51X	FITEM,2,64
FLST,2,4,3	FITEM,2,70
FITEM,2,39	FITEM,2,69
FITEM,2,40	A,P51X
FITEM,2,44	FLST,2,6,3

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FITEM,2,66	FITEM,2,79
FITEM,2,67	A,P51X
FITEM,2,68	FLST,2,4,3
FITEM,2,78	FITEM,2,79
FITEM,2,77	FITEM,2,80
A,P51X	FITEM,2,88
FLST,2,18,3	FITEM,2,87
FITEM,2,69	A,P51X
FITEM,2,70	FLST,2,4,3
FITEM,2,71	FITEM,2,80
FITEM,2,72	FITEM,2,81
FITEM,2,73	FITEM,2,89
FITEM,2,74	FITEM,2,88
FITEM,2,75	A,P51X
FITEM,2,76	FLST,2,4,3
FITEM,2,77	FITEM,2,81
FITEM,2,78	FITEM,2,82
FITEM,2,86	FITEM,2,90
FITEM,2,85	FITEM,2,89
FITEM,2,84	A,P51X
FITEM,2,83	FLST,2,4,3
FITEM,2,82	FITEM,2,82
FITEM,2,81	FITEM,2,83



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FITEM,2,90

A,P51X

FLST,2,4,3

FITEM,2,83

FITEM,2,84

FITEM,2,92

FITEM,2,91

A,P51X

FLST,2,4,3

FITEM,2,84

FITEM,2,85

FITEM,2,93

FITEM,2,92

A,P51X

FLST,2,4,3

FITEM,2,85

FITEM,2,86

FITEM,2,94

FITEM,2,93

A,P51X

SAVE

FLST,2,10,3

FITEM,2,87

FITEM,2,89

FITEM,2,90

FITEM,2,91

FITEM,2,92

FITEM,2,93

FITEM,2,94

FITEM,2,96

FITEM,2,95

A,P51X

SAVE

!\*

!\* glue all areas

!\*

FLST,2,31,5,ORDE,2

FITEM,2,1

FITEM,2,-31

AGLUE,P51X

!\*


/PNUM,KP,0

/PNUM,LINE,0

/PNUM,AREA,1

/PNUM,VOLU,0

/PNUM,NODE,0

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/PNUM,SVAL,0

/NUMBER,0

!\*

/PNUM,ELEM,0

/REPLOT

!\* APLOT

FLST,5,14,5,ORDE,10

FITEM,5,1

FITEM,5,-2

FITEM,5,6

FITEM,5,10

FITEM,5,12

FITEM,5,-15

FITEM,5,21

FITEM,5,-24

FITEM,5,30

FITEM,5,-31

ASEL,S,,P51X

/REPLOT

FLST,5,14,5,ORDE,10

FITEM,5,1

FITEM,5,-2

FITEM,5,6

FITEM,5,10

FITEM,5,12

FITEM,5,-15

FITEM,5,21

FITEM,5,-24

FITEM,5,30

FITEM,5,-31

CM,\_Y,AREA

ASEL,, ,P51X

CM,\_Y1,AREA

CMSEL,S,\_Y

!\*

CMSEL,S,\_Y1

AATT, 1, , 1, 0

CMSEL,S,\_Y

CMDELE,\_Y

CMDELE,\_Y1

!\* ALLSEL,ALL

FLST,5,11,5,ORDE,11

FITEM,5,3

FITEM,5,5

FITEM,5,7

FITEM,5,9

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FITEM,5,11

FITEM,5,16

FITEM,5,19

FITEM,5,-20

FITEM,5,25

FITEM,5,27

FITEM,5,29

ASEL,S, ,P51X

FLST,5,11,5,ORDE,11

FITEM,5,3

FITEM,5,5

FITEM,5,7

FITEM,5,9

FITEM,5,11

FITEM,5,16

FITEM,5,19

FITEM,5,-20

FITEM,5,25

FITEM,5,27

FITEM,5,29

CM\_Y,AREA

ASEL, , ,P51X

CM\_Y1,AREA

!\*

CMSEL,S,\_Y1

AATT, 2, , 1, 0

CMSEL,S,\_Y

CMDELE,\_Y

CMDELE,\_Y1

!\* ALLSEL,ALL

FLST,5,6,5,ORDE,6

FITEM,5,4

FITEM,5,8

FITEM,5,17

FITEM,5,-18

FITEM,5,26

FITEM,5,28

ASEL,S, ,P51X

FLST,5,6,5,ORDE,6

FITEM,5,4

FITEM,5,8


FITEM,5,17

FITEM,5,-18

FITEM,5,26

FITEM,5,28

CM\_Y,AREA

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ASEL, , , P51X

CM,\_Y1,AREA

CMSEL,S,\_Y

!\*

CMSEL,S,\_Y1

AATT, 3, , 1, 0

CMSEL,S,\_Y

CMDELE,\_Y

CMDELE,\_Y1

!\*

ALLSEL,ALL

SAVE

!\*

!\* mesh the areas

!\*

ALLSEL,ALL

APLOT

SMRT,10

FLST,5,31,5,ORDE,2

FITEM,5,1

FITEM,5,-31

CM,\_Y,AREA

ASEL, , , P51X

CHKMSH,'AREA'

CMSEL,S,\_Y

!\*

AMESH,\_Y1

!\*

CMDELE,\_Y

CMDELE,\_Y1

CMDELE,\_Y2

!\*

/PNUM,KP,0

/PNUM,LINE,0

/PNUM,AREA,0

/PNUM,VOLU,0

/PNUM,NODE,0

/PNUM,TABN,0

/PNUM,SVAL,0

/NUMBER,0

!\*


/PNUM,MAT,1

/REPLOT

ALLSEL,ALL

!\* select nodes on the outer surfaces

NSEL,S,LOC,X,0.,0.0001

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NSel,A,LOC,Y,0.,0.0001

NSel,A,LOC,Y,0.2809,0.281

!\* define element for outer surface

!\*

TYPE, 2

MAT, 1

NPLOT

esurf

!\*

!\* create space node

N,50000,0.3,0.5,0,,,

!\* select the nodes and elements that

!\* make up the radiation surfaces

ESEL,S,TYPE,,2

NSLE,R

NSel,S,LOC,X,0.,0.0001

NSel,A,LOC,X,0.4589,0.459

NSel,A,LOC,Y,0.,0.0001

NSel,A,LOC,Y,0.2809,0.281

ESLN,R

NSel,a,node,,50000

FINISH

!\* define radiation matrix

EMIS,1,0.8,

STEF,5.67e-08,

GEOM,1,0,

SPACE,50000,

!\*

VRTXYPE,0,20,

MPRINT,0

WRITE,rad

!\*

ALLSEL,ALL

FINISH

/PREP7

!\*

!\*

TYPE, 3

MAT, 1

REAL,

ESYS, 0

SECNUM,

TSHAP,LINE

!\*

SE,rad, , ,0.0001,

ESEL,S,TYPE,,2

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SAVE

!\* Define effective heat transfer coefficients for

!\* post-fire (vert-20,horiz-up-25, horiz-down-35) MPTEMP

MPTEMP,1,338.71,366.48,394.26,422.04,449.82,477.59,

MPTEMP,7,588.71,755.37,1019.26,

MPDATA,HF,20,1,4.68,5.61,6.18,6.60,6.90,7.13,

MPDATA,HF,20,7,7.64,8.00,8.25,

MPDATA,HF,25,1,5.19,6.34,7.05,7.55,7.92,8.18,

MPDATA,HF,25,7,8.74,9.07,9.17,

MPDATA,HF,35,1,2.34,2.74,2.99,3.17,3.30,3.41,

MPDATA,HF,35,7,3.67,3.89,4.09,

MPLIST

SAVE

FINISH

/SOLU

!\* setup convection coefficients for fire case

ALLSEL,ALL

NSEL,S,LOC,X,0,0.0001

NSEL,A,LOC,X,0.4589,0.459

NSEL,A,LOC,Y,0,0.0001

NSEL,A,LOC,Y,0.2809,0.281

SF,ALL,CONV,19.8,1073

NSEL,ALL

!\*\*\*\*\*

\*\*\*\*\*

!\* Test Heat Generation modelling wood burning

ASEL,S,MAT,,3

ESLA,S

/GO

!\*

\*DIM,burning,TABLE,5,1,0,TIME

!\*

BFE,ALL,HGEN, , %burning%

!\*

!\*\*\*\*\*BFA,ALL,HGEN, %burning%

\*SET,BURNING(1,0,1) , 0.0

\*SET,BURNING(2,0,1) , 0.1

\*SET,BURNING(3,0,1) , 0.2

\*SET,BURNING(4,0,1) , 552.2

\*SET,BURNING(5,0,1) , 552.3

\*SET,BURNING(1,1,1) , 0.0

\*SET,BURNING(2,1,1) , 0.0

\*SET,BURNING(3,1,1) , 7.63e6

\*SET,BURNING(4,1,1) , 7.63e6

\*SET,BURNING(5,1,1) , 0.0

ALLSEL,ALL

SAVE



```

|*****
*****

D,50000,TEMP, 1073

|*****
*****

TUNIF,375,          !REVISED FOR NEW NCT
NUMBER (IC OUTER SHELL)

|*****
*****

SAVE

!*

!* set up run parameters for fire case

!*

ANTYPE,4

!*

TRNOPT,FULL

LUMPM,0

!*

TIME,1800

AUTOTS,-1

DELTIM,0.1,0.1,600,1

KBC,1

!*

TSRES,ERASE

!*

```

```

!*

LSWRITE,2,

!*

!* change boundary conditions for post fire case

!* ALLSEL,ALL

NSEL,S,LOC,X,0.000,0.0001

NSEL,A,LOC,X,0.4589,0.459

SF,ALL,CONV,-20, 311

ALLSEL,ALL

NSEL,S,LOC,Y,0.0,0.0001

SF,ALL,CONV,-35, 311

ALLSEL,ALL

NSEL,S,LOC,Y,0.2809,0.281

SF,ALL,CONV,-25, 311

ALLSEL,ALL

D,50000,TEMP,311

!*

!* apply solar heat flux

!*

ALLSEL,ALL

!* select vertical lines and nodes on the left side

nse|s,loc,x,0


!FLST,5,4,4,ORDE,4

```

!FITEM,5,18	!FITEM,5,35
!FITEM,5,76	!FITEM,5,77
!FITEM,5,94	!FITEM,5,86
!FITEM,5,97	!FITEM,5,108
!LSEL,S,,P51X	!LSEL,S,,P51X
INSL,S,1	INSL,S,1
!FLST,2,97,1,ORDE,9	!FLST,2,97,1,ORDE,9
!FITEM,2,12	!FITEM,2,3
!FITEM,2,17	!FITEM,2,27
!FITEM,2,56	!FITEM,2,57
!FITEM,2,70	!FITEM,2,63
!FITEM,2,72	!FITEM,2,78
!FITEM,2,447	!FITEM,2,795
!FITEM,2,-521	!FITEM,2,-869
!FITEM,2,2039	!FITEM,2,2240
!FITEM,2,-2055	!FITEM,2,-2256
/GO	/GO
!*	!*
F,all,HEAT,0.69	F,all,HEAT,0.69
ALLSEL,ALL	!* select nodes on upper surface
!* select lines and nodes on the right side	ALLSEL,ALL
nsl,s,loc,x,.459,.460	NSEL,S,LOC,Y,0.2809,0.281

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!FITEM,2,79	/SOLU
!FITEM,2,-80	/STATUS,SOLU
!FITEM,2,2257	LSSOLVE,2,3,1
!FITEM,2,-2409	FINISH
!/GO	SAVE
!*	/POST26
F,all,HEAT,2.88	!*
ALLSEL,ALL	!* plot temperature evolution at specified nodes
!* set up run parameters for post fire	!*
TIME,14400 !was 9000	!*
AUTOTS,-1	!* inner wall, top right corner
DELTIM,0.5,0.1,2000,1	NSOL,2,58,TEMP, ,inn_wtr
KBC,1	!*
!*	!*
TSRE S,ERASE	!* inner wall, bottom mid position
!*	NSOL,3,1185,TEMP, ,inn_wbm
TINTP,0.005, , , -1,0.5,-1	!*
!*	!*
OUTRES,ALL,ALL,	!* inner wall, top mid position
TIME,45000	NSOL,4,1720,TEMP, ,inn_wtm
DELTIM,100,10,2000,1	!*
LSWRITE,3,	!*
SAVE	!* outer wall, top mid position

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```

!*

!*

PLVAR,2,3,4,5,,, , , , ,

PRVAR,2,3,4,5,,,

FINISH

!* plot isothermes at certain moments in time

/POST1

SET,LIST,2

SET, , , 1, , , , 17,

/EFACE,1

!*

PLNSOL,TEMP, , 0,

FINISH

/POST1

SET, , , 1, , , , 18,

/EFACE,1

!*

PLNSOL,TEMP, , 0,

SET, , , 1, , , , 20,

/EFACE,1

!*

PLNSOL,TEMP, , 0,

SET, , , 1, , , , 22,

```

```

!*

PLNSOL,TEMP, , 0,

SET, , , 1, , , , 30,

/EFACE,1

!*

PLNSOL,TEMP, , 0,

SET, , , 1, , , , 43,

/EFACE,1

!*

PLNSOL,TEMP, , 0,

SET,PREVIOUS

FINISH

|*****NEW

allsel

/post1

Tmax=0


TimeMAX=0

nmax=0

nset,s,loc,x,0.0525,.4065,

nset,r,loc,y,0.0525,.2285,

```

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nplot

\*GET, ncount, NODE, 0, count

cm,icnodes,node

set,1,1

\*do,t,1,46

tmaxn=0

cmsel,s,icnodes

\*do,i,1,ncount

nodei=node(0,0,0)

\*get,tempi,node,nodei,temp

\*if,tempi,gt,tmaxn,then

tmaxn=tempi

nmaxn=nodei

\*endif

nsel,u,,nodei

\*enddo

\*if,tmaxn,gt,tmax,then

tmax=tmaxn

nmax=nmaxn

\*endif

set,next

\*enddo

tmax=tmax

nmax=nmax

timemax=timemax

allsel

/show,term

/post1,

! Reverse Video

/rgb,index,100,100,100,0

/rgb,index,80,80,80,13

/rgb,index,60,60,60,14

/rgb,index,0,0,0,15

set,1,17


plnsol,temp

/image,save,fig3-4(1800),wmf

set,2,1

/replot

/image,save,fig3-5(1900),wmf

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```

/replot

/image,save,fig3-6(3268),wmf

set,last

/replot

/image,save,fig3-7(45000),wmf

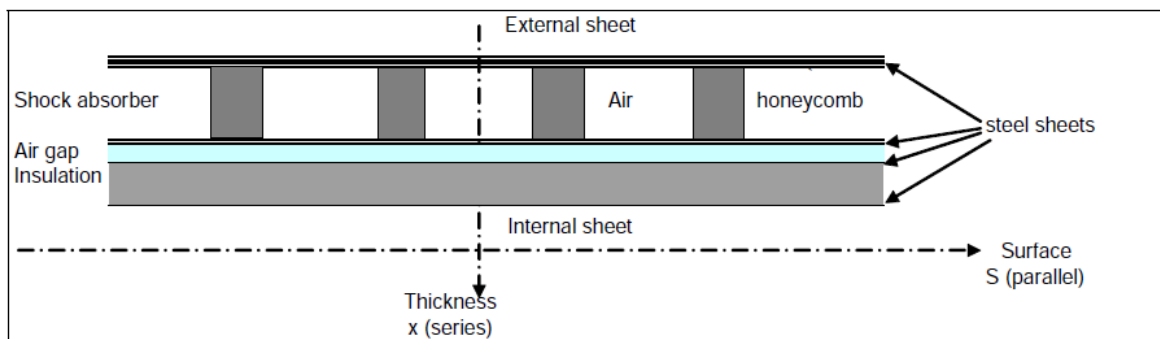
!*****NEW

!/EXIT,ALL

```

### 3.6.3. *NCT Transient Analysis*

The transient analysis uses a one dimensional model of the vertical face of the packaging (thinner part of the packaging) as described in the figure below:



**Figure 3-8 Vertical Face Model**

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The heat flux is set as a sine wave function:

$$Q = \pi/2 \times 800 \sin(\omega \theta) \quad 0 < (\omega \theta) < \pi$$

$$Q = 0 \quad \pi < (\omega \theta) < 2\pi$$

With:  $Q$  = heat energy in g-cal/cm<sup>2</sup>

$\omega$  =  $2\pi / 24$  pulsation

$\theta$  = time in hour

Note that the peak value of  $(\pi/2 \times 800)$  complies with 10CFR 71.71(c)(1), conservatively assuming the highest value of 800 g-cal/cm<sup>2</sup> for the insolation.

$$\int_0^{24 \text{ hr}} Q d\theta = 800 \text{ g-cal/cm}^2$$

Assuming that at each time step, the external surface of the package achieves steady state conditions, the energy balance between the solar heat load, and the convection and radiation exchanges (see section 3.4.1.1), results time dependant solution for the external surface temperature.

The result is plotted on the Figure 3.6.3-1 (blue curve) and is close to a sine wave function. Indeed, when calculating the energy balance equation, it appears that the convention term represents 65% of the exchange, and the radiation term 35%. As the convection term is linearly proportional to the external temperature, this curve is nearly proportional to the solar heat load.

Assume that the external temperature is a sine function with respect to time as follows (and as plotted on Figure 3.6.3-1):

$$T_s = T_{\text{avg}} + T^+ \sin(\omega \theta)$$

With:  $T_{\text{avg}} = 420 \text{ K}$  (maximum value of the blue curve)

$$T^+ = (420 - 311) = 109 \text{ K}$$

The system is thus modeled as a one dimensional model of conduction, with a sinusoidal wave temperature on the external surface as a boundary condition.



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Using equation 4-22 of the “Handbook of Heat Transfer”, Reference 3.6.1.7, the heat equation through a layer of material leads to a temperature of:

$$T(x,\theta) = T_{avg} + T^+ \exp(-L x/d) \sin[L(2 L Fo - x/d)]$$

Using the reference’s notation, it becomes:

$$T(x,\theta) = T_{avg} + T^+ \exp[-(\omega/2\alpha)^{1/2} x] \sin[\omega \theta - (\omega/2\alpha)^{1/2} x]$$

With:  $\alpha = K / \rho C$  = thermal diffusivity,  
 $K$  = conductivity of material,  
 $\rho$  = density of material,  
 $C$  = specific heat of the material,  
 $x$  = thickness thru the material.

Through each layer of material “i” in the TN-B1 packaging, the temperature of the external surface is so decreased by a factor  $\eta$  and lagged by a factor  $\varphi$ :

$$\eta_i = \exp[-(\omega/2\alpha_i)^{1/2} x_i]$$

$$\varphi_i = (\omega/2\alpha_i)^{1/2} x_i$$

Table 3.6.3-1 summarizes the material properties for each component layer through the thickness of the model.

### Equivalent properties of material

The thermal properties ( $K$ ,  $\rho$ ,  $C$ ) of a material equivalent to materials of a system are following the rules:

$$\text{Material in series } K = \frac{e_T}{\sum_i \frac{e_i}{K_i}}$$


$$\text{Material in parallel } K = \frac{1}{S_T} \sum_i S_i K_i$$

$$\text{Material in series } \rho C = \frac{\sum_i \rho_i C_i e_i}{e_T}$$

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$$Materials\ in\ parallel\ \rho\ C = \frac{\sum_i \rho_i C_i S_i}{S_T}$$

The maximum temperature of the cavity surface of the packaging resulting from solving the one dimensional model occurs at ten hours into the cycle and is equal to 350 K. The maximum temperature on the outer surface of the inner container occurs at 8 hours and is equal to 375K. Temperatures are summarized on Table 3.6.3-2.

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**Table 3-6 Material properties**

Component	Material	Thickness x (m)	Surface S (m)	Conductivity K (W/m-K)	Density r (kg/m <sup>3</sup> )	Specific heat C (J/kg-K)	Diffusivity a (m <sup>2</sup> /s)
OC outer sheet	steel	0.004	-	15	7900	477	3.981E-06
Honeycomb <sup>1</sup>	paper	-	0.084 <sup>1</sup>	0.13595	700 <sup>1</sup>	1531 <sup>1</sup>	3.932E-07
	air	-	0.916 <sup>1</sup>	0.0267	1.177	1005	
Shock absorbers	honeycomb	0.108	0.64	0.0359	60	1522	1.737E-06
	air		3.186	0.0267	1.177	1005	
OC inner sheet	steel	0.001	-	15	7900	477	3.981E-06
Air gap	air	0.01	-	0.0267	1.177	1005	2.257E-05
IC outer sheet	steel	0.0015	-	15	7900	477	3.981E-06
IC insulation	Alumina	0.048	-	0.09	250	1046	3.442E-07
IC inner sheet	steel	0.001	-	15	7900	477	3.981E-06


- <sup>1</sup> The honeycomb is assumed to be a combination of paper and air in a parallel system (see below). The proportion of paper and air is determined by the ratio of the densities:

Honeycomb density = 60 kg/m<sup>3</sup>

Paper density = 700 kg/m<sup>3</sup>                      8.4%

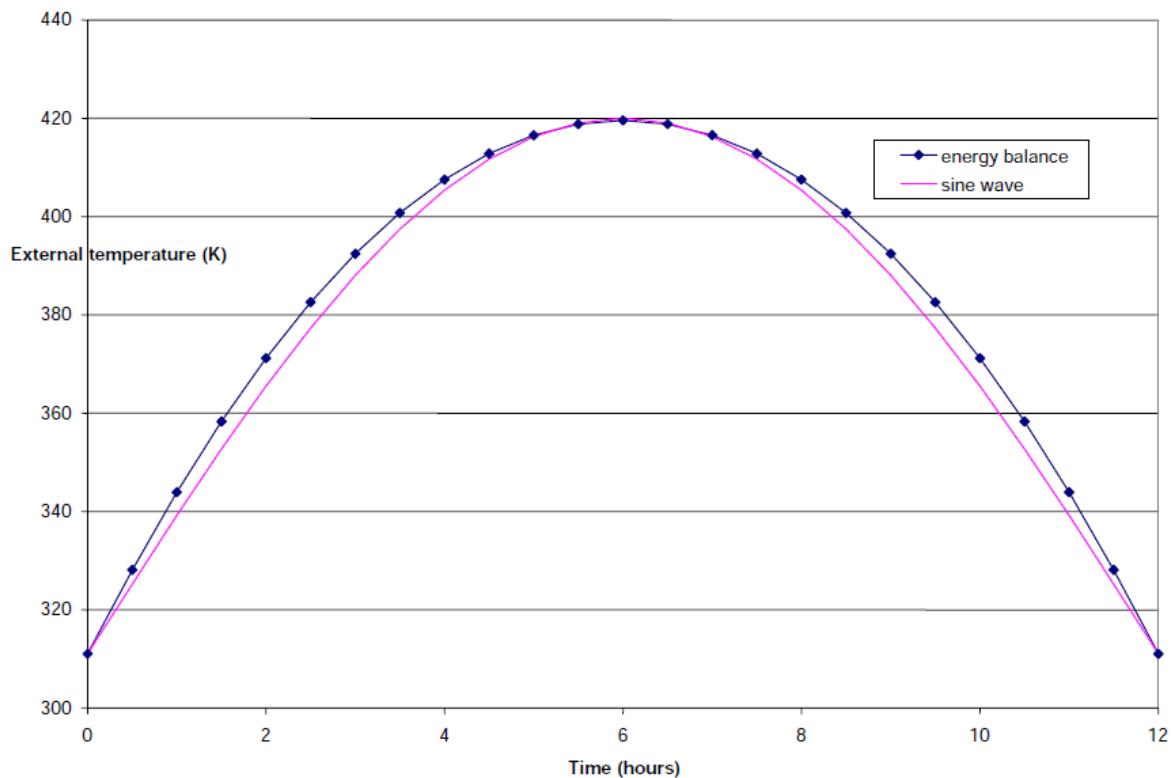
Air density = 1.177 kg/m<sup>3</sup>                      91.6%

Thermal properties of resin impregnated kraft paper (density, conductivity, specific heat) are conservatively assumed to correspond to that of ordinary paper according to Reference 3.6.1.9.

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**Table 3-7 NCT Temperatures Through the Package Thickness**

Time (hour)	Surface temp sin wave Ts	T thru OC Outer	T thru Honeycomb and	T thru OC Inner	T thru Air Gap	T thru IC Inner Shell	T thru Alumina Sillicate
0	311	311	311	311	311	311	311
0.5	325	324	311	311	311	311	311
1	339	338	311	311	311	311	311
1.5	353	351	311	311	311	311	311
2	366	364	312	312	311	311	311
2.5	377	376	321	320	320	319	311
3	388	386	329	329	328	327	311
3.5	397	396	337	337	336	335	311
4	405	404	345	345	343	343	312
4.5	412	410	352	352	350	350	317
5	416	415	358	358	357	356	322
5.5	419	418	364	364	362	362	327
6	420	419	368	368	367	367	332
6.5	419	418	372	372	371	370	336
7	416	415	375	375	373	373	340
7.5	412	411	376	376	375	375	343
8	405	405	377	376	376	<b>375</b>	346
8.5	397	397	376	376	375	375	348
9	388	388	374	374	373	373	349
9.5	377	378	371	371	371	371	<b>350</b>
10	366	366	367	367	367	367	350
10.5	353	353	362	362	362	362	350
11	339	340	357	357	357	357	349
11.5	325	326	350	350	350	350	347
12	311	312	343	343	343	343	344
12.5	311	311	335	335	336	336	342
13	311	311	327	327	328	328	338
13.5	311	311	318	319	319	320	334
14	311	311	311	311	311	311	330
14.5	311	311	311	311	311	311	325
15	311	311	311	311	311	311	320
15.5	311	311	311	311	311	311	315
16	311	311	311	311	311	311	311
16.5	311	311	311	311	311	311	311



**Figure 3-9 Comparison Between Energy Equation Solution with a Sine Wave Equation**

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## 4. CONTAINMENT

### 4.1. DESCRIPTION OF THE CONTAINMENT SYSTEM

#### 4.1.1. *Containment Boundary*

TN-B1 container is limited to use for transporting low enriched uranium, nuclear reactor fuel assemblies and rods. The radioactive material is bound in sintered ceramic pellets having very limited solubility and has minimal propensity to suspend in air. The pellets are sintered at temperatures greater than 1,600°C. These pellets are further sealed into zirconium alloy cladding to form the fuel rod portion of each assembly. The containment boundary for the TN-B1 package is the fuel rod. The components of the fuel rod which constitutes the containment boundary are the zirconium cladding and end caps. The fuel cladding is sealed on each end by end caps which are welded to the cladding. Figure 1-6 Example Fuel Rod (Primary Containment), shows the containment system. The containment system includes the ceramic sintered pellet, clad in sealed zirconium fuel rods which are contained in a stainless steel box which is contained in another stainless steel box.

The fuel rods are manufactured under a Quality Assurance Program meeting the requirements of 10 CFR 71 subpart H. Welds of the fuel rod end caps to the cladding are conducted under a qualified process and verified for integrity by such means as X-ray inspection, ultrasonic testing, or process control. There are no penetrations in the fuel cladding when shipped. The fuel cladding after loading with the pellets is pressurized with helium and end plugs are welded on to close the rod. These welds are designed to withstand the rigorous operating environment of a nuclear reactor. For 11x11 fuel rods, the integrity of the closure welds for the fuel rods are periodically assessed using burst testing. This testing is performed in accordance with guidance provided in a national consensus standard specification for seamless zirconium tubes for fuel rod cladding (ASTM B811 13. *Standard Specification for Wrought Zirconium Alloy Seamless Tubes for Nuclear Reactor Fuel Cladding*, Annex A.1). This testing is considered successful if the ultimate hoop strength at room temperature obtained is equal to or greater than the minimum ultimate strength established from room temperature, longitudinal tensile tests for the same lot. The fact that the ultimate hoop strength is equivalent to or greater than the ultimate tensile strength of the base material provides assurance that finite element analyses assuming only homogenous base material appropriately—and likely conservatively—represents the dynamic structural performance of the fuel cladding. The fuel rod is leak tested after fabrication to demonstrate that it is leak tight ( $<1 \times 10^{-7}$  atm-cc/s).

#### 4.1.2. *Special Requirements for Plutonium*

This section is not applicable since the package is not being used for plutonium shipments.

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## 4.2. GENERAL CONSIDERATIONS

### 4.2.1. *Type A Fissile Packages*

The Type A fissile package is constructed, and prepared for shipment so that there is no loss or dispersal of the radioactive contents and no significant increase in external surface radiation levels and no substantial reduction in the effectiveness of the packaging during normal conditions of transport. The fissile material is bound as a ceramic pellet and contained in a zirconium fuel rod. These rods are leak tested prior to shipment to assure their integrity. Chapter 6.0 demonstrates that the package remains subcritical under normal and hypothetical accident conditions.

### 4.2.2. *Type B Packages*

The Type B fissile package is constructed, and prepared for shipment so that there is no loss or dispersal of the radioactive contents and no significant increase in external surface radiation levels and no substantial reduction in the effectiveness of the packaging during normal conditions of transport.

The package satisfies the quantified release rate of 10 CFR 71.51 by having a release rate less than  $10^{-6} A_2/\text{hr}$  as demonstrated below.

$$A_2 = 0.18 \text{ Ci, therefore } 10^{-6} A_2 = 1.8 \times 10^{-7} \text{ Ci/hr}$$

The mass density of  $\text{UO}_2$  in an aerosol from NUREG/CR-6487, page 17 is  $9 \times 10^{-6} \text{ g/cm}^3$ .

Specific Activity of fuel material is  $1.4 \times 10^{-5} \text{ Ci/g UO}_2$  (7.89 Ci/56kg  $\text{UO}_2$ ).

Leak rate at  $1 \times 10^{-7} \text{ atm-cm}^3/\text{s}$  ( $3.6 \times 10^{-4} \text{ cm}^3/\text{hr}$ ) is equal to  $1 \times 10^{-6} \text{ atm-cm}^3/\text{s}$  ( $3.6 \times 10^{-3} \text{ cm}^3/\text{h}$ ) when pressurized to 10 atm. Assuming that the pressure is further increased due to temperature the leak rate is assumed to increase by an additional factor of 10 so that it is equal to  $3.6 \times 10^{-2} \text{ cm}^3/\text{h}$ .

$$\begin{aligned} \text{Release rate} &= 3.6 \times 10^{-2} \text{ cm}^3/\text{hr} \times 1.4 \times 10^{-5} \text{ Ci/g UO}_2 \times 9 \times 10^{-6} \text{ g/cm}^3 \\ &= 4.5 \times 10^{-12} \text{ Ci/h} \end{aligned}$$

Much less than the  $1.7 \times 10^{-7} \text{ Ci/hr}$  limit.

## 4.3. CONTAINMENT UNDER NORMAL CONDITIONS OF TRANSPORT (TYPE B PACKAGES)

The nature of the contained radioactive material and the structural integrity of the fuel rod cladding including the closure welds are such that there will be no release of radioactivity under



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normal conditions of transport. The welded close containment boundary is not affected by any of the normal conditions of transport as demonstrated in the previous chapters. The pressurization that could be seen by the containment boundary is far below the normal conditions the fuel experiences while in service.

#### 4.4. **CONTAINMENT UNDER HYPOTHETICAL ACCIDENT CONDITIONS (TYPE B PACKAGES)**

The sintered pellet form of the radioactive material and the integrity of the fuel rod cladding are such that there will be no substantial release of radioactivity under the Hypothetical Accident Conditions. Before and after the accident condition testing the rods were helium leak tested demonstrating leak tightness. Similar fuel rods have been tested at temperatures and resulting pressures that will be seen by fuel shipped in the TN-B1.


10 CFR 71.51 requires that no escape of other radioactive material exceeding a total amount  $A_2$  in 1 week, and no external radiation dose rate exceeding 10 mSv/h (1 rem/h) at 1 m (40 in) from the external surface of the package. The following qualitative assessment demonstrates that the performance requirement of 10 CFR 71.51(a)(2) will be satisfied.

Table 1-4 shows the calculated  $A_2$  for the mixture of the maximum radionuclide content in the package is 0.18 Ci. The total radioactivity in the package using the maximum isotopic values is 7.89 Ci. The mass of  $UO_2$  equivalent to an activity of 7.89 Ci is 562 kg (281 kg  $UO_2$ /assembly x 2 assemblies) which yields a mass to activity ratio of 71.2 kg  $UO_2$ /Ci. The mass equivalent  $A_2$  is therefore 12.8 kg  $UO_2$ .

Following the drop test, fuel rods were leak tested and shown to have a very low leak rate of He at a rate of  $5.5 \times 10^{-6} \text{ cm}^3/\text{s}$ . Over one week this is equal to  $3.3 \text{ cm}^3$  ( $5.5\text{E-}6 \text{ cm}^3/\text{s} \times 6.05\text{E}5 \text{ s/wk} = 3.3 \text{ cm}^3/\text{wk}$ ). The tested assembly had 91 fuel rods while the 11x11 has 112 fuel rods. As a result a conservative assumption was made that the amount released would increase proportionately to the number of fuel rods. This was determined to be  $4.1 \text{ cm}^3/\text{wk}$  ( $3.3 \text{ cm}^3/\text{wk} \times 112 \text{ rods}/91 \text{ rods}$ ). Conservatively assuming that the density of the radioactive material is  $10\text{g}/\text{cm}^3$  and using the  $A_2$  mass above of 12.8 kg of  $UO_2$ , the  $UO_2$  would have a volume of  $1,280 \text{ cm}^3/\text{wk}$ . This is much greater than the volume leaked. This calculation is extremely conservative since the  $UO_2$  would predominantly stay in a ceramic form and not be available for dispersion.

Test fuel rods as described in Section 2.0 have been baked at  $800^\circ\text{C}$  for over 30 minutes and did not leak.

Additionally, the large mass, 12.8 kg, of material required to exceed the  $A_2$  would require a catastrophic failure of the rod, significant leak of the inner and outer container.

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Dose rates are less than the 10mSv/hr under any condition because of the low specific activity and low abundance of gamma emitters in the fuel.


Based on this evaluation, it is demonstrated that the package meets the containment requirements of 10 CFR 71.51

#### **4.5. LEAKAGE RATE TESTS FOR TYPE B PACKAGES**

During manufacturing each fuel rod is He leak tested to demonstrate that it is leak tight ( $<1 \times 10^{-7}$  atm-cc/s). There are no leak rate requirements for the inner and outer packaging.

#### **4.6. APPENDIX**

None

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## 5. SHIELDING EVALUATION

The contents of the TN-B1 require no shielding since unirradiated fuel gives off no significant radiation either gamma or neutron. Hence the TN-B1 provides no shielding. The minimal shielding provided by the stainless steel sheet is not required. The dose rate limits established by 10 CFR 71.47(a) for normal conditions of transport (NCT) are verified prior to shipping by direct measurement.

Since there is no shielding provided by the package, there is no shielding change during the Hypothetical Accident Conditions (HAC). Therefore, the higher dose rate allowed by 10 CFR 71.51(a)(2) will be met.

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## 6. CRITICALITY EVALUATION

### 6.1. DESCRIPTION OF CRITICALITY DESIGN

A criticality safety analysis is performed to demonstrate the TN-B1 shipping container safety. The TN-B1 meets applicable IAEA and 10 CFR 71 requirements for a Type B fissile material-shipping container, transporting heterogeneous UO<sub>2</sub> enriched to a maximum of 5.00 wt. percent U-235.


The TN-B1 shipping container design features a stainless steel inner container positioned inside an outer stainless steel container by four evenly spaced stainless steel fixture assemblies. The fixture assemblies cradle the inner container and prevent horizontal or vertical movement. The inner container has two fuel assembly transport compartments, aligned side-by-side and separated by a stainless steel divider. Each transport compartment is lined with polyethylene foam in which the fuel assemblies rest. Additional container details are described in Section 1.2, Package Description. Material manufacturing tolerances are presented in the general arrangement drawings in Section 1.4.1.

The uranium transported in the TN-B1 container is UO<sub>2</sub> pellets enclosed in zirconium alloy cladding. The fuel rods are arranged in 8x8, 9x9, 10x10, or 11x11 square lattice arrays at fixed center- to-center spacing. Fuel rods may also be transported loose with no fixed center-to-center spacing, bundled together in a close packed configuration, or inside a 5-inch diameter stainless steel pipe or protective case.

Water exclusion from the inner container is not required for this package design. The inner container is analyzed in both undamaged and damaged package arrays under optimal moderation conditions and is demonstrated to be safe under Normal Conditions of Transport (NCT) and Hypothetical Accident Condition (HAC) testing.

The criticality analysis for the TN-B1 container is performed at a maximum enrichment of 5.00 wt. percent U-235 for UO<sub>2</sub> or Uranium-Carbide fuel pellets contained in zirconium alloy or stainless steel clad cylindrical rods. The cylindrical fuel rods are arranged in 8x8, 9x9, 10x10, or 11x11 square lattice arrays at fixed center-to-center spacing. Sensitivity analyses are performed by varying fuel parameters (rod pitch, clad ID, clad OD, pellet OD, fuel orientation, polyethylene spacer quantity, and moderator density) to obtain the most reactive configuration. The most reactive configuration is modeled for each authorized payload to demonstrate safety and to validate the fuel parameter ranges specified as loading criteria.


Table 6-1 TN-B1 Fuel Assembly Loading Criteria summarizes the fuel loading criteria for the TN-B1 shipping container.

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**Table 6-1 TN-B1 Fuel Assembly Loading Criteria**

Parameter	Units	Type	Type	Type	Type
Fuel Assembly Type	Rods	8x8	9x9	FANP 10x10	GNF 10x10
UO <sub>2</sub> Density <sup>a2</sup>	g/cm <sup>3</sup>	≤ 10.74	≤ 10.74	≤ 10.74	≤ 10.74
Number of water rods	#	0, 2x2	0, 2-2x2 off-center diagonal, 3x3	0, 2-2x2 off-center diagonal, 3x3	0, 2-2x2 off-center diagonal, 3x3
Number of fuel rods	#	60 - 64	72 - 81	91 - 100	91 - 100
Fuel Rod OD	cm	≥ 1.176	≥ 1.093	≥ 1.000	≥ 1.010
Fuel Pellet OD	cm	≤ 1.05	≤ 0.96	≤ 0.895	≤ 0.895
Cladding Type		Zirconium Alloy	Zirconium Alloy	Zirconium Alloy	Zirconium Alloy
Cladding ID	cm	≤ 1.10	≤ 1.02	≤ 0.933	≤ 0.934
Cladding Thickness	cm	≥ 0.038	≥ 0.036	≥ 0.033	≥ 0.038
Active fuel length	cm	≤ 381	≤ 381	≤ 385	≤ 385
Fuel Rod Pitch	cm	≤ 1.692	≤ 1.51	≤ 1.350	≤ 1.350
U-235 Pellet Enrichment	wt%	≤ 5.0	≤ 5.0	≤ 5.0	≤ 5.0
Maximum Lattice Average Enrichment	wt%	≤ 5.0	≤ 5.0	≤ 5.0	≤ 5.0
Channel Thickness <sup>a1</sup>	cm	0.17 – 0.3048	0.17 – 0.3048	0.17 – 0.3048	0.17 – 0.3048
Part Length Fuel Rods (1/3 through 2/3 normal length)	Max #	None	12	14	14

- a1. Transport with or without channels is acceptable  
a2. Density based on a pellet modeled as a right cylinder.

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**Table 6-1 TN-B1 Fuel Assembly Loading Criteria (continued)**

Parameter	Units	Value
Fuel Assembly Type		11x11
UO <sub>2</sub> Density <sup>b3</sup>	g/cm <sup>3</sup>	≤ 10.763
Number of water rods	#	3x3 center
Number of fuel rods	#	112
Fuel Rod OD	cm	≥ 0.930
Fuel Pellet OD	cm	≤ 0.820
Cladding Type		Zirconium Alloy
Cladding ID	cm	≤ 0.840
Cladding Thickness	cm	≥ 0.045
Fuel Rod Pitch <sup>b1</sup>	cm	≤ 1.195
U-235 Pellet Enrichment	wt%	≤ 5.0
Maximum Lattice Average Enrichment	wt%	≤ 5.0
Fuel Channel Side Thickness <sup>b2</sup>	cm	≤ 0.254
Full Length Fuel Rods		
Quantity	#	92
Active length	cm	≤ 385
Short Part Length Fuel Rods		
Quantity	#	12
Active length	cm	≤ 155.1
Long Part Length Fuel Rods		
Quantity	#	8
Active length	cm	≤ 236.8

b1. Equivalent nominal pitch per Section 6.12.3.1.1.

b2. Transport with or without channels is acceptable.

b3. Density based on a pellet modeled as a right cylinder.

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**Table 6-1 TN-B1 Fuel Assembly Loading Criteria (continued)**

Parameter	Units	Type	Type	Type	Type
Fuel Assembly Type	Rods	8x8	9x9	FANP 10x10	GNF 10x10
Gadolinia Requirements Lattice Average Enrichment <sup>c1</sup>					
≤ 5.0 wt % U-235	#	7 @ 2 wt %	10 @ 2 wt %	12 @ 2 wt %	12 @ 2 wt %
≤ 4.7 wt % U-235	@ wt%	6 @ 2 wt %	8 @ 2 wt %	12 @ 2 wt %	12 @ 2 wt %
≤ 4.6 wt % U-235	Gd <sub>2</sub> O <sub>3</sub>	6 @ 2 wt %	8 @ 2 wt %	10 @ 2 wt %	10 @ 2 wt %
≤ 4.3 wt % U-235		6 @ 2 wt %	8 @ 2 wt %	9 @ 2 wt %	9 @ 2 wt %
≤ 4.2 wt % U-235		6 @ 2 wt %	6 @ 2 wt %	8 @ 2 wt %	8 @ 2 wt %
≤ 4.1 wt % U-235		4 @ 2 wt %	6 @ 2 wt %	8 @ 2 wt %	8 @ 2 wt %
≤ 3.9 wt % U-235		4 @ 2 wt %	6 @ 2 wt %	6 @ 2 wt %	6 @ 2 wt %
≤ 3.8 wt % U-235		4 @ 2 wt %	4 @ 2 wt %	6 @ 2 wt %	6 @ 2 wt %
≤ 3.7 wt % U-235		2 @ 2 wt %	4 @ 2 wt %	6 @ 2 wt %	6 @ 2 wt %
≤ 3.6 wt % U-235		2 @ 2 wt %	4 @ 2 wt %	4 @ 2 wt %	4 @ 2 wt %
≤ 3.5 wt % U-235		2 @ 2 wt %	2 @ 2 wt %	4 @ 2 wt %	4 @ 2 wt %
≤ 3.3 wt % U-235		2 @ 2 wt %	2 @ 2 wt %	2 @ 2 wt %	2 @ 2 wt %
≤ 3.1 wt % U-235		None	2 @ 2 wt %	2 @ 2 wt %	2 @ 2 wt %
≤ 3.0 wt % U-235		None	None	2 @ 2 wt %	2 @ 2 wt %
≤ 2.9 wt % U-235		None	None	None	None
Polyethylene Equivalent Mass (Maximum per Assembly) <sup>c2</sup>	kg	11	11	10.2	10.2

c1. Required gadolinia rods must be distributed symmetrically about the major diagonal

c2. Polyethylene equivalent mass (refer to 6.3.2.2)



**Table 6-1 TN-B1 Fuel Assembly Loading Criteria (continued)**

Parameter	Units	Type
Fuel Assembly Type	Rods	11x11
Gadolinia Requirements Lattice Average Enrichment <sup>d1</sup>		
≤ 5.0 wt % U-235	#	13 @ 2 wt %
≤ 4.8 wt % U-235	@ wt%	12 @ 2 wt %
≤ 4.6 wt % U-235	Gd <sub>2</sub> O <sub>3</sub>	11 @ 2 wt %
≤ 4.4 wt % U-235		10 @ 2 wt %
≤ 4.2 wt % U-235		9 @ 2 wt %
≤ 4.1 wt % U-235		8 @ 2 wt %
≤ 3.9 wt % U-235		7 @ 2 wt %
≤ 3.8 wt % U-235		6 @ 2 wt %
≤ 3.6 wt % U-235		5 @ 2 wt %
≤ 3.5 wt % U-235		4 @ 2 wt %
≤ 3.3 wt % U-235		3 @ 2 wt %
≤ 3.2 wt % U-235		2 @ 2 wt %
≤ 2.9 wt % U-235		None
Polyethylene Equivalent Mass (Maximum per Assembly) <sup>d2</sup>	kg	10.2

d1. Required gadolinia-urania rods shall be distributed symmetrically about the major diagonal and shall not be placed on the periphery.

d2. Polyethylene equivalent mass (refer to 6.3.2.2)

Cylindrical fuel rods containing UO<sub>2</sub>, enriched to 5 wt. percent U-235, are analyzed within the TN-B1 inner container in a 5-inch stainless steel pipe, loose, in a protective case, or bundled together. The fuel rod loading criteria, determined from the criticality evaluation for the TN-B1 shipping container, are shown in *Table 6-2 TN-B1 Fuel Rod Loading Criteria*.

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**Table 6-2 TN-B1 Fuel Rod Loading Criteria**

Parameter	Units	Type						
		8x8 (UO <sub>2</sub> )	9x9 (UO <sub>2</sub> )	10x10 (UO <sub>2</sub> )	11x11 (UO <sub>2</sub> )	CANDU-14 (UC)	CANDU-25 (UC)	Generic PWR (UO <sub>2</sub> )
Fuel Assembly Type								
UO <sub>2</sub> or UC Fuel Density <sup>a</sup>	g/cm <sup>3</sup>	≤10.74	≤10.74	≤10.74	≤10.763	≤13.36	≤13.36	≤10.74
Fuel rod OD	cm	≥1.10	≥1.02	≥1.00	≥0.930	≥1.340	≥0.996	≥1.118
Fuel Pellet OD	cm	≤1.05	≤0.96	≤0.90	≤0.820	≤1.254	≤0.950	≤0.98
Cladding Type		Zirc. Alloy	Zirc. Alloy	Zirc. Alloy	Zirc. Alloy	Zirc. Alloy or SS	Zirc. Alloy or SS	Zirc. Alloy or SS
Cladding ID	cm	≤1.10	≤1.02	≤1.00	≤0.930	≤1.267	≤0.951	≤1.004
Cladding Thickness	cm	≥0.00	≥0.00	≥0.00	≥0.00	≥0.00	≥0.00	≥0.00
Active fuel Length	cm	≤381	≤381	≤385	≤385	≤47.752	≤40.013	≤450
Maximum U-235 Pellet Enrichment	wt. %	≤5.0	≤5.0	≤5.0	≤5.0	≤5.0	≤5.0	≤5.0
Maximum Average fuel rod Enrichment	wt. %	≤5.0	≤5.0	≤5.0	≤5.0	≤5.0	≤5.0	≤5.0
<b>Loose Rod Configuration</b>								
Freely Loose or Strapped Together	#	≤25	≤25	≤25	≤25	N/A	N/A	N/A
Packed in 5" SS Pipe or Protective Case	#	≤22	≤26	≤30	≤30	N/A	N/A	N/A

a. Density based on a pellet modeled as a right cylinder.

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### 6.1.1. *Design Features*

#### 6.1.1.1. **Packaging**

A general discussion of the TN-B1 container design is provided in Section 1.2, Package Description. A detailed set of licensing drawings for the TN-B1 container is provided in Appendix 1.4.1 TN-B1 General Arrangement Drawings. Components important to criticality safety are described below.

The TN-B1 is comprised of two primary components: 1) an inner stainless steel container, and 2) an outer stainless steel container.


The inner stainless steel container is 468.6 cm (184.49 in) in length, 45.9 cm (18.07 in) in width, and 28.6 cm (11.26 in) in height, and provides containment for the uranium inside the cylindrical zirconium alloy tubes. The fuel rods are located inside one of two compartments within the inner container. The compartments are fabricated from 18-gauge (0.122 cm thick) stainless steel, 456.7 cm (179.8 in) in length, 17.6 cm (6.93 in) in width and height. Each compartment is lined with 1.8 cm (0.71 in) thick polyethylene foam and separated from each other by the compartment walls. A 5 cm (1.97 in) thick Alumina Silicate fiber surrounds the compartments to provide thermal insulation, and a 16-gauge (0.15 cm thick) stainless steel sheet surrounds the insulator. The inner container lid consists of an Alumina Silicate layer encased in a 16-gauge (0.15 cm thick) stainless steel sheet. The lid width and length are consistent with the inner container and the overall height is 5.25 cm (2.07 in).

The nominal density of the polyethylene foam is 4 pounds per cubic feet (pcf). Optionally, when transporting FANP 10x10 and 11x11 fuel assemblies, strips of 9 pcf foam may be used under the grid spacers to provide additional support to the fuel assemblies.

The outer container is 506.8 cm (199.53 in) in length, 72.0 cm (28.35 in) in width, and 64.2 cm (25.28 in) in height (with the skids attached the height is 74.2 cm (29.21 in)). The inner container is held rigidly within the outer stainless steel container by four evenly spaced stainless steel fixture assemblies. Shock absorbers, fabricated from a phenol impregnated cardboard material, are placed at six locations above and below the inner container, and twelve locations on either side of the inner container. The wall for the outer container is fabricated from 14 gauge (0.2 cm thick) stainless steel.

### 6.1.2. *Summary Table of Criticality Evaluation*

Table 6-3 Criticality Evaluation Summary, lists the bounding cases evaluated for a given set of conditions. The cases include: fuel assembly transport single package normal and Hypothetical Accident Conditions (HAC), fuel assembly transport package array normal conditions of transport, fuel assembly transport package array HAC, fuel rod transport single package normal


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and hypothetical accident conditions, fuel rod transport package array normal conditions of transport, and fuel rod transport package array HAC.

The criticality analysis for 8x8, 9x9, and 10x10 fuel assemblies (and the corresponding rods) is performed using the KENO V.a module of SCALE 4.4a. The SCALE 4.4a analysis comprises the main body of Chapter 6, as documented in Sections 6.3 through 6.11. The benchmarking analysis for the SCALE 4.4a analysis is documented in Section 6.10, Benchmark Evaluations. The USL for the SCALE 4.4a evaluation is 0.94254.


The criticality analysis for the 11x11 fuel assembly (and the corresponding 11x11 fuel rods) is performed using the KENO V.a module of SCALE 6.1.3. The 11x11 fuel assembly criticality analysis is documented in Appendix B (Section 6.12). The benchmarking analysis for the SCALE 6.1.3 analysis is documented in Section 6.12.9, Benchmark Evaluation for SCALE 6.1.3. A USL of 0.94094 is justified for the 11x11 fuel assembly analysis, and a USL of 0.94047 is justified for the 11x11 fuel rod analysis.

The benchmark USL results for SCALE 4.4a and SCALE 6.1.3 are quite similar, indicating that both programs are acceptable for TN-B1 criticality analysis.

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**Table 6-3 Criticality Evaluation Summary**

<b>Bounding Results for 8x8, 9x9, and 10x10 Fuel (SCALE 4.4a)</b>					
<b>Case</b>	<b>Bounding Fuel Type</b>	<b>k<sub>eff</sub></b>	<b>σ</b>	<b>k<sub>eff</sub> + 2σ</b>	<b>USL</b>
Fuel Assembly Single Package Normal	GNF 10x10 with worst case fuel parameters, 12, 2.0 wt % Gd <sub>2</sub> O <sub>3</sub> fuel rods, and 12 part length fuel rods	0.6673	0.0008	0.6689	0.94254
Fuel Assembly Single Package HAC	GNF 10x10 with worst case fuel parameters, 12, 2.0 wt % Gd <sub>2</sub> O <sub>3</sub> fuel rods, and 12 part length fuel rods	0.6931	0.0010	0.6951	0.94254
Fuel Assembly Package Array Normal	GNF 10x10 with worst case fuel parameters, 12, 2.0 wt % Gd <sub>2</sub> O <sub>3</sub> fuel rods, and 12 part length fuel rods	0.8519	0.0008	0.8535	0.94254
Fuel Assembly Package Array HAC	GNF 10x10 with worst case fuel parameters, 12, 2.0 wt % Gd <sub>2</sub> O <sub>3</sub> fuel rods, and 12 part length fuel rods	0.9378	0.0009	0.9396	0.94254
Fuel Rod Single Package Normal	25 GNF 8x8 fuel rods per container with worst case fuel parameters	0.6365	0.0008	0.6381	0.94254
Fuel Rod Single Package HAC	25 GNF 8x8 fuel rods per container with worst case fuel parameters	0.6532	0.0008	0.6548	0.94254
Fuel Rod Package Array Normal	25 GNF 8x8 fuel rods per container with worst case fuel parameters	0.6365	0.0008	0.6381	0.94254
Fuel Rod Package Array HAC	25 GNF 8x8 fuel rods per container with worst case fuel parameters	0.8731	0.0007	0.8745	0.94254


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**Table 6-3 Criticality Evaluation Summary (continued)**

<b>Bounding Results for 11x11 Fuel (SCALE 6.1.3)</b>					
<b>Case</b>	<b>Bounding Fuel Type</b>	<b>k<sub>eff</sub></b>	<b>σ</b>	<b>k<sub>eff</sub> + 2σ</b>	<b>USL</b>
Fuel Assembly Single Package NCT	11x11 with worst case fuel parameters, 13-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods	0.63082	0.00042	0.63166	0.94094
Fuel Assembly Single Package HAC	11x11 with worst case fuel parameters, 13-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods	0.76615	0.00045	0.76705	0.94094
Fuel Assembly Package Array NCT	11x11 with worst case fuel parameters, 13-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods	0.85303	0.00040	0.85383	0.94094
Fuel Assembly Package Array HAC	11x11 with worst case fuel parameters, 13/13/3-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods <sup>a</sup>	0.93982	0.00043	0.94068	0.94094
Fuel Rod Single Package NCT	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.59145	0.00045	0.59235	0.94047
Fuel Rod Single Package HAC	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.66316	0.00042	0.66400	0.94047
Fuel Rod Package Array NCT	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.59300	0.00042	0.59384	0.94047
Fuel Rod Package Array HAC	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.81947	0.00044	0.82035	0.94047

- a. This configuration contains 13-2.0 wt% Gd<sub>2</sub>O<sub>3</sub> fuel rods in the bottom and middle axial regions and 3-2.0 wt% Gd<sub>2</sub>O<sub>3</sub> fuel rods in the top axial region.
- b. This configuration bounds the 25 loose fuel rod configuration.

A comparison between the nominal fuel parameters and the worst case fuel parameters used in the criticality evaluation is shown in Table 6-4 Nominal vs. Worst Case Fuel Parameters for the TN-B1 Criticality Analysis.


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**Table 6-4 Nominal vs. Worst Case Fuel Parameters for the TN-B1 Criticality Analysis**

Case	Fuel Rod Pitch (cm)	Clad Outer Diameter (cm)	Clad Inner Diameter (cm)	Pellet Outer Diameter (cm)	Pellet Density (g/cm <sup>3</sup> )
<b>11x11</b>					
Nominal	1.195 <sup>a</sup>	Reference 20	Reference 20	Reference 20	< 10.74
Worst Case Modeled for Fuel Assembly Transport	1.2548	0.930	0.840	0.820	10.763
Worst Case Modeled for Fuel Rod Transport	3.52	0.930	0.930	0.820	10.763
<b>FANP 10x10</b>					
Nominal	1.284, 1.2954	1.010, 1.033	0.9020, 0.9217	0.8682, 0.8882	< 10.74
Worst Case Modeled for Fuel Assembly Transport	1.350	1.000	0.9330	0.895	10.74
Worst Case Modeled for Fuel Rod Transport	1.350	1.000	1.000	0.900	10.74
<b>GNF 10x10</b>					
Nominal	1.2954	1.019	0.9322	0.8941	< 10.74
Worst Case Modeled for Fuel Assembly Transport	1.350	1.010	0.9338	0.895	10.74
Worst Case Modeled for Fuel Rod Transport	1.350	1.000	1.000	0.900	10.74
<b>FANP 9x9</b>					
Nominal	1.4478	1.095, 1.0998	0.968, 0.9601	0.94, 0.9398	< 10.74
Worst Case Modeled for Fuel Assembly Transport	1.510	1.093	1.020	0.960	10.74
Worst Case Modeled for Fuel Rod Transport	1.510	1.020	1.020	0.960	10.74
<b>GNF 9x9</b>					
Nominal	1.438	1.110	0.983	0.955	< 10.74
Worst Case Modeled for Fuel Assembly Transport	1.510	1.093	1.020	0.960	10.74
Worst Case Modeled for Fuel Rod Transport	1.510	1.020	1.020	0.960	10.74
<b>GNF 8x8</b>					
Nominal	1.6256	1.2192	1.072	1.044	< 10.74
Worst Case Modeled for Fuel Assembly Transport	1.6923	1.176	1.100	1.050	10.74
Worst Case Modeled for Fuel Rod Transport	1.6923	1.100	1.100	1.050	10.74

a. Equivalent nominal pitch per Section 6.12.3.1.1.



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### 6.1.3. *Criticality Safety Index*

#### 8x8, 9x9, 10x10, and 11x11 fuel assemblies; BWR uranium oxide fuel rods

For the TN-B1, undamaged packages have been analyzed in 21x3x24 arrays and damaged packages have been analyzed in 10x1x10 arrays. Pursuant to 10 CFR 71.59, the number of packages “N” in a 2N array that are subjected to the tests specified in 10 CFR 71.73, or in a 5N array for undamaged packages is used to determine the Criticality Safety Index (CSI). The CSI is determined by dividing the number 50 by the most limiting value of “N” as specified in 10 CFR 71.59.

The TN-B1 criticality analysis demonstrates safety for 5N=1,512 (undamaged) and 2N=100 (damaged) packages. The corresponding Criticality Safety Index (CSI) for criticality control is given by  $CSI = 50/N$ . Since 5N=1,512 and 2N = 100, it follows that the more restrictive N = 50 and  $CSI = 50/50 = 1.0$ . Therefore the maximum allowable number of packages per shipment is  $50/1.0 = 50$ .

#### Uranium carbide and generic PWR uranium oxide fuel rods

Under hypothetical accident conditions, the contents of 2N=64 (8x1x8 array), 48 (4x1x6 array) TN-B1 damaged packages are demonstrated to remain subcritical. Therefore, the CSI for criticality control purposes is 1.6 for an 8x1x8 array and 2.1 for a 4x2x6 array (Ref. 13).

## 6.2. FISSILE MATERIAL CONTENTS

The TN-B1 shall be used to transport UO<sub>2</sub> conforming to the requirements stated in Section 6.1, Table 6-1 and Table 6-3. The uranium isotopic distribution considered in the models used for the criticality safety demonstration is shown in Table 6-5 Uranium Isotopic Distribution.

**Table 6-5 Uranium Isotopic Distribution**

Isotope	Modeled wt. %
U-235	5.00
U-238	95.00

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The criticality analysis conservatively demonstrates safety for UO<sub>2</sub> pellets within cylindrical zirconium alloy tubes, arranged in 8x8, 9x9, 10x10, or 11x11 square assembly lattices. Cylindrical fuel rods containing UO<sub>2</sub>, enriched up to 5 wt. percent U-235, are also conservatively demonstrated safe within the TN-B1 container in a 5-inch stainless steel pipe, loose, in a protective case, or bundled together. The fuel loadings demonstrated safe in the TN-B1 are specified in Table 6-1 and Table 6-3.

### 6.3. **GENERAL CONSIDERATIONS**

Models are generated for single package and package arrays under normal conditions and Hypothetical Accident Conditions (HAC).

#### 6.3.1. ***Model Configuration***

##### 6.3.1.1. **TN-B1 Shipping Container Single Package Model**

The TN-B1 single package models are constructed for both normal conditions of transport and hypothetical accident conditions. The single package models are enveloped with a 30.48 cm layer of full density water for reflection.

##### 6.3.1.1.1. ***Single Package Normal Conditions of Transport Model***

The TN-B1 is comprised of an inner and outer container fabricated from Stainless Steel. The inner container dimensions are shown in Figure 6-4 TN-B1 Inner Container Normal Conditions of Transport Model and Figure 6-5 TN-B1 Container Cross-Section Normal Conditions of Transport Model. It is lined with polyethylene foam having a density of up to 0.080 g/cm<sup>3</sup>. The fuel assemblies rest against the polyethylene foam in a fixed position, and the inner container is positioned within the outer container as shown in Figure 6-5. The inner container has Alumina Silicate thermal insulation between the inner and outer walls. The Alumina Silicate density is approximately 0.25 g/cm<sup>3</sup>. The outer container dimensions are contained in Figure 6-3 and Figure 6-5. The outer container provides protection for the inner container and additional separation between fuel assemblies in adjacent containers. No credit is taken for any of the structural steel between the inner and outer containers. The honeycomb shock absorbers, located between the inner and outer containers, are not explicitly modeled. Instead, water is placed in the space between the inner and outer containers, and its density is varied from 0.0 – 1.0 g/cm<sup>3</sup>. The honeycomb shock absorbers have a density between 0.04 and 0.08 g/cm<sup>3</sup>. The hydrogen number densities for water (1.0 g/cm<sup>3</sup>) and for the honeycomb shock absorber (0.08 g/cm<sup>3</sup>) are 6.677x10<sup>-2</sup> and 2.973x10<sup>-3</sup> atoms/b\*cm, respectively. As a result, water is more effective at thermalizing neutrons than the honeycomb shock absorbers.


Therefore, the use of water at 1.0 g/cm<sup>3</sup> between the inner and outer containers is considered a conservative replacement for the honeycomb shock absorbers.

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The fuel assemblies are modeled inside the inner container, flush with the polyethylene foam. No fuel assembly structures outside the active length of the rod are represented in the models, with the exception of the fuel assembly channel. The fuel assembly structures outside the active fuel length, other than the fuel assembly channel, are composed of materials that absorb neutrons by radiative capture, therefore, neglecting them is conservative. In addition, no grids within the rod active length are represented. The internal grid structure displaces water from between the fuel rods, decreasing the H/X ratio. Since the fuel assemblies are undermoderated, decreasing the H/X ratio decreases system reactivity. Therefore, it is conservative to neglect the internal grid structure in modeling the TN-B1 container. The maximum pellet enrichment and maximum fuel lattice average enrichment is 5.0 wt% U-235. Only 75% credit is taken for gadolinia present in the fuel rods.

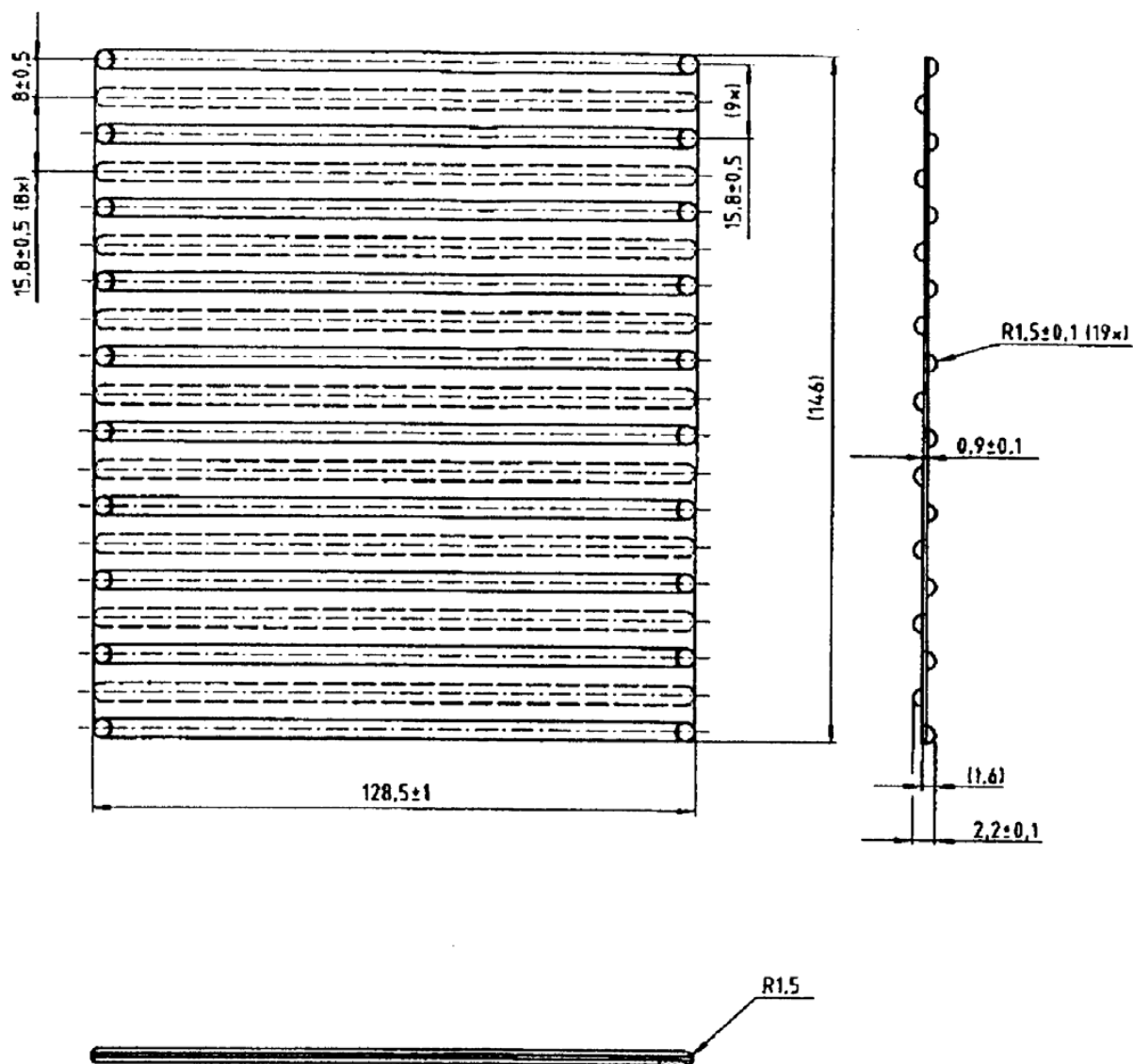
Calculations performed with the package array HAC model determine the fuel assembly modeling for the single package Normal Conditions of Transport (NCT) model. A fuel parameter sensitivity study is conducted and a worst case fuel assembly is developed for each fuel design. The sensitivity study results determine the fuel parameter ranges for the fuel assembly loading criteria shown in Table 6-1 and Table 6-2. The ranges are broad enough to accommodate future fuel assembly design changes. The fuel rod pitch, fuel pellet outer diameter, fuel rod clad inner and outer diameters, fuel rod number, and part length fuel rod number are varied independently in the package array HAC calculations. Reactivity effects are investigated, and the worst case is identified for each parameter perturbation. To validate the ranges for worst case fuel parameter combinations (e.g., worst case pellet OD, clad OD, clad ID, etc.) within the same assembly, a worst case fuel assembly is created for each fuel design considered for transport in the TN-B1 container, by choosing each parameter value that provides the highest system reactivity. Calculations performed with the worst case fuel assemblies validate the parameter ranges to be used as fuel acceptance criteria. Both un-channeled (Figure 6-9 through Figure 6-15) and channeled fuel assemblies, Figure 6-16, are considered in the worst case orientation, subjected to the worst case fuel damage, and the most reactive configuration is chosen for subsequent calculations.

The GNF 10x10 worst case fuel assembly is used for the TN-B1 single package NCT model since it is determined to be the most reactive assembly type in the package array HAC fuel parameter studies. The worst case fuel parameters for the GNF 10x10 assembly are presented in Table 6-11.


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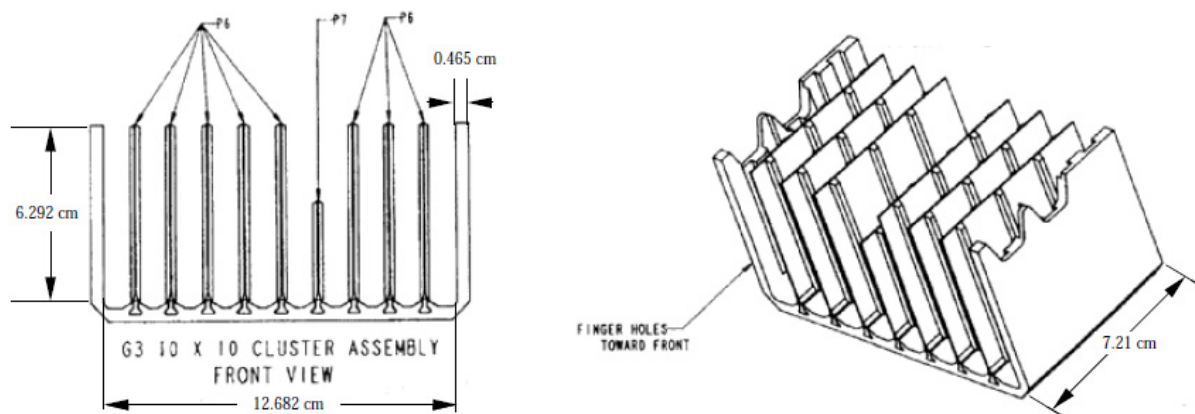
Polyethylene inserts or cluster separators are positioned between fuel rods at various locations along the axis of the fuel assembly to avoid stressing the axial grids during transportation. Two types of inserts, shown in Figure 6-1 and Figure 6-2, are considered for use with the TN-B1 container. Since the polyethylene cluster separators provide a higher volume average density polyethylene inventory, they are chosen for the TN-B1 criticality analysis. Other types of inserts are acceptable provided that their polyethylene inventory is within the limits established using the cluster separators.

The normal condition model utilizes the maximum allowable polyethylene mass and applies it over the full axial length of the fuel. The polyethylene is smeared into the water region surrounding the fuel rods as well as the water region surrounding the fuel assembly normally occupied by the cluster holder.

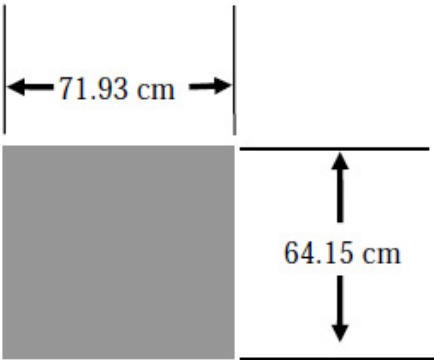
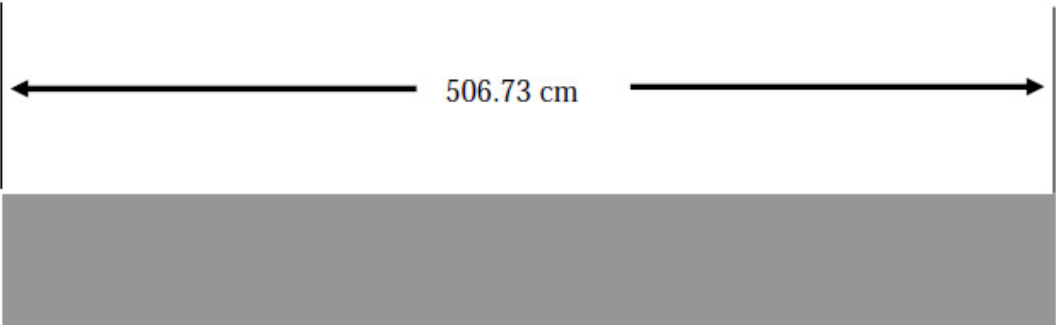


**Figure 6-1 Polyethylene Insert (FANP Design)**

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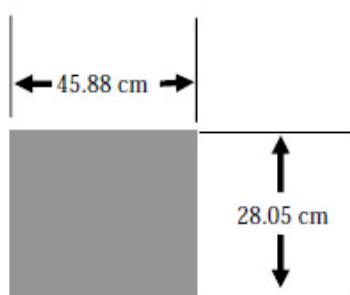
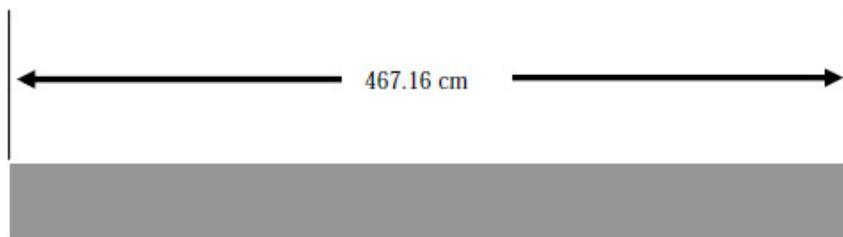
**Figure 6-2 Polyethylene Cluster Separator Assembly (GNF Design)**



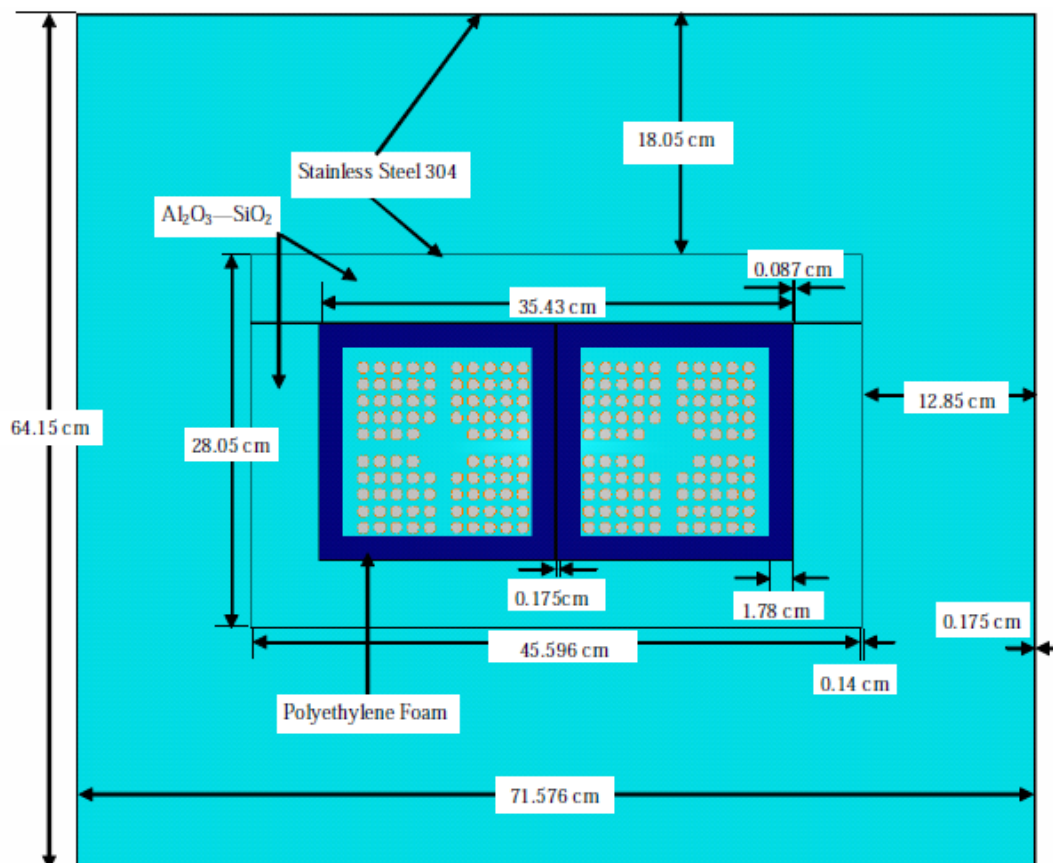
**Figure 6-3 TN-B1 Outer Container Normal Conditions of Transport Model**



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**Figure 6-4 TN-B1 Inner Container Normal Conditions of Transport Model**



**Figure 6-5 TN-B1 Container Cross-Section Normal Conditions of Transport Model**

#### 6.3.1.1.2. ***Single Package Hypothetical Accident Condition Model***

The TN-B1 HAC model inner container dimensions are shown in Figure 6-7 and Figure 6-8. The container deformation modeled for the TN-B1 HAC model includes the damage incurred from the 9-meter drop onto an unyielding surface as well as conservative factors. The TN-B1 inner container length is conservatively reduced by 8.1 cm to bound the damage incurred from the 9-meter drop onto an unyielding surface. The polyethylene foam is assumed to burn away for the HAC single package model. Full density water that provides more reflection capability is assumed to flood the TN-B1 inner container fuel compartment. The Alumina Silicate insulation is assumed to remain in place, since scoping calculations proved it to provide a more reactive configuration. The fuel assemblies are assumed to freely move within the respective compartment resulting in a worst case orientation. The rubber vibro-isolating devices are also assumed to melt when exposed to an external fire, allowing the inner container to shift downward about 2.54 cm. However, scoping calculations reveal no increase in reactivity by

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moving the inner container; therefore, the inner container is positioned within the outer container as shown in Figure 6-8. The inner container horizontal position within the outer container remains the same as the normal condition model, since the stainless steel fixture assemblies remained intact following the 9-meter drop. The outer container dimensions are shown in Figure 6-6 TN-B1 Outer Container Hypothetical Accident Condition Model and Figure 6-8. The outer container length is reduced by 4.7 cm to bound the damage sustained from a 9-meter drop onto an unyielding surface. In addition, the outer container height is reduced by 2.4 cm to bound the damage sustained during the 9-meter drop (Reference 1). No credit is taken for the structural steel between the inner and outer containers. The honeycomb shock absorbers, located between the inner and outer containers, are not explicitly modeled. Instead, water is placed in the space between the inner and outer containers, and its density is varied from 0.0 – 1.0 g/cm<sup>3</sup>. The honeycomb shock absorbers have a density between 0.04 and 0.08 g/cm<sup>3</sup>. The hydrogen number densities for water (1.0 g/cm<sup>3</sup>) and for the honeycomb shock absorber (0.08 g/cm<sup>3</sup>) are  $6.677 \times 10^{-2}$  and  $2.973 \times 10^{-3}$  atoms/b\*cm, respectively. As a result, water is more effective at thermalizing neutrons than the honeycomb shock absorbers. Therefore, the use of water at 1.0 g/cm<sup>3</sup> between the inner and outer containers is considered a conservative replacement for the honeycomb shock absorbers. The reduction in length for the inner and outer containers, the reduction in height for the outer container, the absence of polyethylene foam, the presence of the insulation, and the fuel assembly freedom of movement are consistent with the physical condition of the TN-B1 shipping container after being subjected to the tests specified in 10 CFR Part 71.

Calculations performed with the package array HAC model determine the fuel assembly modeling for the single package HAC model. No fuel assembly structures outside the active length of the rod are represented in the models, with the exception of the fuel assembly channel. The fuel assembly structures outside the active fuel length, other than the fuel assembly channel, are composed of materials that absorb neutrons by radiative capture, therefore, neglecting them is conservative. In addition, no grids within the rod active length are represented. The internal grid structure displaces water from between the fuel rods, decreasing the H/X ratio. Since the fuel assemblies are undermoderated, decreasing the H/X ratio decreases system reactivity. Therefore, it is conservative to neglect the internal grid structure in modeling the TN-B1 container. The maximum pellet enrichment and maximum fuel lattice average enrichment is 5.0 wt% U-235. The gadolinia content of any gadolinia-urania fuel rods is taken to be 75% of the minimum value specified in Table 6-1. The fuel assemblies are modeled inside the inner container, in one of seven orientations shown in Figure 6-9 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 1 through Figure 6-15 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 7. The worst case orientation is chosen for each fuel assembly design considered for transport and used in subsequent calculations. Fuel damage sustained during the 9-meter (30 foot) drop test is

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simulated as a change in fuel rod pitch along the full axial length of each fuel assembly considered for transport. Based on the fuel damage sustained in the TN-B1 shipping container drop test (Reference 1), a 10% reduction in fuel rod pitch over the full length of each fuel assembly, or a 4.1% increase in fuel rod pitch over the full length of each fuel assembly, is determined to be conservative. Both un-channeled (Figure 6-9 through Figure 6-15) and channeled fuel assemblies (Figure 6-16) are considered in the worst case orientation, subjected to the worst case fuel damage, and the most reactive configuration is chosen for subsequent calculations.

The fuel damage sustained during the 9-meter drop test is bounded by performing a fuel parameter sensitivity study and creating a worst case fuel assembly for each fuel design. The sensitivity study results determine the fuel parameter ranges for the fuel assembly loading criteria shown in Table 6-1. The ranges are broad enough to accommodate future fuel assembly design changes. The fuel rod pitch, fuel pellet outer diameter, fuel rod clad inner and outer diameters, fuel rod number, and part length fuel rod number are varied independently in the package array HAC calculations. Reactivity effects are investigated, and the worst case is identified for each parameter perturbation. To validate the ranges for worst case fuel parameter combinations (e.g. worst case pellet OD, clad OD, clad ID, etc.) within the same assembly, a worst case fuel assembly is created for each fuel design considered for transport in the TN-B1 container, by choosing each parameter value that provides the highest system reactivity. Calculations performed with the worst case fuel assemblies validate the parameter ranges to be used as fuel acceptance criteria.

The GNF 10x10 worst case fuel assembly at a 5.0 wt% U-235 enrichment, containing twelve 2 wt % gadolinia-urania fuel rods, and twelve part length fuel rods is used for the TN-B1 single package HAC model since it is determined to be the most reactive assembly in the package array HAC fuel parameter studies. The worst case fuel parameters for the 10x10 assembly are presented in Table 6-11.

Polyethylene inserts (cluster separators) are positioned between fuel rods at various locations along the axis of the fuel assembly to avoid stressing the axial grids during transportation. Two types of inserts, shown in Figure 6-1 and Figure 6-2, are considered for use with the TN-B1 container. Since the polyethylene cluster separators provide a higher volume averaged density polyethylene inventory, they are chosen for the TN-B1 criticality analysis. Other types of inserts are acceptable provided that their polyethylene inventory is within the limits established using the cluster separators.

In the hypothetical accident condition model, the polyethylene inserts are assumed to melt when subjected to the tests specified in 10 CFR Part 71. The polyethylene is assumed to uniformly coat the fuel rods in each fuel assembly forming a cylindrical layer of polyethylene around each fuel rod. Different coating thicknesses are investigated in the package array HAC calculations,

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and a polyethylene mass limit is developed for each fuel assembly type considered for transport. The TN-B1 single package model contains 10x10 worst case fuel assemblies with 10.2 kg of polyethylene per assembly. The polyethylene is smeared into the fuel rod cladding to accommodate the limitations in the lattice cell modeling for cross-section processing in SCALE. A visual representation of the smeared clad/polyethylene mixture compared to a discrete treatment is shown in Figure 6-21 Visual Representation of the Clad/Polyethylene Smeared Mixture versus Discrete Modeling. The polyethylene mass and the volume fractions of polyethylene and zirconium clad for each fuel assembly analyzed are shown in Table 6-13 Polyethylene Mass and Volume Fraction Calculations. The volume fractions in Table 6-13 are entered into the model input standard composition specification area. Mixtures representing the polyethylene inserts between fuel rods are created using the compositions specified, and used in the KENO V.a calculation. The mixtures are also used in the lattice cell description to provide the lump shape and dimensions for resonance cross-section processing, the lattice corrections for cross-section processing, and the information necessary to create flux-weighted cross-sections based on the lattice cell geometry.

#### 6.3.1.2. **Package Array Models**

##### 6.3.1.2.1. ***Package Array Normal Condition Model***

The TN-B1 container package array normal condition model consists of a 21x3x24 array of containers, surrounded by a 30.48 cm layer of full density water for reflection. The container array is fully flooded with water at a density sufficient for optimum moderation. The container and fuel model in the array are those discussed in Section 6.3.1.1.1.

##### 6.3.1.2.2. ***Package Array Hypothetical Accident Condition (HAC) Model***

The TN-B1 package array HAC model consists of either a 14x2x16 or 10x1x10 array of containers, surrounded by a 30.48 cm layer of full density water for reflection. The 14x2x16 array (Sections 6.4.1 – 6.4.10) is initially used under the assumption that the polyethylene foam, on which the fuel assemblies rest, completely burns away during a fire. The 10x1x10 array (Sections 6.4.11 – 6.4.13) assumes the polyethylene foam remains intact following a fire. The container array has no interspersed water between packages in the array and no water in the outer container. These moderator conditions optimize the interaction between packages in the array. Unlike the HAC single package model, the HAC package array model assumes the polyethylene foam remains in place following the tests specified in 10 CFR 71. The presence of polyethylene foam allows increased neutron leakage from the inner container fuel compartment and promotes increased neutron interaction among containers in the array. The inner container fuel compartment space not occupied by the polyethylene foam is fully flooded with water at a density sufficient for optimum moderation. The remaining HAC model container and fuel details are those discussed in Section 6.3.1.1.2.

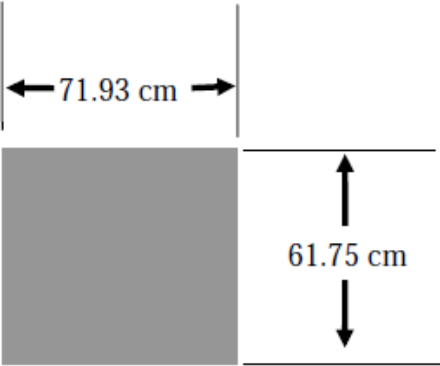
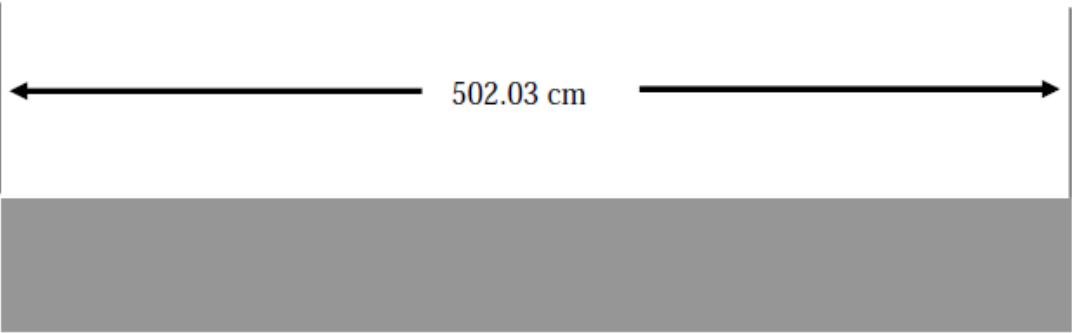
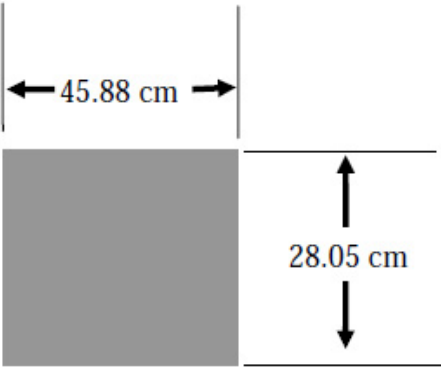
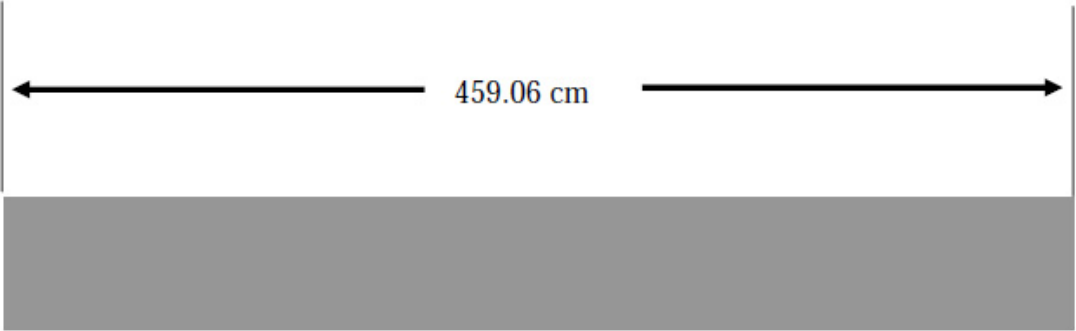
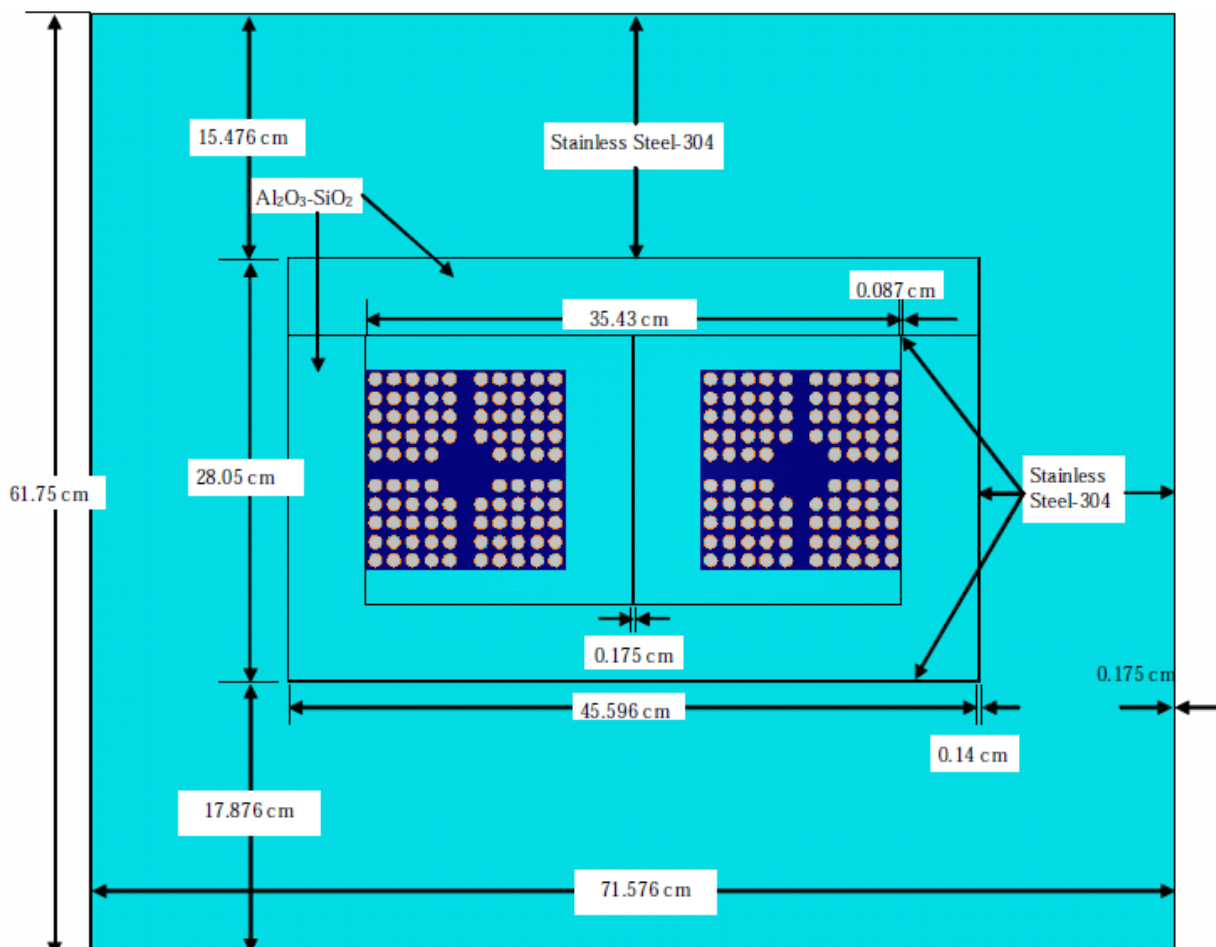


Figure 6-6 TN-B1 Outer Container Hypothetical Accident Condition Model

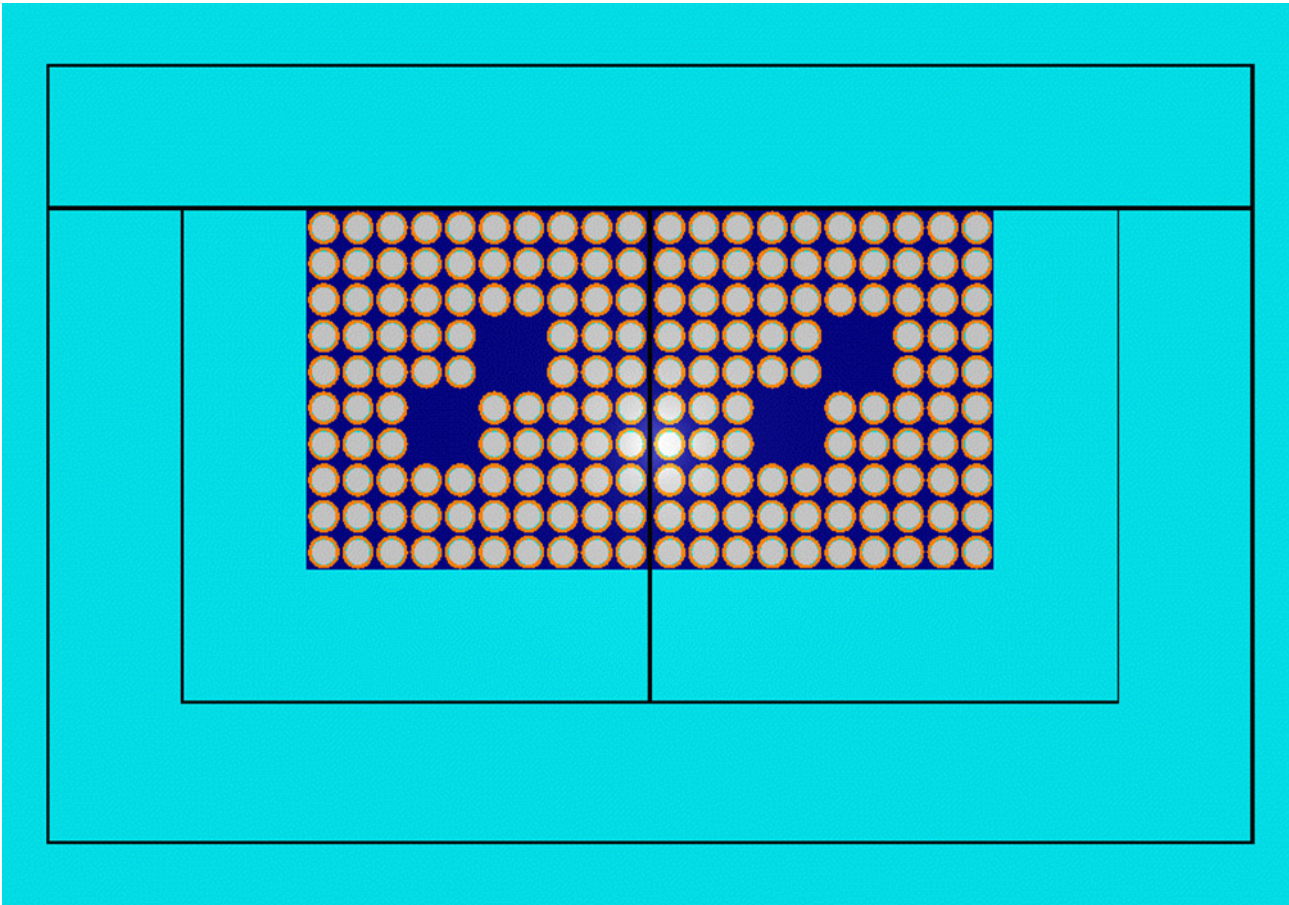


**Figure 6-7 TN-B1 Inner Container Hypothetical Accident Condition Model**

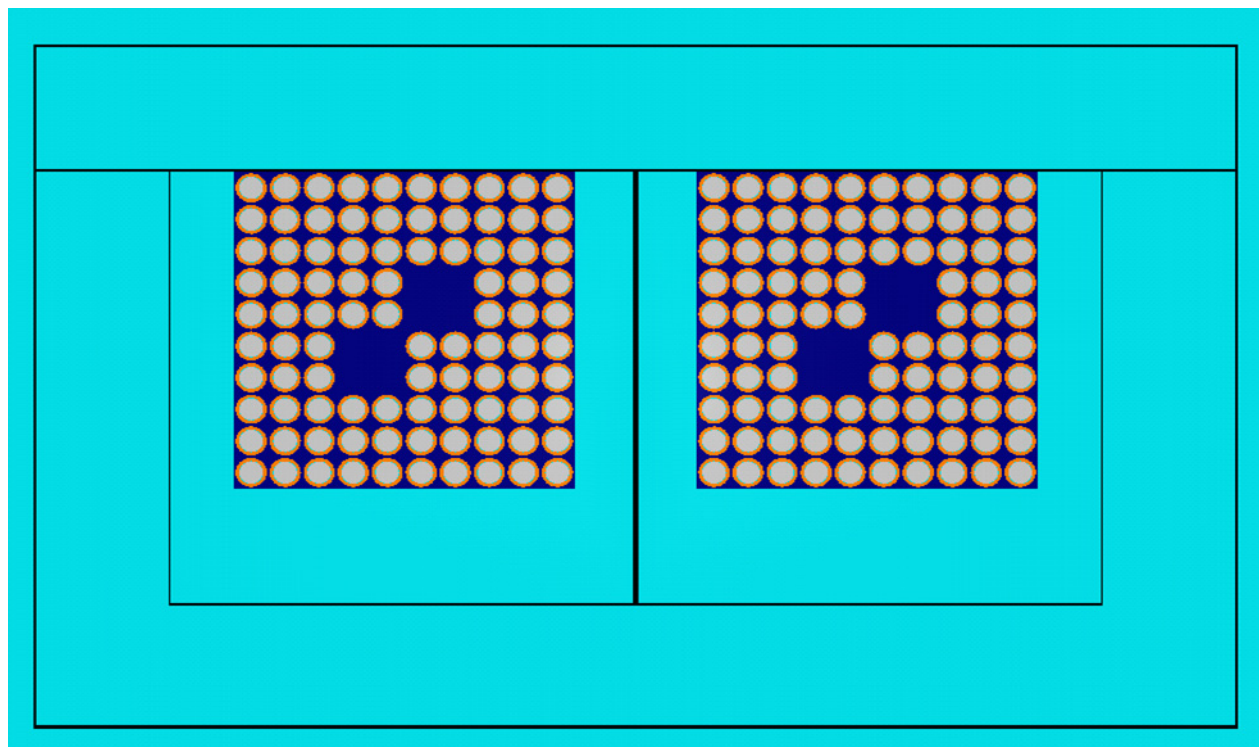




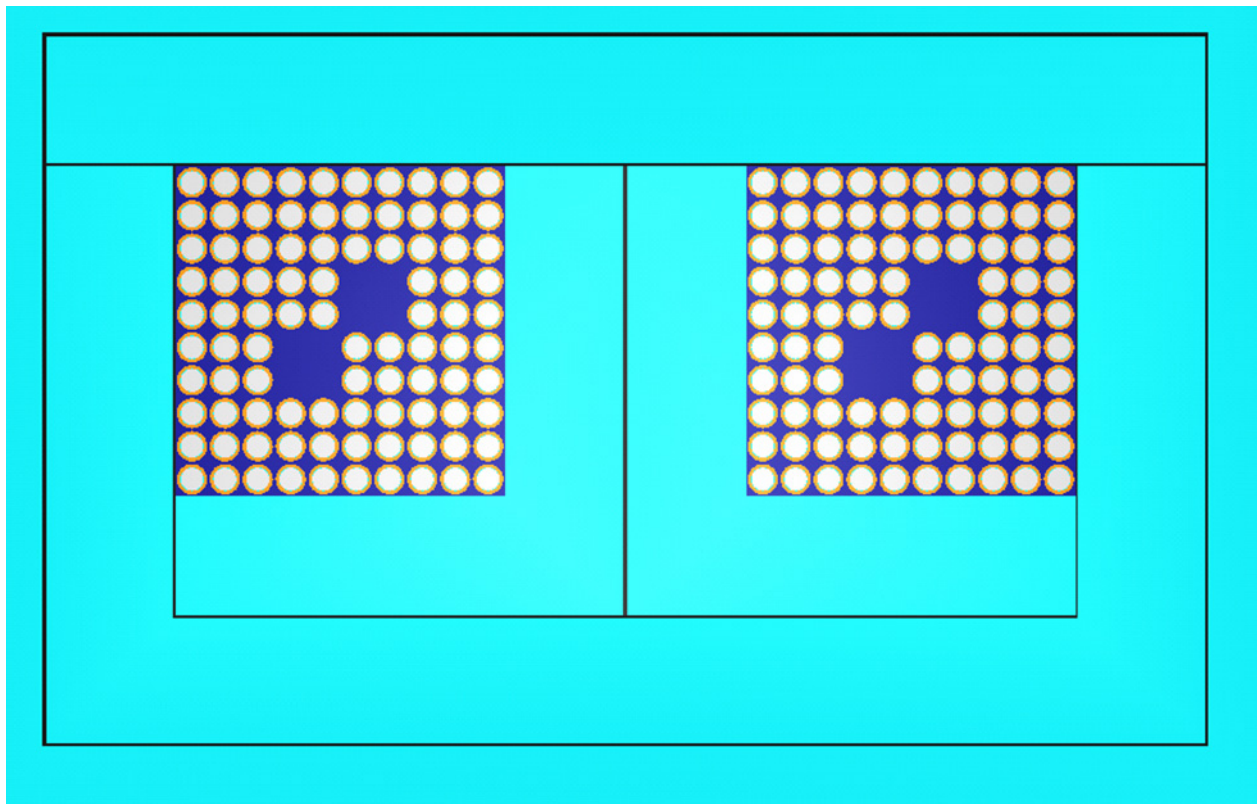
**Figure 6-8 TN-B1 Cross-Section Hypothetical Accident Condition Model**



**Figure 6-9 TN-B1 Hypothetical Accident Condition Model with FuelAssembly Orientation 1**

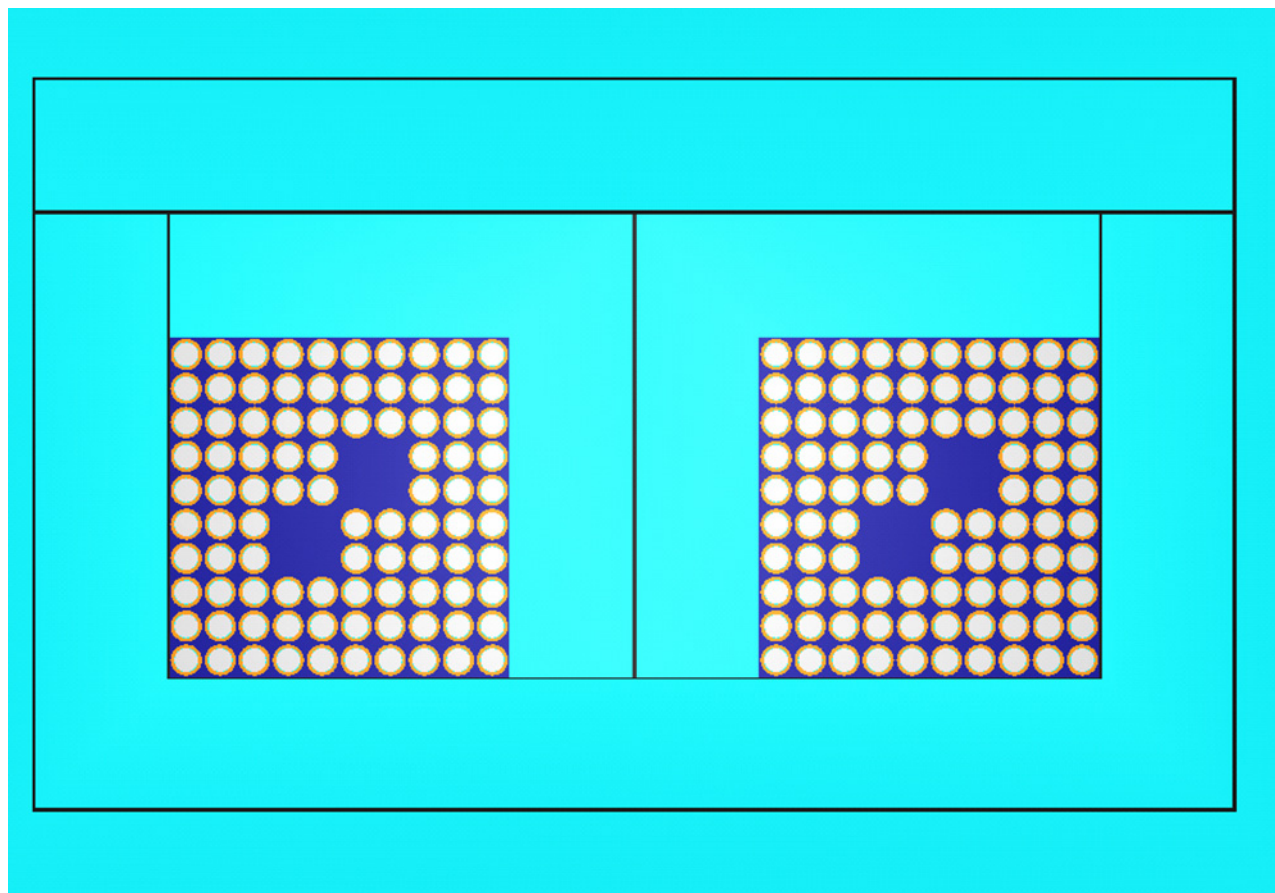


**Figure 6-10 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 2**

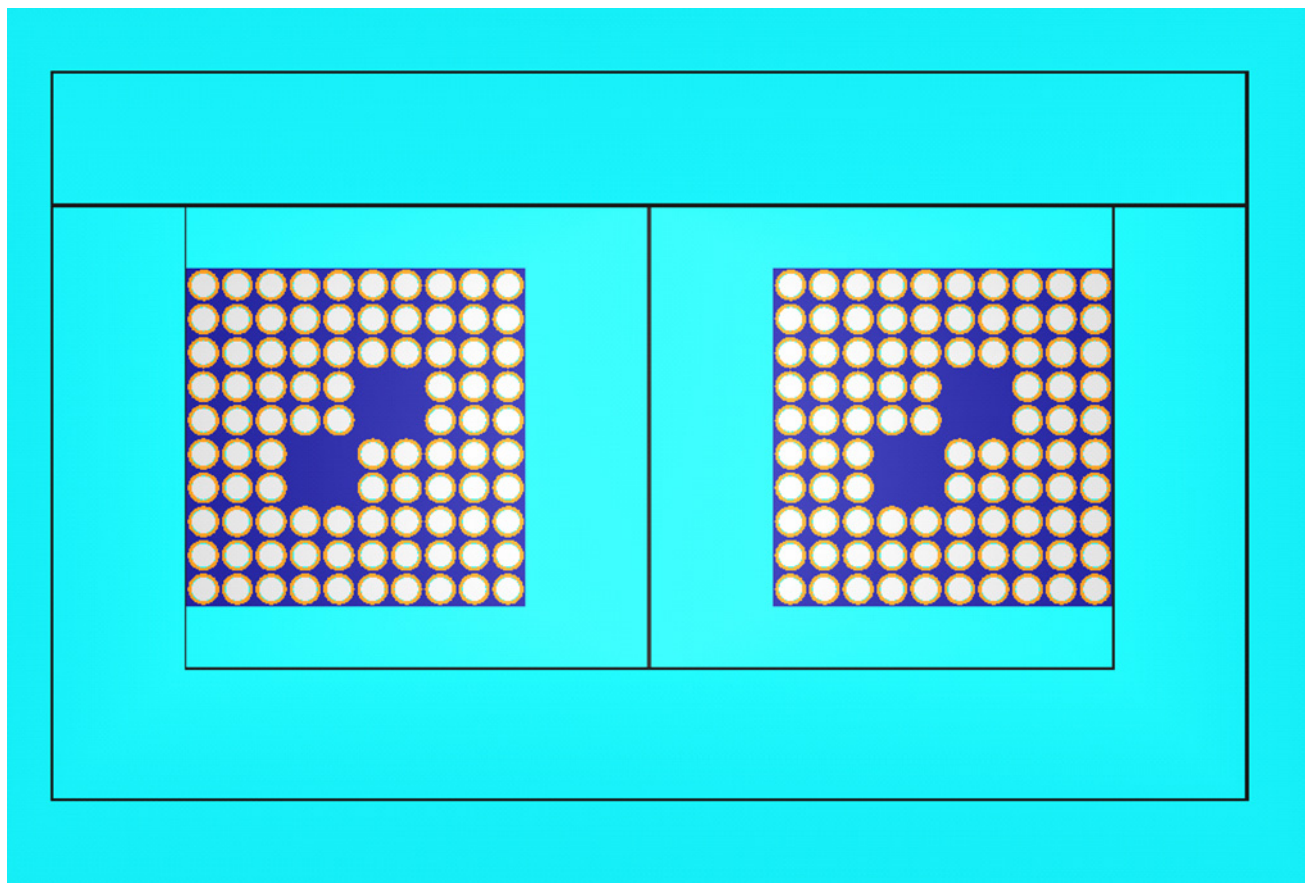


**Figure 6-11 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 3**

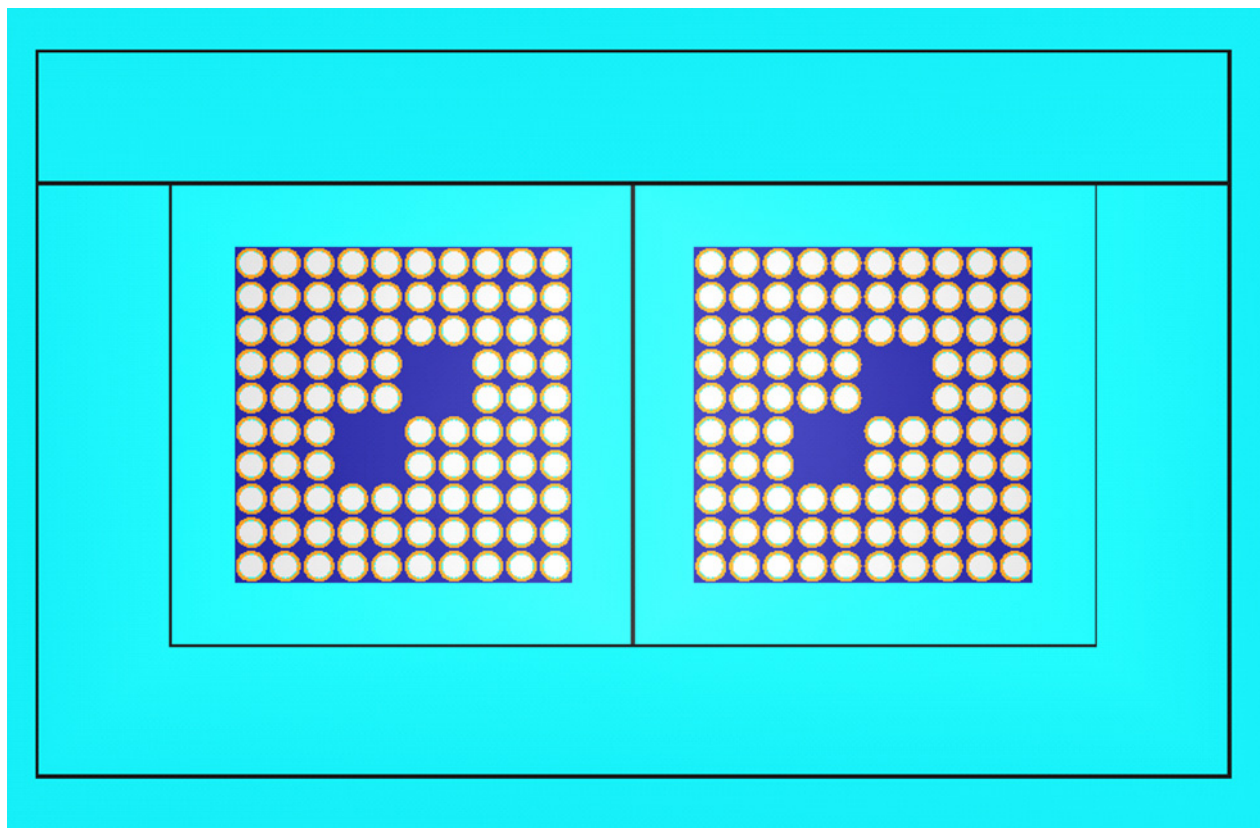




**Figure 6-12 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 4**



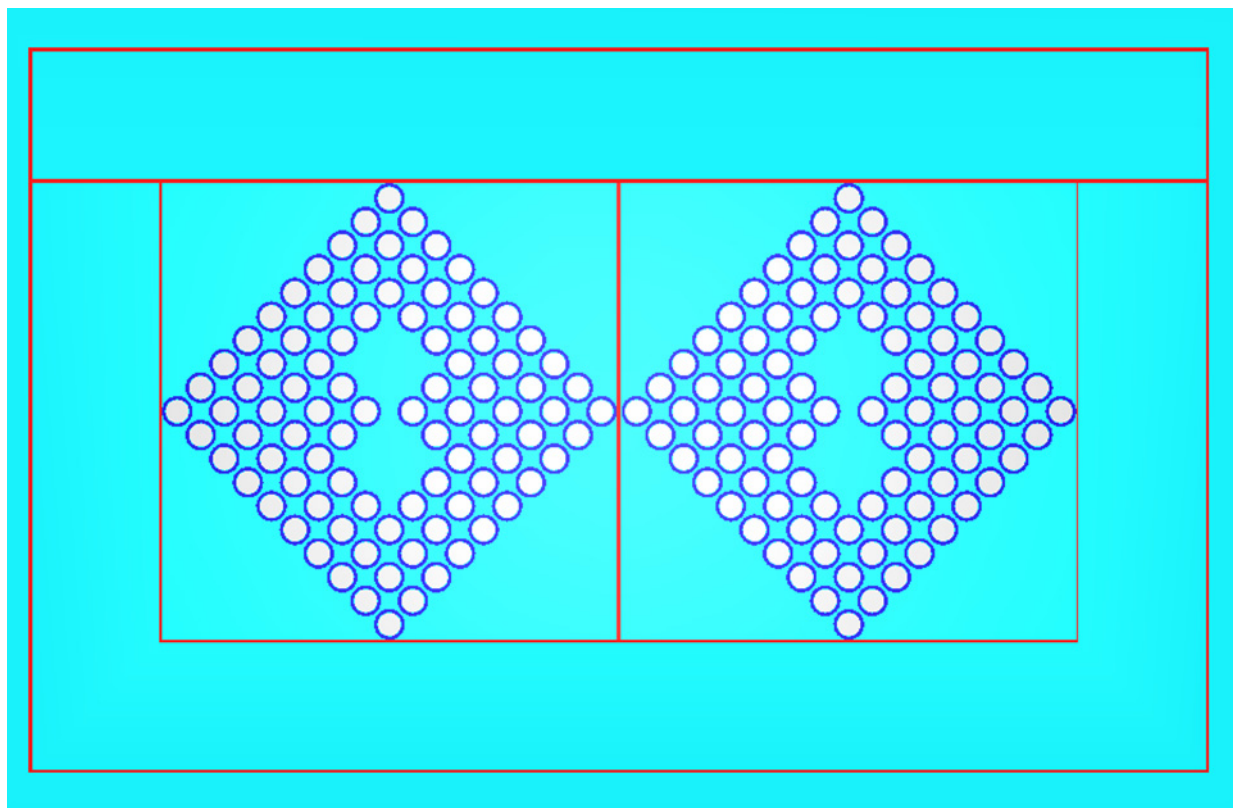
**Figure 6-13 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 5**



**Figure 6-14 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 6**

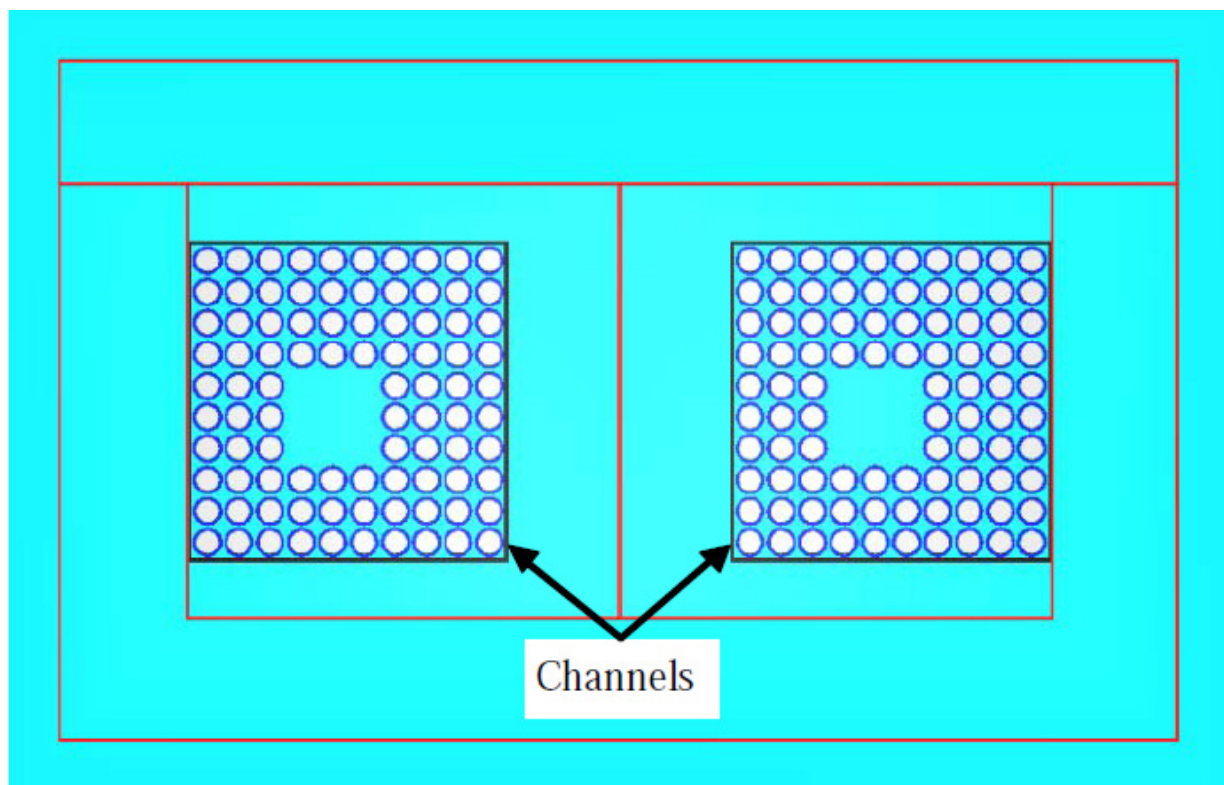


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**Figure 6-15 TN-B1 Hypothetical Accident Condition Model with Fuel Assembly Orientation 7**

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**Figure 6-16 TN-B1 Hypothetical Accident Condition Model with Channels**

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### 6.3.1.3. TN-B1 Fuel Rod Transport Model

The TN-B1 fuel rod transport models are developed for single packages and package arrays under normal transport and hypothetical accident conditions. Cylindrical fuel rods containing UO<sub>2</sub>, enriched to 5 wt. percent U-235, are modeled loose, bundled together, or in the TN-B1 inner container in 5-inch stainless steel pipe or protective case.

#### 6.3.1.3.1. *TN-B1 Single Package Fuel Rod Transport NCT Model*


The TN-B1 single package normal conditions of transport described in Section 6.3.1.1.1 are used for the single package fuel rod transport models.

The fuel rods are modeled inside the inner container, flush with the polyethylene foam. A 0.0152 cm thick polyethylene layer is modeled around each fuel rod to simulate any protective material present. The worst case fuel rod parameters are shown in Table 6-6 TN-B1 Fuel Rod Transport Model Fuel Parameters.

**Table 6-6 TN-B1 Fuel Rod Transport Model Fuel Parameters**

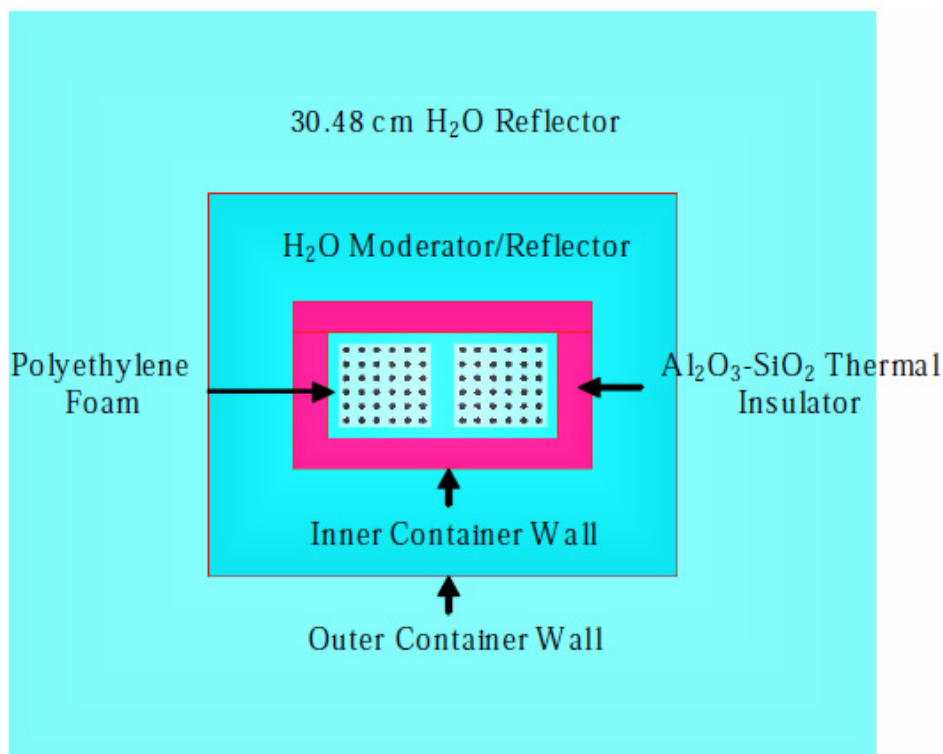
<b>Fuel Rod Type</b>	<b>Pellet OD (cm)</b>	<b>Fuel Rod ID (cm)</b>	<b>Fuel Rod OD (cm)</b>	<b>Fuel Rod Length (cm)</b>
10x10	0.9	1.000	1.000	385
9 x 9	0.9600	1.0200	1.0200	381
8 x 8	1.05	1.1000	1.1000	381

Calculations performed with the fuel rod transport, package array, HAC model determine the fuel assembly modeling for the fuel rod transport, single package, Normal Conditions of Transport (NCT) model. The calculations investigate transporting loose fuel rods, bundled fuel rods, and fuel rods in 5-inch stainless steel pipe within each TN-B1 shipping compartment. A fuel rod pitch sensitivity study is conducted for each fuel rod type to determine the number of fuel rods that can be transported in a loose configuration within the TN-B1 fuel assembly compartment. A square pitch fuel rod array is used for the sensitivity study since scoping calculations showed no statistically significant difference in system reactivity between fuel rods in a square pitch array and those in a triangular pitch array within the container geometry. The pitch sensitivity study results in the minimum and maximum allowable fuel rod quantity for shipping in a loose configuration. The loose rod analysis is used to bound a fuel rod shipment in which fuel rods are strapped or bundled together. A fuel rod pitch sensitivity analysis is also performed to determine the fuel rod quantity that may be transported inside a 5-inch stainless steel pipe. A triangular pitch fuel rod array is used for the sensitivity study since scoping calculations showed it to result in a higher system reactivity than a square pitch rod array inside a 5-inch stainless

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steel pipe. The stainless steel material is conservatively neglected when performing the calculations, therefore, any container with a volume equivalent to or less than the 5-inch stainless steel pipe is acceptable for fuel rod transport, as long as the fuel rod quantity is limited to that for the pipe.

The 8x8 worst case fuel rod is used for the TN-B1 fuel rod transport, single package, NCT model since it is determined to be the most reactive rod in the fuel rod transport, package array, HAC pitch sensitivity studies. The TN-B1 fuel rod transport, single package NCT model is shown in Figure 6-17 TN-B1 Fuel Rod Transport Single Package NCT Model. The worst case fuel parameters for the 8x8 rod are presented in Table 6-6. As shown in Table 6-6, the fuel rod cladding is not modeled for the 8x8 fuel rod. Although the cladding material is removed, the fuel rod external boundary is maintained (i.e. pellet clad gap to fuel rod OD is maintained, polyethylene coating applied to fuel rod OD region).



**Figure 6-17 TN-B1 Fuel Rod Transport Single Package NCT Model**

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#### 6.3.1.3.2. ***TN-B1 Single Package Fuel Rod Transport HAC Model***

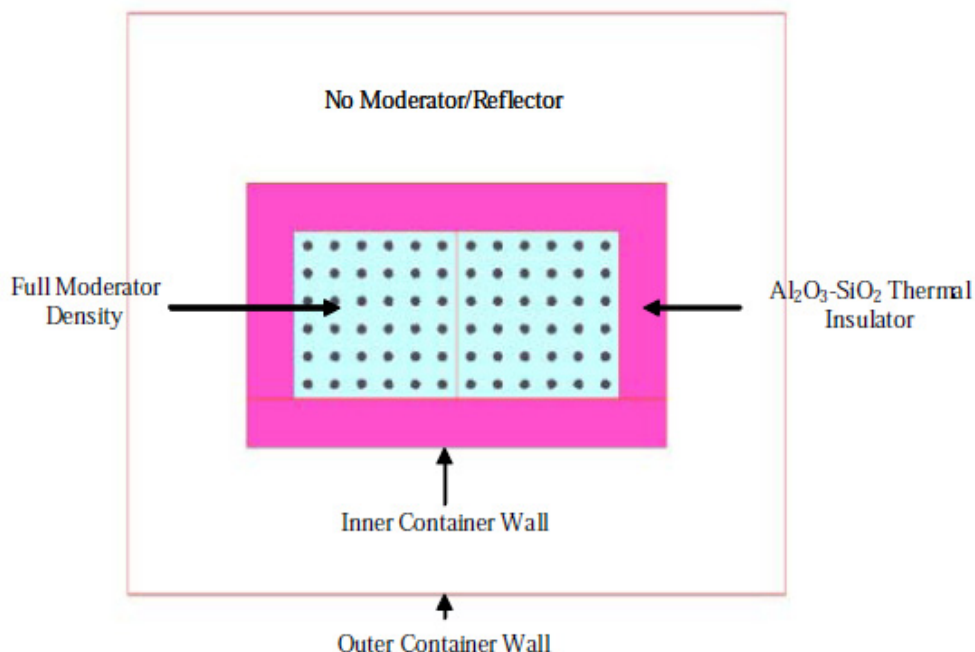
The TN-B1 single package hypothetical accident conditions described in Section 6.3.1.1.2 are used for the single package fuel rod transport models.

The fuel rods are modeled as filling the inner container fuel assembly compartment, since the polyethylene foam is removed due to the HAC. A 0.0152 cm thick polyethylene layer is modeled around each fuel rod to simulate any protective material present. Worst case fuel rod parameters determined from the package array HAC parameter sensitivity analyses (Section 6.3.1.1.2), are used for the fuel rod transport models. The worst case fuel rod parameters are shown in Table 6-6 TN-B1 Fuel Rod Transport Model Fuel Parameters.

Calculations performed with the fuel rod transport, package array, HAC model determine the fuel assembly modeling for the fuel rod transport, single package, HAC model. The calculations investigate transporting loose fuel rods, bundled fuel rods, fuel rods in a 5-inch stainless steel pipe and protective case within each TN-B1 shipping compartment. A fuel rod pitch sensitivity study is conducted for each fuel rod type to determine the number of fuel rods that can be transported in a loose configuration within the TN-B1 fuel assembly compartment. A square pitch fuel rod array is used for the sensitivity study since scoping calculations showed no statistically significant difference in system reactivity between fuel rods in a square pitch array and those in a triangular pitch array within the container geometry. The pitch sensitivity study results in the minimum and maximum allowable fuel rod quantity for shipping in a loose configuration. The loose rod analysis is used to bound a fuel rod shipment in which fuel rods are strapped together. A fuel rod pitch sensitivity analysis is also performed to determine the fuel rod quantity that may be transported inside a 5-inch stainless steel, Type 304 pipe. A triangular pitch fuel rod array is used for the sensitivity study since scoping calculations showed it to result in a higher system reactivity than a square pitch rod array inside a 5-inch stainless steel pipe. The stainless steel material is conservatively neglected when performing the calculations, therefore, any container with a volume equivalent to or less than the 5-inch stainless steel pipe is acceptable for fuel rod transport, as long as the fuel rod quantity is limited to that for the pipe.

The 8x8 worst case fuel rod is used for the TN-B1 fuel rod transport, single package, HAC model since it is determined to be the most reactive rod in the fuel rod transport, package array, HAC pitch sensitivity studies. The TN-B1 fuel rod transport, single package HAC model is shown in Figure 6-18 TN-B1 Fuel Rod Transport Single Package HAC Model. The worst case fuel parameters for the 8x8 rod are presented in Table 6-6. As shown in Table 6-6, the fuel rod cladding is not modeled for the 8x8 fuel rod. Although the cladding material is removed, the fuel rod external boundary is maintained (i.e., pellet clad gap to fuel rod OD is maintained, polyethylene coating applied to fuel rod OD region).

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**Figure 6-18 TN-B1 Fuel Rod Transport Single Package HAC Model**


#### 6.3.1.3.3. ***TN-B1 Package Array Fuel Rod Transport NCT Model***

The TN-B1 package array normal conditions of transport described in Section 6.3.1.2.1 are used for the package array, normal conditions of transport, fuel rod transport models.

The fuel rods are modeled inside the inner container, flush with the polyethylene foam. A 0.0152 cm thick polyethylene layer is modeled around each fuel rod to simulate any protective material present. Worst case fuel rod parameters determined from the package array HAC parameter sensitivity analyses (Section 6.3.1.2.2), are used for the fuel rod transport models. The worst case fuel rod parameters are shown in Table 6-6.

Calculations performed with the fuel rod transport, package array, HAC model determine the fuel assembly modeling for the fuel rod transport, package array, Normal Conditions of Transport (NCT) model. The calculations investigate transporting loose fuel rods, bundled fuel rods, and fuel rods in 5-inch stainless steel pipe within each TN-B1 shipping compartment. A fuel rod pitch sensitivity study is conducted for each fuel rod type to determine the number of fuel rods that can be transported in a loose configuration within the TN-B1 fuel assembly compartment. A square pitch fuel rod array is used for the sensitivity study since scoping calculations showed no statistically significant difference in system reactivity between fuel rods in a square pitch array and those in a triangular pitch array within the container geometry. The pitch sensitivity study results in the minimum and maximum allowable fuel rod quantity for shipping in a loose

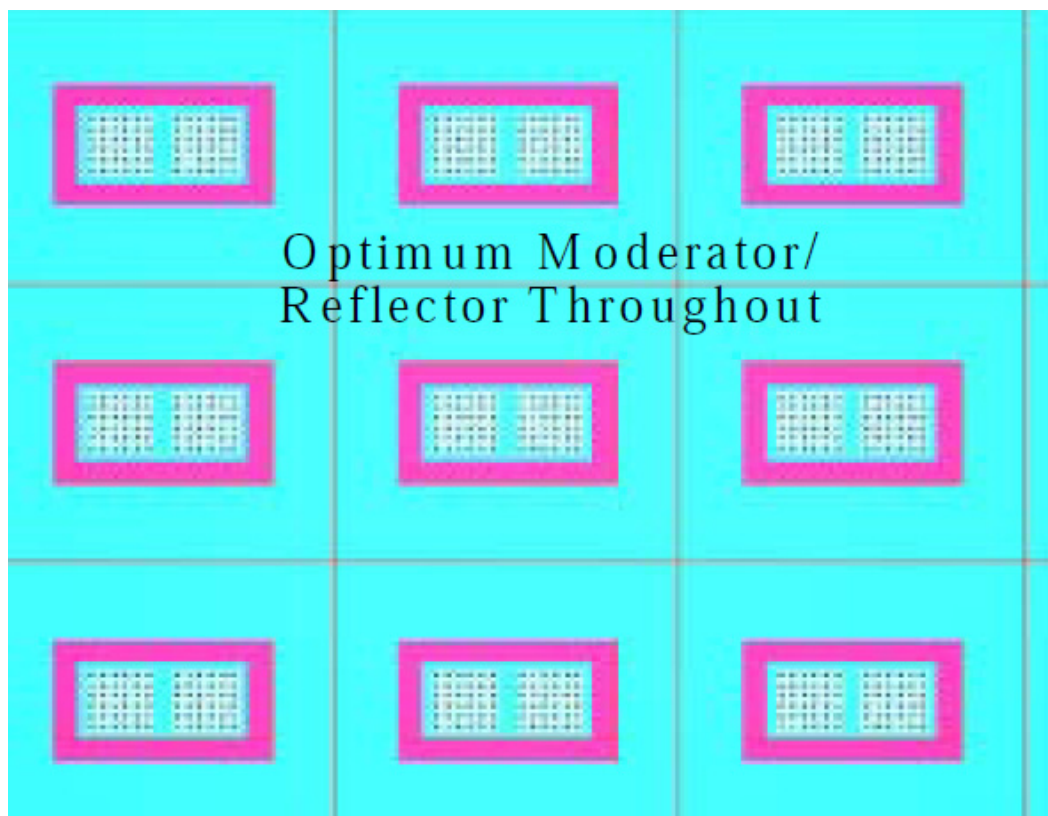


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configuration. The loose rod analysis is used to bound a fuel rod shipment in which fuel rods are strapped or bundled together.

A fuel rod pitch sensitivity analysis is also performed to determine the fuel rod quantity that may be transported inside a 5-inch stainless steel pipe. A triangular pitch fuel rod array is used for the sensitivity study since scoping calculations showed it to result in a higher system reactivity than a square pitch rod array inside a 5-inch stainless steel pipe. The stainless steel material is conservatively neglected when performing the calculations, therefore, any container with a volume equivalent to or less than the 5-inch stainless steel pipe is acceptable for fuel rod transport, as long as the fuel rod quantity is limited to that for the pipe.

The 8x8 worst case fuel rod is used for the TN-B1 fuel rod transport, package array, NCT model since it is determined to be the most reactive rod in the fuel rod transport, package array, HAC pitch sensitivity studies. A portion of the TN-B1 fuel rod transport, 21x3x24 package array, NCT model is shown in Figure 6-19. The worst case fuel parameters for the 8x8 rod are presented in Table 6-6. As shown in Table 6-6, the fuel rod cladding is not modeled for the 8x8 fuel rod. Although the cladding material is removed, the fuel rod external boundary is maintained (i.e., pellet clad gap to fuel rod OD is maintained, polyethylene coating applied to fuel rod OD region).



**Figure 6-19 TN-B1 Fuel Rod Transport Package Array NCT Model**



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#### 6.3.1.3.4. ***TN-B1 Package Array Fuel Rod Transport HAC Model***

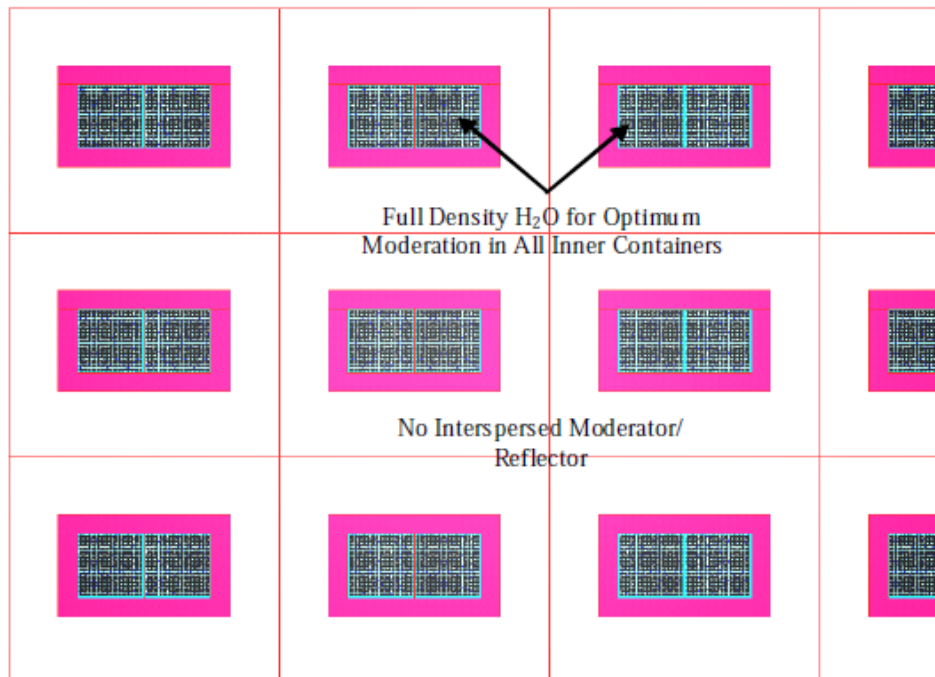
The TN-B1 package array hypothetical accident conditions described in Section 6.3.1.2.2 are used for the package array, HAC, fuel rod transport models.

The fuel rods are modeled filling the inner container for the hypothetical accident conditions. A 0.0152 cm thick polyethylene layer is modeled around each fuel rod to simulate any protective material present. Worst case fuel rod parameters determined from the package array HAC parameter sensitivity analyses (Section 6.3.1.2.2), are used for the fuel rod transport models. The worst case fuel rod parameters are shown in Table 6-6.

Calculations are conducted to investigate transporting loose fuel rods, bundled fuel rods, and fuel rods in 5-inch stainless steel pipe within each TN-B1 shipping compartment. A fuel rod pitch sensitivity study is conducted for each fuel rod type, to determine the number of fuel rods that can be transported in a loose configuration within the TN-B1 fuel assembly compartment. For convenience, a square pitch array is used to conduct the sensitivity study, since scoping calculations revealed little difference in the reactivity between square and triangular pitch arrays. The pitch sensitivity study results in the minimum and maximum allowable fuel rod quantity for shipping rods in a loose configuration. The loose rod analysis is used to bound a fuel rod shipment in which fuel rods are strapped or bundled together.

A fuel rod pitch sensitivity analysis is also performed to determine the fuel rod quantity that may be transported inside a 5-inch stainless steel pipe. Triangular pitch fuel rod arrays are used to find the maximum allowable quantity. The stainless steel material is conservatively neglected when performing the calculations, therefore, any container with a volume equivalent to or less than the 5-inch stainless steel pipe is acceptable for fuel rod transport, as long as the fuel rod quantity is limited to that for the pipe.

The fuel rod type with the most reactive configuration is chosen for the TN-B1 fuel rod transport, package array, HAC model. A portion of the TN-B1 fuel rod transport package array HAC model is shown in Figure 6-20.



**Figure 6-20 TN-B1 Fuel Rod Transport Package Array HAC Model**

## 6.3.2. *Material Properties*


### 6.3.2.1. **Material Tolerances**

Table 6-7 Dimensional Tolerances provides sheet metal thickness dimensional tolerance from ASTM A240 and ASTM A480 (the former refers to the latter for specific tolerances). The table also provides the thicknesses used in the damaged and undamaged container models.

**Table 6-7 Dimensional Tolerances**

<b>Stainless Steel Sheet Gauge</b>	<b>Nominal Thickness (mm)</b>	<b>Permissible Variations* (mm)</b>	<b>Model Thickness Used (in.) [cm] (description)</b>
2 mm.	2.00 mm	± 0.18	0.0689 [0.175] (outer container wall)
1.5 mm	1.50 mm	± 0.15	0.0535 [0.136] (inner container wall)
1.0 mm.	1.00 mm	± 0.13	0.0344 [0.0875] (inner container fuel assembly compartments)

\* ASTM-A240/A240M- 97b, Table A1.2, *Standard Specification for Heat Resisting Chromium and Chromium-Nickel Stainless Steel Plate, Sheet, and Strip for Pressure Vessels*, August 1997.

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#### 6.3.2.2. MATERIAL SPECIFICATIONS

Table 6-8 Material Specifications for the TN-B1 contains the material compositions for the TN-B1 shipping container. The  $\text{UO}_2$  stack density is taken as 98% of theoretical. The presence of  $\text{Gd}_2\text{O}_3$  in the  $\text{UO}_2\text{-Gd}_2\text{O}_3$  pellet reduces the density from 10.74 to 10.67  $\text{g/cm}^3$ .

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**Table 6-8 Material Specifications for the TN-B1**

<b>Material</b>	<b>Density (g/cm<sup>3</sup>)</b>	<b>Constituent</b>	<b>Atomic Density (atoms/b-cm)</b>
U(5.0)O <sub>2</sub> 98% Theoretical Density	10.74	U-235 U-238 O	1.2128x10 <sup>-3</sup> 2.2753x10 <sup>-2</sup> 4.7931x10 <sup>-2</sup>
U(5.0)O <sub>2</sub> -Gd <sub>2</sub> O <sub>3</sub> 98% Theoretical Density 2 wt% Gd <sub>2</sub> O <sub>3</sub> (75% credit for Gd)	10.67	U-235 U-238 O Gd-152 Gd-154 Gd-155 Gd-156 Gd-157 Gd-158 Gd-160	1.18663x10 <sup>-03</sup> 2.22611x10 <sup>-02</sup> 4.76929x10 <sup>-02</sup> 1.06320x10 <sup>-6</sup> 1.15892x10 <sup>-5</sup> 7.86790x10 <sup>-5</sup> 1.08822x10 <sup>-4</sup> 8.31978x10 <sup>-5</sup> 1.32053x10 <sup>-4</sup> 1.16211x10 <sup>-4</sup>
Zirconium	6.49	Zr	4.2846x10 <sup>-2</sup>
Stainless Steel 304	7.94	Fe Cr Ni Mn Si C P	5.8545x10 <sup>-2</sup> 1.7473x10 <sup>-2</sup> 7.7402x10 <sup>-3</sup> 1.7407x10 <sup>-3</sup> 1.7025x10 <sup>-3</sup> 3.1877x10 <sup>-4</sup> 6.9468x10 <sup>-5</sup>
Polyethylene Foam	≤ 0.05 – 0.075	C H	3.4374x10 <sup>-3</sup> 6.8748x10 <sup>-3</sup>
Low Density Polyethylene (LDPE) Insert	0.925	C H	3.9745x10 <sup>-2</sup> 7.9490x10 <sup>-2</sup>
Polyethylene Cluster Assembly	0.949	C H	4.0776x10 <sup>-2</sup> 8.1552x10 <sup>-2</sup>
Alumina Silicate [Al <sub>2</sub> O <sub>3</sub> (49%)- SiO <sub>2</sub> (51%)]	0.25	Al Si O	1.4474x10 <sup>-3</sup> 1.2783x10 <sup>-3</sup> 4.7277x10 <sup>-3</sup>
Paper Honeycomb C <sub>6</sub> H <sub>10</sub> O <sub>5</sub>	0.04 – 0.08	C H O	1.7840x10 <sup>-3</sup> 2.9733x10 <sup>-3</sup> 1.4867x10 <sup>-3</sup>
Full Density Water	1.0	H O	6.6769x10 <sup>-2</sup> 3.3385x10 <sup>-2</sup>

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Polyethylene inserts or polyethylene cluster separators are positioned between fuel rods at various locations along the axis of the fuel assembly to avoid stressing the axial grids during transportation. The inserts are shown in Figure 6-1 while the separators are shown in Figure 6-2. The Low Density Polyethylene (LDPE) insert has a  $0.925 \text{ g/cm}^3$  density and an approximate volume of  $25 \text{ cm}^3$ . Therefore, a 10x10 assembly with 9 polyethylene inserts has a  $225 \text{ cm}^3$  total LDPE volume required for one location along the fuel assembly.

The cluster separator is composed of LDPE ( $0.925 \text{ g/cm}^3$ ) fingers and a High Density Polyethylene (HDPE,  $0.959 \text{ g/cm}^3$ ) holder (The LDPE and HDPE densities are based on accepted industry definitions). The LDPE fingers (10x10) occupy an approximate volume of  $38 \text{ cm}^3$  while the HDPE holder has an approximate volume of  $85 \text{ cm}^3$ . A volume average density of  $0.949 \text{ g/cm}^3$  is calculated for the polyethylene cluster assembly, i.e.

$$\left[ \frac{\{38 \text{ cm}^3 \times 0.925 \text{ g/cm}^3\} + \{85 \text{ cm}^3 \times 0.959 \text{ g/cm}^3\}}{123 \text{ cm}^3} \right]$$


For a 10x10 assembly, two cluster separators, shown in Figure 6-2, are placed at numerous locations along the fuel assembly. A total polyethylene volume of  $246 \text{ cm}^3$  is calculated for each location in which the cluster separators are placed. The TN-B1 criticality calculations use the 10x10 cluster separator characteristics for the fuel types investigated. However, the polyethylene characteristics are only used to establish a polyethylene mass limit so that an accurate measurement of polyethylene characteristics by the user is unnecessary. Other plastics with equivalent hydrogen mass limits are acceptable. The following equation can be used to determine plastic equivalence (e.g., ABS plastic).

$$M_{eq,i} = M_{poly} \times \frac{0.137}{\rho_{mix,i} \times wf_{H,i}}$$

The formula for polyethylene mass equivalence is:

$$\begin{aligned} M_{eq,i} &= M_{poly} \times [(\rho_{mix, poly})(wf_{H, poly})]/[(\rho_{mix,i})(wf_{H,i})] \\ &= M_{poly} \times [(0.949 \text{ g/cm}^3)(0.144)]/[(\rho_{mix,i})(wf_{H,i})] \\ &= M_{poly} \times (0.137 \text{ g/cm}^3)/[(\rho_{mix,i})(wf_{H,i})] \end{aligned}$$

The fuel parameters used to calculate volume fractions for the water and polyethylene mixture in the TN-B1 normal condition model are shown in Table 6-9 TN-B1 Normal Condition Model Fuel Parameters. The volume fractions of polyethylene and water for the worst case fuel assembly

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type analyzed are shown in Table 6-10 TN-B1 Normal Condition Model Polyethylene and Water Volume Fractions and Table 6-11 Single Package Normal and HAC Model Fuel Parameters. The volume fractions in Table 6-10 are entered into the model input standard composition specification area. Mixtures representing the polyethylene inserts between fuel rods are created using the compositions specified, and used in the KENO V.a calculation. The mixtures are also used in the lattice cell description to provide the lump shape and dimensions for resonance cross-section processing, the lattice corrections for cross-section processing, and the information necessary to create cell-weighted cross-sections.

**Table 6-9 TN-B1 Normal Condition Model Fuel Parameters**

Fuel Assembly	Fuel Rod OR (cm)	Number of Fuel Rods	Fuel Rod Pitch (cm)	Fuel Rod Length (cm)	Cluster Separator Volume Surrounding Fuel (cm <sup>3</sup> )	Number of Part Length Fuel Rods
GNF 10x10	0.505	92	1.350	385	10,200	12

**Table 6-10 TN-B1 Normal Condition Model Polyethylene and Water Volume Fractions**

Fuel Assembly	Assembly Volume (cm <sup>3</sup> )	Fuel Rod Volume (cm <sup>3</sup> )	Interstitial Volume (cm <sup>3</sup> )	Polyethylene Volume (cm <sup>3</sup> )	Vf <sub>poly</sub>	Vf <sub>H2O</sub>
GNF 10x10	66,676.46	26,527.22	40,149.24	10,200	0.25405	0.74595

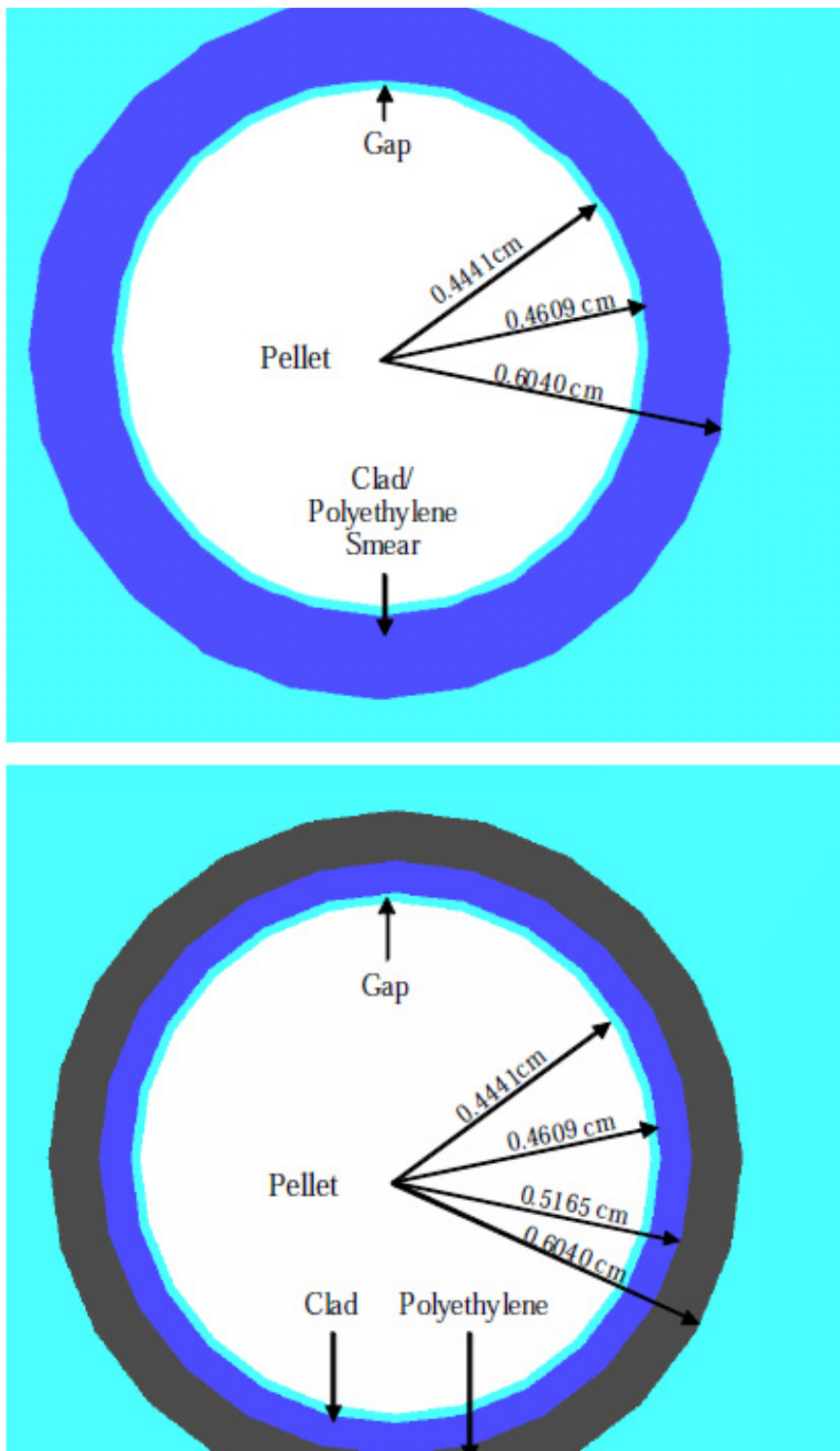
**Table 6-11 Single Package Normal and HAC Model Fuel Parameters**

Fuel Assembly	Partial Fuel Rods (#)	Pitch (cm)	Pellet Diameter (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)
GNF 10X10	12	1.350	0.895	0.9338	1.010


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In the hypothetical accident condition model, the polyethylene inserts are assumed to melt when subjected to the tests specified in 10 CFR Part 71. The polyethylene is assumed to uniformly coat the fuel rods in each fuel assembly forming a cylindrical layer of polyethylene around each fuel rod. Different coating thicknesses are investigated, and a maximum thickness is determined to set a polyethylene mass limit for each fuel assembly type considered for transport. The fuel assembly parameters used to calculate the polyethylene mass limits are shown in Table 6-12 Fuel Assembly Parameters for Polyethylene Mass Calculations. For the fuel parameter sensitivity study and the worst case fuel assembly models, the polyethylene is smeared into the fuel rod cladding to accommodate the limitations in the lattice cell modeling for cross-section processing in SCALE. A visual representation of the smeared clad/polyethylene mixture compared to a discrete treatment is shown in Figure 6-21 Visual Representation of the Clad/Polyethylene Smeared Mixture versus Discrete Modeling. The polyethylene mass and the volume fractions of polyethylene and zirconium clad for each fuel assembly analyzed are shown in Table 6-13 Polyethylene Mass and Volume Fraction Calculations. The volume fractions in Table 6-13 are entered into the model input standard composition specification area. Mixtures representing the polyethylene inserts between fuel rods are created using the compositions specified, and used in the KENO V.a calculation. The mixtures are also used in the lattice cell description to provide the lump shape and dimensions for resonance cross-section processing, the lattice corrections for cross-section processing, and the information necessary to create cell-weighted cross-sections.





**Figure 6-21 Visual Representation of the Clad/Polyethylene Smeared Mixture versus Discrete Modeling**


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**Table 6-12 Fuel Assembly Parameters for Polyethylene Mass Calculations**

Fuel Assembly Design	Fuel Rod OR (cm)	Number of Fuel Rods	Fuel Rod Pitch (cm)	Fuel Rod Length (cm)	Fuel Rod IR (cm)
ATRIUM 10x10	0.5165	91	1.284	383.54	0.4609
GNF 10x10	0.50927	92	1.2954	381	0.46609
Framatome 9x9	0.54991	72	1.4478	381	0.48006
GNF 9x9	0.55499	74	1.43764	381	0.49149
GNF 8x8	0.6096	60	1.6256	381	0.53594

**Table 6-13 Polyethylene Mass and Volume Fraction Calculations**

Radius (cm)	Thickness (cm)	Total Poly Volume <sup>a</sup> (cm <sup>3</sup> )	Total Poly Mass <sup>b</sup> (g)	Volume <sub>poly</sub> Per Fuel Rod <sup>c</sup> (cm <sup>3</sup> )	Volume <sub>clad</sub> Per Fuel Rod <sup>d</sup> (cm <sup>3</sup> )	Vf <sub>clad</sub> <sup>e</sup>	Vf <sub>poly</sub> <sup>f</sup>
<b>Two ATRIUM 10x10 Fuel Assemblies</b>							
0.51650	0.00000	0	0	0.00	65.47985	1.00000	0.00000
0.56504	0.04854	11512.03	10924.92	63.25	65.47985	0.50865	0.49135
0.59071	0.07421	18019.18	17100.20	99.01	65.47985	0.39809	0.60191
0.60395	0.08745	21487	20391.16	118.06	65.47985	0.35676	0.64324
0.61369	0.08000	24087.04	22858.60	132.35	65.47985	0.33100	0.66900
0.62343	0.10693	26729.6	25366.39	146.87	65.47985	0.30836	0.69164
0.63317	0.11667	29414.68	27914.53	161.62	65.47985	0.28833	0.71167
<b>Two GNF 10x10 Fuel</b>							
0.50927	0.00000	0	0	0.00	50.41067	1.00000	0.00000
0.55824	0.04897	11512.03	10924.92	62.57	50.41067	0.44621	0.55379
0.59086	0.08159	19768.04	18759.87	107.43	50.41067	0.31937	0.68063
0.59743	0.08816	21487	20391.16	116.78	50.41067	0.30152	0.69848
0.60723	0.09796	24087.04	22858.6	130.91	50.41067	0.27802	0.72198
0.61703	0.10776	26729.6	25366.39	145.27	50.41067	0.25762	0.74238
0.62683	0.11756	29414.68	27914.53	159.86	50.41067	0.23974	0.76026
<b>Two Framatome 9x9 Fuel Assemblies</b>							
0.5499	0.0000	0	0	0.00	86.11243	1.00000	0.00000
0.6470	0.0971	20021.07	19000	139.04	86.11243	0.38247	0.61753
0.6610	0.1111	23182.3	22000	160.99	86.11243	0.34849	0.65151
0.6702	0.1203	25289.78	24000	175.62	86.11243	0.32901	0.67099
0.6792	0.1293	27397.26	26000	190.26	86.11243	0.31158	0.68842
0.6882	0.1383	29504.74	28000	204.89	86.11243	0.29591	0.70409
0.6970	0.1471	31612.22	30000	219.53	86.11243	0.28174	0.71826

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**Table 6-13 Polyethylene Mass and Volume Fraction Calculations (continued)**

Radius (cm)	Thickness (cm)	Total Poly Volume <sup>a</sup> (cm <sup>3</sup> )	Total Poly Mass <sup>b</sup> (g)	Volume <sub>poly</sub> Per Fuel Rod <sup>c</sup> (cm <sup>3</sup> )	Volume <sub>clad</sub> Per Fuel Rod <sup>d</sup> (cm <sup>3</sup> )	Vf <sub>clad</sub> <sup>e</sup>	Vf <sub>poly</sub> <sup>f</sup>
<b>Two GNF 9x9 Fuel</b>							
0.55499	0.00000	0	0	0.00	79.53889	1.00000	0.00000
0.65344	0.09845	21074.82	20000	142.40	79.53889	0.35839	0.64161
0.66248	0.10749	23182.3	22000	156.64	79.53889	0.33678	0.66322
0.67140	0.11641	25289.78	24000	170.88	79.53889	0.31763	0.68237
0.68020	0.12521	27397.26	26000	185.12	79.53889	0.30054	0.69946
0.68889	0.13390	29504.74	28000	199.36	79.53889	0.28519	0.71481
0.69747	0.14248	31612.22	30000	213.60	79.53889	0.27134	0.72866
<b>Two GNF 8x8 Fuel</b>							
0.60960	0.00000	0	0	0.00	100.9989	1.00000	0.00000
0.71484	0.10524	20021.07	19000	166.84	100.9989	0.37709	0.62291
0.73008	0.12048	23182.3	22000	193.19	100.9989	0.34332	0.65668
0.74006	0.13046	25289.78	24000	210.75	100.9989	0.32398	0.67602
0.74990	0.14030	27397.26	26000	228.31	100.9989	0.30670	0.69330
0.75962	0.15002	29504.74	28000	245.87	100.9989	0.29117	0.70883
0.76922	0.15962	31612.22	30000	263.44	100.9989	0.27714	0.72286

The following example calculations are for two Atrium 10x10 assemblies with a total 21,487 cm<sup>3</sup> polyethylene volume:

- a. Total Polyethylene Volume = (Total Fuel Rod Number)x(2 Fuel Assemblies)x(Polyethylene Area)x(Fuel Rod Length)

$$Volume = (91 \text{ fuel rods}) \times (2 \text{ fuel assemblies}) \times \left\{ (\pi) \left[ (0.60395 \text{ cm})^2 - (0.5165 \text{ cm})^2 \right] \right\} (383.54 \text{ cm}) = 21487 \text{ cm}^3$$

- b. Total Polyethylene Mass = (Total Polyethylene Volume)x(Polyethylene Density)

$$Mass = \left( 21487 \text{ cm}^3 \right) \left( 0.949 \frac{\text{g}}{\text{cm}^3} \right) = 20391.16 \text{ g}$$

- c. Polyethylene Volume per Fuel Rod = Total Polyethylene Volume/Total Fuel Rod Number

$$\frac{Volume_{Poly}}{FuelRod} = \frac{21487 \text{ cm}^3}{(91 \text{ fuel rods}) \times (2 \text{ fuel assemblies})} = 118.06 \text{ cm}^3$$

- d. Clad Volume per Fuel Rod = [(Fuel Rod Area to Outer Clad)-(Fuel Rod Area to Inner Clad)]x Fuel Rod Length

$$\frac{Volume_{clad}}{FuelRod} = (\pi) \left[ (0.5165 \text{ cm})^2 - (0.4609 \text{ cm})^2 \right] (383.54 \text{ cm}) = 65.48 \text{ cm}^3$$

- e. Clad Volume Fraction = Clad Volume/Total Clad and Polyethylene Volumes

$$VF_{clad} = \frac{65.48 \text{ cm}^3}{(118.06 \text{ cm}^3) + (65.48 \text{ cm}^3)} = 0.35676$$

- f. Polyethylene Volume Fraction = Polyethylene Volume/ Total Clad and Polyethylene Volumes

$$VF_{Poly} = \frac{118.06 \text{ cm}^3}{(118.06 \text{ cm}^3) + (65.48 \text{ cm}^3)} = 0.64323$$

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### 6.3.3. *Computer Codes and Cross-Section Libraries*

The calculational methodology employed in the analyses is based on that embodied in SCALE - PC (version 4.4a), as documented in Reference 8. The neutron cross-section library employed in the analyses and the supporting validation analyses was the 44 group ENDF/B-V library distributed with version 4.4a of the SCALE package. Each case was run using the CSAS25 sequence of codes, i.e., BONAMI, NITAWL, and KENO V.a. For each case, 400 generations with 2,500 neutrons per generation were run to ensure proper behavior about the mean value. The methodology and results of the validation of SCALE 4.4a on the PC is outlined in Section 6.10, and results in an Upper Safety Limit (USL) that is the basis for comparison to ensure subcriticality.

For the performance of the Uranium-Carbide and PWR loose rod provision analysis, the GEMER Monte Carlo code was used. GEMER is a Monte Carlo neutron transport code developed by combining geometry and Monte Carlo features from the KENO IV and MERIT Monte Carlo codes and be adding enhance geometry, picture geometry checking and editing features (Ref. 4). Hence, GEMER is the evolution of Geometry Enhanced MERIT. The MERIT code is premised on the Battelle Northwest Laboratory's BMC code and is characterized by its explicit treatment of resolved resonance in material cross section set. Functionally, the GEMER Monte Carlo code is similar in analytic capability to other industry recognized codes such as KENO Va. or MCNP.

Cross sections in GEMER are currently processed from the ENDF/B-IV library in multigroup and resonance parameter formats. Cross-sections are prepared in the 190 energy group format and those in the resonance energy range have the form of resonance parameters. The resonance parameters describe the cross sections in the resonance range and Monte Carlo sampling in this range is done from resonance kernels rather than from broad group cross sections (i.e., explicit treatment of resolved resonance's using a single level Breit-Wigner equation at each collision in the resonance energy range). Thus there is a single unique cross section set associated with each available isotope and dependence is not placed on Dancoff (flux shadowing) correction factors or effective scattering cross sections. This treatment of cross-sections with explicit resonance parameters is especially suited to the analysis of uranium compounds in the form of heterogeneous accumulations, lattices, or systems containing nuclear poisons.

Thermal scattering of hydrogen is represented by the Hayward Kernel  $S(\alpha, \beta)$  data in the ENDF/B-IV library. The types of reactions considered in the GEMER Monte Carlo calculation are fission, capture, elastic, inelastic, and (n, 2n) reactions; absorption is implicitly treated by applying the non-absorption probability to neutron weights on each collision. As part of the solutions, GEMER produces eigenvalue, micro- and macro-group fluxes, reaction rates, cross sections, and neutron balance by isotopes.


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#### **6.3.4. *Demonstration of Maximum Reactivity***

The objectives for the TN-B1 shipping container analysis are to demonstrate package criticality safety and determine fuel loading criteria. To accomplish these objectives, calculations are performed to determine the most reactive fuel configuration inside the TN-B1 assembly compartments. Once the fuel configuration is determined, moderator and reflector conditions are investigated. Finally, package orientation (for arrays) is examined. When the worst case fuel configuration, moderator/reflector conditions, and package orientation are found, the single package and package array calculations under both normal and hypothetical accident conditions are performed.

##### **6.3.4.1. Fuel Assembly Orientation Study (2N=448)**

The package array dimensions for the fuel assembly orientation are 14x2x16 (width x depth x height). Initial calculations are performed to find the worst case fuel assembly orientation inside each TN-B1 fuel compartment. Nominal fuel assembly dimensions are used for these initial calculations (Table 6-4). Note that in all cases with cladding, zirconium is used to conservatively represent any zirconium alloy. The package array HAC model described in Section 6.3.1.2.2 is used and the fuel assembly orientations depicted in Figure 6-9 through Figure 6-15 are applied. In addition, a polyethylene coating covers each fuel rod in the assembly, the fuel assembly is un-channeled, and the moderator density is 1.0 g/cm<sup>3</sup> in the TN-B1 inner container fuel region. The polyethylene foam is assumed to burn away, Alumina Silicate thermal insulator envelopes the inner container, and no water is in either the outer container or between packages in the array. The results of the calculations are shown in Table 6-14 TN-B1 Array HAC Fuel Assembly Orientation. Based on the results in Table 6-14, assembly orientation 6, is bounding for all designs. Therefore, orientation 6 with the assembly centered in each fuel compartment is used in the remaining design calculations. It is also noted that most results in Table 6-14 exceed the 0.94254 USL. For this reason, gadolinia-urania fuel rods are added to the fuel assemblies to provide reactivity hold-down.


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**Table 6-14 TN-B1 Array HAC Fuel Assembly Orientation**

Fuel Assembly	Interspersed Moderator Density (g/cm <sup>3</sup> )	Polyethylene Mass Per Assembly (kg)	Assembly Orientation	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
FANP 10x10	0.0	10.2	1	0.9375	0.0010	0.9395
FANP 10x10	0.0	10.2	2	0.9529	0.0008	0.9545
FANP 10x10	0.0	10.2	3	0.8973	0.0008	0.8989
FANP 10x10	0.0	10.2	4	0.8965	0.0010	0.8985
FANP 10x10	0.0	10.2	5	0.9248	0.0010	0.9268
FANP 10x10	0.0	10.2	6	0.9741	0.0009	<b>0.9759<sup>b</sup></b>
FANP 10x10	0.0	10.2	7	0.9486	0.0009	0.9504
GNF 10x10	0.0	10.2	1	0.9586	0.0010	0.9606
GNF 10x10	0.0	10.2	2	0.9721	0.0009	0.9739
GNF 10x10	0.0	10.2	3	0.9184	0.0008	0.9200
GNF 10x10	0.0	10.2	4	0.9183	0.0009	0.9201
GNF 10x10	0.0	10.2	5	0.9431	0.0008	0.9447
GNF 10x10	0.0	10.2	6	0.9909	0.0010	<b>0.9929<sup>b</sup></b>
GNF 10x10	0.0	10.2	7	0.9652	0.0008	0.9668
FANP 9x9 <sup>a</sup>	0.0	11	1	0.9486	0.0009	0.9504
FANP 9x9	0.0	11	2	0.9559	0.0009	0.9577
FANP 9x9	0.0	11	3	0.9052	0.0008	0.9068
FANP 9x9	0.0	11	4	0.9056	0.0008	0.9072
FANP 9x9	0.0	11	5	0.9293	0.0010	0.9313
FANP 9x9	0.0	11	6	0.9791	0.0008	<b>0.9807<sup>b</sup></b>
FANP 9x9	0.0	11	7	0.9362	0.0009	0.9380
GNF 9x9	0.0	11	1	0.9491	0.0008	0.9507
GNF 9x9	0.0	11	2	0.9577	0.0008	0.9593
GNF 9x9	0.0	11	3	0.9051	0.0008	0.9067
GNF 9x9	0.0	11	4	0.9042	0.0009	0.9060
GNF 9x9	0.0	11	5	0.9287	0.0009	0.9305
GNF 9x9	0.0	11	6	0.9787	0.0008	<b>0.9803<sup>b</sup></b>
GNF 9x9	0.0	11	7	0.9556	0.0008	0.9572
GNF 8x8	0.0	11	1	0.9506	0.0009	0.9524
GNF 8x8	0.0	11	2	0.9563	0.0008	0.9579

- a. The Framatome D-lattice 9x9 assembly was modeled. However, the results presented here are applicable to the C-lattice as well
- b. Limiting case shown in bold



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**Table 6-14 TN-B1 Array HAC Fuel Assembly Orientation (continued)**

Fuel Assembly	Interspersed Moderator Density (g/cm <sup>3</sup> )	Polyethylene Mass Per Assembly (kg)	Assembly Orientation	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
GNF 8x8	0.0	11	3	0.9048	0.0008	0.9064
GNF 8x8	0.0	11	4	0.9052	0.0009	0.9070
GNF 8x8	0.0	11	5	0.9299	0.0009	0.9317
GNF 8x8	0.0	11	6	0.9764	0.0008	<b>0.9780<sup>b</sup></b>
GNF 8x8	0.0	11	7	0.9554	0.0009	0.9572


b. Limiting case shown in bold

#### 6.3.4.2. Fuel Assembly Gadolinia Rod Study (2N=448)

Fuel assemblies with lattice average U-235 enrichments of 5.0 wt% are qualified for transport in the TN-B1 shipping container by crediting the gadolinia-urania fuel rods present in the assembly. The gadolinia-urania fuel rods decrease system reactivity such that the k<sub>eff</sub> + 2σ remains below the 0.94254 USL. The gadolinia content of each gadolinia-urania fuel rod is limited to 75% of the value specified in Table 6-1. Scoping studies are performed using numerous gadolinia-urania fuel rod placement patterns in the orientation 6 models, from the fuel assembly orientation study, to find the pattern that yields the highest reactivity for each fuel assembly type. Of the patterns investigated, three patterns that produce the highest reactivity for each fuel assembly type are shown in Figure 6-22 - Figure 6-24. The calculations are performed using optimum moderator conditions. The results for the 14x2x16 TN-B1 container array transporting 10x10, 9x9, or 8x8 fuel assemblies with gadolinia-urania fuel rods arranged in the patterns displayed in Figure 6-22 - Figure 6-24 are listed in Table 6-15. As shown in Table 6-15, the gadolinia-urania fuel rods hold the system reactivity below the 0.94254 USL. Based on the gadolinia-urania fuel rod pattern optimization calculations:

- Gadolinia-urania fuel rod Pattern G is selected for future FANP 10x10 fuel assembly sensitivity calculations;
- Gadolinia-urania fuel rod Pattern B is selected for future GNF 10x10 fuel assembly sensitivity calculations;
- Gadolinia-urania fuel rod Pattern A is selected for future FANP and GNF 9x9 fuel assembly sensitivity calculations; and
- Gadolinia-urania fuel rod Pattern I is selected for future GNF 8x8 fuel assembly sensitivity calculations.




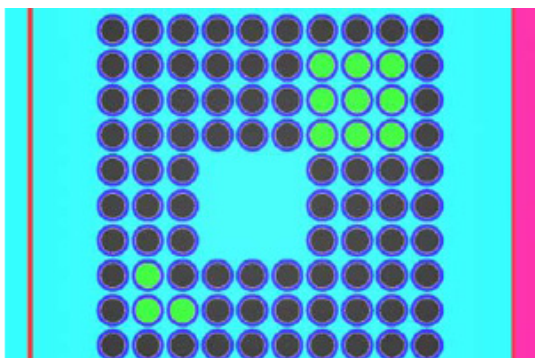
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**Table 6-15 TN-B1 Shipping Container 14x2x16 Array with Gadolinia- Urania Fuel Rods**

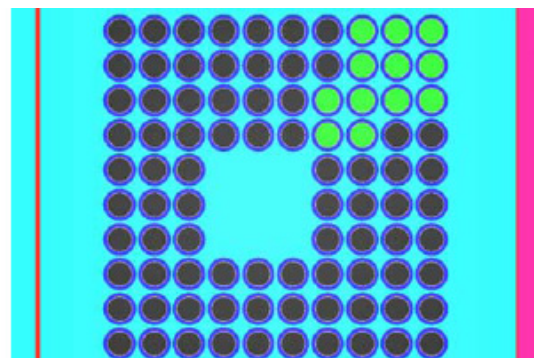
Assembly Type	Pattern Designation	U-235 Enrich (wt%)	Gad Rod #	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
FANP 10x10	B	5.0	12	1.284	0.8882	0.9218	1.033	0.8716	0.0008	0.8732
FANP 10x10	F	5.0	12	1.284	0.8882	0.9218	1.033	0.8699	0.0008	0.8715
FANP 10x10	G	5.0	12	1.284	0.8882	0.9218	1.033	0.8732	0.0008	<b>0.8748</b>
GNF 10x10	B	5.0	12	1.2954	0.8941	0.9322	1.019	0.8886	0.0008	<b>0.8902</b>
GNF 10x10	G	5.0	12	1.2954	0.8941	0.9322	1.019	0.8871	0.0008	0.8887
GNF 10x10	H	5.0	12	1.2954	0.8941	0.9322	1.019	0.8880	0.0009	0.8898
FANP 9x9	A	5.0	10	1.4478	0.9398	0.9601	1.099	0.8644	0.0007	<b>0.8658</b>
FANP 9x9	B	5.0	10	1.4478	0.9398	0.9601	1.099	0.8605	0.0008	0.8621
FANP 9x9	E	5.0	10	1.4478	0.9398	0.9601	1.099	0.8354	0.0009	0.8372
GNF 9x9	A	5.0	10	1.4376	0.9550	0.9830	1.110	0.8579	0.0008	<b>0.8596</b>
GNF 9x9	B	5.0	10	1.4376	0.9550	0.9830	1.110	0.8572	0.0008	0.8588
GNF 9x9	F	5.0	10	1.4376	0.9550	0.9830	1.110	0.8524	0.0009	0.8540
GNF 8x8	E	5.0	7	1.6256	1.0439	1.0719	1.219	0.8779	0.0009	0.8797
GNF 8x8	G	5.0	7	1.6256	1.0439	1.0719	1.219	0.8726	0.0008	0.8742
GNF 8x8	I	5.0	7	1.6256	1.0439	1.0719	1.219	0.8800	0.0009	<b>0.8818</b>

a. Limiting case(s) shown in bold

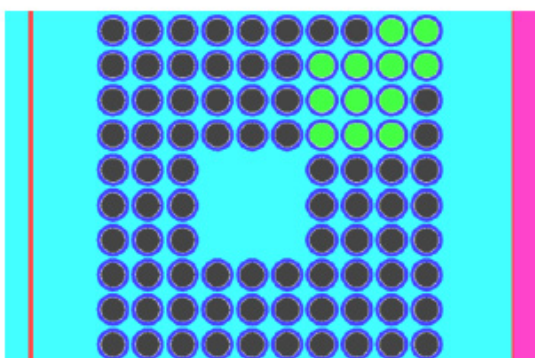
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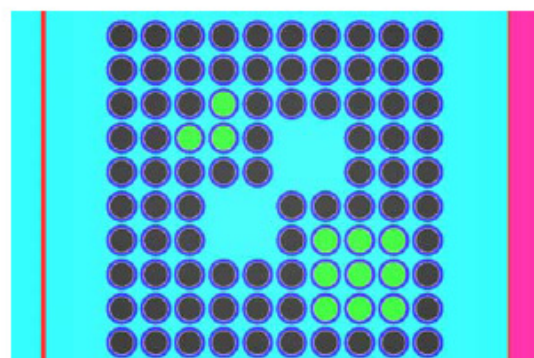
FANP 10x10 5.0 wt% <sup>235</sup>U, Pattern B



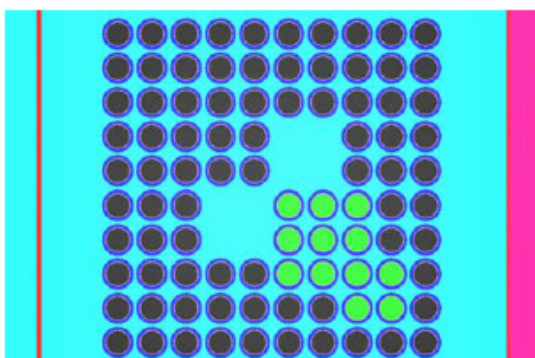
FANP 10x10 5.0 wt% <sup>235</sup>U, Pattern F



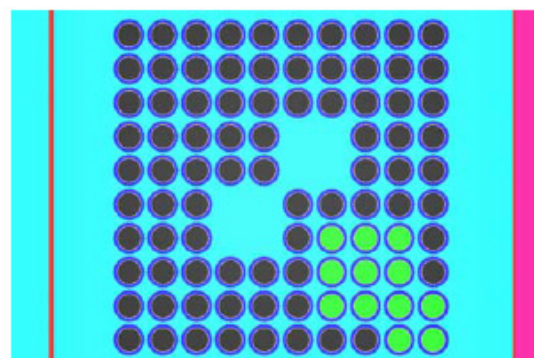
FANP 10x10 5.0 wt% <sup>235</sup>U, Pattern G



GNF 10x10 5.0 wt% <sup>235</sup>U, Pattern B

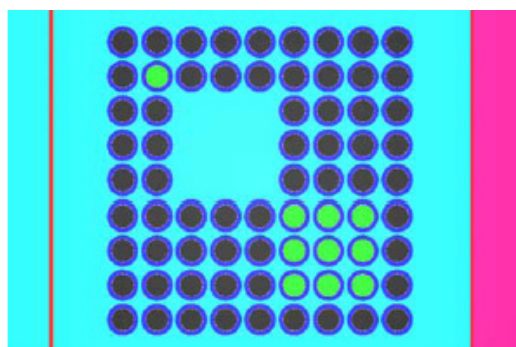


GNF 10x10 5.0 wt% <sup>235</sup>U, Pattern G

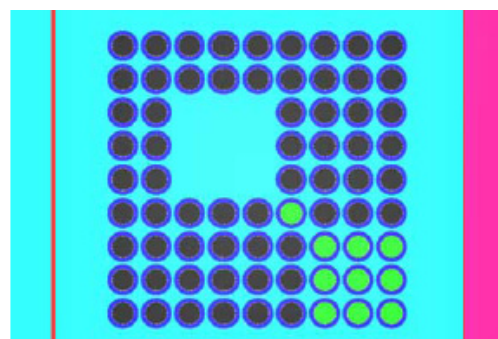


GNF 10x10 5.0 wt% <sup>235</sup>U, Pattern H

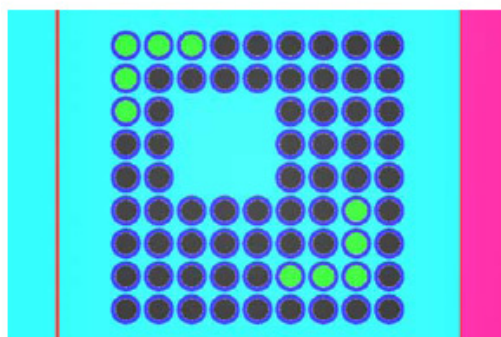
**Figure 6-22 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies at 5.0 wt% <sup>235</sup>U**



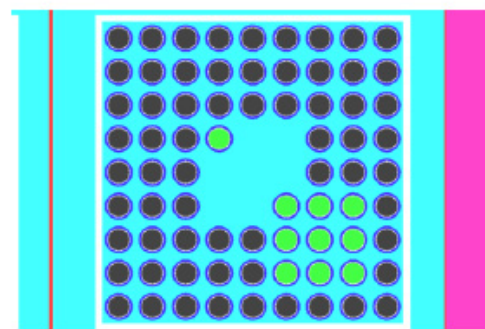
**FANP 9x9 5.0 wt% <sup>235</sup>U, Pattern A**



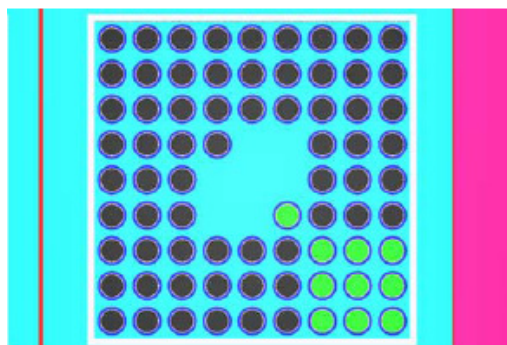
**FANP 9x9 5.0 wt% <sup>235</sup>U, Pattern B**



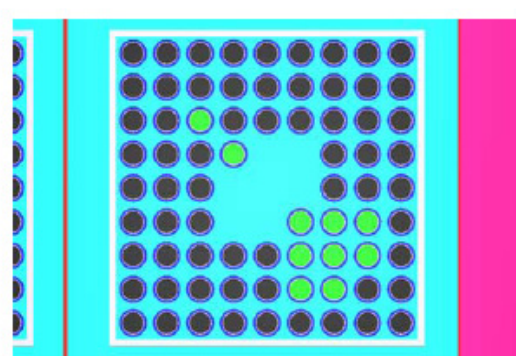
**FANP 9x9 5.0 wt% <sup>235</sup>U, Pattern E**



**GNF 9x9 5.0 wt% <sup>235</sup>U, Pattern A**




**GNF 9x9 5.0 wt% <sup>235</sup>U, Pattern B**

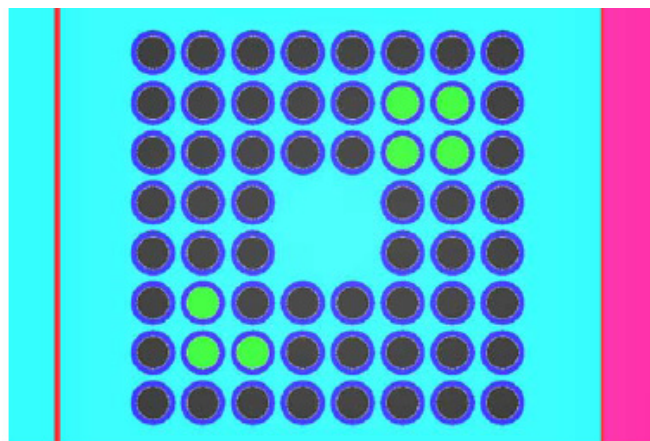


**GNF 9x9 5.0 wt% <sup>235</sup>U, Pattern F**

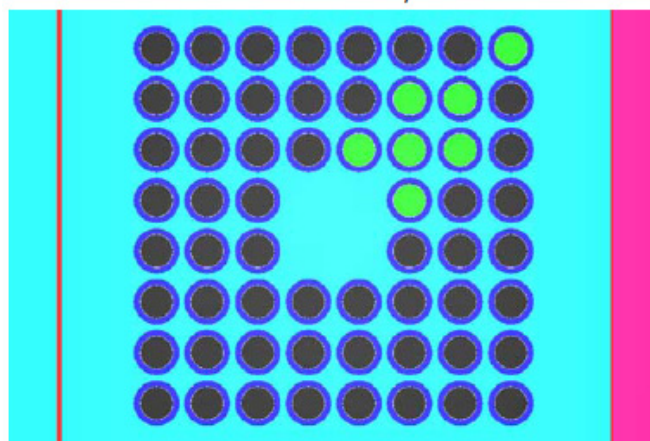
**Figure 6-23 Gadolinia-Urania Fuel Rod Placement Pattern for 9x9 Fuel Assemblies at 5.0 wt% <sup>235</sup>U**



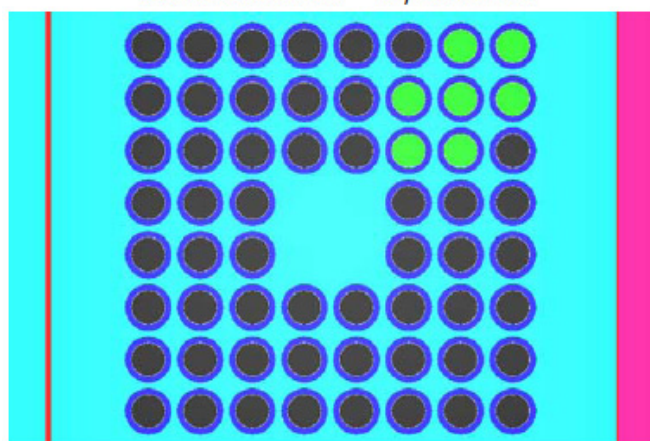
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GNF 8x8 5.0 wt% <sup>235</sup>U, Pattern E




GNF 8x8 5.0 wt% <sup>235</sup>U, Pattern G



GNF 8x8 5.0 wt% <sup>235</sup>U, Pattern I

**Figure 6-24 Gadolinia-Urania Fuel Rod Placement Pattern for 8x8 Fuel Assemblies at 5.0 wt% <sup>235</sup>U**

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#### 6.3.4.3. Fuel Assembly Channel Study (2N=448)


A calculation is performed to determine if the presence of channels around the fuel assembly increases system reactivity. The orientation 6 models with the gadolima-urania fuel rod patterns that produced the highest system reactivity from the previous studies are used and a zirconium channel is placed around each assembly as shown in Figure 6-16 TN-B1 Hypothetical Accident Condition Model with Channels. The channel thickness is varied from 0.17 cm to 0.3048 cm and the impact on reactivity is assessed. The fuel assembly channel is located in the reflector region for each fuel assembly. It has no effect on the assembly H/X ratio since it is not located within the fuel envelope. Therefore, removing it would not have the same impact on system reactivity as removing the internal grid structure. The results are shown in Table 6-16.

Comparing the results in Table 6-16 and Table 6-15 indicates reactivity increases with the presence of channels due to increased neutron leakage from the inner fuel compartment, resulting in increased neutron interaction among containers in the array. Therefore, channels will be included in subsequent calculations.

**Table 6-16 TN-B1 Sensitivity Analysis for Channeled Fuel Assemblies**

Assembly Type	Channel Thickness (cm)	Poly Mass per Assembly (kg)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
FANP 10x10	0.1700	10.2	1.284	0.8882	0.9218	1.033	0.8801	0.0008	0.8817
FANP 10x10	0.2032	10.2	1.284	0.8882	0.9218	1.033	0.8786	0.0008	0.8802
FANP 10x10	0.2540	10.2	1.284	0.8882	0.9218	1.033	0.8815	0.0009	<b>0.8833</b>
FANP 10x10	0.3048	10.2	1.284	0.8882	0.9218	1.033	0.8810	0.0008	0.8826
GNF 10x10	0.1700	10.2	1.2954	0.8941	0.9322	1.019	0.8922	0.0009	0.8940
GNF 10x10	0.2032	10.2	1.2954	0.8941	0.9322	1.019	0.8948	0.0008	0.8964
GNF 10x10	0.2540	10.2	1.2954	0.8941	0.9322	1.019	0.8947	0.0008	0.8963
GNF 10x10	0.3048	10.2	1.2954	0.8941	0.9322	1.019	0.8953	0.0008	<b>0.8969</b>
FANP 9x9	0.1700	11	1.4478	0.9398	0.9601	1.0998	0.8719	0.0009	0.8737
FANP 9x9	0.2032	11	1.4478	0.9398	0.9601	1.0998	0.8724	0.0009	0.8742
FANP 9x9	0.2540	11	1.4478	0.9398	0.9601	1.0998	0.8739	0.0008	0.8756
FANP 9x9	0.3048	11	1.4478	0.9398	0.9601	1.0998	0.8755	0.0009	<b>0.8773</b>

a. Limiting case(s) shown in bold

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**Table 6-16 TN-B1 Sensitivity Analysis for Channeled Fuel Assemblies (continued)**

Assembly Type	Channel Thickness (cm)	Poly Mass per Assembly (kg)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
GNF 9x9	0.1700	11	1.4376	0.9550	0.9830	1.11	0.8626	0.0009	0.8644
GNF 9x9	0.2032	11	1.4376	0.9550	0.9830	1.11	0.8651	0.0009	0.8669
GNF 9x9	0.2540	11	1.4376	0.9550	0.9830	1.11	0.8654	0.0010	<b>0.8674</b>
GNF 9x9	0.3048	11	1.4376	0.9550	0.9830	1.11	0.8659	0.0008	0.8676
GNF 8x8	0.1700	11	1.6256	1.0439	1.0719	1.2192	0.8834	0.0010	0.8854
GNF 8x8	0.2032	11	1.6256	1.0439	1.0719	1.2192	0.8857	0.0008	0.8873
GNF 8x8	0.2540	11	1.6256	1.0439	1.0719	1.2192	0.8884	0.0009	0.8902
GNF 8x8	0.3048	11	1.6256	1.0439	1.0719	1.2192	0.8900	0.0009	<b>0.8918</b>

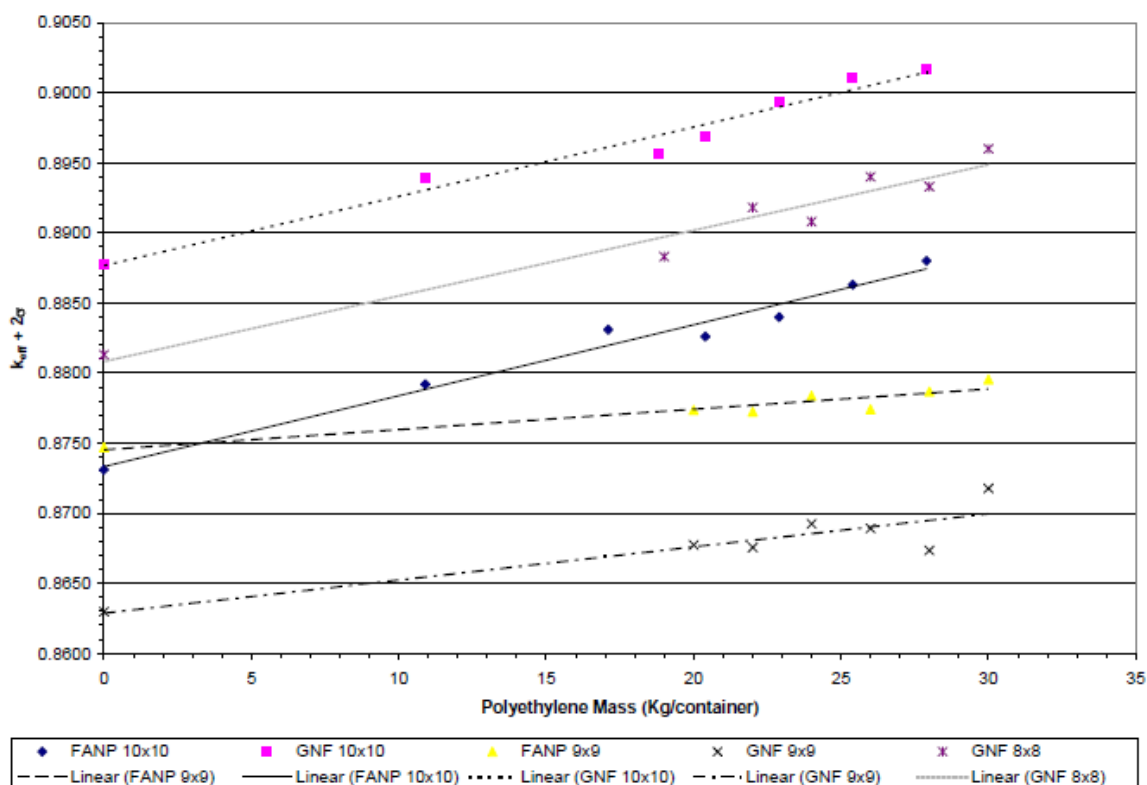
a. Limiting case(s) shown in bold

#### 6.3.4.4. Polyethylene Mass Study (2N=448)

The effect that polyethylene mass has on reactivity for each fuel assembly design is considered for transport in the TN-B1 shipping container. The results of the previous sensitivity studies are taken into consideration for the polyethylene mass study. The worst case channeled (0.3048 cm thick channels) models, used in the previous study, are used for the polyethylene mass study.

The polyethylene and clad volume fractions, shown in Table 6-13, are used in the model material description to represent the polyethylene and clad mixture. They are also used in the lattice cell description for resonance cross-section processing. The polyethylene coating thickness around the fuel rods is varied, and the effect on reactivity is determined. The results of the calculations, Table 6-26, are displayed in Figure 6-25 TN-B1 Array HAC Polyethylene Sensitivity. Although the polyethylene addition increases reactivity, the increase is gradual and the resulting system  $k_{eff}$  remains subcritical. Based on the results in Figure 6-25:

- a polyethylene mass of 10.2 kg/assembly (20.4 kg/container) is chosen for further FANP and GNF 10x10 calculations; and
- an 11 kg/assembly (22 kg/container) polyethylene mass is selected for subsequent FANP 9x9, GNF 9x9, and GNF 8x8 fuel assembly calculations.



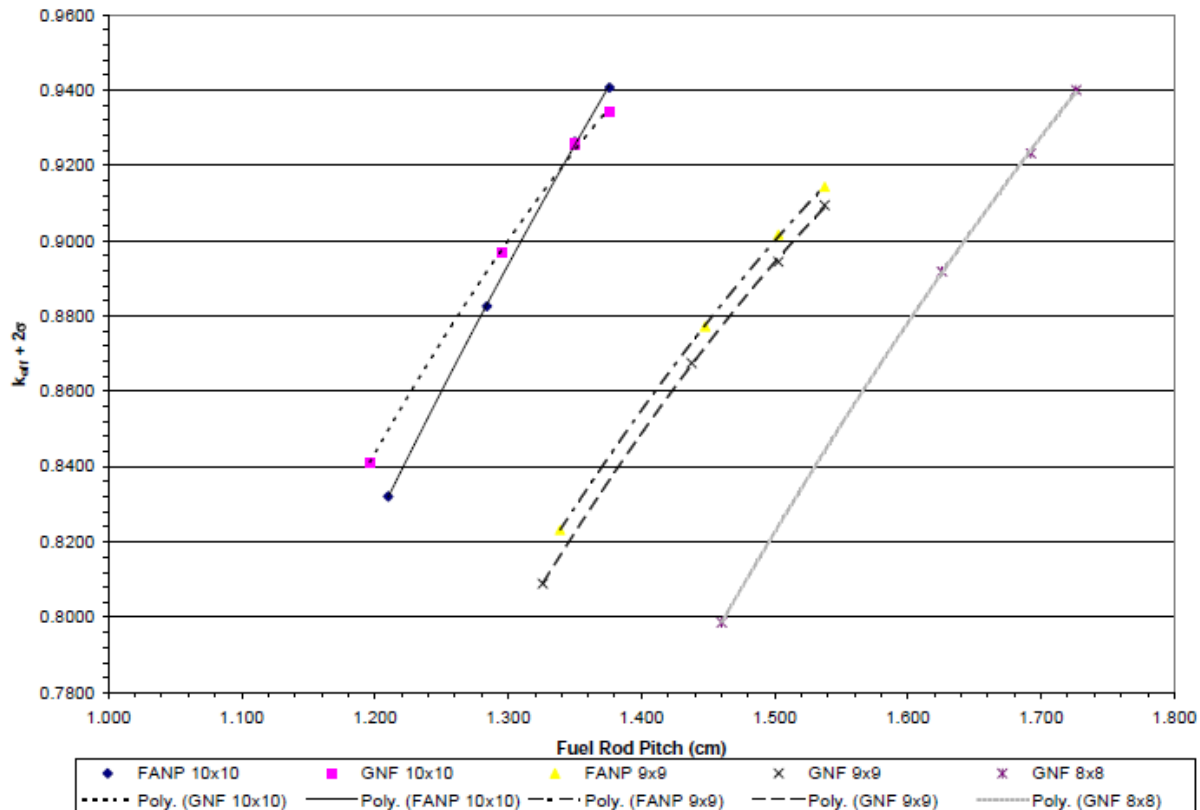
**Figure 6-25 TN-B1 Array HAC Polyethylene Sensitivity**

#### 6.3.4.5. Fuel Rod Pitch Sensitivity Study (2N=448)

A fuel rod pitch sensitivity study is conducted using the worst case models from the polyethylene sensitivity study. The minimum fuel rod pitch is chosen to be at the point that the polyethylene coating on adjacent fuel rods contact. The maximum fuel rod pitch is chosen to be 4.1% greater than the reference fuel designs to bound the damage sustained during the 9 meter drop. The results are shown in Figure 6-26 TN-B1 Fuel Rod Pitch Sensitivity Study. Based on the results in Figure 6-26, the fuel assemblies are under-moderated such that increasing the pitch increases system reactivity. Based on the pitch sensitivity calculations (Table 6-27):

- a 1.350 cm fuel rod pitch is selected as the upper limit for FANP and GNF 10x10 pitch range;
- a 1.510 cm fuel rod pitch is selected as the upper limit for FANP and GNF 9x9 pitch range; and
- a 1.6923 cm fuel rod pitch is selected as the upper limit for GNF 8x8 pitch range.

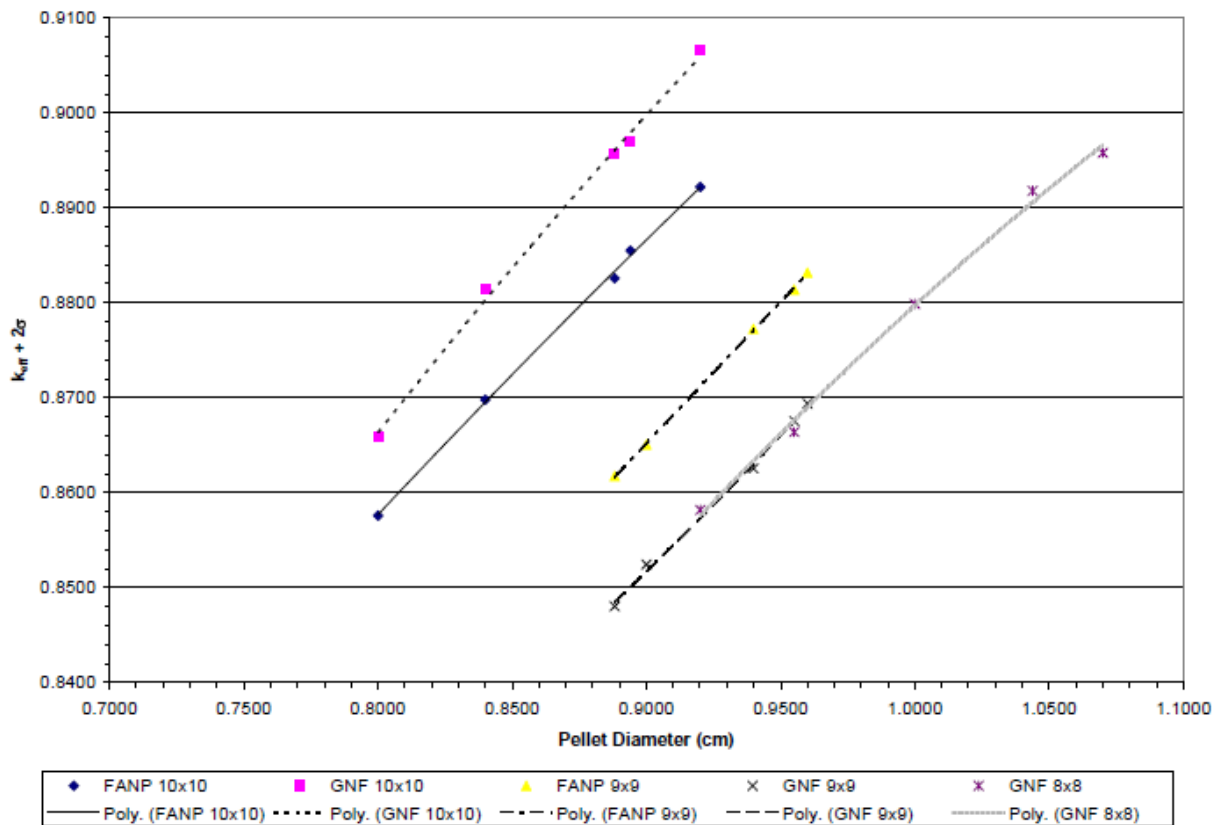




**Figure 6-26 TN-B1 Fuel Rod Pitch Sensitivity Study**

#### 6.3.4.6. Fuel Pellet Diameter Sensitivity Study (2N=448)

With a polyethylene quantity chosen, the worst case orientation known, the channeled fuel effect assessed, and the worst case gadolinia-urania fuel rod patterns identified, a fuel pellet diameter sensitivity study is conducted. For the pellet diameter sensitivity study, the package array HAC model described in Section 6.3.1.2.2 is used for the study, fuel assembly orientation 6 is selected based on the results in Table 6-14, the maximum polyethylene amount for each fuel assembly design is chosen, the worst case gadolinia-urania rod pattern is selected, the inner container fuel compartment is maintained at optimum density water, an Alumina Silicate thermal insulator envelopes the inner container fuel compartment, and water is removed from the outer container and between packages in the array. The results are shown in Figure 6-27 TN-B1 Array HAC Pellet Diameter Sensitivity Study. The results in Figure 6-27, demonstrate that reactivity increases as pellet diameter is increased. Pellet diameters of 0.895 cm for the FANP and GNF 10x10 designs, 0.96 cm for the Framatome and GNF 9x9 designs, and 1.05 cm for the GNF 8x8 design are found acceptable as the upper bounds for the fuel assembly design pellet ranges (Table 6-28).



**Figure 6-27 TN-B1 Array HAC Pellet Diameter Sensitivity Study**

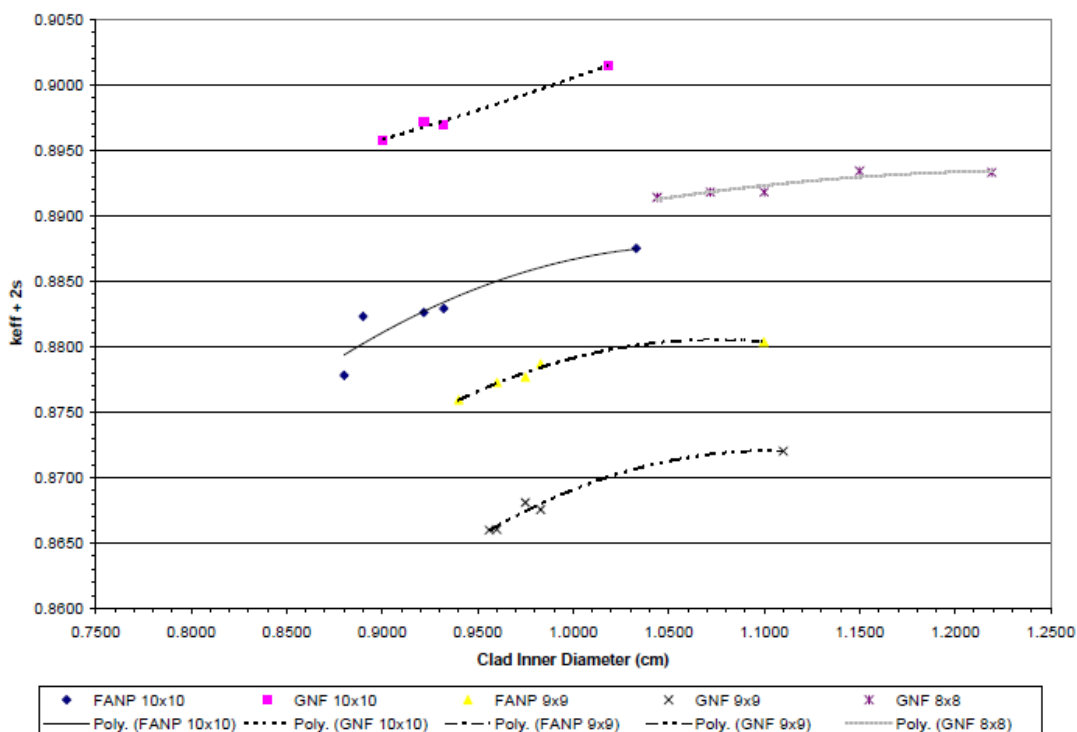
#### 6.3.4.7. Fuel Rod Clad Thickness Sensitivity Study (2N=448)

Two sets of calculations are performed to assess the reactivity sensitivity to changes in cladding thickness. For the clad thickness sensitivity studies, the package array HAC model described in Section 6.3.1.2.2 is used for the study, fuel assembly orientation 6 is selected based on the results in Table 6-14, the maximum polyethylene amount for each fuel assembly design is chosen, the worst case gadolinia-urania rod pattern is selected, the inner container fuel compartment is maintained at optimum density moderation, an Alumina Silicate thermal insulator envelopes the inner container fuel compartment, and water is removed from the outer container and between packages in the array. For the first set of calculations, the inner clad diameter is adjusted to determine the effect on reactivity while the outer clad diameter is fixed at its nominal value shown in Table 6-4. The minimum value for the parameter search range is the pellet OD, while the maximum value for the range is the clad OD. The second set of calculations involves adjustments to the outer clad diameter while the inner clad diameter is held at its nominal value Table 6-4. Figure 6-28 TN-B1 Array HAC Fuel Rod Clad ID Sensitivity Study displays the results for the inner clad diameter sensitivity calculations, and Figure 6-29 TN-B1 Array HAC

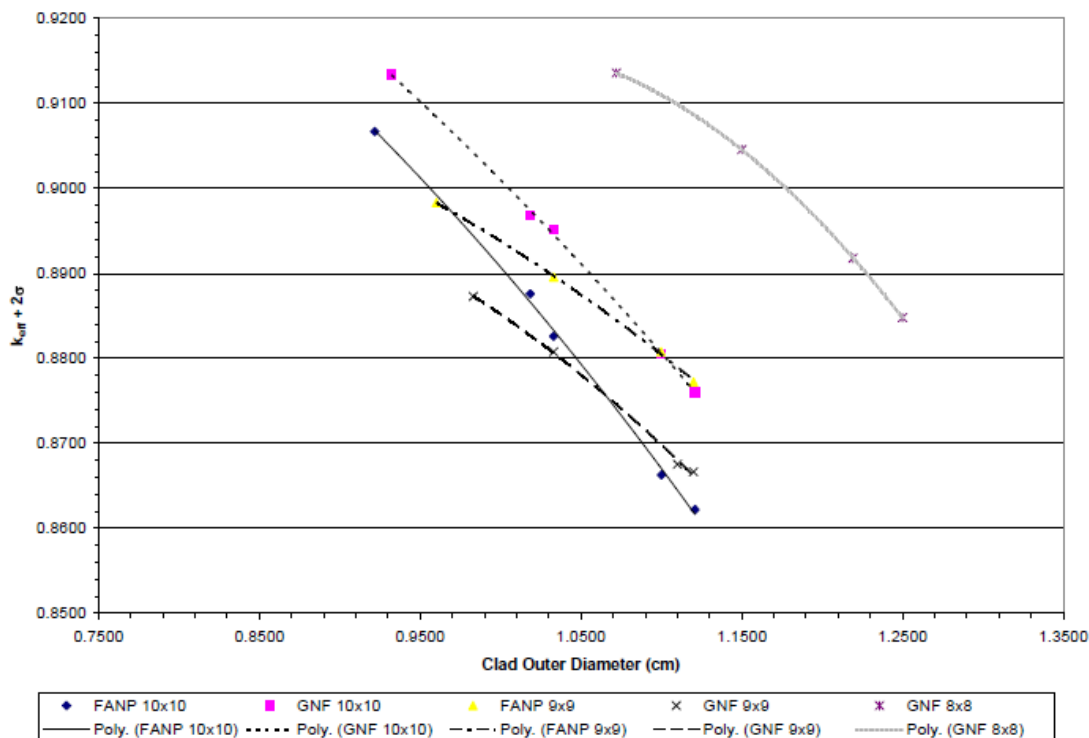
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Fuel Rod Clad OD Sensitivity Study shows the results for the outer clad diameter sensitivity study. Both sets of results demonstrate that a decrease in the clad thickness results in an increase in system reactivity. The results also indicate that reactivity increases as the clad OD is decreased and increases as the clad ID is increased. Based on these results and fabrication constraints (Table 6-30 and Table 6-31):

- a 0.933 cm upper bound clad ID, and a 1.00 cm lower bound clad OD are selected for the FANP and GNF 10x10 parameter ranges;
- a 1.02 cm upper bound clad ID, and a 1.09 cm lower bound clad OD are selected for the FANP and GNF 9x9 parameter ranges; and
- a 1.10 cm upper bound clad ID, and a 1.17 cm lower bound clad OD are selected for the GNF 8x8 parameter range.




**Figure 6-28 TN-B1 Array HAC Fuel Rod Clad ID Sensitivity Study**




**Figure 6-29 TN-B1 Array HAC Fuel Rod Clad OD Sensitivity Study**

#### 6.3.4.8. Worst Case Parameter Fuel Designs (2N=448)

The previous calculations have varied single parameters and assessed the impact on reactivity. Since the ranges investigated are to be a part of the fuel loading criteria, an assessment must be made for more than one parameter change at a time. To validate the parameter ranges selected to appear in the fuel loading criteria, a fuel design is developed by assembling the worst case parameters for each design considered for transport in the TN-B1 container. Table 6-17 TN-B1 Array HAC Worst Case Parameter Fuel Designs contains the worst case parameters for each design. The worst case models from the clad ID and OD sensitivity study are used to conduct the worst case fuel parameter study. The polyethylene is smeared into the fuel rod cladding to accommodate the limitations in the lattice cell modeling for cross-section processing in SCALE. A search for the worst case gadolinia-urania fuel rod pattern is also conducted to validate the worst case fuel design. Numerous patterns were investigated for each fuel assembly with the worst case fuel parameters determined from the sensitivity studies. Of the patterns investigated, three patterns that produce the highest reactivity for each fuel

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assembly type are shown in Figure 6-22 - Figure 6-24. Additional calculations are performed to investigate the number of gadolinia-urania fuel rods needed based on fuel assembly U-235 enrichment. For each fuel assembly U-235 enrichment, a gadolinia-urania fuel rod pattern optimization study is conducted. The three patterns that produce the highest reactivity for each fuel assembly based on U-235 enrichment are shown in Figure 6-30 - Figure 6-32. All results are listed in Table 6-17 and are below the USL of 0.94254. Based on the results listed in Table 6-17, all worst case fuel assembly designs result in maximum system reactivities that are within the statistical uncertainty of one another.

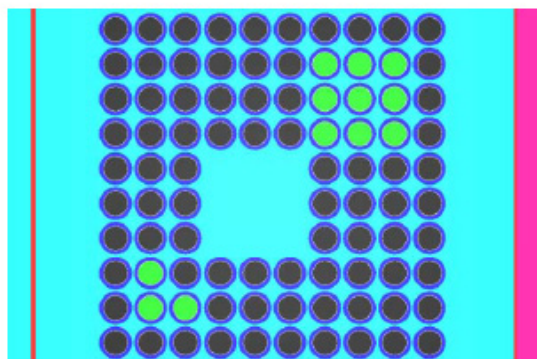
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**Table 6-17 TN-B1 Array HAC Worst Case Parameter Fuel Designs**

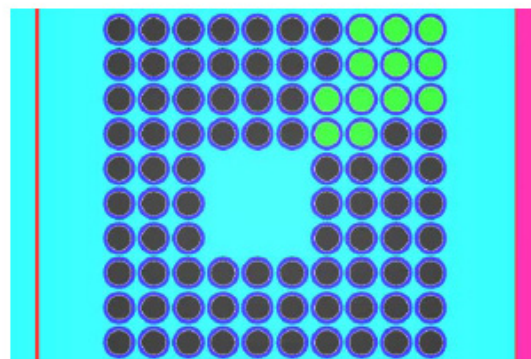
Assembly Type	Gadolinia-Urania Fuel Rod Number	<sup>235</sup> U Enrichment (wt%)	Poly Mass per Assembly (kg)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	keff	σ	keff + 2σ
FANP 10x10	12	5.0	10.2	1.350	0.895	0.933	1.00	0.9368	0.0008	0.9384
FANP 10x10	10	4.6	10.2	1.350	0.895	0.933	1.00	0.9360	0.0009	0.9378
FANP 10x10	9	4.3	10.2	1.350	0.895	0.933	1.00	0.9325	0.0010	0.9345
FANP 10x10	8	4.2	10.2	1.350	0.895	0.933	1.00	0.9366	0.0009	<b>0.9384</b>
FANP 10x10	6	3.9	10.2	1.350	0.895	0.933	1.00	0.9353	0.0007	0.9367
FANP 10x10	4	3.6	10.2	1.350	0.895	0.933	1.00	0.9341	0.0009	0.9359
FANP 10x10	2	3.3	10.2	1.350	0.895	0.933	1.00	0.9305	0.0009	0.9323
FANP 10x10	0	2.9	10.2	1.350	0.895	0.933	1.00	0.9274	0.0008	0.9290
GNF 10x10	12	5.0	10.2	1.350	0.895	0.933	1.00	0.9393	0.0008	0.9409
GNF 10x10	10	4.6	10.2	1.350	0.895	0.933	1.00	0.9349	0.0010	0.9369
GNF 10x10	9	4.3	10.2	1.350	0.895	0.933	1.00	0.9346	0.0008	0.9362
GNF 10x10	8	4.2	10.2	1.350	0.895	0.933	1.00	0.9395	0.0009	<b>0.9413</b>
GNF 10x10	6	3.9	10.2	1.350	0.895	0.933	1.00	0.9377	0.0009	0.9395
GNF 10x10	4	3.6	10.2	1.350	0.895	0.933	1.00	0.9370	0.0008	0.9386
GNF 10x10	2	3.3	10.2	1.350	0.895	0.933	1.00	0.9344	0.0009	0.9362
GNF 10x10	0	2.9	10.2	1.350	0.895	0.933	1.00	0.9317	0.0007	0.9331
FANP 9x9	10	5.0	11	1.510	0.96	1.02	1.09	0.9191	0.0008	0.9207
FANP 9x9	8	4.7	11	1.510	0.96	1.02	1.09	0.9294	0.0008	<b>0.9310</b>
FANP 9x9	6	4.2	11	1.510	0.96	1.02	1.09	0.9242	0.0010	0.9262
FANP 9x9	4	3.8	11	1.510	0.96	1.02	1.09	0.9264	0.0007	0.9278
FANP 9x9	2	3.5	11	1.510	0.96	1.02	1.09	0.9257	0.0007	0.9271
FANP 9x9	0	3.0	11	1.510	0.96	1.02	1.09	0.9214	0.0008	0.9230
GNF 9x9	10	5.0	11	1.510	0.96	1.02	1.09	0.9151	0.0008	0.9167
GNF 9x9	8	4.8	11	1.510	0.96	1.02	1.09	0.9368	0.0009	<b>0.9386</b>
GNF 9x9	6	4.2	11	1.510	0.96	1.02	1.09	0.9294	0.0009	0.9312
GNF 9x9	4	3.8	11	1.510	0.96	1.02	1.09	0.9333	0.0007	0.9347
GNF 9x9	2	3.5	11	1.510	0.96	1.02	1.09	0.9311	0.0008	0.9327
GNF 9x9	0	3.0	11	1.510	0.96	1.02	1.09	0.9290	0.0008	0.9306
GNF 8x8	7	5.0	11	1.6923	1.05	1.10	1.17	0.9356	0.0008	<b>0.9372</b>
GNF 8x8	6	4.7	11	1.6923	1.05	1.10	1.17	0.9323	0.0009	0.9341
GNF 8x8	4	4.1	11	1.6923	1.05	1.10	1.17	0.9305	0.0008	0.9321
GNF 8x8	2	3.7	11	1.6923	1.05	1.10	1.17	0.9321	0.0008	0.9337
GNF 8x8	0	3.1	11	1.6923	1.05	1.10	1.17	0.9311	0.0008	0.9327

a. Limiting case(s) shown in bold

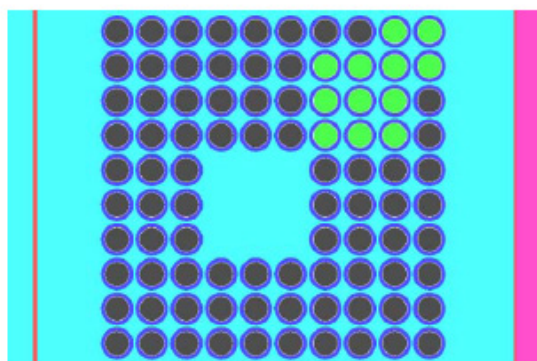




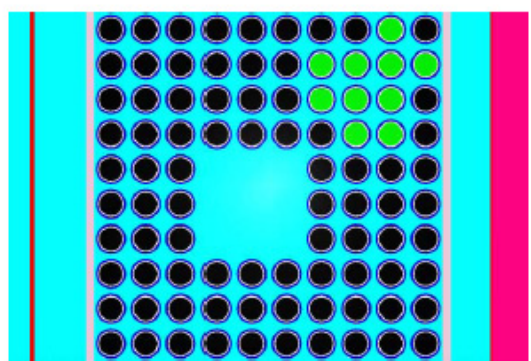
FANP 10x10 5.0 wt%  $^{235}\text{U}$ , Pattern B



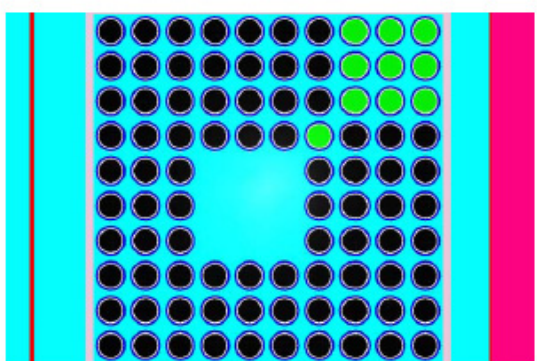
FANP 10x10 5.0 wt%  $^{235}\text{U}$ , Pattern F



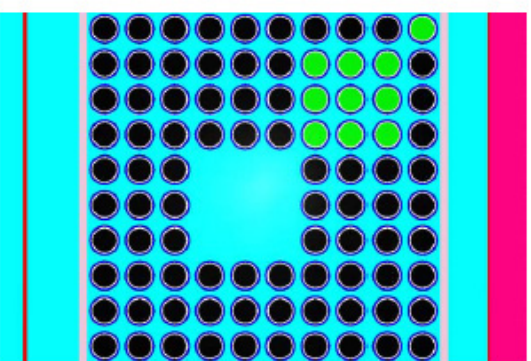
FANP 10x10 5.0 wt%  $^{235}\text{U}$ , Pattern G



FANP 10x10 4.6 wt%  $^{235}\text{U}$ , Pattern E




FANP 10x10 4.6 wt%  $^{235}\text{U}$ , Pattern F

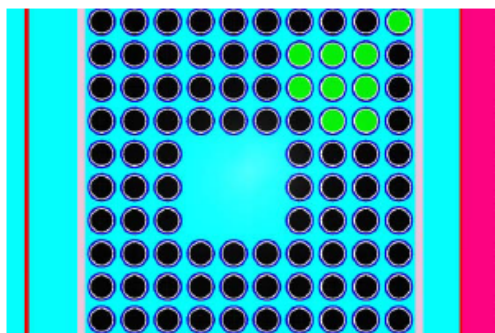


FANP 10x10 4.6 wt%  $^{235}\text{U}$ , Pattern G

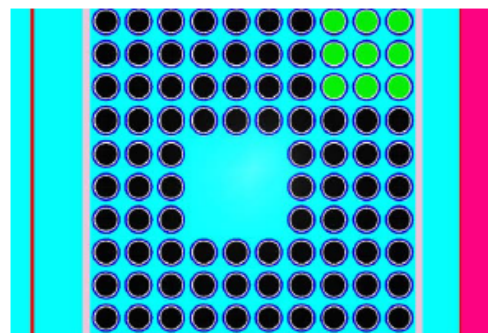
**Figure 6-30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies**



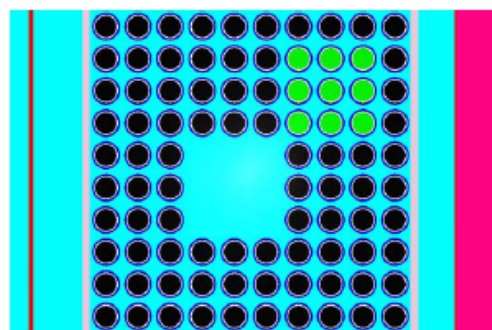
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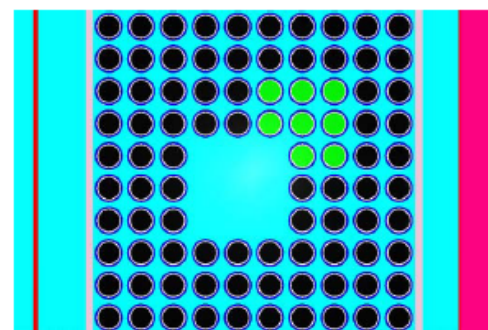
FANP 10x10 4.3 wt%  $^{235}\text{U}$ , Pattern E



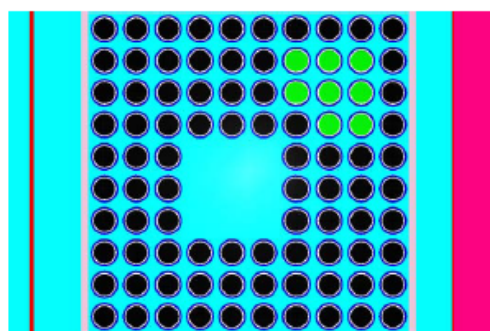
FANP 10x10 4.3 wt%  $^{235}\text{U}$ , Pattern F



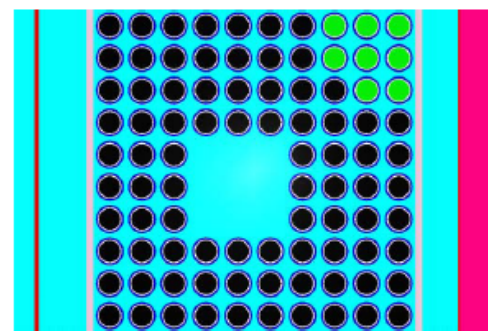
FANP 10x10 4.3 wt%  $^{235}\text{U}$ , Pattern G



FANP 10x10 4.2 wt%  $^{235}\text{U}$ , Pattern D




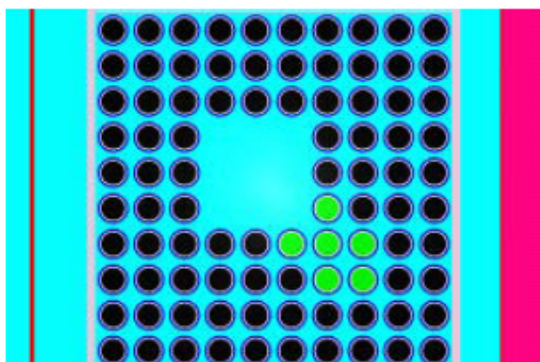
FANP 10x10 4.2 wt%  $^{235}\text{U}$ , Pattern E



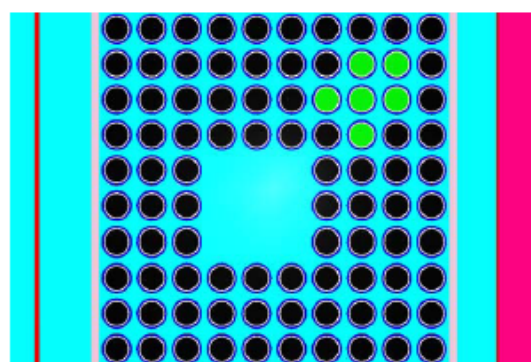
FANP 10x10 4.2 wt%  $^{235}\text{U}$ , Pattern F

**Figure 6 30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies (Continued)**

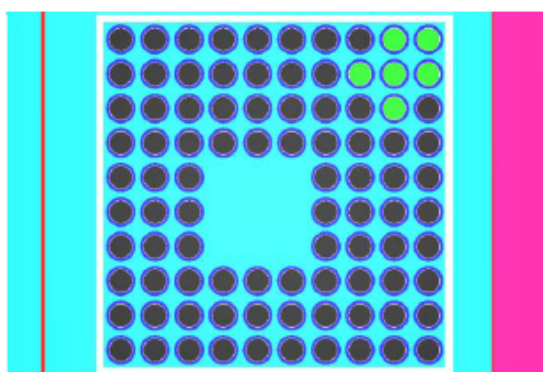
N° FS1-0014159	Rev. 7.0	<b>AREVA TN-B1</b> <b>Docket No. 71-9372</b> <b>Safety Analysis Report</b>	
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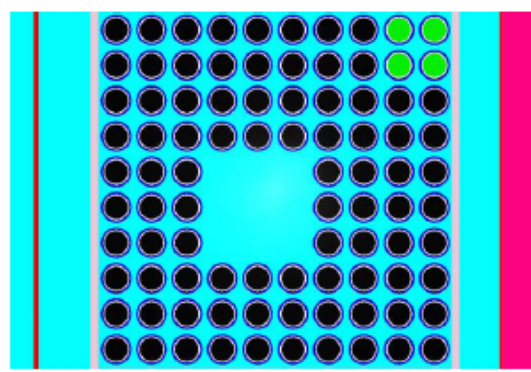
FANP 10x10 3.9 wt% <sup>235</sup>U, Pattern E



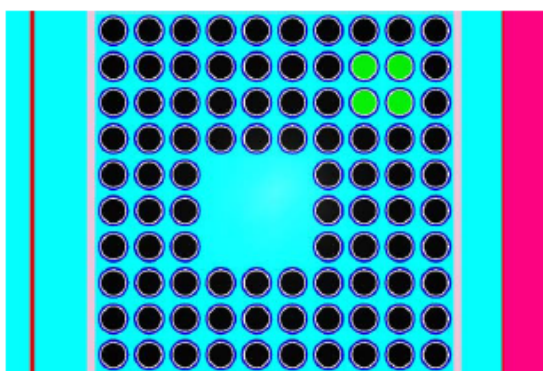
FANP 10x10 3.9 wt% <sup>235</sup>U, Pattern F



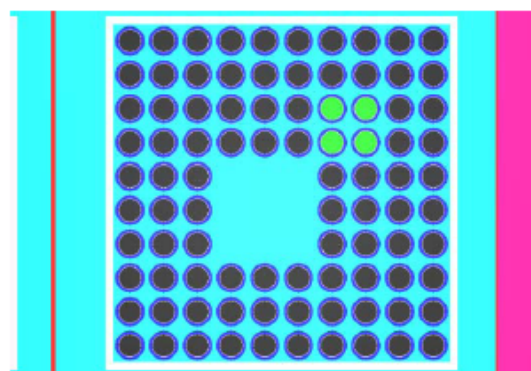
FANP 10x10 3.9 wt% <sup>235</sup>U, Pattern G



FANP 10x10 3.6 wt% <sup>235</sup>U, Pattern H



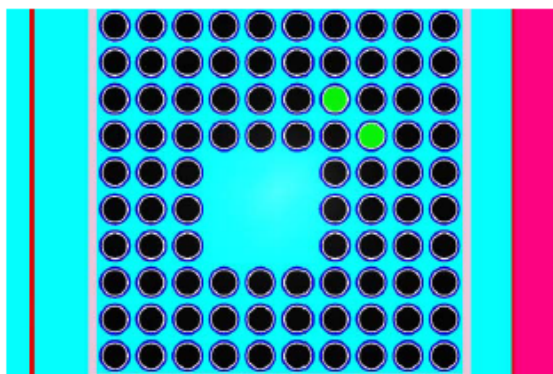
FANP 10x10 3.6 wt% <sup>235</sup>U, Pattern I



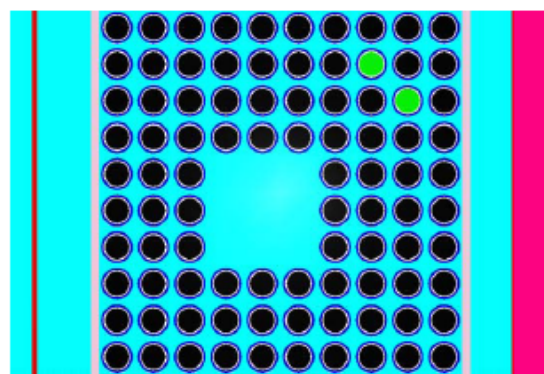
FANP 10x10 3.6 wt% <sup>235</sup>U, Pattern J

**Figure 6 30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies (Continued)**

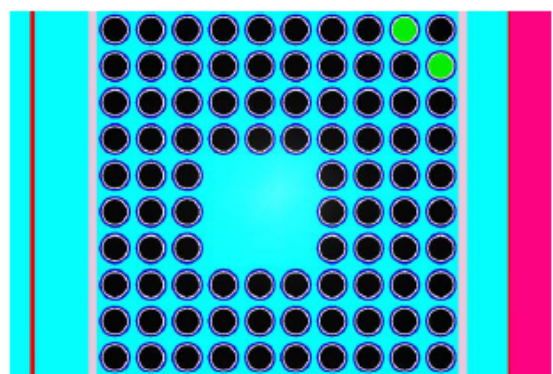




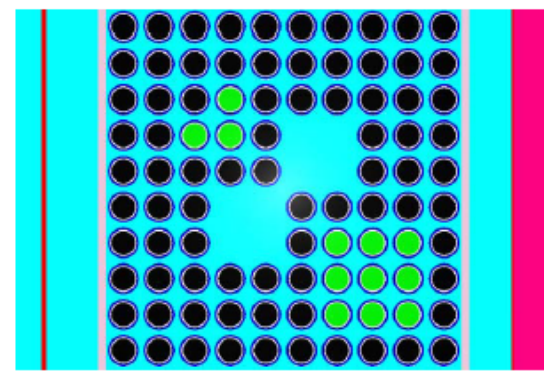
FANP 10x10 3.3 wt%  $^{235}\text{U}$ , Pattern F



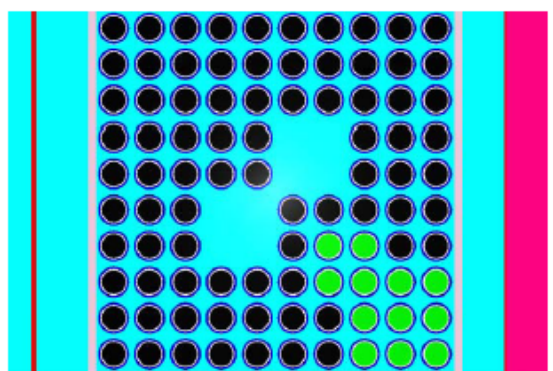
FANP 10x10 3.3 wt%  $^{235}\text{U}$ , Pattern G



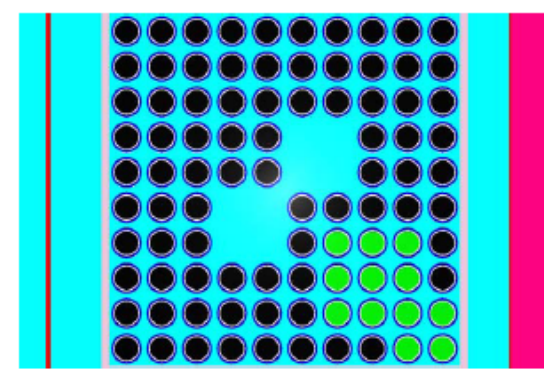
FANP 10x10 3.3 wt%  $^{235}\text{U}$ , Pattern H



GNF 10x10 5.0 wt%  $^{235}\text{U}$ , Pattern B




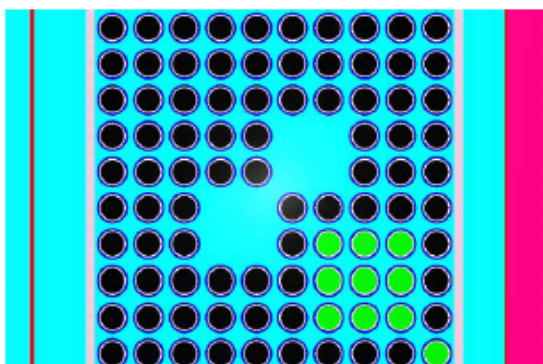
GNF 10x10 5.0 wt%  $^{235}\text{U}$ , Pattern F



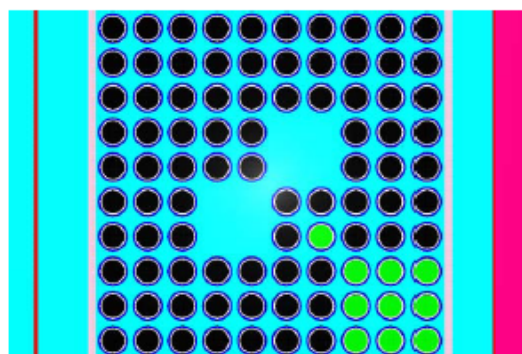
GNF 10x10 5.0 wt%  $^{235}\text{U}$ , Pattern H

**Figure 6 30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies (Continued)**

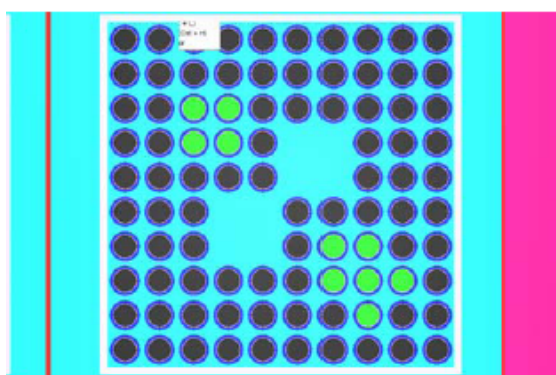
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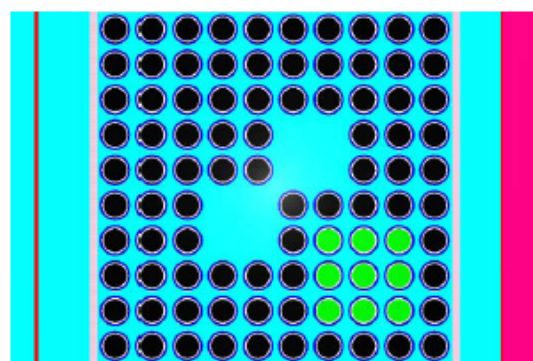
GNF 10x10 4.6 wt%  $^{235}\text{U}$ , Pattern F



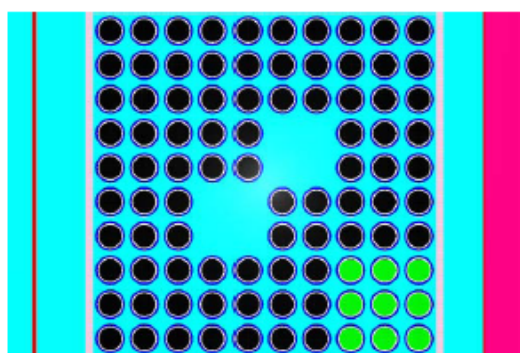
GNF 10x10 4.6 wt%  $^{235}\text{U}$ , Pattern G



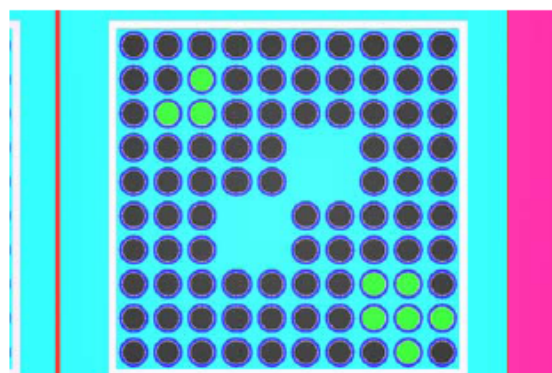
GNF 10x10 4.6 wt%  $^{235}\text{U}$ , Pattern I



GNF 10x10 4.3 wt%  $^{235}\text{U}$ , Pattern F



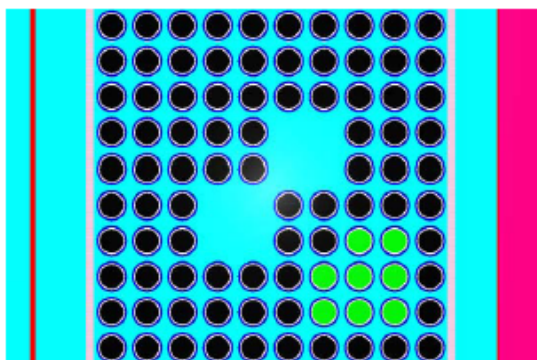
GNF 10x10 4.3 wt%  $^{235}\text{U}$ , Pattern G



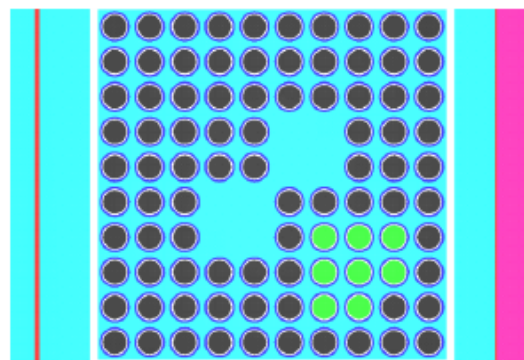
GNF 10x10 4.3 wt%  $^{235}\text{U}$ , Pattern J

**Figure 6 30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies (Continued)**

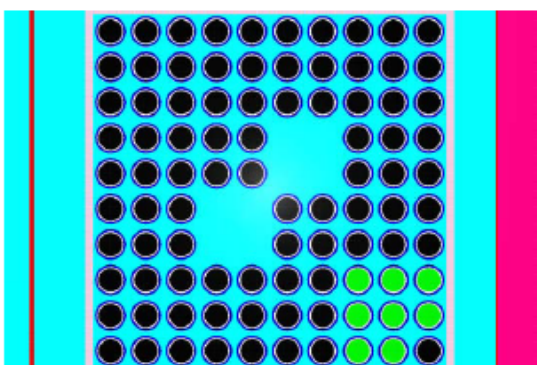




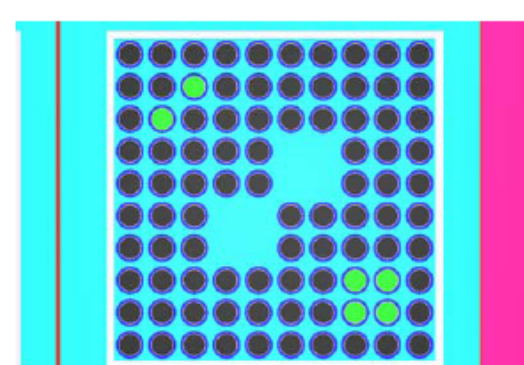
GNF 10x10 4.2 wt% <sup>235</sup>U, Pattern F



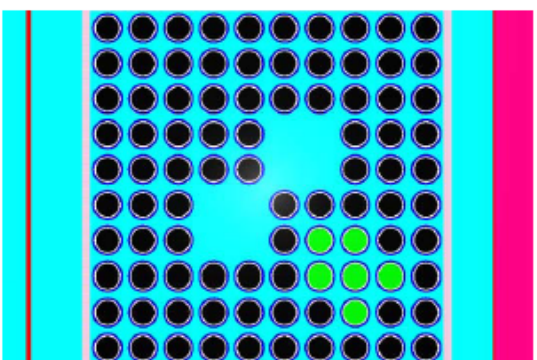
GNF 10x10 4.2 wt% <sup>235</sup>U, Pattern I



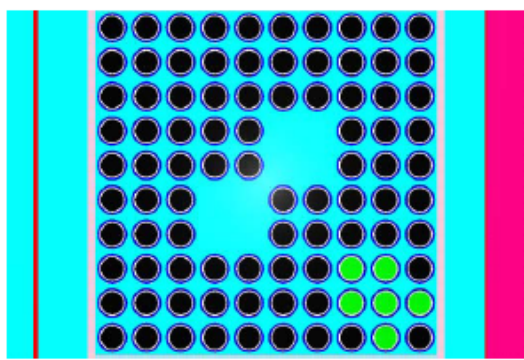
GNF 10x10 4.2 wt% <sup>235</sup>U, Pattern J



GNF 10x10 3.9 wt% <sup>235</sup>U, Pattern G




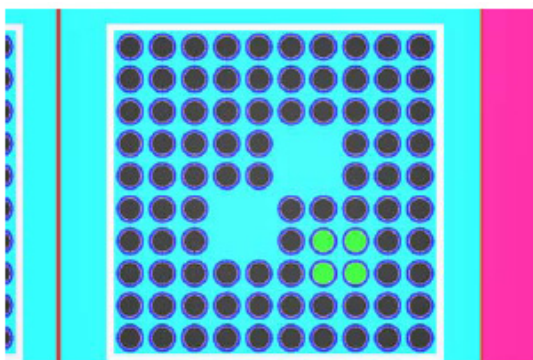
GNF 10x10 3.9 wt% <sup>235</sup>U, Pattern J



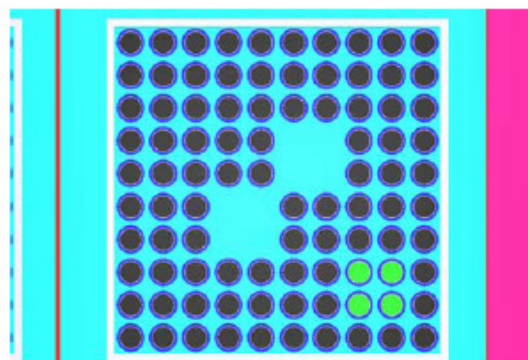
GNF 10x10 3.9 wt% <sup>235</sup>U, Pattern K

**Figure 6 30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies (Continued)**

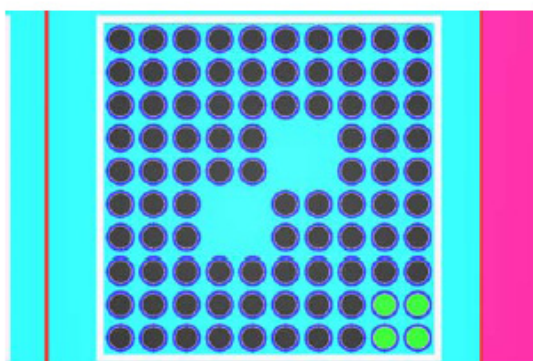
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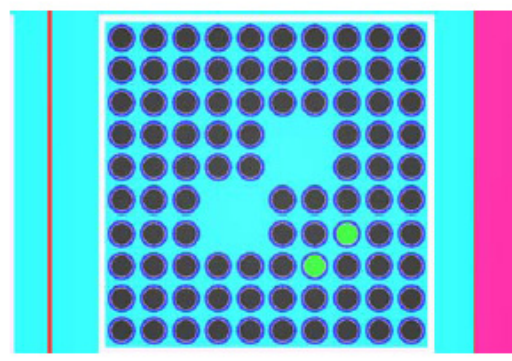
GNF 10x10 3.6 wt%  $^{235}\text{U}$ , Pattern F



GNF 10x10 3.6 wt%  $^{235}\text{U}$ , Pattern G




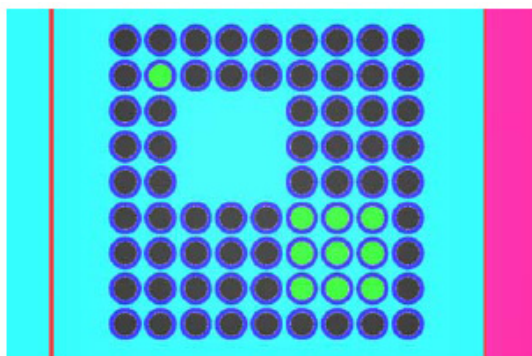
GNF 10x10 3.6 wt%  $^{235}\text{U}$ , Pattern H



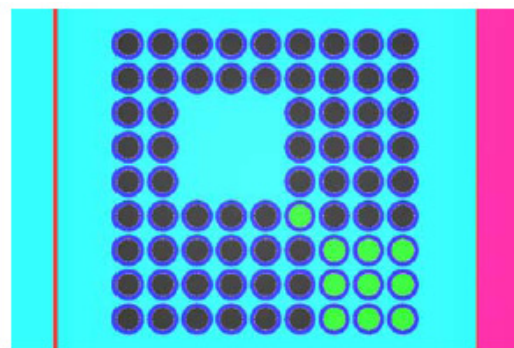
GNF 10x10 3.3 wt%  $^{235}\text{U}$ , Pattern A

**Figure 6 30 Gadolinia-Urania Fuel Rod Placement Pattern for 10x10 Fuel Assemblies (Continued)**

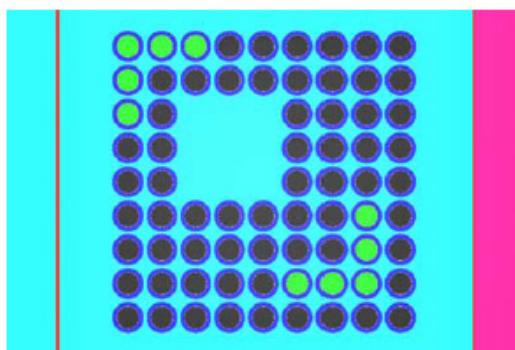
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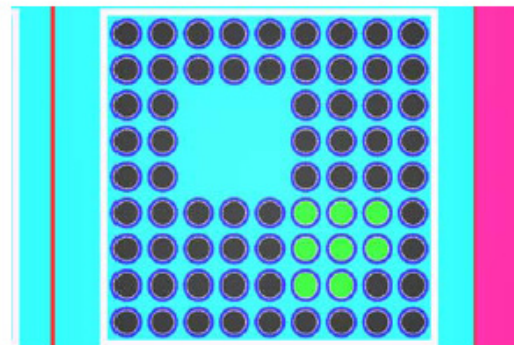
FANP 9x9 5.0 wt% <sup>235</sup>U, Pattern A



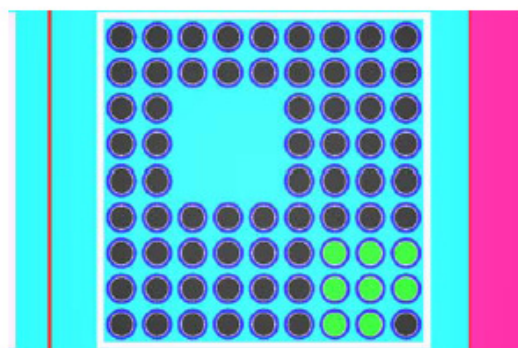
FANP 9x9 5.0 wt% <sup>235</sup>U, Pattern B



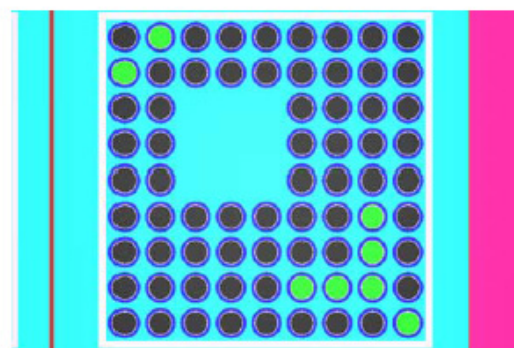
FANP 9x9 5.0 wt% <sup>235</sup>U, Pattern E



FANP 9x9 4.7 wt% <sup>235</sup>U, Pattern A



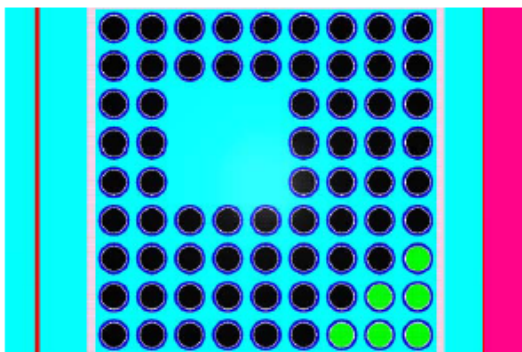
FANP 9x9 4.7 wt% <sup>235</sup>U, Pattern B



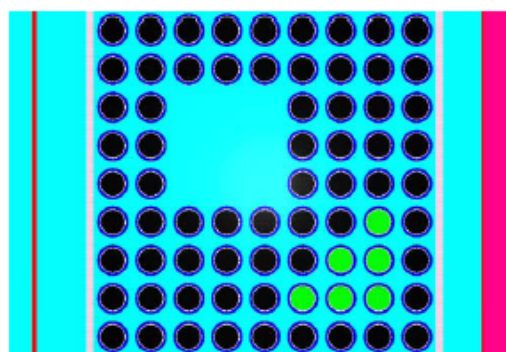
FANP 9x9 4.7 wt% <sup>235</sup>U, Pattern E

**Figure 6-31 Gadolinia-Urania Fuel Rod Placement Pattern for 9x9 Fuel Assemblies**

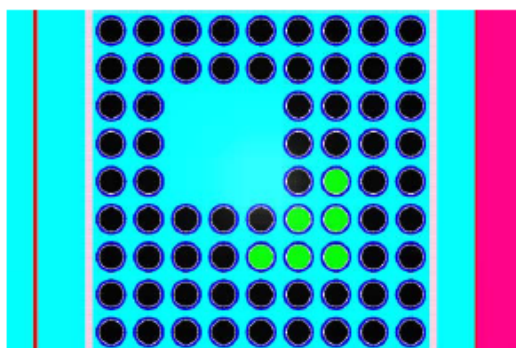




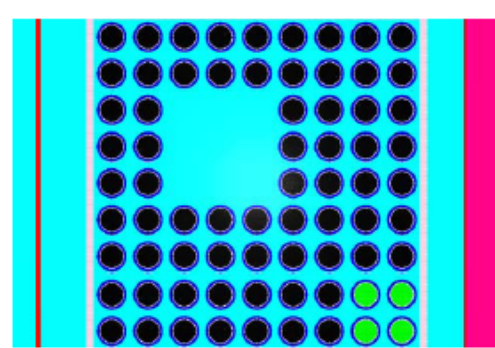
FANP 9x9 4.2 wt% U-235, Pattern A



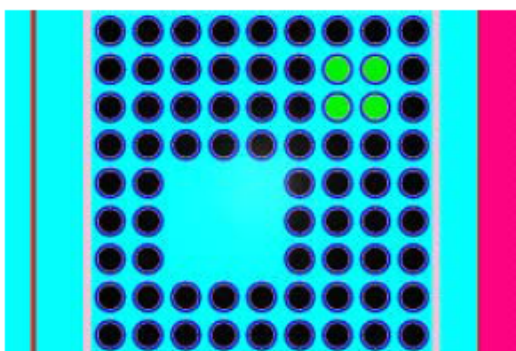
FANP 9x9 4.2 wt% U-235, Pattern B



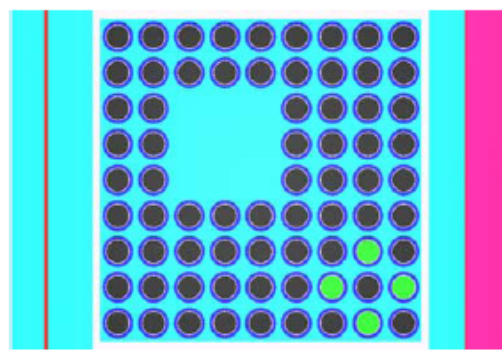
FANP 9x9 4.2 wt% U-235, Pattern C



FANP 9x9 3.8 wt% U-235, Pattern A

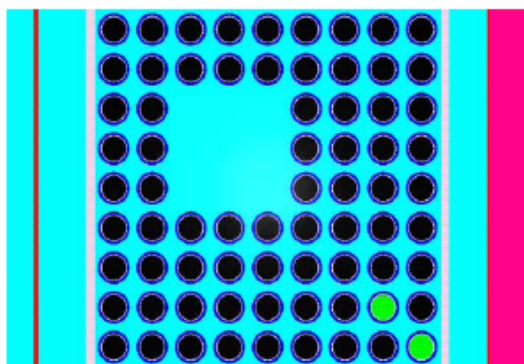


FANP 9x9 3.8 wt% U-235, Pattern B

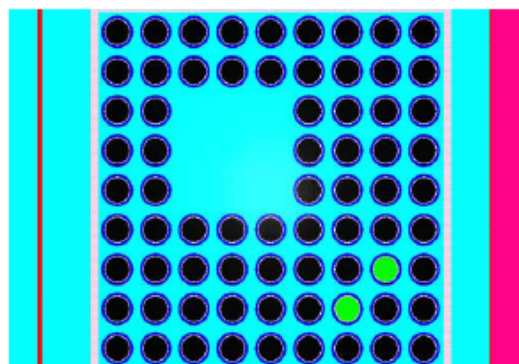


FANP 9x9 3.8 wt% U-235, Pattern F

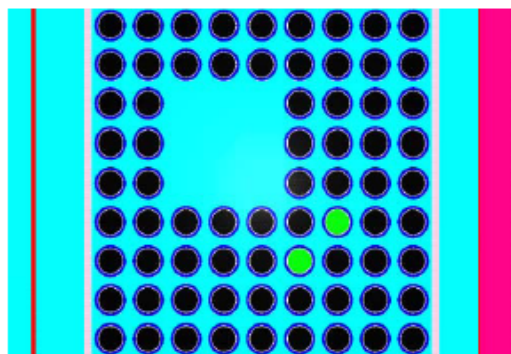
**Figure 6 31 Gadolinia-Urania Fuel Rod Placement Pattern for 9x9 Fuel Assemblies (Continued)**



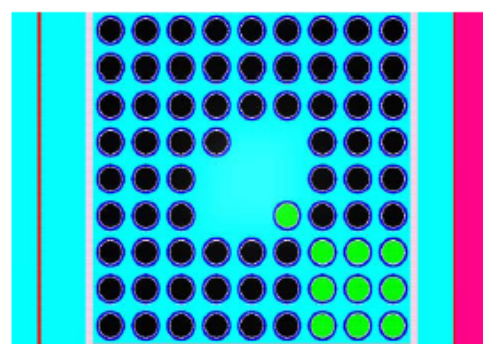
FANP 9x9 3.5 wt% U-235, Pattern B



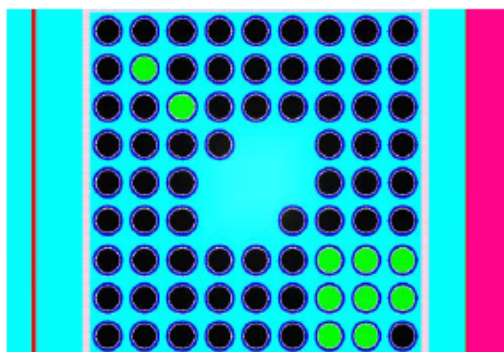
FANP 9x9 3.5 wt% U-235, Pattern C



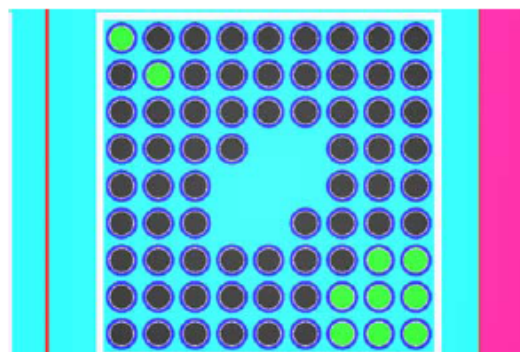
FANP 9x9 3.5 wt% U-235, Pattern D



GNF 9x9 5.0 wt% U-235, Pattern B




GNF 9x9 5.0 wt% U-235, Pattern G

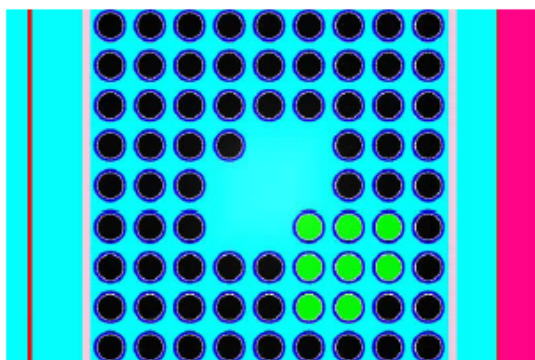


GNF 9x9 5.0 wt% U-235, Pattern H

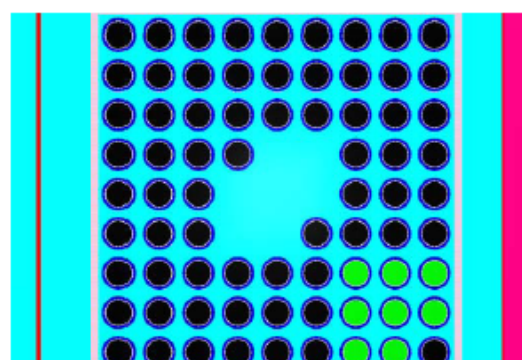
**Figure 6 31 Gadolinia-Urania Fuel Rod Placement Pattern for 9x9 Fuel Assemblies (Continued)**



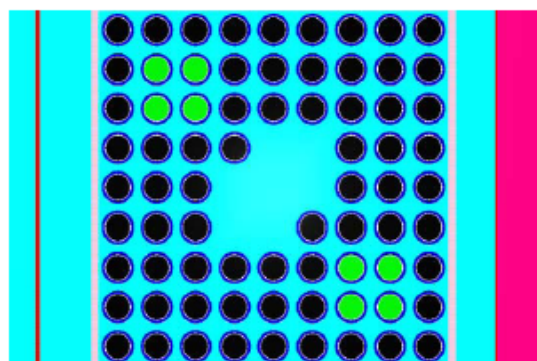
N° FS1-0014159	Rev. 7.0	<b>AREVA TN-B1</b> <b>Docket No. 71-9372</b> <b>Safety Analysis Report</b>	
Handling: None	Page 283/517		



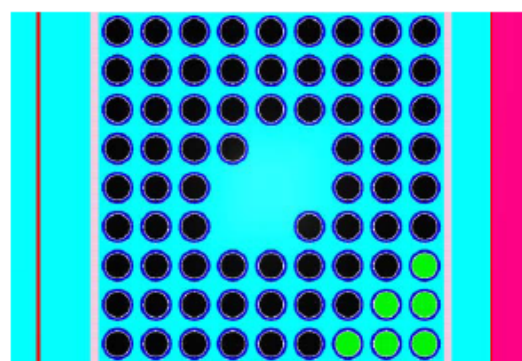
GNF 9x9 4.8 wt% U-235, Pattern A



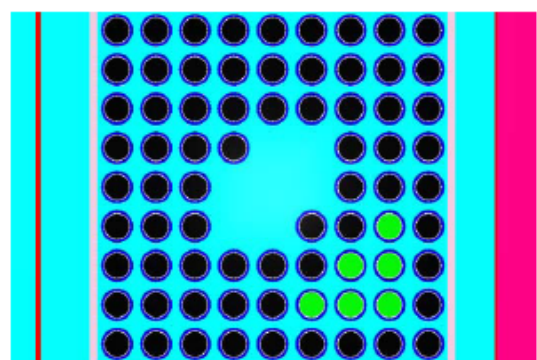
GNF 9x9 4.8 wt% U-235, Pattern B



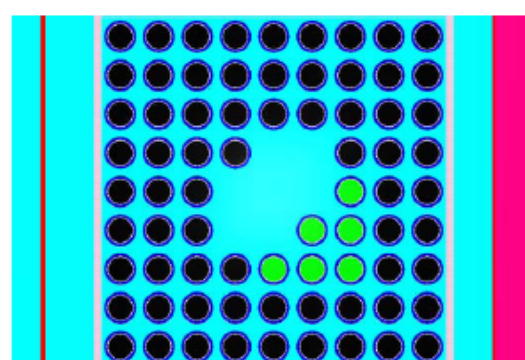
GNF 9x9 4.8 wt% U-235, Pattern H



GNF 9x9 4.2 wt% U-235, Pattern A

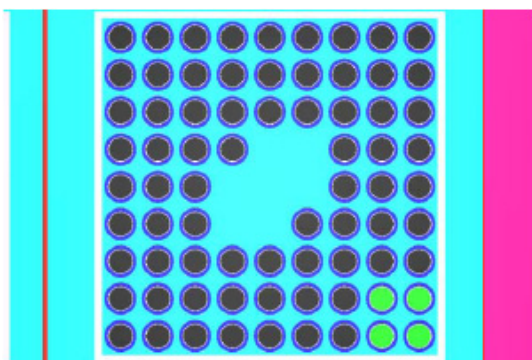


GNF 9x9 4.2 wt% U-235, Pattern B

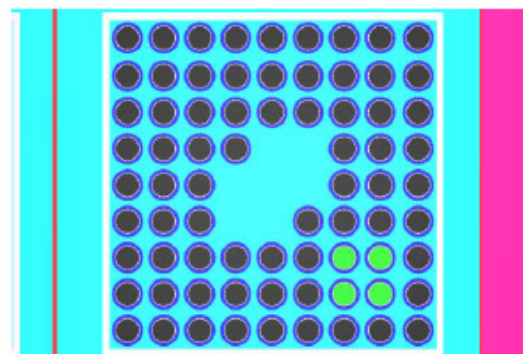


GNF 9x9 4.2 wt% U-235, Pattern C

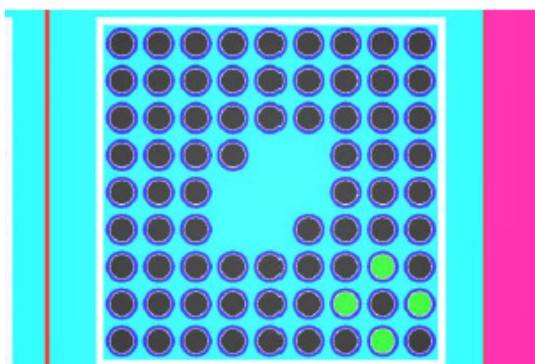
**Figure 6 31 Gadolinia-Urania Fuel Rod Placement Pattern for 9x9 Fuel Assemblies (Continued)**



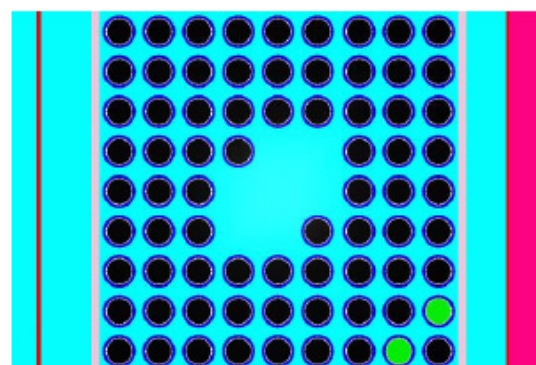
GNF 9x9 3.8 wt%  $^{235}\text{U}$ , Pattern A



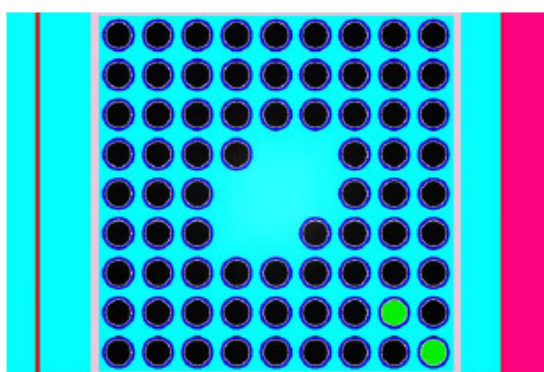
GNF 9x9 3.8 wt%  $^{235}\text{U}$ , Pattern B



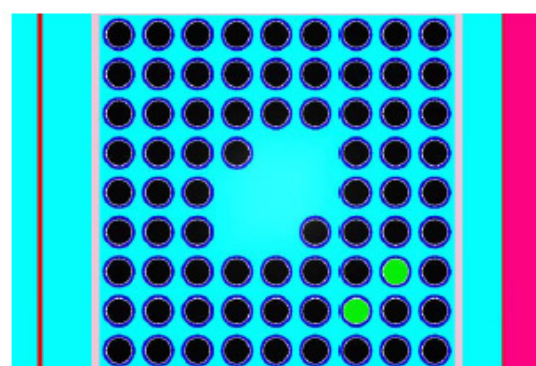
GNF 9x9 3.8 wt%  $^{235}\text{U}$ , Pattern F



GNF 9x9 3.5 wt%  $^{235}\text{U}$ , Pattern A



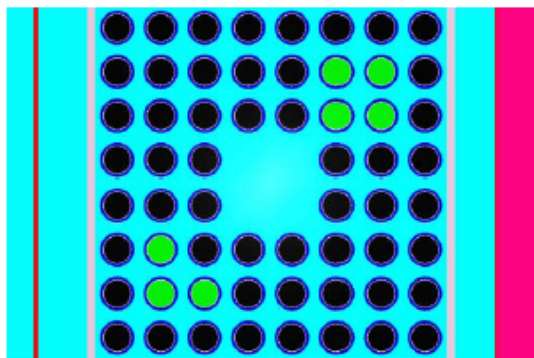
GNF 9x9 3.5 wt%  $^{235}\text{U}$ , Pattern B



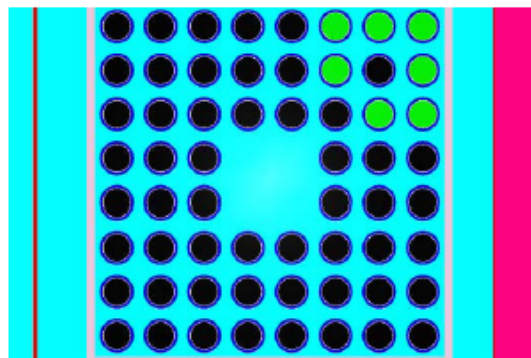
GNF 3.5 wt%  $^{235}\text{U}$ , Pattern C

**Figure 6 31 Gadolinia-Urania Fuel Rod Placement Pattern for 9x9 Fuel Assemblies (Continued)**

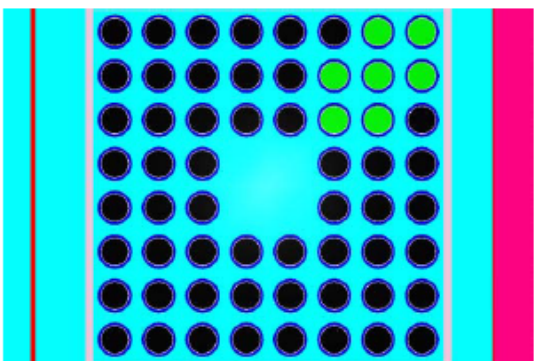




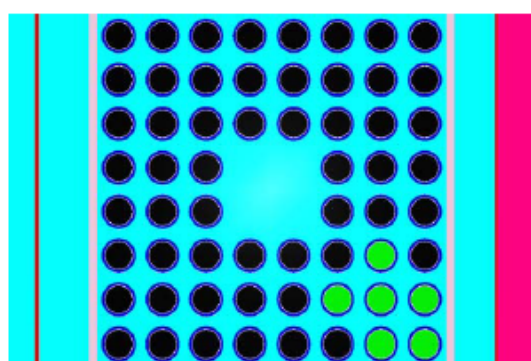
GNF 8x8 5.0 wt%  $^{235}\text{U}$ , Pattern E



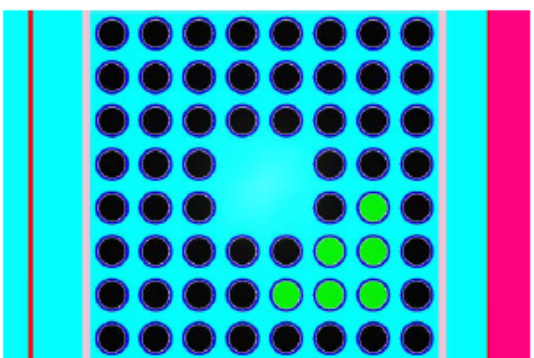
GNF 8x8 5.0 wt%  $^{235}\text{U}$ , Pattern H



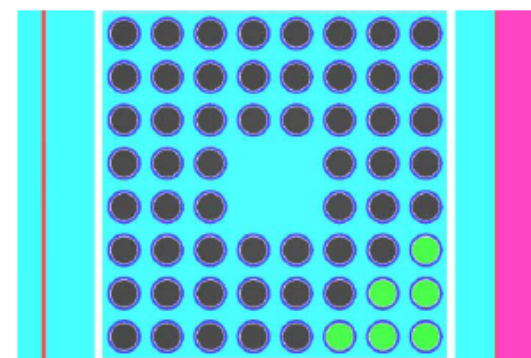
GNF 8x8 5.0 wt%  $^{235}\text{U}$ , Pattern I



GNF 8x8 4.7 wt%  $^{235}\text{U}$ , Pattern B

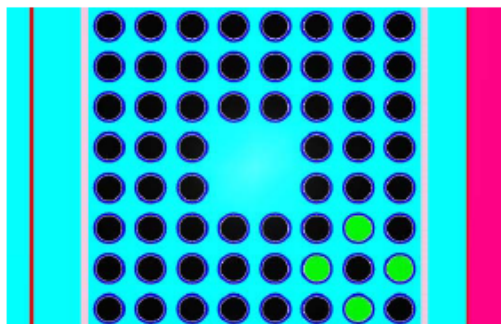


GNF 8x8 4.7 wt%  $^{235}\text{U}$ , Pattern C

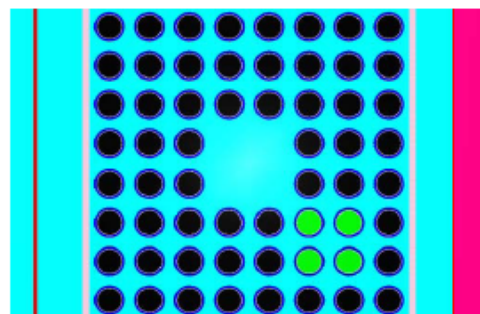


GNF 8x8 4.7 wt%  $^{235}\text{U}$ , Pattern D

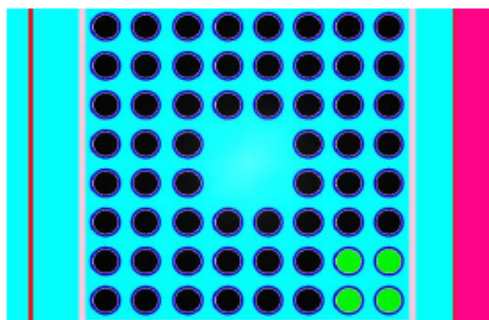
**Figure 6-32 Gadolinia-Urania Fuel Rod Placement Pattern for 8x8 Fuel Assemblies**



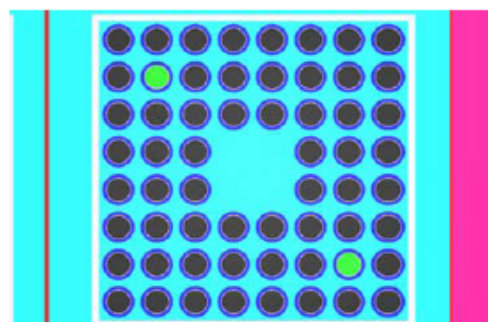
GNF 8x8 4.1 wt% <sup>235</sup>U, Pattern B



GNF 8x8 4.1 wt% <sup>235</sup>U, Pattern C



GNF 8x8 4.1 wt% <sup>235</sup>U, Pattern D



GNF 8x8 3.7 wt% <sup>235</sup>U, Pattern A

**Figure 6 32 Gadolinia-Urania Fuel Rod Placement Pattern for 8x8 Fuel Assemblies (Coninued)**




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#### 6.3.4.9. Part Length Fuel Rod Study (2N=448)


The FANP 10x10, FANP 9x9, GNF 10x10, and GNF 9x9 worst case designs are used to investigate the impact that part length fuel rods have on system reactivity. The worst case part length fuel rod patterns identified by performing scoping studies for the 10x10 designs are shown in Figure 6-33 and Figure 6-34. The worst case part length fuel rod patterns identified by performing scoping studies for the 9x9 designs are shown in Figure 6-35 and Figure 6-36. The fuel rod lengths for the part length rods are half that of the normal rod, and calculations showed that reducing the length further decreases system reactivity. To maintain the same amount of polyethylene when the part length rods are inserted, the polyethylene is redistributed to all rods in the assembly. The worst case models from the moderator density sensitivity study are used to conduct the part length fuel rod study, and the worst case fuel parameters listed in Table 6-17 are utilized. The part length fuel rod study results are contained in Table 6-18. All results for the FANP 9x9, the FANP 10x10, and the GNF 9x9 are below the USL of 0.94254. Several cases for the GNF 10x10 fuel design are above the USL of 0.94254. Therefore, an increased clad thickness is investigated for the 10x10 designs to reduce the system reactivity; these cases are included at the end of Table 6-18. The increased clad thickness for the 10x10 designs reduce system reactivity and all 10x10 results are below the USL of 0.94254. Comparing the results in Table 6-18 with those in Table 6-17 reveals the system reactivity remains about the same for the 9x9 fuel assembly designs with part length fuel rods. The FANP 10x10 and GNF 10x10 fuel designs are more reactive with the part length fuel rod configuration. Based on the results in Table 6-17 and Table 6-18:

- The maximum system reactivity with FANP 10x10 fuel assemblies having part length fuel rods and gadolinia-urania fuel is statistically greater than the maximum system reactivity with FANP 10x10 fuel assemblies having gadolinia-urania fuel and no part length fuel rods. The configuration that yields the highest  $k_{eff} + 2\sigma$  consists of fuel assemblies with a lattice average enrichment of 5.0 wt% U-235, 12 gadolinia-urania fuel rods enriched to 2.0 wt% gadolinia arranged in Pattern G, and 10 part length fuel rods. With the clad thickness for the fuel assemblies increased from 0.0335 cm to 0.0381 cm, the  $k_{eff} + 2\sigma$  for this configuration is 0.9394.
- The maximum system reactivity with GNF 10x10 fuel assemblies having part length fuel rods and gadolinia-urania fuel is statistically greater than the maximum system reactivity with GNF 10x10 fuel assemblies having gadolinia-urania fuel and no part length fuel rods. The configuration that yields the highest  $k_{eff} + 2\sigma$  consists of fuel assemblies with a lattice average enrichment of 5.0 wt% U-235, 12 gadolinia-urania fuel rods enriched to 2.0 wt% gadolinia arranged in Pattern H, and 12 part length fuel rods. With the clad thickness for the fuel assemblies increased from 0.0335 cm to 0.0381 cm, the  $k_{eff} + 2\sigma$  for this configuration is 0.9418.

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- Based on fuel parameter changes made to the 10x10 designs to lower reactivity, a 0.9338 cm upper bound clad ID, and a 1.01 cm lower bound clad OD are established for the GNF 10x10 parameter ranges. The 0.9330 cm upper bound clad ID and 1.00 cm lower bound clad OD may still be used for the FANP 10x10 design since the fuel assembly with this configuration remained below the USL of 0.94254.
- The most reactive FANP 9x9 configuration consists of fuel assemblies with a lattice average enrichment of 4.7 wt% U-235 and 8 gadolinia-urania fuel rods enriched to 2.0 wt% gadolinia arranged in Pattern A and 8 part length rods. The  $k_{eff} + 2\sigma$  for this configuration is 0.9303.
- The most reactive GNF 9x9 configuration consists of fuel assemblies with a lattice average enrichment of 4.7 wt% U-235 and 8 gadolinia-urania fuel rods enriched to 2.0 wt% gadolinia arranged in Pattern B and 8 part length fuel rods. The  $k_{eff} + 2\sigma$  for this configuration is 0.9407.
- The most reactive GNF 8x8 configuration consists of fuel assemblies with a lattice average enrichment of 5.0 wt% U-235, 7 gadolinia-urania fuel rods enriched to 2.0 wt% gadolinia arranged in Pattern I, and no part length fuel rods. The  $k_{eff} + 2\sigma$  for this configuration is 0.9372 (Table 6-17). The GNF 8x8 fuel assembly is not evaluated for part length fuel rods.


The GNF 10x10 assembly is chosen as the overall bounding fuel type since the  $k_{eff} + 2\sigma$  is among the largest numerical values, however, the system reactivity of the 10x10, and 9x9 worst case fuel assembly designs in the 14x2x16 TN-B1 container array are statistically indistinguishable.

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**Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations**

Assembly Type	Number of Part Length Rods	Gadolinia -Urania Fuel Rod Number	<sup>235</sup> U Enrichment (wt%)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	keff	σ	keff + 2σ
FANP 10x10	8	0	2.9	1.350	0.895	0.933	1.00	0.9228	0.0008	0.9244
FANP 10x10	8	2	3.3	1.350	0.895	0.933	1.00	0.9282	0.0008	0.9298
FANP 10x10	8	4	3.6	1.350	0.895	0.933	1.00	0.9332	0.0008	0.9348
FANP 10x10	8	6	3.9	1.350	0.895	0.933	1.00	0.9327	0.0008	0.9343
FANP 10x10	8	8	4.2	1.350	0.895	0.933	1.00	0.9367	0.0008	0.9383
FANP 10x10	8	9	4.3	1.350	0.895	0.933	1.00	0.9282	0.0008	0.9298
FANP 10x10	8	10	4.6	1.350	0.895	0.933	1.00	0.9363	0.0009	0.9381
FANP 10x10	8	12	5.0	1.350	0.895	0.933	1.00	0.9403	0.0008	<b>0.9419</b>
FANP 10x10	10	0	2.9	1.350	0.895	0.933	1.00	0.9224	0.0008	0.9240
FANP 10x10	10	2	3.3	1.350	0.895	0.933	1.00	0.9283	0.0008	0.9299
FANP 10x10	10	4	3.6	1.350	0.895	0.933	1.00	0.9330	0.0007	0.9344
FANP 10x10	10	6	3.9	1.350	0.895	0.933	1.00	0.9333	0.0008	0.9349
FANP 10x10	10	8	4.2	1.350	0.895	0.933	1.00	0.9367	0.0008	0.9383
FANP 10x10	10	9	4.3	1.350	0.895	0.933	1.00	0.9301	0.0008	0.9317
FANP 10x10	10	10	4.6	1.350	0.895	0.933	1.00	0.9379	0.0009	0.9397
FANP 10x10	10	12	5.0	1.350	0.895	0.933	1.00	0.9399	0.0008	<b>0.9415</b>
FANP 10x10	12	0	2.9	1.350	0.895	0.933	1.00	0.9234	0.0008	0.9250
FANP 10x10	12	2	3.3	1.350	0.895	0.933	1.00	0.9281	0.0008	0.9297
FANP 10x10	12	4	3.6	1.350	0.895	0.933	1.00	0.9329	0.0008	0.9345
FANP 10x10	12	6	3.9	1.350	0.895	0.933	1.00	0.9319	0.0008	0.9335
FANP 10x10	12	8	4.2	1.350	0.895	0.933	1.00	0.9356	0.0008	0.9372
FANP 10x10	12	9	4.3	1.350	0.895	0.933	1.00	0.9294	0.0007	0.9308
FANP 10x10	12	10	4.6	1.350	0.895	0.933	1.00	0.9371	0.0008	0.9387
FANP 10x10	12	12	5.0	1.350	0.895	0.933	1.00	0.9404	0.0009	<b>0.9422</b>
FANP 10x10	14	0	2.9	1.350	0.895	0.933	1.00	0.9225	0.0008	0.9241
FANP 10x10	14	2	3.3	1.350	0.895	0.933	1.00	0.9274	0.0008	0.9290
FANP 10x10	14	4	3.6	1.350	0.895	0.933	1.00	0.9326	0.0009	0.9344
FANP 10x10	14	6	3.9	1.350	0.895	0.933	1.00	0.9313	0.0008	0.9329
FANP 10x10	14	8	4.2	1.350	0.895	0.933	1.00	0.9348	0.0010	0.9368
FANP 10x10	14	9	4.3	1.350	0.895	0.933	1.00	0.9310	0.0008	0.9326
FANP 10x10	14	10	4.6	1.350	0.895	0.933	1.00	0.9371	0.0008	0.9387
FANP 10x10	14	12	5.0	1.350	0.895	0.933	1.00	0.9393	0.0009	<b>0.9411</b>


a. Limiting case(s) shown in bold

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**Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations (continued)**

Assembly Type	Number of Part Length Rods	Gadolinia -Urania Fuel Rod Number	<sup>235</sup> U Enrichment (wt%)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	keff	σ	keff + 2σ
GNF 10x10	8	0	2.9	1.350	0.895	0.933	1.00	0.9321	0.0007	0.9335
GNF 10x10	8	2	3.3	1.350	0.895	0.933	1.00	0.9327	0.0007	0.9341
GNF 10x10	8	4	3.6	1.350	0.895	0.933	1.00	0.9395	0.0010	0.9415
GNF 10x10	8	6	3.9	1.350	0.895	0.933	1.00	0.9367	0.0008	0.9383
GNF 10x10	8	8	4.2	1.350	0.895	0.933	1.00	0.9402	0.0008	<b>0.9418</b>
GNF 10x10	8	9	4.3	1.350	0.895	0.933	1.00	0.9369	0.0009	0.9387
GNF 10x10	8	10	4.6	1.350	0.895	0.933	1.00	0.9376	0.0009	0.9394
GNF 10x10	8	12	5.0	1.350	0.895	0.933	1.00	0.9386	0.0010	0.9406
GNF 10x10	10	0	2.9	1.350	0.895	0.933	1.00	0.9300	0.0008	0.9316
GNF 10x10	10	2	3.3	1.350	0.895	0.933	1.00	0.9319	0.0008	0.9335
GNF 10x10	10	4	3.6	1.350	0.895	0.933	1.00	0.9380	0.0009	0.9398
GNF 10x10	10	6	3.9	1.350	0.895	0.933	1.00	0.9347	0.0008	0.9363
GNF 10x10	10	8	4.2	1.350	0.895	0.933	1.00	0.9419	0.0010	<b>0.9439</b>
GNF 10x10	10	9	4.3	1.350	0.895	0.933	1.00	0.9374	0.0008	0.9390
GNF 10x10	10	10	4.6	1.350	0.895	0.933	1.00	0.9385	0.0009	0.9403
GNF 10x10	10	12	5.0	1.350	0.895	0.933	1.00	0.9412	0.0008	0.9428
GNF 10x10	12	0	2.9	1.350	0.895	0.933	1.00	0.9300	0.0007	0.9314
GNF 10x10	12	2	3.3	1.350	0.895	0.933	1.00	0.9316	0.0007	0.9330
GNF 10x10	12	4	3.6	1.350	0.895	0.933	1.00	0.9377	0.0009	0.9395
GNF 10x10	12	6	3.9	1.350	0.895	0.933	1.00	0.9352	0.0008	0.9368
GNF 10x10	12	8	4.2	1.350	0.895	0.933	1.00	0.9408	0.0009	0.9426
GNF 10x10	12	9	4.3	1.350	0.895	0.933	1.00	0.9374	0.0008	0.9390
GNF 10x10	12	10	4.6	1.350	0.895	0.933	1.00	0.9406	0.0009	0.9424
GNF 10x10	12	12	5.0	1.350	0.895	0.933	1.00	0.9415	0.0008	<b>0.9431</b>
GNF 10x10	14	0	2.9	1.350	0.895	0.933	1.00	0.9277	0.0008	0.9293
GNF 10x10	14	2	3.3	1.350	0.895	0.933	1.00	0.9305	0.0008	0.9321
GNF 10x10	14	4	3.6	1.350	0.895	0.933	1.00	0.9374	0.0009	0.9392
GNF 10x10	14	6	3.9	1.350	0.895	0.933	1.00	0.9347	0.0008	0.9363
GNF 10x10	14	8	4.2	1.350	0.895	0.933	1.00	0.9401	0.0009	<b>0.9419</b>
GNF 10x10	14	9	4.3	1.350	0.895	0.933	1.00	0.9370	0.0009	0.9388
GNF 10x10	14	10	4.6	1.350	0.895	0.933	1.00	0.9381	0.0009	0.9399
GNF 10x10	14	12	5.0	1.350	0.895	0.933	1.00	0.9401	0.0008	0.9417


a. Limiting case(s) shown in bold

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**Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations (continued)**

Assembly Type	Number of Part Length Rods	Gadolinia -Urania Fuel Rod Number	<sup>235</sup> U Enrichment (wt%)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	keff	σ	keff + 2σ
FANP 9x9	8	0	3.0	1.510	0.96	1.02	1.09	0.9168	0.0008	0.9184
FANP 9x9	8	2	3.5	1.510	0.96	1.02	1.09	0.9219	0.0008	0.9235
FANP 9x9	8	4	3.8	1.510	0.96	1.02	1.09	0.9234	0.0009	0.9252
FANP 9x9	8	6	4.2	1.510	0.96	1.02	1.09	0.9227	0.0007	0.9241
FANP 9x9	8	8	4.7	1.510	0.96	1.02	1.09	0.9287	0.0008	<b>0.9303</b>
FANP 9x9	8	10	5.0	1.510	0.96	1.02	1.09	0.9165	0.0008	0.9181
FANP 9x9	10	0	3.0	1.510	0.96	1.02	1.09	0.9139	0.0008	0.9155
FANP 9x9	10	2	3.5	1.510	0.96	1.02	1.09	0.9195	0.0008	0.9211
FANP 9x9	10	4	3.8	1.510	0.96	1.02	1.09	0.9189	0.0008	0.9205
FANP 9x9	10	6	4.2	1.510	0.96	1.02	1.09	0.9208	0.0008	0.9224
FANP 9x9	10	8	4.7	1.510	0.96	1.02	1.09	0.9256	0.0009	<b>0.9274</b>
FANP 9x9	10	10	5.0	1.510	0.96	1.02	1.09	0.9135	0.0009	0.9153
FANP 9x9	12	0	3.0	1.510	0.96	1.02	1.09	0.9100	0.0007	0.9114
FANP 9x9	12	2	3.5	1.510	0.96	1.02	1.09	0.9155	0.0007	0.9169
FANP 9x9	12	4	3.8	1.510	0.96	1.02	1.09	0.9168	0.0008	0.9184
FANP 9x9	12	6	4.2	1.510	0.96	1.02	1.09	0.9147	0.0007	0.9161
FANP 9x9	12	8	4.7	1.510	0.96	1.02	1.09	0.9208	0.0008	<b>0.9224</b>
FANP 9x9	12	10	5.0	1.510	0.96	1.02	1.09	0.9087	0.0009	0.9105
GNF 9x9	8	0	3.0	1.510	0.96	1.02	1.09	0.9261	0.0008	0.9277
GNF 9x9	8	2	3.5	1.510	0.96	1.02	1.09	0.9311	0.0008	0.9327
GNF 9x9	8	4	3.8	1.510	0.96	1.02	1.09	0.9303	0.0008	0.9319
GNF 9x9	8	6	4.2	1.510	0.96	1.02	1.09	0.9293	0.0008	0.9309
GNF 9x9	8	8	4.7	1.510	0.96	1.02	1.09	0.9391	0.0008	<b>0.9407</b>
GNF 9x9	8	10	5.0	1.510	0.96	1.02	1.09	0.9140	0.0008	0.9156
GNF 9x9	10	0	3.0	1.510	0.96	1.02	1.09	0.9249	0.0009	0.9267
GNF 9x9	10	2	3.5	1.510	0.96	1.02	1.09	0.9315	0.0008	0.9331
GNF 9x9	10	4	3.8	1.510	0.96	1.02	1.09	0.9287	0.0008	0.9303
GNF 9x9	10	6	4.2	1.510	0.96	1.02	1.09	0.9297	0.0009	0.9315
GNF 9x9	10	8	4.7	1.510	0.96	1.02	1.09	0.9377	0.0008	<b>0.9393</b>
GNF 9x9	10	10	5.0	1.510	0.96	1.02	1.09	0.9048	0.0008	0.9064
GNF 9x9	12	0	3.0	1.510	0.96	1.02	1.09	0.9235	0.0008	0.9251

a. Limiting case(s) shown in bold


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**Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations (continued)**

Assembly Type	Number of Part Length Rods	Gadolinia -Urania Fuel Rod Number	235U Enrichment (wt%)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	keff	$\sigma$	keff + 2 $\sigma$
GNF 9x9	12	2	3.5	1.510	0.96	1.02	1.09	0.9294	0.0009	0.9312
GNF 9x9	12	4	3.8	1.510	0.96	1.02	1.09	0.9288	0.0009	0.9306
GNF 9x9	12	6	4.2	1.510	0.96	1.02	1.09	0.9263	0.0008	0.9279
GNF 9x9	12	8	4.7	1.510	0.96	1.02	1.09	0.9370	0.0009	<b>0.9388</b>
GNF 9x9	12	10	5.0	1.510	0.96	1.02	1.09	0.9056	0.0008	0.9072
FANP 10x10	8	0	2.9	1.350	0.895	0.9338	1.01	0.9203	0.0008	0.9219
FANP 10x10	8	2	3.3	1.350	0.895	0.9338	1.01	0.9150	0.0008	0.9166
FANP 10x10	8	4	3.6	1.350	0.895	0.9338	1.01	0.9290	0.0008	0.9306
FANP 10x10	8	6	3.9	1.350	0.895	0.9338	1.01	0.9303	0.0008	0.9319
FANP 10x10	8	8	4.2	1.350	0.895	0.9338	1.01	0.9292	0.0008	0.9308
FANP 10x10	8	9	4.3	1.350	0.895	0.9338	1.01	0.9293	0.0008	0.9309
FANP 10x10	8	10	4.6	1.350	0.895	0.9338	1.01	0.9335	0.0008	0.9351
FANP 10x10	8	12	5.0	1.350	0.895	0.9338	1.01	0.9353	0.0009	<b>0.9371</b>
FANP 10x10	10	0	2.9	1.350	0.895	0.9338	1.01	0.9218	0.0008	0.9234
FANP 10x10	10	2	3.3	1.350	0.895	0.9338	1.01	0.9265	0.0008	0.9281
FANP 10x10	10	4	3.6	1.350	0.895	0.9338	1.01	0.9320	0.0008	0.9336
FANP 10x10	10	6	3.9	1.350	0.895	0.9338	1.01	0.9311	0.0008	0.9327
FANP 10x10	10	8	4.2	1.350	0.895	0.9338	1.01	0.9345	0.0008	0.9361
FANP 10x10	10	9	4.3	1.350	0.895	0.9338	1.01	0.9296	0.0009	0.9314
FANP 10x10	10	10	4.6	1.350	0.895	0.9338	1.01	0.9369	0.0009	0.9387
FANP 10x10	10	12	5.0	1.350	0.895	0.9338	1.01	0.9376	0.0009	<b>0.9394</b>
FANP 10x10	12	0	2.9	1.350	0.895	0.9338	1.01	0.9216	0.0008	0.9232
FANP 10x10	12	2	3.3	1.350	0.895	0.9338	1.01	0.9256	0.0008	0.9272
FANP 10x10	12	4	3.6	1.350	0.895	0.9338	1.01	0.9314	0.0009	0.9332
FANP 10x10	12	6	3.9	1.350	0.895	0.9338	1.01	0.9319	0.0007	0.9333
FANP 10x10	12	8	4.2	1.350	0.895	0.9338	1.01	0.9345	0.0008	0.9361
FANP 10x10	12	9	4.3	1.350	0.895	0.9338	1.01	0.9277	0.0008	0.9293
FANP 10x10	12	10	4.6	1.350	0.895	0.9338	1.01	0.9347	0.0009	0.9365
FANP 10x10	12	12	5.0	1.350	0.895	0.9338	1.01	0.9370	0.0009	<b>0.9388</b>
FANP 10x10	14	0	2.9	1.350	0.895	0.9338	1.01	0.9207	0.0008	0.9223
FANP 10x10	14	2	3.3	1.350	0.895	0.9338	1.01	0.9247	0.0009	0.9265
FANP 10x10	14	4	3.6	1.350	0.895	0.9338	1.01	0.9291	0.0008	0.9307

a. Limiting case(s) shown in bold




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**Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations (continued)**

Assembly Type	Number of Part Length Rods	Gadolinia -Urania Fuel Rod Number	<sup>235</sup> U Enrichment (wt%)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	keff	σ	keff + 2σ
FANP 10x10	14	6	3.9	1.350	0.895	0.9338	1.01	0.9301	0.0009	0.9319
FANP 10x10	14	8	4.2	1.350	0.895	0.9338	1.01	0.9324	0.0008	0.9340
FANP 10x10	14	9	4.3	1.350	0.895	0.9338	1.01	0.9293	0.0008	0.9309
FANP 10x10	14	10	4.6	1.350	0.895	0.9338	1.01	0.9352	0.0008	0.9368
FANP 10x10	14	12	5.0	1.350	0.895	0.9338	1.01	0.9370	0.0009	<b>0.9388</b>
GNF 10x10	8	0	2.9	1.350	0.895	0.9338	1.01	0.9292	0.0008	0.9308
GNF 10x10	8	2	3.3	1.350	0.895	0.9338	1.01	0.9296	0.0009	0.9314
GNF 10x10	8	4	3.6	1.350	0.895	0.9338	1.01	0.9357	0.0010	0.9377
GNF 10x10	8	6	3.9	1.350	0.895	0.9338	1.01	0.9354	0.0009	0.9372
GNF 10x10	8	8	4.2	1.350	0.895	0.9338	1.01	0.9399	0.0008	<b>0.9415</b>
GNF 10x10	8	9	4.3	1.350	0.895	0.9338	1.01	0.9346	0.0010	0.9366
GNF 10x10	8	10	4.6	1.350	0.895	0.9338	1.01	0.9376	0.0009	0.9394
GNF 10x10	8	12	5.0	1.350	0.895	0.9338	1.01	0.9375	0.0008	0.9391
GNF 10x10	10	0	2.9	1.350	0.895	0.9338	1.01	0.9292	0.0008	0.9308
GNF 10x10	10	2	3.3	1.350	0.895	0.9338	1.01	0.9296	0.0008	0.9312
GNF 10x10	10	4	3.6	1.350	0.895	0.9338	1.01	0.9371	0.0008	0.9387
GNF 10x10	10	6	3.9	1.350	0.895	0.9338	1.01	0.9370	0.0008	0.9386
GNF 10x10	10	8	4.2	1.350	0.895	0.9338	1.01	0.9372	0.0008	0.9388
GNF 10x10	10	9	4.3	1.350	0.895	0.9338	1.01	0.9363	0.0009	0.9381
GNF 10x10	10	10	4.6	1.350	0.895	0.9338	1.01	0.9345	0.0009	0.9363
GNF 10x10	10	12	5.0	1.350	0.895	0.9338	1.01	0.9375	0.0008	<b>0.9391</b>
GNF 10x10	12	0	2.9	1.350	0.895	0.9338	1.01	0.9276	0.0008	0.9292
GNF 10x10	12	2	3.3	1.350	0.895	0.9338	1.01	0.9309	0.0008	0.9325
GNF 10x10	12	4	3.6	1.350	0.895	0.9338	1.01	0.9373	0.0009	0.9391
GNF 10x10	12	6	3.9	1.350	0.895	0.9338	1.01	0.9347	0.0009	0.9365
GNF 10x10	12	8	4.2	1.350	0.895	0.9338	1.01	0.9374	0.0009	0.9392
GNF 10x10	12	9	4.3	1.350	0.895	0.9338	1.01	0.9333	0.0009	0.9351
GNF 10x10	12	10	4.6	1.350	0.895	0.9338	1.01	0.9378	0.0008	0.9394
GNF 10x10	12	12	5.0	1.350	0.895	0.9338	1.01	0.9404	0.0007	<b>0.9418</b>
GNF 10x10	14	0	2.9	1.350	0.895	0.9338	1.01	0.9261	0.0008	0.9277
GNF 10x10	14	2	3.3	1.350	0.895	0.9338	1.01	0.9299	0.0008	0.9315


a. Limiting case(s) shown in bold

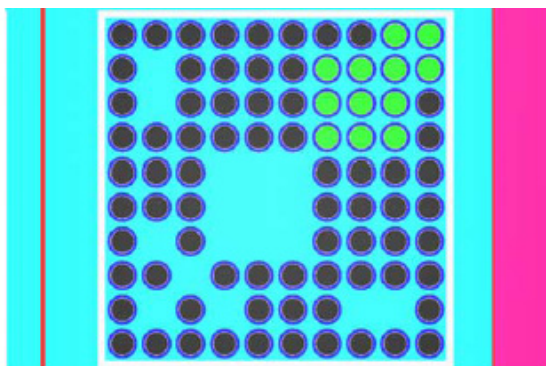
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**Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations (continued)**

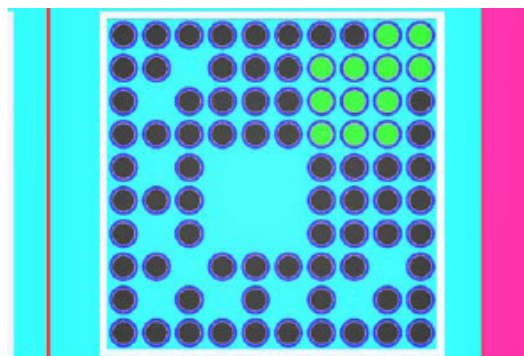
Assembly Type	Number of Part Length Rods	Gadolinia -Urania Fuel Rod Number	<sup>235</sup> U Enrichment (wt%)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
GNF 10x10	14	4	3.6	1.350	0.895	0.9338	1.01	0.9345	0.0008	0.9361
GNF 10x10	14	6	3.9	1.350	0.895	0.9338	1.01	0.9351	0.0009	0.9369
GNF 10x10	14	8	4.2	1.350	0.895	0.9338	1.01	0.9376	0.0009	0.9394
GNF 10x10	14	9	4.3	1.350	0.895	0.9338	1.01	0.9353	0.0008	0.9369
GNF 10x10	14	10	4.6	1.350	0.895	0.9338	1.01	0.9368	0.0009	0.9386
GNF 10x10	14	12	5.0	1.350	0.895	0.9338	1.01	0.9398	0.0008	<b>0.9414</b>

a. Limiting case(s) shown in bold

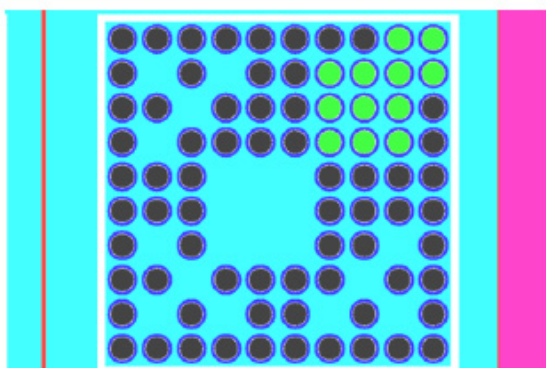
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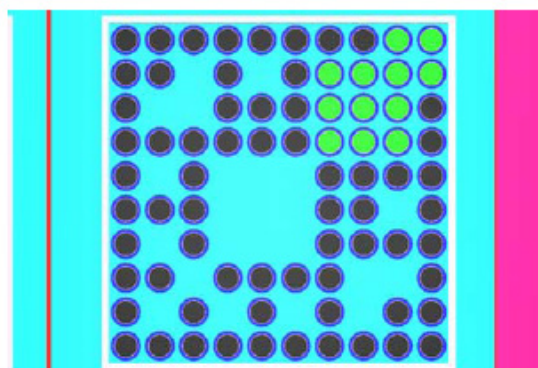
FANP 10x10 5.0 wt%  $^{235}\text{U}$ , 8 Part Length Rods



FANP 10x10 5.0 wt%  $^{235}\text{U}$ , 10 Part Length Rods

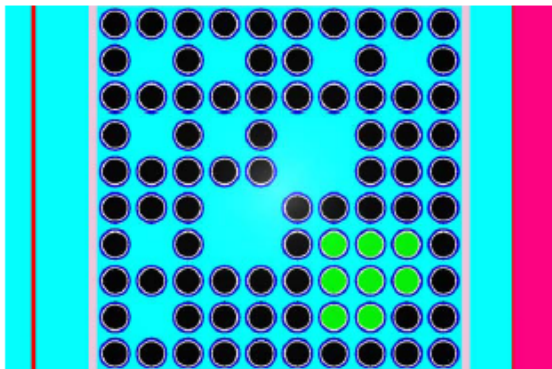


FANP 10x10 5.0 wt%  $^{235}\text{U}$ , 12 Part Length Rods

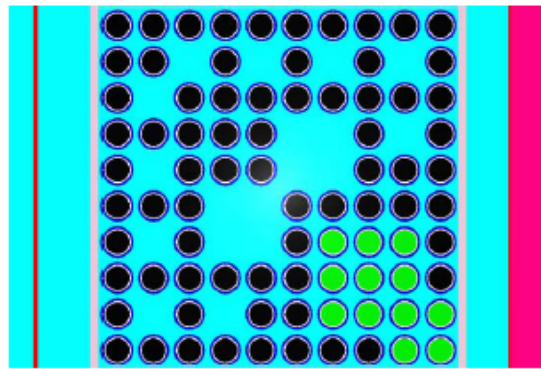


FANP 10x10 5.0 wt%  $^{235}\text{U}$ , 14 Part Length Rods

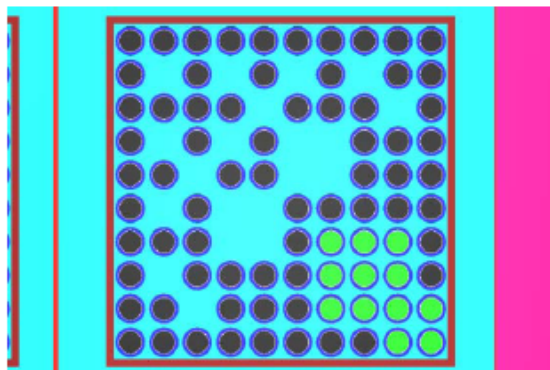
**Figure 6-33 FANP 10x10 Worst Case Fuel Parameters Model with Part Length Fuel Rods**



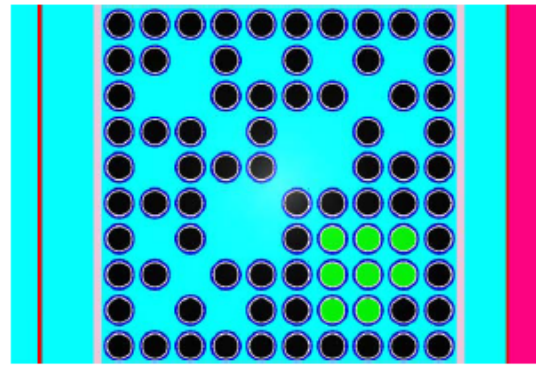
GNF 10x10 4.2 wt%  $^{235}\text{U}$ , 8 Part Length Rods



GNF 10x10 5.0 wt%  $^{235}\text{U}$ , 10 Part Length Rods




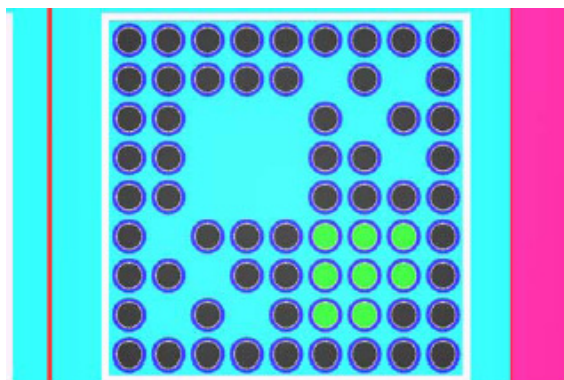
GNF 10x10 5.0 wt%  $^{235}\text{U}$ , 12 Part Length Rods



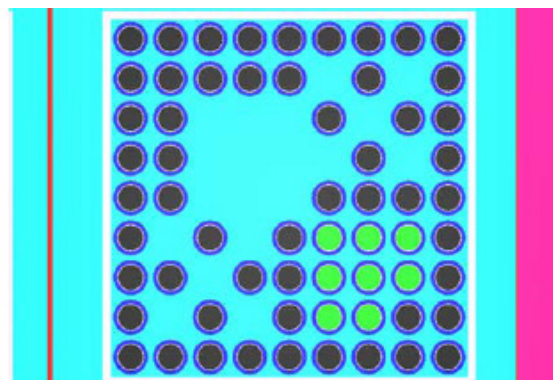
GNF 10x10 4.2 wt%  $^{235}\text{U}$ , 14 Part Length Rods

**Figure 6-34 GNF 10x10 Worst Case Fuel Parameters Model with Part Length Fuel Rods**

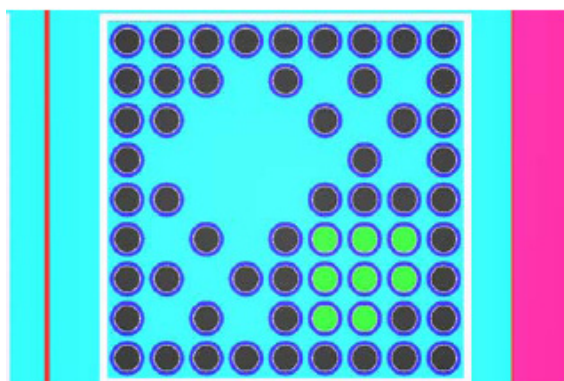
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FANP 9x9 4.7 wt%  $^{235}\text{U}$ , 8 Part Length Rods




FANP 9x9 4.7 wt%  $^{235}\text{U}$ , 10 Part Length Rods

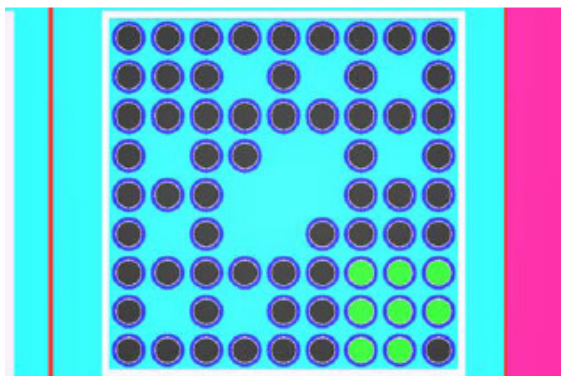


FANP 9x9 4.7 wt%  $^{235}\text{U}$ , 12 Part Length Rods

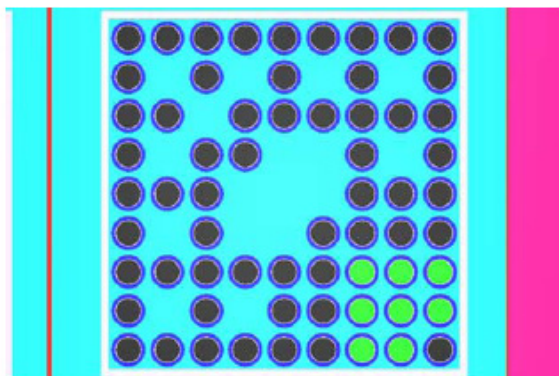
**Figure 6-35 FANP 9x9 Worst Case Fuel Parameters Model with Part Length Fuel Rods**



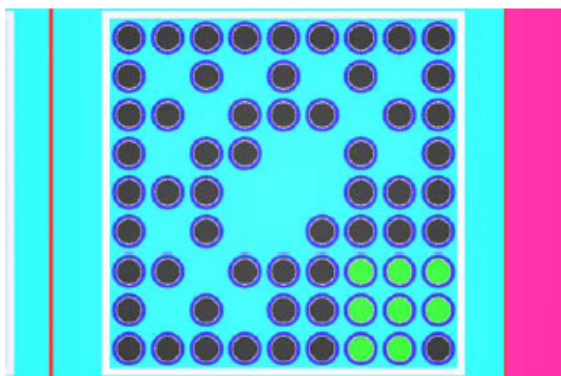
N° FS1-0014159 Rev. 7.0	<div style="text-align: center;"> <b>AREVA TN-B1</b>  <b>Docket No. 71-9372</b>  <b>Safety Analysis Report</b> </div> <div style="text-align: right;">  </div>
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GNF 9x9 4.8 wt% <sup>235</sup>U, 8 Part Length Rods



GNF 9x9 4.8 wt% <sup>235</sup>U, 10 Part Length Rods



GNF 9x9 4.8 wt% <sup>235</sup>U, 12 Part Length Rods

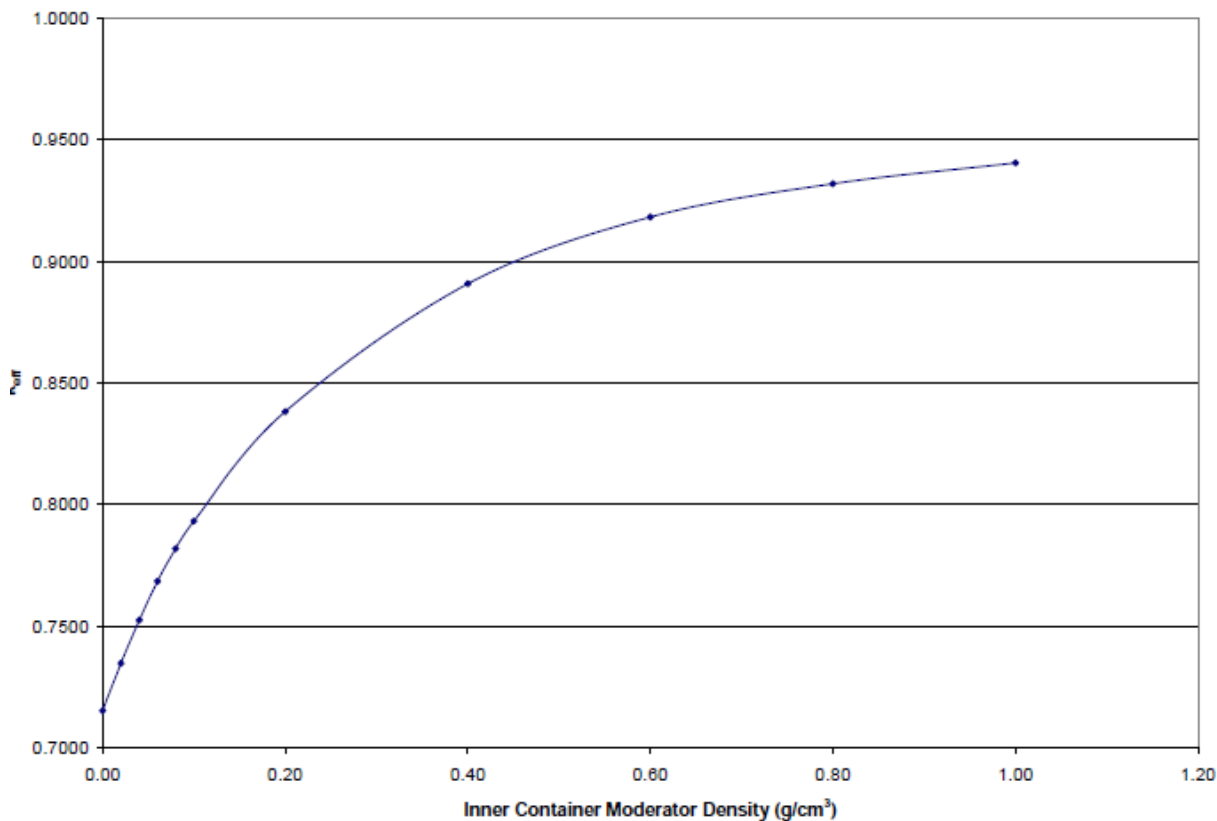
**Figure 6-36 GNF 9x9 Worst Case Fuel Parameters Model with Part Length Fuel Rods**



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#### 6.3.4.10. Moderator Density Study (2N=448)

The worst case design from Table 6-18 TN-B1 Array HAC Part Length Fuel Rod Calculations is used to conduct a moderator density sensitivity analysis. The GNF 10x10 fuel bundle is chosen for the study since it resulted in the highest reactivity in Table 6-18. Previous calculations demonstrated the worst case condition for maximum reactivity is a configuration in which there is no moderator between the TN-B1 shipping packages. The moderator density study is conducted by varying the moderator density inside the inner container fuel compartment. The outer region of the inner container is filled with the Alumina Silicate thermal insulating material. The results of the moderator density study, Table 6-31, are shown in Figure 6-37. As shown in Figure 6-37, all cases peak at full moderator density. Therefore, a moderator density of 1.0 g/cm<sup>3</sup> is chosen as the worst case moderator condition for the TN-B1 inner container fuel compartment.



**Figure 6-37 Moderator Density Sensitivity Study for the TN-B1 HAC Worst Case Parameter Fuel Design**

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#### 6.3.4.11. Material Distribution Reactivity Study (2N=448, 2N=100)

A study is performed to determine the worst packing material distribution within the TN-B1 inner container. The material normally present around the inner container fuel compartment is a thermal insulator consisting of Alumina Silicate. The material normally lining the inner container fuel compartment is a polyethylene foam material which has a density in the range 0.05 – 0.075 g/cm<sup>3</sup>.

The first part of the material distribution study investigates replacing the Alumina Silicate alternately with full density water and void while the inner container fuel compartment is filled with full density water. The GNF 10x10 fuel bundle is chosen for the study since it resulted in the highest reactivity in Table 6-18. In addition, the worst case TN-B1 model is used in a 14x2x16 array (2N=448). The results are shown in Table 6-19. The first three cases in Table 6-19 show the most reactive condition is achieved with the Alumina Silicate thermal insulator in place. Therefore, the Alumina Silicate thermal insulator will remain a part of the worst case TN-B1 model.


The second part of the material distribution study investigates placing the polyethylene foam material in its proper location within the TN-B1 fuel assembly compartment. Until this point, the polyethylene foam was assumed to burn away in the fire that also melted the polyethylene spacers. It should be noted that it is extremely unlikely that this configuration would exist post thermal excursion. The polyethylene foam would be as susceptible to the fire as the polyethylene spacers. However, the incomplete foam burn is considered in this study for conservatism. The GNF 10x10 fuel bundle is chosen for the study since it resulted in the highest reactivity in Table 6-18. In addition, the worst case TN-B1 model is used in a 14x2x16 array (2N=448). The results are shown in Table 6-19. As shown in Table 6-19, the most reactive condition is achieved with the full thickness of ethafoam in place. Since the  $k_{eff}$  values exceed the 0.94254 USL with the polyethylene foam in place, the package array size is reduced to 10x1x10 (2N=100) to meet the acceptance criterion (last row in Table 6-19). The full thickness of ethafoam will be maintained for the remaining TN-B1 calculations since that configuration resulted in the highest  $k_{eff}$  value.

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**Table 6-19 TN-B1 Inner Container Thermal Insulator Region and Polyethylene Foam Material Study**

<b>Fuel Type</b>	<b>Array Size</b>	<b>Inner Container Foam Space</b>	<b>Insulator Space Fill</b>	<b><math>k_{eff}</math></b>	<b><math>\sigma</math></b>	<b><math>k_{eff} + 2\sigma</math></b>
GNF 10x10	14x2x16 (2N=448)	Water	Thermal Ins.	0.9404	0.0007	0.9418
GNF 10x10	14x2x16 (2N=448)	Water	Water	0.7938	0.0009	0.7956
GNF 10x10	14x2x16 (2N=448)	Water	None	0.9362	0.0008	0.9378
GNF 10x10	14x2x16 (2N=448)	¼ Foam Thickness-Water	Thermal Ins.	0.9618	0.0009	0.9636
GNF 10x10	14x2x16 (2N=448)	½ Foam Thickness-Water	Thermal Ins.	0.9808	0.0009	0.9826
GNF 10x10	14x2x16 (2N=448)	5/8 Foam Thickness-Water	Thermal Ins.	0.9902	0.0008	0.9918

<b>Fuel Type</b>	<b>Array Size</b>	<b>Inner Container Foam Space</b>	<b>Insulator Space Fill</b>	<b><math>k_{eff}</math></b>	<b><math>\sigma</math></b>	<b><math>k_{eff} + 2\sigma</math></b>
GNF 10x10	14x2x16 (2N=448)	¾ Foam Thickness-Water	Thermal Ins.	0.9943	0.0008	0.9959
GNF 10x10	14x2x16 (2N=448)	7/8 Foam Thickness-Water	Thermal Ins.	0.9965	0.0008	0.9981
GNF 10x10	14x2x16 (2N=448)	Full Foam Thickness	Thermal Ins.	0.9971	0.0010	<b>0.9991</b>
GNF 10x10	10x1x10 (2N=100)	Full Foam Thickness	Thermal Ins.	0.9378	0.0009	0.9396


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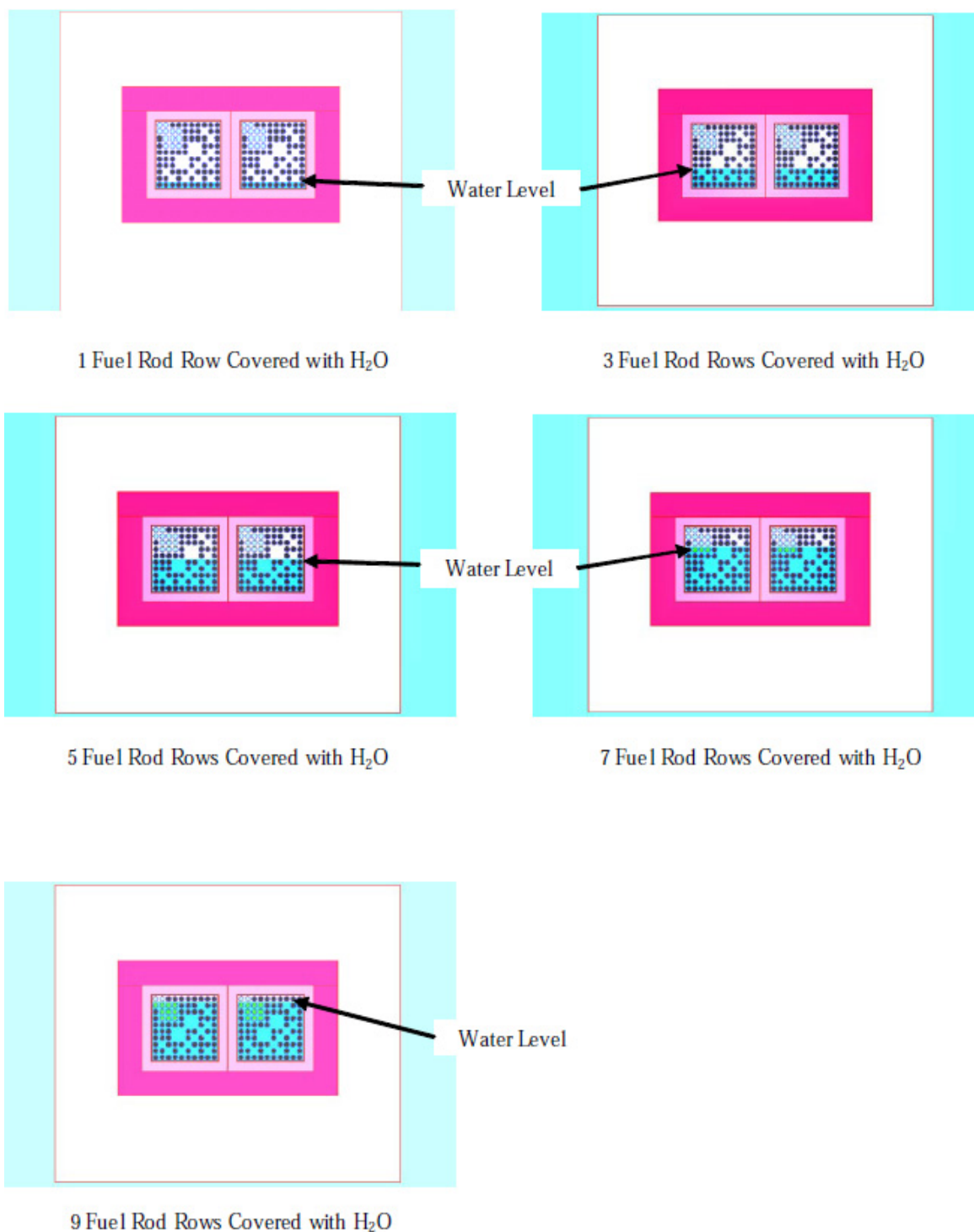
#### 6.3.4.12. Inner Container Partial Flooding Study (2N=100)

Calculations are run in which the fuel bundle rows are partially filled within the TN-B1 inner fuel compartment as shown in Figure 6-39. The GNF 10x10 fuel bundle is chosen for the analysis since it produced the highest reactivity in Figure 6-37. The TN-B1 HAC model from the polyethylene foam study is used with an array size of 10x1x10 (2N=100). The results are shown in Table 6-20. As shown in Table 6-20, the most reactive condition exists when water fully covers each fuel bundle. Therefore, the inner container fuel compartment will be fully flooded with water in the worst case TN-B1 model.


**Table 6-20 TN-B1 Inner Container Partially Filled with Moderator**

<b>Fuel Type</b>	<b>Fuel Rows Filled</b>	<b>Moderator Density (g/cm<sup>3</sup>)</b>	<b>k<sub>eff</sub></b>	<b>σ</b>	<b>k<sub>eff</sub> + 2σ</b>
GNF 10x10	1	1.00	0.6643	0.0007	0.6657
GNF 10x10	3	1.00	0.7678	0.0009	0.7696
GNF 10x10	5	1.00	0.8653	0.0008	0.8669
GNF 10x10	7	1.00	0.9212	0.0008	0.9228
GNF 10x10	9	1.00	0.9355	0.0009	0.9373
GNF 10x10	10	1.00	0.9378	0.0009	<b>0.9396</b>

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**Figure 6-38 TN-B1 Inner Container Fuel Compartment Flooding Cases**

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#### 6.3.4.13. TN-B1 Container Spacing Study (2N=100)

Calculations performed previously assume the TN-B1 shipping containers are resting next to one another with no spacing between them. A container pitch sensitivity study is conducted to determine if reactivity increases as containers are moved away from one another. The HAC model used in the inner container partial flooding study is used for the pitch sensitivity study with an array size of 10x1x10 (2N=100). The GNF 10x10 fuel assemblies with an average lattice enrichment of 5.0 wt% U-235, 12 gadolinia-urania fuel rods enriched to 2.0 wt % gadolinia, and 12 part length fuel rods is used. The worst case fuel parameters listed in Table 6-18 for the GNF 10x10 fuel design are utilized. The edge-to-edge separation is increased from 0 to 10 cm and the reactivity impact is observed. The results shown in Table 6-21 show a decrease in reactivity with increased spacing between containers. Therefore, the most reactive container configuration occurs when there is minimum spacing between containers.

**Table 6-21 TN-B1 Array Spacing Sensitivity Study**

Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Container Pitch (cm)	Pitch (cm)	Pellet Diameter (cm)	Clad ID (cm)	Clad OD (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
GNF 10x10	0.0	71.926	1.350	0.895	0.9338	1.01	0.9378	0.0009	<b>0.9396</b>
GNF 10x10	0.0	74.426	1.350	0.895	0.9338	1.01	0.9259	0.0009	0.9277
GNF 10x10	0.0	76.926	1.350	0.895	0.9338	1.01	0.9122	0.0008	0.9138
GNF 10x10	0.0	81.926	1.350	0.895	0.9338	1.01	0.8865	0.0008	0.8881



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## 6.4. SINGLE PACKAGE EVALUATION

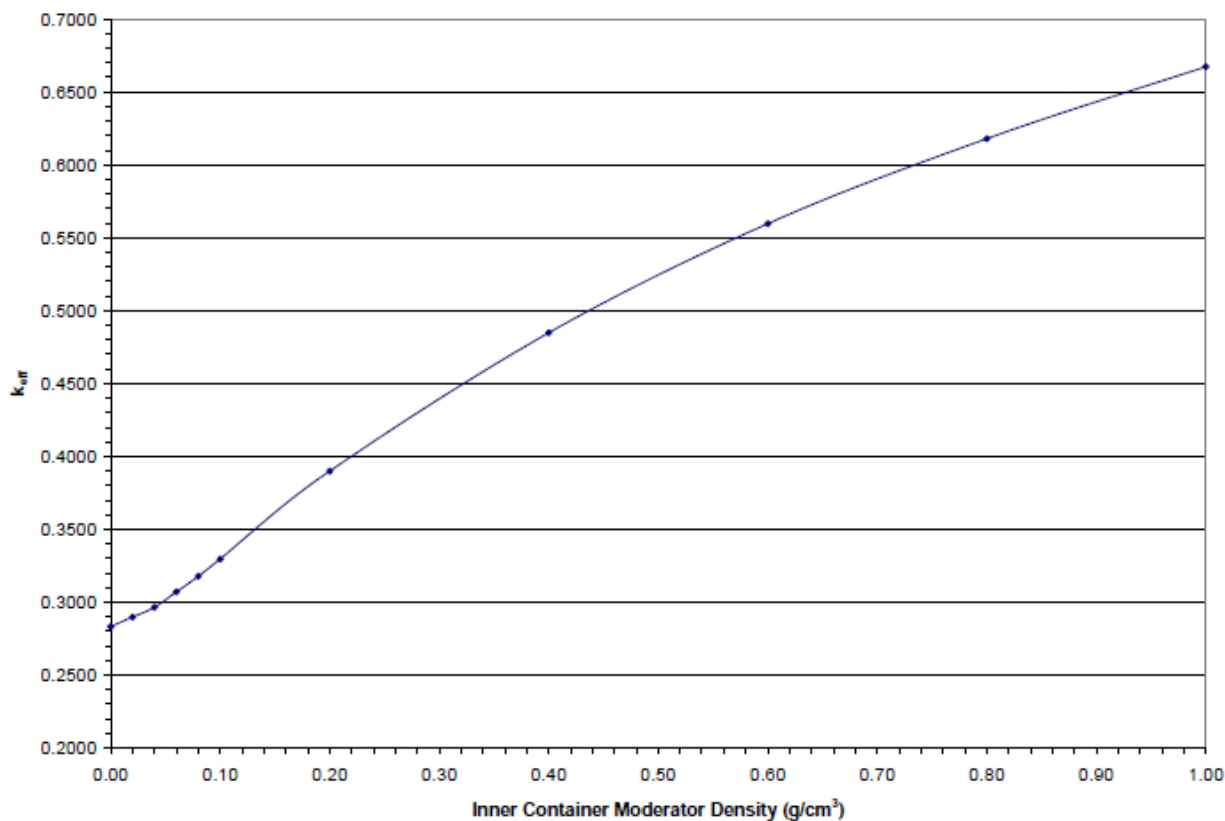
Based on the sensitivity studies performed in this section, the single package and package array normal transport condition and HAC calculations are performed using the GNF 10x10 at an average lattice enrichment of 5.0 wt % U-235, twelve 2.0 wt% gadolinia fuel rods, and 12 part length fuel rods.

### 6.4.1. *Configuration*


The single package model described in Section 6.3.1.1 is used to demonstrate criticality safety of the TN-B1 shipping container using the worst case fuel design. The GNF 10x10 at an average lattice enrichment of 5.0 wt % U-235, twelve 2.0 wt% gadolinia fuel rods, and 12 part length fuel rods is used for the NTC and HAC evaluations. A moderator density study is conducted under both hypothetical accident and normal conditions. In the HAC study, the water density in the inner package is varied while the void in the outer container is maintained. For the normal conditions of transport, the moderator density is uniformly varied.

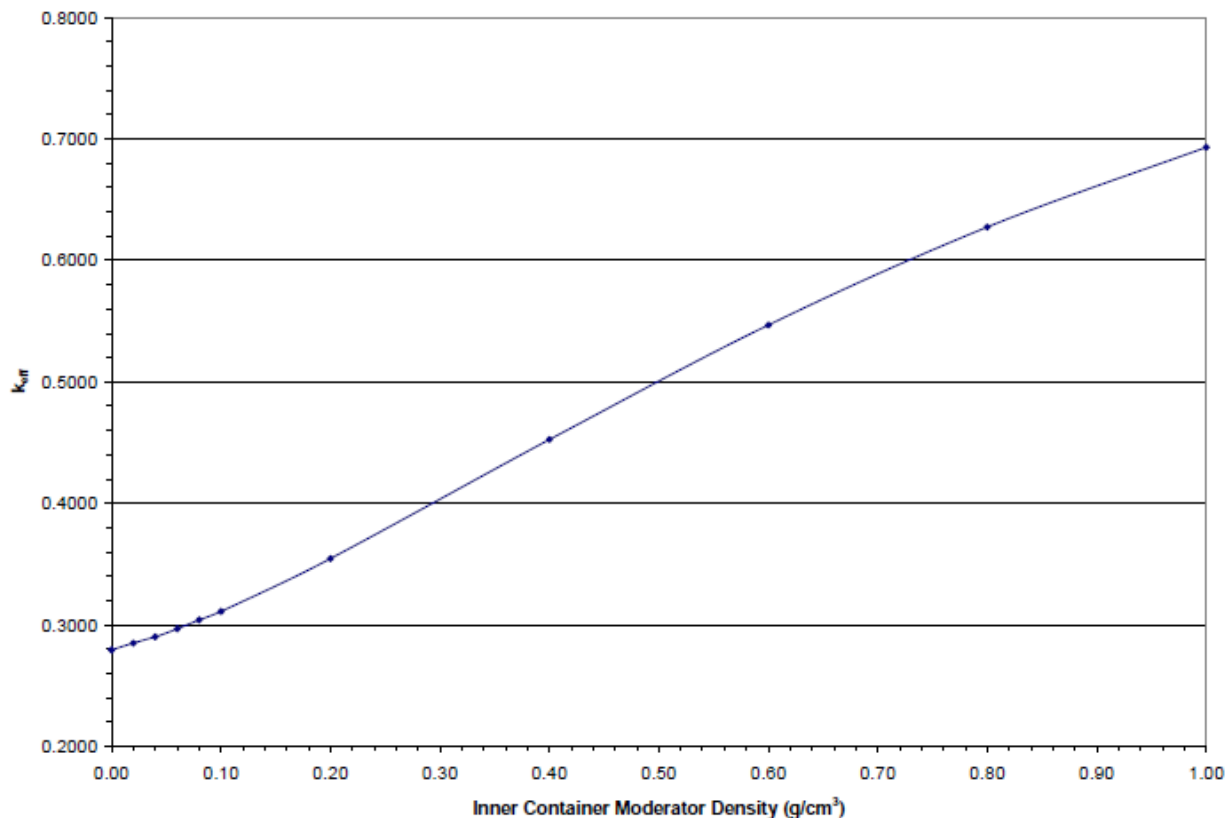
### 6.4.2. *Single Package Results*

The results for the single package normal conditions of transport evaluation are displayed in Figure 6-39. The results for the single package HAC evaluation are shown in Figure 6-40. The results in the figures indicate reactivity for the single package increases with increasing moderator density. The highest  $k_{eff}$  is achieved for both cases at full density moderation in the inner container. The polyethylene foam remains in place for the NTC single package configuration, but the polyethylene foam is removed from the HAC single package configuration. Removing the polyethylene foam in the HAC single package model, decreases neutron leakage which increases reactivity for a single container. In addition, full density moderation is included in the outer container for the single package NTC configuration. In both cases, the  $k_{eff}$  remains far below the USL of 0.94254. The maximum  $k_{eff} + 2\sigma$  for the single package normal conditions of transport case is 0.6689 (Table 6-32), and the maximum  $k_{eff} + 2\sigma$  for the single package HAC case is 0.6951 (Table 6-33). Therefore, criticality safety is established for the single package TN-B1 container.



**Figure 6-39 TN-B1 Single Package Normal Conditions of Transport Results**

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**Figure 6-40 TN-B1 Single Package HAC Results**

## **6.5. EVALUATION OF PACKAGE ARRAYS UNDER NORMAL CONDITIONS OF TRANSPORT**

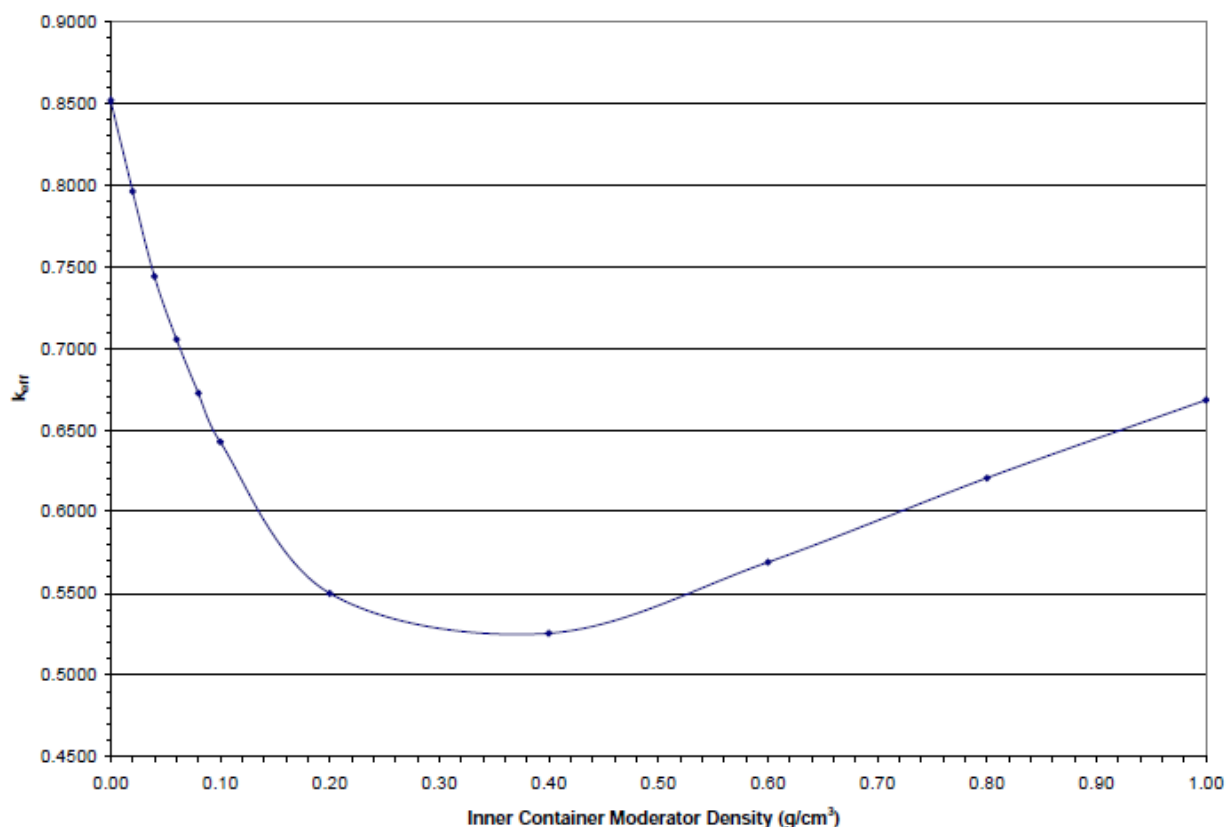
### **6.5.1. *Configuration***

The package array normal condition model described in Section 6.3.1.2.1 is used to demonstrate criticality safety of the TN-B1 shipping container using the GNF 10x10 worst case fuel design at an average lattice enrichment of 5.0 wt % U-235, twelve 2.0 wt% gadolinia fuel rods, and 12 part length fuel rods. The calculation using the normal conditions of transport model involves a moderator density sensitivity study. In the model, the moderator density is uniformly varied and the system reactivity is observed.

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### 6.5.2. *Package Array NCT Results*

The results of the package array normal condition model calculations are shown in Figure 6-41. The reactivity peaks with no moderator present. A decreasing trend continues until the moderator density reaches  $0.4 \text{ g/cm}^3$  at which point reactivity increases almost linearly to full water density. The maximum  $k_{\text{eff}} + 2\sigma$  obtained is 0.8535 (Table 6-34) which is below the USL of 0.94254. Therefore, criticality safety of the TN-B1 shipping container is demonstrated under normal conditions of transport.



**Figure 6-41 TN-B1 Package Array Under Normal Conditions of Transport Results**

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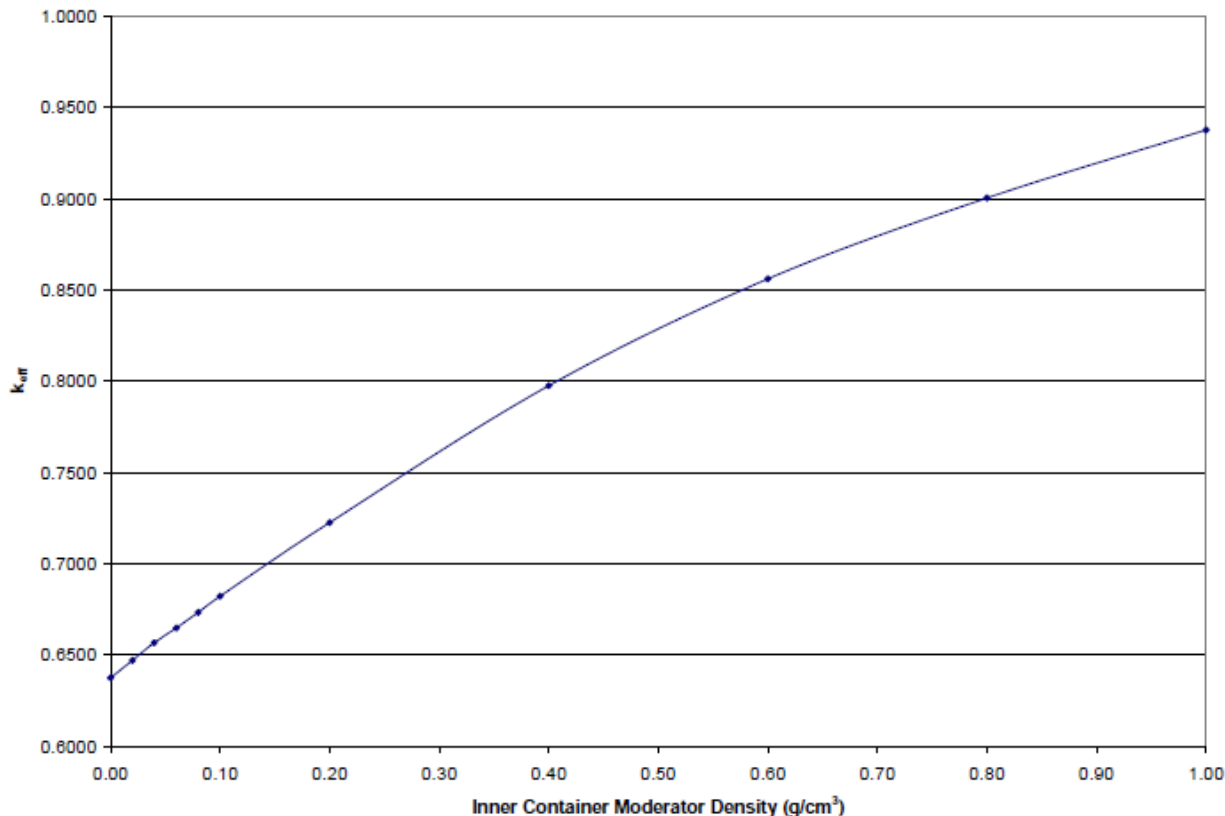
## 6.6. **PACKAGE ARRAYS UNDER HYPOTHETICAL ACCIDENT CONDITIONS**

### 6.6.1. ***Configuration***

The package array hypothetical accident condition model described in Section 6.3.1.2.2 is used to demonstrate criticality safety of a 10x1x10 array (2N=100) of TN-B1 shipping containers using the GNF 10x10 worst case fuel design at an average lattice enrichment of 5.0 wt % U-235, twelve 2.0 wt% gadolinia fuel rods, and 12 part length fuel rods. The calculation using the HAC model involves a moderator density sensitivity study. In the study, no moderator is present in the outer container while the moderator density inside the inner container is varied. The polyethylene foam inside the inner container fuel compartment is modeled because previous calculations demonstrated this configuration to be the most reactive.

### 6.6.2. ***Package Array HAC Results***

The results of the package array (2N=10x1x10=100 array) HAC model calculations are shown in Figure 6-42. The system reactivity begins at its lowest value and increases with increasing interspersed moderator density. This trend highlights the neutronics of the problem. Initially, no moderator, other than the polyethylene surrounding the fuel rods, is present to thermalize neutrons that enter the inner container. As the inner container moderator density increases, higher energy neutrons pass into adjacent containers and thermalize in the vicinity of the fuel creating a more reactive situation. The maximum  $k_{eff} + 2\sigma$  for the package array HAC case is 0.9396 (Table 6-35) which is below the USL of 0.94254. Therefore, criticality safety of the TN-B1 shipping container is demonstrated for the package array under hypothetical accident conditions.



**Figure 6-42 TN-B1 Package Array Hypothetical Accident Condition Results**

#### 6.6.2.1. Pu-239 Effect on Reactivity for the TN-B1 Package Array Hypothetical Accident Condition

Because the fuel scheduled for transport in the TN-B1 could have a small Pu-239 content, the effect on the TN-B1 Package HAC reactivity is investigated. The maximum plutonium concentration ( $3.04 \times 10^{-9}$  gPu-239/gU) listed in Table 1-3 of the SAR is added to the worst case package array HAC model (10x1x10 array), determined in the previous sections, and the  $k_{eff}$  is calculated. The results showed no statistically significant difference between the cases with and without plutonium. The  $k_{eff} \pm 2\sigma$  for the worst case with plutonium is 0.9406. The  $k_{eff} + 2\sigma$  for the worst case without plutonium, calculated in Section 6.6.2, is 0.9396. Both results remain below the USL of 0.94254. Therefore, the plutonium is justifiably neglected in the TN-B1 evaluation.



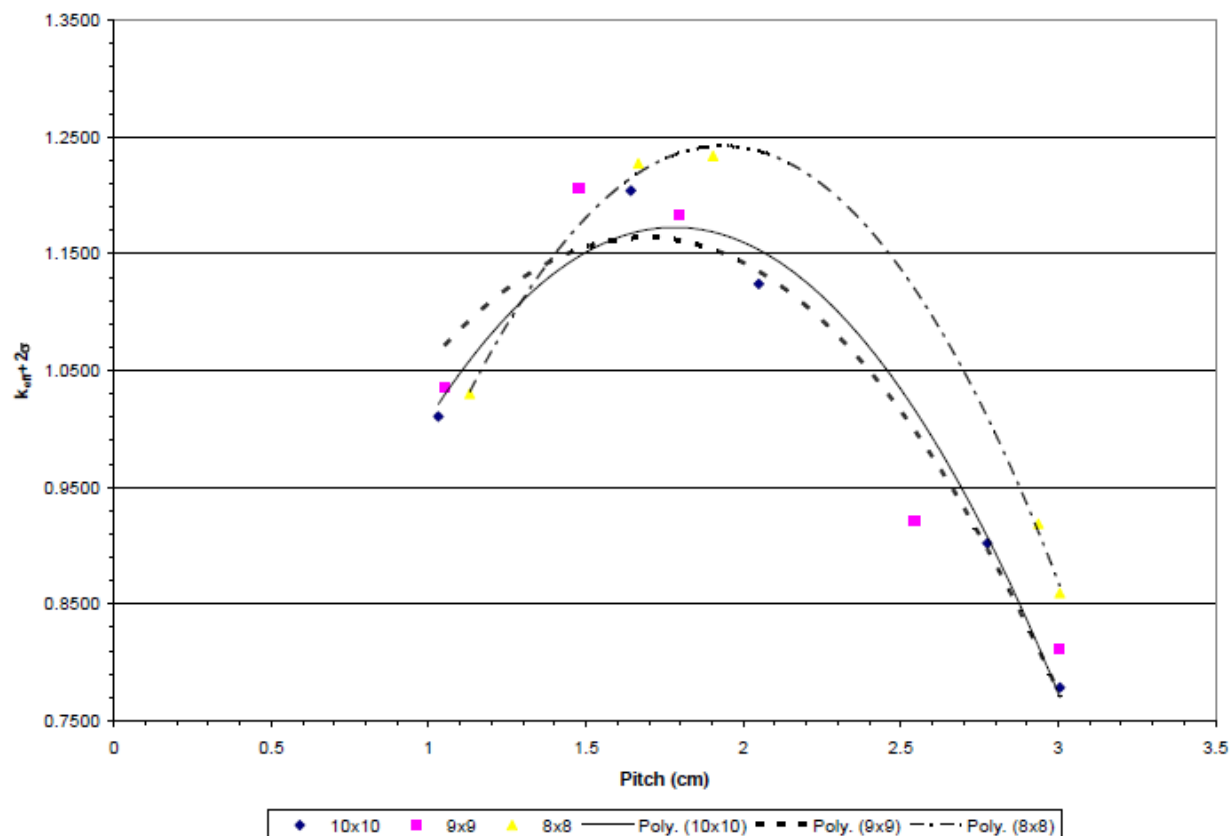
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## 6.7. FUEL ROD TRANSPORT IN THE TN-B1


Studies are conducted to allow transport of UO<sub>2</sub> fuel rods in the TN-B1 container. Several configurations are investigated including: loose fuel rods, fuel rods bundled together, and fuel rods contained in 5-inch stainless steel pipe/protective case. The model uses the 10x10, 9x9, or 8x8 worst case fuel rod designs developed in Section 6.3.4. A 6-mil layer of polyethylene encircles each fuel rod in the model to bound protective packing material that may be used for fuel rod transport.

### 6.7.1. *Loose Fuel Rod Study*

The package array model under hypothetical accident conditions is used for fuel rod calculations in the TN-B1, since it was demonstrated to be more reactive than the normal conditions of transport, package array model. The worst case fuel rods are arranged in a square pitch array inside each TN-B1 transport compartment. Scoping studies indicated little difference between the square and triangular pitch array, therefore the square pitch array is chosen for convenience. The inner container is filled with full density water and the outer container has no water, which facilitates leakage of neutrons into neighboring containers. The fuel rod pitch is varied, and the results are illustrated with curves. The curves are shown Figure 6-43 Fuel Rod Pitch Sensitivity Study and corresponding calculational data listed in Table 6-22 Fuel Rod Pitch Sensitivity Study Results. The results demonstrate that a fully loaded inner compartment in which the rods are all in contact with each other is a supercritical configuration. As a result, a minimum number of fuel rods to ensure subcriticality cannot be established for the TN-B1 shipping container. A maximum fuel rod quantity to ensure subcriticality can be established for the loose configuration. For all three fuel designs, a maximum of 25 fuel rods may be safely transported in each RAJ- II fuel assembly compartment. The 8x8 rod design is limiting as shown in Figure 6-43 and Table 6-22 Fuel Rod Pitch Sensitivity Study Results.



**Figure 6-43 Fuel Rod Pitch Sensitivity Study**

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**Table 6-22 Fuel Rod Pitch Sensitivity Study Results**

Fuel Rod Type	Fuel Rod Pitch (cm)	Fuel Rod Number	Fuel Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
10x10	1.0305	289	0.9	1.000	1.000	1.0092	0.0007	1.0106
10x10	1.6416	100	0.9	1.000	1.000	1.2024	0.0009	1.2042
10x10	2.0484	64	0.9	1.000	1.000	1.1224	0.0009	1.1242
10x10	2.7754	34	0.9	1.000	1.000	0.9005	0.0008	0.9021
10x10	3.0056	25	0.9	1.000	1.000	0.7769	0.0007	0.7783
9x9	1.0505	256	0.9600	1.0200	1.0200	1.0341	0.0007	1.0355
9x9	1.4770	121	0.9600	1.0200	1.0200	1.2045	0.0008	1.2061
9x9	1.7972	81	0.9600	1.0200	1.0200	1.1816	0.0008	1.1832
9x9	2.5432	34	0.9600	1.0200	1.0200	0.9196	0.0008	0.9212
9x9	3.0056	25	0.9600	1.0200	1.0200	0.8096	0.0007	0.8110
8x8	1.1305	225	1.05	1.1000	1.1000	1.0288	0.0007	1.0302
8x8	1.6662	100	1.05	1.1000	1.1000	1.2259	0.0008	1.2275
8x8	1.9035	81	1.05	1.1000	1.1000	1.2328	0.0007	1.2342
8x8	2.9370	30	1.05	1.1000	1.1000	0.9172	0.0008	0.9188
8x8	3.0056	25	1.05	1.1000	1.1000	0.8577	0.0008	0.8593

The results in Table 6-22 Fuel Rod Pitch Sensitivity Study Results are based on calculations performed with full water density inside the inner container. It appears the maximum fuel rod quantity allowable for the 10x10 and 9x9 fuel rods should be 34, while that for the 8x8 fuel rods should be 30. However, the rod configurations at full moderator densities represent an overmoderated condition in which reactivity peaks at a reduced moderator density. Therefore, calculations are performed with 25 fuel rods in each transport compartment for each fuel rod type, and the moderator density inside the inner container is varied from 0.4 g/cm<sup>3</sup> to 1.00 g/cm<sup>3</sup> to investigate the possibility that reactivity peaks at a lower moderator density. The results of these calculations are shown in Table 6-23. The peak reactivity for all the fuel rod types occurs at a moderator density of 0.6 g/cm<sup>3</sup> and are all below the USL of 0.94254. Therefore, criticality safety for loose fuel rod transport with a maximum of 25 rods in each transport compartment is demonstrated.

**Table 6-23 Fuel Rod Maximum Quantity at Reduced Moderator Densities**

Fuel Rod Type	Fuel Rod Pitch (cm)	Fuel Rod Number	Inner Container Moderator Density (g/cm <sup>3</sup> )	Fuel Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
10x10	3.0056	25	0.40	0.9	1.000	1.000	0.7875	0.0009	0.7893
10x10	3.0056	25	0.60	0.9	1.000	1.000	0.8113	0.0008	<b>0.8129</b>
10x10	3.0056	25	0.80	0.9	1.000	1.000	0.8012	0.0007	0.8026
10x10	3.0056	25	1.00	0.9	1.000	1.000	0.7769	0.0007	0.7783
9x9	3.0056	25	0.40	0.9600	1.0200	1.0200	0.8128	0.0008	0.8144
9x9	3.0056	25	0.60	0.9600	1.0200	1.0200	0.8404	0.0008	<b>0.8420</b>
9x9	3.0056	25	0.80	0.9600	1.0200	1.0200	0.8321	0.0008	0.8337
9x9	3.0056	25	1.00	0.9600	1.0200	1.0200	0.8096	0.0007	0.8110
8x8	3.0056	25	0.40	1.05	1.1000	1.1000	0.8529	0.0008	0.8545
8x8	3.0056	25	0.60	1.05	1.1000	1.1000	0.8832	0.0008	<b>0.8848</b>
8x8	3.0056	25	0.80	1.05	1.1000	1.1000	0.8799	0.0009	0.8817
8x8	3.0056	25	1.00	1.05	1.1000	1.1000	0.8577	0.0008	0.8593


a. Limiting case(s) shown in bold

### 6.7.2. *Fuel Rods Bundled Together*

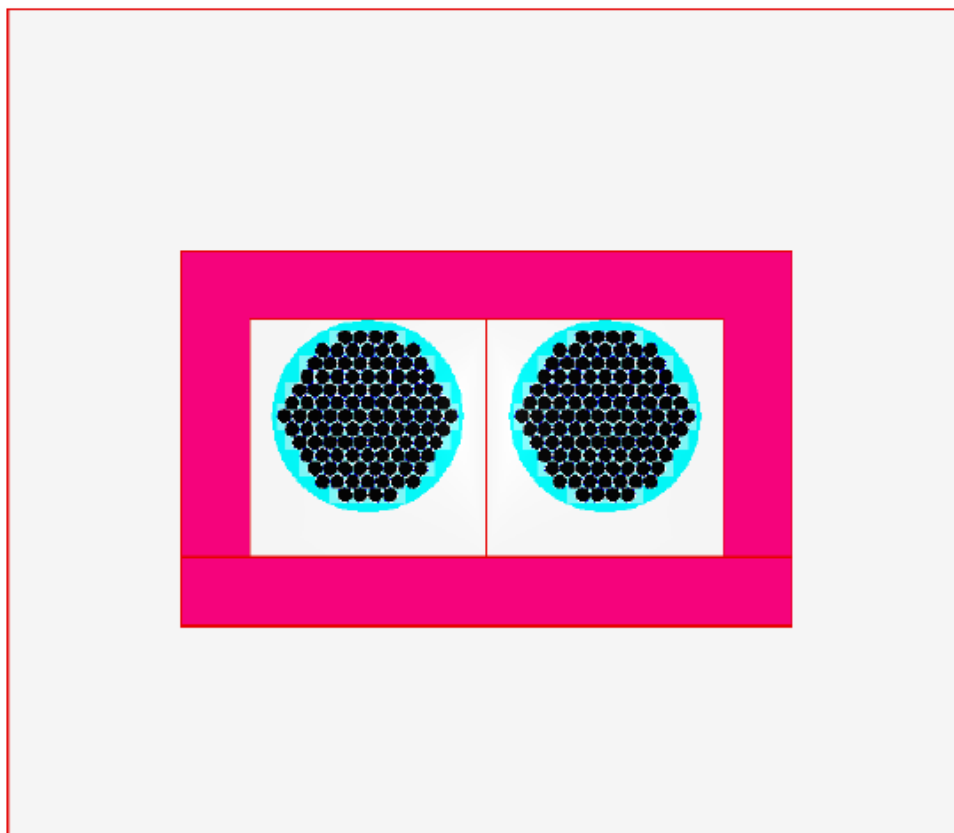
Based on the results in the previous calculation, there is no advantage to bundling fuel rods together since close packed rods do not guarantee subcriticality. Besides, the straps holding the fuel rods together in the bundle may fail during an accident, and the rods could move about the transport compartment without restraint. Therefore, the maximum number of fuel rods allowable in each TN-B1 fuel compartment when fuel rods are transported in bundles is 25 for all types.

### 6.7.3. *Fuel Rods Transported in 5-Inch Stainless Steel Pipe*

A fuel rod pitch sensitivity study is conducted for the transport of fuel rods inside 5-inch stainless steel pipe, residing in the TN-B1 fuel compartment. The package array model under hypothetical accident conditions is used for fuel rod calculations in the TN-B1 container, since it was demonstrated to be more reactive than the normal conditions of transport, package array model. The GNF 10x10, the GNF 9x9, and the GNF 8x8, the UC and PWR worst case fuel rod designs are used for the study. Since the 5-inch stainless steel pipe presents a more difficult volume to accommodate rods in a square pitch, a triangular pitch array is used for the rod configuration. The pipe's stainless steel wall is also neglected for conservatism. The fuel rod configuration inside the pipe is shown in Figure 6-44 for the GNF 8x8 fuel rods. The volume inside the pipe is filled with water at a density sufficient for optimum moderation. The inner fuel


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compartment volume outside the pipe is modeled with no material present to maximize neutron interaction among packages in the array.

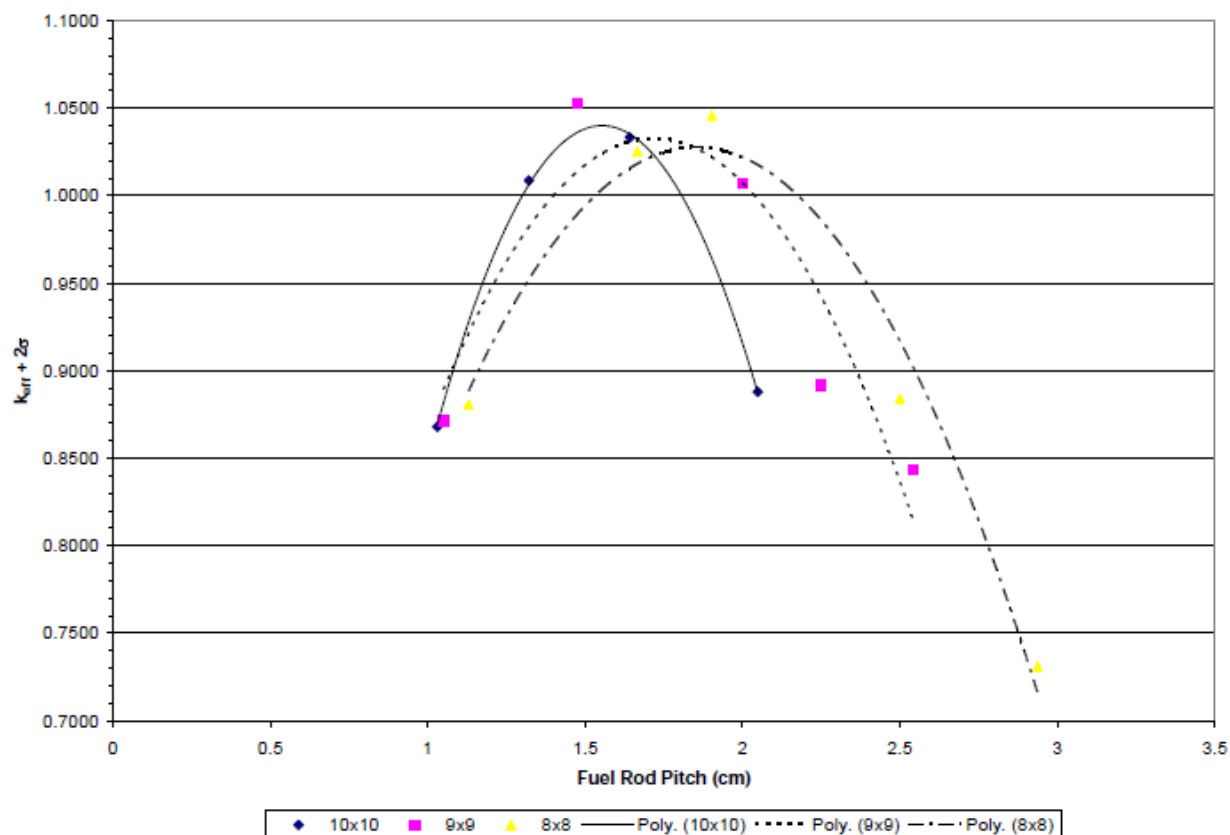


**Figure 6-44 TN-B1 with Fuel Rods in 5-Inch Stainless Steel Pipes for Transport**

The results for fuel rod transport in a SS pipe within the TN-B1 container for the all rod designs are displayed in Figure 6-45. As shown in Figure 6-45, optimum peaks are formed above the USL of 0.94254. Therefore, the stainless steel pipe may be used to ship a limited number of fuel rods. The maximum number of 10x10 fuel rods that may be transported in the stainless steel pipe is 30. The maximum number of 9x9 fuel rods that may be transported in the stainless steel pipe is 26. The maximum number of 8x8 fuel rods that may be transported in the stainless steel pipe is 22. The  $k_{eff} + 2\sigma$  values (Table 6-36) for all fuel rod types with the appropriate fuel rod quantity are below the USL of 0.94254. Therefore, criticality safety is demonstrated for fuel rod transport inside a SS pipe within the TN-B1 container.

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The optimum peak for the 10x10 fuel rods is greater than that for the 9x9 or 8x8 fuel rods in the SS pipe. Since the reactivity peak for the 8x8 fuel rod in the loose rod study is greater than that for the 10x10 fuel rods in the SS pipe, it is chosen as the bounding fuel assembly type.



**Figure 6-45 TN-B1 Fuel Rod Transport in Stainless Steel Pipe**




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This addendum to the TN-B1 SAR includes analysis of Uranium-Carbide and UO<sub>2</sub> PWR rods inside the 5" stainless steel pipe. Loose rods in the product container are evaluated in this analysis ACEL's CANDU Uranium-Carbide (UC) or generic Uranium-Dioxide (UO<sub>2</sub>) fuel rods with a maximum U-235 (pellet) enrichment of 5.0%. The analysis is also applicable to UC or UO<sub>2</sub> fuel rods with GdO<sub>2</sub> or boron, provided that the maximum enrichment and dimensional limits are met since the presence of GdO<sub>2</sub> or boron in the fuel rods will result in a reduction in the applicable neutron multiplication factors. The same applies to fuel rods clad with stainless steel since stainless steel (with the same or greater clad thickness) is a better neutron absorber than zircaloy.

Three different fuel rods have been considered in this analysis, as designated by the labels "CANDU-14", "CANDU-25" and "PWR". The CANDU-14 and CANDU-25 types are those corresponding to the fuel rods in typical CANDU 14 element and 25 element fuel bundle assemblies (Table 6-2). The PWR type is that corresponding to generic PWR fuel rods.

The optimum condition for interspersed water in 8x1x8 and 4x2x6 arrays of damaged containers has been determined as in the case for the infinite arrays of undamaged containers by scoping calculations independently varying the W/F ratios inside the product containers and the interspersed water outside. The results of the scoping calculations are that the optimum interspersed water is again the 0.0 case, presumably because the fuel region inside the Product Containers is already fully moderated by the water and plastic sleeving surrounding the fuel rods.

Based on the results of the horizontally infinite arrays of damaged packages, calculations have been made for the 8x1x8 arrays of damaged TN-B1 containers for most reactive water-to-fuel ratios inside the product containers without interspersed water outside the product containers. Tables 6-24 and 6-25 show the results for three types of rods. The maximum  $k_{eff} + 2\sigma$  for 8x1x8 arrays of TN-B1 containers is 0.9131 which occurs for loose CANDU-14 UC fuel rods at a W/F ratio of 2.68. As in the case for the horizontally infinite arrays of undamaged TN-B1, this result also bounds the  $k_{eff}$  values of the CANDU-25 UC fuel rod and generic PWR UO<sub>2</sub> designs.

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**Table 6-24 Results for 8x1x8 Array of Containers with Loose Fuel Rods**

Type of Rods	W/F Ratio	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
CANDU-14 (UC)	2.12	0.90794	0.00076	0.90946
CANDU-25 (UC)	2.68	0.91162	0.00074	<b>0.91310</b>
PWR (UO <sub>2</sub> )	2.24	0.85480	0.00074	0.85628

**Table 6-25 Results for 4x2x6 Array of Containers with Loose Fuel Rods**

Type of Rods	W/F Ratio	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
CANDU-14 (UC)	2.12	0.82820	0.00073	0.82966
CANDU-25 (UC)	2.68	0.83361	0.00072	<b>0.83505</b>
PWR (UO <sub>2</sub> )	2.24	0.77301	0.00075	0.77451

#### **6.7.4. Fuel Rods Transported in Stainless Steel Protective Case**

The fuel rod pitch sensitivity study conducted for the transport of fuel rods inside the 5-inch stainless steel pipe described in Section 6.7.3 bounds the transport of fuel rods in the protective case. The protective case cross-section is 89 mm (3.50 inches) by 80 mm (3.15 inches). Based on this small cross-sectional area, the total number of fuel rods that will fit in the protective case is less than the total for the 5-inch pipe. Based on the calculations for the stainless steel pipe, the maximum number of 10x10 fuel rods that may be transported in the protective case is 30, the maximum number of 9x9 fuel rods that may be transported in in the protective case is 26, the maximum number of 8x8 fuel rods that may be transported in in the protective case is 22.

#### **6.7.5. Single Package Fuel Rod Transport Evaluation**

##### **6.7.5.1. Configuration**

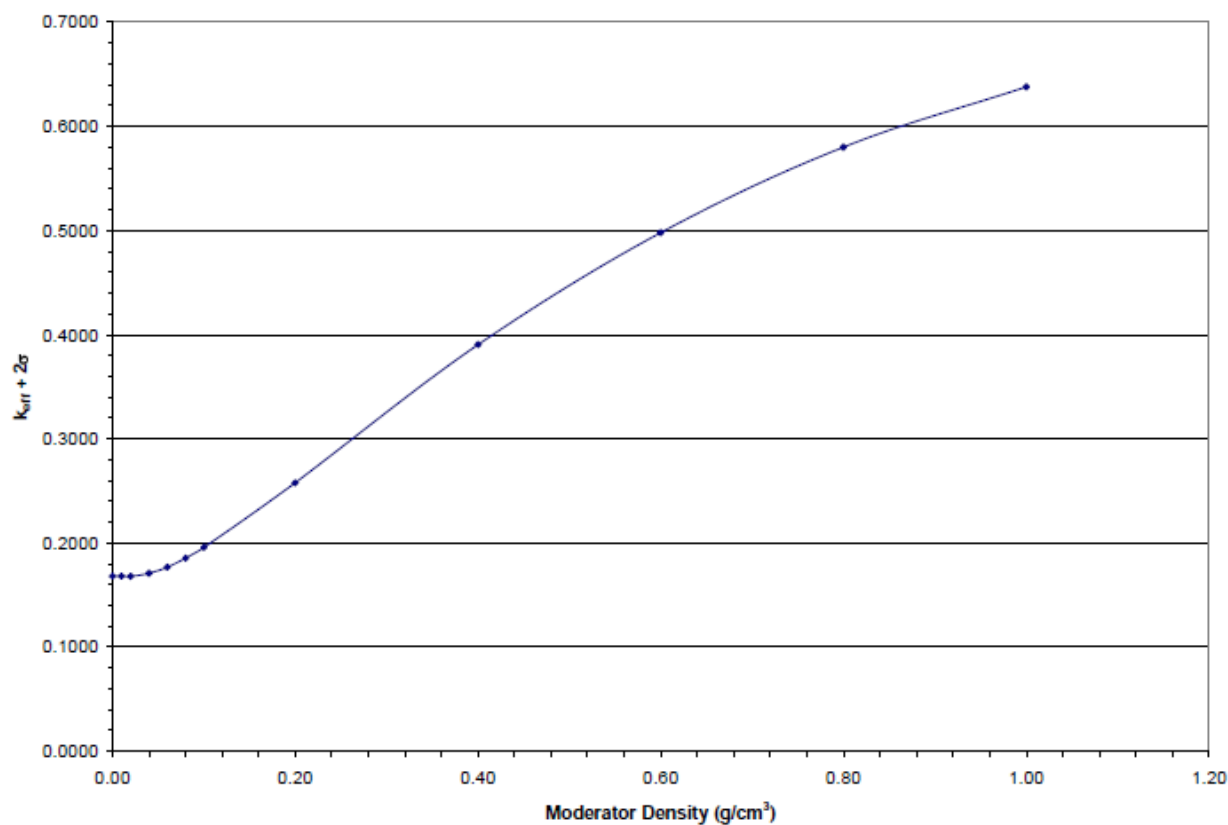
The single package model described in Section 6.3.1.1 is used to demonstrate criticality safety of the TN-B1 shipping container using the worst case fuel design. The single package is evaluated under both normal conditions of transport and hypothetical accident conditions. The

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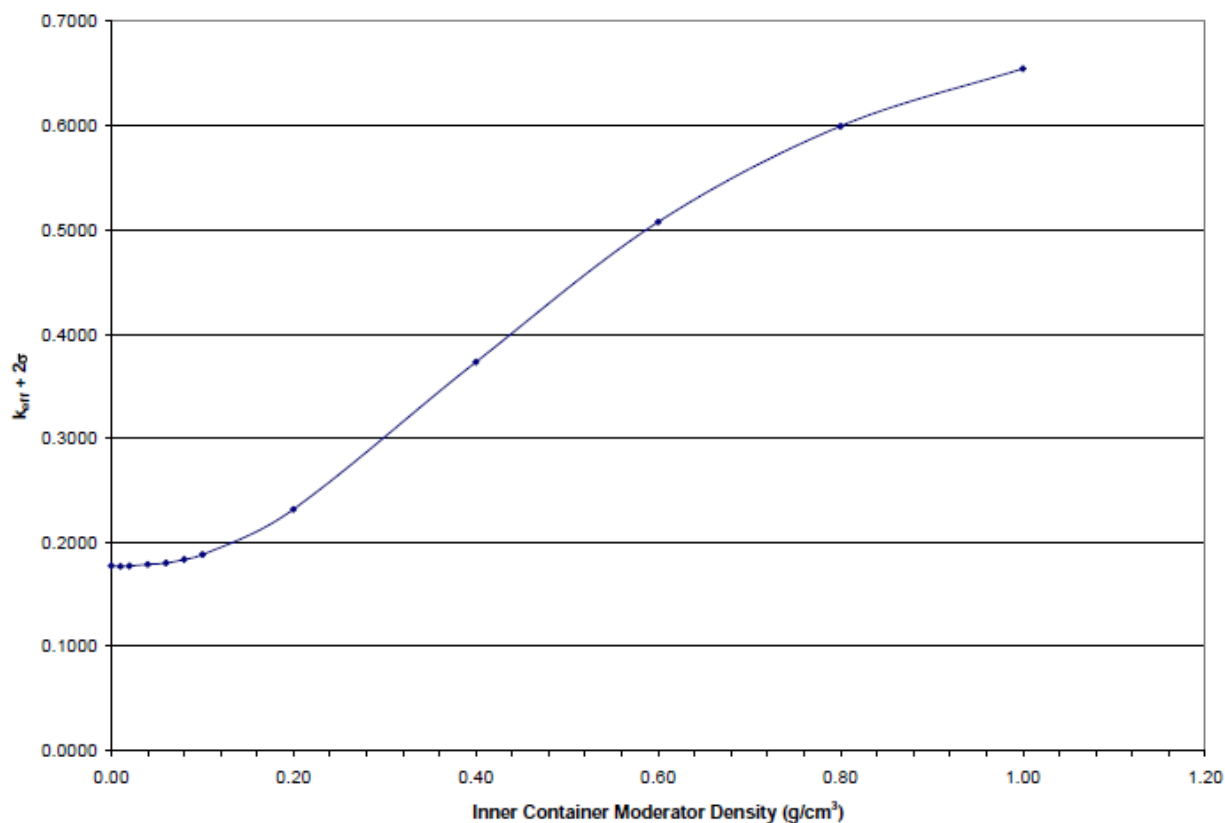
evaluation consists of a moderator density sensitivity study. For the normal conditions of transport model, the moderator density is uniformly varied. In contrast, the moderator density is fixed in the inner container for the hypothetical accident condition model, and the moderator in the outer container is varied. Based on the results in Table 6-22, the GNF 8x8 worst case fuel rod design is used for the study since it produced the highest reactivity peak among all fuel rods considered.

#### 6.7.5.2. Single Package Fuel Rod Transport Result

The results for the single package, loose fuel rod, normal conditions of transport evaluation are displayed in Figure 6-46. The results for the single package, loose fuel rod, HAC evaluation are shown in Figure 6-47. The results in the figures indicate reactivity for the single package increases with increasing moderator density. The highest  $k_{\text{eff}}$  is achieved for both cases at full density moderation. In both cases, the  $k_{\text{eff}}$  remains far below the USL of 0.94254. The maximum  $k_{\text{eff}} + 2\sigma$  for the single package normal conditions of transport case is 0.6381 (Table 6-37), and the maximum  $k_{\text{eff}} + 2\sigma$  for the single package HAC case is 0.6548 (Table 6-38). Therefore, criticality safety is established for the single package TN-B1 container transporting up to 25 loose fuel rods.




**Figure 6-46 TN-B1 Fuel Rod Single Package Under Normal Conditions of Transport**



**Figure 6-47 TN-B1 Fuel Rod Transport Single Package HAC**

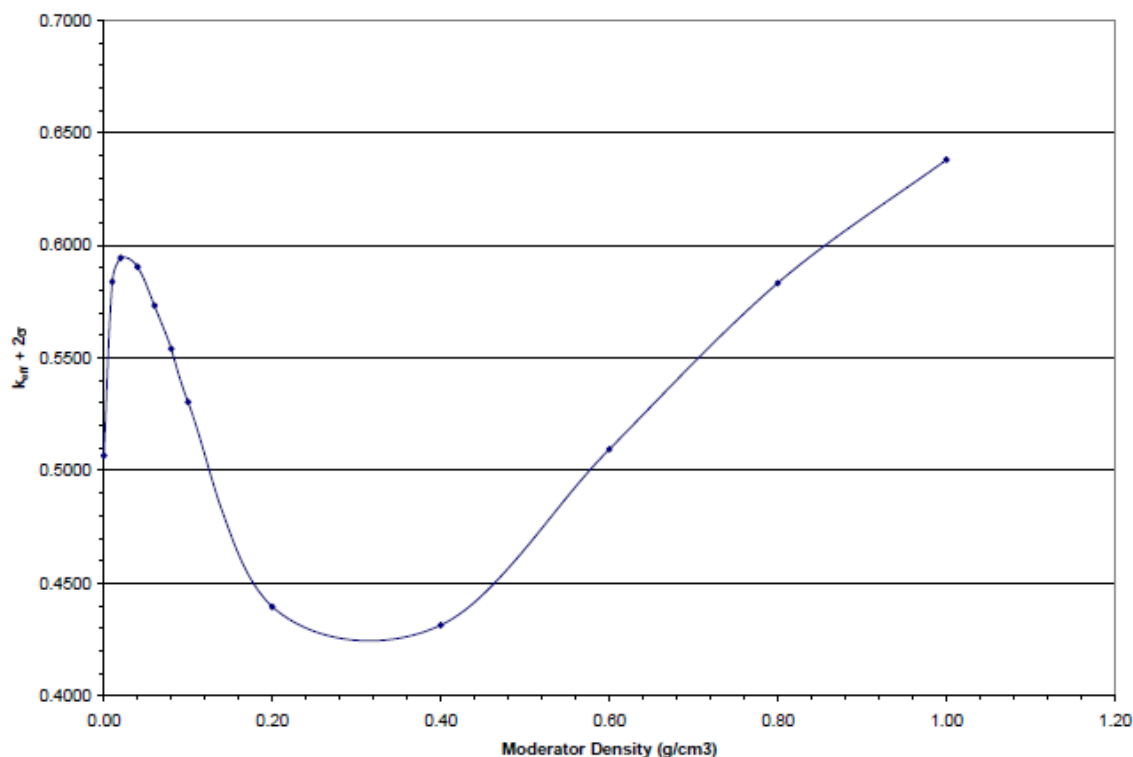
#### **6.7.6. *Evaluation of Package Arrays with Fuel Rods Under Normal Conditions of Transport***

The package array normal condition model described in Section 6.3.1.2.1 is used to demonstrate criticality safety of the TN-B1 shipping container when transporting fuel rods. Based on the results in Table 6-22, the GNF 8x8 worst case fuel rod design is used for the study since it produced the highest reactivity peak among all fuel rod designs considered. The calculation using the package array normal conditions of transport model for fuel rod transport involves a moderator density sensitivity study. In the model, the moderator density is uniformly varied and the system reactivity is observed.

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#### 6.7.6.1. Package Array NCT Fuel Rod Transport Results

The results of the package array fuel rod transport normal condition model calculations are shown in Figure 6-48. As shown, the reactivity initially increases then decreases as the moderator density increases until a density of 0.4 g/cm<sup>3</sup> is reached, then it increases essentially linearly until full density is reached. The maximum  $k_{eff} + 2\sigma$  obtained is 0.6381 (Table 6-39) which is below the USL of 0.94254. Therefore, criticality safety of the TN-B1 shipping container with fuel rods is demonstrated under normal conditions of transport.



**Figure 6-48 TN-B1 Package Array Under Normal Conditions of Transport with Loose Fuel Rods**



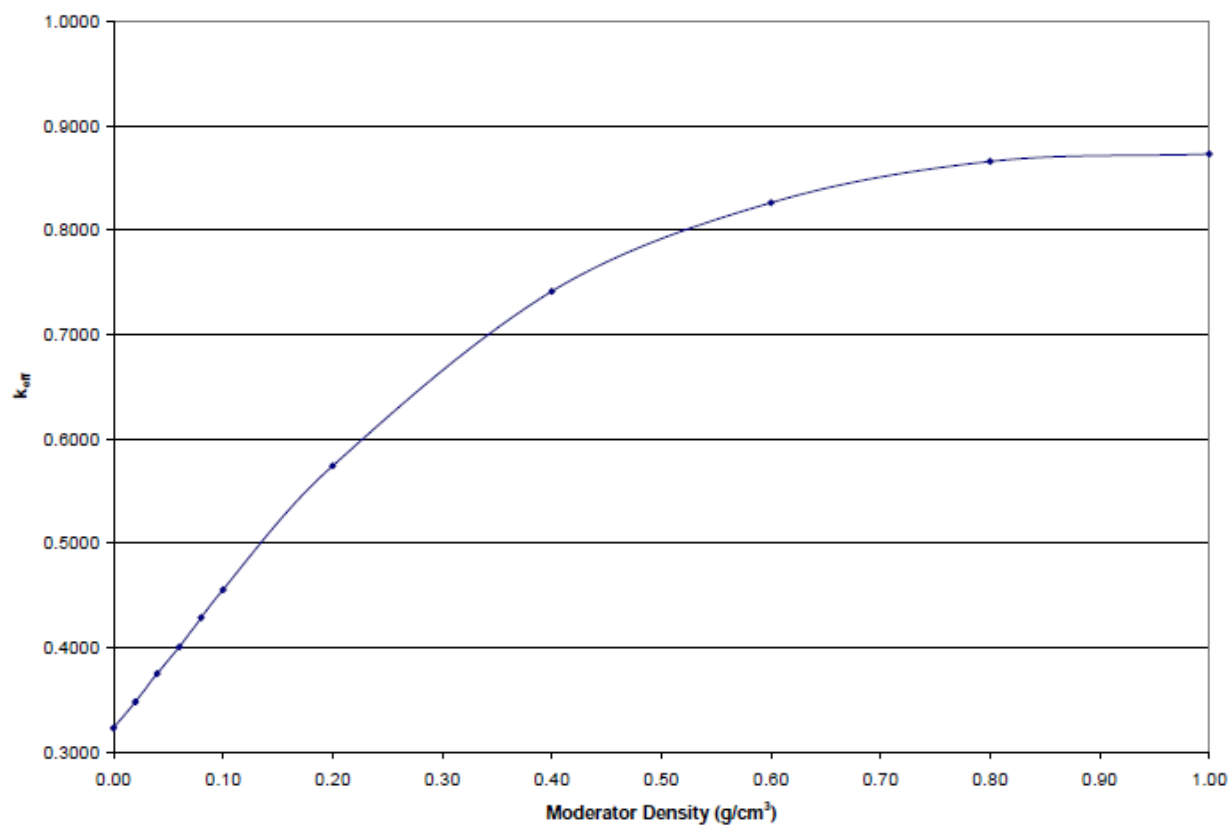
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#### **6.7.7. Fuel Rod Transport Package Arrays Under Hypothetical Accident Conditions**

The package array hypothetical accident condition model described in Section 6.3.1.2.2 is used to demonstrate criticality safety of a 10x1x10 array (2N=100) of TN-B1 shipping containers when transporting loose fuel rods. Based on the results in Table 6-22, the GNF 8x8 worst case fuel rod design is used for the study since it produced the highest reactivity peak among the fuel rod designs considered. The calculation using the HAC model involves a moderator density sensitivity study. In the study, there is no interspersed moderator, and the moderator density inside the inner container is varied. The polyethylene foam lines the inner container fuel compartment since the configuration resulted in the most reactive conditions.

##### **6.7.7.1. Package Array HAC Fuel Rod Transport Results**

The results of the package array HAC model calculations are shown in Figure 6-49. The reactivity begins at its lowest value and increases with increasing internal moderator density until a peak is reached at a density of 0.6 g/cm<sup>3</sup>. The maximum  $k_{eff} + 2\sigma$  for the package array fuel rod transport HAC case is 0.8745 (Table 6-40), which is below the USL of 0.94254. Therefore, criticality safety of the TN-B1 shipping container is demonstrated for the package array under hypothetical accident conditions when fuel rods are being transported



**Figure 6-49 TN-B1 Fuel Rod Transport Under HAC**

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## 6.8. FISSILE MATERIAL PACKAGES FOR AIR TRANSPORT

This package is not intended for the air transport of fissile material.

## 6.9. CONCLUSION

Based on the calculations that have been documented, the TN-B1 shipping container is qualified to transport UO<sub>2</sub> fuel assemblies, including 10x10, 9x9, and 8x8 BWR designs, in accordance with the criticality safety requirements of the IAEA and 10 CFR 71. The fuel assemblies may be channeled or un-channeled.

The calculations documented in Chapter 6.0 also demonstrate a finite 10x1x10 array of damaged, or a 21x3x24 array of un-damaged packages remains below a  $k_{eff}$  of 0.95 with optimum interspersed moderation. Therefore, the calculations support a CSI of 1.0.

In addition, the calculations demonstrate UO<sub>2</sub> fuel rods may be packaged within the TN-B1 inner container in 5-inch stainless steel pipe/protective case, loose, or bundled together. The UO<sub>2</sub> fuel rods may consist of 10x10, 9x9, or 8x8 fuel rod designs.

The calculations documented in Chapter 6.0 also demonstrate the 10x10 fuel assemblies may be transported with 8, 10, 12, or 14 part length fuel rods, and 9x9 fuel assemblies may be transported with 8, 10 and 12 part length fuel rods.

## 6.10. BENCHMARK EVALUATIONS

### 6.10.1. *Applicability of Benchmark Experiments*

The criticality calculation method is verified by comparison with critical experiment data which is sufficiently diverse to establish that the method bias and uncertainty will apply to conditions considered in the TN-B1 shipping container criticality analysis. A set of 27 critical experiments are analyzed using SCALE-PC to demonstrate its applicability to criticality analysis and to establish a set of Upper Subcritical Limits (USLs) that define acceptance criteria. Benchmark experiments are selected with compositions, configurations, and nuclear characteristics that are comparable to those encountered in the TN-B1 shipping container loaded with fuel as described in Table 6-1. The critical experiments are described in detail in References 2-5 and 9-12 and summarized in Section 6.11.10.

The critical experiments consisted of water moderated, oxide fuel arrays in square lattices. Fourteen experiments were 15x8 fuel rod lattices, with 4.31 weight percent (w/o) U-235 enrichment, and different absorber plates in the water gaps between rods. The absorber plates include aluminum, Type 304L stainless steel, Type 304L stainless steel with various boron enrichments, zircaloy-4, and Boral™. Thirteen experiments were 15x15 fuel rod lattices using multiple enrichments, no absorbers between rod clusters, and gadonium absorber integral to the

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fuel in most cases (9 cases). The lattice arrays in these experiments had enrichments of 2.46, 2.73, 2.74, 2.75, 2.76, 2.77, or 2.78 w/o U-235. Comparison with these experiments demonstrates the applicability of the criticality calculation method.

### 6.10.2. *Bias Determination*

A set of Upper Subcritical Limits is determined using the results from the 27 critical experiments and USL Method 1, Confidence Band with Administrative Margin, described in Section 4.0 of NUREG/CR-6361 (Reference 7). The USL Method 1 applies a statistical calculation of the method bias and its uncertainty plus an administrative margin ( $0.05 \Delta k$ ) to a linear fit of the critical experiment benchmark data. The USLs are determined as a function of the critical experiment system parameters; enrichment, water-to-fuel ratio, hydrogen-to- U-235 ratio, pin pitch, average energy of the lethargy causing fission, and the average energy group causing fission.

- The following equation is determined for the USL as a function of enrichment:

$$\text{USL} = 0.9388 + (8.6824 \times 10^{-4}) x \quad \text{for all } x$$

*The variance of the equation fit is  $3.6827 \times 10^{-6}$ . The applicable range for enrichment is  $2.46 \leq x \leq 4.31$ .*
- The following equation is determined for the USL as a function of water-to-fuel ratio:

$$\text{USL} = 0.9398 + (6.6864 \times 10^{-4}) x \quad \text{for all } x$$

*The variance of the equation fit is  $3.8188 \times 10^{-6}$ . The applicable range for water-to-fuel ratio is  $1.8714 \leq x \leq 3.8832$ .*
- The following equation is determined for the USL as a function of hydrogen-to-U-235:

$$\text{USL} = 0.9380 + (1.4976 \times 10^{-5}) x \quad \text{for all } x$$

*The variance of the equation fit is  $4.1692 \times 10^{-6}$ . The applicable range for hydrogen-to-U-235 ratio is  $200.56 \leq x \leq 255.92$ .*
- The following equation is determined for the USL as a function of pin pitch:

$$\text{USL} = 0.9387 + (1.4894 \times 10^{-3}) x \quad \text{for all } x$$

*The variance of the equation fit is  $3.7993 \times 10^{-6}$ . The applicable range for pin pitch is  $1.6358 \leq x \leq 2.54$ .*
- The following equation is determined for the USL as a function of average energy of the lethargy causing fission:

$$\text{USL} = 0.9423 - (3.8725 \times 10^{-3}) x \quad \text{for all } x$$

*The variance of the equation fit is  $4.1339 \times 10^{-6}$ . The applicable range for average energy of the lethargy causing fission is  $0.1127 \leq x \leq 0.3645$ .*

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- The following equation is determined for the USL as a function of the average energy group causing fission:

$$USL = 0.9281 + (3.9834 \times 10^{-4})x \quad \text{for all } x$$

*The variance of the equation fit is  $4.0641 \times 10^{-6}$ . The applicable range for the average energy group causing fission is  $32.89 \leq x \leq 35.77$ .*

Of the preceding equations, the USL as a function of enrichment is the best correlated to the data since the variance of the equation fit is the smallest. Therefore, the USL as a function of enrichment is used to determine a minimum USL for each fuel assembly type considered for use with the TN-B1 shipping container (Table 6-1). Figure 6-50 shows the USL as a function of enrichment. USL values are calculated as a function of enrichment for each candidate fuel design. All candidate fuel designs have the same maximum enrichment of 5.0 wt. percent U-235. Although the 5.0 wt. percent U-235 enrichment falls outside the range of applicability, ANSI/ANS-8.1 (Reference 6) allows the range of applicability to be extended beyond the range of conditions represented by the benchmarks, as long as that extrapolation is not large. As outlined in Reference 7,  $k(x)-w(x)$  is used to extend the USL curve beyond the range of applicability. Figure 6-50 displays the USL curve extrapolation using  $k(x)-w(x)$ ; the extrapolated USL value corresponding to the 5.0 wt. percent U-235 enrichment is 0.94323. Since the extrapolated value results in a higher USL than the maximum enrichment within the range of applicability would produce, the USL corresponding to the 4.31 wt. percent U-235 enrichment is conservatively selected. Therefore, the USL for the TN-B1 shipping container is 0.94254.

The following equation is used to develop the  $k_{eff}$  for the transportation of fuel in the TN-B1 shipping container:

$$K_{eff} = k_{case} + 2\sigma$$

where:

$k_{case}$  = KENO V.a  $k_{eff}$  for a particular case of interest

$\sigma$  = uncertainty in calculated KENO V.a  $k_{eff}$  for a particular case of interest

The  $k_{eff}$  for each container configuration analyzed in the TN-B1 shipping container criticality analysis is compared to the minimum USL (0.94254) to ensure subcriticality.

The GEMER program has been validated against experiments that have uranium form, chemical composition and moderation/reflection conditions similar to those of this application. For low-enriched  $UO_2$  lattice systems without poison, the calculational bias and bias uncertainty of GEMER is given by (Ref. 13):

$$b^* = 0.017$$

A minimum margin of subcriticality is applied as:

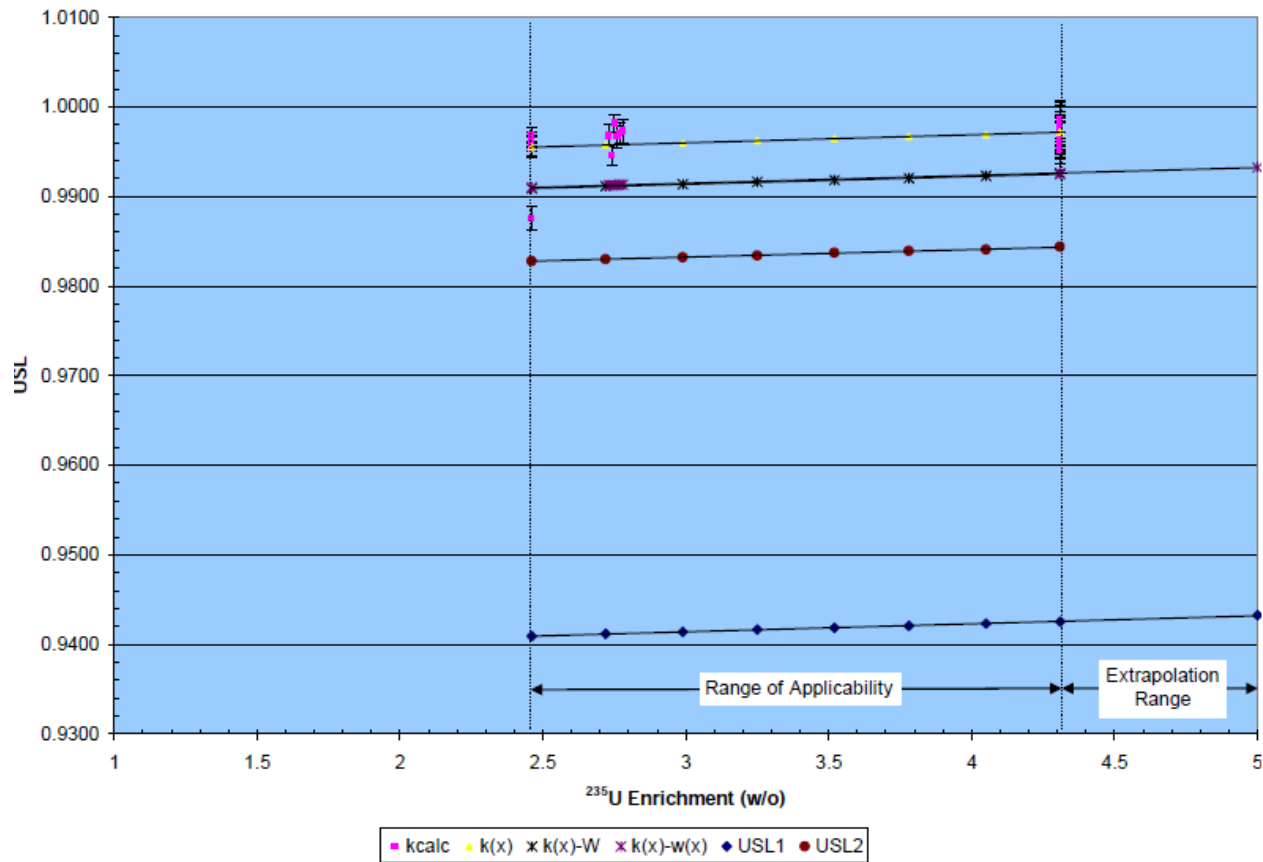
$$\Delta k_m = 0.05$$

Since the GEMER validation benchmarks for heterogeneous  $\text{UO}_2$  systems do not include uranium-carbide (UC) fuel types in the Area of Applicability (AOA), an additional margin  $\Delta k_{AOA} = 0.01$  will be applied for loose UC rods since no UC critical benchmarks are currently available.

Therefore,

For  $\text{UO}_2$  Rods:  $USL = 1 + b^* - \Delta k_m = 1 + (-0.017) - 0.05 = 0.933$

For UC Rods:  $USL = 1 + b^* - \Delta k_m - \Delta k_{AOA} = 1 + (-0.017) - 0.05 - 0.01 = 0.923$



**Figure 6-50 USL as a Function of Enrichment**



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## 6.11. APPENDIX A

### 6.11.1. *Single Package Normal Conditions of Transport Input*

```


=CSAS25                PARM=SIZE=500000
TN-B1 CONTAINER, HAC, NO INTERSPERSED H2O, 100% INNER H2O DENSITY, 5.0 W/O
235U, 12 GAD RODS, SINGLE PACKAGE
44GROUPNDF5            LATTICECELL
UO2                    1  DEN=10.74    1.0  293 92235 5.0 92238 95.0  END
ZR                     2  1.00                293                END H2O
3  1.00                293                END ARBMUO2            10.74
2 1 1 1 92000 1

                        8016 2 4 0.97840 293 92235    5.0
                        92238 95.0  END

ARBMGD2O3              7.407 2 0 1 1 64000 2
                        8016 3 4 0.02160 293                END
H2O                    5  1.00                293                END
SS304                  6  1.00                293                END
POLYETHYLENE           7  DEN=0.080000 1.0  293                END
POLYETHYLENE           8  DEN=0.949 0.25405 293                END
H2O                    8  DEN=1.00  0.74595 293                END
H2O                    9  1.00                293                END
ARBMAL2O3              0.25 2 0 1 0 13027 2 8016 3 10  0.49  END
ARBMSIO2              0.25 2 0 1 0 14000 1 8016 2 10  0.51  END
ZR                     11 1.00                293                END
END COMP
SQUAREPITCH 1.3500 0.8950 1 8 1.01000 2 0.9338 0                END MORE
DATA
RES=4 CYLINDER 0.4475  DAN(4)=2.3197146E-01
END MORE DATA
TN-B1 CONTAINER, HAC, NO INTERSPERSED H2O, 100% INNER H2O DENSITY, 5.0 W/O
235U, 12 GAD RODS, SINGLE PACKAGE
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES RUN=YES END PARM READ GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS
CUBOID      6  1  2P0.0875    2P228.34  2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID      9  1  2P17.713    2P228.34  2P8.829
'INSERT FOAM POLYETHYLENE
HOLE        4          -8.9003        0.00    0.00
HOLE        5           8.9003        0.00    0.00
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID      6  1  2P17.800    2P228.34    8.829  -8.9165

```

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'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS CUBOID  
10 1 2P22.798 2P228.34 8.829 -13.839

'DEFINE THE INNER WALLS OF THE BOX ENDS

CUBOID 6 1 2P22.798 2P228.48 8.829 -13.979

'DEFINE INNER CORE OF BOX ENDS

CUBOID 10 1 2P22.798 2P233.44 8.829 -13.979

'DEFINE OUTER WALLS OF THE INNER BOX

CUBOID 6 1 2P22.938 2P233.58 8.829 -13.979

UNIT 2

COM=!INNER BOX LID!

'DEFINE INNER CORE OF INNER BOX LID

CUBOID 10 1 2P22.798 2P233.44 2P2.48

'DEFINE WALLS FOR INNER BOX LID

CUBOID 6 1 2P22.938 2P233.58 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID! ARRAY

1 3\*0

UNIT 4

COM=!FOAM POLYETHYLENE FOR LEFT ASSEMBLY COMPARTMENT!

CUBOID 9 1 2P7.055 2P228.34 2P7.055

HOLE 70 -6.7500 -192.50 -6.750

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENTS CUBOID 7  
1 2P8.8126 2P228.34 2P8.829

UNIT 5

COM=!FOAM POLYETHYLENE FOR RIGHT ASSEMBLY COMPARTMENT!

CUBOID 9 1 2P7.055 2P228.34 2P7.055

HOLE 70 -6.7500 -192.50 -6.750

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENT CUBOID  
7 1 2P8.8126 2P228.34 2P8.829

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.4475 192.5 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER


0 1 0.4669 192.5  
0

'DEFINE THE FUEL ROD CLADDING/POLY

YCYLINDER 2 1 0.5050 192.5 0

'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE

CUBOID 8 1 2P0.6750 192.5 0 2P0.6750

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UNIT 20

COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE! CUBOID

8 1 2P0.6750 192.5 0 2P0.6750

UNIT 40

COM=!5 W/O FUEL PINS W (2.0 WT % X 0.75) GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 4 1 0.4475 192.5 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER 0

1 0.4669 192.5 0

'DEFINE THE FUEL ROD CLADDING/POLY

YCYLINDER 2 1 0.5050 192.5 0

'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE

CUBOID 8 1 2P0.6750 192.5 0 2P0.6750

UNIT 50

COM=!LOWER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!

ARRAY 2 3\*0

UNIT 60

COM=!UPPER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!

ARRAY 3 3\*0

UNIT 70

COM=!COMPLETE FUEL ASSEMBLY!

ARRAY 4 3\*0

REFLECTOR 11 1 2R0.3048 2R0.0 2R0.3048 1

GLOBAL

UNIT 400

COM=!OUTER CONTAINER BODY AND LID!

'DEFINE INNER REGION OF THE OUTER CONTAINER

CUBOID 3 1 2P35.788 2P253.188 2P31.900

'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER HOLE 3

-22.938 -233.58 -14.024

'DEFINE WALLS OF THE OUTER CONTAINER AND LID

CUBOID 6 1 2P35.963 2P253.363 2P32.075


'GLOBAL

'UNIT 500

'ARRAY 10 3\*0

REFLECTOR 5 1 6R30.48 1

END GEOM

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```

READ ARRAY
ARA=1 NUX=1 NUY=1 NUZ=2
FILL 1 2
END FILL
ARA=2 NUX=10 NUY=1 NUZ=10
FILL 10 10 10 10 10 10 10 10 10 40 40
      10 10 20 10 10 10 40 40 40 40 40
      10 20 10 10 10 10 40 40 40 40 10
      10 10 10 20 20 10 40 40 40 40 10
      10 20 10 20 20 10 10 10 10 10 10
      10 10 20 10 10 20 20 10 10 10 10
      10 20 10 20 10 20 20 10 10 10 10
      10 10 10 10 20 10 10 10 20 10 10
      10 20 10 20 10 20 10 20 10 10 10
      10 10 10 10 10 10 10 10 10 10 10
END FILL
ARA=3 NUX=10 NUY=1 NUZ=10
FILL 10 10 10 10 10 10 10 10 10 40 40
      10 10 10 10 10 10 40 40 40 40 40
      10 10 10 10 10 10 40 40 40 40 10
      10 10 10 20 20 10 40 40 40 40 10
      10 10 10 20 20 10 10 10 10 10 10
      10 10 10 10 10 20 20 10 10 10 10
      10 10 10 10 10 20 20 10 10 10 10
      10 10 10 10 10 10 10 10 10 10 10
      10 10 10 10 10 10 10 10 10 10 10
      10 10 10 10 10 10 10 10 10 10 10
END FILL
ARA=4 NUX=1 NUY=2 NUZ=1
FILL 50 60
END FILL
ARA=10 NUX=21 NUY=3 NUZ=24

FILL F400
END FILL END
ARRAY

READ BNDS ALL VACUUM END
BNDS
END DATA END

```

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### 6.11.2. *Single Package Hypothetical Accident Conditions Input*

```
=CSAS25          PARM=SIZE=500000
TN-B1 CONTAINER, HAC, 12 PART LENGTH RODS, 12 GAD RODS, 1.350 CM PITCH,
PATTERN H, SINGLE PACKAGE
44GROUPNDF5          LATTICECELL
UO2          1  DEN=10.74 1.0 293 92235 5.0 92238 95.0          END
ZR          2          0.26380 293          END
POLYETHYLENE 2  DEN=0.949 0.73620 293          END H2O
3 0.01 293          END ARBMUO2          10.74
2 1 1 1 92000 1
          8016 2 4 0.97840 293 92235 5.0
          92238 95.0 END
ARBMGD2O3 7.407 2 0 1 1 64000 2
          8016 3 4 0.02160 293          END H2O
5 1.00 293          END SS304 6
1.00 293          END H2O 7
1.00 293          END H2O 8
1.00 293          END ZR 9
1.00 293          END ARBMAL2O3 0.25 2
0 1 0 13027 2 8016 3 10 0.49          END ARBMSIO2 0.25 2 0 1 0
14000 1 8016 2 10 0.51          END END COMP
SQUAREPITCH 1.3500 0.8950 1 7 1.19720 2 0.9338 0          END MORE DATA
RES=4 CYLINDER 0.4475 DAN(4)=2.2023524E-01
END MORE DATA
TN-B1 CONTAINER, HAC, 12 PART LENGTH RODS, 12 GAD RODS, 1.350 CM PITCH,
PATTERN H, SINGLE PACKAGE
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES RUN=YES END PARM
READ GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS CUBOID
6 1 2P0.0875 225.20 -228.34 2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 7 1 2P17.713 225.20 -228.34 2P8.829
'PLACE THE FUEL ASSEMBLIES INSIDE INNER BOX
HOLE 70 -15.290 -192.50 -6.477
HOLE 70 2.336 -192.50 -6.477
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 6 1 2P17.800 225.20 -228.34 8.829 -8.9165
'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS
CUBOID 10 1 2P22.798 225.20 -228.34 8.829 -13.839
'DEFINE THE INNER WALLS OF THE BOX ENDS
CUBOID 6 1 2P22.798 225.34 -228.48 8.829 -13.979
'DEFINE INNER CORE OF BOX ENDS -8.1CM IN Y FOR TOTAL DEFORMATION
CUBOID 10 1 2P22.798 225.34 -233.44 8.829 -13.979
```

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'DEFINE OUTER WALLS OF THE INNER BOX -8.1CM IN Y FOR TOTAL DEFORMATION  
CUBOID 6 1 2P22.938 225.48 -233.58 8.829 -13.979

UNIT 2

COM=!INNER BOX LID!

'DEFINE INNER CORE OF INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION

CUBOID 10 1 2P22.798 2P229.39 2P2.48

'DEFINE WALLS FOR INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION

CUBOID 6 1 2P22.938 2P229.53 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID! ARRAY

1 3\*0

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.4475 192.5 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER

0 1 0.4669 192.5 0

'DEFINE THE FUEL ROD CLADDING/POLY

YCYLINDER 2 1 0.5986 192.5 0

'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE

CUBOID 7 1 2P0.6750 192.5 0 2P0.6750

UNIT 20

COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!

CUBOID 7 1 2P0.6750 192.5 0 2P0.6750

UNIT 30

COM=!ARRAY FOR COMPLETE FUEL ASSEMBLY!

ARRAY 2 3\*0

REFLECTOR 9 1 2R0.3048 2R0.0 2R0.3048 1

UNIT 40

COM=!5 W/O FUEL PINS W (2.0 WT % X 0.75) GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 4 1 0.4475 192.5 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER


0 1 0.4669 192.5 0

'DEFINE THE FUEL ROD CLADDING/POLY

YCYLINDER 2 1 0.5986 192.5 0

'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE

CUBOID 7 1 2P0.6750 192.5 0 2P0.6750

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UNIT 50

COM=!LOWER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!

ARRAY 2 3\*0

UNIT 60

COM=!UPPER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR! ARRAY

3 3\*0

UNIT 70

COM=!COMPLETE FUEL ASSEMBLY! ARRAY

4 3\*0

REFLECTOR 9 1 2R0.3048 2R0.0 2R0.3048 1

GLOBAL UNIT 400

COM=!OUTER CONTAINER BODY AND LID!

'DEFINE INNER REGION OF THE OUTER CONTAINER

'MINUS 4.7CM IN Y AND -2.4CM IN Z FOR TOTAL DEFORMATION

CUBOID 0 1 2P35.788 247.960 -253.190 29.500 -31.900

'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER HOLE

3 -22.938 -229.53 -14.024

'DEFINE WALLS OF THE OUTER CONTAINER AND LID

CUBOID 6 1 2P35.963 248.135 -253.365 29.675 -32.075

'GLOBAL

'UNIT 500

'ARRAY 10 3\*0

REFLECTOR 5 1 6R30.48 1

END GEOM

READ ARRAY

ARA=1 NUX=1 NUY=1 NUZ=2

FILL 1 2

END FILL

ARA=2 NUX=10 NUY=1 NUZ=10

FILL 10 10 10 10 10 10 10 10 10 40 40

10 10 20 10 10 10 40 40 40 40

10 20 10 10 10 10 40 40 40 10

10 10 10 20 20 10 40 40 40 10

10 20 10 20 20 10 10 10 10 10

10 10 20 10 10 20 20 10 10 10

10 20 10 20 10 20 20 10 10 10

10 10 10 10 20 10 10 10 20 10

10 20 10 20 10 20 10 20 10 10

10 10 10 10 10 10 10 10 10 10



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END FILL

ARA=3 NUX=10 NUY=1 NUZ=10

FILL 10 10 10 10 10 10 10 10 10 40 40  
10 10 10 10 10 10 10 40 40 40 40  
10 10 10 10 10 10 10 40 40 40 10  
10 10 10 20 20 10 40 40 40 10  
10 10 10 20 20 10 10 10 10 10  
10 10 10 10 10 20 20 10 10 10  
10 10 10 10 10 20 20 10 10 10  
10 10 10 10 10 10 10 10 10 10  
10 10 10 10 10 10 10 10 10 10  
10 10 10 10 10 10 10 10 10 10

END FILL

ARA=4 NUX=1 NUY=2 NUZ=1

FILL 50 60

END FILL END ARRAY

READ BNDS ALL=VACUUM

END BNDS

END DATA END

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Handling: None	Page 337/517		

### 6.11.3. *Package Array Normal Conditions of Transport Input*

```
=CSAS25          PARM=SIZE=500000
TN-B1 CONTAINER, HAC, NO INTERSPERSED H2O, 100% INNER H2O DENSITY, 5.0 W/O
235U, 12 GAD RODS, 21 X 3 X 24 ARRAY
44GROUPNDF5          LATTICECELL
UO2          1  DEN=10.74      1.0  293 92235 5.0 92238 95.0  END
ZR           2  1.00              293              END
H2O          3  1.00              293              END
ARBMUO2      10.74 2 1 1 1 92000 1
                                8016 2 4 0.97840 293 92235 5.0
                                92238 95.0  END
ARBMGD2O3    7.407 2 0 1 1 64000 2
                                8016 3 4 0.02160 293              END
H2O          5  1.00              293              END
SS304        6  1.00              293              END
POLYETHYLENE 7  DEN=0.080000 1.0  293              END
POLYETHYLENE 8  DEN=0.949 0.25405 293              END
H2O          8  DEN=1.00 0.74595 293              END
H2O          9  1.00              293              END
ARBMAL2O3    0.25 2 0 1 0 13027 2 8016 3 10 0.49  END
ARBMSIO2     0.25 2 0 1 0 14000 1 8016 2 10 0.51  END
ZR           11 1.00              293              END
END COMP
SQUAREPITCH 1.3500 0.8950 1 8 1.01000 2 0.9338 0  END
MORE DATA
RES=4 CYLINDER 0.4475  DAN(4)=2.3197146E-01
END MORE DATA
TN-B1 CONTAINER, HAC, NO INTERSPERSED H2O, 100% INNER H2O DENSITY, 5.0 W/O
235U, 12 GAD RODS, 21 X 3 X 24 ARRAY
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES RUN=YES END PARM READ
GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS
CUBOID      6  1  2P0.0875  2P228.34  2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID      9  1  2P17.713  2P228.34  2P8.829
'INSERT FOAM POLYETHYLENE
HOLE        4      -8.9003      0.00      0.00
HOLE        5       8.9003      0.00      0.00
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID      6  1  2P17.800  2P228.34  8.829 -8.9165
'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS
CUBOID     10  1  2P22.798  2P228.34  8.829 -13.839
'DEFINE THE INNER WALLS OF THE BOX ENDS
```

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CUBOID 6 1 2P22.798 2P228.48 8.829 -13.979

'DEFINE INNER CORE OF BOX ENDS

CUBOID 10 1 2P22.798 2P233.44 8.829 -13.979

'DEFINE OUTER WALLS OF THE INNER BOX

CUBOID 6 1 2P22.938 2P233.58 8.829 -13.979

UNIT 2

COM=!INNER BOX LID!

'DEFINE INNER CORE OF INNER BOX LID

CUBOID 10 1 2P22.798 2P233.44 2P2.48

'DEFINE WALLS FOR INNER BOX LID

CUBOID 6 1 2P22.938 2P233.58 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID!

ARRAY 1 3\*0

UNIT 4

COM=!FOAM POLYETHYLENE FOR LEFT ASSEMBLY COMPARTMENT!

CUBOID 9 1 2P7.055 2P228.34 2P7.055

HOLE 70 -6.7500 -192.50 -6.750

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENTS

CUBOID 7 1 2P8.8126 2P228.34 2P8.829

UNIT 5

COM=!FOAM POLYETHYLENE FOR RIGHT ASSEMBLY COMPARTMENT!

CUBOID 9 1 2P7.055 2P228.34 2P7.055

HOLE 70 -6.7500 -192.50 -6.750

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENT

CUBOID 7 1 2P8.8126 2P228.34 2P8.829

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.4475 192.5 0

'DEFINE THE PELLET-CLAD GAP

YCYLINDER 0 1 0.4669 192.5 0

'DEFINE THE FUEL ROD CLADDING/POLY

YCYLINDER 2 1 0.5050 192.5 0


'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE

CUBOID 8 1 2P0.6750 192.5 0 2P0.6750

UNIT 20

COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!

CUBOID 8 1 2P0.6750 192.5 0 2P0.6750

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UNIT 40

COM=!5 W/O FUEL PINS W (2.0 WT % X 0.75) GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 4 1 0.4475 192.5 0

'DEFINE THE PELLET-CLAD GAP

YCYLINDER 0 1 0.4669 192.5 0

'DEFINE THE FUEL ROD CLADDING/POLY

YCYLINDER 2 1 0.5050 192.5 0

'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE

CUBOID 8 1 2P0.6750 192.5 0 2P0.6750

UNIT 50

COM=!LOWER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!

ARRAY 2 3\*0

UNIT 60

COM=!UPPER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!

ARRAY 3 3\*0

UNIT 70

COM=!COMPLETE FUEL ASSEMBLY! ARRAY

4 3\*0

REFLECTOR 11 1 2R0.3048 2R0.0 2R0.3048 1

UNIT 400

COM=!OUTER CONTAINER BODY AND LID!

'DEFINE INNER REGION OF THE OUTER CONTAINER

CUBOID 3 1 2P35.788 2P253.188 2P31.900

'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER

HOLE 3 -22.938 -233.58 -14.024

'DEFINE WALLS OF THE OUTER CONTAINER AND LID

CUBOID 6 1 2P35.963 2P253.363 2P32.075

GLOBAL UNIT

500

ARRAY 10 3\*0

REFLECTOR 5 1 6R30.48 1

END GEOM

READ ARRAY


ARA=1 NUX=1 NUY=1 NUZ=2

FILL 1 2

END FILL

ARA=2 NUX=10 NUY=1 NUZ=10

FILL 10 10 10 10 10 10 10 10 40 40

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```

10 10 20 10 10 10 40 40 40 40
10 20 10 10 10 10 40 40 40 10
10 10 10 20 20 10 40 40 40 10
10 20 10 20 20 10 10 10 10 10
10 10 20 10 10 20 20 10 10 10
10 20 10 20 10 20 20 10 10 10
10 10 10 10 20 10 10 10 20 10
10 20 10 20 10 20 10 20 10 10
10 10 10 10 10 10 10 10 10 10

```

END FILL

ARA=3 NUX=10 NUY=1 NUZ=10

```

FILL 10 10 10 10 10 10 10 10 40 40
10 10 10 10 10 10 40 40 40 40
10 10 10 10 10 10 40 40 40 10
10 10 10 20 20 10 40 40 40 10
10 10 10 20 20 10 10 10 10 10
10 10 10 10 10 20 20 10 10 10
10 10 10 10 10 20 20 10 10 10
10 10 10 10 10 10 10 10 10 10
10 10 10 10 10 10 10 10 10 10
10 10 10 10 10 10 10 10 10 10

```

END FILL

ARA=4 NUX=1 NUY=2 NUZ=1

FILL 50 60

END FILL

ARA=10 NUX=21 NUY=3 NUZ=24

FILL F400

END FILL

END ARRAY

READ BNDS ALL = VACUUM END

BNDS

END DATA END

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#### 6.11.4. *Package Array Hypothetical Accident Conditions Input*

##### 6.11.4.1. **GNF 10x10**

```
=CSAS25          PARM=SIZE=500000
TN-B1 CONTAINER, HAC, 100% H2O DENSITY, WORSTCASE, GNF 10x10, 10 X 1 X 10
ARRAY
44GROUPNDF5          LATTICECELL
UO2          1  DEN=10.74 1.0 293 92235 5.0 92238 95.0          END
ZR          2          0.26380 293          END
POLYETHYLENE 2  DEN=0.949 0.73620 293          END
H2O          3  0.01 293          END
ARBMUO2      10.74 2 1 1 1 92000 1
                        8016 2 4 0.97840 293 92235 5.0
                        92238 95.0 END
ARBMGD2O3    7.407 2 0 1 1 64000 2
                        8016 3 4 0.02160 293          END
H2O          5  1.00 293          END
SS304        6  1.00 293          END
H2O          7  1.00 293          END
POLYETHYLENE 8  DEN=0.080000 1.0 293          END
ZR          9  1.00 293          END
ARBMAL2O3    0.25 2 0 1 0 13027 2 8016 3 10 0.49          END
ARBMSIO2     0.25 2 0 1 0 14000 1 8016 2 10 0.51          END
END COMP
SQUAREPITCH 1.3500 0.8950 1 7 1.19720 2 0.9338 0          END
MORE DATA
RES=4 CYLINDER 0.4475 DAN(4)=2.2023524E-01
END MORE DATA
TN-B1 CONTAINER, HAC, 100% H2O DENSITY, WORSTCASE, GNF 10x10, 10 X 1 X 10
ARRAY
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES RUN=YES END PARM READ
GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS CUBOID
      6 1 2P0.0875 225.20 -228.34 2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 7 1 2P17.713 225.20 -228.34 2P8.829
'INSERT FOAM POLYETHYLENE AND FUEL
HOLE 4 -8.9001 0.00 0.00
HOLE 5 8.9001 0.00 0.00
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 6 1 2P17.800 225.20 -228.34 8.829 -8.9165
```

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'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS  
CUBOID 10 1 2P22.798 225.20 -228.34 8.829 -13.839  
'DEFINE THE INNER WALLS OF THE BOX ENDS  
CUBOID 6 1 2P22.798 225.34 -228.48 8.829 -13.979  
'DEFINE INNER CORE OF BOX ENDS -8.1CM IN Y FOR TOTAL DEFORMATION  
CUBOID 10 1 2P22.798 225.34 -233.44 8.829 -13.979  
'DEFINE OUTER WALLS OF THE INNER BOX -8.1CM IN Y FOR TOTAL DEFORMATION  
CUBOID 6 1 2P22.938 225.48 -233.58 8.829 -13.979

#### UNIT 2

COM=!INNER BOX LID!  
'DEFINE INNER CORE OF INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION  
CUBOID 10 1 2P22.798 2P229.39 2P2.48  
'DEFINE WALLS FOR INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION  
CUBOID 6 1 2P22.938 2P229.53 2P2.62

#### UNIT 3

COM=!INNER BOX WITH ENDS AND LID!  
ARRAY 1 3\*0

#### UNIT 4

COM=!FOAM POLYETHYLENE FOR LEFT ASSEMBLY COMPARTMENT!  
CUBOID 7 1 2P7.055 225.20 -228.34 2P7.055  
HOLE 70 -6.7500 -192.50 -6.750  
'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENTS  
CUBOID 8 1 2P8.8126 225.20 -228.34 2P8.829

#### UNIT 5

COM=!FOAM POLYETHYLENE FOR RIGHT ASSEMBLY COMPARTMENT!  
CUBOID 7 1 2P7.055 225.20 -228.34 2P7.055  
HOLE 70 -6.7500 -192.50 -6.750  
'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENT  
CUBOID 8 1 2P8.8126 225.20 -228.34 2P8.829

#### UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!  
'DEFINE THE FUEL PELLET  
YCYLINDER 1 1 0.4475 192.5 0  
'DEFINE THE PELLET-CLAD GAP YCYLINDER 0 1  
0.4669 192.5 0  
'DEFINE THE FUEL ROD CLADDING/POLY  
YCYLINDER 2 1 0.5986 192.5 0  
'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE  
CUBOID 7 1 2P0.6750 192.5 0 2P0.6750



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UNIT 20  
COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!  
CUBOID 7 1 2P0.6750 192.5 0 2P0.6750

UNIT 30  
COM=!ARRAY FOR COMPLETE FUEL ASSEMBLY! ARRAY  
2 3\*0  
REFLECTOR 9 1 2R0.3048 2R0.0 2R0.3048 1

UNIT 40  
COM=!5 W/O FUEL PINS W (2.0 WT % X 0.75) GAD!  
'DEFINE THE FUEL PELLET  
CYLINDER 4 1 0.4475 192.5 0  
'DEFINE THE PELLET-CLAD GAP  
CYLINDER 0 1 0.4669 192.5 0  
'DEFINE THE FUEL ROD CLADDING/POLY  
CYLINDER 2 1 0.5986 192.5 0  
'DEFINE THE FUEL ROD PITCH FILLED WITH POLYETHYLENE  
CUBOID 7 1 2P0.6750 192.5 0 2P0.6750


UNIT 50  
  
COM=!LOWER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!  
ARRAY 2 3\*0

UNIT 60  
COM=!UPPER HALF FUEL ASSEMBLY WITH CLUSTER SEPARATOR!  
ARRAY 3 3\*0

UNIT 70  
COM=!COMPLETE FUEL ASSEMBLY!  
ARRAY 4 3\*0  
REFLECTOR 9 1 2R0.3048 2R0.0 2R0.3048 1

UNIT 400  
COM=!OUTER CONTAINER BODY AND LID!  
'DEFINE INNER REGION OF THE OUTER CONTAINER  
'MINUS 4.7CM IN Y AND -2.4CM IN Z FOR TOTAL DEFORMATION  
CUBOID 0 1 2P35.788 247.960 -253.190 29.500 -31.900  
'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER  
HOLE 3 -22.938 -229.53 -14.024  
'DEFINE WALLS OF THE OUTER CONTAINER AND LID  
CUBOID 6 1 2P35.963 248.135 -253.365 29.675 -32.075

GLOBAL  
UNIT 500  
ARRAY 10 3\*0

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REFLECTOR 5 1 6R30.48 1  
END GEOM

READ ARRAY

ARA=1 NUX=1 NUY=1 NUZ=2

FILL 1 2

END FILL

ARA=2 NUX=10 NUY=1 NUZ=10

FILL 10 10 10 10 10 10 10 10 10 40 40  
10 10 20 10 10 10 40 40 40 40  
10 20 10 10 10 10 40 40 40 10  
10 10 10 20 20 10 40 40 40 10  
10 20 10 20 20 10 10 10 10 10  
10 10 20 10 10 20 20 10 10 10  
10 20 10 20 10 20 20 10 10 10  
10 10 10 10 20 10 10 10 20 10  
10 20 10 20 10 20 10 20 10 10  
10 10 10 10 10 10 10 10 10 10

END FILL

ARA=3 NUX=10 NUY=1 NUZ=10

FILL 10 10 10 10 10 10 10 10 10 40 40  
10 10 10 10 10 10 40 40 40 40  
10 10 10 10 10 10 40 40 40 10  
10 10 10 20 20 10 40 40 40 10  
10 10 10 20 20 10 10 10 10 10  
10 10 10 10 10 20 20 10 10 10  
10 10 10 10 10 20 20 10 10 10  
10 10 10 10 10 10 10 10 10 10  
10 10 10 10 10 10 10 10 10 10  
10 10 10 10 10 10 10 10 10 10

END FILL

ARA=4 NUX=1 NUY=2 NUZ=1

FILL 50 60

END FILL

ARA=10 NUX=10 NUY=1 NUZ=10

FILL F400

END FILL END

ARRAY

READ BNDS ALL=VACUUM END

BNDS


END DATA END

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### 6.11.5. Single Package Loose Rods Normal Conditions of Transport Input

```
=CSAS25                PARM=SIZE=500000
TN-B1 CONTAINER, 8, NTC, 100% H2O, 2.8150 CM PITCH, LOOSE FUEL RODS, SINGLE
PACKAGE
44GROUPNDF5           LATTICECELL
UO2                   1  DEN=10.74    1.0  293 92235 5.0 92238 95.0  END
POLYETHYLENE          2  DEN=0.925    1.0  293                                END
H2O                   3  1.00                293                                END
UO2                   4  DEN=10.4799 1.0  293 92235 3.25 92238 96.75 END
GD                    4  DEN=0.17374 1.0  293                                END
O                    4  DEN=0.026514 1.0  293                                END
H2O                   5  1.00                293                                END
SS304                 6  1.00                293                                END
H2O                   8  1.00                293                                END
H2O                   9  1.00                293                                END
ARBMAL2O3             0.25 2 0 1 0 13027 2 8016 3 10 0.49  END
ARBMSIO2              0.25 2 0 1 0 14000 1 8016 2 10 0.51  END
ZR                    11 1.00                293                                END
END COMP
SQUAREPITCH 2.8150 1.0500 1 8 1.13048 2 1.100 0          END
TN-B1 CONTAINER, 8, NTC, 100% H2O, 2.8150 CM PITCH, LOOSE FUEL RODS, SINGLE
PACKAGE
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES END PARM READ GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS
CUBOID      6  1  2P0.0875    2P228.34  2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID      3  1  2P17.713    2P228.34  2P8.829
'INSERT FOAM POLYETHYLENE
HOLE        4      -8.9003      0.00    0.00
HOLE        5       8.9003      0.00    0.00
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID      6  1  2P17.800    2P228.34    8.829  -8.9165
'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS
CUBOID     10  1  2P22.798    2P228.34    8.829  -13.839
'DEFINE THE INNER WALLS OF THE BOX ENDS
CUBOID      6  1  2P22.798    2P228.48    8.829  -13.979
'DEFINE INNER CORE OF BOX ENDS
CUBOID     10  1  2P22.798    2P233.44    8.829  -13.979
'DEFINE OUTER WALLS OF THE INNER BOX
CUBOID      6  1  2P22.938    2P233.58    8.829  -13.979
```

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UNIT 2

COM=!INNER BOX LID!

'DEFINE INNER CORE OF INNER BOX LID

CUBOID 10 1 2P22.798 2P233.44 2P2.48

'DEFINE WALLS FOR INNER BOX LID

CUBOID 6 1 2P22.938 2P233.58 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID! ARRAY

1 3\*0

UNIT 4

COM=!FOAM POLYETHYLENE FOR LEFT ASSEMBLY COMPARTMENT!

CUBOID 3 1 2P7.0378 2P228.34 2P7.054

HOLE 30 -7.0376 -191.77 -7.0376

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENTS

CUBOID 7 1 2P8.8126 2P228.34 2P8.829

UNIT 5

COM=!FOAM POLYETHYLENE FOR RIGHT ASSEMBLY COMPARTMENT!

CUBOID 3 1 2P7.0378 2P228.34 2P7.054

HOLE 30 -7.0376 -191.77 -7.0376

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENT

CUBOID 7 1 2P8.8126 2P228.34 2P8.829

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER 0 1  
0.55000 381 0

'DEFINE THE FUEL ROD CLADDING YCYLINDER

2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 2P1.40750 381 0 2P1.40750

UNIT 20


COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!

CUBOID 8 1 2P1.40750 381 0 2P1.40750

UNIT 30

COM=!ARRAY FOR COMPLETE FUEL ASSEMBLY!

ARRAY 2 3\*0

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```

UNIT 400
COM=!OUTER CONTAINER BODY AND LID!
'DEFINE INNER REGION OF THE OUTER CONTAINER
CUBOID 3 1 2P35.788 2P253.188 2P31.900
'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER
HOLE 3 -22.938 -233.58 -14.024
'DEFINE WALLS OF THE OUTER CONTAINER AND LID
CUBOID 6 1 2P35.963 2P253.363 2P32.075

```

```

GLOBAL
UNIT 500
ARRAY 10 3*0
REFLECTOR 5 1 6R30.48 1
END GEOM

```

```

READ ARRAY
ARA=1 NUX=1 NUY=1 NUZ=2
FILL 1 2
END FILL
ARA=2 NUX=5 NUY=1 NUZ=5

```

```

FILL 10 10 10 10 10
      10 10 10 10 10
      10 10 10 10 10
      10 10 10 10 10
      10 10 10 10 10
END FILL
ARA=10 NUX=21 NUY=3 NUZ=24
FILL F400
END FILL
END ARRAY

```

```

READ BNDS ALL=VACUUM END
BNDS
END DATA
END

```

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### 6.11.6. Single Package Loose Fuel Rods Hypothetical Accident Conditions Input

```
=CSAS25                PARM=SIZE=500000
TN-B1 CONTAINER, 8, HAC, 100% H2O, WORST CASE MODEL, 3.0056 CM PITCH, LOOSE FUEL
RODS, SINGLE PACKAGE
44GROUPNDF5            LATTICECELL
UO2                    1  DEN=10.74 1.0 293 92235 5.0 92238 95.0      END
POLYETHYLENE          2  DEN=0.925 1.0 293                          END
H2O                   3  1.00 293                                END
UO2                   4  DEN=10.4799 1.0 293 92235 3.25 92238 96.75END
GD                    4  DEN=0.17374 1.0 293                      END
O                    4  DEN=0.026514 1.0 293                    END
H2O                   5  1.00 293                                END
SS304                 6  1.00 293                                END
H2O                   7  DEN=1.00 1.0 293                      END
H2O                   8  DEN=1.00 1.0 293                      END
ZR                    9  1.00 293                                END
ARBMAL2O3             0.25 2 0 1 0 13027 2 8016 3 10 0.49      END
ARBMSIO2              0.25 2 0 1 0 14000 1 8016 2 10 0.51      END
END COMP

SQUAREPITCH 3.0056 1.0500 1 8 1.13048 2 1.100 0              END
TN-B1 CONTAINER, 8, HAC, 100% H2O, WORST CASE MODEL, 3.0056 CM PITCH, LOOSE FUEL
RODS, SINGLE PACKAGE
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES ENDPARM READ
GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS
CUBOID 6 1 2P0.0875 225.20 -228.34 2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 7 1 2P17.713 225.20 -228.34 2P8.829
'PLACE THE FUEL ASSEMBLIES INSIDE INNER BOX
HOLE 30 -16.413 -190.50 -7.514
HOLE 30 1.386 -190.50 -7.514
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 6 1 2P17.800 225.20 -228.34 8.829 -8.9165
'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS
CUBOID 10 1 2P22.798 225.20 -228.34 8.829 -13.839
'DEFINE THE INNER WALLS OF THE BOX ENDS
CUBOID 6 1 2P22.798 225.34 -228.48 8.829 -13.979
'DEFINE INNER CORE OF BOX ENDS -8.1CM IN Y FOR TOTAL DEFORMATION
CUBOID 10 1 2P22.798 225.34 -233.44 8.829 -13.979
'DEFINE OUTER WALLS OF THE INNER BOX -8.1CM IN Y FOR TOTAL DEFORMATION
CUBOID 6 1 2P22.938 225.48 -233.58 8.829 -13.979
```

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UNIT 2

COM=!INNER BOX LID!

'DEFINE INNER CORE OF INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION

CUBOID 10 1 2P22.798 2P229.39 2P2.48

'DEFINE WALLS FOR INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION

CUBOID 6 1 2P22.938 2P229.53 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID! ARRAY

1 3\*0

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER

0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 2P1.50280 381 0 2P1.50280

UNIT 20

COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!

CUBOID 8 1 2P1.50280 381 0 2P1.50280

UNIT 30

COM=!ARRAY FOR COMPLETE FUEL ASSEMBLY!

ARRAY 2 3\*0

GLOBAL UNIT 400

COM=!OUTER CONTAINER BODY AND LID!

'DEFINE INNER REGION OF THE OUTER CONTAINER

'MINUS 4.7CM IN Y AND -2.4CM IN Z FOR TOTAL DEFORMATION

CUBOID 0 1 2P35.788 247.960 -253.190 29.500 -31.900

'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER

HOLE 3 -22.938 -229.53 -14.024

'DEFINE WALLS OF THE OUTER CONTAINER AND LID

CUBOID 6 1 2P35.963 248.135 -253.365 29.675 -32.075

'GLOBAL

'UNIT 500

'ARRAY 10 3\*0

REFLECTOR 5 1 6R30.48 1

END GEOM



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```

READ ARRAY
ARA=1 NUX=1 NUY=1 NUZ=2
FILL 1 2
END FILL
ARA=2 NUX=5 NUY=1 NUZ=5
FILL 10 10 10 10 10
      10 10 10 10 10
      10 10 10 10 10
      10 10 10 10 10
      10 10 10 10 10
END FILL
ARA=10 NUX=14 NUY=2 NUZ=16
FILL F400
END FILL END
ARRAY

READ BNDS ALL=VACUUM END
BNDS
END DATA


```

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### 6.11.7. Package Array Loose Fuel Rods Normal Conditions of Transport Input

```
=CSAS25                PARM=SIZE=500000
TN-B1 CONTAINER, 8, NTC, 100% H2O, 2.8150 CM PITCH, LOOSE FUEL RODS, 21 x 3x 24
44GROUPNDF5            LATTICECELL
UO2                    1  DEN=10.74      1.0  293 92235 5.0 92238 95.0 END
POLYETHYLENE          2  DEN=0.925      1.0  293                                END
H2O                   3  1.00                                293                                END
UO2                   4  DEN=10.4799  1.0  293 92235 3.25 92238 96.75
END GD                4  DEN=0.17374  1.0  293
END
O                      4  DEN=0.026514 1.0  293
END H2O               5  1.00                                293
END SS304             6  1.00                                293
END POLYETHYLENE      7  DEN=0.067967 1.0  293
END H2O              8  1.00                                293
END H2O              9  1.00                                293
END ARBMAL2O3         0.25 2 0 1 0 13027 2 8016 3 10 0.49
END ARBMSIO2         0.25 2 0 1 0 14000 1 8016 2 10 0.51
END ZR               11 1.00                                293
END END COMP
SQUAREPITCH 2.8150 1.0500 1 8 1.13048 2 1.100 0                                END
TN-B1 CONTAINER, 8, NTC, 100% H2O, 2.8150 CM PITCH, LOOSE FUEL RODS, 21 x 3 x 24
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES END PARM READ GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS
CUBOID 6 1 2P0.0875 2P228.34 2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 3 1 2P17.713 2P228.34 2P8.829
'INSERT FOAM POLYETHYLENE
HOLE 4 -8.9003 0.00 0.00
HOLE 5 8.9003 0.00 0.00
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 6 1 2P17.800 2P228.34 8.829 -8.9165
'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS
CUBOID 10 1 2P22.798 2P228.34 8.829 -13.839
'DEFINE THE INNER WALLS OF THE BOX ENDS
CUBOID 6 1 2P22.798 2P228.48 8.829 -13.979
'DEFINE INNER CORE OF BOX ENDS
CUBOID 10 1 2P22.798 2P233.44 8.829 -13.979
'DEFINE OUTER WALLS OF THE INNER BOX
CUBOID 6 1 2P22.938 2P233.58 8.829 -13.979
```

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UNIT 2

COM=!INNER BOX LID!

'DEFINE INNER CORE OF INNER BOX LID

CUBOID 10 1 2P22.798 2P233.44 2P2.48

'DEFINE WALLS FOR INNER BOX LID

CUBOID 6 1 2P22.938 2P233.58 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID! ARRAY 1 3\*0

UNIT 4

COM=!FOAM POLYETHYLENE FOR LEFT ASSEMBLY COMPARTMENT!

CUBOID 3 1 2P7.0378 2P228.34 2P7.054

HOLE 30 -7.0376 -191.77 -7.0376

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENTS

CUBOID 7 1 2P8.8126 2P228.34 2P8.829

UNIT 5

COM=!FOAM POLYETHYLENE FOR RIGHT ASSEMBLY COMPARTMENT!

CUBOID 3 1 2P7.0378 2P228.34 2P7.054

HOLE 30 -7.0376 -191.77 -7.0376

'FOAM POLYETHYLENE FOR ASSEMBLY COMPARTMENT

CUBOID 7 1 2P8.8126 2P228.34  
2P8.829

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 2P1.40750 381 0  
2P1.40750

UNIT 20

COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!

CUBOID 8 1 2P1.40750 381 0  
2P1.40750

UNIT 30

COM=!ARRAY FOR COMPLETE FUEL ASSEMBLY! ARRAY 2 3\*0

UNIT 400

COM=!OUTER CONTAINER BODY AND LID!

'DEFINE INNER REGION OF THE OUTER CONTAINER

CUBOID 3 1 2P35.788 2P253.188  
2P31.900

'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER

HOLE 3 -22.938 -233.58 -14.024

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'DEFINE WALLS OF THE OUTER CONTAINER AND LID  
CUBOID        6        1        2P35.963        2P253.363        2P32.075  
  
GLOBAL UNIT 500  
ARRAY 10 3\*0  
REFLECTOR 5        1        6R30.48        1  
END GEOM  
  
READ ARRAY  
ARA=1 NUX=1 NUY=1 NUZ=2  
FILL 1 2  
END FILL  
ARA=2 NUX=5        NUY=1 NUZ=5  
  
FILL 10 10 10 10 10  
10 10 10 10 10  
10 10 10 10 10  
10 10 10 10 10  
10 10 10 10 10  
END FILL  
ARA=10 NUX=21 NUY=3        NUZ=24  
FILL F400  
END FILL  
END ARRAY  
  
READ BNDS ALL=VACUUM END BNDS  
END DATA  
END


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### 6.11.8. Package Array Loose Fuel Rods Hypothetical Accident Conditions Input

```
=CSAS25                PARM=SIZE=500000
TN-B1 CONTAINER, 8, HAC, 100% H2O, WORST CASE MODEL, 3.0056 CM PITCH, LOOSE
FUEL RODS, 10 X 1 X 10 ARRAY
44GROUPNDF5            LATTICECELL
UO2                    1  DEN=10.74 1.0 293 92235 5.0 92238 95.0      END
POLYETHYLENE          2  DEN=0.925 1.0 293                          END
H2O                   3  1.00 293                                END
H2O                   5  1.00 293                                END
SS304                 6  1.00 293                                END
POLYETHYLENE          7  DEN=0.08000 1.0 293                      END
H2O                   8  DEN=1.00 1.0 293                      END
ZR                    9  1.00 293                                END
ARBMAL2O3             0.25 2 0 1 0 13027 2 8016 3 10 0.49        END
ARBMSIO2              0.25 2 0 1 0 14000 1 8016 2 10 0.51        END
END COMP
SQUAREPITCH 3.0056 1.0500 1 8 1.13048 2 1.100 0                END
TN-B1 CONTAINER, 8, HAC, 100% H2O, WORST CASE MODEL, 3.0056 CM PITCH, LOOSE
FUEL RODS, 10 X 1 X 10 ARRAY
READ PARM TME=400 GEN=400 NPG=2500 NSK=50 NUB=YES END PARM READ GEOM

UNIT 1
COM=!CONTAINER INNER BOX!
'DEFINE GEOMETRY FOR SEPARATOR PLATE BETWEEN ASSEMBLY COMPARTMENTS
CUBOID 6 1 2P0.0875 225.20 -228.34 2P8.829
'DEFINE REGION FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 7 1 2P17.713 225.20 -228.34
2P8.829
'PLACE THE FUEL ASSEMBLIES INSIDE INNER BOX
HOLE 30 -15.913 -190.50 -7.014
HOLE 30 1.886 -190.50 -7.014
'DEFINE WALLS FOR ASSEMBLY COMPARTMENTS WITHIN INNER BOX
CUBOID 6 1 2P17.800 225.20 -228.34
8.829 -8.9165
'DEFINE REGION OUTSIDE THE WALLS OF THE ASSEMBLY COMPARTMENTS
CUBOID 10 1 2P22.798 225.20 -228.34 8.829 -
13.839
'DEFINE THE INNER WALLS OF THE BOX ENDS
CUBOID 6 1 2P22.798 225.34 -228.48 8.829 -13.979
'DEFINE INNER CORE OF BOX ENDS -8.1CM IN Y FOR TOTAL DEFORMATION
CUBOID 10 1 2P22.798 225.34 -233.44 8.829 -13.979
'DEFINE OUTER WALLS OF THE INNER BOX -8.1CM IN Y FOR TOTAL DEFORMATION
CUBOID 6 1 2P22.938 225.48 -233.58 8.829 -13.979

UNIT 2
COM=!INNER BOX LID!
'DEFINE INNER CORE OF INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION
CUBOID 10 1 2P22.798 2P229.39 2P2.48
```

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'DEFINE WALLS FOR INNER BOX LID -8.1CM IN Y FOR TOTAL DEFORMATION  
CUBOID 6 1 2P22.938 2P229.53 2P2.62

UNIT 3

COM=!INNER BOX WITH ENDS AND LID! ARRAY 1 3\*0

UNIT 10

COM=!5 W/O FUEL PINS W/O GAD!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 2P1.50280 381 0 2P1.50280

UNIT 20

COM=!SPACE WITHIN FUEL ASSEMBLY LATTICE!

CUBOID 8 1 2P1.50280 381 0 2P1.50280

UNIT 30

COM=!ARRAY FOR COMPLETE FUEL ASSEMBLY! ARRAY 2 3\*0

UNIT 40

COM=!5 W/O FUEL PINS W/O GAD LEFT SIDE FOAM!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 1.50280 -1.00280 381 0 2P1.50280

UNIT 46

COM=!5 W/O FUEL PINS W/O GAD LEFT SIDE TOP FOAM!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0


'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 1.50280 -1.00280 381 0 1.00280 -1.50280

UNIT 47

COM=!5 W/O FUEL PINS W/O GAD LEFT SIDE BOTTOM FOAM!

'DEFINE THE FUEL PELLET

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YCYLINDER 1 1 0.52500 381 0  
 'DEFINE THE PELLETT-CLAD GAP YCYLINDER 0 1 0.55000 381 0  
 'DEFINE THE FUEL ROD CLADDING YCYLINDER 2 1 0.56524 381 0  
 'DEFINE THE FUEL ROD PITCH FILLED WITH WATER  
 CUBOID 8 1 1.50280 -1.00280 381 0 1.50280 -1.00280

#### UNIT 50

COM=!5 W/O FUEL PINS W/O GAD RIGHT SIDE FOAM!

'DEFINE THE FUEL PELLETT

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLETT-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 1.00280 -1.50280 381 0 2P1.50280

#### UNIT 56

COM=!5 W/O FUEL PINS W/O GAD RIGHT SIDE TOP FOAM!

'DEFINE THE FUEL PELLETT

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLETT-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 1.00280 -1.50280 381 0 1.00280 -1.50280

#### UNIT 57

COM=!5 W/O FUEL PINS W/O GAD RIGHT BOTTOM SIDE FOAM!

'DEFINE THE FUEL PELLETT

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLETT-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 1.00280 -1.50280 381 0 1.50280 -1.00280

#### UNIT 60

COM=!5 W/O FUEL PINS W/O GAD TOP SIDE FOAM!

'DEFINE THE FUEL PELLETT

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLETT-CLAD GAP


YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER



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CUBOID 8 1 2P1.50280 381 0 1.00280 -1.50280

UNIT 70

COM=!5 W/O FUEL PINS W/O GAD BOTTOM SIDE FOAM!

'DEFINE THE FUEL PELLET

YCYLINDER 1 1 0.52500 381 0

'DEFINE THE PELLET-CLAD GAP

YCYLINDER 0 1 0.55000 381 0

'DEFINE THE FUEL ROD CLADDING

YCYLINDER 2 1 0.56524 381 0

'DEFINE THE FUEL ROD PITCH FILLED WITH WATER

CUBOID 8 1 2P1.50280 381 0 1.50280 -1.00280

UNIT 400

COM=!OUTER CONTAINER BODY AND LID!

'DEFINE INNER REGION OF THE OUTER CONTAINER

'MINUS 4.7CM IN Y AND -2.4CM IN Z FOR TOTAL DEFORMATION

CUBOID 0 1 2P35.788 247.960 -253.190 29.500 -31.900

'INNER CONTAINER PLACEMENT WITHIN OUTER CONTAINER

HOLE 3 -22.938 -229.53 -14.024

'DEFINE WALLS OF THE OUTER CONTAINER AND LID

CUBOID 6 1 2P35.963 248.135 -253.365 29.675 -32.075

GLOBAL UNIT 500

ARRAY 10 3\*0

REFLECTOR 5 1 6R30.48 1

END GEOM

READ ARRAY

ARA=1 NUX=1 NUY=1 NUZ=2

FILL 1 2

END FILL

ARA=2 NUX=5 NUY=1 NUZ=5

FILL 47 70 70 70 57

40 10 10 10 50

40 10 10 10 50

40 10 10 10 50

46 60 60 60 56

END FILL

ARA=10 NUX=10 NUY=1 NUZ=10


FILL F400

END FILL END ARRAY

READ BNDS ALL=VACUUM

END BNDS


END DATA END

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### 6.11.9. Data Tables for Figures in TN-B1 CSE


**Table 6-26 Data for Figure 6-25 TN-B1 Array HAC Polyethylene Sensitivity**

Output File Name	Case Description	Interspersed Moderator Density (g/cm <sup>3</sup> )	Polyethylene Mass (kg)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_a10_no interspersedh2o_polyethylenesensitivity_1.284cmpitch_14X2X16	Atrium 10XP+	0.00	0	0.8715	0.0008	0.8731
"	Atrium 10XP+	0.00	10.9	0.8774	0.0009	0.8792
"	Atrium 10XP+	0.00	17.1	0.8813	0.0009	0.8831
"	Atrium 10XP+	0.00	20.4	0.8810	0.0008	0.8826
"	Atrium 10XP+	0.00	22.9	0.8822	0.0009	0.8840
"	Atrium 10XP+	0.00	25.4	0.8847	0.0008	0.8863
"	Atrium 10XP+	0.00	27.9	0.8860	0.001	0.8880
rajll_hac_g10_no interspersedh2o_polyethylenesensitivity_pitch1.2954cm_14X2X16	GNF 10 x 10	0.00	0	0.8863	0.0007	0.8877
"	GNF 10 x 10	0.00	10.9	0.8923	0.0008	0.8939
"	GNF 10 x 10	0.00	17.1	0.8940	0.0008	0.8956
"	GNF 10 x 10	0.00	20.4	0.8955	0.0007	0.8969
"	GNF 10 x 10	0.00	22.9	0.8975	0.0009	0.8993
"	GNF 10 x 10	0.00	25.4	0.8994	0.0008	0.9010
"	GNF 10 x 10	0.00	27.9	0.9001	0.0008	0.9017

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**Table 6-26 Data for Figure 6-25 TN-B1 Array HAC Polyethylene Sensitivity (continued)**

Output File Name	Case Description	Interspersed Moderator Density (g/cm <sup>3</sup> )	Polyethylene Mass (kg)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_polysen s	FANP 9x9	0.00	0	0.8728	0.0009	0.8746
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_polysen s	FANP 9x9	0.00	20	0.8756	0.0009	0.8774
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_channel s	FANP 9x9	0.00	22	0.8755	0.0009	0.8773
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_polysen s	FANP 9x9	0.00	24	0.8769	0.0007	0.8783
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_polysen s	FANP 9x9	0.00	26	0.8758	0.0008	0.8774
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_polysen s	FANP 9x9	0.00	28	0.8766	0.0008	0.8782
rajll_hac_f9_10g ad rods_refassy_ 14x2x16_polysen s	FANP 9x9	0.00	30	0.8776	0.0009	0.8794
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_polyse ns	GNF 9x9	0.00	0	0.8612	0.0008	0.8628
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_polyse ns	GNF 9x9	0.00	20	0.8661	0.0009	0.8679
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_chann els	GNF 9x9	0.00	22	0.8659	0.0008	0.8676
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_polyse ns	GNF 9x9	0.00	24	0.8676	0.0007	0.8690
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_polyse ns	GNF 9x9	0.00	26	0.8670	0.0009	0.8688
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_polyse ns	GNF 9x9	0.00	28	0.8656	0.0009	0.8674
rajll_hac_g9_10g ad rods_refassy_ 14X2X16_polyse ns	GNF 9x9	0.00	30	0.8702	0.0008	0.8718

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**Table 6-26 Data for Figure 6-25 TN-B1 Array HAC Polyethylene Sensitivity (continued)**

Output File Name	Case Description	Interspersed Moderator Density (g/cm <sup>3</sup> )	Polyethylene Mass (kg)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
rajll_hac_g8_noi nterspersedh2o_ polyethylenesens itivity_1.6256cm_ 14X2X16	GNF 8x8	0.00	0	0.8795	0.0009	0.8813
"	GNF 8x8	0.00	19	0.8865	0.0009	0.8883
"	GNF 8x8	0.00	22	0.8900	0.0009	0.8918
"	GNF 8x8	0.00	24	0.8892	0.0008	0.8908
"	GNF 8x8	0.00	26	0.8924	0.0008	0.8940
"	GNF 8x8	0.00	28	0.8915	0.0009	0.8933
"	GNF 8x8	0.00	30	0.8942	0.0009	0.8960

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**Table 6-27 Data for Figure 6-26 TN-B1 Fuel Rod Pitch Sensitivity Study**

<b>Output File Name</b>	<b>Interspersed Moderator Density (g/cm<sup>3</sup>)</b>	<b>Polyethylene Mass (kg)</b>	<b>Pitch (cm)</b>	<b>keff</b>	<b>σ</b>	<b>keff + 2σ</b>
rajll_hac_a10_nointerspersedh2o_pitchsensitivity_14X2X16	0.00	20.4	1.210	0.8301	0.0010	0.8321
“	0.00	20.4	1.284	0.8810	0.0008	0.8826
“	0.00	20.4	1.350	0.9245	0.0009	0.9263
“	0.00	20.4	1.376	0.9391	0.0008	0.9407
rajll_hac_g10_nointerspersedh2o_pitchsensitivity_14X2X16	0.00	20.4	1.1960	0.8394	0.0009	0.8412
“	0.00	20.4	1.2954	0.8955	0.0007	0.8969
“	0.00	20.4	1.350	0.9241	0.0008	0.9257
“	0.00	20.4	1.3760	0.9328	0.0008	0.9344
rajll_hac_f9_10gad rods_refassy_14x2x16_pitch	0.00	22	1.3389	0.8219	0.0008	0.8235
“	0.00	22	1.4478	0.8755	0.0009	0.8773
“	0.00	22	1.5028	0.8998	0.0008	0.9014
rajll_hac_f9_10gad rods_refassy_14x2x16_channels	0.00	22	1.5376	0.9126	0.0009	0.9144
rajll_hac_g9_10gad rods_refassy_14X2X16_pitchsens	0.00	22	1.3260	0.8073	0.0008	0.8089
“	0.00	22	1.4376	0.8659	0.0008	0.8676
“	0.00	22	1.5028	0.8929	0.0008	0.8944
rajll_hac_g9_10gad rods_refassy_14X2X16_channels	0.00	22	1.5376	0.9076	0.0009	0.9095
rajll_hac_g8_nointerspersedh2o_pitchsensitivity_14X2X16	0.00	22	1.4603	0.7968	0.0009	0.7986
“	0.00	22	1.6256	0.8900	0.0009	0.8918
“	0.00	22	1.6923	0.9216	0.0008	0.9232
“	0.00	22	1.7264	0.9384	0.0008	0.9400

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**Table 6-28 Data for Figure 6-27 TN-B1 Array HAC Pellet Diameter Sensitivity Study**


Output File Name	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pellet Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_a10_nointer spersedh2o_pelletod sensitivity_14X2X16	0	0.8000	0.8560	0.0008	0.8576
"	0	0.8400	0.8680	0.0009	0.8698
"	0	0.8882	0.8810	0.0008	0.8826
"	0	0.8941	0.8839	0.0008	0.8855
"	0	0.9200	0.8906	0.0008	0.8922
rajll_hac_g10_nointer spersedh2o_pelletod sensitivity_14X2X16	0	0.8000	0.8641	0.0009	0.8659
"	0	0.8400	0.8796	0.0009	0.8814
"	0	0.8882	0.8941	0.0008	0.8957
"	0	0.8941	0.8955	0.0007	0.8969
"	0	0.9200	0.9050	0.0008	0.9066
rajll_hac_f9_10gadro ds_refassy_14x2x16 _pelletod	0	0.8882	0.8600	0.0008	0.8616
"	0	0.9000	0.8633	0.0009	0.8651
rajll_hac_f9_10gadro ds_refassy_14x2x16 _channels	0	0.9398	0.8755	0.0009	0.8773
rajll_hac_f9_10gadro ds_refassy_14x2x16 _pelletod	0	0.9550	0.8799	0.0008	0.8815
"	0	0.9600	0.8817	0.0007	0.8831
rajll_hac_g9_10gadr ods_refassy_14X2X1 6_pelletodsens	0	0.8882	0.8462	0.0008	0.8478
"	0	0.9000	0.8509	0.0009	0.8527
"	0	0.9398	0.8609	0.0008	0.8625
rajll_hac_g9_10gadr ods_refassy_14X2X1 6_channels	0	0.9550	0.8659	0.0008	0.8676
rajll_hac_g9_10gadr ods_refassy_14X2X1 6_pelletodsens	0	0.9600	0.8678	0.0008	0.8694
rajll_hac_g8_nointers persedh2o_pelletods ensitivity_14X2X16	0	0.9200	0.8566	0.0008	0.8582
"	0	0.9550	0.8648	0.0008	0.8664
"	0	1.0000	0.8783	0.0008	0.8799
"	0	1.0439	0.8900	0.0009	0.8918
"	0	1.0700	0.8940	0.0009	0.8958

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**Table 6-29 Data for Figure 6-28 TN-B1 Array HAC Fuel Rod Clad ID Sensitivity Study**


Output File Name	Moderator Density (g/cm <sup>3</sup> )	Clad Inner Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_a10_nointerspersedh2o_cladidsensitivity_14X2X16	0	0.8800	0.8760	0.0009	0.8778
"	0	0.8900	0.8805	0.0009	0.8823
"	0	0.9218	0.8810	0.0008	0.8826
"	0	0.9322	0.8813	0.0008	0.8829
"	0	1.0330	0.8855	0.0010	0.8875
rajll_hac_g10_nointerspersedh2o_cladidsensitivity_14X2X16	0	0.9000	0.8937	0.0010	0.8957
"	0	0.9218	0.8956	0.0008	0.8972
"	0	0.9322	0.8955	0.0007	0.8969
"	0	1.0185	0.8999	0.0008	0.9015
rajll_hac_f9_10gad rods_refassy_14x2x16_cladid	0	0.9400	0.8742	0.0009	0.8759
rajll_hac_f9_10gad rods_refassy_14x2x16_channels	0	0.9601	0.8755	0.0009	0.8773
rajll_hac_f9_10gad rods_refassy_14x2x16_cladid	0	0.9750	0.8760	0.0009	0.8777
"	0	0.9830	0.8768	0.0009	0.8786
"	0	1.0998	0.8789	0.0008	0.8804
rajll_hac_g9_10gad rods_refassy_14X2X16_cladid	0	0.9560	0.8641	0.0008	0.8657
"	0	0.9600	0.8643	0.0008	0.8659
"	0	0.9750	0.8660	0.0009	0.8678
rajll_hac_g9_10gad rods_refassy_14X2X16_channels	0	0.9830	0.8659	0.0008	0.8676
rajll_hac_g9_10gad rods_refassy_14X2X16_cladid	0	1.1100	0.8702	0.0008	0.8718
rajll_hac_g8_nointerspersedh2o_cladidsensitivity_14X2X16	0	1.0440	0.8894	0.001	0.8914
"	0	1.0719	0.8900	0.0009	0.8918
"	0	1.1000	0.8900	0.0009	0.8918
"	0	1.1500	0.8918	0.0008	0.8934
"	0	1.2192	0.8917	0.0008	0.8933



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**Table 6-30 Data for Figure 6-29 TN-B1 Array HAC Fuel Rod Clad OD Sensitivity Study**

Output File Name	Moderator Density (g/cm <sup>3</sup> )	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_a10_nointerspersedh2o_cladodsensitivity_14X2X16	0	0.9218	0.9051	0.0008	0.9067
"	0	1.0185	0.8858	0.0009	0.8876
"	0	1.0330	0.8810	0.0008	0.8826
"	0	1.1000	0.8647	0.0008	0.8663
"	0	1.1210	0.8604	0.0009	0.8622
rajll_hac_g10_nointerspersedh2o_cladodsensitivity_14X2X16	0	0.9322	0.9118	0.0008	0.9134
"	0	1.0185	0.8955	0.0007	0.8969
"	0	1.0330	0.8935	0.0008	0.8951
"	0	1.1000	0.8790	0.0008	0.8806
"	0	1.1210	0.8742	0.0009	0.8760
rajll_hac_f9_10gadrod_refassy_14x2x16_cladod	0	0.9601	0.8967	0.0008	0.8984
"	0	1.0330	0.8876	0.0008	0.8892
"	0	1.0998	0.8792	0.0008	0.8808
rajll_hac_f9_10gadrod_refassy_14x2x16_channels	0	1.1200	0.8755	0.0009	0.8773
rajll_hac_g9_10gadrod_refassy_14X2X16_cladod	0	0.9830	0.8857	0.0008	0.8873
"	0	1.0330	0.8791	0.0009	0.8809
rajll_hac_g9_10gadrod_refassy_14X2X16_channels	0	1.1100	0.8659	0.0008	0.8676
rajll_hac_g9_10gadrod_refassy_14X2X16_cladod	0	1.1200	0.8644	0.0010	0.8664
rajll_hac_g8_nointerspersedh2o_cladodsensitivity_14X2X16	0	1.0719	0.9120	0.0008	0.9136
"	0	1.1500	0.9030	0.0008	0.9046
"	0	1.2192	0.8900	0.0009	0.8918
"	0	1.2500	0.8832	0.0008	0.8848

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**Table 6-31 Data For Figure 6-37 Moderator Density Sensitivity Study for the TN-B1 HAC Worst Case Parameter Fuel Design**

<b>Output File Name</b>	<b>Moderator Density (g/cm<sup>3</sup>)</b>	<b>Clad Inner Diameter (cm)</b>	<b>Clad Outer Diameter (cm)</b>	<b>k<sub>eff</sub></b>	<b>σ</b>	<b>k<sub>eff</sub> + 2σ</b>
rajll_hac_g10_worst case_moderatordensity_14 X2X16	0.00	0.9338	1.010	0.7154	0.0006	0.7166
“	0.02	0.9338	1.010	0.7349	0.0007	0.7363
“	0.04	0.9338	1.010	0.7526	0.0007	0.7540
“	0.06	0.9338	1.010	0.7686	0.0006	0.7698
“	0.08	0.9338	1.010	0.7820	0.0007	0.7834
“	0.10	0.9338	1.010	0.7933	0.0008	0.7949
“	0.20	0.9338	1.010	0.8383	0.0007	0.8397
“	0.40	0.9338	1.010	0.8908	0.0007	0.8922
“	0.60	0.9338	1.010	0.9182	0.0009	0.9200
“	0.80	0.9338	1.010	0.9319	0.0008	0.9335
“	1.00	0.9338	1.010	0.9404	0.0007	0.9418

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**Table 6-32 Data for Figure 6-39 TN-B1 Single Package Normal Conditions of Transport Results**

Output File Name	Fuel Assembly Type	Moderator Density (g/cm <sup>3</sup> )	Gadolinia Rod (#)	Pitch (cm)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_normal_g10_5 .0wtpct235u_h2ode nsitysensitivity_12g ad rods_singlepack age	GNF 10 x 10	0.00	12	1.35	0.895	0.9338	1.010	0.2833	0.0005	0.2843
“	GNF 10 x 10	0.02	12	1.35	0.895	0.9338	1.010	0.2899	0.0005	0.2909
“	GNF 10 x 10	0.04	12	1.35	0.895	0.9338	1.010	0.2966	0.0006	0.2978
“	GNF 10 x 10	0.06	12	1.35	0.895	0.9338	1.010	0.3071	0.0006	0.3083
“	GNF 10 x 10	0.08	12	1.35	0.895	0.9338	1.010	0.3178	0.0006	0.3190
“	GNF 10 x 10	0.10	12	1.35	0.895	0.9338	1.010	0.3297	0.0005	0.3307
“	GNF 10 x 10	0.20	12	1.35	0.895	0.9338	1.010	0.3899	0.0006	0.3911
“	GNF 10 x 10	0.40	12	1.35	0.895	0.9338	1.010	0.4848	0.0008	0.4864
“	GNF 10 x 10	0.60	12	1.35	0.895	0.9338	1.010	0.5597	0.0008	0.5613
“	GNF 10 x 10	0.80	12	1.35	0.895	0.9338	1.010	0.6180	0.0007	0.6194
“	GNF 10 x 10	1.00	12	1.35	0.895	0.9338	1.010	0.6673	0.0008	0.6689

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**Table 6-33 Data for Figure 6-40 TN-B1 Single Package HAC Results**

Output File Name	Fuel Assembly Type	Inner Container Moderator Density (g/cm <sup>3</sup> )	Gadolinia Fuel Rods (#)	Pitch (cm)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_g10__worstcase_moderator density_singlepackage	GNF 10 x 10	0.00	12	1.35	0.895	0.9338	1.010	0.2794	0.0005	0.2804
“	GNF 10 x 10	0.02	12	1.35	0.895	0.9338	1.010	0.2850	0.0005	0.2860
“	GNF 10 x 10	0.04	12	1.35	0.895	0.9338	1.010	0.2902	0.0005	0.2912
“	GNF 10 x 10	0.06	12	1.35	0.895	0.9338	1.010	0.2967	0.0006	0.2979
“	GNF 10 x 10	0.08	12	1.35	0.895	0.9338	1.010	0.3041	0.0006	0.3053
“	GNF 10 x 10	0.10	12	1.35	0.895	0.9338	1.010	0.3111	0.0005	0.3121
“	GNF 10 x 10	0.20	12	1.35	0.895	0.9338	1.010	0.3546	0.0006	0.3558
“	GNF 10 x 10	0.40	12	1.35	0.895	0.9338	1.010	0.4526	0.0007	0.4540
“	GNF 10 x 10	0.60	12	1.35	0.895	0.9338	1.010	0.5468	0.0008	0.5484
“	GNF 10 x 10	0.80	12	1.35	0.895	0.9338	1.010	0.6274	0.0008	0.6290
rajll_hac_g10_100pct20density_worstcase_singlepackage	GNF 10 x 10	1.00	12	1.35	0.895	0.9338	1.010	0.6931	0.0010	0.6951

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**Table 6-34 Data for Figure 6-41 TN-B1 Package Array Under Normal Conditions of Transport Results**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Part Length Fuel Rods (#)	Pitch (cm)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_normal_g10_5.0wtpct235u_h2odensitysensitivity_12gad rods_21X3X24	GNF 10 x 10	0.00	12	1.35	0.895	0.9338	1.010	0.8519	0.0008	0.8535
“	GNF 10 x 10	0.02	12	1.35	0.895	0.9338	1.010	0.7962	0.0007	0.7976
“	GNF 10 x 10	0.04	12	1.35	0.895	0.9338	1.010	0.7441	0.0007	0.7455
“	GNF 10 x 10	0.06	12	1.35	0.895	0.9338	1.010	0.7054	0.0008	0.7070
“	GNF 10 x 10	0.08	12	1.35	0.895	0.9338	1.010	0.6726	0.0008	0.6742
“	GNF 10 x 10	0.10	12	1.35	0.895	0.9338	1.010	0.6427	0.0008	0.6443
“	GNF 10 x 10	0.20	12	1.35	0.895	0.9338	1.010	0.5500	0.0008	0.5516
“	GNF 10 x 10	0.40	12	1.35	0.895	0.9338	1.010	0.5254	0.0007	0.5268
“	GNF 10 x 10	0.60	12	1.35	0.895	0.9338	1.010	0.5690	0.0007	0.5704
“	GNF 10 x 10	0.80	12	1.35	0.895	0.9338	1.010	0.6206	0.0007	0.6220
“	GNF 10 x 10	1.00	12	1.35	0.895	0.9338	1.010	0.6683	0.0008	0.6699

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**Table 6-35 Data for Figure 6-42 TN-B1 Package Array Hypothetical Accident Condition Results**

Output File Name	Fuel Assembly Type	Inner Container Moderator Density (g/cm <sup>3</sup> )	Gadolinia-urania Fuel Rods (#)	Pitch (cm)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_g10_12partlengththords_worstcase_moderatordensity_10X1X10	GNF 10 x 10	0.00	12	1.35	0.895	0.9338	1.010	0.6375	0.0007	0.6389
“	GNF 10 x 10	0.02	12	1.35	0.895	0.9338	1.010	0.6470	0.0007	0.6484
“	GNF 10 x 10	0.04	12	1.35	0.895	0.9338	1.010	0.6567	0.0007	0.6581
“	GNF 10 x 10	0.06	12	1.35	0.895	0.9338	1.010	0.6648	0.0007	0.6662
“	GNF 10 x 10	0.08	12	1.35	0.895	0.9338	1.010	0.6734	0.0007	0.6748
“	GNF 10 x 10	0.10	12	1.35	0.895	0.9338	1.010	0.6822	0.0007	0.6836
“	GNF 10 x 10	0.20	12	1.35	0.895	0.9338	1.010	0.7226	0.0007	0.7240
“	GNF 10 x 10	0.40	12	1.35	0.895	0.9338	1.010	0.7976	0.0007	0.7990
“	GNF 10 x 10	0.60	12	1.35	0.895	0.9338	1.010	0.8561	0.0009	0.8579
“	GNF 10 x 10	0.80	12	1.35	0.895	0.9338	1.010	0.9005	0.0008	0.9021
“	GNF 10 x 10	1.00	12	1.35	0.895	0.9338	1.010	0.9378	0.0009	0.9396

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**Table 6-36 Data for Figure 6-45 TN-B1 Fuel Rod Transport in Stainless Steel Pipe**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pitch (cm)	Fuel Rod (#)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_8_worstcase_ssp_ipe_14x2x16	8x8	1.000	1.1305	110	1.05	1.1000	1.1000	0.8793	0.0007	0.8807
"	8x8	1.000	1.6662	52	1.05	1.1000	1.1000	1.0235	0.0009	1.0253
"	8x8	1.000	1.9035	43	1.05	1.1000	1.1000	1.0440	0.0008	1.0456
rajll_hac_8_worstcase_ssp_ipe_22fuelrod s_14x2x16	8x8	1.000	2.5	22	1.05	1.1000	1.1000	0.8823	0.0008	0.8839
rajll_hac_8_worstcase_ssp_ipe_14x2x16	8x8	1.000	2.937	14	1.05	1.1000	1.1000	0.7294	0.0008	0.7310
rajll_hac_9_worstcase_ssp_ipe_14x2x16	9x9	1.000	1.0505	140	0.9600	1.0200	1.0200	0.8701	0.0006	0.8713
"	9x9	1.000	1.4770	72	0.9600	1.0200	1.0200	1.0515	0.0008	1.0531
"	9x9	1.000	2	38	0.9600	1.0200	1.0200	1.0056	0.0009	1.0074
rajll_hac_9_worstcase_ssp_ipe_26fuelrod s_14x2x16	9x9	1.000	2.25	26	0.9600	1.0200	1.0200	0.8900	0.0008	0.8916
rajll_hac_9_worstcase_ssp_ipe_14x2x16	9x9	1.000	2.5432	22	0.9600	1.0200	1.0200	0.8416	0.0010	0.8436



**Table 6-36 Data for Figure 6-45 TN-B1 Fuel Rod Transport in Stainless Steel Pipe(continued)**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pitch (cm)	Fuel Rod (#)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_10_worstcase_ssp_ipe_14x2x16	10x10	1.000	1.0305	144	0.9	1.000	1.000	0.8666	0.0007	0.8680
"	10x10	1.000	1.3213	84	0.9	1.000	1.000	1.0070	0.0008	1.0086
"	10x10	1.000	1.6416	56	0.9	1.000	1.000	1.0310	0.0011	1.0332
"	10x10	1.000	2.0484	30	0.9	1.000	1.000	0.8863	0.0008	0.8879

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**Table 6-37 Data for Figure 6-46 TN-B1 Fuel Rod Single Package Under Normal Conditions of Transport**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pitch (cm)	Fuel Rod Number (#)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_normal_8_worstcasefuel_fuelrodtransport_moderator_density_sensitivity_single_package	8x8	0.00	2.815	25	1.05	1.1000	1.1000	0.1675	0.0004	0.1683
“	8x8	0.01	2.815	25	1.05	1.1000	1.1000	0.1675	0.0004	0.1683
“	8x8	0.02	2.815	25	1.05	1.1000	1.1000	0.1672	0.0004	0.1680
“	8x8	0.04	2.815	25	1.05	1.1000	1.1000	0.1702	0.0004	0.1710
“	8x8	0.06	2.815	25	1.05	1.1000	1.1000	0.1757	0.0005	0.1767
“	8x8	0.08	2.815	25	1.05	1.1000	1.1000	0.1845	0.0005	0.1855
“	8x8	0.10	2.815	25	1.05	1.1000	1.1000	0.1949	0.0004	0.1957
“	8x8	0.20	2.815	25	1.05	1.1000	1.1000	0.2567	0.0005	0.2577
“	8x8	0.40	2.815	25	1.05	1.1000	1.1000	0.3890	0.0007	0.3904
“	8x8	0.60	2.815	25	1.05	1.1000	1.1000	0.4967	0.0007	0.4981
“	8x8	0.80	2.815	25	1.05	1.1000	1.1000	0.5783	0.0009	0.5801
“	8x8	1.00	2.815	25	1.05	1.1000	1.1000	0.6365	0.0008	0.6381

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**Table 6-38 Data for Figure 6-47 TN-B1 Fuel Rod Transport Single Package HAC**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pitch (cm)	Fuel Rod Number (#)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_8_worstcase_fuelrod_transport_mode_ratordensitysensitivity_singlepackage	8x8	0.00	3.0056	25	1.05	1.1000	1.1000	0.1769	0.0004	0.1777
“	8x8	0.01	3.0056	25	1.05	1.1000	1.1000	0.1761	0.0004	0.1769
“	8x8	0.02	3.0056	25	1.05	1.1000	1.1000	0.1767	0.0004	0.1775
“	8x8	0.04	3.0056	25	1.05	1.1000	1.1000	0.1778	0.0005	0.1788
“	8x8	0.06	3.0056	25	1.05	1.1000	1.1000	0.1794	0.0004	0.1802
“	8x8	0.08	3.0056	25	1.05	1.1000	1.1000	0.1829	0.0004	0.1837
“	8x8	0.10	3.0056	25	1.05	1.1000	1.1000	0.1876	0.0004	0.1884
“	8x8	0.20	3.0056	25	1.05	1.1000	1.1000	0.2306	0.0005	0.2316
“	8x8	0.40	3.0056	25	1.05	1.1000	1.1000	0.3718	0.0007	0.3732
“	8x8	0.60	3.0056	25	1.05	1.1000	1.1000	0.5062	0.0007	0.5076
“	8x8	0.80	3.0056	25	1.05	1.1000	1.1000	0.5980	0.0008	0.5996
“	8x8	1.00	3.0056	25	1.05	1.1000	1.1000	0.6532	0.0008	0.6548

**Table 6-39 Data for Figure 6-48 TN-B1 Package Array Under Normal Conditions of Transport with Loose Fuel Rods**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pitch (cm)	Fuel Rod Number (#)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_normal_8_worstcasefuel_fuelrodtransport_moderator_density_sensitivity_21X3X24	8x8	0.00	2.815	25	1.05	1.1000	1.1000	0.5055	0.0006	0.5067
"	8x8	0.01	2.815	25	1.05	1.1000	1.1000	0.5827	0.0006	0.5839
"	8x8	0.02	2.815	25	1.05	1.1000	1.1000	0.5931	0.0007	0.5945
"	8x8	0.04	2.815	25	1.05	1.1000	1.1000	0.5891	0.0007	0.5905
"	8x8	0.06	2.815	25	1.05	1.1000	1.1000	0.5719	0.0007	0.5733
"	8x8	0.08	2.815	25	1.05	1.1000	1.1000	0.5523	0.0009	0.5541
"	8x8	0.10	2.815	25	1.05	1.1000	1.1000	0.5291	0.0007	0.5305
"	8x8	0.20	2.815	25	1.05	1.1000	1.1000	0.4383	0.0006	0.4395
"	8x8	0.40	2.815	25	1.05	1.1000	1.1000	0.4300	0.0007	0.4314
"	8x8	0.60	2.815	25	1.05	1.1000	1.1000	0.5079	0.0008	0.5095
"	8x8	0.80	2.815	25	1.05	1.1000	1.1000	0.5817	0.0008	0.5833
"	8x8	1.00	2.815	25	1.05	1.1000	1.1000	0.6365	0.0008	0.6381

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**Table 6-40 Data for Figure 6-49 TN-B1 Fuel Rod Transport Under HAC**

Output File Name	Fuel Assembly Type	Interspersed Moderator Density (g/cm <sup>3</sup> )	Pitch (cm)	Fuel Rod Number (#)	Pellet OD (cm)	Clad Inner Diameter (cm)	Clad Outer Diameter (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
rajll_hac_8_worstcase_fuelrodtransport_100pct_h2o_density_10x1x10	8x8	0.00	3.0056	25	1.05	1.1000	1.1000	0.3230	0.0005	0.3240
"	8x8	0.01	3.0056	25	1.05	1.1000	1.1000	0.3479	0.0005	0.3489
"	8x8	0.02	3.0056	25	1.05	1.1000	1.1000	0.3752	0.0007	0.3766
"	8x8	0.04	3.0056	25	1.05	1.1000	1.1000	0.4007	0.0006	0.4019
"	8x8	0.06	3.0056	25	1.05	1.1000	1.1000	0.4287	0.0006	0.4299
"	8x8	0.08	3.0056	25	1.05	1.1000	1.1000	0.4556	0.0006	0.4568
"	8x8	0.10	3.0056	25	1.05	1.1000	1.1000	0.5743	0.0009	0.5761
"	8x8	0.20	3.0056	25	1.05	1.1000	1.1000	0.7416	0.0009	0.7434
"	8x8	0.40	3.0056	25	1.05	1.1000	1.1000	0.8264	0.0008	0.8280
"	8x8	0.60	3.0056	25	1.05	1.1000	1.1000	0.8660	0.0008	0.8676
"	8x8	0.80	3.0056	25	1.05	1.1000	1.1000	0.8731	0.0007	0.8745
"	8x8	1.00	3.0056	25	1.05	1.1000	1.1000	0.3752	0.0007	0.3766

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### 6.11.10. *Summary of Experiments*

This document provides a summary of the experiments used in Reference 3 to determine the SCALE 4.4a bias. Trending data is either from the original experiments or calculated herein, i.e., H/U values, have been added to the data. Note that in most cases the experimental  $k_{\text{eff}} \pm \sigma$  from Reference 3 do not have a reference. If data from the original experiment and/or data from the International Handbook of Evaluated Criticality Safety Benchmark Experiments (see Reference 4) provided these values, it was so noted or additional values provided.

The USL method of NUREG/CR-6361 (Reference 7) has the tacit assumption that the experimental  $k$  is 1.0000. Likewise, it does not account for the uncertainty in the experimental values. It is recommended that the procedure discussed in NUREG/CR-6698, "Guide for Validation of Nuclear Criticality Safety Calculational Methodology," be considered. The document has the following definitions for the calculated' values used for the bias evaluation:

$$k_{\text{norm}} = k_{\text{calc}}/k_{\text{exp}} \text{ and}$$

$$\sigma_{\text{norm}} = [(\sigma_{\text{calc}})^2 + (\sigma_{\text{exp}})^2]^{1/2}$$

This will normalize the calculated to experimental to account for uncertainties in the experimental values.

**Note:** The reference numbers quoted in the following sections are references listed in each section, rather than those listed in Section 6.11.

#### 6.11.10.1. **Critical Configurations**

##### 6.11.10.1.1. ***Water-Moderated U(4.31)O<sub>2</sub> Fuel Rods in 2.54-cm Square-Pitched Arrays***

##### References:

1. "Critical Separation Between Subcritical Clusters of 4.29 Wt% U-235 Enriched UO<sub>2</sub> Rods in Water With Fixed Neutron Poisons," S.R. Bierman, B. M. Durst, E.D. Clayton, Battelle Pacific Northwest Laboratories, NUREG/CR-0073(PNL-2695).
2. "Water-Moderated U(4.31)O<sub>2</sub> Fuel Rods in 2.54-cm Square-Pitched Arrays," V.F. Dean, Evaluator, International Handbook of Evaluated Criticality Safety Benchmark Experiments," NEA/NSC/DOC(95)03, Sept 2001, Nuclear Energy Agency.
3. "Software Validation Document, EMF-2670, PC-SCALE 4.4a V&V", C.D. Manning, EMF-2670, Rev. 1, 11/26/2002, Framatome ANP.

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Reference 3 uses the data from this set of experiments as part of a heterogeneous uranium oxide set of benchmark calculations. Table 6 of that reference provides some information on the experimental configuration and Tables 7 and 9 provide results for the 238 and 44 group Scale 4.4a cross-sections, respectively. Table 6-41 Summary of Information for Experiment below provides a summary of the benchmark information from References 1 and 2. The rod and oxide dimensional and material information came from Reference 1. The enrichment quoted in Reference 1 was changed in Reference 2 due to a later chemical analysis of the fuel rods used in the experiment. Thus, the table uses the 4.31 value from Reference 2 rather than 4.29 quoted in Reference 1. The temperatures of the experiments were not included in Reference 1 and were not explicitly noted at the time of the experiment. The authors of Reference 2 obtained logbooks from similar experiments at PNL that showed temperatures ranging from ~18°C to ~25°C. From these data Reference 2 inferred an average value of ~22°C which is listed here. The value used in the calculations of Reference 3 is not currently known. The temperature value is used to calculate the hydrogen atom density and a deviation of a few degrees will not significantly change the results. The U and H atom densities used a value of Avogadro's number of 0.6022142E-24. The H/U value applies only to the fuel cluster. Table 6-44 Urania Gadolinia Experiment Summary<sup>a</sup> contains cases using cell-weighted models, 'x' added to case ID. These are included for completeness and should not be included in the normal benchmarking trending.

**Table 6-41 Summary of Information for Experiment**

Pellet OD, cm	1.2649	Enrichment, wt%	4.31 <sup>a</sup>	$V_{H_2O}/V_{oxide}$	3.883228
Rod ID, cm	1.2827	Oxide Density, g/cm <sup>3</sup>	94.9	U-235 Atom Density	1.0125E-03
Rod OD, cm	1.4147	Temperature, °C	22 <sup>b</sup>	H Atom Density	0.066724
Rod Pitch, cm	2.54	Water Density, g/cm <sup>3</sup>	0.9978	H/U	255.92
Clad Material	Aluminum	Boron, ppm	0.0		

- a) Redefined from 4.29 in Reference 2 due to fuel evaluation after publication of Reference 1.
- b) Not defined in Reference 1, assumed in Reference 2 based upon inference from data notebooks of experiments.



**Table 6-42 Parameters for Benchmark Cases for SCALE 4.4a 44 Group Cross-Section Set**

Case ID <sup>c</sup>	Lattice <sup>a</sup>	Spacing <sup>a</sup> between clusters, cm		Experimental $k_{eff}$ and $\sigma$				SCALE 4.4a 44 Group Cross- Section Calculated $k_{eff}$ and $\sigma$				Absorber Plates in Water Gap
		Rod- rod	Cell- cell	$k_{eff}^b$	$\sigma^b$	$k_{eff}^c$	$\sigma^c$	$k_{eff}^d$	$\sigma^d$	AFG <sup>d</sup>	EALF <sup>d</sup> (ev)	
c004.out	15x8	11.72	10.62	1.0000	0.0020	0.9997	0.0020	0.9971	0.0008	35.772	0.112667	None
c005b.out	15x8	10.77	9.64	1.0000	0.0180	0.9997	0.0020	0.9960	0.0008	35.763	0.112942	0.625 cm Al plates
c006b.out	15x8	10.72	9.59	1.0000	0.0019	0.9997	0.0020	0.9960	0.0008	35.768	0.112841	0.625 cm Al plates
c007a.out	15x8	9.76	8.63	1.0000	0.0021	0.9997	0.0020	0.9966	0.0008	35.768	0.112705	0.302 cm SS 304L plates
c008b.out	15x8	9.22	8.09	1.0000	0.0021	0.9997	0.0020	0.9948	0.0008	35.755	0.113485	0.302 cm SS 304L plates
c009b.out	15x8	8.08	6.95	1.0000	0.0021	0.9997	0.0020	0.9963	0.0008	35.748	0.113698	0.298 cm 304L plates with 1.05 wt% B
c010b.out	15x8	6.60	5.47	1.0000	0.0021	0.9997	0.0020	0.9980	0.0008	35.728	0.114519	0.298 cm 304L plates with 1.05 wt% B
c011b.out	15x8	7.90	6.77	1.0000	0.0021	0.9997	0.0020	0.9983	0.0009	35.750	0.113450	0.298 cm 304L plates with 1.62 wt% B
c012b.out	15x8	5.76	4.63	1.0000	0.0021	0.9997	0.0020	0.9975	0.0007	35.729	0.114508	0.298 cm 304L plates with 1.62 wt% B
c013b.out	15x8	9.65	8.52	1.0000	0.0021	0.9997	0.0020	0.9956	0.001	35.768	0.112832	0.485 cm, SS 304L plates
c014b.out	15x8	8.58	7.45	1.0000	0.0021	0.9997	0.0020	0.9970	0.0009	35.745	0.113819	0.485 cm, SS 304L plates
c029b.out	15x8	10.90	9.77	1.0000	0.0021	0.9997	0.0020	0.9967	0.0008	35.770	0.112874	0.652 cm, Zircaloy-4 plates
c030b.out	15x8	10.86	9.73	1.0000	0.0021	0.9997	0.0020	0.9977	0.0009	35.767	0.112860	0.652 cm, Zircaloy-4 plates
c031b.out	15x8	7.672	6.55	1.0000	0.0021	0.9997	0.0020	0.9975	0.0008	35.727	0.114536	0.723 cm, Boral plates, 28.7 wt% B

- a) From Reference 1. The 'rod surface-to-rod' surface spacing is reported in Reference 1. Reference 2 (p. 9) provides the cell-to-cell spacing for selected experiments from Reference 1 as: (rod-rod) – (pitch) + (rod diameter). This formula was applied to all above values even though some 'rod-rod' may be 'array plate-to-plate'.
- b) Values from Reference 3, Table 6, p. 42. Source of  $\sigma$  values is not listed in this reference.
- c) Values from Reference 2, p. 23 based upon calculational uncertainties in parameters and assumptions in the benchmark models of the reference. Note that Reference 2 only includes 4 of the cases from Reference 1 listed above. Here it is assumed that the values listed above apply to all cases.
- d) From Reference 3, Table 9, p. 61 for 44 group cross-sections. Table 7 in this reference has values for 238 group cross-sections

**Table 6-43 Parameters for Benchmark Cases for SCALE 4.4a 238 Group Cross-Section Set**

Case ID <sup>c</sup>	Lattice <sup>a</sup>	Cluster Spacing <sup>a</sup> , cm		Experimental $k_{eff}$ and $\sigma$				SCALE 4.4a 238 Group Cross-Section Calculated $k_{eff}$ and $\sigma$				Absorber Plates in Water Gap
		Rod-rod	Cell-cell	$k_{eff}^b$	$\sigma^b$	$k_{eff}^c$	$\sigma^c$	$k_{eff}^d$	$\sigma^d$	AFG <sup>d</sup>	EALF <sup>d</sup> (ev)	
c001x.out <sup>e</sup>	10x11.51	0.0	0.0	1.0000	0.0021	0.9997	0.0020	0.9987	0.0008	208.112	0.108721	--
c002x.out	8x16.37	0.0	0.0	1.0000	0.0021	0.9997	0.0020	0.9993	0.0008	208.157	0.108277	--
c003x.out	9x13.35	0.0	0.0	1.0000	0.0021	0.9997	0.0020	1.0015	0.0010	208.136	0.108496	--
c004.out	15x8	11.72	10.62	1.0000	0.0020	0.9997	0.0020	0.9930	0.0010	207.568	0.114058	None
c005b.out	15x8	10.77	9.64	1.0000	0.0180	0.9997	0.0020	0.9931	0.0008	207.550	0.114504	0.625 cm Al plates
c006b.out	15x8	10.72	9.59	1.0000	0.0019	0.9997	0.0020	0.9941	0.0009	207.508	0.114748	0.625 cm Al plates
c007a.out	15x8	9.76	8.63	1.0000	0.0021	0.9997	0.0020	0.9944	0.0008	207.547	0.114468	0.302 cm SS 304L plates
c007x.out	15x8	9.76	8.63	1.0000	0.0021	0.9997	0.0020	1.0010	0.0008	208.273	0.107285	0.302 cm SS 304L plates
c008b.out	15x8	9.22	8.09	1.0000	0.0021	0.9997	0.0020	0.9931	0.0007	207.487	0.114939	0.302 cm SS 304L plates
c008x.out	15x8	9.22	8.09	1.0000	0.0021	0.9997	0.0020	0.9981	0.0008	208.220	0.107758	0.302 cm SS 304L plates
c009b.out	15x8	8.08	6.95	1.0000	0.0021	0.9997	0.0020	0.9928	0.0008	207.472	0.114907	0.298 cm 304L plates with 1.05 wt% B
c010b.out	15x8	6.60	5.47	1.0000	0.0021	0.9997	0.0020	0.9952	0.0009	207.373	0.115896	0.298 cm 304L plates with 1.05 wt% B
c011b.out	15x8	7.90	6.77	1.0000	0.0021	0.9997	0.0020	0.9964	0.0008	207.507	0.114703	0.298 cm 304L plates with 1.62 wt% B
c012b.out	15x8	5.76	4.63	1.0000	0.0021	0.9997	0.0020	0.9938	0.0009	207.364	0.116224	0.298 cm 304L plates with 1.62 wt% B
c013b.out	15x8	9.65	8.52	1.0000	0.0021	0.9997	0.0020	0.9953	0.0008	207.495	0.114944	0.485 cm, SS 304L plates
c013x.out	15x8	9.65	8.52	1.0000	0.0021	0.9997	0.0020	1.0002	0.0009	208.270	0.107272	0.485 cm, SS 304L plates
c014b.out	15x8	8.58	7.45	1.0000	0.0021	0.9997	0.0020	0.9942	0.0009	207.484	0.115038	0.485 cm, SS 304L plates
c014x.out	15x8	8.580	7.45	1.0000	0.0021	0.9997	0.0020	1.0018	0.0008	208.211	0.107849	0.485 cm, SS 304L plates
c029b.out	15x8	10.90	9.77	1.0000	0.0021	0.9997	0.0020	0.9942	0.0008	207.549	0.114428	0.652 cm, Zircaloy-4 plates
c030b.out	15x8	10.86	9.73	1.0000	0.0021	0.9997	0.0020	0.9946	0.0008	207.508	0.114783	0.652 cm, Zircaloy-4 plates
c031b.out	15x8	7.672	6.55	1.0000	0.0021	0.9997	0.0020	0.9951	0.0008	207.387	0.115885	0.723 cm, Boral plates, 28.7 wt% B

- a) From Reference 1. The 'rod surface-to-rod' surface spacing is reported in Reference 1. Reference 2 (p. 9) provides the cell-to-cell spacing for selected experiments from Reference 1 as: (rod-rod) – (pitch) + (rod diameter). This formula was applied to all above values even though some 'rod-rod' may be 'array plate-to-plate'.
- b) Values from Reference 3, Table 6, p. 42. Source of  $\sigma$  values is not listed in this reference.
- c) Values from Reference 2, p. 23 based upon calculational uncertainties in parameters and assumptions in the benchmark models of the reference. Note that Reference 2 only includes 4 of the cases from Reference 1 listed above. Here it is assumed that the values listed above apply to all cases.
- d) From Reference 3, Table 9, p. 61 for 44 group cross-sections. Table 7 in this reference has values for 238 group cross-sections. e) From Reference 3, Table 6. The 'x' before '.out' means the case is a cell weighted model.
- e) From Reference 3, Table 6. The 'x' before '.out' means the case is a cell weighted model.


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#### 6.11.10.1.2. *Urania Gadolinia Experiments*

##### References:

4. FANP Doc: 32-5012895-00, "Validation Report – SCALEPC-44A Urania-Gadolinia Experiments," R.S. Harding.
5. "Urania Gadolinia: Nuclear Model Development and Critical Experiment Benchmark," L.W. Newman, Babcock & Wilcox for DOE, DOE/ET/34212-41, BAW-1910, April 1984.
6. "Development and Demonstration of An Advanced Extended-Burnup Fuel Assembly Design Incorporating Urania-Gadolinia," L.W. Newman, Babcock & Wilcox for DOE, DOE/ET/34212-41, BAW-1681-2, August 1982.

Reference 4 uses the experimental data from References 5 and 6 to construct benchmark cases for SCALE 4.4a. Table 6-44 Urania Gadolinia Experiment Summary<sup>a</sup> summarizes the experimental configuration data that form the basis for the KENO V.a models. Table 6-46 Urania Gadolinia Critical Experiment Trending Data provides trending parameters for this set of experiments. Table 6-45 Experimental Parameters for Calculating U-235 and H Atom Densities lists the basis for the H/U values tabulated in Table 6-46 Urania Gadolinia Critical Experiment Trending Data. Table 6-47 Urania Gadolinia Benchmark  $k_{\text{eff}}$  Data provides the experimental and calculated results for the 44 and 238 group SCALE 4.4a cross-section sets from Reference 3.

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**Table 6-44 Urania Gadolinia Experiment Summary<sup>a</sup>**

Parameter	Rod 1	Rod 2	Rod 3
U-235 wt%	4.02	2.459	1.944
Gadolinia Wt%	-	-	4
Pellet density <sup>b</sup> , g/cm <sup>3</sup>	9.46	10.218	10.328
Pellet OD, cm	1.1265	1.03	1.0296
Rod ID, cm	1.1265	1.044	1.0439
Rod OD, cm	1.2078	1.206	1.2065
Rod Pitch, cm	1.6358	1.6358	1.6358
Clad Material	SS	Al	Al
V <sub>fuel/cell</sub>	0.996654	0.833229	0.832582
V <sub>H2O/cell</sub>	1.530044	1.533399	1.532452
Water boron factor <sup>c</sup>	0.99928		
Temperature <sup>d</sup> , °C	22		
Water density, g/cm <sup>3</sup>	0.99777		

- a) From Reference 4.
- b) Based upon rod mass and fuel volume in rod.
- c) A factor to correct water density from 25 °C to 20 °C. Boron ppm is based upon 25 °C measurements. See Reference 4, p. 9.
- d) Not specified explicitly for this set of experiments. This value is inferred from temperature data in Reference 7.

**Table 6-45 Experimental Parameters for Calculating U-235 and H Atom Densities**

Case ID	Number of Different Type Rods in each Critical Configuration(Reference 1 Table						Core Volume <sup>a</sup>		Atom Density <sup>a</sup>	
	2.46 Wt%	4.02 Wt%	1.94 Wt% (Gd)	Water	Misc	Core Total	Fuel	Water	U-235	H
core01.out	4808	-	-	153	-	4961	4006.16	7765.83	5.67711E-04	0.066676
core03.out	4788	-	-	137	16	4941	3989.50	7692.42	5.67711E-04	0.066676
core05.out	4780	-	28	153	-	4961	4006.15	7765.90	5.67061E-04	0.066676
core5a.out	4776	-	32	153	-	4961	4006.14	7765.91	5.66968E-04	0.066676
core5b.out	4780	-	28	153	-	4961	4006.15	7765.90	5.67061E-04	0.066676
core08.out	4772	-	36	153	-	4961	4006.14	7765.92	5.66875E-04	0.066676
core10.out	4772	-	36	137	16	4961	4006.14	7723.11	5.66875E-04	0.066676
core12a.out	3920	888	-	153	-	4961	4151.29	7768.81	6.21492E-04	0.066676
core14.out	3920	860	28	153	-	4961	4146.69	7768.79	6.19146E-04	0.066676
core16.out	3920	852	36	153	-	4961	4145.38	7768.78	6.18475E-04	0.066676
core18.out	3676	944	-	180	-	4800	4003.79	7553.60	6.27210E-04	0.066676
core19.out	3676	928	16	180	-	4800	4001.17	7553.58	6.25815E-04	0.066676
core20.out	3676	912	32	180	-	4800	3998.54	7553.57	6.24420E-04	0.066676

a) Calculated values. Atom densities based upon Avogadro's number of 0.6022142E-24

**Table 6-46 Urania Gadolinia Critical Experiment Trending Data**

Case Name	Clad <sup>a</sup>	Lattice <sup>a</sup>	wt% 235 <sup>a</sup>	Boron, ppm <sup>a</sup>	Vh2o/Vfuel <sup>b</sup>	H/U <sup>b</sup>	k <sub>eff</sub> <sup>c</sup>	Sigma <sup>c</sup>	Rod Configurations <sup>a</sup>
core01.out	Al	15x15	2.46	1337.9	1.9385	227.67	1.0002	0.0005	0
core03.out	Al	15x15	2.46/1.94	1239.3	1.9282	226.46	1.0000	0.0006	20-4%Gd
core05.out	Al	15x15	2.46/1.94	1208.0	1.9385	227.93	0.9999	0.0006	28-4%Gd
core5a.out	Al	15x15	2.46/1.94	1191.3	1.9385	227.97	0.9999	0.0006	32-4%Gd
core5b.out	Al	15x15	2.46/1.94	1207.1	1.9385	227.93	0.9999	0.0006	28-4%Gd
core08.out	Al	15x15	2.46/1.94	1170.7	1.9385	228.01	1.0083	0.0012	36-4%Gd
core10.out	Al	15x15	2.46/1.94	1177.1	1.9278	226.75	1.0001	0.0009	36-4%Gd+3 void rods
core12a.out	SS/Al	15x15	4.02/2.46	1899.3	1.8714	200.77	1.0000	0.0007	4.02 inner/2.456 outer
core14.out	SS/Al	15x15	4.02/2.46/1.94	1653.8	1.8735	201.76	1.0030	0.0009	28-4%Gd
core16.out	SS/Al	15x15	4.02/2.46/1.94	1579.4	1.8741	202.04	1.0001	0.0010	36-4%Gd
core18.out	SS/Al	16x16	4.02/2.46	1776.8	1.8866	200.56	1.0002	0.0011	CE Large Guide Tubes
core19.out	SS/Al	16x16	4.02/2.46/1.94	1628.3	1.8878	201.14	1.0002	0.0010	16-4%Gd
core20.out	SS/Al	16x16	4.02/2.46/1.94	1499.0	1.8891	201.72	1.0002	0.0010	Zone + 32-4%

- a) Reference 4.  
b) Calculated values from Table 5.  
c) Reference 3, Table 6. The source of these values is not documented in the reference.

**Table 6-47 Urania Gadolinia Benchmark  $k_{eff}$  Data**

Case ID	Experimental $k_{eff}$ and $\sigma$		SCALE 4.4a 44 Group Cross-Section Calculated $k_{eff}$ and $\sigma$				SCALE 4.4a 238 Group Cross-Section Calculated $k_{eff}$ and $\sigma$			
	$k_{eff}^a$	$\sigma^a$	$k_{eff}^b$	$\sigma^b$	AFG <sup>b</sup>	EALF <sup>b</sup> (ev)	$k_{eff}^b$	$\sigma^b$	AFG <sup>b</sup>	EALF <sup>b</sup> (ev)
core01.out	1.0002	0.0005	0.9955	0.0006	33.8930	0.2530	0.9952	0.0007	197.6190	0.2567
core03.out	1.0000	0.0006	0.9963	0.0006	33.9190	0.2499	0.9943	0.0006	197.6810	0.2547
core05.out	0.9999	0.0006	0.9968	0.0006	33.9280	0.2493	0.9935	0.0006	197.6840	0.2543
core5a.out	0.9999	0.0006	0.9963	0.0005	33.9270	0.2494	0.9940	0.0006	197.6850	0.2547
core5b.out	0.9999	0.0006	0.9959	0.0006	33.9160	0.2504	0.9941	0.0007	197.6280	0.2558
core08.out	1.0083	0.0012	0.9958	0.0006	33.9200	0.2503	0.9928	0.0005	197.7470	0.2534
core10.out	1.0001	0.0009	0.9956	0.0006	33.9130	0.2512	0.9922	0.0007	197.6080	0.2562
core12a.out	1.0000	0.0007	0.9982	0.0006	32.8910	0.3644	0.9950	0.0006	193.1960	0.3697
core14.out	1.0030	0.0009	0.9976	0.0007	33.0670	0.3421	0.9942	0.0007	193.8910	0.3488
core16.out	1.0001	0.0010	0.9969	0.0007	33.1010	0.3376	0.9941	0.0007	194.1570	0.3412
core18.out	1.0002	0.0011	0.9975	0.0007	32.8960	0.3645	0.9950	0.0007	193.2390	0.3684
core19.out	1.0002	0.0010	0.9973	0.0006	33.0140	0.3489	0.9941	0.0007	193.6610	0.3553
core20.out	1.0002	0.0010	0.9969	0.0007	33.1050	0.3382	0.9950	0.0006	194.0850	0.3425

a) Values from Reference 3, Table 6, p. 42. Source of  $\sigma$  values is not documented in this reference.

b) From Reference 3, Table 9, p. 61 for 44 group cross-sections. Table 7 in this reference has values for 238 group cross-sections



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### 6.11.10.1.3. ***Critical Experiments Supporting Close Proximity Water Storage of Power Reactor Fuel***

#### **References:**

7. FANP Doc. 32-5012896-00, "Validation Report – SCALEPC-44A Close Proximity Experiments," R.S. Harding.
8. "Critical Experiments Supporting Close Proximity Water Storage of Power Reactor Fuel," M.N. Baldwin, et.al., BAW-1484-7, July 1979.


Reference 7 uses the experimental data from Reference 8 to construct benchmark cases for SCALE 4.4a. Table 6-48 Close Proximity Experiment Summary<sup>a</sup> summarizes the experimental configuration data that form the basis for the KENO V.a models. Table 6-49 Close Proximity Experiment Trending Data provides trending parameters for this set of experiments. Table 6-50 Close Proximity Experiment  $k_{eff}$  Data provides the experimental and calculated results for the 44 and 238 group SCALE 4.4a cross-section sets from Reference 3.

**Table 6-48 Close Proximity Experiment Summary<sup>a</sup>**

U-235 wt%	2.459	Fuel Lattice	14x14
Pellet Density <sup>b</sup> , g/cm <sup>3</sup>	10.218	Clad Material	Al
Pellet OD, cm	1.030	V <sub>fuel</sub> /cell	0.8332
Rod ID, cm	1.044	V <sub>h2o</sub> /cell	1.5342
Rod OD, cm	1.206	V <sub>h2o</sub> /V <sub>f</sub>	1.8413
Rod Pitch, cm	1.636		

a) From Reference 7.

b) Based upon rod mass and fuel volume in rod

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**Table 6-49 Close Proximity Experiment Trending Data**

Case ID	Cluster Spacing <sup>a</sup> , cm	Temp <sup>a</sup> , °C	Boron <sup>a</sup> , ppm	Boron Factors <sup>b</sup>	Water density <sup>b</sup> , g/cm <sup>3</sup>	Atom Density <sup>c</sup>		H/U <sup>c</sup>	Absorbers <sup>a</sup>
						U-235	H		
ac1p1.out	--	21	0	0.999788	0.99799	5.6991E-04	0.066725	215.57	--
ac1p2.out	0.000	18.5	1037	1.000298	0.99850	5.6991E-04	0.066793	215.79	--
ac1p3.out	1.636	18	764	1.000392	0.99860	5.6991E-04	0.066806	215.83	H2O
ac1p4.out	1.636	17	0	1.000572	0.99878	5.6991E-04	0.066830	215.91	84 B4C pins/H2O
ac1p5.out	3.272	17.5	0	1.000483	0.99869	5.6991E-04	0.066818	215.87	64 B4C pins/H2O
ac1p6.out	3.272	17.5	0	1.000483	0.99869	5.6991E-04	0.066818	215.87	64 B4C pins/H2O
ac1p7.out	4.908	17.5	0	1.000483	0.99869	5.6991E-04	0.066818	215.87	34 B4C pins/H2O
ac1p8.out	4.908	17.5	0	1.000483	0.99869	5.6991E-04	0.066818	215.87	34 B4C pins/H2O
ac1p9.out	6.544	17.5	0	1.000483	0.99869	5.6991E-04	0.066818	215.87	H2O
ac1p10.out	6.544	24.5	143	0.998967	0.99718	5.6991E-04	0.066616	215.22	H2O
acp11a.out	1.636	25.5	510	0.999712	0.99692	5.6991E-04	0.066648	215.32	0.462 cm, SS 304/H2O
acp11b.out	1.636	26	514	0.998578	0.99992	5.6991E-04	0.066773	215.73	0.462 cm, SS 304/H2O
acp11c.out	1.636	25.5	501	0.999712	0.99692	5.6991E-04	0.066648	215.32	0.462 cm, SS 304/H2O
acp11d.out	1.636	25.5	493	0.998840	0.99692	5.6991E-04	0.066590	215.14	0.462 cm, SS 304/H2O
acp11e.out	1.636	25	474	0.999712	0.99404	5.6991E-04	0.066456	214.70	0.462 cm, SS 304/H2O
acp11f.out	1.636	25	462	0.998840	0.99404	5.6991E-04	0.066398	214.52	0.462 cm, SS 304/H2O
acp11g.out	1.636	25.5	432	0.999712	0.99992	5.6991E-04	0.066849	215.97	0.462 cm, SS 304/H2O
ac1p12.out	3.272	26	217	0.998578	0.99679	5.6991E-04	0.066564	215.05	0.462 cm, SS 304/H2O
ac1p13.out	1.636	20	15	1.000000	0.99821	5.6991E-04	0.066754	215.67	0.645 cm, BAI 1.614 wt% B/H2O
acp13a.out	1.636	17	28	1.000572	0.99878	5.6991E-04	0.066830	215.91	0.645 cm, BAI 1.614 wt% B/H2O
ac1p14.out	1.636	18	92	1.000392	0.99860	5.6991E-04	0.066806	215.83	0.645 cm, BAI 1.614 wt% B/H2O
ac1p15.out	1.636	18	395	1.000392	0.99860	5.6991E-04	0.066806	215.83	0.645 cm, BAI 1.614 wt% B/H2O

- a) Reference 8.
- b) Boron factors to correct water density from 25°C to 20°C. Boron ppm is based upon 25°C measurements. See Reference 7, Table 3.0-1, p. 46. Water density from standard tables.
- c) Calculated values based upon Avogadro's number of 0.6022142E-24

**Table 6-49 Close Proximity Experiment Trending Data (continued)**

Case ID	Cluster Spacing <sup>a</sup> , cm	Temp <sup>a</sup> , °C	Boron <sup>a</sup> , ppm	Boron Factors <sup>b</sup>	Water density <sup>b</sup> , g/cm <sup>3</sup>	Atom Density <sup>c</sup>		H/U <sup>c</sup>	Absorbers <sup>a</sup>
						U-235	H		
ac1p16.out	3.272	17.5	121	1.000483	0.99878	5.6991E-04	0.066824	215.89	0.645 cm, BAI 1.614 wt% B/H <sub>2</sub> O
ac1p17.out	1.636	17.5	487	1.000483	0.99878	5.6991E-04	0.066824	215.89	0.645 cm, BAI 1.614 wt% B/H <sub>2</sub> O
ac1p18.out	3.272	18	197	1.000392	0.99860	5.6991E-04	0.066806	215.83	0.645 cm, BAI 1.614 wt% B/H <sub>2</sub> O
ac1p19.out	1.636	17.5	634	1.000483	0.99878	5.6991E-04	0.066824	215.89	0.645 cm, BAI 1.614 wt% B/H <sub>2</sub> O
ac1p20.out	3.272	17.5	320	1.000483	0.99878	5.6991E-04	0.066824	215.89	0.645 cm, BAI 1.614 wt% B/H <sub>2</sub> O
ac1p21.out	6.544	16.5	72	1.000740	0.99992	5.6991E-04	0.066918	216.19	0.645 cm, BAI 1.614 wt% B/H <sub>2</sub> O

- a) Reference 8.
- b) Boron factors to correct water density from 25°C to 20°C. Boron ppm is based upon 25°C measurements. See Reference 7, Table 3.0-1, p. 46. Water density from standard tables.
- c) Calculated values based upon Avogadro's number of 0.6022142E-24

**Table 6-50 Close Proximity Experiment  $k_{eff}$  Data**

Case ID	Experimental $k_{eff}$ and $\sigma$		SCALE 4.4a 44 Group Cross-Section Calculated $k_{eff}$ and $\sigma$				SCALE 4.4a 238 Group Cross-Section Calculated $k_{eff}$ and $\sigma$			
	$k_{eff}^a$	$\sigma^a$	$k_{eff}^b$	$\sigma^b$	AFG <sup>b</sup>	EALF <sup>b</sup> (ev)	$k_{eff}^b$	$\sigma^b$	AFG <sup>b</sup>	EALF <sup>b</sup> (ev)
acp1.out	1.0002	0.0005	0.9931	0.0008	34.8710	0.1712	0.9889	0.0009	201.9510	0.1761
acp2.out	1.0001	0.0005	0.9956	0.0008	33.9420	0.2484	0.9939	0.0008	197.6580	0.2540
acp3.out	1.0000	0.0006	0.9963	0.0006	34.5210	0.1960	0.9934	0.0007	200.5280	0.2002
acp4.out	0.9999	0.0006	0.9897	0.0008	34.6110	0.1910	0.9875	0.0008	200.7350	0.1946
acp5.out	1.0000	0.0007	0.9883	0.0008	34.9500	0.1662	0.9873	0.0008	202.4670	0.1689
acp6.out	1.0097	0.0012	0.9884	0.0007	34.8840	0.1716	0.9872	0.0007	201.9760	0.1760
acp7.out	0.9998	0.0009	0.9900	0.0007	35.2100	0.1496	0.9867	0.0008	203.6900	0.1527
acp8.out	1.0083	0.0012	0.9906	0.0008	35.1720	0.1526	0.9874	0.0007	203.3420	0.1573
acp9.out	1.0030	0.0009	0.9906	0.0006	35.3620	0.1411	0.9879	0.0007	204.4120	0.1438
acp10.out	1.0001	0.0009	0.9913	0.0007	35.2090	0.1494	0.9883	0.0008	203.7410	0.1528
acp11a.out	1.0000	0.0006	0.9955	0.0007	34.4600	0.2001	0.9919	0.0006	200.2820	0.2046
acp11b.out	1.0007	0.0007	0.9942	0.0007	34.4640	0.1997	0.9916	0.0009	200.2900	0.2043
acp11c.out	1.0007	0.0006	0.9943	0.0008	34.4550	0.2007	0.9915	0.0008	200.1800	0.2060
acp11d.out	1.0007	0.0006	0.9939	0.0006	34.4290	0.2035	0.9920	0.0009	200.1670	0.2063
acp11e.out	1.0007	0.0006	0.9952	0.0007	34.4350	0.2030	0.9918	0.0006	200.0830	0.2078
acp11f.out	1.0007	0.0006	0.9947	0.0008	34.4360	0.2033	0.9916	0.0006	200.0020	0.2089
acp11g.out	1.0007	0.0006	0.9941	0.0007	34.4200	0.2054	0.9908	0.0007	199.9760	0.2096
acp12.out	1.0000	0.0007	0.9911	0.0007	34.8740	0.1702	0.9889	0.0008	202.2960	0.1727
acp13.out	1.0000	0.0010	0.9922	0.0007	34.5220	0.1963	0.9906	0.0009	200.3490	0.2013
acp13a.out	1.0000	0.0010	0.9901	0.0008	34.5020	0.1979	0.9884	0.0007	200.2550	0.2031
acp14.out	1.0001	0.0010	0.9905	0.0007	34.4720	0.2005	0.9891	0.0009	200.1840	0.2045
acp15.out	0.9998	0.0016	0.9881	0.0008	34.4020	0.2057	0.9823	0.0007	199.8980	0.2102
acp16.out	1.0001	0.0006	0.9860	0.0007	34.8250	0.1737	0.9841	0.0007	202.0010	0.1769
acp17.out	1.0007	0.0019	0.9897	0.0007	34.3970	0.2061	0.9874	0.0007	199.9490	0.2097
acp18.out	1.0002	0.0011	0.9869	0.0007	34.8410	0.1728	0.9859	0.0008	202.0310	0.1759
acp19.out	1.0002	0.0010	0.9910	0.0007	34.4010	0.2052	0.9888	0.0006	199.9530	0.2096
acp20.out	1.0003	0.0011	0.9889	0.0006	34.8410	0.1726	0.9869	0.0008	202.0440	0.1758
acp21.out	0.9997	0.0013	0.9868	0.0008	35.1290	0.1544	0.9854	0.0007	203.3850	0.1570

a) Values from Reference 3, Table 6, p. 42. Generally obtained from Tables 8 and 9 of Reference 8; acp11 series of values not documented in Reference 3.

b) From Reference 3, Table 9, p. 61 for 44 group cross-sections. Table 7 in this reference has values for 238 group cross-sections.

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#### 6.11.10.1.4. ***Critical Experiments Supporting Underwater Storage of Tightly Packed Configurations of Spent Fuel Pins***

##### **References:**

9. FANP Doc. 32-5012897-00, "Validation Report – SCALEPC-44A Consolidation Experiments," R.S. Harding
10. "Critical Experiments Supporting Underwater Storage of Tightly Packed Configurations of Spent Fuel Pins," G.S. Hoovler, et.al., BAW-1645-4, November, 1981.

Reference 9 uses the experimental data from Reference 10 to construct benchmark cases for SCALE 4.4a. Table 6-51 Tightly Packed Configuration Experiment Summary<sup>a</sup> summarizes the experimental configuration data that form the basis for the KENO V.a models. Table 6-52 Tightly Packed Configuration Experiment Trending Data provides trending parameters for this set of experiments. Table 6-53 Tightly Packed Configuration Experiment  $k_{eff}$  Data provides the experimental and calculated results for the 44 and 238 group SCALE 4.4a cross-section sets from Reference 3.

**Table 6-51 Tightly Packed Configuration Experiment Summary<sup>a</sup>**

U-235 wt%	2.459	Fuel Volume, cm <sup>3</sup>	0.833229
Pellet Density <sup>b</sup> , g/cm <sup>3</sup>	10.233	Pitch, cm	Vh20/Ffuel
U-235 atom density <sup>c</sup>	5.7075E-04	1.2093	0.149022
Pellet OD, cm	1.0300	1.2090	0.383292
Rod ID, cm	1.0440	1.4097	1.014058
Rod OD, cm	1.2060		
Clad Material	Al		

- a) From Reference 9.
- b) Based upon rod mass and fuel volume in rod, note this is the same 2.459 wt% fuel used in the previous 2 benchmark cases. The difference in densities has not been discussed.
- c) Calculated values based upon Avogadro's number of 0.6022142E-24.

**Table 6-52 Tightly Packed Configuration Experiment Trending Data**

Case ID	Rod Pitch <sup>a</sup> , cm	Lattice <sup>a</sup>	Cluster Spacing <sup>a</sup> , cm	Temp <sup>a</sup> , °C	Boron <sup>a</sup> , ppm	Boron Factor <sup>b</sup>	Water density <sup>b</sup>	V <sub>h2o</sub> /V <sub>fuel</sub> <sup>c</sup>	H atom density <sup>c</sup>	H/U <sub>f</sub> <sup>c</sup>
rcon01.out	1.2093	15x17 tria <sup>d</sup>	1.778x1.945	22.5	435	0.999451	0.99767	0.1490	0.066681	17.41
rcon02.out	1.2093	15x17 tria	1.778x1.945	23.5	426	0.999214	0.99742	0.1490	0.066648	17.40
rcon03.out	1.2093	15x17 tria	1.778x1.945	24.0	406	0.999091	0.99730	0.1490	0.066632	17.40
rcon04.out	1.2093	15x17 tria	1.778x1.945	22.5	383	0.999451	0.99767	0.1490	0.066681	17.41
rcon05.out	1.2093	15x17 tria	1.778x1.945	23.0	354	0.999334	0.99754	0.1490	0.066665	17.41
rcon06.out	1.2093	15x17 tria	1.778x1.945	23.0	335	0.999334	0.99754	0.1490	0.066665	17.41
rcon07.out	1.2093	15x17 tria	2.539x2.709	20.0	361	1.000000	0.99821	0.1490	0.066754	17.43
rcon09.out	1.2090	15x15 sq	1.7780	21.0	886	0.999788	0.99799	0.3833	0.066725	44.81
rcon10.out	1.2090	15x15 sq	1.7780	21.0	871	0.999788	0.99799	0.3833	0.066725	44.81
rcon11.out	1.2090	15x15 sq	1.7780	22.0	852	0.999566	0.99777	0.3833	0.066695	44.79
rcon12.out	1.2090	15x15 sq	1.7780	21.0	834	0.999788	0.99799	0.3833	0.066725	44.81
rcon13.out	1.2090	15x15 sq	1.7780	21.0	815	0.999788	0.99799	0.3833	0.066725	44.81
rcon14.out	1.2090	15x15 sq	1.7780	22.0	781	0.999566	0.99777	0.3833	0.066695	44.79
rcon15.out	1.2090	15x15 sq	1.7780	22.0	746	0.999566	0.99777	0.3833	0.066695	44.79
rcon16.out	1.4097	13x13 sq	1.7920	22.5	1156	0.999451	0.99767	1.0141	0.066681	118.47
rcon17.out	1.4097	13x13 sq	1.7920	22.5	1141	0.999451	0.99767	1.0141	0.066681	118.47
rcon18.out	1.4097	13x13 sq	1.7920	23.0	1123	0.999334	0.99754	1.0141	0.066665	118.44
rcon19.out	1.4097	13x13 sq	1.7920	23.0	1107	0.999334	0.99754	1.0141	0.066665	118.44
rcon20.out	1.4097	13x13 sq	1.7920	23.0	1093	0.999334	0.99754	1.0141	0.066665	118.44
rcon21.out	1.4097	13x13 sq	1.7920	23.0	1068	0.999334	0.99754	1.0141	0.066665	118.44
rcon28.out	1.4097	15x17 tria	3.807x2.976	18.5	121	1.000298	0.99850	1.0141	0.066793	17.44

- a) Reference 9.
- b) Boron factors to correct water density from 25°C to 20°C. Boron ppm is based upon 25 °C measurements. See Reference 10, Table 3.0-1, p. 46. Water density from standard tables.
- c) Calculated values based upon Avogadro's number of 0.6022142E-24.
- d) Triangular pitch for array.



**Table 6-53 Tightly Packed Configuration Experiment keff Data**

Case ID	Experimental $k_{eff}$ and $\sigma$		SCALE 4.4a 44 Group Cross-Section Calculated $k_{eff}$ and $\sigma$				SCALE 4.4a 238 Group Cross-Section Calculated $k_{eff}$ and $\sigma$			
	$k_{eff}^a$	$\sigma^a$	$k_{eff}^b$	$\sigma^b$	AFG <sup>b</sup>	EALF <sup>b</sup> (ev)	$k_{eff}^b$	$\sigma^b$	AFG <sup>b</sup>	EALF <sup>b</sup> (ev)
rcon01.out	1.0007	0.0006	0.9999	0.0008	28.9400	2.4011	0.9910	0.0007	170.1330	2.4368
rcon02.out	1.0007	0.0006	1.0009	0.0007	28.9020	2.4444	0.9909	0.0008	169.9770	2.4688
rcon03.out	1.0007	0.0006	0.9973	0.0008	28.8680	2.4872	0.9882	0.0007	169.6020	2.5454
rcon04.out	1.0007	0.0006	1.0008	0.0007	28.8990	2.4644	0.9899	0.0007	169.6960	2.5284
rcon05.out	1.0007	0.0006	0.9995	0.0008	28.8970	2.4706	0.9899	0.0008	169.6200	2.5435
rcon06.out	1.0007	0.0006	0.9980	0.0007	28.8900	2.4915	0.9906	0.0008	169.5520	2.5553
rcon07.out	1.0007	0.0006	0.9982	0.0008	29.8910	1.6259	0.9904	0.0008	175.2760	1.6431
rcon09.out	1.0007	0.0006	0.9977	0.0006	29.8930	1.4607	1.0092	0.0007	180.0400	1.1271
rcon10.out	1.0007	0.0006	0.9966	0.0008	29.8760	1.4759	0.9884	0.0006	176.1470	1.4891
rcon11.out	1.0007	0.0006	0.9959	0.0007	29.8450	1.4982	0.9909	0.0008	176.1150	1.4922
rcon12.out	1.0007	0.0006	0.9980	0.0008	29.8490	1.4979	0.9876	0.0007	175.8550	1.5240
rcon13.out	1.0007	0.0006	0.9969	0.0007	29.8430	1.5074	0.9897	0.0007	175.8220	1.5280
rcon14.out	1.0007	0.0006	0.9963	0.0007	29.8310	1.5207	0.9894	0.0007	175.7230	1.5402
rcon15.out	1.0007	0.0006	0.9975	0.0008	29.8450	1.5180	0.9915	0.0007	175.7200	1.5399
rcon16.out	1.0007	0.0006	0.9948	0.0007	32.7100	0.4216	0.9892	0.0007	175.7140	1.5415
rcon17.out	1.0007	0.0006	0.9952	0.0006	32.6820	0.4276	0.9894	0.0006	191.3680	0.4309
rcon18.out	1.0007	0.0006	0.9939	0.0006	32.6400	0.4370	0.9909	0.0007	191.2180	0.4360
rcon19.out	1.0007	0.0006	0.9965	0.0006	32.6540	0.4344	0.9897	0.0007	191.0430	0.4426
rcon20.out	1.0007	0.0006	0.9967	0.0007	32.6370	0.4391	0.9915	0.0007	190.9880	0.4447
rcon21.out	1.0007	0.0006	0.9959	0.0008	32.6220	0.4427	0.9903	0.0007	190.8780	0.4485
rcon28.out	1.0007	0.0006	0.9968	0.0008	31.0790	1.0062	0.9915	0.0008	190.7670	0.4529

a) Values from Reference 3, Table 6, p. 42. Source of value not documented in this reference.

b) From Reference 3, Table 9, p. 61 for 44 group cross-sections. Table 7 in this reference has values for 238 group cross-sections



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#### 6.11.10.1.5. *Reduced Density Moderation Between Fuel Clusters with 4.738 Wt% Fuel*

##### References:

11. FANP Doc. 32-5012894-00, "Validation Report – SCALEPC-44A Dissolution Experiments," R.S. Harding.
12. "Dissolution and Storage Experimental Program with U[4.75]O<sub>2</sub> Rods," Transactions of the American Nuclear Society, Vol. 33, pg. 362.

Reference 11 uses the experimental data from Reference 12 to construct benchmark cases for SCALE 4.4a. Table 6-54 Reduced Density Moderation Experiments Summary and Trending Parameters<sup>a</sup> summarizes the experimental configuration data that form the basis for the KENO V.a models and provides trending parameters that are constant for the series of experiments. Table 6-55 Reduced Density Moderation Experiments Trending Data and  $k_{\text{eff}}$  Data provides trending parameters for this set of experiments. It also provides the experimental and calculated results for the 44 and 238 group SCALE 4.4a cross-section sets from Reference 3.

**Table 6-54 Reduced Density Moderation Experiments Summary and Trending Parameters<sup>a</sup>**

U-235 wt%	4.738	Temperature, °C	22
Pellet Density, g/cm <sup>3</sup>	10.38	Water density, g/cm <sup>3</sup>	0.99777
Pellet OD, cm	0.7900	Fuel Volume, cm <sup>3</sup>	0.49017
Rod ID, cm	0.8200	Water Volume, cm <sup>3</sup>	1.12852
Rod OD, cm	0.9400	V <sub>h2o</sub> /V <sub>fuel</sub>	2.30232
Rod Pitch, cm	1.3500	U-235 atom density <sup>b</sup>	1.1155E-03
Clad Material	Al alloy	H atom density <sup>b</sup>	0.066676
Lattice	18x18	H/U	1.3761E+02

a) From Reference 11.

b) Calculated values based upon Avogadro's number of 0.6022142E-24.

**Table 6-55 Reduced Density Moderation Experiments Trending Data and  $k_{eff}$  Data**

Case ID	Cluster Spacing <sup>a</sup> , cm	Spacing Material <sup>a</sup> [Material (density)]	Experimental $k_{eff}$ and $\sigma$		SCALE 4.4a 44 Group Cross-Section Calculated $k_{eff}$ and $\sigma$				SCALE 4.4a 238 Group Cross-Section Calculated $k_{eff}$ and $\sigma$			
			$k_{eff}^b$	$\sigma^b$	$k_{eff}^c$	$\sigma^c$	AFG <sup>c</sup>	EALF <sup>c</sup> (ev)	$k_{eff}^c$	$\sigma^c$	AFG <sup>c</sup>	EALF <sup>c</sup> (ev)
mdis01.out	0.0	-	1.0000	0.0014	0.9914	0.0008	33.5390	0.2824	0.9885	0.0010	195.994	0.2879
mdis02.out	2.5	H2O	1.0000	0.0014	0.9871	0.0009	33.6720	0.2644	0.9862	0.0008	196.836	0.2685
mdis03.out	2.5	Air/Box	1.0000	0.0014	0.9841	0.0011	33.6720	0.2647	0.9805	0.0008	196.750	0.2702
mdis04.out	2.5	Polyst(0.0323)/Box	1.0000	0.0014	0.9902	0.0008	33.8040	0.2514	0.9884	0.0008	197.439	0.2559
mdis05.out	2.5	Polyeth(0.2879)/Box	1.0000	0.0014	0.9908	0.0010	33.9160	0.2407	0.9891	0.0009	198.001	0.2442
mdis06.out	2.5	Polyeth(0.5540)/Box	1.0000	0.0014	1.0008	0.0010	34.0370	0.2295	0.9963	0.0008	198.539	0.2344
mdis07.out	2.5	H2O/Box	1.0000	0.0014	0.9917	0.0009	34.1100	0.2242	0.9886	0.0008	198.827	0.2288
mdis08.out	5.0	H2O	1.0000	0.0014	0.9873	0.0010	33.8000	0.2497	0.9840	0.0009	197.504	0.2545
mdis09.out	5.0	Air/Box	1.0000	0.0014	0.9869	0.0010	33.8110	0.2485	0.9861	0.0009	197.586	0.2524
mdis10.out	5.0	Polyst(0.0323)/Box	1.0000	0.0014	0.9938	0.0008	34.0940	0.2225	0.9912	0.0008	198.934	0.2267
mdis11.out	5.0	Polyeth(0.2879)/Box	1.0000	0.0014	1.0031	0.0010	34.3010	0.2048	0.9997	0.0008	200.018	0.2076
mdis12.out	5.0	Polyeth(0.5540)/Box	1.0000	0.0014	-	-	-	-	1.0027	0.0009	200.577	0.1984
mdis13.out	5.0	H2O/Box	1.0000	0.0014	0.9907	0.0008	34.4280	0.1951	0.9878	0.0008	200.547	0.1988
mdis14.out	10.0	H2O	1.0000	0.0014	0.9890	0.0008	33.9850	0.2294	0.9854	0.0009	198.552	0.2333
mdis15.out	10.0	Air/Box	1.0000	0.0014	0.9894	0.0009	34.0150	0.2266	0.9842	0.0008	198.647	0.2315
mdis16.out	10.0	Polyst(0.0323)/Box	1.0000	0.0014	1.0013	0.0008	34.4450	0.1907	0.9970	0.0009	200.792	0.1948
mdis17.out	10.0	Polyeth(0.2879)/Box	1.0000	0.0014	0.9985	0.0008	34.5970	0.1788	0.9951	0.0009	201.537	0.1831
mdis18.out	10.0	Polyeth(0.5540)/Box	1.0000	0.0014	0.9965	0.0008	34.6430	0.1740	0.9923	0.0009	201.894	0.1774
mdis19.out	10.0	H2O/Box	1.0000	0.0014	0.9931	0.0009	34.6530	0.1737	0.9888	0.0008	201.908	0.1772

- a) References 11 and 12.
- b) Values from Reference 3, Table 6, p. 42. Source of value not documented in this reference.
- c) From Reference 3, Table 9, p. 61 for 44 group cross-sections. Table 7 in this reference has values for 238 group cross-sections

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## 6.12. **APPENDIX B: 11X11 FUEL ASSEMBLY CRITICALITY ANALYSIS**

Appendix B documents the criticality analysis of the TN-B1 package with a payload of two 11x11 fuel assemblies. Appendix B also documents the criticality analysis for up to 25 11x11 loose rods or up to 30 11x11 rods in a 5-in stainless steel pipe. The format of Appendix B follows the same general outline of the main body of the report and references the main body of the report for information common to both analyses.

The 11x11 fuel assembly criticality analysis is performed using SCALE 6.1.3 (Reference 15), while the analysis for the 8x8, 9x9, and 10x10 was performed using SCALE 4.4a (Reference 8). SCALE 6.1.3 is used for the 11x11 fuel assembly analysis to take advantage of the ENDF/B-VII cross section data and resonance cross section processing using CENTRM. In addition, SCALE 6.1.3 allows fuel resonance parameters to be computed separately for fuel rods with and without gadolinia without the use of the MORE DATA card, which simplifies input preparation. Benchmarking for SCALE 6.1.3 is documented in Section 6.12.9, *Benchmark Evaluation for SCALE 6.1.3*. A USL of 0.94094 is justified for the 11x11 fuel assembly analysis, and a USL of 0.94047 is justified for the 11x11 fuel rod analysis.

### 6.12.1. ***Description of the Criticality Design***

#### 6.12.1.1. **Design Features**

Refer to Section 6.1.1, *Design Features*, for a description of the design features of the package.

A criticality safety analysis is performed to demonstrate the TN-B1 shipping container safety for the 11x11 fuel assembly and associated rods when detached from the fuel assembly. The TN-B1 meets applicable IAEA and 10 CFR 71 requirements for a Type B fissile material-shipping container, transporting heterogeneous UO<sub>2</sub> enriched to a maximum of 5.00 wt. percent U-235.

Water exclusion from the inner container is not required for this package design. The inner container is analyzed in both undamaged and damaged package arrays under optimal moderation conditions and is demonstrated to be safe under Normal Conditions of Transport (NCT) and Hypothetical Accident Condition (HAC) testing.

Sensitivity analyses are performed by varying fuel parameters (rod pitch, clad ID, clad OD, pellet OD, fuel orientation, polyethylene quantity, and moderator density) to obtain the most reactive configuration.


Table 6-56 TN-B1 11x11 Fuel Assembly General Loading Criteria summarizes the general fuel loading criteria for the TN-B1 shipping container for 11x11 fuel. Table 6-57 TN-B1 11x11 Fuel Assembly Gadolinia Loading Criteria summarizes the gadolinia loading requirements for 11x11 fuel assemblies. The gadolinia loading requirements are provided as a minimum number of

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2.0 wt.% gadolinia-urania rods required per lattice. Required gadolinia-urania rods shall be distributed symmetrically about the major diagonal and shall not be placed on the periphery.

Plastic inserts may optionally be used between fuel rods at various locations along the axis of the fuel assembly to avoid stressing the axial grids during transportation. An example of a plastic insert is shown in Figure 6-1 Polyethylene Insert (FANP Design). The option for plastic inserts is bounded by allowing up to 10.2 kg of polyethylene equivalent mass per fuel assembly. In addition to the inserts, the polyethylene equivalent mass also includes all polyethylene/plastic components within the inner compartment, excluding the foam liner. Polyethylene equivalent mass is defined in Section 6.3.2.2.

Cylindrical fuel rods containing  $\text{UO}_2$ , enriched to 5 wt. percent U-235, are analyzed within the TN-B1 inner container in a 5-inch stainless steel pipe (which bounds fuel rods in a protective case), or loose within the cavity (which bounds fuel rods bundled together). No gadolinia credit is taken in the fuel rod analysis. Only bare pellet columns (i.e., no cladding) are conservatively modeled. The fuel rod loading criteria are provided in Table 6-58 TN-B1 11x11 Fuel Rod Loading Criteria.

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**Table 6-56 TN-B1 11x11 Fuel Assembly General Loading Criteria**

Parameter	Units	Value
UO <sub>2</sub> Density <sup>c</sup>	g/cm <sup>3</sup>	≤ 10.763
Number of water rods	#	3x3 center
Number of fuel rods	#	112
Fuel Rod OD	cm	≥ 0.930
Fuel Pellet OD	cm	≤ 0.820
Cladding Type		Zirconium Alloy
Cladding ID	cm	≤ 0.840
Cladding Thickness	cm	≥ 0.045
Fuel Rod Pitch <sup>a</sup>	cm	≤ 1.195
U-235 Pellet Enrichment	wt%	≤ 5.0
Maximum Lattice Average Enrichment	wt%	≤ 5.0
Fuel Channel Side Thickness <sup>b</sup>	cm	≤ 0.254
Full Length Fuel Rods		
Quantity	#	92
Active length	cm	≤ 385
Short Part Length Fuel Rods		
Quantity	#	12
Active length	cm	≤ 155.1
Long Part Length Fuel Rods		
Quantity	#	8
Active length	cm	≤ 236.8

- a. Equivalent nominal pitch per Section 6.12.3.1.1.  
b. Transport with or without channels is acceptable.  
c. Density based on a pellet modeled as a right cylinder.

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**Table 6-57 TN-B1 11x11 Fuel Assembly Gadolinia Loading Criteria**

Parameter	Units	Type
Gadolinia Requirements		
Lattice Average Enrichment <sup>a</sup>	# @ wt% Gd <sub>2</sub> O <sub>3</sub>	
≤ 5.0 wt % U-235		13 @ 2 wt %
≤ 4.8 wt % U-235		12 @ 2 wt %
≤ 4.6 wt % U-235		11 @ 2 wt %
≤ 4.4 wt % U-235		10 @ 2 wt %
≤ 4.2 wt % U-235		9 @ 2 wt %
≤ 4.1 wt % U-235		8 @ 2 wt %
≤ 3.9 wt % U-235		7 @ 2 wt %
≤ 3.8 wt % U-235		6 @ 2 wt %
≤ 3.6 wt % U-235		5 @ 2 wt %
≤ 3.5 wt % U-235		4 @ 2 wt %
≤ 3.3 wt % U-235		3 @ 2 wt %
≤ 3.2 wt % U-235		2 @ 2 wt %
≤ 2.9 wt % U-235		None
Polyethylene Equivalent Mass (Maximum per Assembly) <sup>b</sup>	kg	10.2

- Required gadolinia rods shall be distributed symmetrically about the major diagonal and shall not be placed on the periphery.
- Polyethylene equivalent mass (refer to Section 6.3.2.2)

**Table 6-58 TN-B1 11x11 Fuel Rod Loading Criteria**

Parameter	Units	Type
UO <sub>2</sub> Density <sup>a</sup>	g/cm <sup>3</sup>	≤ 10.763
Fuel Rod OD	cm	≥ 0.930
Fuel Pellet OD	cm	≤ 0.820
Cladding Type		Zirc. Alloy
Cladding ID	cm	≤ 0.930
Cladding Thickness	cm	≥ 0.00
Active Fuel Length	cm	≤ 385
Maximum U-235 Pellet Enrichment	wt. %	≤ 5.0
Maximum Average Fuel Rod Enrichment	wt. %	≤ 5.0
Rods loose in cavity (or bundled)	#	≤ 25 (50 per package)
Rods in pipe component (or protective case)	#	≤ 30 (60 per package)

- Density based on a pellet modeled as a right cylinder.

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#### 6.12.1.2. Summary Table of Criticality Evaluation

The following configurations are allowed for transport in the TN-B1:

- Up to two 11x11 fuel assemblies
- Up to 25 11x11 fuel rods loose within each package cavity (50 rods per package). This configuration bounds 25 fuel rods bundled together.
- Up to 30 11x11 fuel rods contained within a 5-in stainless steel pipe within each package cavity (up to 60 rods per package). This configuration bounds the stainless steel protective case, which has a smaller cross sectional area than the stainless steel pipe component.

Calculations are performed for the NCT and HAC single package, NCT array, and HAC array. Water moderation is allowed for both NCT and HAC cases. The HAC array size is 10x1x10 (100 packages) and the NCT array size is 21x3x24 (1,512 packages). The most reactive results are summarized in Table 6-59 Criticality Evaluation Summary for 11x11 Fuel Assemblies and Rods. Note that because the NCT single package case is flooded, it meets the requirements of 71.55(b).



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**Table 6-59 Criticality Evaluation Summary for 11x11 Fuel Assemblies and Rods**

Case	Bounding Fuel Type	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$	USL
Fuel Assembly Single Package NCT	11x11 with worst case fuel parameters, 13-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods	0.63082	0.00042	0.63166	0.94094
Fuel Assembly Single Package HAC	11x11 with worst case fuel parameters, 13-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods	0.76615	0.00045	0.76705	0.94094
Fuel Assembly Package Array NCT	11x11 with worst case fuel parameters, 13-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods	0.85303	0.00040	0.85383	0.94094
Fuel Assembly Package Array HAC	11x11 with worst case fuel parameters, 13/13/3-2.0 wt% Gd <sub>2</sub> O <sub>3</sub> fuel rods <sup>a</sup>	0.93982	0.00043	0.94068	0.94094
Fuel Rod Single Package NCT	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.59145	0.00045	0.59235	0.94047
Fuel Rod Single Package HAC	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.66316	0.00042	0.66400	0.94047
Fuel Rod Package Array NCT	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.59300	0.00042	0.59384	0.94047
Fuel Rod Package Array HAC	30 fuel rods in stainless steel pipe (2 per container) with worst case fuel parameters <sup>b</sup>	0.81947	0.00044	0.82035	0.94047

- a. This configuration contains 13-2.0wt% Gd<sub>2</sub>O<sub>3</sub> fuel rods in the bottom and middle axial regions and 3-2.0 wt% Gd<sub>2</sub>O<sub>3</sub> fuel rods in the top axial region.
- b. This configuration bounds the 25 loose fuel rod configuration.

### 6.12.1.3. Criticality Safety Index

Undamaged packages have been analyzed in 21x3x24 arrays and damaged packages have been analyzed in 10x1x10 arrays. Pursuant to 10 CFR 71.59, the number of packages “N” in a 2N array that are subjected to the tests specified in 10 CFR 71.73, or in a 5N array for undamaged packages is used to determine the Criticality Safety Index (CSI). The CSI is determined by dividing the number 50 by the most limiting value of “N” as specified in 10 CFR 71.59.

The TN-B1 criticality analysis demonstrates safety for 5N=1,512 (undamaged) and 2N=100 (damaged) packages. The corresponding CSI for criticality control is given by  $CSI = 50/N$ . Since 5N=1,512 and 2N = 100, it follows that the more restrictive N = 50 and  $CSI = 50/50 = 1.0$ . Therefore the maximum allowable number of packages per shipment is  $50/1.0 = 50$ .

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Note that the NCT array size is significantly larger than the array size needed to justify a CSI of 1.0 (5N = 250 minimum). The large NCT array size is selected to be consistent with the NCT array size utilized in the 8x8, 9x9, and 10x10 fuel assembly analysis.

### 6.12.2. *Fissile Material Contents*

The following fissile material contents are allowed for transport in the TN-B1:

- Up to two 11x11 fuel assemblies
- Up to 25 11x11 fuel rods loose within each package cavity (50 rods per package). This configuration bounds 25 fuel rods bundled together.
- Up to 30 11x11 fuel rods contained within a 5-in stainless steel pipe component within each package cavity (up to 60 rods per package). This configuration bounds the stainless steel protective case, which has a smaller cross sectional area than the stainless steel pipe component.

Details of the 11x11 fuel assembly and loose rods are defined in Table 6-56 TN-B1 11x11 Fuel Assembly General Loading Criteria, Table 6-57 TN-B1 11x11 Fuel Assembly Gadolinia Loading Criteria, and Table 6-58 TN-B1 11x11 Fuel Rod Loading Criteria.


### 6.12.3. *General Considerations*

#### 6.12.3.1. **Model Configuration**

##### 6.12.3.1.1. *Fuel Assembly Model*

Details of the development of the bounding fuel assembly model are provided in Section 6.12.3.5, *Parameter Selection for 11x11 Fuel Assembly Model*. Following is a summary of the bounding fuel assembly. The bounding fuel assembly has the following parameters:

- NCT: pitch = 1.195 cm, HAC: pitch = 1.2548 cm (5% pitch expansion)
- NCT: dry pellet/cladding gap, HAC: wet pellet/cladding gap
- NCT/HAC: pellet density = 10.763 g/cm<sup>3</sup>
- NCT/HAC: pellet diameter = 0.820 cm
- NCT/HAC: cladding outer diameter = 0.930 cm
- NCT/HAC: cladding inner diameter = 0.840 cm
- NCT/HAC: 10.2 kg of polyethylene equivalent mass per fuel assembly. The polyethylene equivalent mass includes all polyethylene/plastic components within the inner compartment, excluding the foam liner.
- NCT/HAC (single package): 5% enrichment with 13 gadolinia-urania rods per lattice with the loading pattern as shown on Figure 6-52 Bounding Fuel Assembly Model.
- HAC (package array): 5% enrichment with 13 gadolinia-urania rods in the bottom and middle axial regions and 3.3% enrichment with 3 gadolinia-urania rods per lattice in the

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top region with the loading pattern as shown on Figure 6-52 Bounding Fuel Assembly Model.

- NCT/HAC: Zirconium water channel (i.e., the zirconium channel in the center of the fuel assembly) modeled as water
- NCT/HAC: Models developed with and without the zirconium fuel channel. A range of thicknesses is examined (0 to 0.254 cm).

The 11x11 fuel assembly has variable rod pitch, with the upper section of the assembly having a constant 1.195 cm nominal pitch and the lower section of the assembly having rods at varying pitches (see Reference 20). Both sections have the same total distance across the assembly. The variation from the nominal pitch in the lower region is very small, <1 mm. Since the overall envelope of the fuel region is unchanged, the H/U-235 ratio for this region is the same as a region which is fully rodded (i.e., no water spaces from partial length rods). As such, no reactivity change is anticipated for this region and the pitch throughout the entire assembly is modeled with an equivalent nominal pitch of 1.195 cm in the NCT models.

In the HAC models, the fuel assembly is modeled with a 5% expanded pitch. The average pitch expansion determined analytically using an LS-Dyna model is 2.6% (Reference 19). The expansion of the outside perimeter of the fuel assembly determined analytically using an LS-Dyna model is 3.4% (Reference 19). The pitch expansion modeled bounds the LS-Dyna computed pitch expansion. The 5% pitch expansion also bounds the 4.1% pitch expansion experimentally determined by drop testing an FANP 10x10 fuel assembly (Reference 1).

The 11x11 fuel assembly has three distinct axial regions (lattices) due to the presence of long and short partial-length rods. There are 92 full-length rods, 8 long partial-length rods, and 12 short partial-length rods. In the center of the fuel assembly is a 3x3 water hole. The bottom lattice has  $92+8+12 = 112$  rods, the middle lattice has  $92+8 = 100$  rods, and the top lattice has 92 rods. The loading pattern of the three rod types is depicted in Figure 6-51 Position of Fuel Rods in 11x11 Assembly.

The fuel rod lengths are modeled conservatively. The length of the partial length fuel rods is increased by 9.2 cm to account for upper and lower plenum regions and for additional conservatism. An additional 4 cm is conservatively added to the length of all fuel rods. This results in modeled lengths of 155.1 cm, 236.8 cm, and 385 cm for the short partial-length rods, long partial-length rods and full-length rods, respectively. All fuel is modeled such that the bottom of the fuel column for all rod types is even with each other.

No fuel assembly structures outside the active region of the assembly are represented in the models. The neglected structures are composed of materials that absorb neutrons by radiative capture; therefore, not considering these structures in the analysis is conservative. In addition, no grids are represented within the rod active length. The internal grid structure displaces water from between the fuel rods, decreasing moderation. Because the fuel assemblies are

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undermoderated, decreasing the moderation decreases system reactivity. Therefore, it is conservative to neglect the internal grid structure in the TN-B1 container analysis.

Some fuel rods contain gadolinia to hold-down the reactivity. The minimum allowed gadolinia loading is 2 wt.%. In the KENO models, only 75% credit is taken for the gadolinia. The number of gadolinia-urania rods required is a function of the lattice average enrichment. Gadolinium may be present in the full-length and/or partial-length rods. For any given fuel assembly, the number of gadolinia-urania rods in each lattice is a constant, although the pattern may vary from lattice to lattice. The most reactive gadolinia-urania rod pattern is independently determined for each lattice within a fuel assembly.

The gadolinia-urania rod patterns are developed using the following rules: (1) the gadolinia-urania rods shall be symmetric about the major diagonal, and (2) gadolinia-urania rods shall not be placed on the periphery. For the NCT models and for the HAC single package, the final gadolinia-urania loading pattern for each of the three lattices for the bounding fuel assembly (5 wt.% enrichment, 13 gadolinia-urania rods per lattice) is illustrated in Figure 6-52 Bounding Fuel Assembly Model. For the HAC package array, the final gadolinia-urania loading pattern for each of the three lattices for the bounding fuel assembly (5 wt.% enrichment, 13 gadolinia-urania rods in the bottom and middle lattices and 3.3 wt.% enrichment, 3 gadolinia-urania rods in the top lattice) is also illustrated in Figure 6-52 Bounding Fuel Assembly Model. The bounding enrichment and gadolinia-urania loading pattern is developed in Section 6.12.3.5, Parameter Selection for 11x11 Fuel Assembly Model.

Each fuel assembly may contain up to 10.2 kg of polyethylene equivalent mass, as defined in Section 6.3.2.2. In the NCT models, 10.2 kg of polyethylene is homogenized with the water inside the fuel assembly boundary. **In the HAC models, the polyethylene is modeled in two configurations: (1) homogenized with the cladding (which approximates polyethylene melted onto the cladding), and (2) homogenized with the water inside the fuel assembly boundary.** The 10.2 kg of polyethylene equivalent mass does not include the polyethylene liner foam, which is treated separately.

#### 6.12.3.1.2. **Single Package NCT Model with 11x11 Fuel**

The geometrical parameters of the single package NCT models are the same as defined in Section 6.3.1.1.1, *Single Package Normal Conditions of Transport Model*. Refer to that section for detailed information about the packaging dimensions. Key dimensions of the NCT single package model are also illustrated on Figure 6-3 TN-B1 Outer Container Normal Conditions of Transport Model, Figure 6-4 TN-B1 Inner Container Normal Conditions of Transport Model, and Figure 6-5 TN-B1 Container Cross-Section Normal Conditions of Transport Model.

The fuel is modeled with or without fuel channels, which extend axially along the active fuel length. The polyethylene liner foam is modeled in close contact with the fuel assembly, and the

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fuel assembly is centered within each compartment. The foam has a nominal density of 4 pounds per cubic feet (pcf), although small strips of 9 pcf foam may be used under the grid spacers to provide extra support. The foam is conservatively modeled with a density of 5 pcf on the sides and top and 10 pcf on the bottom. Modeling 10 pcf foam across the entire bottom bounds the actual density.

In the NCT single package models, full-density water is modeled between the inner and outer containers to maximize reflection, and the entire package is reflected with 30.48 cm of water. Note that the single package NCT case with full flooding meets the requirements of 71.55(b).

#### 6.12.3.1.3. **Single Package HAC Model with 11x11 Fuel**

The geometrical parameters of the single package HAC models are the same as defined in Section 6.3.1.1.2, *Single Package Hypothetical Accident Condition Model*. Refer to that section for detailed information about the packaging dimensions, which include damage from drop testing. Key dimensions of the HAC single package model are also illustrated on Figure 6-6 TN-B1 Outer Container Hypothetical Accident Condition Model, Figure 6-7 TN-B1 Inner Container Hypothetical Accident Condition Model, and Figure 6-8 TN-B1 Cross-Section Hypothetical Accident Condition Model.

The fuel is modeled with or without fuel channels, which extend axially along the active fuel length. The polyethylene foam liner is modeled as intact, partially burned, or totally burned to determine the most reactive condition. The burned foam is replaced with water. Fuel assemblies are also modeled in various orientations to determine the most reactive condition. Full-density water is modeled between the inner and outer containers to maximize reflection, and the entire package is reflected with 30.48 cm of water.

#### 6.12.3.1.4. **NCT Array Model with 11x11 Fuel**

The NCT array model is similar to the NCT single package model. The NCT array size is selected to be 21x3x24 (1,512 packages) to be consistent with the original analysis documented in Section 6.3.1.2.1, *Package Array Normal Condition Model*. The minimum NCT array size needed to justify a CSI = 1.0 is only 250 packages, a significantly smaller number.

Per Regulatory Guide 7.9 (Reference 16), if a water spray test has demonstrated that water would not leak into the package, water inleakage need not be assumed for the NCT array analysis. While little or no water intrusion is expected as a result of the water spray test, because a water spray test has not been performed, water inleakage is assumed in the NCT array models. Consistent with the approach in Section 6.3.1.2.1, *Package Array Normal Condition Model*, in the NCT array cases, the water density is modeled at a constant value in all void spaces within the package. Cases are run in which the water density ranges from 0 to 1.0 g/cm<sup>3</sup>. The entire package array is reflected with 30.48 cm of water.

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#### 6.12.3.1.5. **HAC Array Model with 11x11 Fuel**

The HAC array model is similar to the HAC single package model. The HAC array size is selected to be 10x1x10 (100 packages), consistent with a CSI = 1.0.

Potential burn scenarios of the polyethylene liner foam are investigated. As a result of a postulated fire, some or all of the foam may burn. In general, partial burn scenarios are the most reactive, and numerous partial burn scenarios are investigated. The most reactive assembly orientation is determined with and without the fuel channel.

In the HAC array analysis, the water density inside the inner container is allowed to vary independently from the water density between the inner and outer containers. The entire package array is reflected with 30.48 cm of water.

#### 6.12.3.1.6. **11x11 Rod Models**

An analysis is also performed for 11x11 rods (i.e., rods that are not part of a fuel assembly). Two separate analyses are performed for 11x11 rods: (1) rods loose (or bundled) within the liner, and (2) rods contained within a 5-in stainless steel pipe or protective case.

Up to 25 loose rods are placed directly into each liner, or up to 50 loose rods per package. Loose rods may also be bundled together, as bundles of close-packed rods would be less reactive than rods that are allowed to achieve an optimum pitch. In this loose rod configuration, the pitch is varied until optimum moderation is achieved, see *Figure 6-53 Twenty-five 11x11 Fuel Rods in Liner*.

Up to 30 loose rods are placed in a 5-in schedule 40 stainless steel pipe (see General Arrangement Drawing 0028B98, *Shipping Container Loose Fuel Rods*). Two pipes may be placed in the package, for a total of 60 rods per package. The pipe itself is conservatively modeled as water. A pitch study is performed for rods in the pipe component, and a triangular pitch is used to better match the circular pipe boundary, see *Figure 6-54 Thirty 11x11 Fuel Rods in Pipe Component*. Studies are also performed for less than 30 rods in the pipe, and various moderation scenarios are investigated to determine the most reactive configuration.

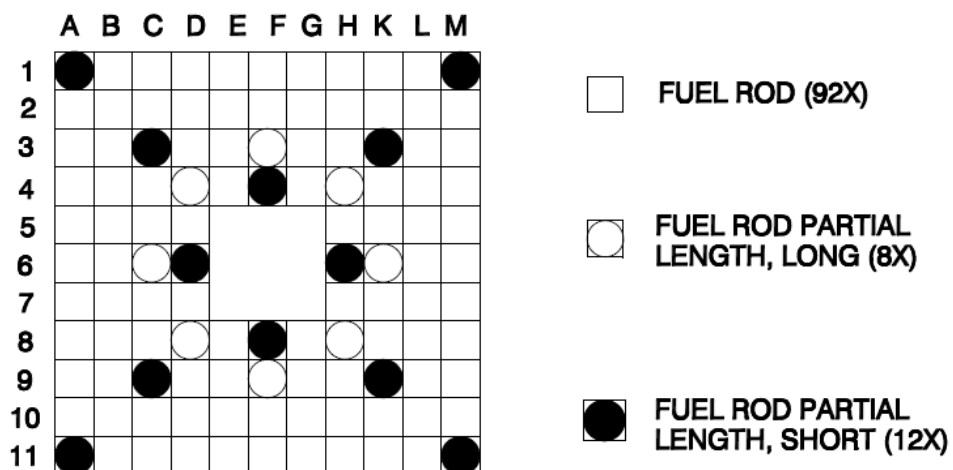
The analysis of 30 rods in a pipe component bounds 30 rods in the protective case (see General Arrangement Drawing 105E3773, *RAJ-II Protective Case Licensing Drawing*). The pipe component has an inner diameter of 5.047-in, or an area of 20.0 in<sup>2</sup>. The protective case has a cavity cross sectional area of 80 mm x 89 mm (3.15-in x 3.50-in), or an area of 11.0 in<sup>2</sup>. Therefore, the pipe component has a significantly larger area than the protective case and allows greater pitch expansion and moderation. Therefore, the analysis of 30 rods in the pipe component bounds 30 rods in the protective case.

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Each fuel rod is placed in a plastic sleeve 6-mil (0.006-in) thick, and this plastic sleeve is modeled explicitly around each fuel rod. In addition, the cladding is replaced with water to conservatively increase moderation. The pellet/cladding gap is also filled with water.

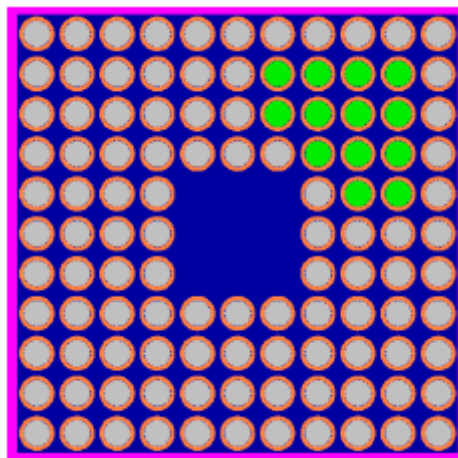


### POSITIONS OF THE FUEL RODS

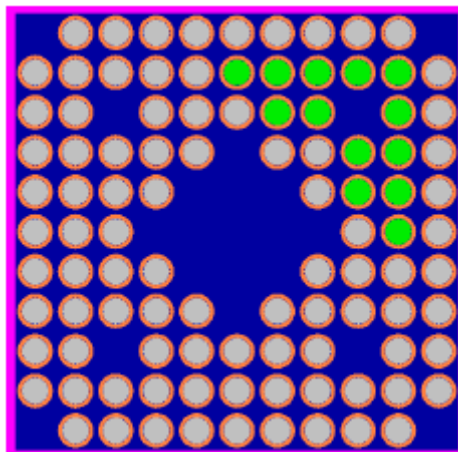


**Figure 6-51 Position of Fuel Rods in 11x11 Assembly**

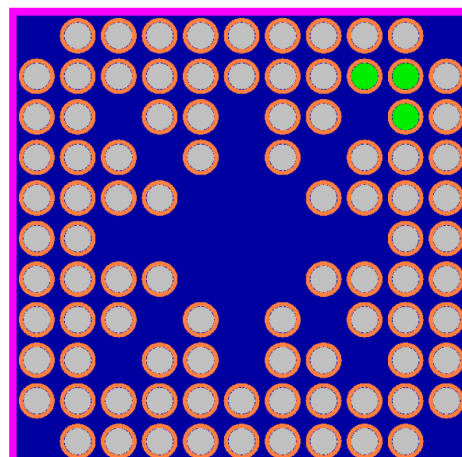
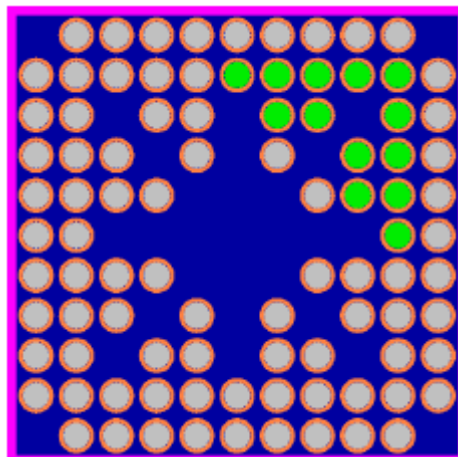
Bottom Axial Layer (5 wt%, Full and Partial Length Rods)



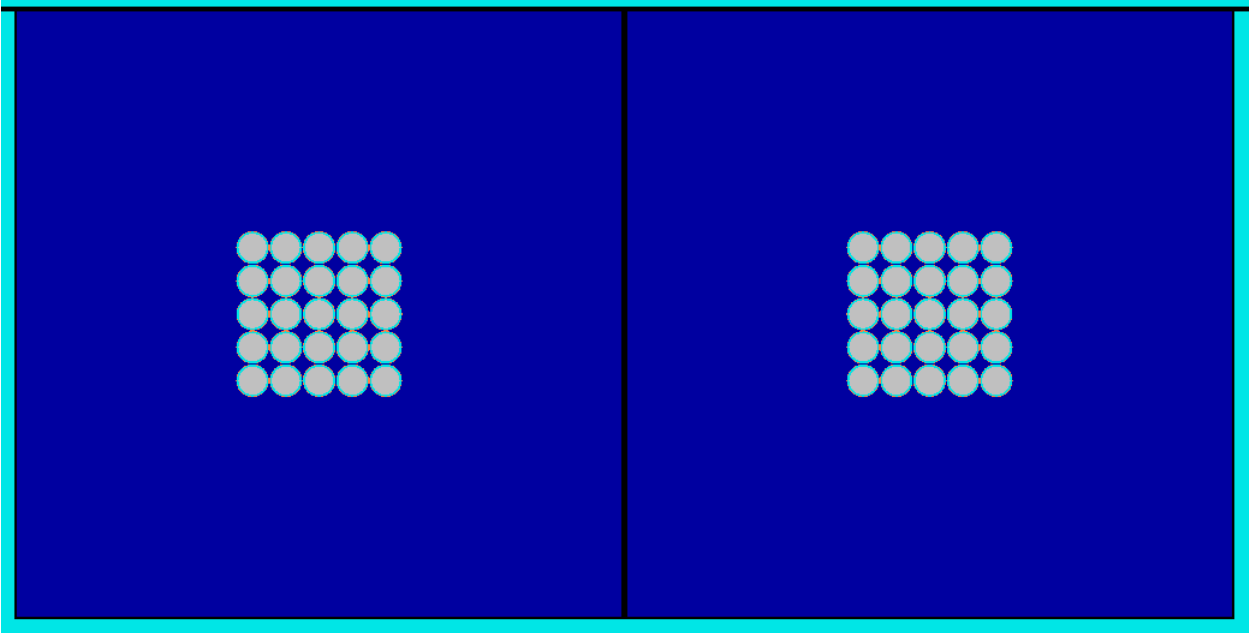
Middle Axial Layer (5 wt%, Full and Long Partial-Length Rods)



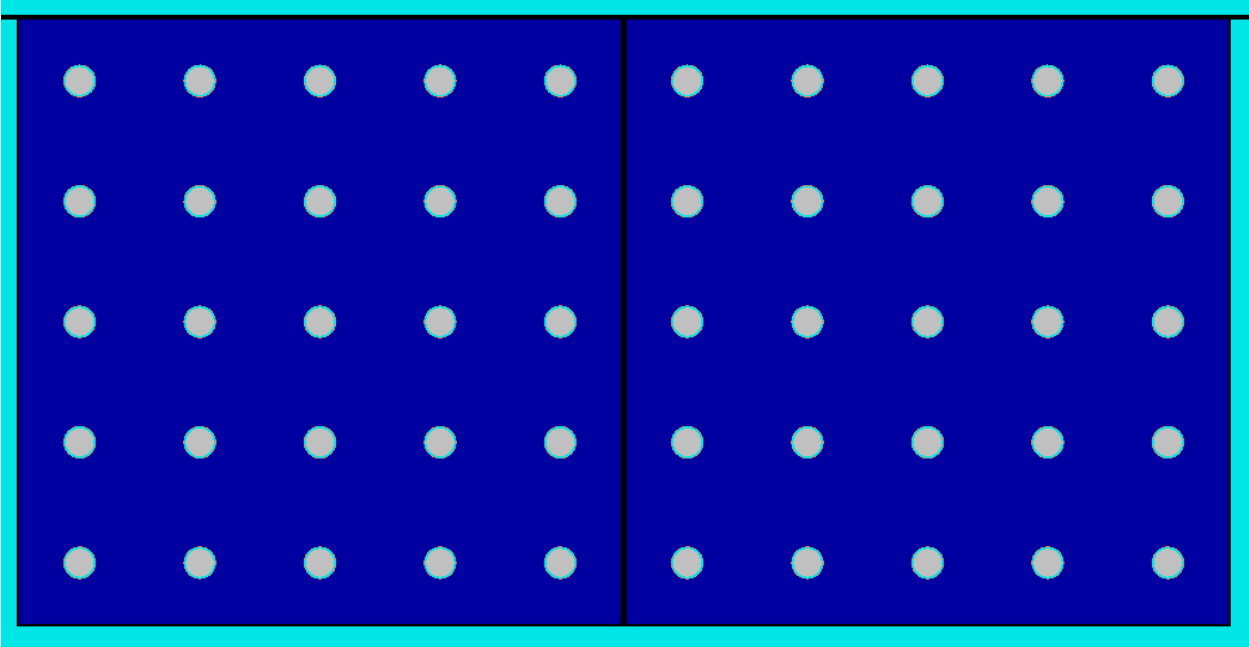
Top Axial Layer (Full-length Rods Only)  
 5 wt% (NCT, HAC single package)      3.3 wt% (HAC package array)



**Figure 6-52 Bounding Fuel Assembly Model**




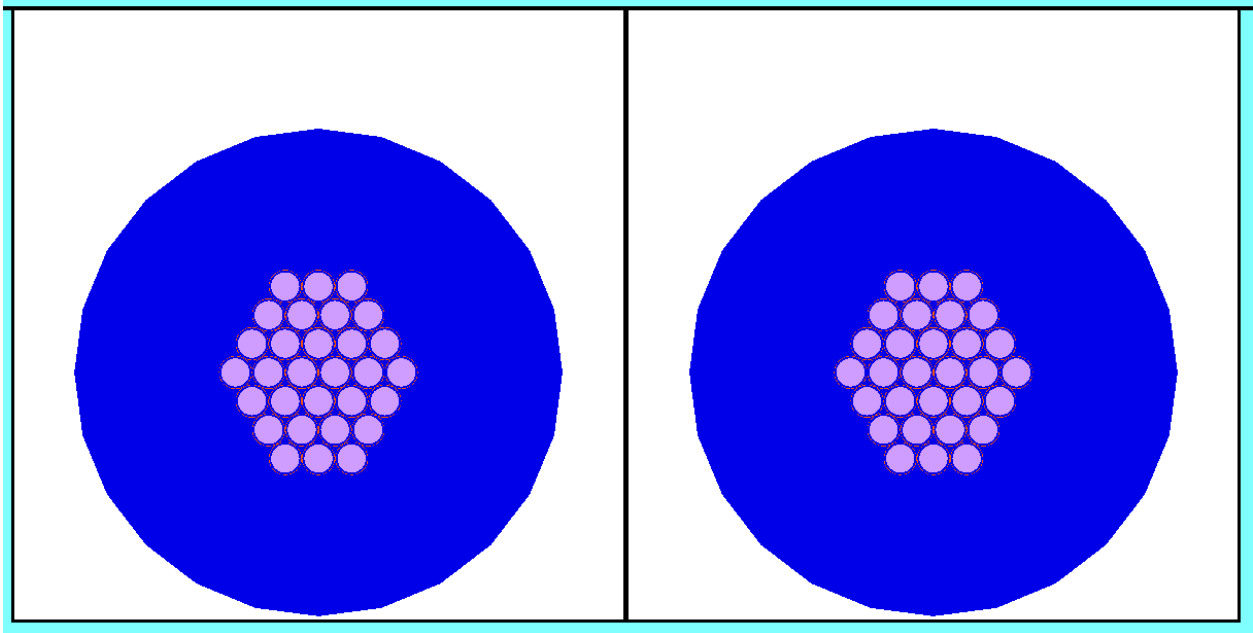
Close-packed loose fuel rods



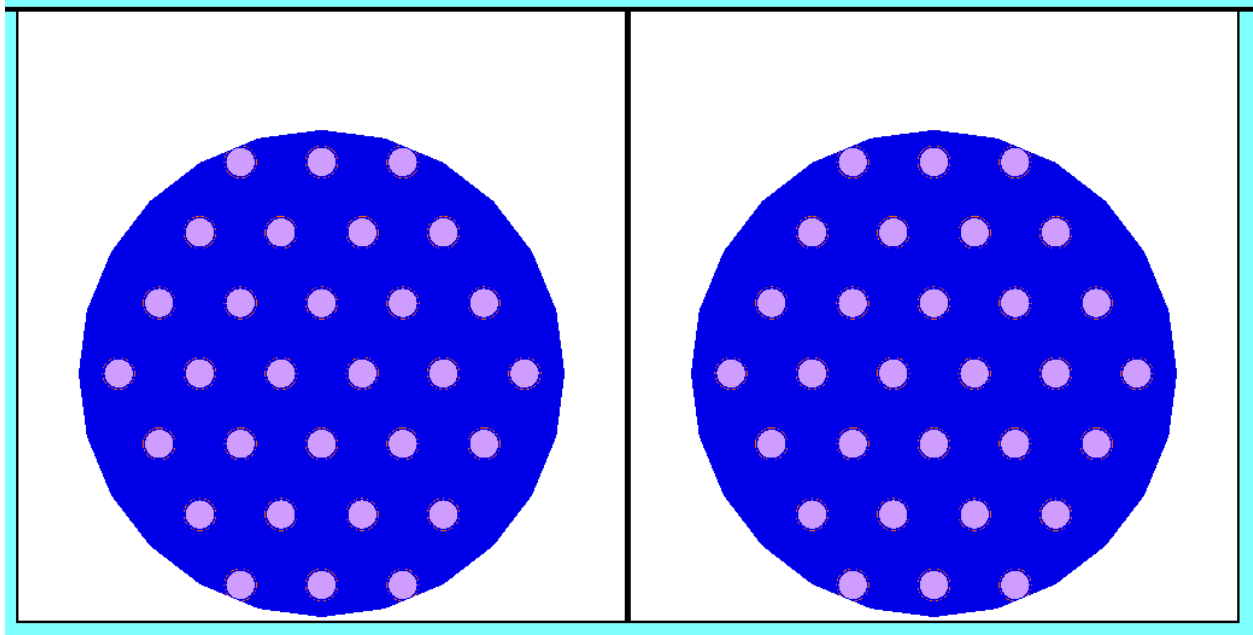
Maximum-pitch loose fuel rods

**Figure 6-53 Twenty-five 11x11 Fuel Rods in Liner**

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Close-packed fuel rods in pipe component



Maximum-pitch fuel rods in pipe component

**Figure 6-54 Thirty 11x11 Fuel Rods in Pipe Component**

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### 6.12.3.2. Material Properties

The material specifications for the 11x11 analysis are summarized in Table 6-60 Material Specifications for the TN-B1 11x11 Analysis. The atomic densities are extracted directly from the SCALE 6.1.3/KENO V.a output files. Because ENFD/B-VII cross sections are used, elements are represented as individual isotopes.

The UO<sub>2</sub> stack density is modeled as 10.763 g/cm<sup>3</sup>. The UO<sub>2</sub> material is included as material 1 in the sample problem inputs given in Section 6.12.10 *Sample Input Files*.

The 4 pcf polyethylene foam is conservatively modeled with a density of 5 pcf. Likewise, the 9 pcf polyethylene foam is conservatively modeled with a density of 10 pcf.

The presence of Gd<sub>2</sub>O<sub>3</sub> in the UO<sub>2</sub>-Gd<sub>2</sub>O<sub>3</sub> pellet reduces the density from 10.763 to 10.691 g/cm<sup>3</sup>. The UO<sub>2</sub>-Gd<sub>2</sub>O<sub>3</sub> is included as material 4 (see the sample problem input in Section 6.12.10.1). In the input, the UO<sub>2</sub>- Gd<sub>2</sub>O<sub>3</sub> is defined with two entries: UO<sub>2</sub> is defined as the first entry of material 4 with a density of 10.763 g/cm<sup>3</sup>. Gd<sub>2</sub>O<sub>3</sub> is defined as the second entry of material 4 with a density of 7.407 g/cm<sup>3</sup>. Using these densities and the volume fractions calculated below, the density of the UO<sub>2</sub>- Gd<sub>2</sub>O<sub>3</sub> material is calculated by SCALE 6.1.3 to be 10.691 g/cm<sup>3</sup>.

A 2 wt% gadolinia-urania rod is used in the analysis. However, only 75% of this is credited in the current analysis. Thus the gadolinium weight percent in the gadolinia-urania fuel is 1.5%. The fraction of the UO<sub>2</sub> and Gd<sub>2</sub>O<sub>3</sub> of the gadolinia fuel material is determined using the following equation:

$$wf_{Gd} = \frac{\rho_{Gd} * VF_{Gd}}{\rho_{UO2} * VF_{UO2} + \rho_{Gd} * VF_{Gd}}$$

where  $wf_{Gd}$  = weight fraction of Gd<sub>2</sub>O<sub>3</sub> in the gadolinia-urania fuel = 0.015

$\rho_{Gd}$  = density of Gd<sub>2</sub>O<sub>3</sub> = 7.407 g/cm<sup>3</sup>

$VF_{Gd}$  = volume fraction of Gd<sub>2</sub>O<sub>3</sub>

$\rho_{UO2}$  = density of UO<sub>2</sub> = 10.96 g/cm<sup>3</sup> \* 0.982 theoretical density = 10.763 g/cm<sup>3</sup>

$VF_{UO2}$  = volume fraction of UO<sub>2</sub> = 1 -  $VF_{Gd}$

Substituting (1- $VF_{Gd}$ ) for  $VF_{UO2}$  into the above equation and solving for  $VF_{Gd}$  gives

$$VF_{Gd} = \frac{wf_{Gd} * \rho_{UO2}}{\rho_{Gd} + wf_{Gd} * \rho_{UO2} - wf_{Gd} * \rho_{Gd}} = 0.0216$$

$$VF_{UO2} = 0.9784$$

### 6.12.3.3. Computer Codes and Cross-Section Libraries

The KENO V.a module of SCALE 6.1.3 is used in the analysis (CSAS5 sequence). The 238 group ENDF/B-VII cross section set is used in all input files. CENTRM is used for resonance processing. This cross section set and resonance processing methodology is used in the

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benchmark set described in Section 6.12.9, *Benchmark Evaluation for SCALE 6.1.3*. For each case, a minimum of 3 million active histories are run to ensure proper behavior about the mean value (e.g, 1,550 generations, 2,000 neutrons per generation, skipping the first 50 generations).

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**Table 6-60 Material Specifications for the TN-B1 11x11 Analysis**

Material	Density (g/cm <sup>3</sup> )	Constituent	Atomic Density (atoms/b-cm)
U(5.0)O <sub>2</sub>	10.763 <sup>a</sup>	O-16	4.80319E-02
		U-235	1.21538E-03
		U-238	2.28006E-02
U(5.0)O <sub>2</sub> -Gd <sub>2</sub> O <sub>3</sub> 2 wt% Gd <sub>2</sub> O <sub>3</sub> (75% credit for Gd)	10.691 <sup>b</sup>	O-16	4.77918E-02
		Gd-152	1.06321E-06
		Gd-154	1.15888E-05
		Gd-155	7.86763E-05
		Gd-156	1.08818E-04
		Gd-157	8.31949E-05
		Gd-158	1.32049E-04
		Gd-160	1.16207E-04
		U-235	1.18913E-03
		U-238	2.23081E-02
		Zr-90	2.20431E-02
		Zr-91	4.80708E-03
Zirconium	6.49	Zr-92	7.34772E-03
		Zr-94	7.44626E-03
		Zr-96	1.19963E-03
		H-1 (in poly)	6.86951E-03
		C	3.43476E-03
Polyethylene Foam (4 pcf)	≤ 0.08	H-1 (in poly)	1.37390E-02
		C	6.86951E-03
Polyethylene Foam (9 pcf)	≤ 0.16	H-1 (in poly)	8.14896E-02
		C	4.07448E-02
Polyethylene	0.949	O-16	4.72705E-03
		Al-27	1.44724E-03
Alumina Silicate [Al <sub>2</sub> O <sub>3</sub> (49%)- SiO <sub>2</sub> (51%)]	0.25	Si-28	1.17870E-03
		Si-29	5.98790E-05
		Si-30	3.95189E-05
		H-1 (in water)	6.68734E-02
Full Density Water	1.0	O-16	3.34367E-02

- a UO<sub>2</sub> is represented in the input file as material 1 with a density of 10.763 g/cm<sup>3</sup> (see sample input files in Section 6.12.10 *Sample Input Files*).
- b UO<sub>2</sub>-Gd<sub>2</sub>O<sub>3</sub> is represented in the input file as material 4 with two entries. Entry 1 of material 4 is UO<sub>2</sub> with a density 10.763 g/cm<sup>3</sup> and volume fraction of 0.9784 as calculated in Section 6.12.3.2. Entry 2 of material 4 is Gd<sub>2</sub>O<sub>3</sub> with a density of 7.407 g/cm<sup>3</sup> and a volume fraction of 0.0216. The density of the UO<sub>2</sub>-Gd<sub>2</sub>O<sub>3</sub> material is calculated by SCALE 6.1.3 to be 10.691 g/cm<sup>3</sup> using the input densities and volume fractions (see the sample input file in Section 6.12.10.1).



**Table 6-60 Material Specifications for the TN-B1 11x11 Analysis (continued)**

Material	Density (g/cm <sup>3</sup> )	Constituent	Atomic Density (atoms/b-cm)
Stainless Steel 304	7.94	C	3.18488E-04
		Si-28	1.57010E-03
		Si-29	7.97625E-05
		Si-30	5.26416E-05
		P-31	6.94688E-05
		Cr-50	7.59178E-04
		Cr-52	1.46402E-02
		Cr-53	1.65988E-03
		Cr-54	4.13224E-04
		Mn-55	1.74072E-03
		Fe-54	3.45419E-03
		Fe-56	5.36980E-02
		Fe-57	1.22946E-03
		Fe-58	1.63927E-04
		Ni-58	5.28417E-03
		Ni-60	2.02017E-03
		Ni-61	8.74631E-05
		Ni-62	2.77870E-04
		Ni-64	7.04349E-05

#### 6.12.3.4. Demonstration of Maximum Reactivity

##### 6.12.3.4.1. 11x11 Fuel Assembly Analysis

The most reactive condition occurs for the HAC array. Therefore, the following discussion refers to the parameters of the HAC array analysis for the 11x11 fuel assembly. A parameter study is performed for the 11x11 fuel assembly to determine the most reactive fuel assembly parameters. The results are:

- The zirconium water channel (i.e., the zirconium channel in the center of the fuel assembly) is modeled as water, which increases moderation and hence increases reactivity.
- The zirconium fuel channel (i.e., the zirconium channel on the outside of the fuel assembly) is modeled at the maximum thickness of 0.254 cm. It is demonstrated that when the fuel channel is modeled at the maximum thickness the system reactivity peaks with a 1.2 cm thick polyethylene liner. If the zirconium fuel channel is modeled as water, maximum reactivity occurs with a 1.5 cm thick polyethylene liner. The two results are statistically the same. Therefore, fuel may be shipped either with or without the fuel channel.
- The polyethylene equivalent mass for each fuel assembly is limited to 10.2 kg. This mass of polyethylene is smeared into the fuel cladding. Polyethylene is a superior moderator

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than water and increases the reactivity. For the final most reactive HAC configurations, a second modeling approach was considered in which the mass of polyethylene is smeared into the moderator.

- Pitch is increased 5% to 1.2548 cm. Increasing the pitch increases moderation and hence the reactivity. A structural analysis of the 11x11 fuel assembly has demonstrated that the average pitch expansion is a lower value of 2.6%, and the expansion of the fuel assembly outer perimeter is a lower value of 3.4% (Reference 19).
- Maximum pellet diameter of 0.820 cm. This is the largest allowed pellet diameter, as increasing the pellet diameter increases the reactivity.
- Maximum cladding inner diameter of 0.840 cm and minimum cladding outer diameter of 0.930 cm. These parameters control the thickness of the cladding, and reactivity increases as the cladding thickness decreases because replacing cladding with water increases moderation.
- While a range of enrichments and the number of gadolinia-urania rods are considered, the most reactive combination for uniform axial enrichment is 5.0% enriched fuel with 13 gadolinia-urania rods in each lattice in the pattern shown in Figure 6-52 Bounding Fuel Assembly Model. Numerous patterns are considered and the most reactive pattern is selected. Gadolinia-urania rods shall be placed symmetrically about the major diagonal and shall not be placed on the periphery. When axial variation of enrichment is considered, the reactivity of the HAC array configuration increases slightly if an enrichment of 5.0% with 13 gadolinia-urania rods is used in the bottom and middle lattices and an enrichment of 3.3% with 3 gadolinia-urania rods is used in the top lattice.
- The minimum  $Gd_2O_3$  loading in a Gadolinia-urania rod is 2.0%, and only 75% credit is taken for the gadolinia.
- The density of  $UO_2$  is modeled as  $10.763 \text{ g/cm}^3$ . Reactivity increases with increasing fuel loading.
- The lengths of the full-length, long partial-length, and short partial-length rods are conservatively increased by at least 4 cm to increase the fuel loading.
- Water is modeled in the pellet/cladding gap.

Other system parameters are also selected to maximize reactivity:

- Outer dimensions of the package reduced to account for HAC damage, which reduces the overall size of the array and increases reactivity.
- Inner container flooded with full-density water to maximize moderation.
- The region between the inner and outer containers is modeled as void to maximize neutron interactions between packages. Modeling water in this region reduces the reactivity.
- All rods of a fuel assembly are fully flooded to maximize moderation. Uncovering some fuel rods reduces the reactivity.

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- Thermal insulator modeled as Alumina-Silica. This bounds modeling this region as either void or water.
- Polyethylene foam liner modeled as partially burned with a thickness of 1.2 cm. Modeling the foam liner as partially burned is more reactive than either complete foam burn or no foam burn scenarios. In general, modeling some polyethylene foam increases neutron interactions between arrays of packages because the mean free path for a neutron in low-density foam is much higher than neutrons in full-density water.
- A fuel assembly orientation study demonstrates that the system is most reactive in a partial foam burn scenario with the fuel assemblies centered within the compartments with the fuel channel present.
- Trace amounts of plutonium are shown to have a negligible effect on reactivity and thus is neglected.

With the conservative modeling choices noted above,  $k_{\text{eff}} + 2\sigma$  is 0.94068, which is below the fuel assembly USL of 0.94094.

#### 6.12.3.4.2. **11x11 Fuel Rod Analysis**

The most reactive condition occurs for the HAC array. Therefore, the following discussion refers to the parameters of the HAC array analysis for 11x11 fuel rods.

Two scenarios are considered:

1. 25 fuel rods loose within each liner cavity. This analysis bounds fuel rods that are bundled together, as bundled fuel rods would have significantly less moderation than loose rods, and credit could not be taken for the bundles remaining intact in an accident.
2. 30 fuel rods within a 5-in stainless steel pipe. This analysis bounds fuel rods transported in a protective case, as the protective case has a much smaller cross sectional area than the pipe and would allow significantly less moderation.

In both analyses, conservative fuel rod parameters are used:

- A 6-mil thick polyethylene sleeve is modeled around each fuel rod. Polyethylene is a superior moderator than water and increases the reactivity.
- Maximum pellet diameter of 0.820 cm. This is the largest allowed pellet diameter, as increasing the pellet diameter increases the reactivity.
- The cladding is modeled as water to conservatively increase moderation. The pellet-cladding gap is also modeled as water.
- No gadolinia is credited.
- The density of  $\text{UO}_2$  is modeled as  $10.763 \text{ g/cm}^3$ . Reactivity increases with increasing fuel loading.

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- The lengths of the rods are conservatively modeled as 385 cm, which increases the fuel loading compared to partial-length rods.

In the loose rod analysis, 25 loose rods are modeled per compartment (50 rods per package). The rods are arranged in a 5x5 square array within each compartment.

- A pitch study is performed and optimum moderation is achieved. Therefore, reactivity will decrease for less than 25 rods.
- A moderation study is performed and the system is most reactive with 0.6 g/cm<sup>3</sup> moderator.
- A foam liner partial burn study is performed and the system is most reactive with 0.4 cm liner thickness.
- The most reactive case has  $k_{\text{eff}} + 2\sigma = 0.72725$ , which is significantly less than the rod USL of 0.94047.

In the 5-in pipe analysis, 30 rods are modeled per pipe (60 rods per package). The rods are arranged in a triangular pitch within the pipe.

- The wall of the pipe is conservatively modeled as water. A pitch study is performed in which the pitch is allowed to expand until the rods extend to the outer diameter of the pipe. This geometry is not physically possible because the rods would contact the inner diameter of the pipe before this level of moderation is achieved. The case with the maximum pitch expansion is the most reactive, indicating the system is likely undermoderated.
- A moderator density study indicates that the system is most reactive with full-density water within the pipe to maximize moderation.
- Moderation is increased by removing rods for the same pitch and reactivity decreased compared to modeling 30 rods in the pipe. Moderation is also increased by removing rods and increasing the pitch and reactivity decreased compared to modeling 30 rods in the pipe. Therefore, less than 30 rods per pipe is bounded by 30 rods per pipe.
- Several different orientations of the pipes within the package are investigated, along with partial foam burn. It is demonstrated that complete foam burn with the pipes shifted to the center of the package is the most reactive condition.
- The insulation is modeled as Alumina-Silica consistent with the 11x11 fuel assembly HAC models.
- The outer region is modeled as void consistent with the 11x11 fuel assembly HAC array models.
- The region outside the pipes but inside the inner container is modeled with a density of 0.1 g/cm<sup>3</sup>, which is slightly more reactive than modeling this region as void.
- The most reactive case has  $k_{\text{eff}} + 2\sigma = 0.82035$ , which is significantly less than the rod USL of 0.94047.

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#### 6.12.3.5. Parameter Selection for 11x11 Fuel Assembly Model

Prior to performing the criticality analysis for the 11x11 fuel assembly, the parameters of the fuel assembly are selected to determine the most reactive fuel assembly configuration. The following parameter studies are performed:

1. Fuel assembly orientation
2. Fuel assembly zirconium channel
3. Zirconium water channel
4. Polyethylene equivalent mass
5. Fuel rod pitch
6. Fuel pellet diameter
7. Fuel rod cladding thickness
8. Inner container partial flooding
9. Thermal insulator material
10. Polyethylene foam liner thickness (i.e., partial burn)
11. Number of gadolinia-urania rods as a function of enrichment.

The parameter selection model is based upon the 10x10 HAC array model because this model results in reactivities close to the USL. The single package and NCT array reactivities do not approach the USL. In the baseline HAC model used in this parameter study, the inner container is fully flooded while the region between the inner and outer containers is void. This moderation condition is used as the starting point of the analysis because it has been shown to be the most reactive moderation condition for 8x8, 9x9, and 10x10 fuel (see Section 6.6, *Package Arrays Under Hypothetic Accident Conditions*), and 11x11 fuel behaves in a similar manner.

The parameters listed above are optimized using an HAC array model and are applicable for NCT single package, HAC single package, and NCT array analyses with the exception of fuel assembly orientation, fuel assembly zirconium channel, and polyethylene foam liner thickness. These parameters are re-evaluated for NCT single package, HAC single package, and NCT array analyses because the system response for these parameters may be different than an HAC array.

Basic bounding data for the 11x11 fuel assembly is summarized in Table 6-61 11x11 Fuel Assembly Data. A simple diagram of the assembly showing the location of the fuel rods is shown in Figure 6-51 Position of Fuel Rods in 11x11 Assembly.

The maximum pellet enrichment and maximum fuel lattice average enrichment is 5 wt% U-235. A density of 10.763 g/cm<sup>3</sup> is used in the analysis.

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#### 6.12.3.5.1. ***Fuel Assembly Orientation Study***

The 10x1x10 HAC array model described in Section 6.12.3.1.5, *HAC Array Model with 11x11 Fuel*, is used to perform initial calculations to find the worst case fuel assembly orientation inside each TN-B1 fuel compartment. The base model features 5 wt.% UO<sub>2</sub> rods (no gadolinium), nominal fuel dimensions, and no fuel channel or water channel. 10.2 kg of polyethylene per fuel assembly is modeled as smeared into the fuel rod cladding to increase moderation. The nominal pitch of 1.195 cm is modeled. The polyethylene foam liner is assumed to have burned away in this initial model, which allows a greater degree of fuel orientation scenarios (the fuel must be centered in the compartment if the foam does not burn). All analyzed orientations are shown in Figure 6-55 Fuel Assembly Orientation 1 through Figure 6-66 Fuel Assembly Orientation 12.

The results of the calculations are provided in Table 6-62 11x11 Fuel Assembly Orientation Results. Based on these results, assembly orientation 7 is bounding. Orientation 7 has one assembly centered in the fuel compartment and one assembly shifted toward the center of the container. Orientation 7 is used in the subsequent studies in which the foam is assumed to have completely burned away. When partial foam is present (partial foam burn), it is later demonstrated that centered fuel assemblies are more reactive.

It is noted that many of the results in Table 6-62 11x11 Fuel Assembly Orientation Results exceed the 0.94094 USL because no gadolinia-urania rods are modeled. For this reason, 12 gadolinia-urania fuel rods are added to each lattice (i.e., bottom, middle, top) in a reasonably conservative pattern to provide reactivity hold-down in subsequent parametric models. This initial gadolinia-urania loading pattern is preliminary. The final gadolinia-urania loading pattern is explicitly derived at the end of the parametric study as a function of lattice enrichment.

#### 6.12.3.5.2. ***Fuel Assembly Zirconium Channel Study***

The 11x11 fuel assembly may have a zirconium channel around the outer boundary of the fuel assembly. The channel thickness is  $\leq 0.254$  cm. A range of channel thicknesses from 0 cm (no channel) to the maximum value of 0.254 cm are investigated to determine the most reactive condition.

The fuel assembly channel is located in the reflector region for each assembly. It has no effect on the assembly H/U-235 ratio since it is not located within the fuel envelope. Calculations are performed both with and without a water gap between the fuel rods and the channel.

Since the orientation used in the current analysis (Orientation 7) has the assembly in the right compartment shifted left to the center of the shipping container, it is necessary to shift the assembly further to the right to accommodate the presence of the channel (and water gap if present).

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The results are shown in Table 6-63 11x11 Zirconium Channel Study Results. The results indicate that reactivity increases with the presence of channels due to increased leakage from the inner fuel compartment, resulting in increased neutron interaction among containers in the array. In addition, reactivity increases with increasing thickness of the channel. Furthermore, including a water gap between the fuel pin cells and the channel has no statistically significant effect on reactivity. Therefore, the model using the thickest channel of 0.254 cm with no gap between the pin cells and the channel is chosen for further study. Note that additional channel studies are performed when the polyethylene foam liner is considered.

#### 6.12.3.5.3. ***Zirconium Water Channel Study***


The limiting channeled model discussed in the previous section is used to determine if the presence of the zirconium water channel within the fuel assembly increases system reactivity. A zirconium channel is placed within the center 3x3 water hole. The channel is modeled with an inner width and thickness as described in Reference 20. The result for including a water channel within the 11x11 assembly is listed in Table 6-64 11x11 Zirconium Water Channel Study Results. By including the zirconium water channel, system reactivity is decreased; therefore, it is conservative to neglect the zirconium water channel within the TN-B1 analysis.

#### 6.12.3.5.4. ***Polyethylene Mass Study***

The effect that polyethylene mass has on reactivity is considered. Plastic inserts may be used during transportation, which are inserted into the fuel assembly. Other plastic materials may also be used, such as spacer straps. A polyethylene equivalent mass limit of 10.2 kg is set for the 11x11 fuel assembly. To verify that this limit is sufficiently conservative for the 11x11 assembly, a polyethylene mass study is performed in which the polyethylene mass is varied from 0 kg per assembly to 10.2 kg per assembly. This study uses the limiting channeled model discussed in Section 6.12.3.5.2, *Fuel Assembly Zirconium Channel Study*, as the base case. The polyethylene mass is smeared into the fuel cladding for all fuel rods, which simulates the polyethylene melted onto the fuel rods. The volume fractions as a function of polyethylene mass are provided in Table 6-65 11x11 Polyethylene Mass and Volume Fraction Calculations.

The results of the polyethylene mass study are provided in Table 6-66 11x11 Polyethylene Mass Sensitivity Analysis Results. This shows that the value of 10.2 kg/assembly bounds the results given for lower polyethylene mass values.



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#### 6.12.3.5.5. ***Fuel Rod Pitch Study***

A fuel rod pitch sensitivity study is conducted using the limiting channeled model. The minimum fuel rod pitch is chosen to be at the point that the polyethylene coating on adjacent fuel rods contact. The maximum fuel rod pitch is chosen to be 5% greater than the nominal pitch. A structural analysis of the 11x11 fuel assembly has demonstrated that the average pitch expansion is a lower value of 2.6%, and the expansion of the fuel assembly outer perimeter is a lower value of 3.4% (Reference 19).

The results of the fuel rod pitch study are provided in Table 6-67 11x11 Fuel Rod Pitch Sensitivity Analysis Results. The results show that the fuel assemblies are under-moderated such that increasing the rod pitch increases system reactivity from minimum pitch to the maximum of 5% increase.

#### 6.12.3.5.6. ***Fuel Pellet Diameter Study***

A fuel pellet diameter sensitivity study is conducted. Using the limiting channeled model, the diameter of the fuel pellet is varied from 0.780 cm to 0.820 cm. The results are provided in Table 6-68 11x11 Pellet Diameter Sensitivity Analysis Results. The results show that reactivity increases as the pellet diameter is increased. A pellet diameter of 0.820 cm is selected as the upper bound for the 11x11 fuel assembly pellet range.

#### 6.12.3.5.7. ***Fuel Rod Cladding Thickness Study***

Two sets of calculations are performed to assess the reactivity sensitivity to changes in cladding thickness. For both sets of calculations the limiting channeled model is used.

For the first set of calculations, the inner cladding diameter is adjusted to determine the effect on reactivity while the outer cladding diameter is fixed at 0.940 cm. The minimum value for the parameter search range is the pellet outer diameter, while the maximum value for the range is the cladding outer diameter. The volume fractions of the zircaloy and polyethylene cladding material are calculated with the cladding/polyethylene outer diameter remaining fixed to correspond to the cladding thickness being considered. The volume fractions for the calculations varying the clad inner diameter are given in Table 6-69 11x11 Zirc and Polyethylene Volume Fractions, Varying Cladding ID. Results are provided in Table 6-70 11x11 Cladding ID Sensitivity Analysis Results.

The second set of calculations involves adjustments to the outer cladding diameter while the inner cladding diameter is held at 0.826 cm. The range of cladding outer diameters studied is between 0.826 cm (cladding inner diameter) and 1.0386 cm (maximum outer diameter for nominal rod pitch and a polyethylene limit of 10.2 kg/assembly). The cladding/polyethylene outer diameter and the volume fractions of the zircaloy and polyethylene cladding material are calculated with the cladding inner diameter remaining fixed to correspond to the cladding

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thickness being considered. The volume fractions for the calculations varying the clad outer diameter are given in Table 6-71 11x11 Zirc and Polyethylene Volume Fractions, Varying Cladding OD. Results are provided in Table 6-72 11x11 Cladding OD Sensitivity Analysis Results.

Both sets of results demonstrate that a decrease in the cladding thickness results in an increase in system reactivity as cladding is replaced by water. Based on these results and future design considerations, 0.840 cm (maximum) and 0.930 cm (minimum) are selected for the cladding inner diameter and cladding outer diameter, respectively, or a minimum cladding thickness of 0.045 cm. For these dimensions and 10.2 kg polyethylene,  $VF_{\text{clad}} = 0.31309$  and  $VF_{\text{poly}} = 0.68691$ .

#### 6.12.3.5.8. **Inner Container Partial Flooding Study**

The fuel assembly parameters utilized in the inner container partial flooding study (and subsequent studies) are consistent with the conclusions of the previous sections:

- Assembly orientation 7 (for full foam burn)
- 0.254 cm thick fuel assembly channel included
- Zirconium water channel not included
- 10.2 kg polyethylene per fuel assembly
- Pitch increased 5% to 1.2548 cm
- Pellet diameter of 0.820 cm
- Cladding inner diameter of 0.840 cm
- Cladding outer diameter of 0.930 cm

The fuel bundle rows are partially filled within the TN-B1 inner fuel compartment, as shown in Figure 6-67 Inner Container Partial Flooding. For this study, the polyethylene foam is modeled as water within the inner compartment (i.e, complete foam burn). The conclusions of the study remain valid if polyethylene material is used in this region.

The results of the partial flooding study are provided in Table 6-73 11x11 Sensitivity Analysis Results for Partially Flooded Inner Container. The results show that the most reactive conditions exist when water fully covers each fuel bundle. Therefore, the inner container fuel compartment will be fully flooded with water in the worst case TN-B1 model.

#### 6.12.3.5.9. **Thermal Insulator Material Study**

The most reactive case from Table 6-73 11x11 Sensitivity Analysis Results for Partially Flooded Inner Container is used to conduct a study to determine if the system is more reactive with or

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without the thermal insulating material. The material normally present around the inner container fuel compartment is a thermal insulator consisting of Alumina Silicate.

The Alumina Silicate insulator is replaced with full density water or void. The inner container fuel compartment for both scenarios is filled with full density water. Results for the thermal insulator sensitivity study are given in Table 6-74 11x11 Sensitivity Analysis Results for Thermal Insulator Material and show that the most reactive condition is achieved with the Alumina Silicate thermal insulator in place. Therefore, the Alumina Silicate thermal insulator will remain a part of the worst case TN-B1 model. This is consistent with the physical condition of the TN-B1 shipping container after being subjected to the tests specified in 10 CFR Part 71 (see Sections 3.2.2 and 3.5.2)

#### 6.12.3.5.10. ***Polyethylene Foam Liner Study***

In the previous cases in this section, the polyethylene foam liner is assumed to burn away in the fire. In this study, incomplete foam burn is modeled to determine the effect on the reactivity.

The material normally lining the inner container fuel compartment is a polyethylene foam material that has a density of 4 pcf on the top and sides. The 11x11 fuel assembly is supported at the grid spacers using small strips of 9 pcf foam, although the majority of the bottom foam is only 4 pcf. Because sensitivity studies show that reactivity increases slightly with increasing foam density, 5 pcf foam (0.08 g/cm<sup>3</sup>) is modeled on the top and sides and 10 pcf foam (0.16 g/cm<sup>3</sup>) is modeled on the bottom.

To determine the effect on reactivity from the liner, the thickness of the foam is varied within the container from no liner to a liner that fills the container from the walls to the fuel assembly. The presence of the liner affects the most reactive assembly orientation determined in Section 6.12.3.5.1, *Fuel Assembly Orientation Study*, because for a full liner thickness (i.e., no foam burn) the fuel assemblies must be centered. The presence or absence of the channel also affects the fuel assembly orientation, as well as the system response. For these reasons, four different scenarios are considered:

1. Fuel assemblies centered, fuel channel included
2. Fuel assemblies centered, no fuel channel
3. Fuel assembly in left compartment centered, fuel assembly in right compartment shifted to the left, fuel channel included.
4. Fuel assembly in left compartment centered, fuel assembly in right compartment shifted to the left, no fuel channel.

For the channeled fuel, the maximum thickness of the liner is from the container wall to the fuel assembly channel. For the unchanneled fuel, the maximum thickness of the liner is from the container wall to the fuel assembly envelope.

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In the models with the fuel channel in the right compartment shifted to the left, the fuel assembly is placed directly next to the foam liner on the center wall between compartments. The assembly is shifted towards the center of the right compartment as the liner thickness is increased. Including the fuel channel also increases the distance between the fuel assemblies when the right fuel assembly is shifted to the left.

Results of the polyethylene liner sensitivity studies are given in Table 6-75 11x11 Sensitivity Analysis Results for Polyethylene Liner. Several observations may be made from these results:

1. Modeling partial foam burn is more reactive than either no foam burn or complete foam burn. In fact, the increase in reactivity is such that the USL is violated. The placement and number of gadolinia-urania rods is determined in the next section for a variety of U-235 enrichments such that reactivity will remain below the USL of 0.94094.
2. Reactivity increases as the thickness of the liner increases until the liner fills ~75% of the compartment.
3. For thinner polyethylene liners, the presence of the fuel channel results in a higher system reactivity than using unchanneled fuel. For thicker liners, the unchanneled fuel results in higher system reactivity.
4. In general, for partial foam burn scenarios, shifting the right fuel assembly toward the center of the container results in lower system reactivity than having both fuel assemblies centered within their respective compartments.

The highest system reactivity is observed for a centered, channeled configuration with a 1.2-cm-thick polyethylene liner. This configuration will be used for further calculations, including the gadolinia-urania rod analysis versus U-235 enrichment. It is noted that the maximum reactivity for unchanneled assemblies is observed in the centered orientation with a 1.5-cm-thick polyethylene liner. The reactivity of these systems is statistically the same.

Finally, as mentioned above, the 9 pcf foam is intended to be located in discrete locations only and not a continuous liner. Thus a calculation replacing the 9 pcf bottom foam liner with 4 pcf foam is performed to show that modeling 9 pcf foam is more reactive. This calculation is performed for the most limiting case in the above analyses:


9 pcf foam liner (poly\_liner\_120\_cc.out):  $k_{\text{eff}} + 2\sigma = 0.95276$

Replacing 9 pcf foam with 4 pcf foam (poly\_liner\_120\_DENS.out):  $k_{\text{eff}} + 2\sigma = 0.95170$

Modeling the bottom as 9 pcf foam is slightly more reactive than modeling the bottom as 4 pcf foam, although the effect is small and within the uncertainty of the Monte Carlo method. For this reason, using small 9 pcf foam strips with FANP 10x10 fuel assemblies is also acceptable.

#### 6.12.3.5.11. **Gadolinia-Urania Fuel Rod Restriction Study**

The parameter studies described in the preceding sections are performed using a 12 gadolinia-urania rods in a preliminary loading pattern. As the results in the polyethylene foam liner study

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exceed the USL, the number of gadolinia-urania rods is increased and the loading pattern is modified to lower the system reactivity.

As a result of the parameter studies performed in Sections 6.12.3.5.1 through 6.12.3.5.10, the most reactive system parameters for the 10x10 HAC array are:


- Assembly orientation 6 (fuel assemblies centered for partial foam burn)
- 0.254 cm thick fuel assembly channel included
- Zirconium water channel not included
- 10.2 kg polyethylene per fuel assembly smeared into the cladding (note that for the final most reactive HAC configurations, a second modeling approach was considered in which the polyethylene per fuel assembly was smeared into the moderator)
- Pitch increased 5% to 1.2548 cm
- Pellet diameter of 0.820 cm
- Cladding inner diameter of 0.840 cm
- Cladding outer diameter of 0.930 cm
- Fuel assemblies fully flooded (i.e., no uncovered fuel rods)
- Thermal insulator modeled as Alumina-Silica
- Polyethylene foam liner modeled as partially burned with a thickness of 1.2 cm

A 2 wt% gadolinia rod is used in the analysis. However, only 75% of this is credited in the current analysis. Thus the gadolinia weight percent in the gadolinia-urania fuel is 1.5%. See Section 6.12.3.2, *Material Properties*.

The 11x11 assembly has three distinct axial layers, corresponding to the lengths of the different types of fuel rods. The three axial regions are defined as follows:

- Bottom: This layer includes all full-length and all partial-length rods. No water holes from partial-length rods are present.
- Middle: This layer includes the full-length rods and the long partial-length rods. 12 water holes are located in the short partial-length rod locations.
- Top: This layer includes only full-length rods. 12 water holes are located in the short partial-length rod locations and 8 water holes are located in the long partial-length rod locations.

While the minimum required number of gadolinia-urania fuel rods is constant in each axial region, the gadolinia-urania fuel rod placement patterns may vary between each of the three axial regions. The approach is to initially model each of the axial regions as a full-length fuel assembly (i.e., 385 cm long) and vary the gadolinia-urania fuel rod placement in patterns expected to maximize reactivity. These initial models are referred to as “2-D” models because

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the geometry across each axial region of the fuel assembly is modeled over the full fuel assembly length. In general, reactivity is maximized when the gadolinia-urania fuel rods are clustered together, which reduces their worth due to self-shielding. In these initial “2-D” models, the reactivities should not be compared against the USL, as the intent is only to observe trends in the reactivities from the various loading patterns rather than absolute reactivities.

Two restrictions on the gadolinia-urania fuel rod placement patterns are stipulated:

1. Gadolinia-urania rods shall be distributed symmetrically about the major diagonal, and.
2. Gadolinia-urania rods shall not be placed on the periphery.

After the gadolinia-urania fuel rod placement patterns are independently determined for each of the three axial layers using the “2-D” models, final models are developed in which the 11x11 fuel assembly is modeled explicitly with each optimized axial layer to demonstrate the reactivity is below the USL.

The number of required gadolinia-urania rods and the loading pattern changes with enrichment. The following combinations of enrichment and number of gadolinia-urania rods are demonstrated to be acceptable:

1. 5.0 wt.%, 13 gadolinia-urania rods
2. 4.8 wt.%, 12 gadolinia-urania rods
3. 4.6 wt.%, 11 gadolinia-urania rods
4. 4.4 wt.%, 10 gadolinia-urania rods
5. 4.2 wt.%, 9 gadolinia-urania rods
6. 4.1 wt.%, 8 gadolinia-urania rods
7. 3.9 wt.%, 7 gadolinia-urania rods
8. 3.8 wt.%, 6 gadolinia-urania rods
9. 3.6 wt.%, 5 gadolinia-urania rods
10. 3.5 wt.%, 4 gadolinia-urania rods
11. 3.3 wt.%, 3 gadolinia-urania rods
12. 3.2 wt.%, 2 gadolinia-urania rods
13. 2.9 wt.%, 0 gadolinia-urania rods

Numerous gadolinia-urania rod loading patterns are investigated for each enrichment. The three most reactive loading patterns for each enrichment are illustrated on Figure 6-68 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 5.0 wt% <sup>235</sup>U, 13 Gd Rods through Figure 6-79 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.2 wt% <sup>235</sup>U, 2 Gd Rods. For brevity, all loading patterns considered are not provided.

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The “2-D” results corresponding to the loading patterns described in the previous paragraph are provided for each enrichment in Table 6-76 11x11 Bottom Layer “2-D” Analysis Results, Table 6-77 11x11 Middle Layer “2-D” Analysis Results, and Table 6-78 11x11 Top Layer “2-D” Analysis Results.


The most reactive loading patterns determined using the “2-D” models are combined in a full 3-D model of the 11x11 fuel assembly with each axial layer modeled explicitly. The final results for each enrichment are provided in Table 6-79 11x11 Gadolinia-Urania Analysis Results. The most reactive case for each enrichment is highlighted in boldface.

Because the number of gadolinia-urania rods are selected for each enrichment for the reactivity to be close to the USL, the maximum reactivities for each enrichment are necessarily similar. The maximum  $k + 2\sigma$  is 0.93810 and occurs for the 5.0% enrichment with 13 gadolinia-urania rods. The minimum  $k + 2\sigma$  is 0.93404 and occurs for 3.2% enrichment and 2 gadolinia-urania rods. Although all of the enrichment/gadolinia-urania rod configurations have similar reactivities, the 5.0% enrichment assembly with 13 gadolinia-urania rods per lattice is used as the bounding fuel assembly configuration because it is slightly more reactive than the other configurations.

The most reactive results from Table 6-79 11x11 Gadolinia-Urania Analysis Results are also summarized in Table 6-1 TN-B1 Fuel Assembly Loading Criteria.

It is recognized that each axial region may contain a different U-235 enrichment. To address this in a bounding configuration, the most reactive “2-D” configuration for each axial region, regardless of enrichment, was combined to form a 3-D model. This model contained 5 wt% enrichment with 13 gadolinia-urania rods in the bottom and middle axial regions. The upper axial region contained 3.3 wt% enrichment with 3 gadolinia-urania rods. The result of this model is also provided in Table 6-79 11x11 Gadolinia-Urania Analysis Results.




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**Table 6-61 11x11 Fuel Assembly Data**

Parameter	Units	Value
Fuel Assembly Type		11x11
UO <sub>2</sub> Density <sup>b</sup>	g/cm <sup>3</sup>	≤ 10.763
Number of water rods	#	3x3 center
Number of fuel rods	#	112
Fuel Rod OD	cm	≥ 0.930
Fuel Pellet OD	cm	≤ 0.820
Cladding Type		Zirconium Alloy
Cladding ID	cm	≤ 0.840
Cladding Thickness	cm	≥ 0.045
Fuel Rod Pitch <sup>a</sup>	cm	≤ 1.195
U-235 Pellet Enrichment	wt%	≤ 5.0
Maximum Lattice Average Enrichment	wt%	≤ 5.0
Fuel Channel Side Thickness	cm	≤ 0.254
Full Length Fuel Rods		
Quantity	#	92
Active length	cm	≤ 385
Short Part Length Fuel Rods		
Quantity	#	12
Active length	cm	≤ 155.1
Long Part Length Fuel Rods		
Quantity	#	8
Active length	cm	≤ 236.8

a. Equivalent nominal pitch per Section 6.12.3.1.1.

b. Density based on a pellet modeled as a right cylinder.

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**Table 6-62 11x11 Fuel Assembly Orientation Results**


Filename	Assembly Orientation	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
atrium-11_orient01.out	1	0.94507	0.00052	0.94611
atrium-11_orient02.out	2	0.94906	0.00044	0.94994
atrium-11_orient03.out	3	0.89695	0.00054	0.89803
atrium-11_orient04.out	4	0.89730	0.00046	0.89822
atrium-11_orient05.out	5	0.91769	0.00045	0.91859
atrium-11_orient06.out	6	0.96649	0.00048	0.96745
<b>atrium-11_orient07.out</b>	<b>7</b>	<b>0.98108</b>	<b>0.00051</b>	<b>0.98210</b>
atrium-11_orient08.out	8	0.94141	0.00046	0.94233
atrium-11_orient09.out	9	0.93333	0.00046	0.93425
atrium-11_orient10.out	10	0.97212	0.00047	0.97306
atrium-11_orient11.out	11	0.95334	0.00047	0.95428
atrium-11_orient12.out	12	0.89655	0.00044	0.89743

**Table 6-63 11x11 Zirconium Channel Study Results**

Filename	Channel Thickness (cm)	Water gap between fuel cells and channel?	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
atrium-11_5wt_12gd.out	0	No	0.88494	0.00052	0.88598
channel_1100.out	0.110	No	0.88585	0.00052	0.88689
channel_1300.out	0.130	No	0.88759	0.00047	0.88853
channel_1600.out	0.160	No	0.88784	0.00048	0.88880
channel_1700.out	0.170	No	0.88769	0.00046	0.88861
channel_1800.out	0.180	No	0.88824	0.00048	0.88920
channel_2000.out	0.200	No	0.88741	0.00044	0.88829
<b>channel_2540.out</b>	<b>0.254</b>	<b>No</b>	<b>0.88852</b>	<b>0.00044</b>	<b>0.88940</b>
channel_1300_h2ogap.out	0.130	Yes	0.88756	0.00044	0.88844
channel_1800_h2ogap.out	0.180	Yes	0.88804	0.00047	0.88898
channel_2540_h2ogap.out	0.254	Yes	0.88845	0.00047	0.88939

**Table 6-64 11x11 Zirconium Water Channel Study Results**

Filename	Water channel included?	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
<b>channel_2540.out</b>	<b>No</b>	<b>0.88852</b>	<b>0.00044</b>	<b>0.88940</b>
water_channel.out	Yes	0.88691	0.00045	0.88781

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**Table 6-65 11x11 Polyethylene Mass and Volume Fraction Calculations**

Clad IR, cm	0.4130	Clad OR, cm	0.4700
Number of full length fuel rods	92	Length, cm	385.0
Number of long partial length fuel rods	8	Length, cm	236.8
Number of short partial length fuel rods	12	Length, cm	155.1

Poly Mass (kg/assembly)	Clad/Poly Radius (cm)	Thickness (cm)	VF <sub>clad</sub>	VF <sub>poly</sub>
0	0.4700	0	1.00000	0.00000
2.5	0.4923	0.0223	0.70108	0.29892
5.0	0.5136	0.0436	0.53994	0.46006
6.0	0.5218	0.0518	0.49487	0.50513
7.0	0.5300	0.0600	0.45618	0.54382
8.0	0.5380	0.0680	0.42339	0.57661
8.5	0.5419	0.0719	0.40891	0.59109
9.0	0.5459	0.0759	0.39495	0.60505
10.2	0.5552	0.0852	0.36557	0.63443

The following example is for a total polyethylene mass of 10.2 kg/assembly.

Total Polyethylene Volume = [(Total Fuel Rod Number) \* (Polyethylene Area) \* (Fuel Rod Length)]

Full length: Volume =  $92\text{rods} * \{\pi[(0.5552\text{cm})^2 - (0.4700\text{cm})^2]\} * 385\text{cm} = 9719.6 \text{ cm}^3$

Long Partial length: Volume =  $8\text{rods} * \{\pi[(0.5552\text{cm})^2 - (0.4700\text{cm})^2]\} * 236.8 = 519.8 \text{ cm}^3$

Short Partial length: Volume =  $12\text{rods} * \{\pi[(0.5552\text{cm})^2 - (0.4700\text{cm})^2]\} * 155.1 = 510.7 \text{ cm}^3$

Total: Volume =  $9719.6 + 519.8 + 510.7 = 10750.1 \text{ cm}^3$

Total Polyethylene Mass = (Total Polyethylene Volume) \* (Polyethylene Density), where density of polyethylene equivalent =  $0.949 \text{ g/cm}^3$ .

Total: Mass =  $(10750 \text{ cm}^3) * 0.949 \text{ g/cm}^3 = 10202 \text{ g}$

Total Cladding Volume = [(Total Fuel Rod Number) \* (Cladding Area) \* (Fuel Rod Length)]

Full length: Volume<sub>clad</sub> =  $92\text{rods} * \{\pi[(0.4700\text{cm})^2 - (0.4130\text{cm})^2]\} * 385\text{cm} = 5600.6 \text{ cm}^3$

Long Partial length: Volume<sub>clad</sub> =  $8\text{rods} * \{\pi[(0.4700\text{cm})^2 - (0.4130\text{cm})^2]\} * 236.8\text{cm} = 299.5 \text{ cm}^3$

Short Partial length: Volume<sub>clad</sub> =  $12\text{rods} * \{\pi[(0.4700\text{cm})^2 - (0.4130\text{cm})^2]\} * 155.1\text{cm} = 294.3 \text{ cm}^3$


Total: Volume<sub>clad</sub> =  $5600.6 \text{ cm}^3 + 299.5 \text{ cm}^3 + 294.3 \text{ cm}^3 = 6194.4 \text{ cm}^3$

Cladding Volume Fraction = Cladding Volume / (Cladding Volume + Polyethylene Volume)

Total: VF<sub>clad</sub> =  $6194.4\text{cm}^3 / (6194.4\text{cm}^3 + 10750.1\text{cm}^3) = 0.36557$

Polyethylene Volume Fraction = Polyethylene Volume / (Cladding Volume + Polyethylene Volume)

Total: VF<sub>poly</sub> =  $10750.1\text{cm}^3 / (6194.4\text{cm}^3 + 10750.1\text{cm}^3) = 0.63443$

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**Table 6-66 11x11 Polyethylene Mass Sensitivity Analysis Results**

Filename	Polyethylene Mass (kg/assy)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
poly_mass_000.out	0	0.88276	0.00048	0.88372
poly_mass_025.out	2.5	0.88425	0.00044	0.88513
poly_mass_050.out	5.0	0.88581	0.00046	0.88673
poly_mass_055.out	5.5	0.88496	0.00045	0.88586
poly_mass_060.out	6.0	0.88630	0.00044	0.88718
poly_mass_070.out	7.0	0.88715	0.00047	0.88809
poly_mass_080.out	8.0	0.88731	0.00046	0.88823
poly_mass_085.out	8.5	0.88718	0.00045	0.88808
poly_mass_090.out	9.0	0.88790	0.00048	0.88886
<b>channel_2540.out</b>	<b>10.2</b>	<b>0.88852</b>	<b>0.00044</b>	<b>0.88940</b>

**Table 6-67 11x11 Fuel Rod Pitch Sensitivity Analysis Results**

Filename	Change in pin pitch (%)	Pin Pitch (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
ppitch_min.out	-6.9	1.1130	0.83959	0.00050	0.84059
channel_2540.out	0	1.1950	0.88852	0.00044	0.88940
ppitch_041%.out	4.1	1.2440	0.91450	0.00046	0.91542
<b>ppitch_050%.out</b>	<b>5.0</b>	<b>1.2548</b>	<b>0.91996</b>	<b>0.00043</b>	<b>0.92082</b>

**Table 6-68 11x11 Pellet Diameter Sensitivity Analysis Results**

Filename	Pellet diameter (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
prad_7800.out	0.7800	0.88441	0.00049	0.88539
prad_8000.out	0.8000	0.88746	0.00048	0.88842
prad_8098.out	0.8098	0.88859	0.00052	0.88963
channel_2540.out	0.8110	0.88852	0.00044	0.88940
prad_8122.out	0.8122	0.88934	0.00043	0.89020
prad_8160.out	0.8160	0.88828	0.00044	0.88916
prad_8180.out	0.8180	0.88917	0.00047	0.89011
<b>prad_8200.out</b>	<b>0.8200</b>	<b>0.88952</b>	<b>0.00048</b>	<b>0.89048</b>


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**Table 6-69 11x11 Zirc and Polyethylene Volume Fractions, Varying Cladding ID**

Fuel assembly parameters				
Cladding/Polyethylene OD, cm	1.1104		Clad OD, cm	0.9400
Number of full length fuel rods	92		Length, cm	385.0
Number of long partial length fuel rods	8		Length, cm	236.8
Number of short partial length fuel rods	12		Length, cm	155.1
Volume Fraction Calculations				
Clad inner diameter (cm)	Clad thickness (cm)	VF <sub>clad</sub>		VF <sub>poly</sub>
0.8110	0.0645	0.39265		0.60735
0.8140	0.0630	0.38746		0.61254
0.8220	0.0590	0.37307		0.62693
0.8260	0.0570	0.36557		0.63443
0.8300	0.0550	0.35785		0.64215
0.8400	0.0500	0.33751		0.66249
0.8500	0.0450	0.31558		0.68442
0.8700	0.0350	0.26613		0.73387
0.9000	0.0200	0.17400		0.82600
0.9400	0.0000	0.00000		1.00000

**Table 6-70 11x11 Cladding ID Sensitivity Analysis Results**

Filename	Clad inner diameter (cm)	Clad thickness (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
clad_thick_ID_8110.out	0.8110	0.0645	0.88528	0.00050	0.88628
clad_thick_ID_8140.out	0.8140	0.0630	0.88633	0.00043	0.88719
clad_thick_ID_8220.out	0.8220	0.0590	0.88821	0.00046	0.88913
channel_2540.out	0.8260	0.0570	0.88852	0.00044	0.88940
clad_thick_ID_8300.out	0.8300	0.0550	0.88957	0.00049	0.89055
<b>clad_thick_ID_8400.out</b>	<b>0.8400</b>	<b>0.0500</b>	<b>0.89191</b>	<b>0.00044</b>	<b>0.89279</b>
clad_thick_ID_8500.out	0.8500	0.0450	0.89424	0.00043	0.89510
clad_thick_ID_8700.out	0.8700	0.0350	0.90020	0.00048	0.90116
clad_thick_ID_9000.out	0.9000	0.0200	0.90712	0.00049	0.90810
clad_thick_ID_9400.out	0.9400	0.0000	0.91706	0.00051	0.91808


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**Table 6-71 11x11 Zirc and Polyethylene Volume Fractions, Varying Cladding OD**

Fuel assembly parameters				
Clad ID, cm	0.8260			
Number of full length fuel rods	92		Length, cm	385.0
Number of long partial length fuel rods	8		Length, cm	236.8
Number of short partial length fuel rods	12		Length, cm	155.1
Volume Fraction Calculations				
Clad outer diameter (cm)	Clad/poly diameter (cm)	Clad thickness (cm)	VF <sub>clad</sub>	VF <sub>poly</sub>
0.8260	1.0158	0.0000	0.00000	1.00000
0.8400	1.0272	0.0070	0.06255	0.93745
0.8750	1.0560	0.0245	0.19255	0.80745
0.9100	1.0852	0.0420	0.29437	0.70563
0.9200	1.0936	0.0470	0.31950	0.68050
0.9300	1.1020	0.0520	0.34320	0.65680
0.9350	1.1062	0.0545	0.35454	0.64546
0.9400	1.1104	0.0570	0.36557	0.63443
0.9450	1.1146	0.0595	0.37630	0.62370
0.9650	1.1316	0.0695	0.41613	0.58387
0.9850	1.1488	0.0795	0.45171	0.54829
1.0050	1.1660	0.0895	0.48392	0.51608
1.0358	1.1926	0.1049	0.52783	0.47217
1.0386	1.1950	0.1063	0.53156	0.46844

**Table 6-72 11x11 Cladding OD Sensitivity Analysis Results**

Filename	Clad outer diameter (cm)	Clad thickness (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
clad_thick_OD_08260.out	0.8260	0.0000	0.91500	0.00049	0.91598
clad_thick_OD_08400.out	0.8400	0.0070	0.91269	0.00046	0.91361
clad_thick_OD_08750.out	0.8750	0.0245	0.90355	0.00045	0.90445
clad_thick_OD_09100.out	0.9100	0.0420	0.89627	0.00046	0.89719
clad_thick_OD_09200.out	0.9200	0.0470	0.89326	0.00049	0.89424
<b>clad_thick_OD_09300.out</b>	<b>0.9300</b>	<b>0.0520</b>	<b>0.89093</b>	<b>0.00045</b>	<b>0.89183</b>
clad_thick_OD_09350.out	0.9350	0.0545	0.89064	0.00048	0.89160
channel_2540.out	0.9400	0.0570	0.88852	0.00044	0.88940
clad_thick_OD_09450.out	0.9450	0.0595	0.88782	0.00045	0.88872
clad_thick_OD_09650.out	0.9650	0.0695	0.88298	0.00044	0.88386
clad_thick_OD_09850.out	0.9850	0.0795	0.87673	0.00049	0.87771
clad_thick_OD_10050.out	1.0050	0.0895	0.87196	0.00043	0.87282
clad_thick_OD_10358.out	1.0358	0.1049	0.86185	0.00045	0.86275
clad_thick_OD_10386.out	1.0386	0.1063	0.86097	0.00048	0.86193

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**Table 6-73 11x11 Sensitivity Analysis Results for Partially Flooded Inner Container**

Filename	# of fuel rows filled	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
rows_filled_01.out	1	0.74005	0.00042	0.74089
rows_filled_03.out	3	0.79346	0.00049	0.79444
rows_filled_05.out	5	0.85444	0.00050	0.85544
rows_filled_07.out	7	0.90061	0.00051	0.90163
rows_filled_09.out	9	0.92082	0.00049	0.92180
rows_filled_10.out	10	0.92672	0.00046	0.92764
<b>worst_param.out</b>	<b>11 (all)</b>	<b>0.92675</b>	<b>0.00053</b>	<b>0.92781</b>

**Table 6-74 11x11 Sensitivity Analysis Results for Thermal Insulator Material**


Filename	Insulator material	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
<b>worst_param.out</b>	<b>Alumina Silicate</b>	<b>0.92675</b>	<b>0.00053</b>	<b>0.92781</b>
therm_ins_to_void.out	Void	0.92225	0.00050	0.92325
therm_ins_to_water.out	Full density water	0.81792	0.00047	0.81886



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**Table 6-75 11x11 Sensitivity Analysis Results for Polyethylene Liner**


Filename	Assembly Orientation in Right Compartment <sup>a</sup>	Fuel Channel?	Polyethylene liner thickness (cm)	Fraction polyethylene-to-water	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
poly_liner_000_cc.out	centered	yes	0	0.00	0.92303	0.00048	0.92399
poly_liner_020_cc.out	centered	yes	0.2	0.12	0.93105	0.00044	0.93193
poly_liner_040_cc.out	centered	yes	0.4	0.24	0.93749	0.00044	0.93837
poly_liner_060_cc.out	centered	yes	0.6	0.36	0.94371	0.00048	0.94467
poly_liner_080_cc.out	centered	yes	0.8	0.48	0.94852	0.00045	0.94942
poly_liner_100_cc.out	centered	yes	1.0	0.60	0.95019	0.00048	0.95115
poly_liner_110_cc.out	centered	yes	1.1	0.66	0.95110	0.00043	0.95196
<b>poly_liner_120_cc.out</b>	<b>centered</b>	<b>yes</b>	<b>1.2</b>	<b>0.72</b>	<b>0.95170</b>	<b>0.00053</b>	<b>0.95276</b>
poly_liner_130_cc.out	centered	yes	1.3	0.78	0.95173	0.00046	0.95265
poly_liner_140_cc.out	centered	yes	1.4	0.84	0.95052	0.00052	0.95156
poly_liner_150_cc.out	centered	yes	1.5	0.91	0.94894	0.00049	0.94992
poly_liner_160_cc.out	centered	yes	1.6	0.97	0.94711	0.00046	0.94803
poly_liner_full_cc.out	centered	yes	1.657	1.00	0.94532	0.00047	0.94626
poly_liner_000_cn.out	centered	no	0	0.00	0.91624	0.00044	0.91712
poly_liner_020_cn.out	centered	no	0.2	0.10	0.92450	0.00046	0.92542
poly_liner_040_cn.out	centered	no	0.4	0.21	0.93153	0.00054	0.93261
poly_liner_060_cn.out	centered	no	0.6	0.31	0.93777	0.00047	0.93871
poly_liner_080_cn.out	centered	no	0.8	0.42	0.94333	0.00044	0.94421
poly_liner_100_cn.out	centered	no	1.0	0.52	0.94711	0.00047	0.94805
poly_liner_110_cn.out	centered	no	1.1	0.58	0.94890	0.00052	0.94994
poly_liner_120_cn.out	centered	no	1.2	0.63	0.94997	0.00048	0.95093
poly_liner_130_cn.out	centered	no	1.3	0.68	0.95090	0.00045	0.95180
poly_liner_140_cn.out	centered	no	1.4	0.73	0.95126	0.00046	0.95218
poly_liner_150_cn.out	centered	no	1.5	0.78	0.95131	0.00048	0.95227
poly_liner_160_cn.out	centered	no	1.6	0.84	0.95026	0.00046	0.95118
poly_liner_170_cn.out	centered	no	1.7	0.89	0.94880	0.00044	0.94968
poly_liner_180_cn.out	centered	no	1.8	0.94	0.94747	0.00049	0.94845
poly_liner_full_cn.out	centered	no	1.911	1.00	0.94407	0.00045	0.94497

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Handling: None	Page 435/517		

**Table 6-75 11x11 Sensitivity Analysis Results for Polyethylene Liner (continued)**


Filename	Assembly Orientation in Right Compartment <sup>a</sup>	Fuel Channel?	Polyethylene liner thickness (cm)	Fraction polyethylene-to-water	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
poly_liner_000_sc.out	shifted	yes	0	0.00	0.92677	0.00046	0.92769
poly_liner_020_sc.out	shifted	yes	0.2	0.12	0.93087	0.00046	0.93179
poly_liner_040_sc.out	shifted	yes	0.4	0.24	0.93609	0.00048	0.93705
poly_liner_060_sc.out	shifted	yes	0.6	0.36	0.94036	0.00048	0.94132
poly_liner_080_sc.out	shifted	yes	0.8	0.48	0.94399	0.00043	0.94485
poly_liner_100_sc.out	shifted	yes	1.0	0.60	0.94757	0.00043	0.94843
poly_liner_110_sc.out	shifted	yes	1.1	0.66	0.94835	0.00049	0.94933
poly_liner_120_sc.out	shifted	yes	1.2	0.72	0.94879	0.00050	0.94979
poly_liner_130_sc.out	shifted	yes	1.3	0.78	0.94907	0.00046	0.94999
poly_liner_140_sc.out	shifted	yes	1.4	0.84	0.94794	0.00044	0.94882
poly_liner_150_sc.out	shifted	yes	1.5	0.91	0.94746	0.00047	0.94840
poly_liner_160_sc.out	shifted	yes	1.6	0.97	0.94647	0.00047	0.94741
poly_liner_full_sc.out	shifted	yes	1.657	1.00	0.94563	0.00049	0.94661
poly_liner_000_sn.out	shifted	no	0	0.00	0.92242	0.00046	0.92334
poly_liner_020_sn.out	shifted	no	0.2	0.10	0.92716	0.00046	0.92808
poly_liner_040_sn.out	shifted	no	0.4	0.21	0.93223	0.00047	0.93317
poly_liner_060_sn.out	shifted	no	0.6	0.31	0.93651	0.00046	0.93743
poly_liner_080_sn.out	shifted	no	0.8	0.42	0.94041	0.00052	0.94145
poly_liner_100_sn.out	shifted	no	1.0	0.52	0.94493	0.00044	0.94581
poly_liner_110_sn.out	shifted	no	1.1	0.58	0.94662	0.00045	0.94752
poly_liner_120_sn.out	shifted	no	1.2	0.63	0.94676	0.00051	0.94778
poly_liner_130_sn.out	shifted	no	1.3	0.68	0.94852	0.00046	0.94944
poly_liner_140_sn.out	shifted	no	1.4	0.73	0.94822	0.00053	0.94928
poly_liner_150_sn.out	shifted	no	1.5	0.78	0.94861	0.00050	0.94961
poly_liner_160_sn.out	shifted	no	1.6	0.84	0.94844	0.00047	0.94938
poly_liner_170_sn.out	shifted	no	1.7	0.89	0.94764	0.00046	0.94856
poly_liner_180_sn.out	shifted	no	1.8	0.94	0.94698	0.00043	0.94784
poly_liner_full_sn.out	shifted	no	1.911	1.00	0.94500	0.00045	0.94590

<sup>a</sup> Assembly in left compartment is centered.

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**Table 6-76 11x11 Bottom Layer “2-D” Analysis Results**

Filename	<sup>235</sup> U wt%	# Gd rods	Pattern	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
FandPLRs_02gd_A1.out	3.2	2	A1	0.94716	0.00040	0.94796
FandPLRs_02gd_A2.out	3.2	2	A2	0.94639	0.00050	0.94739
FandPLRs_02gd_A4.out	3.2	2	A4	0.94728	0.00048	0.94824
FandPLRs_03gd_A1.out	3.3	3	A1	0.94725	0.00050	0.94825
FandPLRs_03gd_A2.out	3.3	3	A2	0.94549	0.00048	0.94645
FandPLRs_03gd_B3.out	3.3	3	B3	0.94248	0.00044	0.94336
FandPLRs_04gd_A1.out	3.5	4	A1	0.94881	0.00045	0.94971
FandPLRs_04gd_A2.out	3.5	4	A2	0.95449	0.00045	0.95539
FandPLRs_04gd_B3.out	3.5	4	B3	0.95015	0.00047	0.95109
FandPLRs_05gd_A1.out	3.6	5	A1	0.94950	0.00050	0.95050
FandPLRs_05gd_A2.out	3.6	5	A2	0.94886	0.00047	0.94980
FandPLRs_05gd_B3.out	3.6	5	B3	0.94806	0.00045	0.94896
FandPLRs_06gd_A2.out	3.8	6	A2	0.95510	0.00043	0.95596
FandPLRs_06gd_A3.out	3.8	6	A3	0.95397	0.00049	0.95495
FandPLRs_06gd_E1.out	3.8	6	E1	0.94981	0.00051	0.95083
FandPLRs_07gd_A1.out	3.9	7	A1	0.94977	0.00048	0.95073
FandPLRs_07gd_A3.out	3.9	7	A3	0.95213	0.00046	0.95305
FandPLRs_07gd_E1.out	3.9	7	E1	0.95044	0.00043	0.95130
FandPLRs_08gd_A2.out	4.1	8	A2	0.95357	0.00048	0.95453
FandPLRs_08gd_B1.out	4.1	8	B1	0.95784	0.00048	0.95880
FandPLRs_08gd_E1.out	4.1	8	E1	0.95462	0.00046	0.95554
FandPLRs_09gd_A2.out	4.2	9	A2	0.95166	0.00043	0.95252
FandPLRs_09gd_B1.out	4.2	9	B1	0.95356	0.00049	0.95454
FandPLRs_09gd_B3.out	4.2	9	B3	0.95174	0.00051	0.95276
FandPLRs_10gd_A2.out	4.4	10	A2	0.95594	0.00044	0.95682
FandPLRs_10gd_C2.out	4.4	10	C2	0.95262	0.00041	0.95344
FandPLRs_10gd_C5.out	4.4	10	C5	0.95048	0.00057	0.95162
FandPLRs_11gd_B1.out	4.6	11	B1	0.95642	0.00044	0.95730
FandPLRs_11gd_B2.out	4.6	11	B2	0.95622	0.00044	0.95710
FandPLRs_11gd_B3.out	4.6	11	B3	0.95311	0.00046	0.95403
FandPLRs_12gd_A2.out	4.8	12	A2	0.95692	0.00051	0.95794
FandPLRs_12gd_B2.out	4.8	12	B2	0.95926	0.00043	0.96012
FandPLRs_12gd_D2.out	4.8	12	D2	0.95303	0.00044	0.95391
FandPLRs_13gd_A2.out	5.0	13	A2	0.95763	0.00051	0.95865
FandPLRs_13gd_C3.out	5.0	13	C3	0.96128	0.00047	0.96222
FandPLRs_13gd_F1.out	5.0	13	F1	0.95707	0.00048	0.95803

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**Table 6-77 11x11 Middle Layer “2-D” Analysis Results**

Filename	<sup>235</sup> U wt%	# Gd rods	Pattern	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
LPL_02gd_A1.out	3.2	2	A1	0.92677	0.00042	0.92761
LPL_02gd_A4.out	3.2	2	A4	0.92843	0.00044	0.92931
LPL_02gd_B2.out	3.2	2	B2	0.92546	0.00043	0.92632
LPL_03gd_A1.out	3.3	3	A1	0.92806	0.00054	0.92914
LPL_03gd_B1.out	3.3	3	B1	0.92046	0.00046	0.92138
LPL_03gd_B2.out	3.3	3	B2	0.92201	0.00047	0.92295
LPL_04gd_A1.out	3.5	4	A1	0.92787	0.00044	0.92875
LPL_04gd_A3.out	3.5	4	A3	0.92828	0.00044	0.92916
LPL_04gd_A4.out	3.5	4	A4	0.92391	0.00042	0.92475
LPL_05gd_A1.out	3.6	5	A1	0.92765	0.00046	0.92857
LPL_05gd_A2.out	3.6	5	A2	0.92282	0.00045	0.92372
LPL_05gd_C1.out	3.6	5	C1	0.92173	0.00048	0.92269
LPL_06gd_A1.out	3.8	6	A1	0.92793	0.00050	0.92893
LPL_06gd_B1.out	3.8	6	B1	0.92678	0.00049	0.92776
LPL_06gd_B3.out	3.8	6	B3	0.92691	0.00044	0.92779
LPL_07gd_A1.out	3.9	7	A1	0.92803	0.00050	0.92903
LPL_07gd_A2.out	3.9	7	A2	0.92316	0.00043	0.92402
LPL_07gd_B1.out	3.9	7	B1	0.92456	0.00045	0.92546
LPL_08gd_A1.out	4.1	8	A1	0.92721	0.00048	0.92817
LPL_08gd_A2.out	4.1	8	A2	0.92866	0.00046	0.92958
LPL_08gd_A3.out	4.1	8	A3	0.92578	0.00047	0.92672
LPL_09gd_A1.out	4.2	9	A1	0.92598	0.00047	0.92692
LPL_09gd_B1.out	4.2	9	B1	0.92178	0.00047	0.92272
LPL_09gd_C2.out	4.2	9	C2	0.92164	0.00048	0.92260
LPL_10gd_A1.out	4.4	10	A1	0.92439	0.00044	0.92527
LPL_10gd_A2.out	4.4	10	A2	0.92657	0.00051	0.92759
LPL_10gd_A4.out	4.4	10	A4	0.92394	0.00047	0.92488
LPL_11gd_A1.out	4.6	11	A1	0.92859	0.00043	0.92945
LPL_11gd_B2.out	4.6	11	B2	0.92733	0.00051	0.92835
LPL_11gd_D3.out	4.6	11	D3	0.92450	0.00050	0.92550
LPL_12gd_A2.out	4.8	12	A2	0.93001	0.00047	0.93095
LPL_12gd_B1.out	4.8	12	B1	0.92732	0.00041	0.92814
LPL_12gd_C4.out	4.8	12	C4	0.92629	0.00050	0.92729
LPL_13gd_A3.out	5.0	13	A3	0.92655	0.00048	0.92751
LPL_13gd_B1.out	5.0	13	B1	0.93041	0.00046	0.93133
LPL_13gd_B2.out	5.0	13	B2	0.92729	0.00045	0.92819

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**Table 6-78 11x11 Top Layer “2-D” Analysis Results**


Filename	<sup>235</sup> U wt%	# Gd rods	Pattern	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
FLRs_02gd_A1.out	3.2	2	A1	0.91924	0.00052	0.92028
FLRs_02gd_A3.out	3.2	2	A3	0.92021	0.00041	0.92103
FLRs_02gd_B1.out	3.2	2	B1	0.91673	0.00046	0.91765
FLRs_03gd_A1.out	3.3	3	A1	0.92143	0.00045	0.92233
FLRs_03gd_A3.out	3.3	3	A3	0.91417	0.00042	0.91501
FLRs_03gd_B1.out	3.3	3	B1	0.91395	0.00041	0.91477
FLRs_04gd_A1.out	3.5	4	A1	0.92048	0.00044	0.92136
FLRs_04gd_A3.out	3.5	4	A3	0.92087	0.00042	0.92171
FLRs_04gd_B1.out	3.5	4	B1	0.91466	0.00039	0.91544
FLRs_05gd_A1.out	3.6	5	A1	0.92043	0.00040	0.92123
FLRs_05gd_A2.out	3.6	5	A2	0.91335	0.00044	0.91423
FLRs_05gd_B2.out	3.6	5	B2	0.91304	0.00046	0.91396
FLRs_06gd_A1.out	3.8	6	A1	0.91807	0.00050	0.91907
FLRs_06gd_A3.out	3.8	6	A3	0.91919	0.00046	0.92011
FLRs_06gd_B2.out	3.8	6	B2	0.91962	0.00045	0.92052
FLRs_07gd_A1.out	3.9	7	A1	0.91680	0.00046	0.91772
FLRs_07gd_A2.out	3.9	7	A2	0.91669	0.00044	0.91757
FLRs_07gd_B3.out	3.9	7	B3	0.91214	0.00046	0.91306
FLRs_08gd_A1.out	4.1	8	A1	0.91598	0.00044	0.91686
FLRs_08gd_B2.out	4.1	8	B2	0.91602	0.00041	0.91684
FLRs_08gd_D1.out	4.1	8	D1	0.91573	0.00044	0.91661
FLRs_09gd_A1.out	4.2	9	A1	0.91535	0.00045	0.91625
FLRs_09gd_A2.out	4.2	9	A2	0.91121	0.00045	0.91211
FLRs_09gd_B3.out	4.2	9	B3	0.90798	0.00044	0.90886
FLRs_10gd_A1.out	4.4	10	A1	0.91104	0.00046	0.91196
FLRs_10gd_B2.out	4.4	10	B2	0.91242	0.00042	0.91326
FLRs_10gd_B4.out	4.4	10	B4	0.91047	0.00050	0.91147
FLRs_11gd_A1.out	4.6	11	A1	0.91520	0.00054	0.91628
FLRs_11gd_A2.out	4.6	11	A2	0.91331	0.00044	0.91419
FLRs_11gd_A3.out	4.6	11	A3	0.91092	0.00042	0.91176
FLRs_12gd_A2.out	4.8	12	A2	0.91032	0.00045	0.91122
FLRs_12gd_B2.out	4.8	12	B2	0.91117	0.00045	0.91207
FLRs_12gd_B4.out	4.8	12	B4	0.91193	0.00047	0.91287
FLRs_13gd_A2.out	5.0	13	A2	0.91337	0.00051	0.91439
FLRs_13gd_A3.out	5.0	13	A3	0.91117	0.00045	0.91207
FLRs_13gd_A4.out	5.0	13	A4	0.91044	0.00043	0.91130

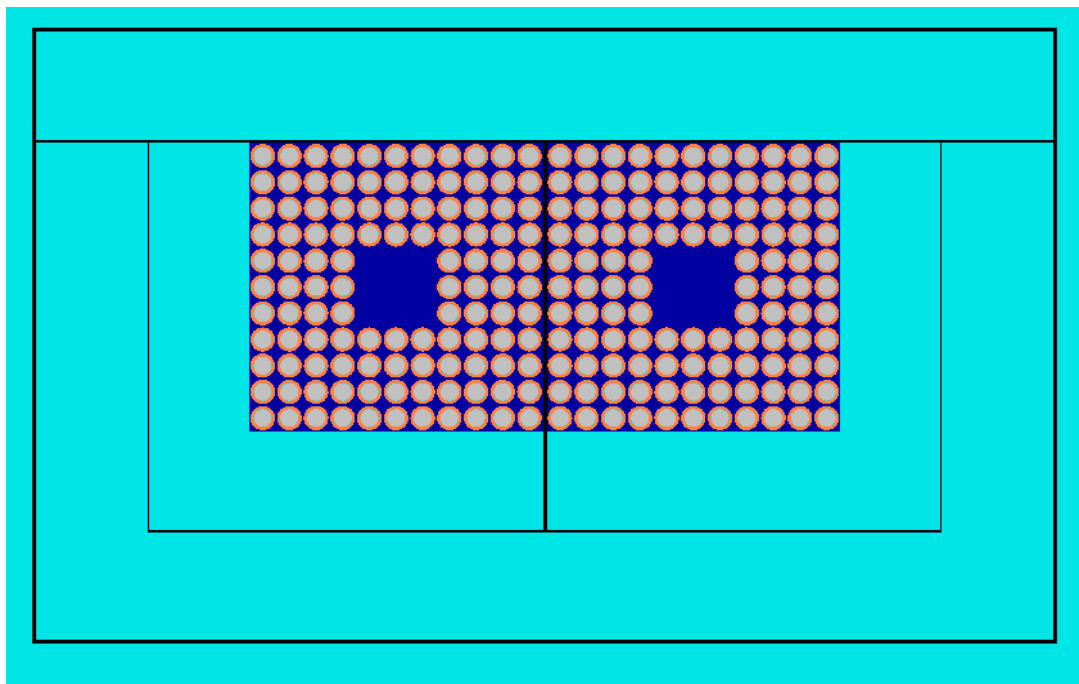
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**Table 6-79 11x11 Gadolinia-Urania Analysis Results**

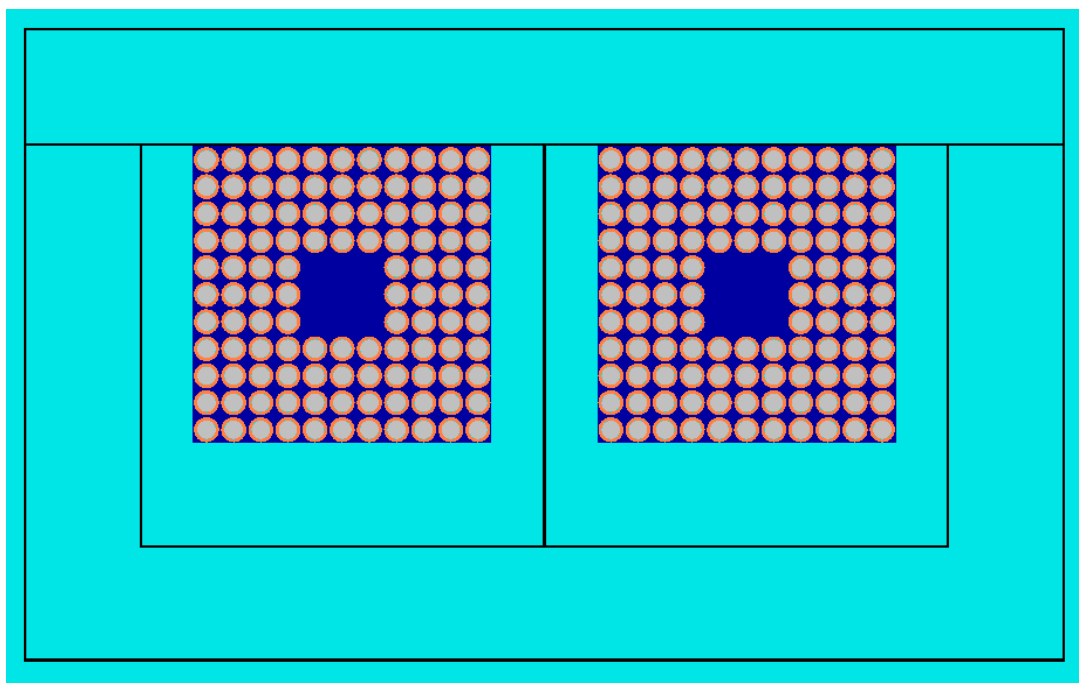
Filename	<sup>235</sup> U wt%	# Gd rods	Axial Pattern			k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
			Bottom	Middle	Top			
<b>5wt_13gd.out</b>	<b>5.0</b>	<b>13</b>	<b>C3</b>	<b>B1</b>	<b>A2</b>	<b>0.93710</b>	<b>0.00050</b>	<b>0.93810</b>
48wt_12gd_fullA2.out	4.8	12	B2	A2	A2	0.93550	0.00046	0.93642
48wt_12gd_fullB2.out	4.8	12	B2	A2	B2	0.93576	0.00048	0.93672
<b>48wt_12gd_fullB4.out</b>	<b>4.8</b>	<b>12</b>	<b>B2</b>	<b>A2</b>	<b>B4</b>	<b>0.93599</b>	<b>0.00047</b>	<b>0.93693</b>
<b>46wt_11gd_B1-A1-A1.out</b>	<b>4.6</b>	<b>11</b>	<b>B1</b>	<b>A1</b>	<b>A1</b>	<b>0.93582</b>	<b>0.00047</b>	<b>0.93676</b>
46wt_11gd_B1-B2-A1.out	4.6	11	B1	B2	A1	0.93548	0.00050	0.93648
46wt_11gd_B2-A1-A1.out	4.6	11	B2	A1	A1	0.93493	0.00047	0.93587
46wt_11gd_B2-B2-A1.out	4.6	11	B2	B2	A1	0.93476	0.00042	0.93560
44wt_10gd_fullA1.out	4.4	10	A2	A2	A1	0.93413	0.00048	0.93509
<b>44wt_10gd_fullB2.out</b>	<b>4.4</b>	<b>10</b>	<b>A2</b>	<b>A2</b>	<b>B2</b>	<b>0.93471</b>	<b>0.00044</b>	<b>0.93559</b>
<b>42wt_09gd.out</b>	<b>4.2</b>	<b>9</b>	<b>B1</b>	<b>A1</b>	<b>A1</b>	<b>0.93341</b>	<b>0.00045</b>	<b>0.93431</b>
41wt_08gd_fullA1.out	4.1	8	B1	A2	A1	0.93683	0.00046	<b>0.93775</b>
41wt_08gd_fullB2.out	4.1	8	B1	A2	B2	0.93662	0.00042	0.93746
41wt_08gd_fullD1.out	4.1	8	B1	A2	D1	0.93589	0.00043	0.93675
<b>39wt_07gd_fullA1.out</b>	<b>3.9</b>	<b>7</b>	<b>A3</b>	<b>A1</b>	<b>A1</b>	<b>0.93417</b>	<b>0.00046</b>	<b>0.93509</b>
39wt_07gd_fullA2.out	3.9	7	A3	A1	A2	0.93403	0.00047	0.93497
38wt_06gd_A2_A1_A3.out	3.8	6	A2	A1	A3	0.93579	0.00046	0.93671
<b>38wt_06gd_A2_A1_B2.out</b>	<b>3.8</b>	<b>6</b>	<b>A2</b>	<b>A1</b>	<b>B2</b>	<b>0.93684</b>	<b>0.00043</b>	<b>0.93770</b>
38wt_06gd_A2_B1_B2.out	3.8	6	A2	B1	B2	0.93620	0.00045	0.93710
38wt_06gd_A2_B3_A3.out	3.8	6	A2	B3	A3	0.93662	0.00050	0.93762
38wt_06gd_A2_B3_B2.out	3.8	6	A2	B3	B2	0.93545	0.00050	0.93645
38wt_06gd_A3_A1_B2.out	3.8	6	A3	A1	B2	0.93551	0.00051	0.93653
38wt_06gd_A3_B3_A3.out	3.8	6	A3	B3	A3	0.93520	0.00048	0.93616
36wt_05gd_botA1.out	3.6	5	A1	A1	A1	0.93372	0.00048	0.93468
<b>36wt_05gd_botA2.out</b>	<b>3.6</b>	<b>5</b>	<b>A2</b>	<b>A1</b>	<b>A1</b>	<b>0.93420</b>	<b>0.00043</b>	<b>0.93506</b>
36wt_05gd_botB3.out	3.6	5	B3	A1	A1	0.93364	0.00046	0.93456
35wt_04gd_A2-A1-A1.out	3.5	4	A2	A1	A1	0.93596	0.00048	0.93692
35wt_04gd_A2-A1-A3.out	3.5	4	A2	A1	A3	0.93602	0.00050	0.93702
35wt_04gd_A2-A3-A1.out	3.5	4	A2	A3	A1	0.93499	0.00043	0.93585
<b>35wt_04gd_A2-A3-A3.out</b>	<b>3.5</b>	<b>4</b>	<b>A2</b>	<b>A3</b>	<b>A3</b>	<b>0.93664</b>	<b>0.00044</b>	<b>0.93752</b>
<b>33wt_03gd.out</b>	<b>3.3</b>	<b>3</b>	<b>A1</b>	<b>A1</b>	<b>A1</b>	<b>0.93355</b>	<b>0.00047</b>	<b>0.93449</b>
32wt_02gd_A1-A1-A1.out	3.2	2	A1	A1	A1	0.93268	0.00046	0.93360
32wt_02gd_A1-A4-A1.out	3.2	2	A1	A4	A1	0.93236	0.00043	0.93322
32wt_02gd_A1-A4-A3.out	3.2	2	A1	A4	A3	0.93310	0.00045	0.93400
32wt_02gd_A2-A4-A3.out	3.2	2	A2	A4	A3	0.93276	0.00044	0.93364
32wt_02gd_A4-A1-A3.out	3.2	2	A4	A1	A3	0.93285	0.00043	0.93371
<b>32wt_02gd_A4-A4-A1.out</b>	<b>3.2</b>	<b>2</b>	<b>A4</b>	<b>A4</b>	<b>A1</b>	<b>0.93320</b>	<b>0.00042</b>	<b>0.93404</b>
32wt_02gd_A4-A4-A3.out	3.2	2	A4	A4	A3	0.93290	0.00054	0.93398
<b>29wt_00gd.out</b>	<b>2.9</b>	<b>0</b>	<b>---</b>	<b>---</b>	<b>---</b>	<b>0.93500</b>	<b>0.00048</b>	<b>0.93596</b>
<b>5-5-33wt_13-13-3gd.out</b>	<b>5 / 5 / 3.3<sup>a</sup></b>	<b>13 / 13 / 3<sup>a</sup></b>	<b>C3</b>	<b>B1</b>	<b>A1</b>	<b>0.93855</b>	<b>0.00044</b>	<b>0.93943</b>

<sup>a</sup> Values provided for each lattice region as bottom / middle / top.

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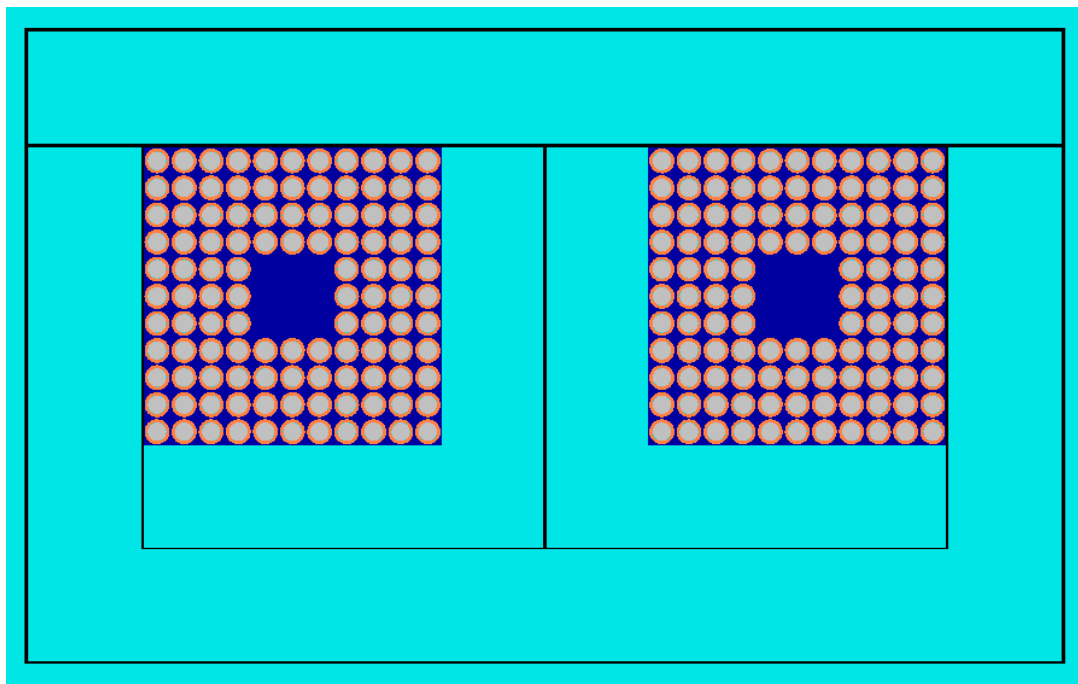


**Figure 6-55 Fuel Assembly Orientation 1**

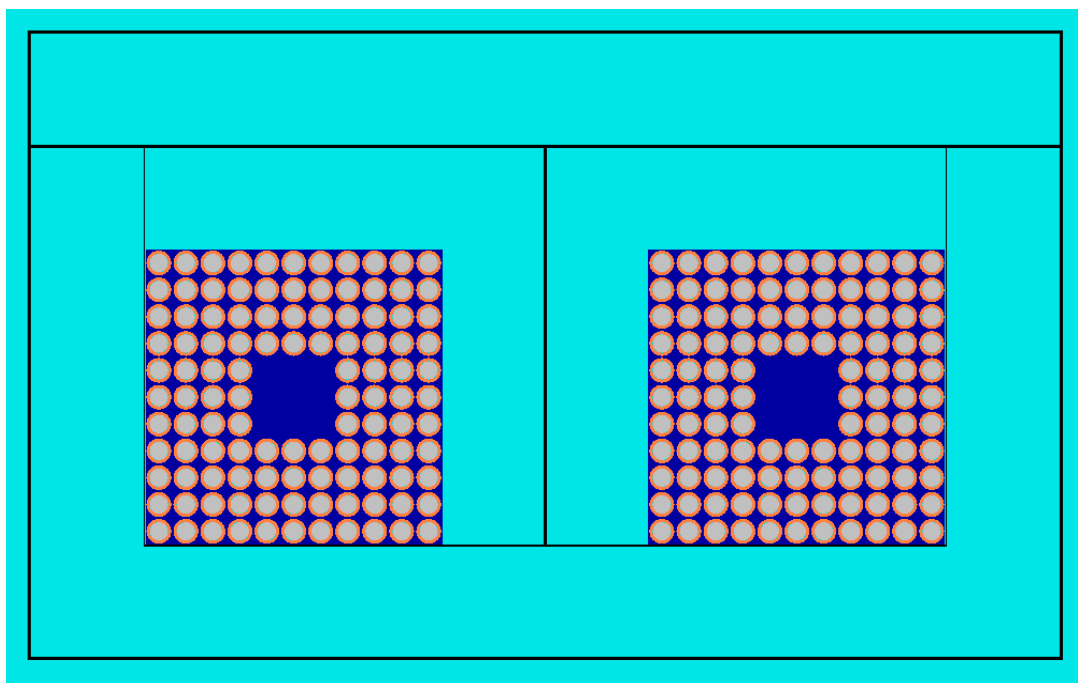


**Figure 6-56 Fuel Assembly Orientation 2**

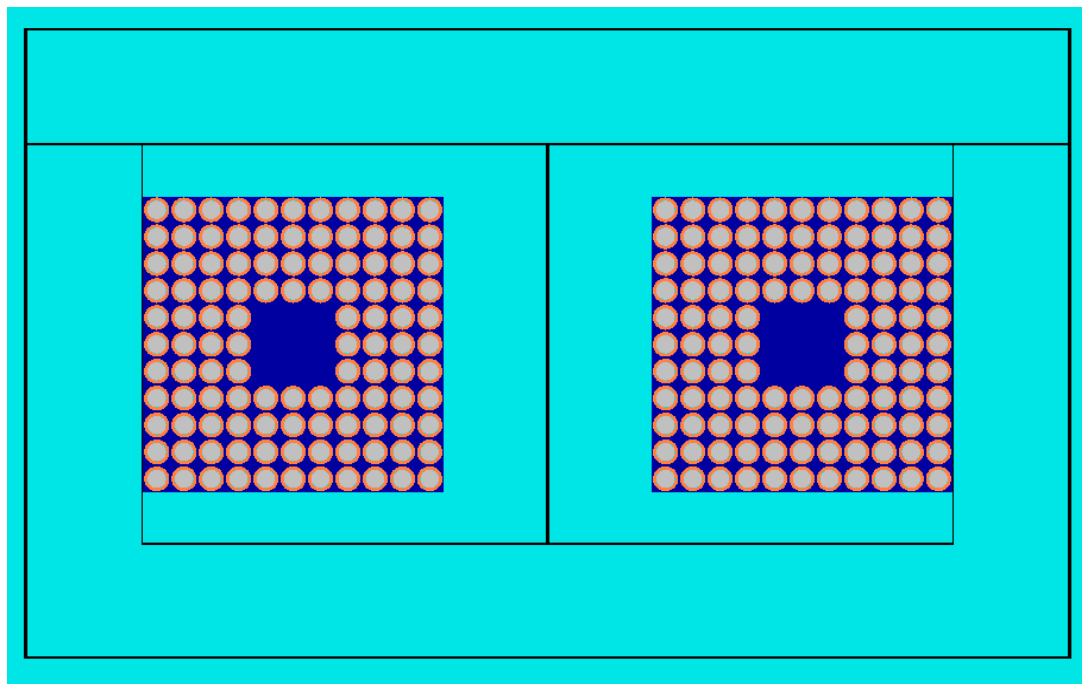




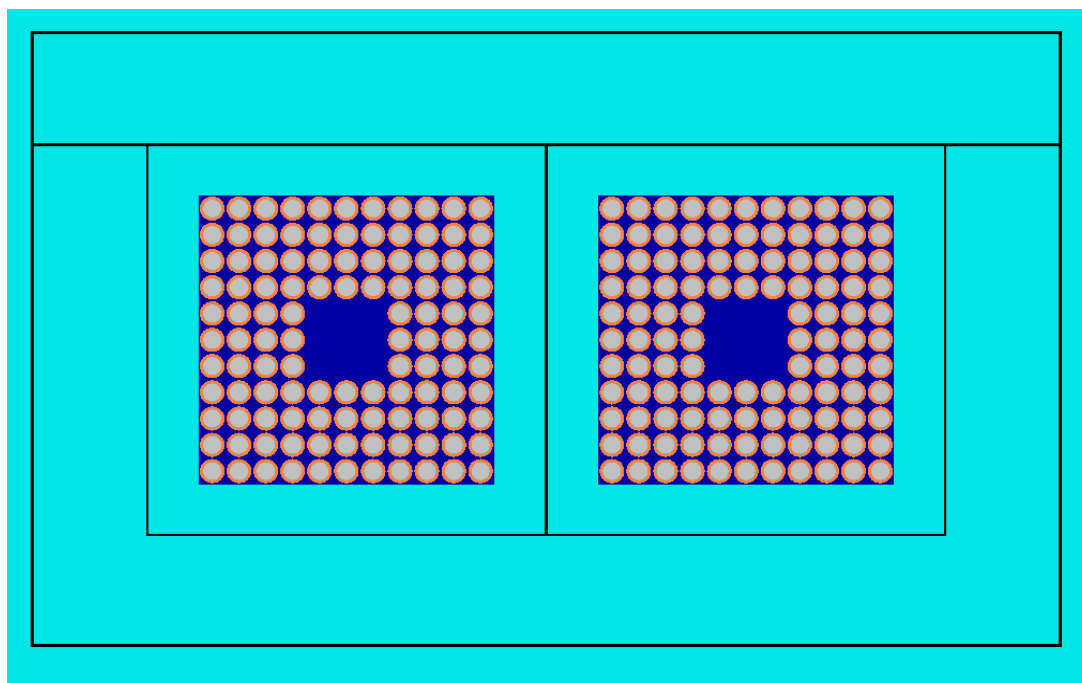
**Figure 6-57 Fuel Assembly Orientation 3**



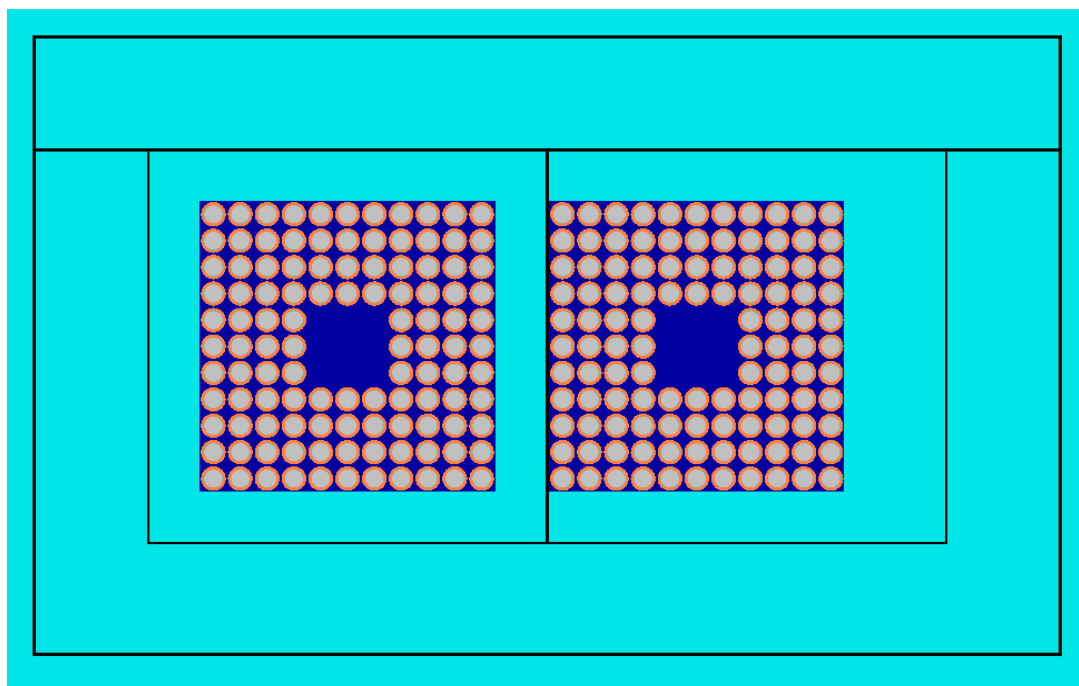
**Figure 6-58 Fuel Assembly Orientation 4**



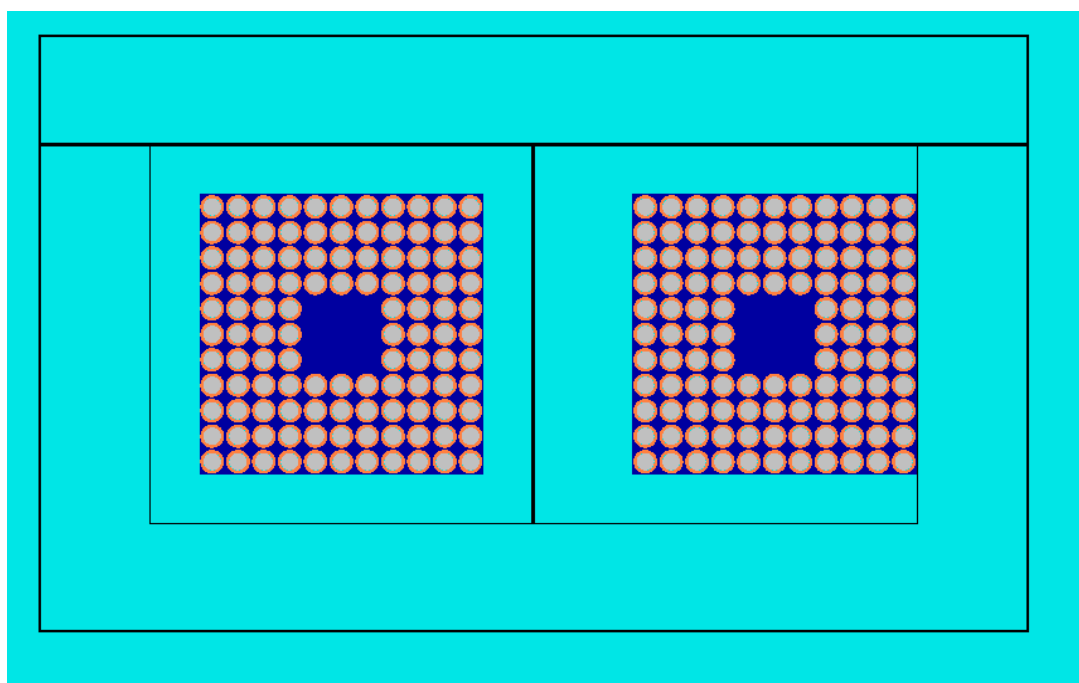
**Figure 6-59 Fuel Assembly Orientation 5**



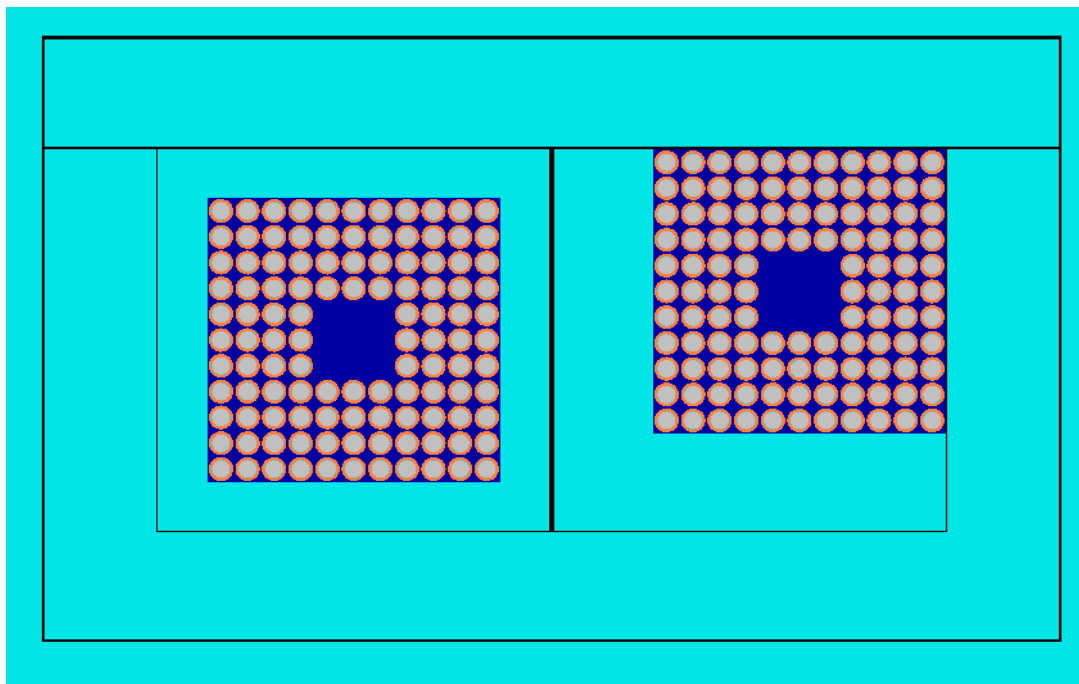
**Figure 6-60 Fuel Assembly Orientation 6**



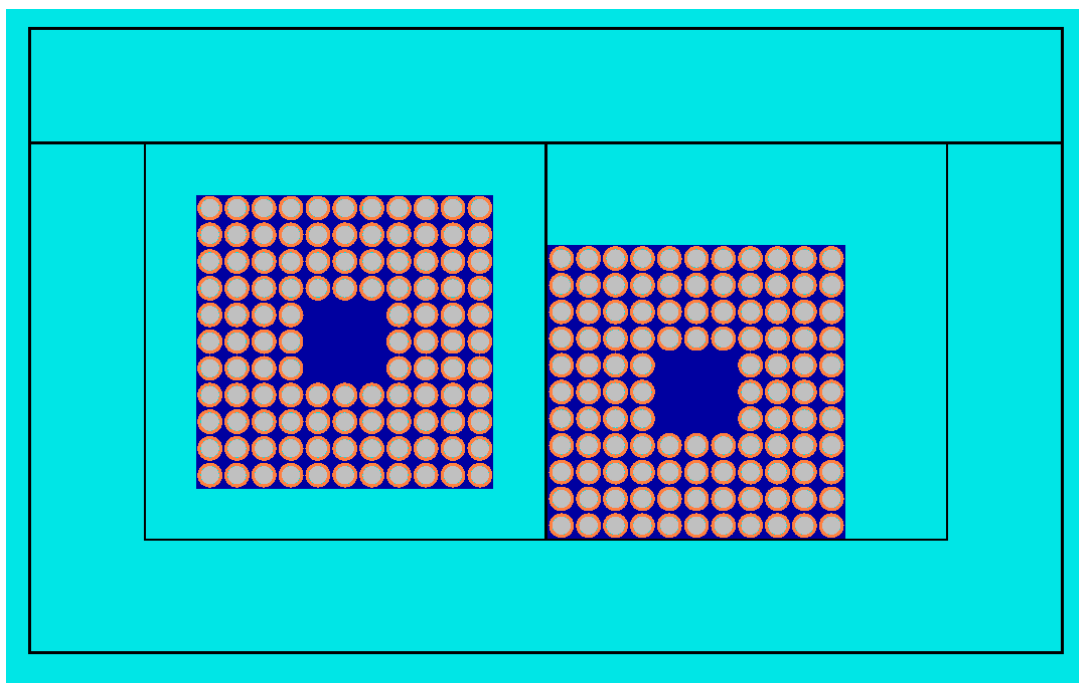
**Figure 6-61 Fuel Assembly Orientation 7**



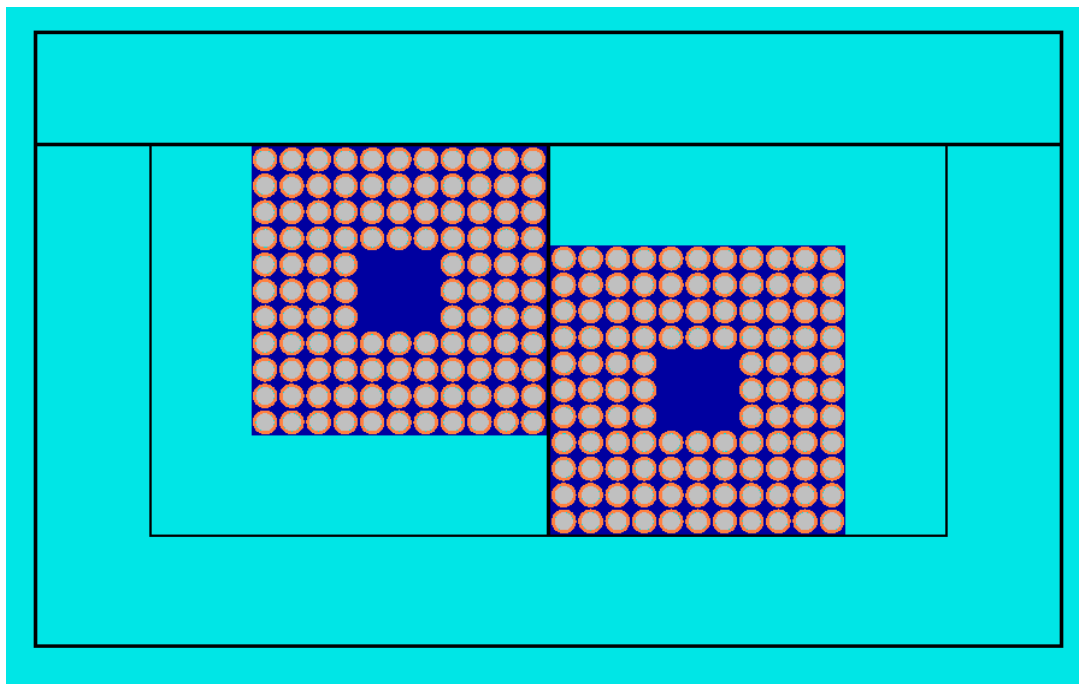
**Figure 6-62 Fuel Assembly Orientation 8**



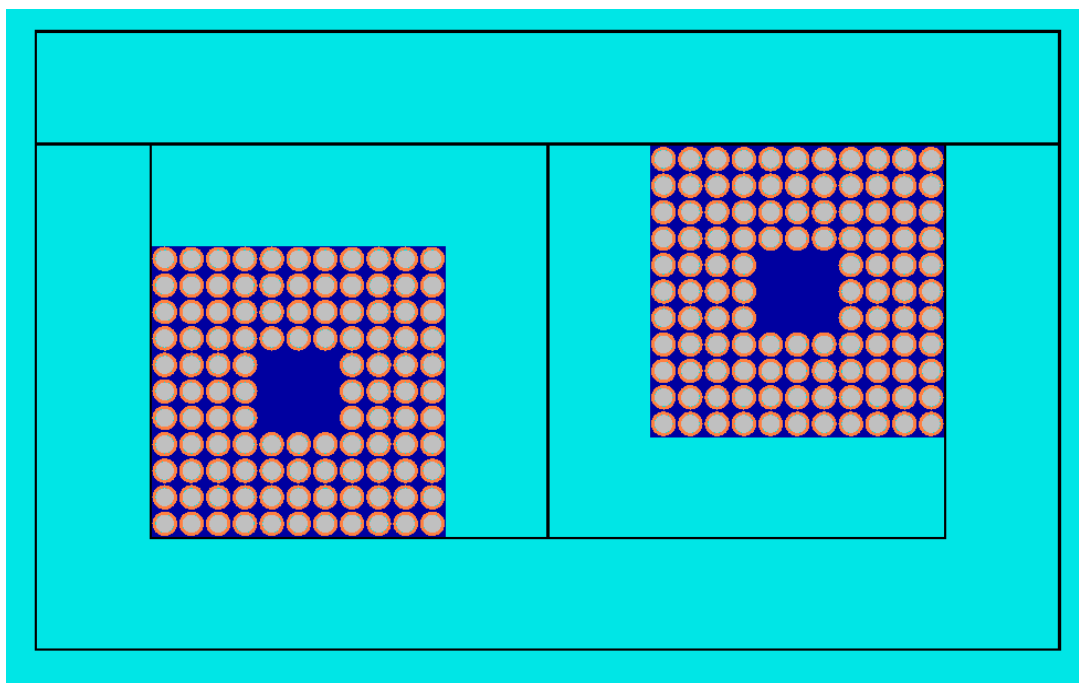
**Figure 6-63 Fuel Assembly Orientation 9**



**Figure 6-64 Fuel Assembly Orientation 10**

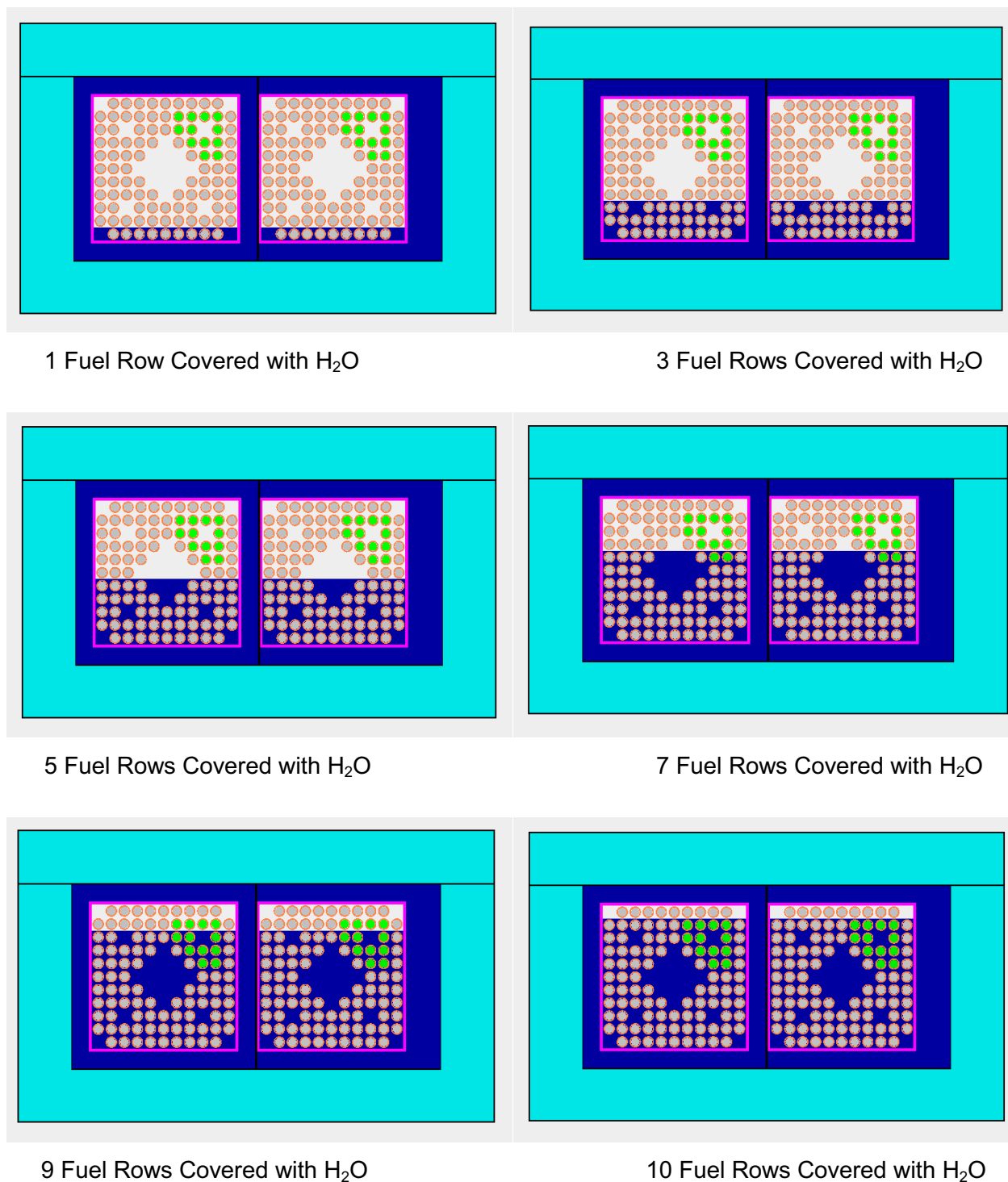


**Figure 6-65 Fuel Assembly Orientation 11**



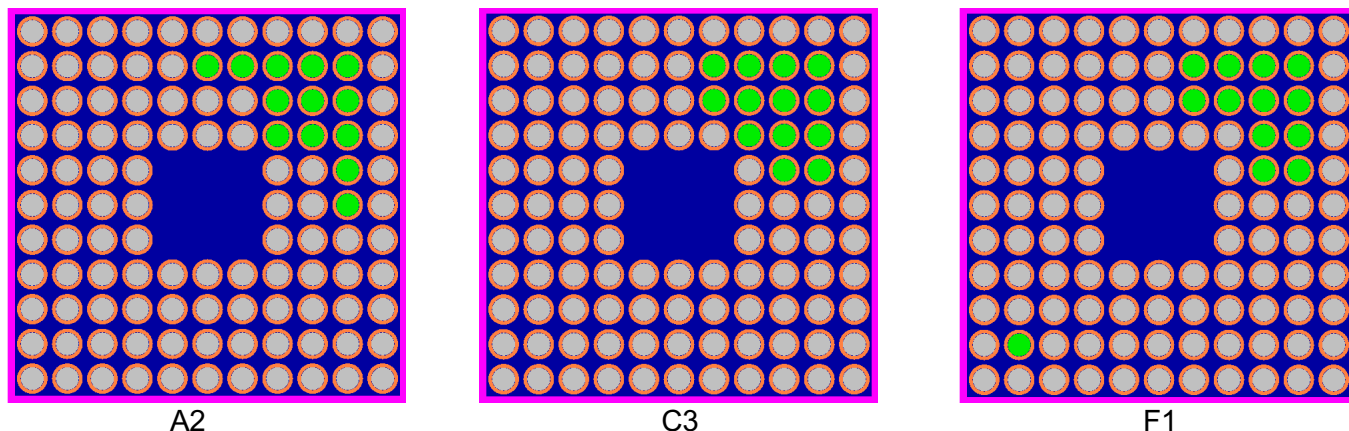
**Figure 6-66 Fuel Assembly Orientation 12**

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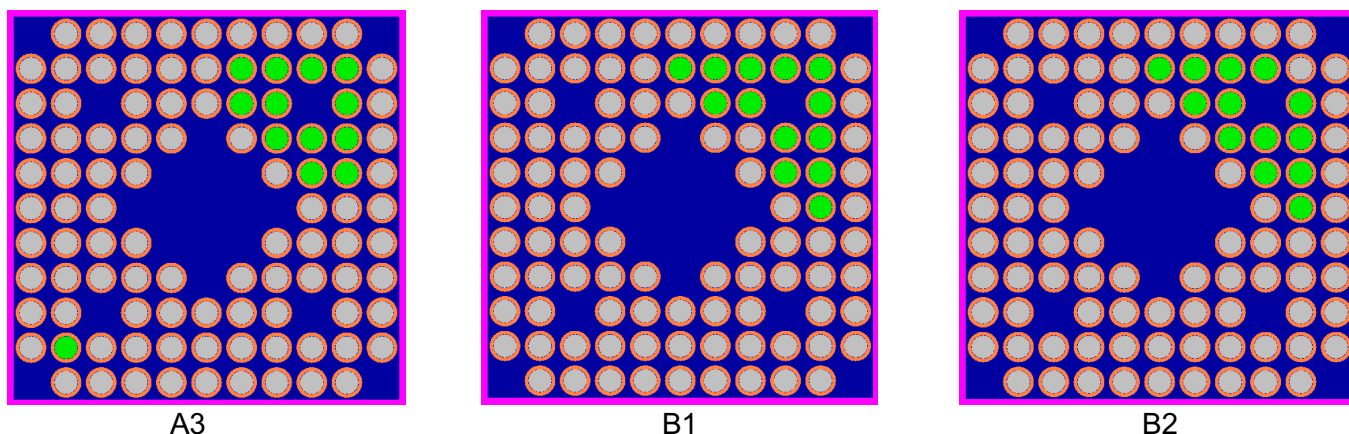


**Figure 6-67 Inner Container Partial Flooding**

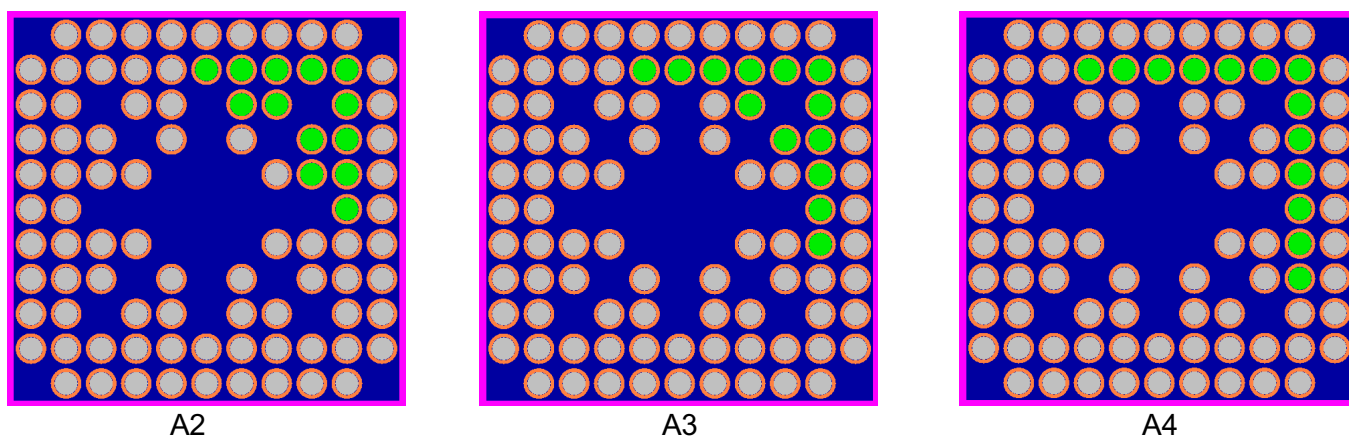
Bottom Axial Layer (Full and Partial Length Rods):



Middle Axial Layer (Full and Long Partial Length Rods):



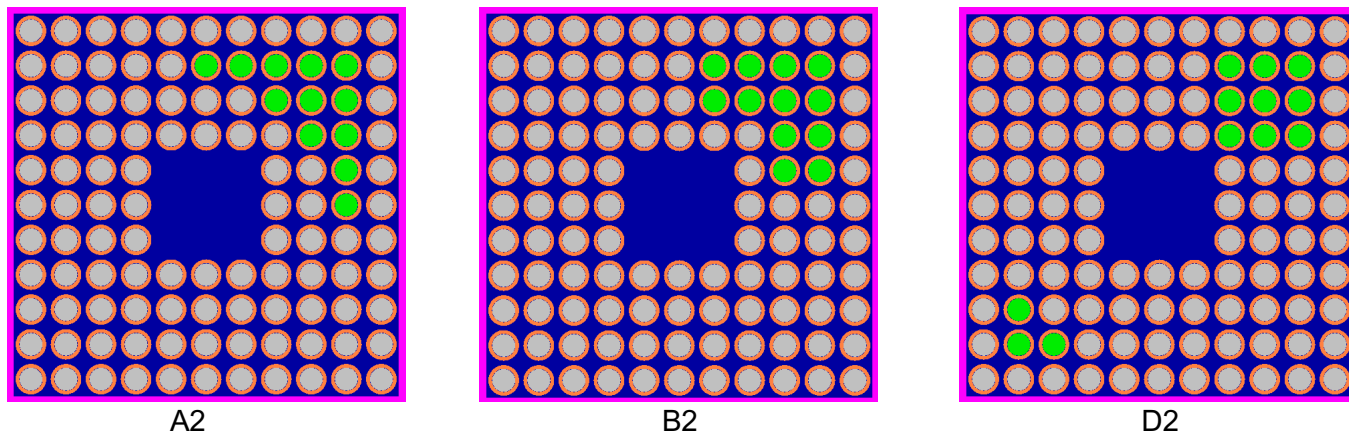
Top Axial Layer (Full Length Rods Only):



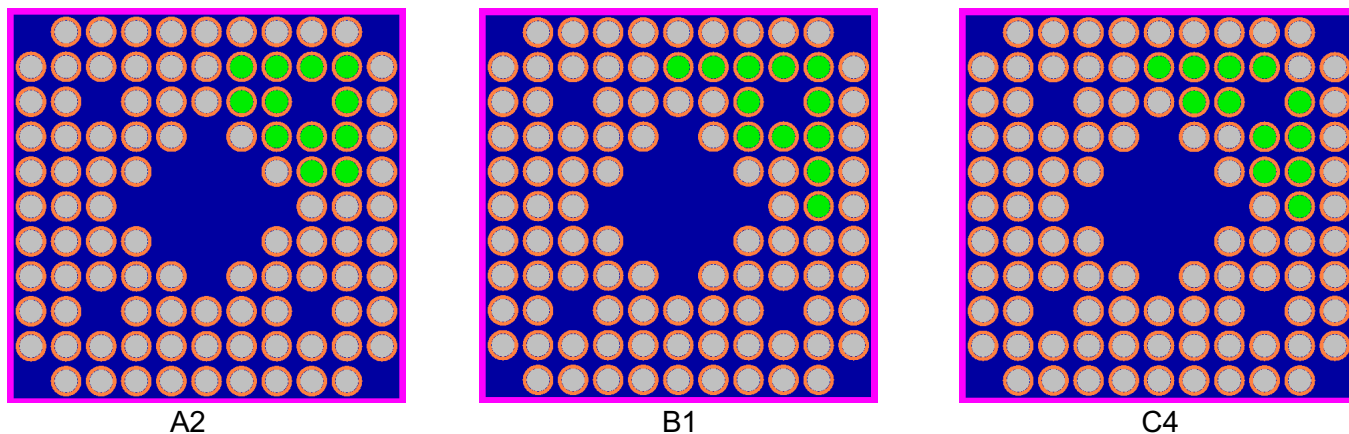
**Figure 6-68 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 5.0 wt% <sup>235</sup>U, 13 Gd Rods**



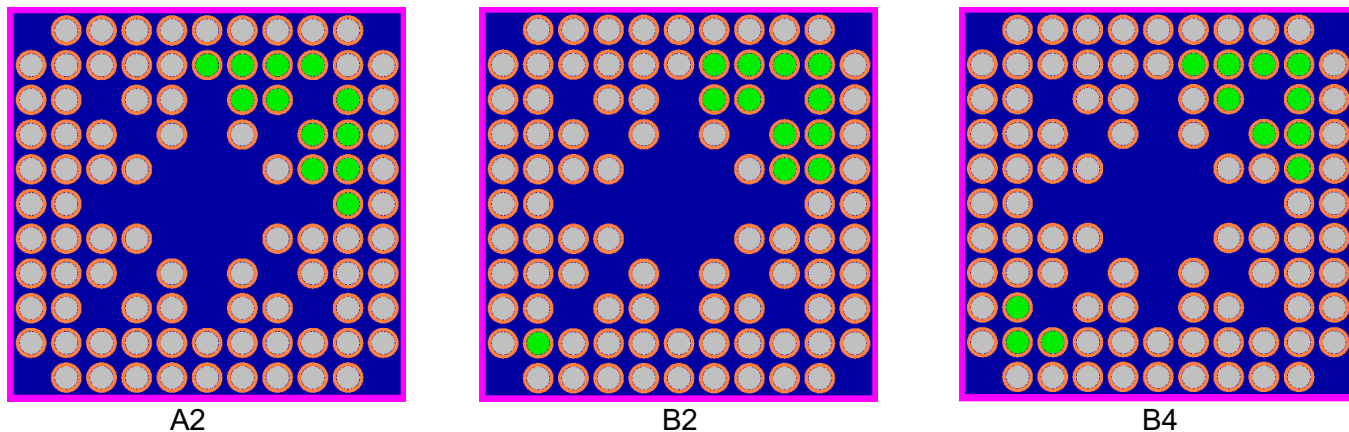
Bottom Axial Layer (Full and Partial Length Rods):




Middle Axial Layer (Full and Long Partial Length Rods):



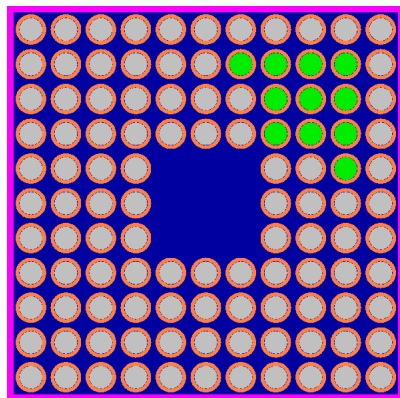
Top Axial Layer (Full Length Rods Only):



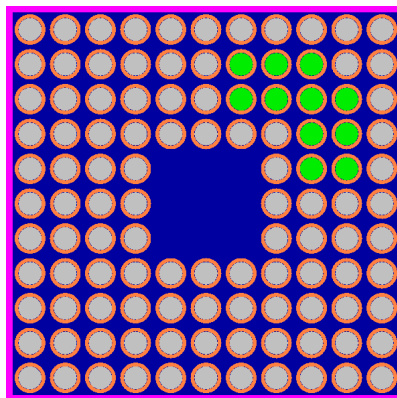
**Figure 6-69 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 4.8 wt% <sup>235</sup>U, 12 Gd Rods**

N° FS1-0014159 Rev. 7.0	<div style="text-align: center;"> <b>AREVA TN-B1</b>  <b>Docket No. 71-9372</b>  <b>Safety Analysis Report</b> </div> <div style="text-align: right;">  </div>
Handling: None Page 449/517	

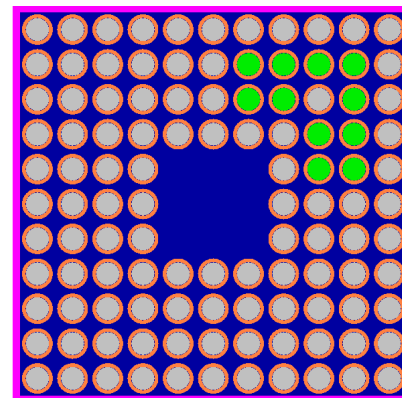
Bottom Axial Layer (Full and Partial Length Rods):



B1

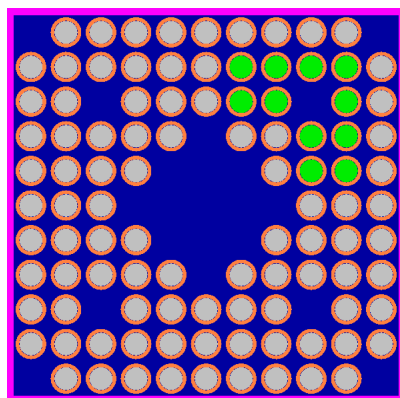


B2

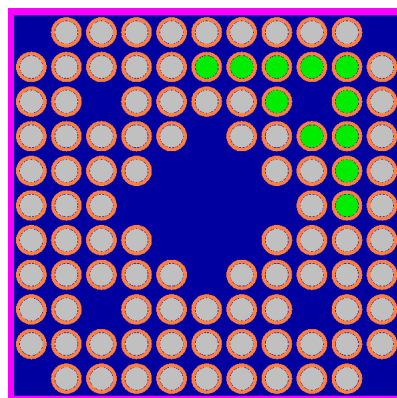


B3

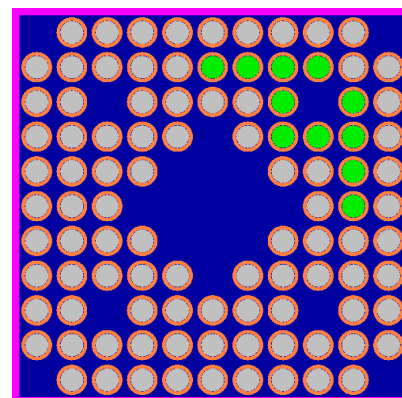
Middle Axial Layer (Full and Long Partial Length Rods):



A1

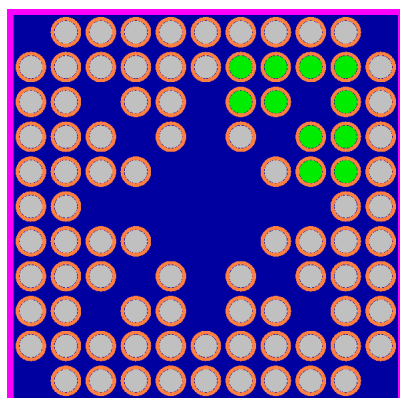


B2

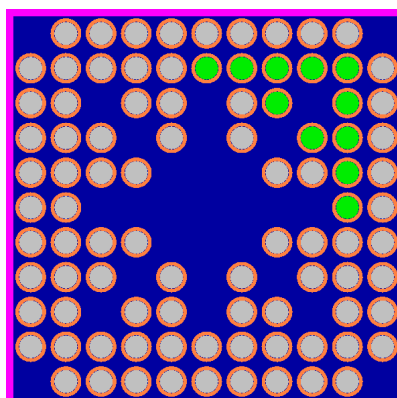


D3

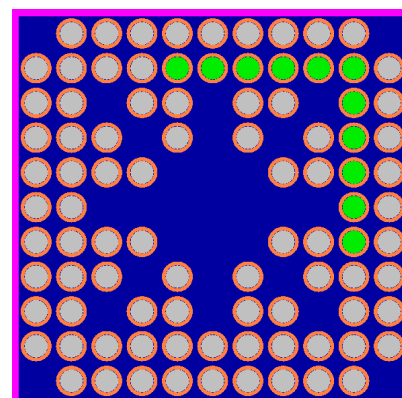
Top Axial Layer (Full Length Rods Only):



A1



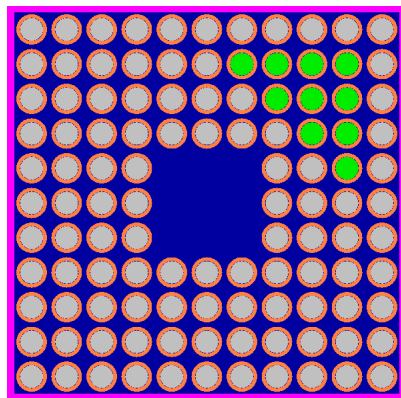
A2



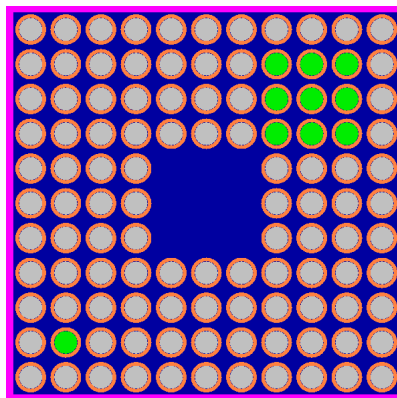
A3

**Figure 6-70 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 4.6 wt% <sup>235</sup>U, 11 Gd Rods**

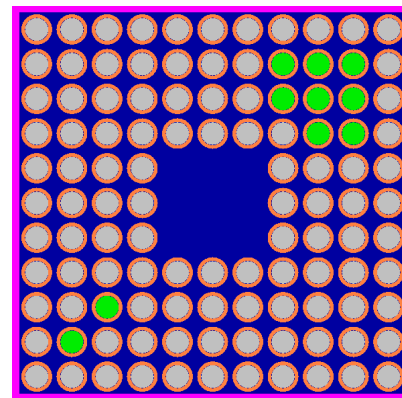
Bottom Axial Layer (Full and Partial Length Rods):



A2

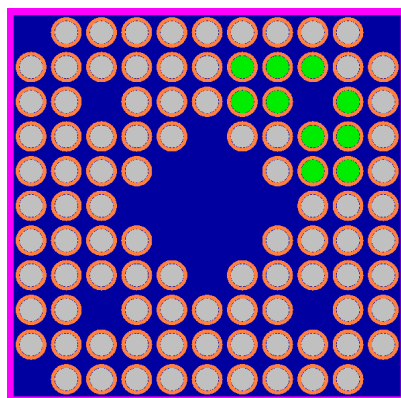


C2

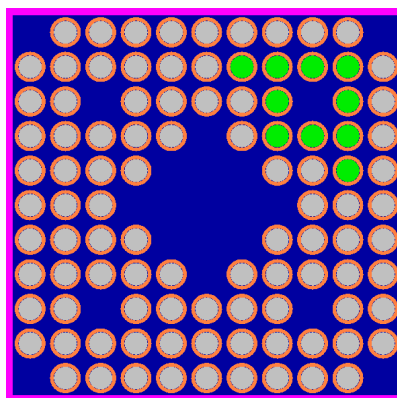


C5

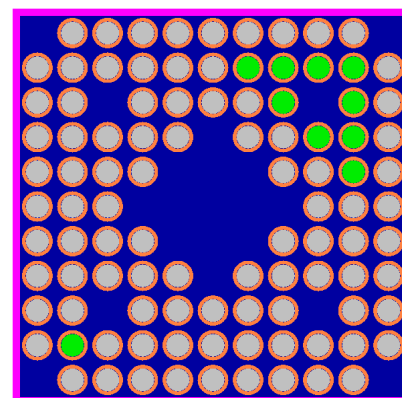
Middle Axial Layer (Full and Long Partial Length Rods):



A1

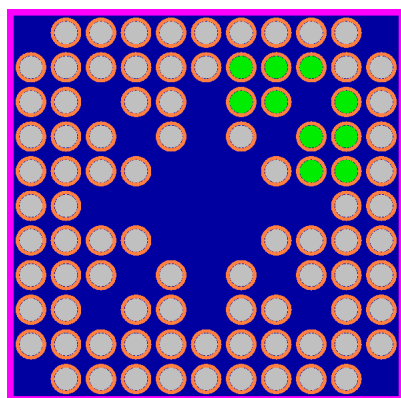


A2

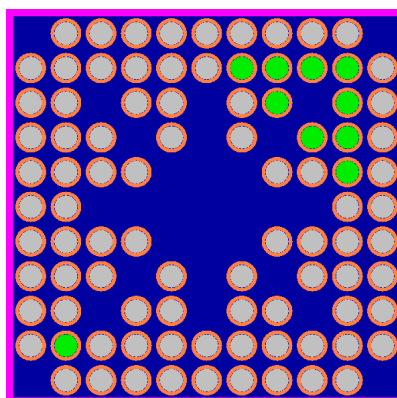


A4

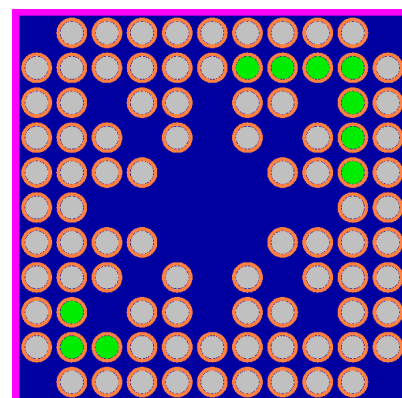
Top Axial Layer (Full Length Rods Only):



A1



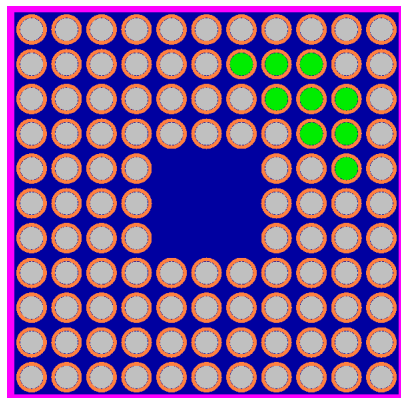
B2



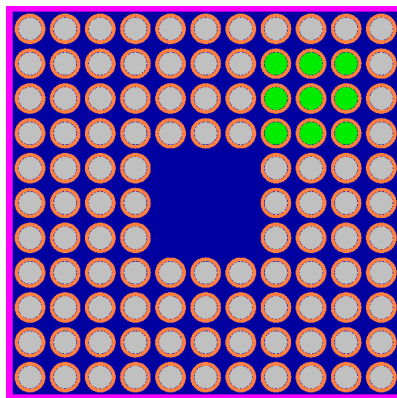
B4

**Figure 6-71 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 4.4 wt% <sup>235</sup>U, 10 Gd Rods**

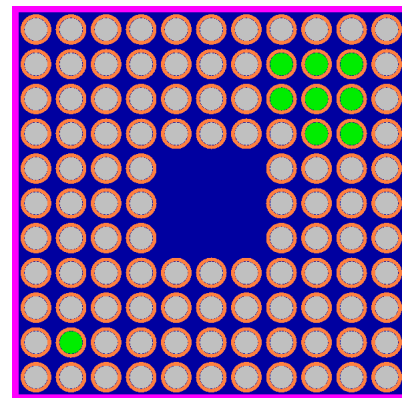
Bottom Axial Layer (Full and Partial Length Rods):



A2

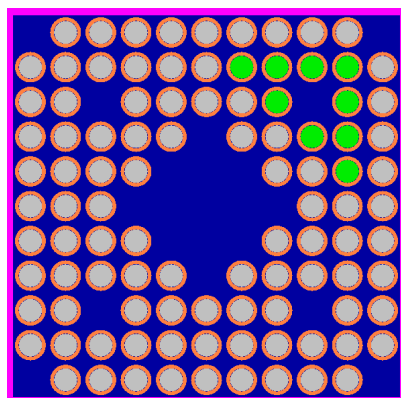


B1

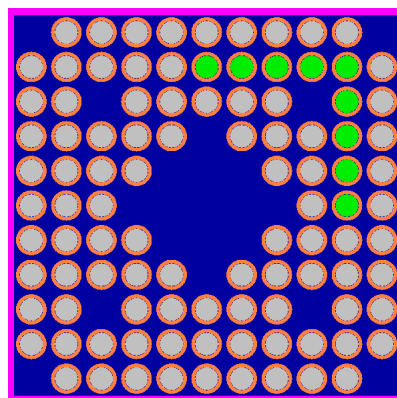


B3

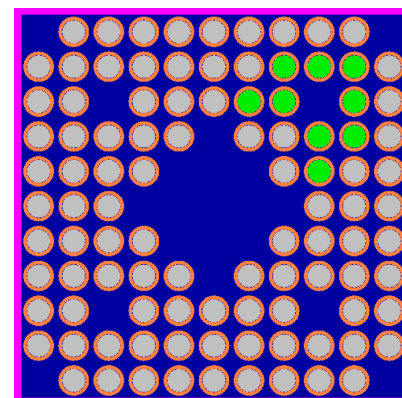
Middle Axial Layer (Full and Long Partial Length Rods):



A1

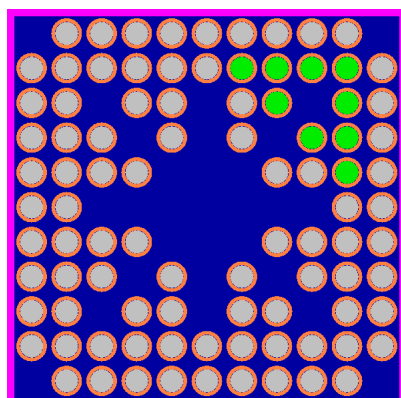


B1

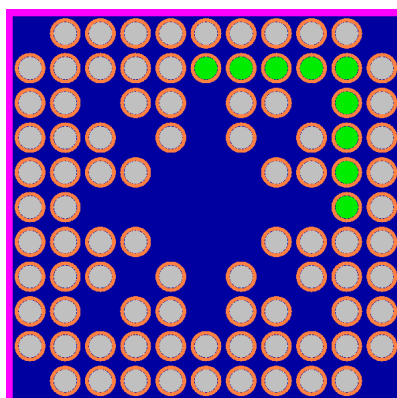


C2

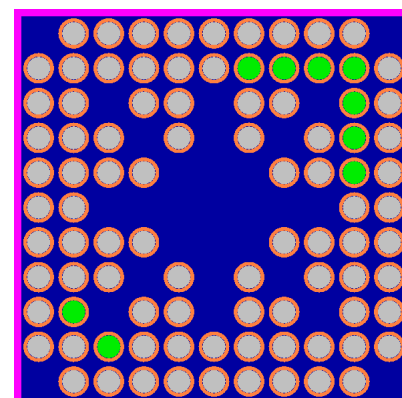
Top Axial Layer (Full Length Rods Only):



A1



A2

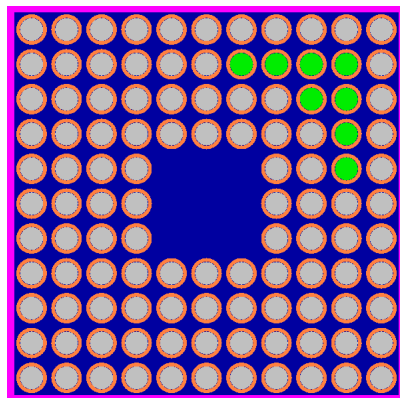


B3

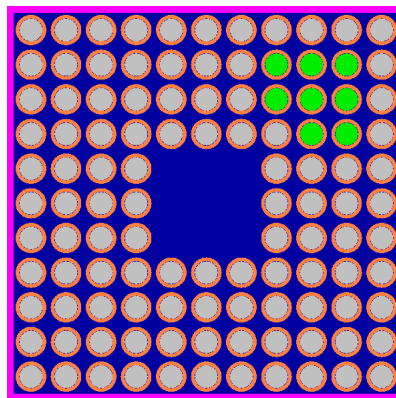
**Figure 6-72 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 4.2 wt%  $^{235}\text{U}$ , 9 Gd Rods**



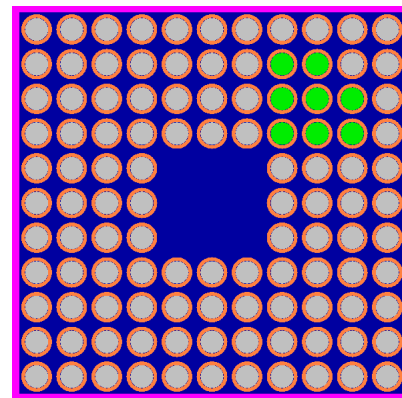
Bottom Axial Layer (Full and Partial Length Rods):



A2

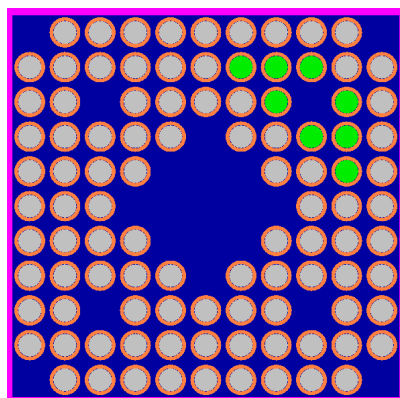


B1

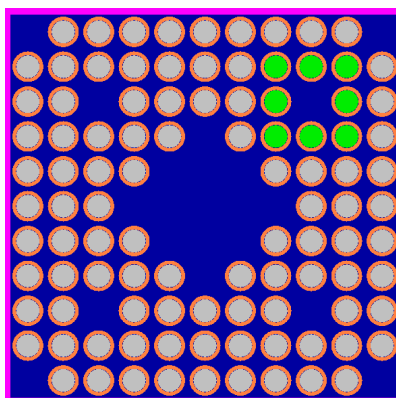


E1

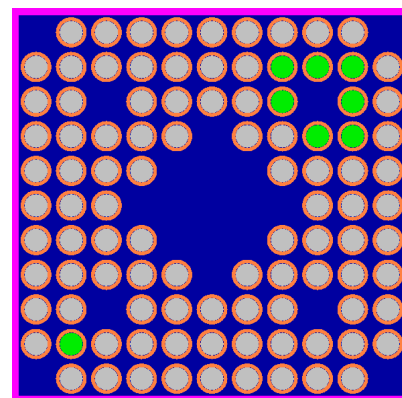
Middle Axial Layer (Full and Long Partial Length Rods):



A1

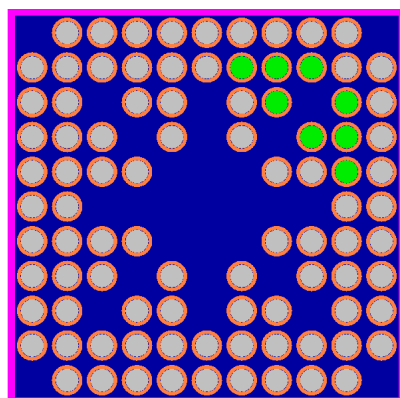


A2

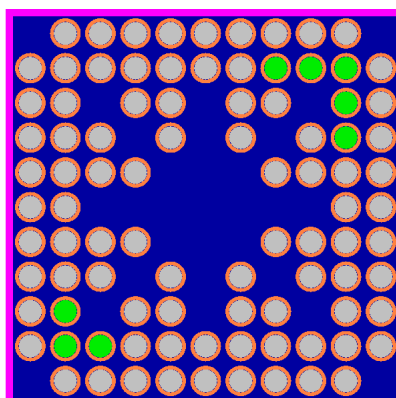


A3

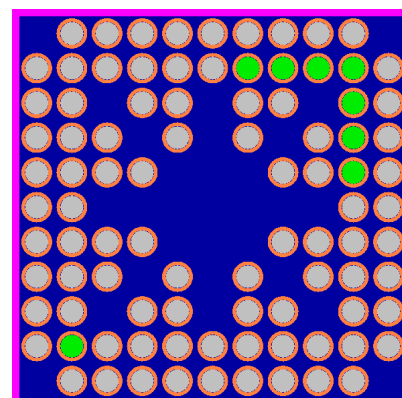
Top Axial Layer (Full Length Rods Only):



A1



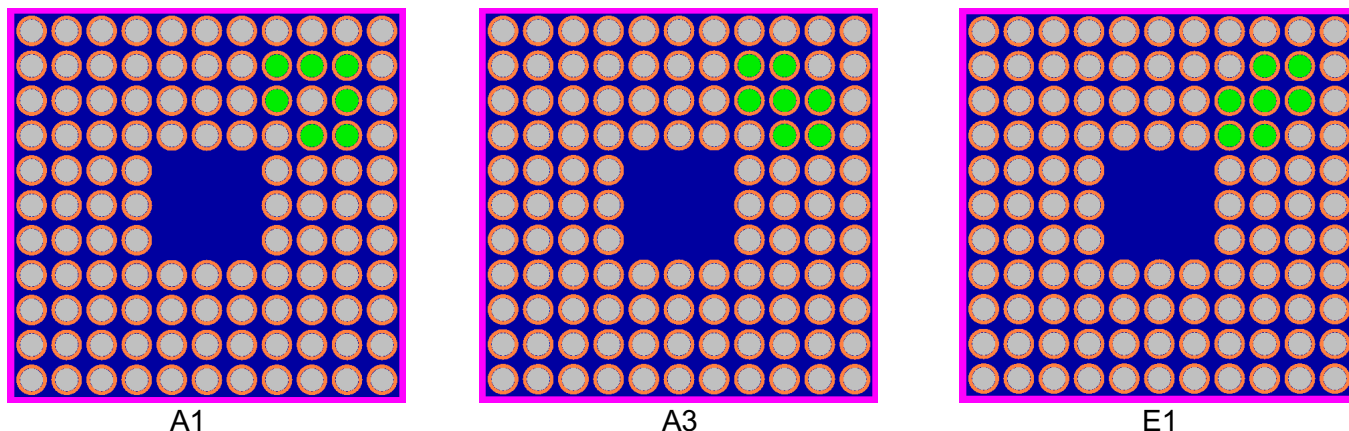
B2



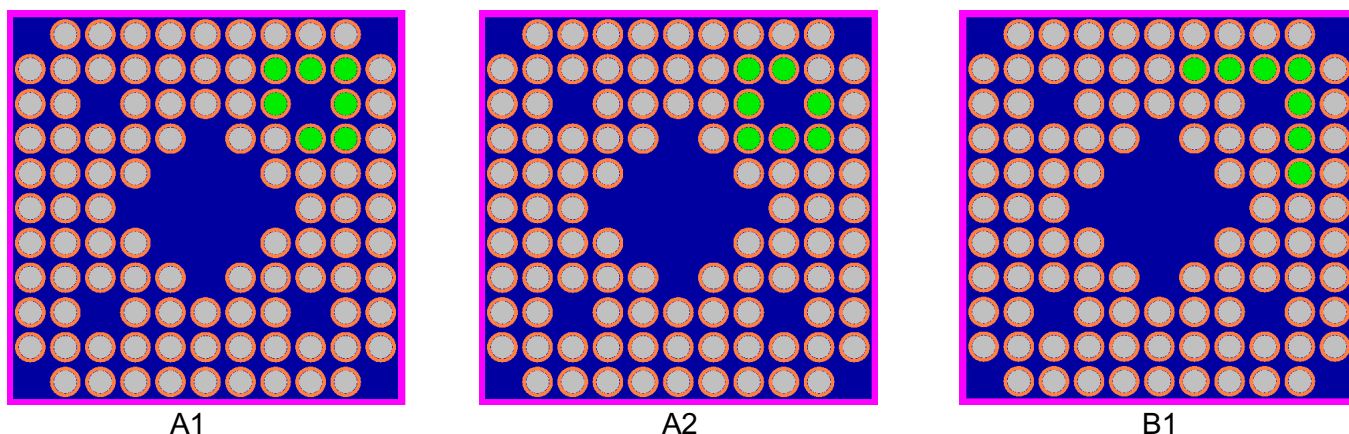
D1

**Figure 6-73 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 4.1 wt% <sup>235</sup>U, 8 Gd Rods**

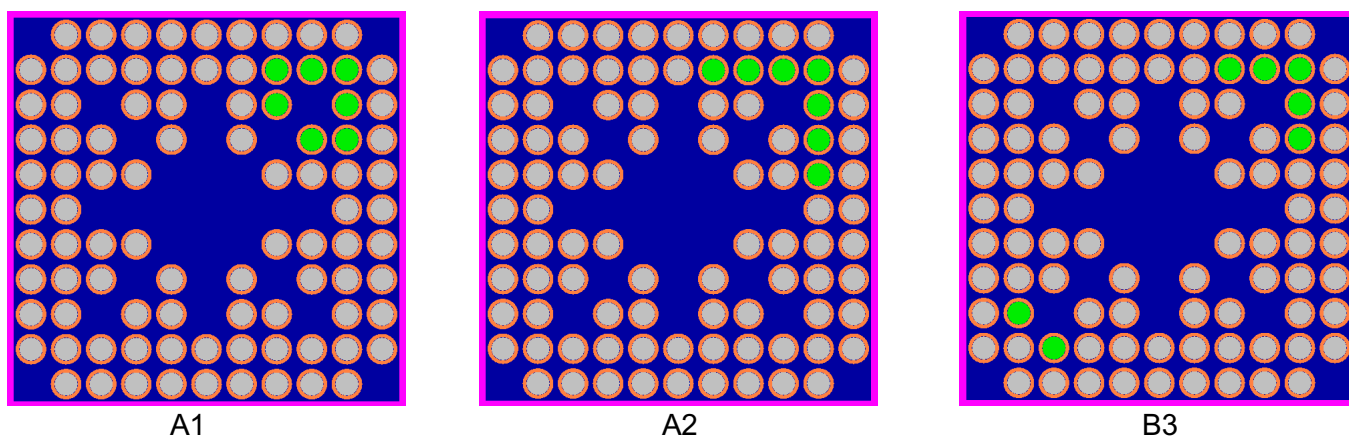
Bottom Axial Layer (Full and Partial Length Rods):




Middle Axial Layer (Full and Long Partial Length Rods):



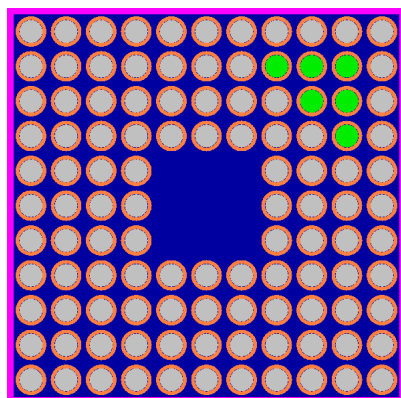
Top Axial Layer (Full Length Rods Only):



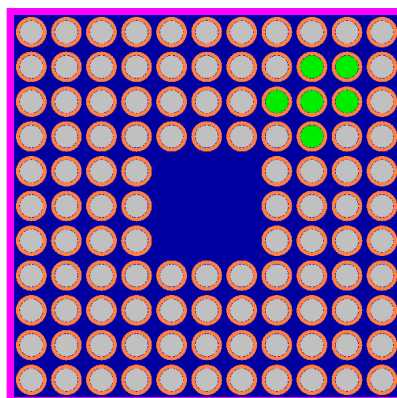
**Figure 6-74 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.9 wt% <sup>235</sup>U, 7 Gd Rods**

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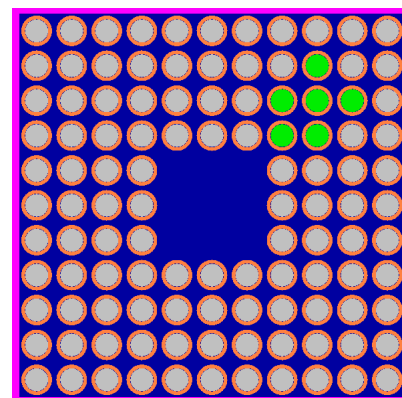
Bottom Axial Layer (Full and Partial Length Rods):



A2

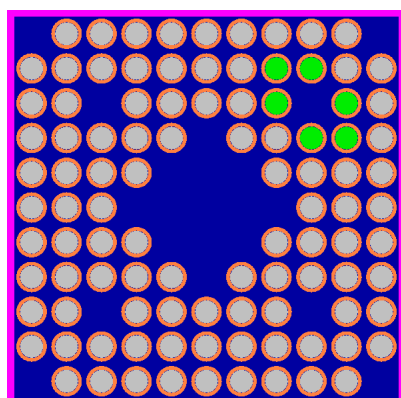


A3

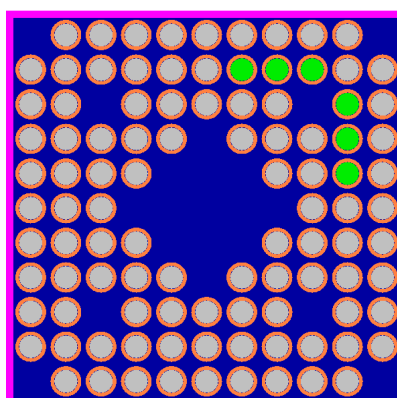


E1

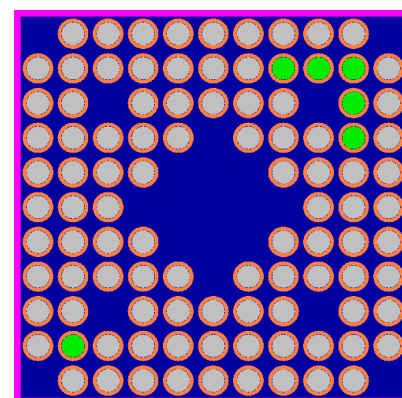
Middle Axial Layer (Full and Long Partial Length Rods):



A1

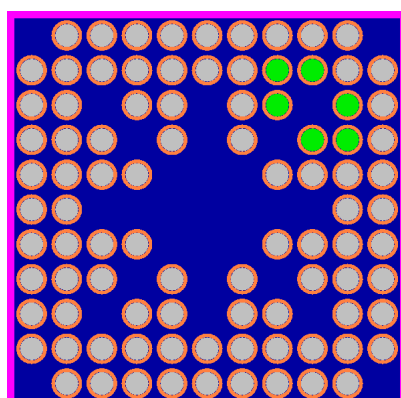


B1

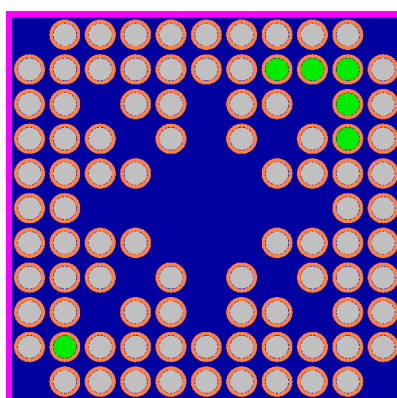


B3

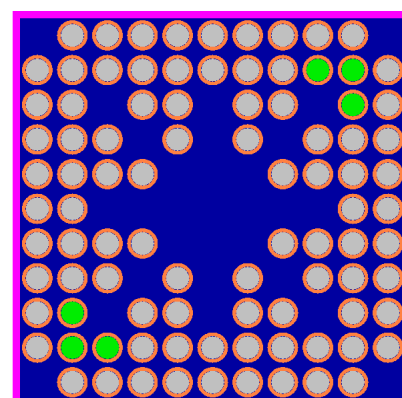
Top Axial Layer (Full Length Rods Only):



A1



A3

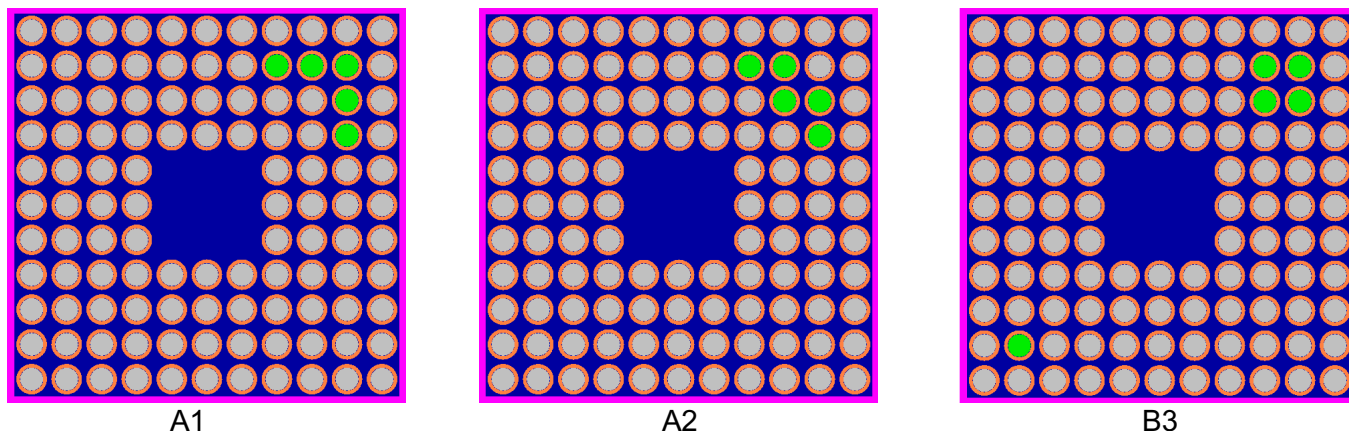


B2

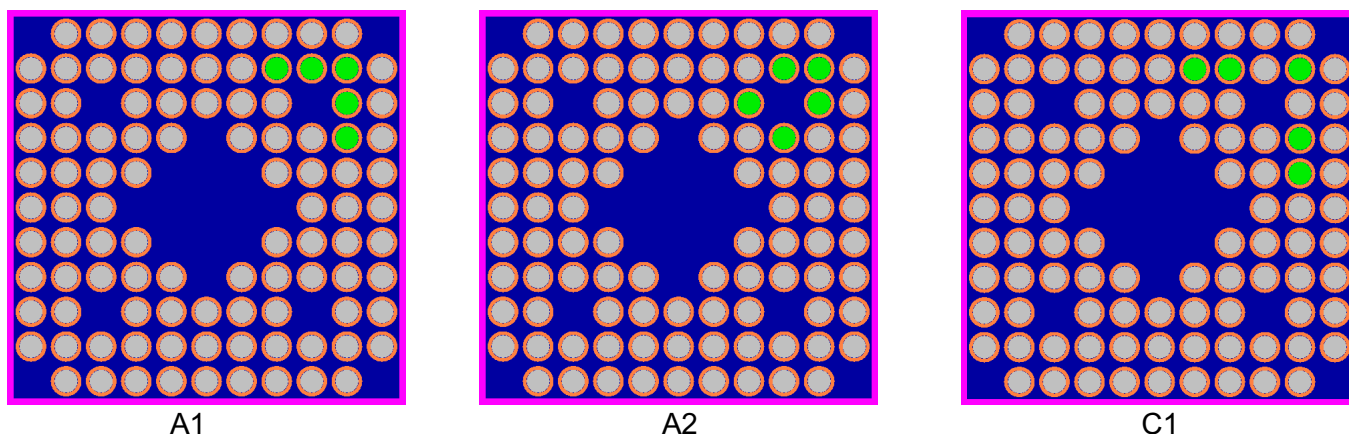
**Figure 6-75 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.8 wt%  $^{235}\text{U}$ , 6 Gd Rods**



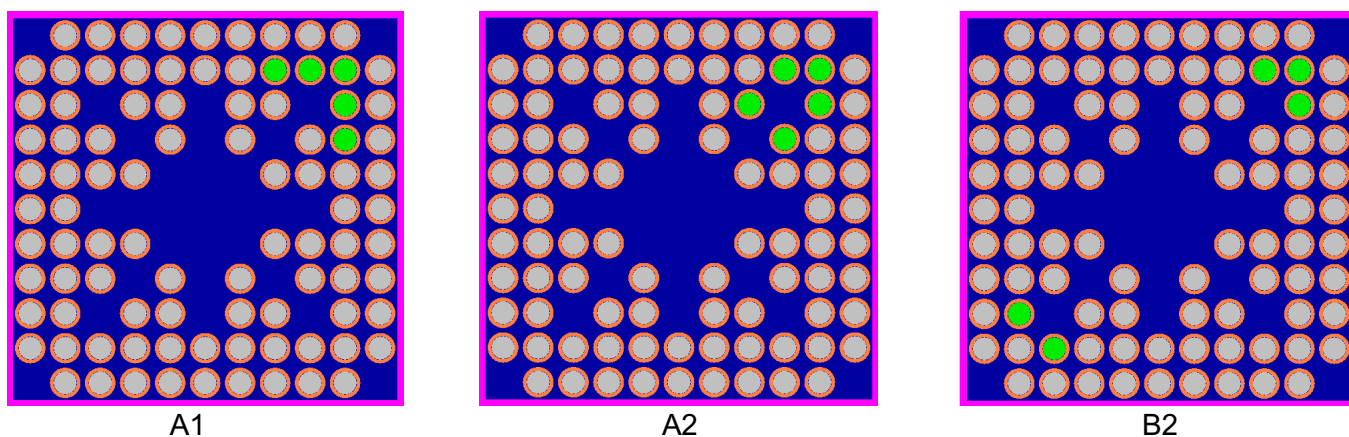
Bottom Axial Layer (Full and Partial Length Rods):



Middle Axial Layer (Full and Long Partial Length Rods):

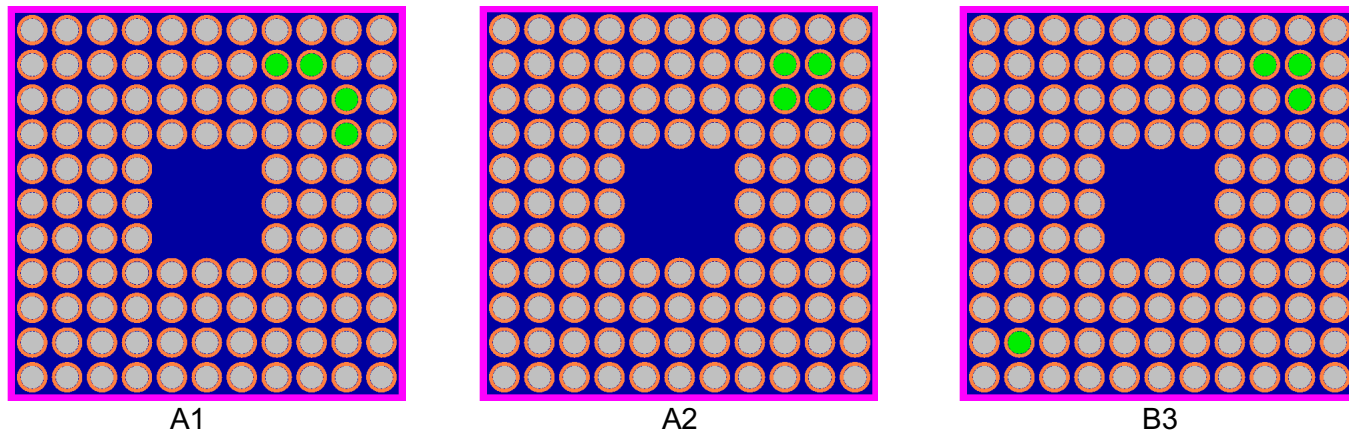


Top Axial Layer (Full Length Rods Only):

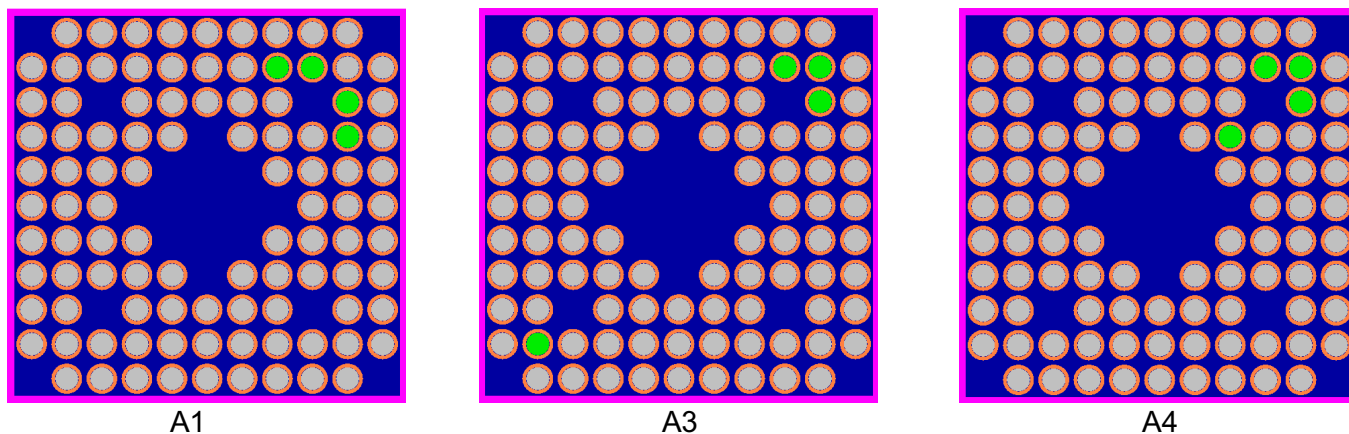


**Figure 6-76 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.6 wt% <sup>235</sup>U, 5 Gd Rods**

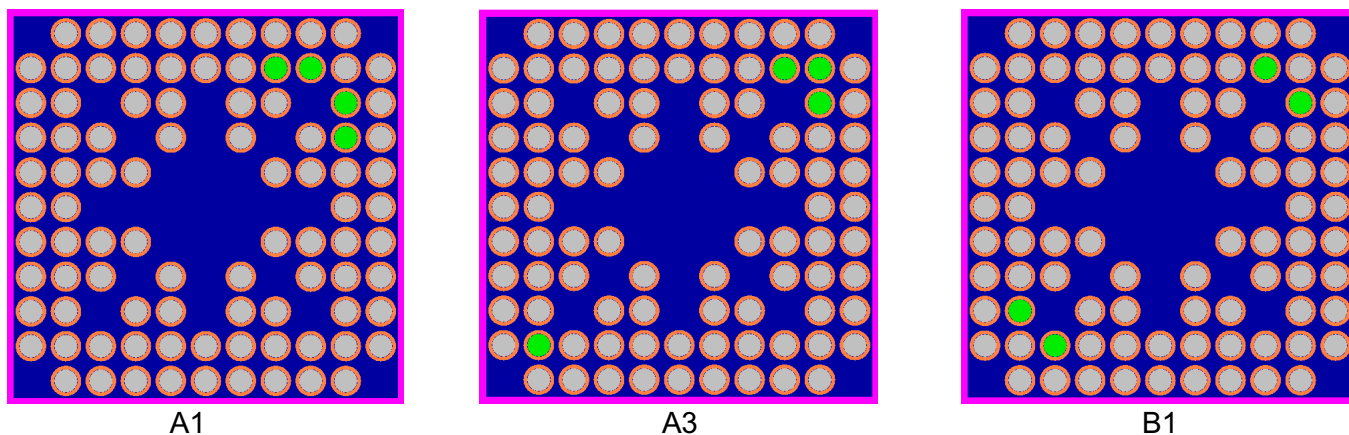
Bottom Axial Layer (Full and Partial Length Rods):



Middle Axial Layer (Full and Long Partial Length Rods):

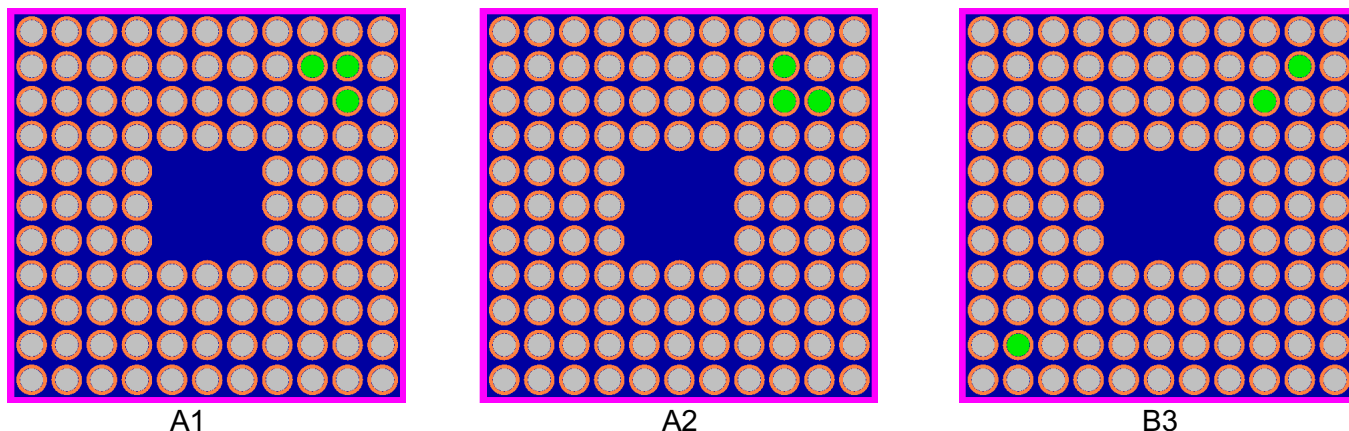


Top Axial Layer (Full Length Rods Only):

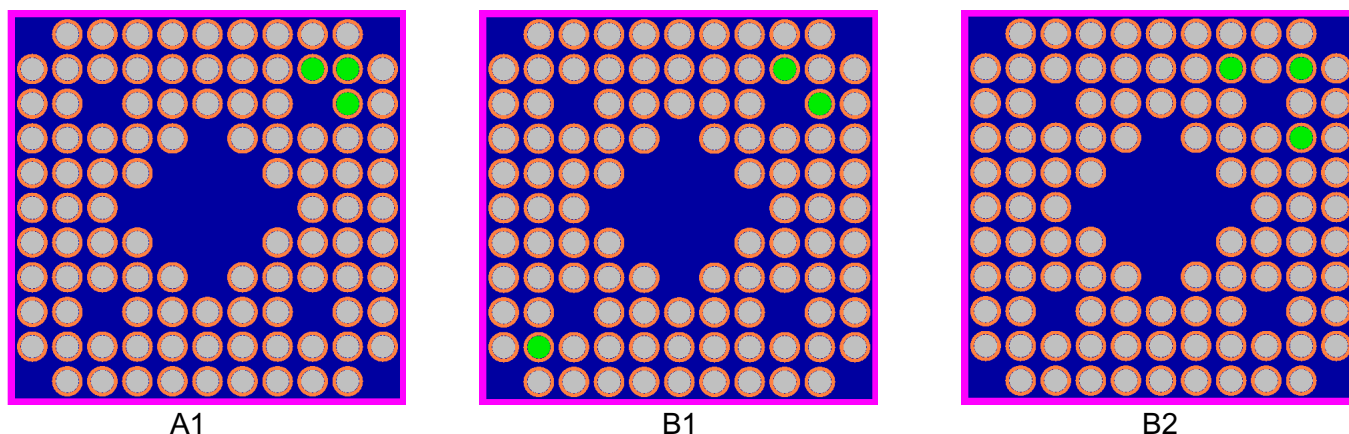


**Figure 6-77 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.5 wt% <sup>235</sup>U, 4 Gd Rods**

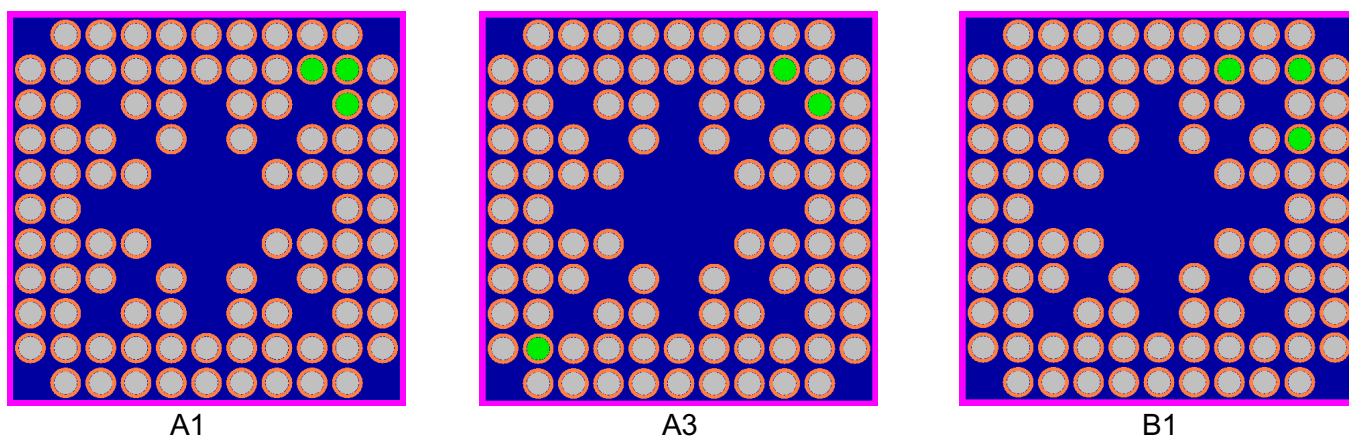
Bottom Axial Layer (Full and Partial Length Rods):



Middle Axial Layer (Full and Long Partial Length Rods):

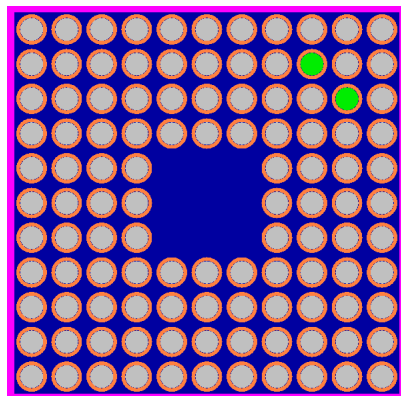


Top Axial Layer (Full Length Rods Only):

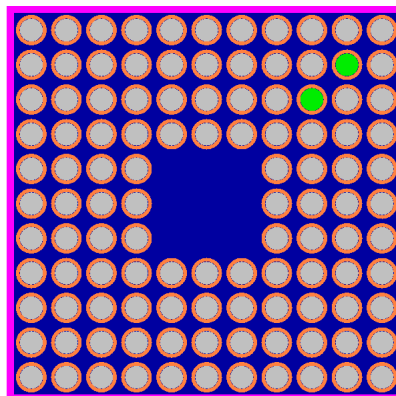


**Figure 6-78 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.3 wt% <sup>235</sup>U, 3 Gd Rods**

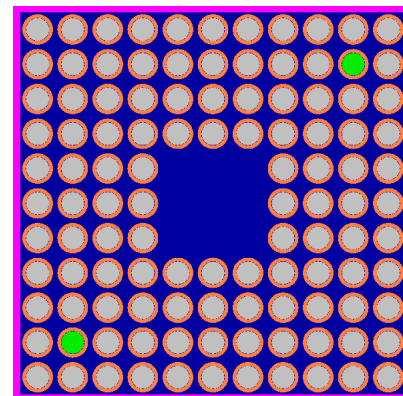
Bottom Axial Layer (Full and Partial Length Rods):



A1

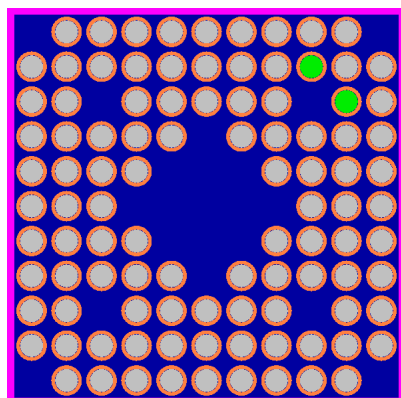


A2

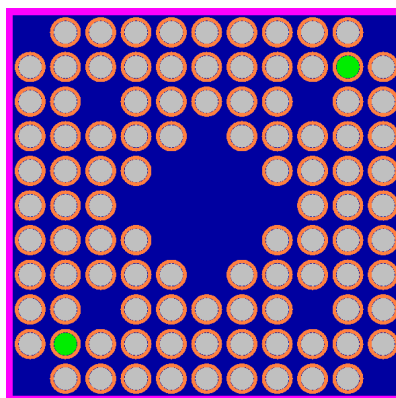


A4

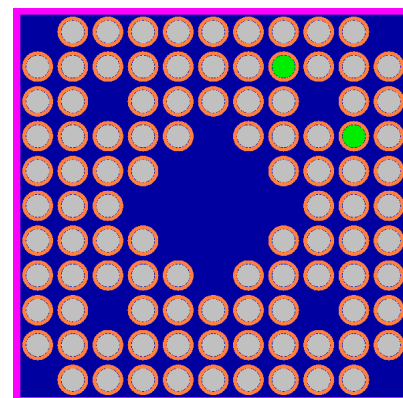
Middle Axial Layer (Full and Long Partial Length Rods):



A1

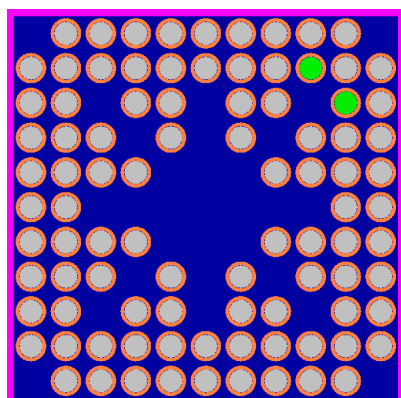


A4

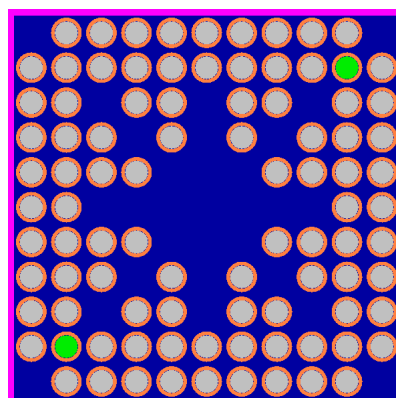


B2

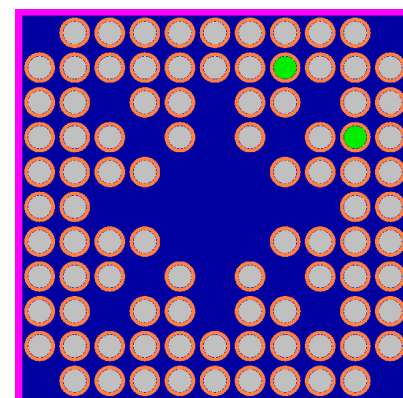
Top Axial Layer (Full Length Rods Only):



A1



A3



B1

**Figure 6-79 Most Reactive Gadolinia-Urania Fuel Rod Patterns: 3.2 wt% <sup>235</sup>U, 2 Gd Rods**



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#### 6.12.4. *Single Package Evaluation*

This section describes the evaluation of a single package model under NCT and HAC.

The fuel assembly model is described in Section 6.12.3.1.1, *Fuel Assembly Model*, and is based on the parametric study performed in Section 6.12.3.5, *Parameter Selection for 11x11 Fuel Assembly Model*.

##### 6.12.4.1. **NCT Single Package Evaluation**

In the NCT single package evaluation, the package is modeled using NCT geometry, as described in Section 6.12.3.1.2, *Single Package NCT Model with 11x11 Fuel*. Because the fuel is undamaged under NCT, the nominal fuel rod pitch of 1.195 cm is used. Also, the full-thickness polyethylene liner is in place, which centers the fuel assemblies in each compartment.

In the NCT models, the 10.2 kg of polyethylene allowed per fuel assembly is modeled as homogenized with the water between the fuel rods, in the water holes resulting from partial-length fuel rods, and within the center water channel. The volume fractions for the polyethylene and water materials in the moderator are calculated as follows:

$$\begin{aligned} \text{Total Assembly Volume} &= [(\# \text{ Fuel Rod Number/side}) * (\text{fuel rod pitch})]^2 * (\text{Full length Fuel Rod Length}) \\ \text{Volume}_{\text{assy}} &= (11 \text{ rods/side} * 1.195 \text{ cm})^2 * 385 \text{ cm} = 66524.54 \text{ cm}^3 \end{aligned}$$

$$\begin{aligned} \text{Fuel Rod Volume per Assembly} &= \# \text{ fuel rods} * \pi(\text{Fuel Rod Outer Clad})^2 * \text{Fuel Rod Length} \\ \text{Full length:} \quad \text{Volume}_{\text{fuel\_rod\_full}} &= 92 * \pi * (0.465 \text{ cm})^2 * 385 \text{ cm} = 24060.48 \text{ cm}^3 \\ \text{Long Partial length:} \quad \text{Volume}_{\text{fuel\_rod\_LPLR}} &= 8 * \pi * (0.465 \text{ cm})^2 * 236.8 \text{ cm} = 1286.85 \text{ cm}^3 \\ \text{Short Partial length:} \quad \text{Volume}_{\text{fuel\_rod\_SPLR}} &= 12 * \pi * (0.465 \text{ cm})^2 * 155.1 \text{ cm} = 1264.30 \text{ cm}^3 \\ \text{Volume}_{\text{fuel}} &= \text{Volume}_{\text{fuel\_rod\_full}} + \text{Volume}_{\text{fuel\_rod\_LPLR}} + \text{Volume}_{\text{fuel\_rod\_SPLR}} \\ \text{Volume}_{\text{fuel}} &= 24060.48 \text{ cm}^3 + 1286.85 \text{ cm}^3 + 1264.30 \text{ cm}^3 = 26611.63 \text{ cm}^3 \end{aligned}$$

$$\begin{aligned} \text{Water + Polyethylene Volume per Assembly} &= \text{Volume}_{\text{assy}} - \text{Volume}_{\text{fuel}} \\ \text{Volume}_{\text{h2o+poly}} &= 66524.54 \text{ cm}^3 - 26611.63 \text{ cm}^3 = 39912.92 \text{ cm}^3 \end{aligned}$$

$$\begin{aligned} \text{Total Polyethylene Volume} &= \text{Total Polyethylene Mass} / \text{Polyethylene Density} \\ \text{Volume}_{\text{poly}} &= 10.2 \text{ kg} / (0.949 \text{ g} / \text{cm}^3) = 10748.16 \text{ cm}^3 \end{aligned}$$

$$\begin{aligned} \text{Water Volume Fraction} &= [(\text{Water + Polyethylene Volume}) - (\text{Polyethylene Volume})] / (\text{Water + Polyethylene Volume}) \\ \text{VF}_{\text{h2o}} &= (39912.92 \text{ cm}^3 - 10748.16 \text{ cm}^3) / (39912.92 \text{ cm}^3) = \mathbf{0.73071} \end{aligned}$$

$$\begin{aligned} \text{Polyethylene Volume Fraction} &= \text{Polyethylene Volume} / (\text{Water + Polyethylene Volume}) \\ \text{VF}_{\text{poly}} &= 10748.16 \text{ cm}^3 / (39912.92 \text{ cm}^3) = \mathbf{0.26929} \end{aligned}$$

In the NCT model, void is modeled in the pellet-to-cladding gap.

Since the foam liner remains intact for NCT, the full thickness liner is used in the model. A moderator density study, in which the density of the water within the inner container is varied, is performed for the single package model for both channeled and un-channeled fuel assemblies. The region between the inner and outer containers is modeled as full-density water to maximize reflection.

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Results from the NCT single package model are provided in Table 6-80 11x11 Single Package NCT Results. The most reactive configuration is observed for the channeled fuel assemblies for a moderator density of 1.0 g/cm<sup>3</sup> within the inner container. The maximum  $k_{\text{eff}} + 2\sigma$  value for the single package normal conditions of transport case is 0.63166, which is far below the USL of 0.94094. This case also meets the requirements of 71.55(b). Therefore, criticality safety is established for the NCT single package TN-B1 container.

The assembly containing varying axial enrichment was also considered in the final configuration for the NCT single package model. The  $k_{\text{eff}} + 2\sigma$  value calculated for the 5-5-3.3wt% assembly was 0.63085, which is lower than that found for the assembly with uniform axial enrichment. Therefore, criticality safety is maintained for the NCT single package TN-B1 container for assemblies having varying axial enrichment.

#### 6.12.4.2. HAC Single Package Evaluation

In the HAC single package evaluation, the package is modeled using HAC geometry, as described in Section 6.12.3.1.3, *Single Package HAC Model with 11x11 Fuel*. Because the fuel could be damaged under HAC, the fuel rod pitch is expanded 5% to 1.2548 cm. In the HAC models, the 10.2 kg of polyethylene is assumed to be smeared into the fuel rod cladding. The fuel assemblies are allowed to move within the inner container compartments. Also, water is modeled in the pellet-to-cladding gap.

The Alumina Silicate thermal insulation is modeled between the inner and outer walls. This is consistent with the physical condition of the TN-B1 shipping container after being subjected to the tests specified in 10 CFR Part 71 (see Sections 3.2.2 and 3.5.2).

In an initial series of cases, the polyethylene foam liner is assumed to completely burn away. Full density water that provides more reflection capability is assumed to flood the TN-B1 inner container fuel compartment. Since the foam liner is assumed to burn away, the most reactive assembly orientation from Section 6.12.3.5.1, *Fuel Assembly Orientation Study*, is used: orientation 7 with the assembly in the right compartment shifted toward the center of the container. An evaluation to determine the most reactive configuration regarding the presence of the assembly channel is performed. The results are provided in Table 6-81 11x11 Single Package HAC Results, Channel Study. The most reactive configuration for the HAC single package model is found for the unchanneled assemblies.

Next, a moderator density study is performed without the fuel channel. Full density water is modeled within the outer container to maximize reflection and the moderator density is varied only within the inner container. Results are provided in Table 6-82 11x11 Single Package HAC Results, Complete Foam Burn. The maximum  $k_{\text{eff}} + 2\sigma$  value is 0.76705.

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Because the HAC array cases are more reactive with liner partial foam burn rather than complete foam burn (see Section 6.12.3.5.10, *Polyethylene Foam Liner Study*), additional cases are investigated in which only partial foam burn is modeled. For each of the orientations shown below, a study is performed for both a channeled and unchanneled fuel assembly:

1. Both assemblies centered within the compartment.
2. The assembly in the left compartment centered and the assembly in the right compartment shifted toward the left.
3. Both assemblies shifted toward the center of the package.

Results for these three configurations are provided in Table 6-83 11x11 Single Package HAC Results, Partial Foam Burn, Centered Assemblies, Table 6-84 11x11 Single Package HAC Results, Partial Foam Burn, Right Assembly Shifted, and Table 6-85 11x11 Single Package HAC Results, Partial Foam Burn, Both Assemblies Shifted. It is observed that for the HAC single package, complete foam burn is more reactive than partial foam burn, as the liner region is acting primarily as a reflector. Conversely, in the HAC array, lower-density foam in this region increases neutron interactions between packages because the neutron mean free path is much higher in the low-density foam compared to water, thus increasing the reactivity.

The maximum  $k_{\text{eff}} + 2\sigma$  value for the HAC single package is 0.76705 (complete foam burn), which is far below the USL of 0.94094. Therefore, criticality safety is established for the HAC single package TN-B1 container.


The assembly containing varying axial enrichment was also considered in the final configuration for the HAC single package model. The  $k_{\text{eff}} + 2\sigma$  value calculated for the 5-5-3.3wt% assembly was 0.76703, which is lower than that found for the assembly with uniform axial enrichment. Therefore, criticality safety is maintained for the HAC single package TN-B1 container for assemblies having varying axial enrichment.

A second modeling approach in which the melted polyethylene is smeared into the moderator instead of the clad material was considered. Using the uniform axial configuration, the  $k_{\text{eff}} + 2\sigma$  value calculated for this approach was 0.76595, which is lower than that found for the modeling approach in which the polyethylene is smeared into the clad.

#### 6.12.4.3. Single Package Results

The NCT and HAC single package results are provided in Table 6-80 11x11 Single Package NCT Results through Table 6-85 11x11 Single Package HAC Results, Partial Foam Burn, Both Assemblies Shifted.




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**Table 6-80 11x11 Single Package NCT Results**

Filename	Moderator Density (g/cm <sup>3</sup> )	Fuel Channel?	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
NCT_single_pkg_dens000_ch.out	0.00	yes	0.39691	0.00035	0.39761
NCT_single_pkg_dens002_ch.out	0.02	yes	0.40084	0.00034	0.40152
NCT_single_pkg_dens004_ch.out	0.04	yes	0.40647	0.00035	0.40717
NCT_single_pkg_dens006_ch.out	0.06	yes	0.41135	0.00035	0.41205
NCT_single_pkg_dens008_ch.out	0.08	yes	0.41558	0.00034	0.41626
NCT_single_pkg_dens010_ch.out	0.10	yes	0.42054	0.00038	0.42130
NCT_single_pkg_dens020_ch.out	0.20	yes	0.44513	0.00038	0.44589
NCT_single_pkg_dens040_ch.out	0.40	yes	0.49541	0.00041	0.49623
NCT_single_pkg_dens060_ch.out	0.60	yes	0.54417	0.00039	0.54495
NCT_single_pkg_dens080_ch.out	0.80	yes	0.58873	0.00042	0.58957
<b>NCT_single_pkg_dens100_ch.out</b>	<b>1.0</b>	<b>yes</b>	<b>0.63082</b>	<b>0.00042</b>	<b>0.63166</b>
NCT_single_pkg_dens000_nc.out	0.00	no	0.39511	0.00035	0.39581
NCT_single_pkg_dens002_nc.out	0.02	no	0.39933	0.00033	0.39999
NCT_single_pkg_dens004_nc.out	0.04	no	0.40421	0.00032	0.40485
NCT_single_pkg_dens006_nc.out	0.06	no	0.40812	0.00037	0.40886
NCT_single_pkg_dens008_nc.out	0.08	no	0.41323	0.00036	0.41395
NCT_single_pkg_dens010_nc.out	0.10	no	0.41763	0.00035	0.41833
NCT_single_pkg_dens020_nc.out	0.20	no	0.44306	0.00036	0.44378
NCT_single_pkg_dens040_nc.out	0.40	no	0.49112	0.00042	0.49196
NCT_single_pkg_dens060_nc.out	0.60	no	0.53836	0.00044	0.53924
NCT_single_pkg_dens080_nc.out	0.80	no	0.58375	0.00043	0.58461
NCT_single_pkg_dens100_nc.out	1.0	no	0.62383	0.00045	0.62473

**Table 6-81 11x11 Single Package HAC Results, Channel Study**

Filename	Fuel Channel?	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
HAC_single_pkg_5wt13gd_NL_shft_ch.out	yes	0.75955	0.00046	0.76047
<b>HAC_single_pkg_5wt13gd_NL_shft_un.out</b>	<b>no</b>	<b>0.76615</b>	<b>0.00045</b>	<b>0.76705</b>


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**Table 6-82 11x11 Single Package HAC Results, Complete Foam Burn**

Filename	Moderator Density (g/cm <sup>3</sup> )	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
HAC_single_dens000.out	0.00	0.40245	0.00032	0.40309
HAC_single_dens002.out	0.02	0.40749	0.00037	0.40823
HAC_single_dens004.out	0.04	0.41194	0.00034	0.41262
HAC_single_dens006.out	0.06	0.41873	0.00034	0.41941
HAC_single_dens008.out	0.08	0.42450	0.00039	0.42528
HAC_single_dens010.out	0.10	0.43117	0.00036	0.43189
HAC_single_dens020.out	0.20	0.46946	0.00044	0.47034
HAC_single_dens040.out	0.40	0.55640	0.00041	0.55722
HAC_single_dens060.out	0.60	0.63868	0.00045	0.63958
HAC_single_dens080.out	0.80	0.70904	0.00049	0.71002
<b>HAC_single_dens100.out</b>	<b>1.0</b>	<b>0.76615</b>	<b>0.00045</b>	<b>0.76705</b>

**Table 6-83 11x11 Single Package HAC Results, Partial Foam Burn, Centered Assemblies**

Filename	Fuel Channel?	Polyethylene liner thickness (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
HAC_single_liner_000_cc.out	yes	0.00	0.74267	0.00044	0.74355
HAC_single_liner_020_cc.out	yes	0.20	0.73742	0.00048	0.73838
HAC_single_liner_040_cc.out	yes	0.40	0.73191	0.00048	0.73287
HAC_single_liner_060_cc.out	yes	0.60	0.72604	0.00045	0.72694
HAC_single_liner_080_cc.out	yes	0.80	0.71675	0.00047	0.71769
HAC_single_liner_100_cc.out	yes	1.00	0.70796	0.00046	0.70888
HAC_single_liner_120_cc.out	yes	1.20	0.69876	0.00046	0.69968
HAC_single_liner_140_cc.out	yes	1.40	0.68730	0.00045	0.68820
HAC_single_liner_160_cc.out	yes	1.60	0.67801	0.00046	0.67893
HAC_single_liner_full_cc.out	yes	1.657	0.67415	0.00043	0.67501
HAC_single_liner_000_cn.out	no	0.00	0.74453	0.00044	0.74541
HAC_single_liner_020_cn.out	no	0.20	0.74112	0.00047	0.74206
HAC_single_liner_040_cn.out	no	0.40	0.73562	0.00052	0.73666
HAC_single_liner_060_cn.out	no	0.60	0.72960	0.00043	0.73046
HAC_single_liner_080_cn.out	no	0.80	0.72258	0.00046	0.72350
HAC_single_liner_100_cn.out	no	1.00	0.71394	0.00047	0.71488
HAC_single_liner_120_cn.out	no	1.20	0.70495	0.00047	0.70589
HAC_single_liner_140_cn.out	no	1.40	0.69502	0.00045	0.69592
HAC_single_liner_160_cn.out	no	1.60	0.68508	0.00046	0.68600
HAC_single_liner_180_cn.out	no	1.80	0.67373	0.00046	0.67465
HAC_single_liner_full_cn.out	no	1.911	0.66622	0.00044	0.66710

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**Table 6-84 11x11 Single Package HAC Results, Partial Foam Burn, Right Assembly Shifted**

Filename	Fuel Channel?	Polyethylene liner thickness (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
HAC single liner 000 o7c.out	yes	0.00	0.75663	0.00051	0.75765
HAC single liner 020 o7c.out	yes	0.20	0.74806	0.00048	0.74902
HAC single liner 040 o7c.out	yes	0.40	0.73805	0.00044	0.73893
HAC single liner 060 o7c.out	yes	0.60	0.72806	0.00045	0.72896
HAC single liner 080 o7c.out	yes	0.80	0.71742	0.00043	0.71828
HAC single liner 100 o7c.out	yes	1.00	0.70778	0.00049	0.70876
HAC single liner 120 o7c.out	yes	1.20	0.69759	0.00049	0.69857
HAC single liner 140 o7c.out	yes	1.40	0.68722	0.00048	0.68818
HAC single liner 160 o7c.out	yes	1.60	0.67707	0.00041	0.67789
HAC single liner full o7c.out	yes	1.657	0.67415	0.00043	0.67501
HAC single liner 000 o7n.out	no	0.00	0.76504	0.00045	0.76594
HAC single liner 020 o7n.out	no	0.20	0.75573	0.00046	0.75665
HAC single liner 040 o7n.out	no	0.40	0.74562	0.00051	0.74664
HAC single liner 060 o7n.out	no	0.60	0.73538	0.00048	0.73634
HAC single liner 080 o7n.out	no	0.80	0.72495	0.00048	0.72591
HAC single liner 100 o7n.out	no	1.00	0.71591	0.00051	0.71693
HAC single liner 120 o7n.out	no	1.20	0.70519	0.00047	0.70613
HAC single liner 140 o7n.out	no	1.40	0.69323	0.00048	0.69419
HAC single liner 160 o7n.out	no	1.60	0.68341	0.00042	0.68425
HAC single liner 180 o7n.out	no	1.80	0.67444	0.00044	0.67532
HAC single liner full o7n.out	no	1.911	0.66622	0.00044	0.66710

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**Table 6-85 11x11 Single Package HAC Results, Partial Foam Burn, Both Assemblies Shifted**


Filename	Fuel Channel?	Polyethylene liner thickness (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
HAC single liner 000 INc.out	yes	0.00	0.75356	0.00046	0.75448
HAC single liner 020 INc.out	yes	0.20	0.74478	0.00045	0.74568
HAC single liner 040 INc.out	yes	0.40	0.73510	0.00053	0.73616
HAC single liner 060 INc.out	yes	0.60	0.72556	0.00048	0.72652
HAC single liner 080 INc.out	yes	0.80	0.71580	0.00046	0.71672
HAC single liner 100 INc.out	yes	1.00	0.70685	0.00049	0.70783
HAC single liner 120 INc.out	yes	1.20	0.69602	0.00043	0.69688
HAC single liner 140 INc.out	yes	1.40	0.68519	0.00045	0.68609
HAC single liner 160 INc.out	yes	1.60	0.67717	0.00046	0.67809
HAC single liner full INc.out	yes	1.657	0.67415	0.00043	0.67501
HAC single liner 000 INn.out	no	0.00	0.76073	0.00047	0.76167
HAC single liner 020 INn.out	no	0.20	0.75191	0.00046	0.75283
HAC single liner 040 INn.out	no	0.40	0.74247	0.00042	0.74331
HAC single liner 060 INn.out	no	0.60	0.73316	0.00046	0.73408
HAC single liner 080 INn.out	no	0.80	0.72331	0.00045	0.72421
HAC single liner 100 INn.out	no	1.00	0.71371	0.00044	0.71459
HAC single liner 120 INn.out	no	1.20	0.70294	0.00048	0.70390
HAC single liner 140 INn.out	no	1.40	0.69257	0.00045	0.69347
HAC single liner 160 INn.out	no	1.60	0.68392	0.00047	0.68486
HAC single liner 180 INn.out	no	1.80	0.67264	0.00048	0.67360
HAC single liner full INn.out	no	1.911	0.66622	0.00044	0.66710

## 6.12.5. Evaluation of Package Arrays Under Normal Conditions of Transport

### 6.12.5.1. Configuration

The NCT package array model is described in Section 6.12.3.1.4, *NCT Array Model with 11x11 Fuel*. The NCT model consists of a 21x3x24 array of containers, surrounded by a 30.48-cm layer of full density water for reflection. The container array is fully flooded with water at a density needed to achieve the most reactive configuration. The model is comprised of both the inner and outer containers fabricated from Stainless Steel. The inner container has Alumina Silicate thermal insulation between the inner and outer walls. No credit is taken for any of the structural steel between the inner and outer containers.

The worst case fuel assembly parameters from Section 6.12.3.5, *Parameter Selection for 11x11 Fuel Assembly Model*, are used in the package array model. Nominal pitch is utilized to reflect the NCT condition. The package array evaluation is made for the most reactive lattice with axial uniform enrichment: average lattice enrichment of 5.0 wt% and thirteen 2.0 wt% gadolinia fuel rods. Void is modeled in the pellet-to-cladding gap.


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For the NCT package array evaluation, the fuel assemblies are held centered within the inner container compartments by the polyethylene liner. The top and sides of the liner are modeled with 0.08 g/cm<sup>3</sup> polyethylene, while the bottom liner is modeled with 0.16 g/cm<sup>3</sup> polyethylene. The maximum allowable polyethylene equivalent mass (10.2 kg) is smeared into the water region surrounding the fuel rods, in the water holes resulting from partial-length fuel rods, and within the center water channel. The volume fractions for the polyethylene and water materials in the moderator are provided in Section 6.12.3.6.1, *NCT Single Package Evaluation*.

Since the foam liner remains intact for NCT, the full thickness liner is used in the model. A moderator density study, in which the density of the water within the inner and outer containers is varied, is performed for the package array model for both channeled and un-channeled fuel assemblies.

Results from the NCT package array model are provided in Table 6-86 11x11 NCT Array Results. The most reactive configuration is observed for the channeled fuel assemblies. The reactivity peaks with no water within the package, although moderation is provided by the 10.2 kg of polyethylene. The maximum  $k_{\text{eff}} + 2\sigma$  value obtained for the package array for normal conditions of transport case is 0.85383, which is below the USL of 0.94094. Therefore, criticality safety of the TN-B1 shipping container is demonstrated under normal conditions of transport.

The assembly containing varying axial enrichment was also considered in the final configuration for the NCT package array model. The  $k_{\text{eff}} + 2\sigma$  value calculated for the 5-5-3.3wt% assembly was 0.84759, which is lower than that found for the assembly with uniform axial enrichment. Therefore, criticality safety is maintained for the NCT single package TN-B1 container for assemblies having varying axial enrichment.

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### 6.12.5.2. Results

The NCT array results are provided in Table 6-86 11x11 NCT Array Results.

**Table 6-86 11x11 NCT Array Results**

Filename	Moderator Density (g/cm <sup>3</sup> )	Fuel Channel?	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
<b>NCT_array_pkg_dens000_ch.out</b>	<b>0.00</b>	<b>yes</b>	<b>0.85303</b>	<b>0.00040</b>	<b>0.85383</b>
NCT_array_pkg_dens002_ch.out	0.02	yes	0.79296	0.00042	0.79380
NCT_array_pkg_dens004_ch.out	0.04	yes	0.74358	0.00045	0.74448
NCT_array_pkg_dens006_ch.out	0.06	yes	0.70552	0.00041	0.70634
NCT_array_pkg_dens008_ch.out	0.08	yes	0.67097	0.00039	0.67175
NCT_array_pkg_dens010_ch.out	0.10	yes	0.64318	0.00042	0.64402
NCT_array_pkg_dens020_ch.out	0.20	yes	0.55071	0.00039	0.55149
NCT_array_pkg_dens040_ch.out	0.40	yes	0.51469	0.00037	0.51543
NCT_array_pkg_dens060_ch.out	0.60	yes	0.54575	0.00043	0.54661
NCT_array_pkg_dens080_ch.out	0.80	yes	0.58996	0.00040	0.59076
NCT_array_pkg_dens100_ch.out	1.00	yes	0.63123	0.00049	0.63221
NCT_array_pkg_dens000_nc.out	0.00	no	0.83671	0.00041	0.83753
NCT_array_pkg_dens002_nc.out	0.02	no	0.78267	0.00036	0.78339
NCT_array_pkg_dens004_nc.out	0.04	no	0.73327	0.00039	0.73405
NCT_array_pkg_dens006_nc.out	0.06	no	0.69606	0.00040	0.69686
NCT_array_pkg_dens008_nc.out	0.08	no	0.66290	0.00037	0.66364
NCT_array_pkg_dens010_nc.out	0.10	no	0.63378	0.00042	0.63462
NCT_array_pkg_dens020_nc.out	0.20	no	0.53825	0.00039	0.53903
NCT_array_pkg_dens040_nc.out	0.40	no	0.50125	0.00038	0.50201
NCT_array_pkg_dens060_nc.out	0.60	no	0.53365	0.00040	0.53445
NCT_array_pkg_dens080_nc.out	0.80	no	0.57891	0.00041	0.57973
NCT_array_pkg_dens100_nc.out	1.00	no	0.62302	0.00042	0.62386

### 6.12.6. Evaluation of Package Arrays Under Hypothetical Accident Conditions

#### 6.12.6.1. Configuration

The parameter study documented in Section 6.12.3.5, *Parameter Selection for 11x11 Fuel Assembly Model*, is performed for an HAC array of 10x1x10 packages. Therefore, the cases documented in that section are part of the HAC array analysis. The conclusions are summarized below:

- Assembly orientation 6 (fuel assemblies centered for partial foam burn)
- 0.254 cm thick fuel assembly zirconium channel included
- Zirconium water channel not included

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- 10.2 kg polyethylene per fuel assembly (smeared into the cladding)
- Pitch increased 5% to 1.2548 cm
- Pellet diameter of 0.820 cm
- Cladding inner diameter of 0.840 cm
- Cladding outer diameter of 0.930 cm
- Fuel assemblies fully flooded (i.e, no uncovered fuel rods)
- Thermal insulator modeled as Alumina-Silica
- Polyethylene foam liner modeled as partially burned with a thickness of 1.2 cm
- Enrichment of 5.0% with 13 gadolinia-urania rods with the loading pattern shown in Figure 6-52 Bounding Fuel Assembly Model for the uniform axial enrichment.

The maximum  $k_{\text{eff}} + 2\sigma$  value for the HAC package array case with uniform axial enrichment from Table 6-79 11x11 Gadolinia-Urania Analysis Results is 0.93810, which is below the USL of 0.94094. This case is developed with full-water moderation of the inner container and void between the inner and outer containers, which is assumed to be the most reactive moderation condition. This assumption is proven in the following sets of cases.


In the first set of cases, the moderator density within the inner container is varied while void is modeled between the inner and outer containers. In the second set of cases, the moderator density within the inner container is modeled as full-density water while variable density water is modeled between the inner and outer containers. The results are provided in Table 6-87 11x11 HAC Array Results. It is observed that the most reactive condition is with a fully moderated inner container and void outer container. When water is added to the outer container, the reactivity drops considerably, as the water in the outer container effectively isolates the packages from one another.

Note that in the calculations uranium in the mixture is modeled as only U-235 and U-238. However, as indicated in Table 1-3,  $3.04 \times 10^{-9}$  gPu-239/gU may be present in the fuel matrix. Because Pu-239 is fissile, it is added to the worst case package array HAC model with uniform axial enrichment.

The results indicate no statistically significant difference between the cases with and without plutonium. The  $k_{\text{eff}} + 2\sigma$  for the case with plutonium is 0.93871. The  $k_{\text{eff}} + 2\sigma$  for the case without plutonium is 0.93810. Therefore, the plutonium is justifiably neglected in the TN-B1 evaluation.

As noted in Section 6.12.3.5.11 and Table 6-79 11x11 Gadolinia-Urania Analysis Results, the assembly containing varying axial enrichment resulted in a slightly higher reactivity than did the assembly with axial uniform enrichment. The maximum  $k_{\text{eff}} + 2\sigma$  value for the HAC package




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array case with varying axial enrichment from Table 6-79 11x11 Gadolinia-Urania Analysis Results is 0.93943, which is below the USL of 0.94094. This case is developed with full-water moderation of the inner container and void between the inner and outer containers, which is assumed to be the most reactive moderation condition. Therefore, criticality safety is maintained for the HAC package array for assemblies having varying axial enrichment.

A second modeling approach in which the melted polyethylene is smeared into the moderator instead of the clad material was considered. Using the varying axial enrichment configuration, the  $k_{\text{eff}} + 2\sigma$  value calculated for this approach was 0.94068, which is higher than that found for the modeling approach in which the polyethylene is smeared into the clad. This value remains below the USL of 0.94094 and criticality safety is maintained for the HAC package array.

#### 6.12.6.2. Results

Results for the HAC array cases are provided in Table 6-87 11x11 HAC Array Results.

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**Table 6-87 11x11 HAC Array Results**

Filename	Inner Container Moderator Density (g/cm <sup>3</sup> )	Outer Region Water Density (g/cm <sup>3</sup> )	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
HAC_array_dens000.out	0.00	0.00	0.62068	0.00040	0.62148
HAC_array_dens002.out	0.02	0.00	0.63438	0.00036	0.63510
HAC_array_dens004.out	0.04	0.00	0.64665	0.00036	0.64737
HAC_array_dens006.out	0.06	0.00	0.65966	0.00037	0.66040
HAC_array_dens008.out	0.08	0.00	0.67109	0.00040	0.67189
HAC_array_dens010.out	0.10	0.00	0.68278	0.00047	0.68372
HAC_array_dens020.out	0.20	0.00	0.73465	0.00045	0.73555
HAC_array_dens040.out	0.40	0.00	0.81370	0.00052	0.81474
HAC_array_dens060.out	0.60	0.00	0.87019	0.00050	0.87119
HAC_array_dens080.out	0.80	0.00	0.90958	0.00044	0.91046
<b>HAC_array_dens100.out</b>	<b>1.0</b>	<b>0.00</b>	<b>0.93710</b>	<b>0.00050</b>	<b>0.93810</b>
<b>HAC_array_outH2O_dens000.out</b>	<b>1.0</b>	<b>0.00</b>	<b>0.93710</b>	<b>0.00050</b>	<b>0.93810</b>
HAC_array_outH2O_dens002.out	1.0	0.02	0.91440	0.00047	0.91534
HAC_array_outH2O_dens004.out	1.0	0.04	0.88722	0.00045	0.88812
HAC_array_outH2O_dens006.out	1.0	0.06	0.86099	0.00047	0.86193
HAC_array_outH2O_dens008.out	1.0	0.08	0.83691	0.00045	0.83781
HAC_array_outH2O_dens010.out	1.0	0.10	0.81614	0.00075	0.81764
HAC_array_outH2O_dens020.out	1.0	0.20	0.75097	0.00046	0.75189
HAC_array_outH2O_dens040.out	1.0	0.40	0.70736	0.00045	0.70826
HAC_array_outH2O_dens060.out	1.0	0.60	0.69956	0.00045	0.70046
HAC_array_outH2O_dens080.out	1.0	0.80	0.69831	0.00046	0.69923
HAC_array_outH2O_dens100.out	1.0	1.0	0.69970	0.00044	0.70058

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### 6.12.7. *Transport of 11x11 Rods*

The analysis for transport of 8x8, 9x9, and 10x10 fuel assembly rods within the TN-B1 is documented in *Section 6.7, Fuel Rod Transport in the TN-B1*. However, a review of Table 6-2 TN-B1 Fuel Rod Loading Criteria indicates the 11x11 fuel rod is outside the criteria set in terms of theoretical density and fuel rod outer diameter. All other parameters fall within the values provided for the 10x10 UO<sub>2</sub> fuel rods. Therefore, studies are conducted to allow transport of the 11x11 fuel assembly UO<sub>2</sub> fuel rods in the TN-B1 container. Two configurations are investigated, which include the loose fuel rods and fuel rods contained in a 5-inch stainless steel pipe. It was shown in *Section 6.7, Fuel Rod Transport in the TN-B1*, that these two configurations bound the configurations in which fuel rods are bundled together or contained within a protective case.

Fuel parameters modeled to reflect the 11x11 rod worst case parameters are shown below:

UO<sub>2</sub> pellet density = 10.763 g/cm<sup>3</sup>

UO<sub>2</sub> enrichment = 5.0 wt% <sup>235</sup>U

Pellet diameter = 0.820 cm

Rod outer diameter = 0.930 cm

Cladding thickness = 0.00 cm

Fuel length = 385 cm

Similar to *Section 6.7, Fuel Rod Transport in the TN-B1*, a 6-mil layer of polyethylene encircles each fuel rod in the model to bound protective packing material that may be used for fuel rod transport. Moderator is modeled between the fuel and polyethylene. The cladding is conservatively modeled as water to further increase moderation. Water is modeled in the pellet/cladding gap.

The approach is to investigate 25 loose rods in each liner cavity and 30 rods in each pipe component under HAC array conditions, and select the most reactive rod configuration for the single package and NCT array analyses.

#### 6.12.7.1. **11x11 Fuel Rods Loose or Bundled**

In the following analysis, the fuel rods are placed loose within the liner without any hardware to restrict movement. Twenty-five (25) fuel rods per compartment, or 50 fuel rods per package, are considered. This analysis bounds fuel rods that are bundled together, as bundled fuel rods would have significantly less moderation than loose rods, and credit could not be taken for the bundles remaining intact in an accident.

The package array model under HAC is used for the loose fuel rod calculations since it is demonstrated to be more reactive than the NCT package array model for the 11x11 fuel assembly. The inner container is initially filled with 60% density water since this was determined to be the optimum density for moderation in the original loose rod analysis, see Table 6-23 Fuel

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Rod Maximum Quantity at Reduced Moderator Densities, and system behavior will be similar for the 11x11 rod. The outer container has no water, which facilitates leakage of neutrons into neighboring containers. Twenty-five (25) worst case fuel rods are arranged in a square pitch inside each TN-B1 transport compartment.

A fuel rod pitch sensitivity study is performed for the 11x11 loose fuel rods in which the fuel rod pitch is varied from close packed to filling the inner compartment, as shown in Figure 6-80 Twenty-five 11x11 Fuel Rods in the TN-B1 Container. The results are provided in Table 6-88 Fuel Rod Pitch Sensitivity Study Results with 11x11 Fuel Rods. The maximum reactivity for 25 11x11 fuel rods in the TN-B1 package HAC array occurs for a fuel rod pitch of 3.20 cm. Because reactivity decreases for a larger pitch of 3.52 cm, optimum moderation is achieved for a 3.20 cm pitch.

The fuel rod pitch sensitivity study is initially performed using a water density of  $0.60 \text{ g/cm}^2$  since this was found to be the limiting case the original analysis. To ensure that this moderator density produces the highest reactivity for the 11x11 rods, a moderator density study is performed. A rod pitch of 3.20 cm is used for the density study. The results are provided in Table 6-89 Moderator Density Sensitivity Study Results with 11x11 Fuel Rods. These results demonstrate that a moderator density of  $0.60 \text{ g/cm}^3$  produces the highest reactivity for 25 11x11 fuel rods in the TN-B1 package array.

Finally, a study considering the polyethylene liner within the inner container is performed. The most reactive fuel rod pitch of 0.320 cm is used along with a moderator density of  $0.60 \text{ g/cm}^3$ . The liner thickness is varied from no liner to the maximum liner that could accommodate the fuel rod array used. Thicker liners would require a smaller rod pitch, which would lower the system reactivity. The foam liner is modeled with a density of  $0.08 \text{ g/cm}^3$  on the top and sides and  $0.16 \text{ g/cm}^3$  on the bottom. The results are provided in Table 6-90 Polyethylene Liner Sensitivity Study Results with 11x11 Fuel Rods. These results show that a 0.4 cm thick polyethylene liner produces the highest reactivity for 25 11x11 fuel rods in the TN-B1 package array, with  $k_{\text{eff}} + 2\sigma = 0.72725$ . It is noted that there is very little variation in the results for the different liner thicknesses.

Compared to the 11x11 fuel assembly (maximum  $k_{\text{eff}} + 2\sigma = 0.93810$ ), the reactivity of 25 loose rods is comparatively quite low due to insufficient fissile mass.

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**Table 6-88 Fuel Rod Pitch Sensitivity Study Results with 11x11 Fuel Rods**

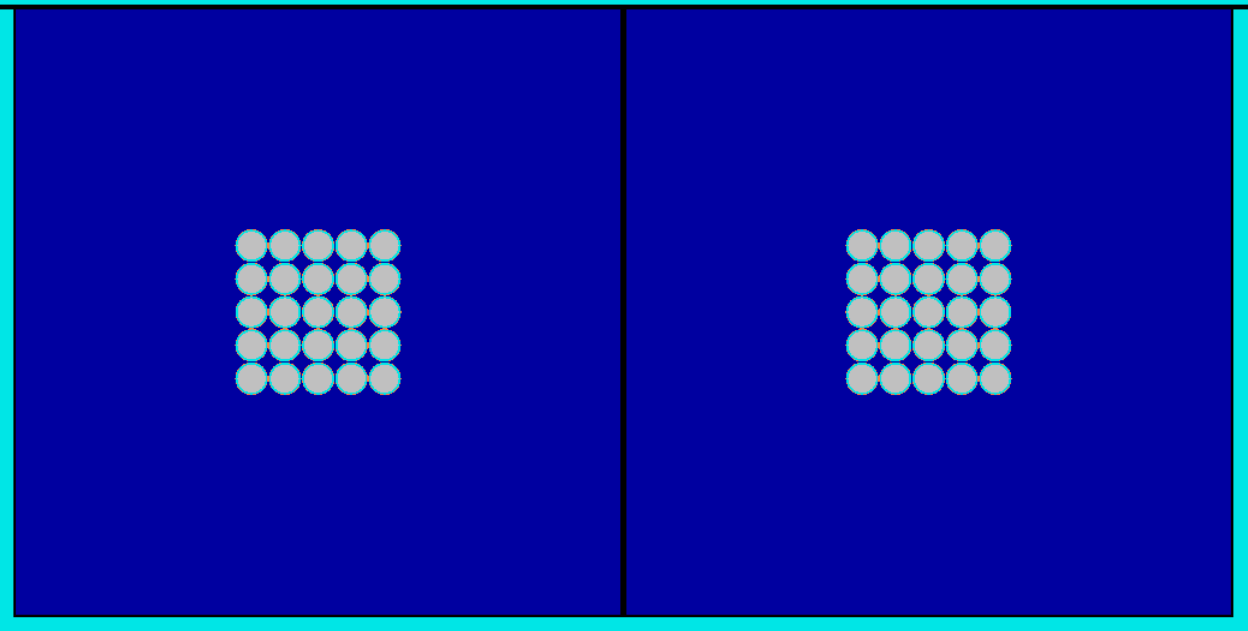
Filename	Fuel Rod Pitch (cm)	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
atrium11_pitch_097_MOD.out	0.970	0.42032	0.00037	0.42106
atrium11_pitch_120_MOD.out	1.200	0.47105	0.00037	0.47179
atrium11_pitch_160_MOD.out	1.600	0.55238	0.00042	0.55322
atrium11_pitch_200_MOD.out	2.000	0.62411	0.00047	0.62505
atrium11_pitch_240_MOD.out	2.400	0.67726	0.00044	0.67814
atrium11_pitch_280_MOD.out	2.800	0.71075	0.00044	0.71163
atrium11_pitch_301_MOD.out	3.0056	0.71942	0.00046	0.72034
<b>atrium11_pitch_320_MOD.out</b>	<b>3.200</b>	<b>0.72369</b>	<b>0.00043</b>	<b>0.72455</b>
atrium11_pitch_352_MOD.out	3.520	0.71850	0.00039	0.71928

**Table 6-89 Moderator Density Sensitivity Study Results with 11x11 Fuel Rods**

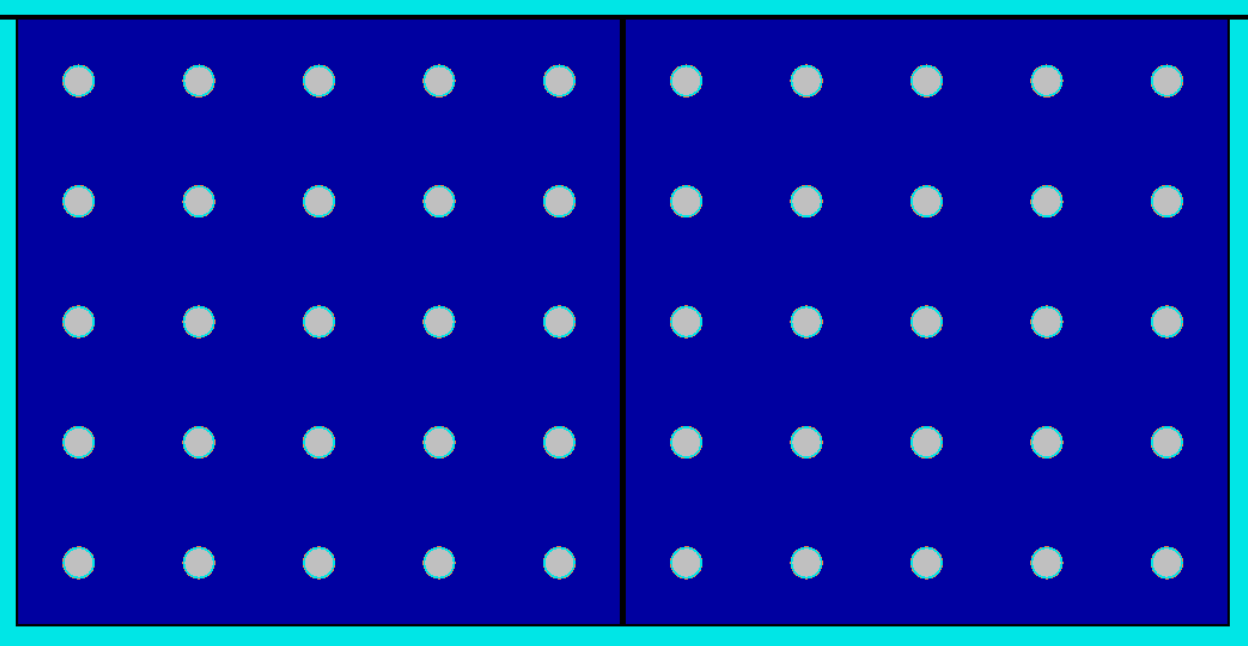
Filename	Moderator Density (g/cm <sup>3</sup> )	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
atrium11_dens000_MOD.out	0.00	0.19803	0.00020	0.19843
atrium11_dens010_MOD.out	0.10	0.38208	0.00034	0.38276
atrium11_dens020_MOD.out	0.20	0.53767	0.00045	0.53857
atrium11_dens040_MOD.out	0.40	0.68478	0.00040	0.68558
<b>atrium11_pitch_320_MOD.out</b>	<b>0.60</b>	<b>0.72369</b>	<b>0.00043</b>	<b>0.72455</b>
atrium11_dens080_MOD.out	0.80	0.71880	0.00041	0.71962
atrium11_dens100_MOD.out	1.00	0.69309	0.00040	0.69389

**Table 6-90 Polyethylene Liner Sensitivity Study Results with 11x11 Fuel Rods**

Filename	Polyethylene Liner Thickness (cm)	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
looseRods_liner_000.out	0.00	0.72386	0.00042	0.72470
looseRods_liner_020.out	0.20	0.72532	0.00042	0.72616
<b>looseRods_liner_040.out</b>	<b>0.40</b>	<b>0.72635</b>	<b>0.00045</b>	<b>0.72725</b>
looseRods_liner_060.out	0.60	0.72565	0.00044	0.72653
looseRods_liner_080.out	0.80	0.72477	0.00046	0.72569
looseRods_liner_full.out	0.8125	0.72512	0.00041	0.72594



Close-packed loose fuel rods



Maximum-pitch loose fuel rods

**Figure 6-80 Twenty-five 11x11 Fuel Rods in the TN-B1 Container**

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#### 6.12.7.2. 11x11 Fuel Rods in 5-in Pipe or Protective Case

In the following analysis, the fuel rods are placed in a 5-in stainless steel pipe. Thirty (30) fuel rods per pipe, or 60 fuel rods per package, are considered. This analysis bounds fuel rods transported in a protective case, as the protective case has a much smaller cross sectional area than the pipe and would allow significantly less moderation.

The package array model under HAC is used for the fuel rod in a stainless steel pipe calculations since it is demonstrated to be more reactive than the NCT package array model for the 11x11 fuel assembly. The volume inside the pipe is filled with water. The inner compartment volume outside the pipe as well as the volume in the outer container is initially modeled with no material present to maximize neutron interactions among packages in the array. This configuration is consistent with complete liner foam burn. The pipe's stainless steel wall is neglected for conservatism.

A 5-in schedule 40 pipe has an outer diameter of 5.563-in and a wall thickness of 0.258-in. The pipe is modeled as a moderator-filled cylinder with a 5.563-in diameter, which allows additional volume for pitch expansion because the pitch is allowed to expand until the outer diameter of the pipe is reached. The number of rods allowed in a pipe is limited to 30, consistent with the 10x10 fuel rod design. A triangular pitch is used in the models, similar to the 10x10 rod analysis, and a fuel rod pitch sensitivity study is performed. The fuel rod pitch is varied from close packed to filling the pipe, as shown in Figure 6-81 Thirty 11x11 Fuel Rods in a 5-in Stainless Steel Pipe. The results are given in Table 6-91 Fuel Rod Pitch Sensitivity Study Results with 11x11 Fuel Rods in a Pipe. The maximum reactivity for 30 fuel rods per pipe is found with a fuel rod pitch of 2.3606 cm, which is the maximum possible pitch.

The fuel rod pitch sensitivity study is executed using a water density of  $1.0 \text{ g/cm}^3$ . To ensure that this moderator density produces the highest reactivity for the 11x11 rods, a moderator density study is performed. A rod pitch of 2.3606 cm is used for the moderator density study. The results are provided in Table 6-92 Moderator Density in Pipe Sensitivity Study Results with 11x11 Fuel Rods. These results show that a moderator density of  $1.0 \text{ g/cm}^3$  produces the highest reactivity for 30 11x11 fuel rods in the pipe within the TN-B1 package array.

Because the rods in the pipe are likely undermoderated, it is possible that reducing the number of rods within the pipe may increase reactivity by increasing moderation. Using the maximum reactivity configuration from above (fuel rod pitch of 2.3606 cm, density of  $1.0 \text{ g/cm}^3$ ), several calculations are performed to show that 30 rods remain bounding. First, two rods are removed from near center for a total of 28 rods in the pipe. Next, four rods are removed from the periphery for a total of 26 rods in the pipe. Removing these rods also allows the pitch to increase slightly. Calculations are performed for the original pitch, corresponding to 30 rods, and with the maximum pitch allowed with 26 rods. Finally, 22 rods are considered by removing




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four periphery rods and four interior rods. This configuration is considered for both the original pitch corresponding to 30 rods and the maximum pitch allowed with 22 rods. The configurations at maximum pitch are shown in Figure 6-82 Configurations for Sensitivity Study on Number of 11x11 Fuel Rods in a 5-in Stainless Steel Pipe. Results are provided in Table 6-93 Number of Rods Sensitivity Study Results with 11x11 Fuel in Pipe and indicate that reducing the number of rods in the pipe decreases reactivity.

The outer diameter of the pipe is 5.563-in, which is significantly less than the width of the fuel compartment of the TN-B1 container. Thus, the location of the pipe within the compartment is considered. Only the horizontal positioning of the pipe is varied since it is shown in the 11x11 fuel assembly orientation study (see Section 6.12.3.5.1, Fuel Assembly Orientation Study) that reactivity is not sensitive to the vertical variation. The original configuration has both pipes centered horizontally and nearly flush with the bottom of the compartment, as illustrated in Figure 6-81 Thirty 11x11 Fuel Rods in a 5-in Stainless Steel Pipe. Configurations considered in this sensitivity study are provided in Figure 6-83 Configurations for Sensitivity Study on Placement of the 5-in Stainless Steel Pipe. Results are given in Table 6-94 Pipe Placement Sensitivity Study Results and demonstrate that shifting both pipes toward the center of the container (case shift2) produces the highest reactivity.

A study considering the polyethylene liner within the inner container is also performed. The most reactive fuel rod pitch of 2.3606 cm is used along with a moderator density of  $1.0 \text{ g/cm}^3$ . The liner thickness is varied from no liner to the maximum liner that could accommodate the pipe. The foam liner is modeled with a density of  $0.08 \text{ g/cm}^3$  on the top and sides and  $0.16 \text{ g/cm}^3$  on the bottom. It is observed when the liner has completely burned away (see Table 6-94 Pipe Placement Sensitivity Study Results) that maximum reactivity occurs when both pipes are shifted toward the center of the container (shifted in). Thus, when the liner is modeled, two pipe orientations are investigated: (1) pipes centered within their compartment, and (2) pipes shifted in. It is noted that for the study with the pipes shifted in, the location of the pipe shifts with the liner thickness. The results for the liner study are provided in Table 6-95 Polyethylene Liner Sensitivity Study Results with 11x11 Fuel Rods in Pipe and indicate that the most reactive configuration occurs when the liner has completely burned away with the pipes shifted in. It is noted that there is very little variation in the results for the different liner thicknesses.

Finally, a second moderator density study is performed in which moderator is modeled outside the pipe but within the inner container with varying density. The most reactive model from the polyethylene liner study is used for this study: both pipes shifted in and no foam liner present. The results for the study are provided in Table 6-96 Inner Container (Outside Pipe) Moderator Density Sensitivity Study. The results indicate that adding moderator with a density of  $\leq 0.1 \text{ g/cm}^3$  outside of the pipe increases reactivity. The maximum  $k_{\text{eff}} + 2\sigma$  is 0.82035 for  $0.1 \text{ g/cm}^3$  water in this region.

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**Table 6-91 Fuel Rod Pitch Sensitivity Study Results with 11x11 Fuel Rods in a Pipe**


Filename	Fuel Rod Pitch (cm)	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
30rods_pipe_pitch0961_MOD.out	0.9606	0.48115	0.00043	0.48201
30rods_pipe_pitch1261_MOD.out	1.2606	0.58817	0.00045	0.58907
30rods_pipe_pitch1561_MOD.out	1.5606	0.68463	0.00044	0.68551
30rods_pipe_pitch1861_MOD.out	1.8606	0.75785	0.00047	0.75879
30rods_pipe_pitch2048_MOD.out	2.0484	0.78662	0.00048	0.78758
30rods_pipe_pitch2161_MOD.out	2.1606	0.79855	0.00046	0.79947
<b>30rods_pipe_pitch2361_MOD.out</b>	<b>2.3606</b>	<b>0.80614</b>	<b>0.00047</b>	<b>0.80708</b>

**Table 6-92 Moderator Density in Pipe Sensitivity Study Results with 11x11 Fuel Rods**

Filename	Moderator Density (g/cm <sup>3</sup> )	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
30rods_pipe_dens00_MOD.out	0.00	0.21700	0.00022	0.21744
30rods_pipe_dens01_MOD.out	0.10	0.30569	0.00028	0.30625
30rods_pipe_dens02_MOD.out	0.20	0.41873	0.00036	0.41945
30rods_pipe_dens04_MOD.out	0.40	0.60063	0.00040	0.60143
30rods_pipe_dens06_MOD.out	0.60	0.71187	0.00049	0.71285
30rods_pipe_dens08_MOD.out	0.80	0.77393	0.00045	0.77483
<b>30rods_pipe_pitch2361_MOD.out</b>	<b>1.00</b>	<b>0.80614</b>	<b>0.00047</b>	<b>0.80708</b>

**Table 6-93 Number of Rods Sensitivity Study Results with 11x11 Fuel in Pipe**

Filename	# Rods	Pitch, cm	$k_{eff}$	$\sigma$	$k_{eff} + 2\sigma$
atrium11_22rods_pipe.out	22	2.3606	0.70613	0.00043	0.70699
atrium11_22rods_pipe_maxPitch.out	22	2.5326	0.69620	0.00042	0.69704
atrium11_26rods_pipe.out	26	2.3606	0.76348	0.00044	0.76436
atrium11_26rods_pipe_maxPitch.out	26	2.5326	0.76328	0.00045	0.76418
atrium11_28rods_pipe.out	28	2.3606	0.78038	0.00049	0.78136
<b>30rods_pipe_pitch2361_MOD.out</b>	<b>30</b>	<b>2.3606</b>	<b>0.80614</b>	<b>0.00047</b>	<b>0.80708</b>

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**Table 6-94 Pipe Placement Sensitivity Study Results**


Filename	Left pipe placement <sup>a</sup>	Right pipe placement <sup>a</sup>	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
30rods_pipe_pitch2361_MOD.out	center	center	0.80614	0.00047	0.80708
pipe_shift_up.out	up	up	0.80580	0.00043	0.80666
pipe_shift1.out	center	in	0.80947	0.00044	0.81035
<b>pipe_shift2.out</b>	<b>in</b>	<b>in</b>	<b>0.81502</b>	<b>0.00048</b>	<b>0.81598</b>
pipe_shift3.out	center	out	0.80487	0.00046	0.80579
pipe_shift4.out	out	out	0.80268	0.00044	0.80356

<sup>a</sup> Refers to horizontal position for all cases except pipe\_shift\_up. Pipe\_shift\_up shifts the pipes vertically to the center of the compartment.

**Table 6-95 Polyethylene Liner Sensitivity Study Results with 11x11 Fuel Rods in Pipe**

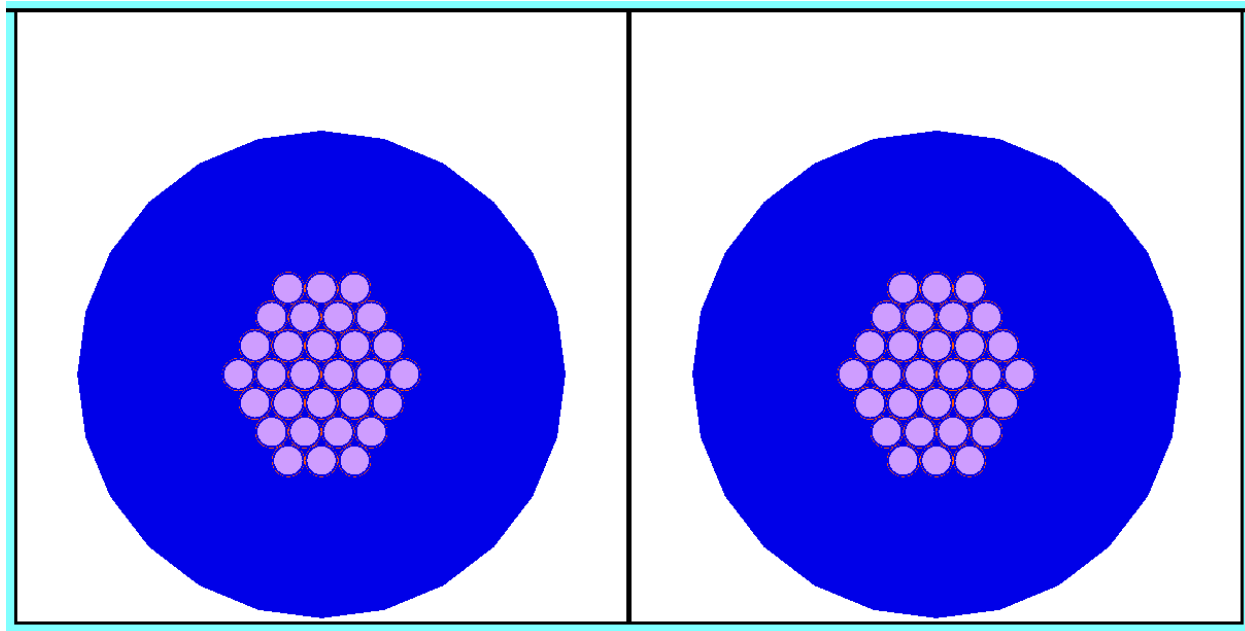
Filename	Polyethylene Liner Thickness (cm)	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
Pipes centered in compartments				
liner_000_cntr_void.out	0.00	0.80563	0.00044	0.80651
liner_020_cntr_void.out	0.20	0.80736	0.00044	0.80824
liner_040_cntr_void.out	0.40	0.80834	0.00046	0.80926
liner_060_cntr_void.out	0.60	0.80886	0.00044	0.80974
liner_080_cntr_void.out	0.80	0.80879	0.00049	0.80977
liner_100_cntr_void.out	1.00	0.80915	0.00042	0.80999
liner_120_cntr_void.out	1.20	0.81128	0.00046	0.81220
liner_140_cntr_void.out	1.40	0.81179	0.00050	0.81279
liner_160_cntr_void.out	1.60	0.81144	0.00048	0.81240
liner_full_cntr_void.out	1.74	0.81198	0.00046	0.81290
Pipes shifted in towards center of container <sup>a</sup>				
<b>liner_000_shift2_void.out</b>	<b>0.00</b>	<b>0.81557</b>	<b>0.00047</b>	<b>0.81651</b>
liner_020_shift2_void.out	0.20	0.81443	0.00056	0.81555
liner_040_shift2_void.out	0.40	0.81373	0.00041	0.81455
liner_060_shift2_void.out	0.60	0.81281	0.00047	0.81375
liner_080_shift2_void.out	0.80	0.81336	0.00043	0.81422
liner_100_shift2_void.out	1.00	0.81343	0.00042	0.81427
liner_120_shift2_void.out	1.20	0.81293	0.00046	0.81385
liner_140_shift2_void.out	1.40	0.81259	0.00047	0.81353
liner_160_shift2_void.out	1.60	0.81278	0.00046	0.81370
liner_full_shift2_void.out	1.74	0.81198	0.00046	0.81290

<sup>a</sup> For these cases, the pipes are shifted towards the center of the package as in "shift2" of Figure 6-83; however, the pipes are centered vertically in the compartment instead of being shifted to the bottom of the compartment.

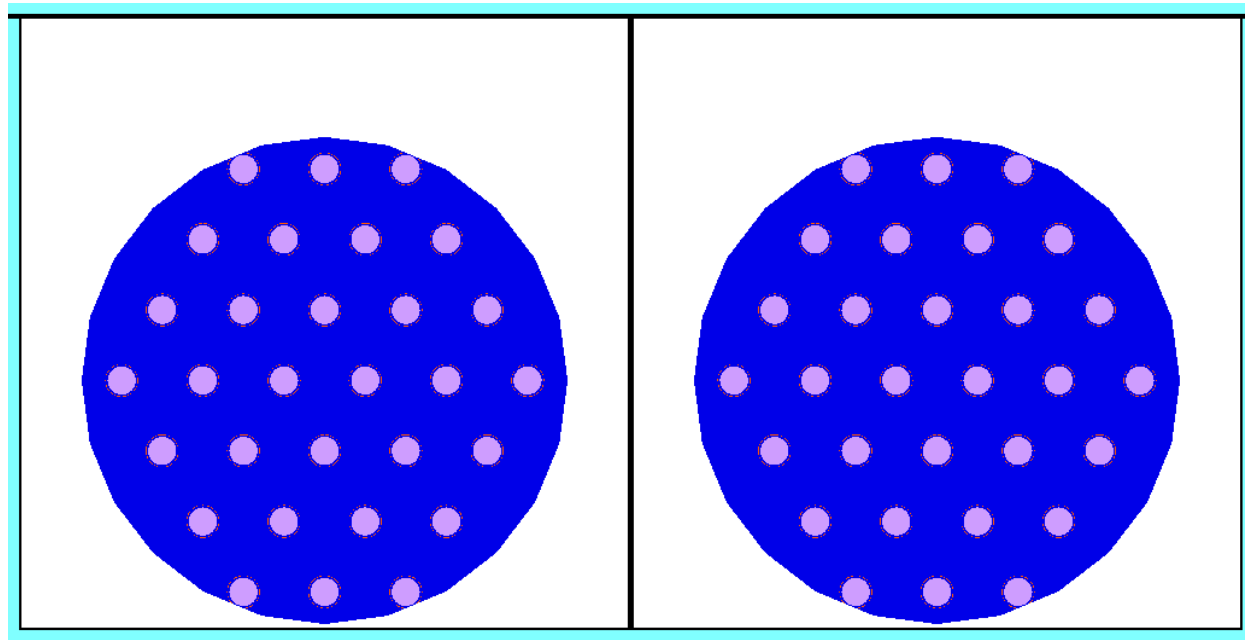
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**Table 6-96 Inner Container (Outside Pipe) Moderator Density Sensitivity Study**

Filename	Moderator Density (g/cm <sup>3</sup> )	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
liner_000_shift2_void.out <sup>a</sup>	0.00	0.81557	0.00047	0.81651
mod_in_pipe10_dens002.out:	0.02	0.81657	0.00043	0.81743
mod_in_pipe10_dens004.out:	0.04	0.81709	0.00046	0.81801
mod_in_pipe10_dens006.out:	0.06	0.81827	0.00049	0.81925
mod_in_pipe10_dens008.out:	0.08	0.81871	0.00046	0.81963
<b>mod_in_pipe10_dens010.out:</b>	<b>0.10</b>	<b>0.81947</b>	<b>0.00044</b>	<b>0.82035</b>
mod_in_pipe10_dens020.out:	0.20	0.81481	0.00045	0.81571
mod_in_pipe10_dens040.out:	0.40	0.80142	0.00047	0.80236
mod_in_pipe10_dens060.out:	0.60	0.78567	0.00046	0.78659
mod_in_pipe10_dens080.out:	0.80	0.76915	0.00041	0.76997
mod_in_pipe10_dens100.out:	1.0	0.75750	0.00044	0.75838

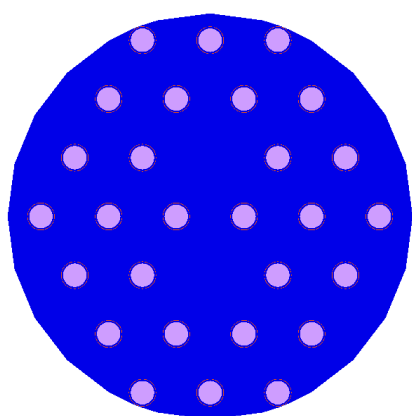


Close-packed fuel rods in pipe

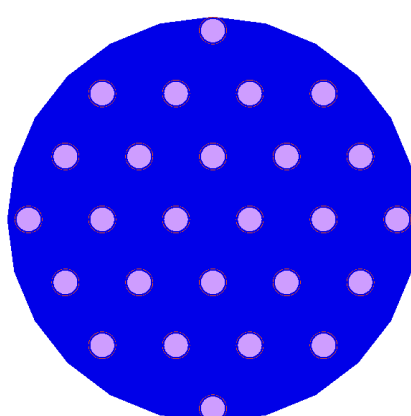


Maximum-pitch fuel rods in pipe

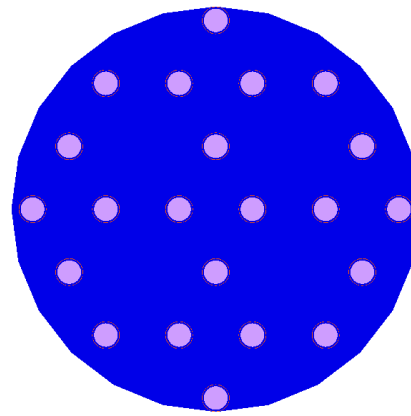
**Figure 6-81 Thirty 11x11 Fuel Rods in a 5-in Stainless Steel Pipe**



28 fuel rods




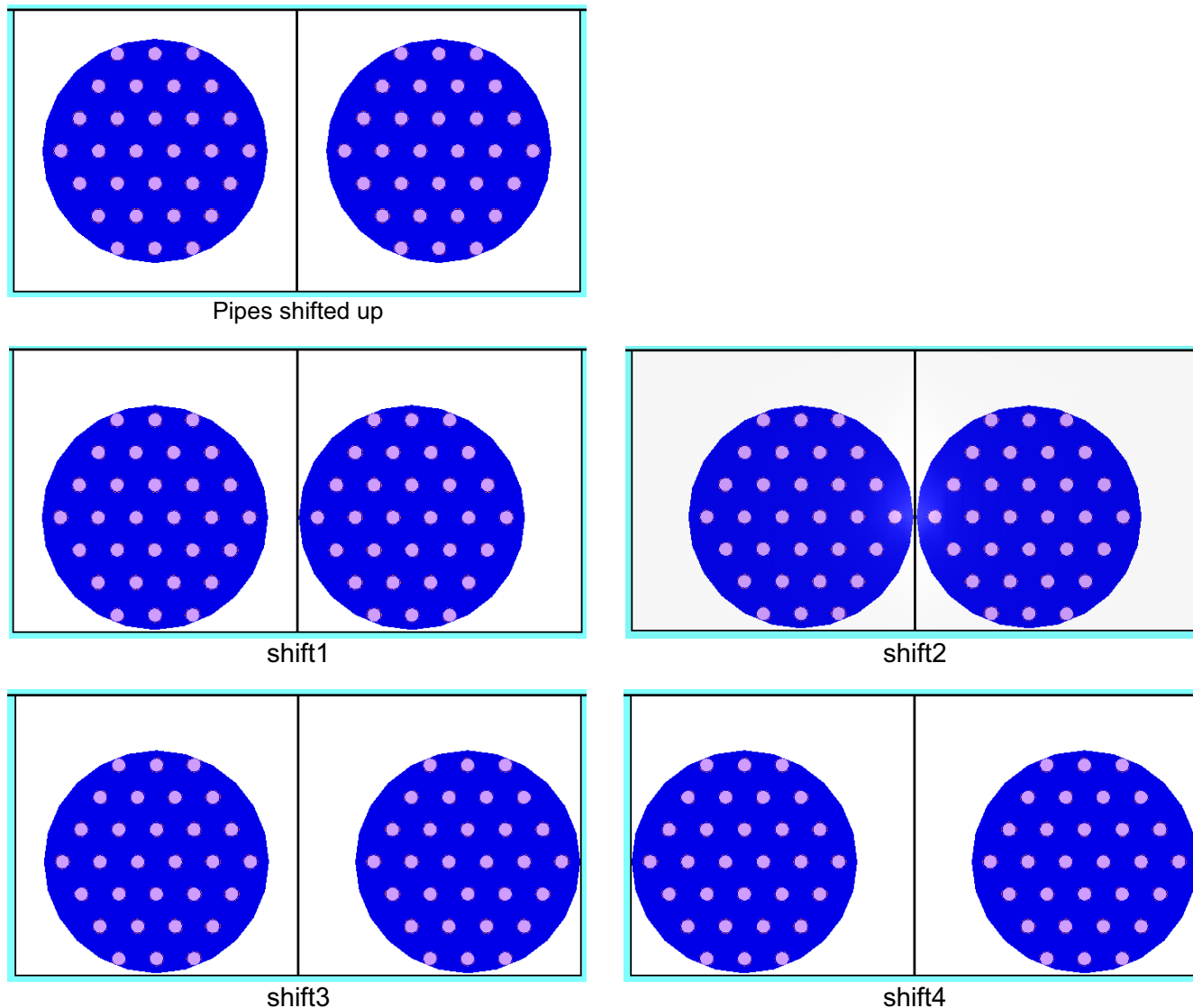
26 fuel rods



22 fuel rods

**Figure 6-82 Configurations for Sensitivity Study on Number of 11x11 Fuel Rods in a 5-in Stainless Steel Pipe**

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**Figure 6-83 Configurations for Sensitivity Study on Placement of the 5-in Stainless Steel Pipe**

#### 6.12.7.3. Single Package Evaluation for 11x11 Rods

This section describes the evaluation of a single package model under NCT and HAC. Fuel parameters modeled to reflect the 11x11 rod worst case parameters shown below:

UO<sub>2</sub> pellet density = 10.763 g/cm<sup>3</sup>

UO<sub>2</sub> enrichment = 5.0 wt% <sup>235</sup>U

Pellet diameter = 0.820 cm

Rod outer diameter = 0.930 cm

Cladding thickness = 0.00 cm

Fuel length = 385 cm



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A 6-mil layer of polyethylene encircles each fuel rod. Comparing the most reactive HAC array results from the loose rod analysis ( $k_{\text{eff}} + 2\sigma = 0.72725$ , see Section 6.12.7.1) and pipe analysis ( $k_{\text{eff}} + 2\sigma = 0.82035$ , see Section 6.12.7.2), the pipe analysis is significantly more reactive than the loose rod analysis. Therefore, single package cases are performed only for 30 rods in the 5-in pipe. The NCT and HAC single package models are the same basic models used for the 11x11 fuel assembly analysis, with the fuel assembly replaced with the pipe. The single package model is enveloped with a 30.48-cm layer of full density water for reflection.

#### NCT Single Package

For the NCT single package evaluation, the stainless steel pipes are held centered within the inner container compartments by the polyethylene liner. The top and sides of the liner are modeled with  $0.08 \text{ g/cm}^3$  polyethylene, while the bottom liner is modeled with  $0.16 \text{ g/cm}^3$  polyethylene. A liner thickness of 1.74 cm is used, which corresponds to available space between the walls of the compartment and the pipe. The space between the pipe and the liner is filled with moderator.

A moderator density study, in which the density of the water within the inner container is varied, is performed for the NCT single package model. Results from the NCT single package model are provided in Table 6-97 TN-B1 NCT Single Package Evaluation with 11x11 Fuel Rods. The maximum  $k_{\text{eff}} + 2\sigma$  value for the single package normal conditions of transport case is 0.59235 for a moderator density of  $1 \text{ g/cm}^3$ , which is far below the rod USL of 0.94047. Therefore, criticality safety is established for the NCT single package TN-B1 container with 11x11 fuel rods.


#### HAC Single Package

For the HAC single package evaluation, the polyethylene foam liner is assumed to burn away and the pipes are allowed to move within the inner container compartments. Thus, the most reactive pipe configuration from Section 6.12.7.2, 11x11 Fuel Rods in 5-in Pipe or Protective Case, is used: shift2 with both pipes shifted toward the center of the container.

The Alumina Silicate thermal insulation is modeled between the inner and outer walls. This is consistent with the physical condition of the TN-B1 shipping container after being subjected to the tests specified in 10 CFR Part 71 (see Sections 3.2.2 and 3.5.2).

Two moderator density studies are performed for the most reactive configuration determined above. For the first study, the moderator density is varied within the inner container. For the second study, the moderator density in the pipe is fixed at  $1.0 \text{ g/cm}^3$  while the density outside the pipe but within the inner container is varied.

Results from the HAC single package model are provided in Table 6-98 TN-B1 HAC Single Package Evaluation with 11x11 Fuel Rods, Moderator Density Varied within Pipe and Inner Container for the moderator density study in which all water within the pipe and inner container is uniformly varied. Results of fixing the moderator density in the pipe at  $1.0 \text{ g/cm}^3$  and varying

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the water density outside the pipe are provided in Table 6-99 TN-B1 HAC Single Package Evaluation with 11x11 Fuel Rods: Moderator Density Varied Outside Pipe. The maximum  $k_{\text{eff}} + 2\sigma$  value for the single package HAC case is 0.66400 and occurs with maximum density water inside the pipe and inner container. This result is far below the rod USL of 0.94047. Therefore, criticality safety is established for the HAC single package TN-B1 container with 11x11 fuel rods.

**Table 6-97 TN-B1 NCT Single Package Evaluation with 11x11 Fuel Rods**

Filename	Moderator Density (g/cm <sup>3</sup> )	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
NCT_single_pipe_dens000.out	0.00	0.23555	0.00027	0.23609
NCT_single_pipe_dens002.out	0.02	0.23929	0.00027	0.23983
NCT_single_pipe_dens004.out	0.04	0.24207	0.00027	0.24261
NCT_single_pipe_dens006.out	0.06	0.24581	0.00027	0.24635
NCT_single_pipe_dens008.out	0.08	0.25086	0.00030	0.25146
NCT_single_pipe_dens010.out	0.10	0.25585	0.00027	0.25639
NCT_single_pipe_dens020.out	0.20	0.28698	0.00031	0.28760
NCT_single_pipe_dens040.out	0.40	0.37263	0.00036	0.37335
NCT_single_pipe_dens060.out	0.60	0.45991	0.00039	0.46069
NCT_single_pipe_dens080.out	0.80	0.53590	0.00039	0.53668
<b>NCT_single_pipe_dens100.out</b>	<b>1.0</b>	<b>0.59145</b>	<b>0.00045</b>	<b>0.59235</b>

**Table 6-98 TN-B1 HAC Single Package Evaluation with 11x11 Fuel Rods, Moderator Density Varied within Pipe and Inner Container**

Filename	Moderator Density (g/cm <sup>3</sup> )	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
HAC_single_pipe_dens000_shift.out	0.00	0.23969	0.00025	0.24019
HAC_single_pipe_dens002_shift.out	0.02	0.24037	0.00025	0.24087
HAC_single_pipe_dens004_shift.out	0.04	0.24167	0.00027	0.24221
HAC_single_pipe_dens006_shift.out	0.06	0.24368	0.00026	0.24420
HAC_single_pipe_dens008_shift.out	0.08	0.24749	0.00026	0.24801
HAC_single_pipe_dens010_shift.out	0.10	0.25197	0.00029	0.25255
HAC_single_pipe_dens020_shift.out	0.20	0.29275	0.00030	0.29335
HAC_single_pipe_dens040_shift.out	0.40	0.41304	0.00036	0.41376
HAC_single_pipe_dens060_shift.out	0.60	0.52597	0.00041	0.52679
HAC_single_pipe_dens080_shift.out	0.80	0.60853	0.00042	0.60937
<b>HAC_single_pipe_dens100_shift.out</b>	<b>1.0</b>	<b>0.66316</b>	<b>0.00042</b>	<b>0.66400</b>

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**Table 6-99 TN-B1 HAC Single Package Evaluation with 11x11 Fuel Rods: Moderator Density Varied Outside Pipe**

Filename	Moderator Density (g/cm <sup>3</sup> )	k <sub>eff</sub>	σ	k <sub>eff</sub> + 2σ
HAC_single_pipe_dens000_in100.out	0.00	0.57214	0.00044	0.57302
HAC_single_pipe_dens002_in100.out	0.02	0.57475	0.00043	0.57561
HAC_single_pipe_dens004_in100.out	0.04	0.57776	0.00047	0.57870
HAC_single_pipe_dens006_in100.out	0.06	0.58142	0.00041	0.58224
HAC_single_pipe_dens008_in100.out	0.08	0.58505	0.00043	0.58591
HAC_single_pipe_dens010_in100.out	0.10	0.58731	0.00043	0.58817
HAC_single_pipe_dens020_in100.out	0.20	0.60136	0.00040	0.60216
HAC_single_pipe_dens040_in100.out	0.40	0.62405	0.00043	0.62491
HAC_single_pipe_dens060_in100.out	0.60	0.64117	0.00043	0.64203
HAC_single_pipe_dens080_in100.out	0.80	0.65417	0.00044	0.65505
<b>HAC_single_pipe_dens100_shift.out</b>	<b>1.0</b>	<b>0.66316</b>	<b>0.00042</b>	<b>0.66400</b>

#### 6.12.7.4. NCT Array Evaluation for 11x11 Rods

The NCT package array model is the same basic model used for the 11x11 fuel assembly analysis, with the fuel assembly replaced with the pipe. The model consists of a 21x3x24 array of containers, surrounded by a 30.48-cm layer of full density water for reflection. The container array is fully flooded with water at a density sufficient for maximum reactivity.

Fuel parameters modeled to reflect the 11x11 rod worst case parameters shown below:

UO<sub>2</sub> pellet density = 10.763 g/cm<sup>3</sup>

UO<sub>2</sub> enrichment = 5.0 wt% <sup>235</sup>U

Pellet diameter = 0.820 cm

Rod outer diameter = 0.930 cm

Cladding thickness = 0.00 cm

Fuel length = 385 cm

A 6-mil layer of polyethylene encircles each fuel rod. Comparing the most reactive HAC array results from the loose rod analysis (k<sub>eff</sub> + 2σ = 0.72725, see Section 6.12.7.1) and pipe analysis (k<sub>eff</sub> + 2σ = 0.82035, see Section 6.12.7.2), the pipe analysis is significantly more reactive than the loose rod analysis. Therefore, NCT array cases are performed only for 30 rods in the 5-in pipe.

For the NCT package array evaluation, the fuel assemblies are held centered within the inner container compartments by the polyethylene liner. A liner thickness of 1.74 cm is used, which corresponds to available space between the walls of the compartment and the pipe. The space between the pipe and the liner is filled with moderator. The top and sides of the liner are

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modeled with 0.08 g/cm<sup>3</sup> polyethylene, while the bottom liner is modeled with 0.16 g/cm<sup>3</sup> polyethylene.

A moderator density study, in which the density of the water within the inner and outer containers is uniformly varied, is performed for the package array model. Results from the NCT package array model with 11x11 fuel rods are provided in Table 6-100 TN-B1 NCT Package Array Evaluation with 11x11 Fuel Rods. The maximum  $k_{\text{eff}} + 2\sigma$  value for the package array NCT case is 0.59384 and occurs for full density water, which is far below the rod USL of 0.94047. Therefore, criticality safety is established for the TN-B1 container package array with 11x11 fuel rods for the NCT array.

**Table 6-100 TN-B1 NCT Package Array Evaluation with 11x11 Fuel Rods**

Filename	Moderator Density (g/cm <sup>3</sup> )	$k_{\text{eff}}$	$\sigma$	$k_{\text{eff}} + 2\sigma$
NCT_array_pipe_dens000.out	0.00	0.49959	0.00032	0.50023
NCT_array_pipe_dens002.out	0.02	0.53972	0.00036	0.54044
NCT_array_pipe_dens004.out	0.04	0.52833	0.00037	0.52907
NCT_array_pipe_dens006.out	0.06	0.51110	0.00035	0.51180
NCT_array_pipe_dens008.out	0.08	0.48967	0.00037	0.49041
NCT_array_pipe_dens010.out	0.10	0.46815	0.00037	0.46889
NCT_array_pipe_dens020.out	0.20	0.38906	0.00033	0.38972
NCT_array_pipe_dens040.out	0.40	0.38840	0.00036	0.38912
NCT_array_pipe_dens060.out	0.60	0.46283	0.00040	0.46363
NCT_array_pipe_dens080.out	0.80	0.53585	0.00042	0.53669
<b>NCT_array_pipe_dens100.out</b>	<b>1.0</b>	<b>0.59300</b>	<b>0.00042</b>	<b>0.59384</b>

#### 6.12.7.5. HAC Array Evaluation for 11x11 Rods

The loose rod evaluation documented in Section 6.12.7.1, *11x11 Fuel Rods Loose or Bundled* and Section 6.12.7.2, *11x11 Fuel Rods in 5-in Pipe or Protective Case*, are for a 10x1x10 HAC array. Comparing the most reactive HAC array results from the loose rod analysis ( $k_{\text{eff}} + 2\sigma = 0.72725$ ) and pipe analysis ( $k_{\text{eff}} + 2\sigma = 0.82035$ ), the pipe analysis is significantly more reactive than the loose rod analysis.

The most reactive pipe configuration is with a moderator density of 1.0 g/cm<sup>3</sup> inside the pipe and 0.10 g/cm<sup>3</sup> within the inner container, outside the pipe. The polyethylene foam liner is assumed to have burned away, which allows the pipes to relocate toward the center of the package. The region between the inner and outer container is conservatively modeled as void. The maximum  $k_{\text{eff}} + 2\sigma$  value for the HAC package array case is 0.82035, which is below the USL of 0.94047. Therefore, criticality safety of the TN-B1 shipping container with 11x11 fuel rods is demonstrated for the HAC array.

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### 6.12.8. *Fissile Material Packages for Air Transport*

This package is not intended for the air transport of fissile material.

### 6.12.9. *Benchmark Evaluation for SCALE 6.1.3*

The 11x11 fuel assembly and fuel rod analyses are performed using the KENO V.a module of SCALE 6.1.3. Therefore, the SCALE 4.4a benchmarking evaluation documented in Section 6.10, *Benchmark Evaluations*, is repeated using SCALE 6.1.3. The SCALE 6.1.3 benchmark evaluation uses the 238 group ENDF/B-VII cross section library with CENTRM resonance processing.

A USL of 0.94094 is justified for the 11x11 fuel assembly, and a USL of 0.94047 is justified for the 11x11 fuel rod analysis.

#### 6.12.9.1. **Applicability of Benchmark Experiments**

A total of 58 experiments are selected for benchmarking SCALE 6.1.3 as it is applied to the current evaluation. The 27 experiments originally used in the TN-B1 criticality analysis (see Section 6.10, *Benchmark Evaluations*) are retained in the current analysis:

- 14 configurations: These configurations are water moderated U(4.31 wt. %)O<sub>2</sub> fuel rods in 2.54 cm square-pitched arrays reflected by aluminum, steel, borated steel, zircaloy or boron plates. These experiments use an open top carbon steel tank configuration with acrylic support plates, polyethylene lattice plates and aluminum plates as support structure.
- 13 configurations: These configurations are borated water moderated U(2.459 wt. %)O<sub>2</sub> fuel rods in 1.636 cm square-pitched arrays. 12 experiments replaced some of the fuel rods with U(4.02 wt. %)O<sub>2</sub> fuel rods and/or U(1.94 wt%)O<sub>2</sub>-4 wt% Gd<sub>2</sub>O<sub>3</sub> rods. These experiments use an aluminum tank and aluminum plates as support structure.

An additional 31 experiments from the International Handbook of Evaluated Criticality Safety Benchmark Experiments (ICSBEP) (Reference 17) are selected to supplement the original 27 experiments identified above. The additional experiments are:

- 8 configurations (cases 1-8) from LEU-COMP-THERM-001: These configurations are water moderated U(2.35 wt. %)O<sub>2</sub> fuel rods in 2.032 cm square-pitched arrays. These experiments use an open top carbon steel tank configuration with acrylic support plates, polyethylene lattice plates and aluminum plates as support structure.
- 4 configurations (cases 1-3, 5) from LEU-COMP-THERM-002: These configurations are water moderated U(4.31 wt. %)O<sub>2</sub> fuel rods in 2.54 cm square-pitched arrays. These experiments use an open top carbon steel tank configuration with acrylic support plates, polyethylene lattice plates and aluminum plates as support structure.
- 3 configurations (cases 1, 16, 17) from LEU-COMP-THERM-008: These configurations are borated water moderated U(2.459 wt. %)O<sub>2</sub> fuel rods in 1.636 cm square-pitched

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arrays. These experiments use an aluminum tank and aluminum plates as support structure.

- 12 configurations (cases 10-17, 19-22) from LEU-COMP-THERM-017: These configurations are water moderated U(2.35 wt. %)O<sub>2</sub> fuel rods in square pitched arrays reflected by steel. These experiments use an open top carbon steel tank configuration with acrylic support plates, polyethylene lattice plates and aluminum plates as support structure.
- 2 configurations (cases 1-2) from LEU-COMP-THERM-042: These configurations are water moderated, square-pitched arrays of U(2.35 wt. %)O<sub>2</sub> fuel rods separated by absorption plates made of steel (case 1) or borated steel (case 2) and reflected by steel walls. These experiments use open top carbon steel tank configuration with acrylic support plates, polyethylene lattice plates and aluminum plates as support structure.
- 2 configurations (cases 1-2) from LEU-COMP-THERM-050: These configurations are water moderated, square-pitched arrays of U(4.738 wt. %)O<sub>2</sub> fuel rods with a central zircaloy tank that contains water. These experiments use stainless steel tank configuration with aluminum alloy (AG3M) and stainless steel as support structure.

Table 6-101 Data for Selected Experiments summarizes pertinent data from each of the experiments selected to be included in the USL calculations. Table 6-102 Experiment K<sub>eff</sub> shows the experiment k<sub>eff</sub> and associated uncertainty for all selected experiments. When available, the experiment k<sub>eff</sub> and uncertainty values given Table 6-102 Experiment K<sub>eff</sub> are the benchmark values from the references. The benchmark values are determined in the ICSBEP evaluation by accounting for modeling approximations and simplifications.

The cases are run for 1550 generations, 2000 neutrons per generation, and 50 generations skipped. The results are summarized in Table 6-103 SCALE 6.1.3 Results. This table also includes parameters of interest in the USL evaluation: enrichment, moderator-to-fuel volume ratio, pin pitch, energy of the average lethargy of fission (EALF), and average energy group causing fission (AFG).

#### 6.12.9.2. Bias Determination

A set of Upper Subcritical Limits is determined using the results from the critical experiments and the USL Method 1, Confidence Band with Administrative Margin. The USL Method 1 applies a statistical calculation of the method bias and its uncertainty plus an administrative margin (0.05 Δk) to a linear fit of the critical experiment benchmark data. The USLs are determined as a function of the critical experiment system parameters; enrichment, water-to-fuel volume ratio, pin pitch, EALF, and AFG.

While the original TN-B1 benchmarking also included a USL equation as a function of hydrogen-to-U-235 ratio, the additional experiments did not include this data. It is not expected that this parameter would result in a considerably better correlation of the data than the other parameters



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considered; therefore, a USL equation is not determined as a function of hydrogen-to-U-235 ratio for the current analysis.

The USL method has the tacit assumption that the experimental k is 1.0000 (Reference 7). Likewise, it does not account for the uncertainty in the experimental values. NUREG/CR-6698 (Reference 18) discusses a process for which a different value of the experiment  $k_{eff}$  is considered, as well as the experimental uncertainty. The document has the following definitions for the calculated values used for the bias evaluation:

$$k_{norm} = \frac{k_{calc}}{k_{exp}}$$

$$\sigma_{norm} = \sqrt{\sigma_{calc}^2 + \sigma_{exp}^2}$$

The above equations normalize the calculated to experimental data to account for uncertainties in the experiment values.  $K_{norm}$  and  $\sigma_{norm}$  for each case are provided in Table 6-104 Data Needed for Calculation of USL for SCALE 6.1.3 and are used to generate the USL equations. Table 6-104 also summarizes the data from Table 6-101 through Table 6-103 needed for the USL calculations.

Using the data in Table 6-104, a set of USL equations are generated using the USLSTATS code provided with SCALE 6.1.3. This is run using a 0.05  $\Delta k$  margin. The USLSTATS computed results for each parameter are:

- The following equation is determined for the USL as a function of enrichment:  
 $USL = 0.9404 + (6.8308 \times 10^{-4})x$  for all x.  
*The variance of the equation fit is  $2.2323 \times 10^{-6}$ .*  
*The applicable range for enrichment is  $2.35 \leq x \leq 4.738$   $^{235}\text{U}$  wt%*
- The following equation is determined for the USL as a function of water-to-fuel volume ratio:  
 $USL = 0.9403 + (8.5180 \times 10^{-4})x$  for all x.  
*The variance of the equation fit is  $2.0192 \times 10^{-6}$ .*  
*The applicable range for water-to-fuel ratio is  $1.60 \leq x \leq 3.8832$*
- The following equation is determined for the USL as a function of pin pitch:  
 $USL = 0.9385 + (2.0457 \times 10^{-3})x$  for all x.  
*The variance of the equation fit is  $1.9479 \times 10^{-6}$ .*  
*The applicable range for pin pitch is  $1.30 \leq x \leq 2.54$  cm*
- The following equation is determined for the USL as a function of average energy of the lethargy causing fission:  
 $USL = 0.9440 - (9.1167 \times 10^{-3})x$  for all x.  
*The variance of the equation fit is  $2.1023 \times 10^{-6}$ .*  
*The applicable range for average energy of lethargy causing fission is  $0.0936 \leq x \leq 0.3547$  eV*



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- The following equation is determined for the USL as a function of the average energy group causing fission:  

$$USL = 0.9126 + (1.4681 \times 10^{-4})x \quad \text{for all } x.$$
*The variance of the equation fit is  $2.0426 \times 10^{-6}$ .*  
*The applicable range for average energy group causing fission is  $193.58 \leq x \leq 209.32$*

Of the preceding equations, the USL as a function of pin pitch is the best correlation to the data since the variance of the equation fit is the smallest (Reference 7, Section 4.1.3). Therefore, the USL as a function of pin pitch is used to determine a minimum USL for the 11x11 fuel assembly for use with the TN-B1 shipping container. Figure 6-84: USL as a Function of Pin Pitch shows the USL as a function of pin pitch. The nominal pin pitch used in the 11x11 fuel assembly criticality analyses is 1.195 cm. Although the 1.195 cm pin pitch falls outside the range of applicability, ANSI/ANS-8.1 (Reference 6) allows the range of applicability to be extended beyond the range of conditions represented by the benchmarks, as long as that extrapolation is not large. As outlined in (Reference 7),  $k(x)-w(x)$  is used to extend the USL curve beyond the range of applicability. Figure 6-84: USL as a Function of Pin Pitch displays the USL curve extrapolation using  $k(x)-w(x)$ ; the extrapolated USL value corresponding to the 1.195 cm pin pitch is 0.94094. Since the extrapolated value results in a lower USL than the minimum pin pitch within the range of applicability would produce (0.94116), the USL corresponding to the 1.195 cm pin pitch is selected. Therefore, the USL for the TN-B1 shipping container with 11x11 fuel assemblies is 0.94094.


It is also noted that the most reactive 11x11 fuel assembly is for the HAC cases where the pitch is increased by 5%. Increasing the pin pitch would bring the value closer to the range of applicability but would also increase the USL. Thus, using the lower USL corresponding to the nominal pitch is conservative.

This USL value is not directly applicable to the 11x11 fuel rod analysis since the pin pitch is not maintained. Per Section 6.12.7, *Transport of 11x11 Rods*, the pin pitch for the fuel rod study varies from 0.970 cm to 3.520 cm for the loose rods and from 0.9606 cm to 2.3606 cm for the rods in the pipe.

The USL corresponding to the smallest pin pitch in the fuel rod analysis is 0.94047. The smallest pin pitch corresponds to the lowest USL. Use of a lower USL for configurations with larger pitch values is conservative. Therefore, an upper subcritical limit of 0.94047 is used for comparison of all fuel rod configurations to ensure subcriticality.

The following equation is used to develop the  $k_{eff}$  for the transportation of fuel in the TN-B1 shipping container:

$$k_{eff} = k_{case} + 2\sigma$$

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where:


$k_{case}$  = KENO V.a (SCALE 6.1.3)  $k_{eff}$  for a particular case of interest

$\sigma$  = uncertainty in the calculated KENO V.a (SCALE 6.1.3)  $k_{eff}$  for a particular case of interest

The  $k_{eff}$  for each container configuration analyzed in the TN-B1 shipping container criticality analysis is compared to the USL of 0.94094 for the 11x11 fuel assembly or 0.94047 for the 11x11 fuel rod analysis to ensure subcriticality.

**Table 6-101 Data for Selected Experiments**

Experiment Name	Assembly Separation (cm)	Boron Conc. (ppm)	Enrichment ( $^{235}\text{U}$ wt%)	Mod/fuel ratio	Pin Pitch (cm)	Comments
<u>See Section 6.11.10.1</u>						
c004	10.62	-	4.31	3.8832	2.54	no absorber plates
c005b	9.64	-	4.31	3.8832	2.54	0.625 cm Al plates
c006b	9.59	-	4.31	3.8832	2.54	0.625 cm Al plates
c007a	8.63	-	4.31	3.8832	2.54	0.302 cm SS 304L plates
c008b	8.09	-	4.31	3.8832	2.54	0.302 cm SS 304L plates
c009b	6.95	-	4.31	3.8832	2.54	0.298 cm SS 304L plates with 1.05 wt% B
c010b	5.47	-	4.31	3.8832	2.54	0.298 cm SS 304L plates with 1.05 wt% B
c011b	6.77	-	4.31	3.8832	2.54	0.298 cm SS 304L plates with 1.62 wt% B
c012b	4.63	-	4.31	3.8832	2.54	0.298 cm SS 304L plates with 1.62 wt% B
c013b	8.52	-	4.31	3.8832	2.54	0.485 cm SS 304L plates
c014b	7.45	-	4.31	3.8832	2.54	0.485 cm SS 304L plates
c029b	9.77	-	4.31	3.8832	2.54	0.652 cm Zircaloy-4 plates
c030b	9.73	-	4.31	3.8832	2.54	0.652 cm Zircaloy-4 plates
c031b	6.55	-	4.31	3.8832	2.54	0.723 cm Boral plates, 28.7 wt% B
<u>See Section 6.11.10.2 <sup>a</sup></u>						
core01	-	1337.9	2.459	1.9385	1.6358	4808-2.46 wt%
core03	-	1239.3	2.457	1.9282	1.6358	4788-2.46 wt%, 20 Gd
core05	-	1208.0	2.456	1.9385	1.6358	4780-2.46 wt%, 28 Gd
core05a	-	1191.3	2.456	1.9385	1.6358	4776-2.46 wt%, 32 Gd
core05b	-	1207.1	2.456	1.9385	1.6358	4780-2.46 wt%, 28 Gd
core08	-	1170.7	2.455	1.9385	1.6358	4772-2.46 wt%, 36 Gd
core10	-	1177.1	2.455	1.9278	1.6358	4772-2.46 wt%, 36 Gd
core12a	-	1899.3	2.747	1.8714	1.6358	3920-2.46 wt%, 888-4.02 wt%
core14	-	1653.8	2.735	1.8735	1.6358	3920-2.46 wt%, 860-4.02 wt%, 28 Gd
core16	-	1579.4	2.732	1.8741	1.6358	3920-2.46 wt%, 852-4.02 wt%, 36 Gd
core18	-	1776.8	2.778	1.8866	1.6358	3676-2.46 wt%, 944-4.02 wt%
core19	-	1628.3	2.771	1.8878	1.6358	3676-2.46 wt%, 928-4.02 wt%, 16 Gd
core20	-	1499.0	2.764	1.8891	1.6358	3676- 2.46 wt%, 912-4.02 wt%, 32 Gd

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**Table 6–101 Data for Selected Experiments Continued**

Experiment Name	Assembly Separation (cm)	Boron Conc. (ppm)	Enrichment ( <sup>235</sup> U wt%)	Mod/fuel ratio	Pin Pitch (cm)	Comments
<u>LEU-COMP-THERM-001, Table 10-2</u>						
-001	-	-	2.35	2.918	2.032	
-002	11.92	-	2.35	2.918	2.032	
-003	8.41	-	2.35	2.918	2.032	
-004	10.05	-	2.35	2.918	2.032	
-005	6.39	-	2.35	2.918	2.032	
-006	8.01	-	2.35	2.918	2.032	
-007	4.46	-	2.35	2.918	2.032	
-008	7.57	-	2.35	2.918	2.032	
<u>LEU-COMP-THERM-002, Table 10-2</u>						
-001	-	-	4.31	3.882	2.540	contains partial rows
-002	-	-	4.31	3.882	2.540	contains partial rows
-003	-	-	4.31	3.882	2.540	contains partial rows
-005	7.11	-	4.31	3.882	2.540	13x8 array
<u>LEU-COMP-THERM-008, Table 10-2</u>						
-001	-	1511	2.459	1.841	1.636	no water rods
-016	-	1158	2.459	1.841	1.636	water rods in lines
-017	-	921	2.459	1.841	1.636	water rods in lines
<u>LEU-COMP-THERM-017, Table 10-2</u>						
-010	9.89	-	2.35	2.918	2.032	steel plates
-011	10.44	-	2.35	2.918	2.032	steel plates
-012	10.44	-	2.35	2.918	2.032	steel plates
-013	9.60	-	2.35	2.918	2.032	steel plates
-014	8.75	-	2.35	2.918	2.032	steel plates
-015	8.57	-	2.35	1.600	1.684	steel plates
-016	9.17	-	2.35	1.600	1.684	steel plates
-017	9.10	-	2.35	1.600	1.684	steel plates
-019	8.87	-	2.35	1.600	1.684	steel plates
-020	8.65	-	2.35	1.600	1.684	steel plates
-021	8.13	-	2.35	1.600	1.684	steel plates
-022	7.26	-	2.35	1.600	1.684	steel plates
<u>LEU-COMP-THERM-042, Table 10-2</u>						
-001	8.28	-	2.35	1.600	1.684	steel plates
-002	4.80	-	2.35	1.600	1.684	borated steel plates
<u>LEU-COMP-THERM-050, Table 10-2</u>						
-001	-	-	4.738	2.032	1.300	center water tank
-002	-	-	4.738	2.032	1.300	center water tank


<sup>a</sup> Enrichment listed for these experiments are the average of all rods in the core. The number of rods for each enrichment is listed in the “comments” column with the <sup>235</sup>U enrichment of the Gd rods being 1.94 wt%.

**Table 6-102 Experiment  $K_{eff}$**

Case	$k_{eff}^a$	$\sigma$
<u>See Table 6-43</u>		
c004	0.9997	0.0020
c005b	0.9997	0.0020
c006b	0.9997	0.0020
c007a	0.9997	0.0020
c008b	0.9997	0.0020
c009b	0.9997	0.0020
c010b	0.9997	0.0020
c011b	0.9997	0.0020
c012b	0.9997	0.0020
c013b	0.9997	0.0020
c014b	0.9997	0.0020
c029b	0.9997	0.0020
c030b	0.9997	0.0020
c031b	0.9997	0.0020
<u>See Table 6-47</u>		
core01	1.0002	0.0005
core03	1.0000	0.0006
core05	0.9999	0.0006
core05a	0.9999	0.0006
core05b	0.9999	0.0006
core08	1.0083	0.0012
core10	1.0001	0.0009
core12a	1.0000	0.0007
core14	1.0030	0.0009
core16	1.0001	0.0010
core18	1.0002	0.0011
core19	1.0002	0.0010
core20	1.0002	0.0010


Case	$k_{eff}^a$	$\sigma$
<u>LEU-COMP-THERM-001, Section 3.5</u>		
-001	0.9998	0.0031
-002	0.9998	0.0031
-003	0.9998	0.0031
-004	0.9998	0.0031
-005	0.9998	0.0031
-006	0.9998	0.0031
-007	0.9998	0.0031
-008	0.9998	0.0031
<u>LEU-COMP-THERM-002, Section 3.5</u>		
-001	0.9997	0.0020
-002	0.9997	0.0020
-003	0.9997	0.0020
-005	0.9997	0.0020
<u>LEU-COMP-THERM-008, Section 3.5</u>		
-001	1.0007	0.0012
-016	1.0007	0.0012
-017	1.0007	0.0012
<u>LEU-COMP-THERM-017, Section 3.5</u>		
-010	1.0000	0.0031
-011	1.0000	0.0031
-012	1.0000	0.0031
-013	1.0000	0.0031
-014	1.0000	0.0031
-015	1.0000	0.0028
-016	1.0000	0.0028
-017	1.0000	0.0028
-019	1.0000	0.0028
-020	1.0000	0.0028
-021	1.0000	0.0028
-022	1.0000	0.0028
<u>LEU-COMP-THERM-042, Section 3.5</u>		
-001	1.0000	0.0016
-002	1.0000	0.0016
<u>LEU-COMP-THERM-050, Section 3.5</u>		
-001	1.0004	0.0010
-002	1.0004	0.0010

<sup>a</sup> When available, the experiment  $k_{eff}$  and uncertainty values given in this table are the benchmark values from the references. The benchmark values are determined in the ICSBEP evaluations by accounting for modeling approximations and simplifications.

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**Table 6-103 SCALE 6.1.3 Results**

Experiment	Enrichment ( <sup>235</sup> U wt%)	Mod/fuel ratio	Pitch (cm)	EALF (eV)	AFG	k <sub>eff</sub>	σ
<u>TN-B1 original set</u>							
c004	4.31	3.8832	2.54	0.1117	207.75	0.9970	0.0005
c005b	4.31	3.8832	2.54	0.1117	207.74	0.9980	0.0005
c006b	4.31	3.8832	2.54	0.1119	207.73	0.9981	0.0005
c007a	4.31	3.8832	2.54	0.1120	207.72	0.9983	0.0005
c008b	4.31	3.8832	2.54	0.1121	207.70	0.9966	0.0005
c009b	4.31	3.8832	2.54	0.1125	207.67	0.9983	0.0004
c010b	4.31	3.8832	2.54	0.1133	207.58	0.9985	0.0005
c011b	4.31	3.8832	2.54	0.1127	207.64	0.9968	0.0005
c012b	4.31	3.8832	2.54	0.1133	207.57	0.9982	0.0005
c013b	4.31	3.8832	2.54	0.1119	207.72	0.9972	0.0005
c014b	4.31	3.8832	2.54	0.1125	207.67	0.9970	0.0005
c029b	4.31	3.8832	2.54	0.1115	207.77	0.9982	0.0005
c030b	4.31	3.8832	2.54	0.1117	207.74	0.9983	0.0005
c031b	4.31	3.8832	2.54	0.1131	207.60	0.9987	0.0005
core01	2.459	1.9385	1.6358	0.2460	198.01	0.9962	0.0004
core03	2.457	1.9282	1.6358	0.2452	198.04	0.9959	0.0004
core05	2.456	1.9385	1.6358	0.2447	198.06	0.9961	0.0004
core05a	2.456	1.9385	1.6358	0.2446	198.06	0.9964	0.0004
core05b	2.456	1.9385	1.6358	0.2453	198.03	0.9955	0.0004
core08	2.455	1.9385	1.6358	0.2450	198.04	0.9965	0.0004
core10	2.455	1.9278	1.6358	0.2451	198.03	0.9960	0.0003
core12a	2.747	1.8714	1.6358	0.3547	193.58	0.9971	0.0005
core14	2.735	1.8735	1.6358	0.3355	194.24	0.9965	0.0005
core16	2.732	1.8741	1.6358	0.3287	194.49	0.9962	0.0004
core18	2.778	1.8866	1.6358	0.3525	193.66	0.9977	0.0005
core19	2.771	1.8878	1.6358	0.3406	194.06	0.9973	0.0004
core20	2.764	1.8891	1.6358	0.3289	194.48	0.9970	0.0004
<u>LEU-COMP-THERM-001</u>							
-001	2.35	2.918	2.032	0.097	208.94	0.9982	0.0004
-002	2.35	2.918	2.032	0.096	209.05	0.9968	0.0005
-003	2.35	2.918	2.032	0.095	209.16	0.9965	0.0004
-004	2.35	2.918	2.032	0.096	209.07	0.9978	0.0004
-005	2.35	2.918	2.032	0.094	209.22	0.9955	0.0004
-006	2.35	2.918	2.032	0.095	209.12	0.9976	0.0004
-007	2.35	2.918	2.032	0.094	209.32	0.9965	0.0006
-008	2.35	2.918	2.032	0.095	209.19	0.9963	0.0004
<u>LEU-COMP-THERM-002</u>							
-001	4.31	3.882	2.540	0.113	207.60	0.9978	0.0005
-002	4.31	3.882	2.540	0.113	207.62	0.9995	0.0005
-003	4.31	3.882	2.540	0.113	207.64	0.9976	0.0005
-005	4.31	3.882	2.540	0.112	207.76	0.9985	0.0005
<u>LEU-COMP-THERM-008</u>							
-001	2.459	1.841	1.636	0.280	196.49	0.9969	0.0004
-016	2.459	1.841	1.636	0.228	198.96	0.9985	0.0004
-017	2.459	1.841	1.636	0.199	200.56	0.9972	0.0004

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**Table 6-103 SCALE 6.1.3 Results (continued)**

Experiment	Enrichment ( <sup>235</sup> U wt%)	Mod/fuel ratio	Pitch (cm)	EALF (eV)	AFG	k <sub>eff</sub>	σ
<u>LEU-COMP-THERM-017</u>							
-010	2.35	2.918	2.032	0.100	208.54	0.9973	0.0004
-011	2.35	2.918	2.032	0.098	208.74	0.9970	0.0005
-012	2.35	2.918	2.032	0.097	208.90	0.9979	0.0004
-013	2.35	2.918	2.032	0.095	209.09	0.9980	0.0004
-014	2.35	2.918	2.032	0.095	209.19	0.9977	0.0004
-015	2.35	1.600	1.684	0.178	201.72	0.9972	0.0004
-016	2.35	1.600	1.684	0.172	202.13	0.9977	0.0005
-017	2.35	1.600	1.684	0.167	202.50	0.9984	0.0004
-019	2.35	1.600	1.684	0.162	202.82	0.9978	0.0005
-020	2.35	1.600	1.684	0.161	202.93	0.9967	0.0004
-021	2.35	1.600	1.684	0.159	203.04	0.9970	0.0005
-022	2.35	1.600	1.684	0.158	203.12	0.9964	0.0005
<u>LEU-COMP-THERM-042</u>							
-001	2.35	1.600	1.684	0.169	202.34	0.9968	0.0005
-002	2.35	1.600	1.684	0.176	201.86	0.9961	0.0004
<u>LEU-COMP-THERM-050</u>							
-001	4.738	2.032	1.300	0.200	200.35	0.9978	0.0005
-002	4.738	2.032	1.300	0.191	200.90	0.9965	0.0005

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**Table 6-104 Data Needed for Calculation of USL for SCALE 6.1.3**

Experiment	Enrichment ( <sup>235</sup> U wt%)	Mod/fuel ratio	Pitch (cm)	EALF (eV)	AFG	k <sub>norm</sub> <sup>a</sup>	σ <sub>norm</sub> <sup>b</sup>
<u>TN-B1 original set</u>							
c004	4.31	3.8832	2.54	0.1117	207.75	0.9973	0.0021
c005b	4.31	3.8832	2.54	0.1117	207.74	0.9983	0.0021
c006b	4.31	3.8832	2.54	0.1119	207.73	0.9984	0.0021
c007a	4.31	3.8832	2.54	0.1120	207.72	0.9986	0.0021
c008b	4.31	3.8832	2.54	0.1121	207.70	0.9969	0.0021
c009b	4.31	3.8832	2.54	0.1125	207.67	0.9986	0.0020
c010b	4.31	3.8832	2.54	0.1133	207.58	0.9988	0.0021
c011b	4.31	3.8832	2.54	0.1127	207.64	0.9971	0.0021
c012b	4.31	3.8832	2.54	0.1133	207.57	0.9985	0.0021
c013b	4.31	3.8832	2.54	0.1119	207.72	0.9975	0.0021
c014b	4.31	3.8832	2.54	0.1125	207.67	0.9973	0.0021
c029b	4.31	3.8832	2.54	0.1115	207.77	0.9985	0.0021
c030b	4.31	3.8832	2.54	0.1117	207.74	0.9986	0.0021
c031b	4.31	3.8832	2.54	0.1131	207.60	0.9990	0.0021
core01	2.459	1.9385	1.6358	0.2460	198.01	0.9960	0.0006
core03	2.457	1.9282	1.6358	0.2452	198.04	0.9959	0.0007
core05	2.456	1.9385	1.6358	0.2447	198.06	0.9962	0.0007
core05a	2.456	1.9385	1.6358	0.2446	198.06	0.9965	0.0007
core05b	2.456	1.9385	1.6358	0.2453	198.03	0.9956	0.0007
core08	2.455	1.9385	1.6358	0.2450	198.04	0.9882	0.0013
core10	2.455	1.9278	1.6358	0.2451	198.03	0.9959	0.0010
core12a	2.747	1.8714	1.6358	0.3547	193.58	0.9971	0.0009
core14	2.735	1.8735	1.6358	0.3355	194.24	0.9935	0.0010
core16	2.732	1.8741	1.6358	0.3287	194.49	0.9961	0.0011
core18	2.778	1.8866	1.6358	0.3525	193.66	0.9975	0.0012
core19	2.771	1.8878	1.6358	0.3406	194.06	0.9971	0.0011
core20	2.764	1.8891	1.6358	0.3289	194.48	0.9968	0.0011
<u>LEU-COMP-THERM-001</u>							
-001	2.35	2.918	2.032	0.097	208.94	0.9984	0.0031
-002	2.35	2.918	2.032	0.096	209.05	0.9970	0.0031
-003	2.35	2.918	2.032	0.095	209.16	0.9967	0.0031
-004	2.35	2.918	2.032	0.096	209.07	0.9980	0.0031
-005	2.35	2.918	2.032	0.094	209.22	0.9957	0.0031
-006	2.35	2.918	2.032	0.095	209.12	0.9978	0.0031
-007	2.35	2.918	2.032	0.094	209.32	0.9967	0.0031
-008	2.35	2.918	2.032	0.095	209.19	0.9965	0.0031



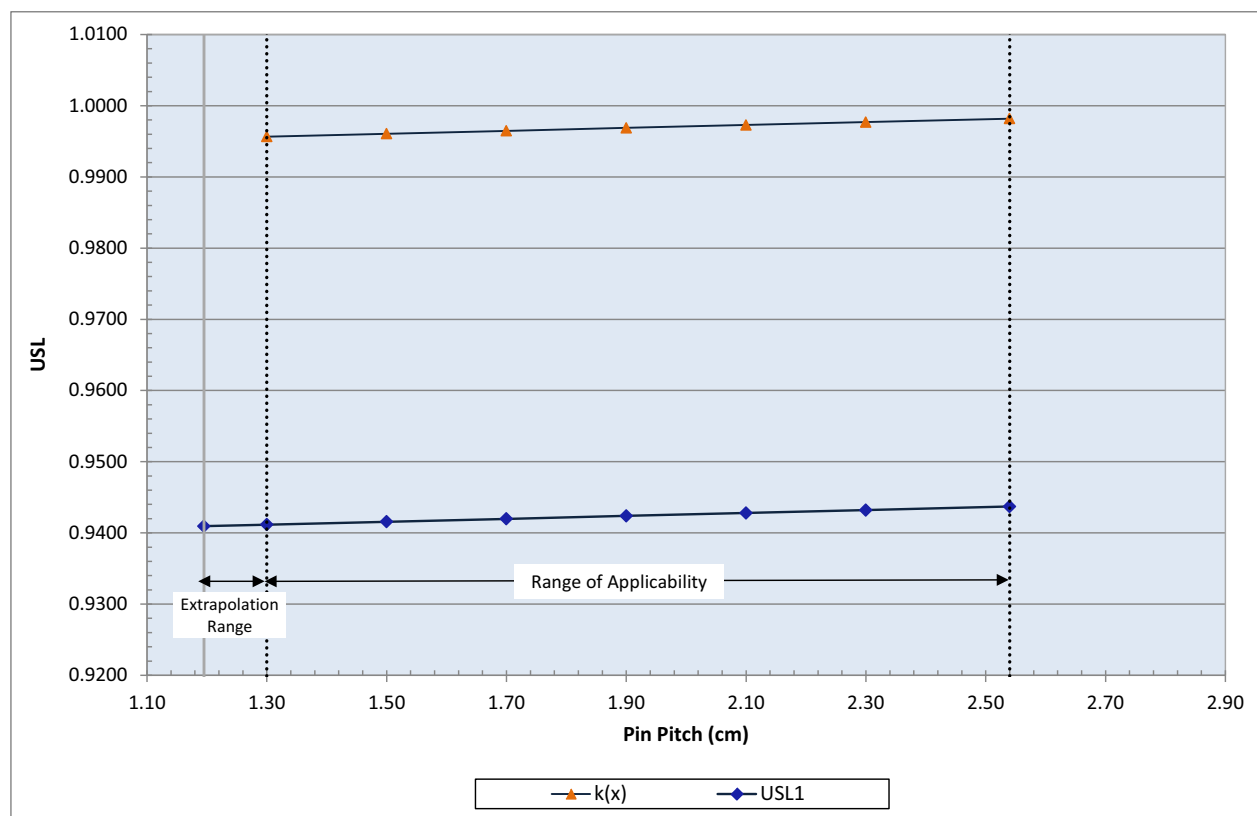
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**Table 6-104 Data Needed for Calculation of USL for SCALE 6.1.3 (continued)**

Experiment	Enrichment ( <sup>235</sup> U wt%)	Mod/fuel ratio	Pitch (cm)	EALF (eV)	AFG	k <sub>norm</sub> <sup>a</sup>	σ <sub>norm</sub> <sup>b</sup>
<u>LEU-COMP-THERM-002</u>							
-001	4.31	3.882	2.540	0.113	207.60	0.9981	0.0021
-002	4.31	3.882	2.540	0.113	207.62	0.9998	0.0021
-003	4.31	3.882	2.540	0.113	207.64	0.9979	0.0021
-005	4.31	3.882	2.540	0.112	207.76	0.9988	0.0021
<u>LEU-COMP-THERM-008</u>							
-001	2.459	1.841	1.636	0.280	196.49	0.9962	0.0013
-016	2.459	1.841	1.636	0.228	198.96	0.9978	0.0013
-017	2.459	1.841	1.636	0.199	200.56	0.9965	0.0013
<u>LEU-COMP-THERM-017</u>							
-010	2.35	2.918	2.032	0.100	208.54	0.9973	0.0031
-011	2.35	2.918	2.032	0.098	208.74	0.9970	0.0031
-012	2.35	2.918	2.032	0.097	208.90	0.9979	0.0031
-013	2.35	2.918	2.032	0.095	209.09	0.9980	0.0031
-014	2.35	2.918	2.032	0.095	209.19	0.9977	0.0031
-015	2.35	1.600	1.684	0.178	201.72	0.9972	0.0028
-016	2.35	1.600	1.684	0.172	202.13	0.9977	0.0028
-017	2.35	1.600	1.684	0.167	202.50	0.9984	0.0028
-019	2.35	1.600	1.684	0.162	202.82	0.9978	0.0028
-020	2.35	1.600	1.684	0.161	202.93	0.9967	0.0028
-021	2.35	1.600	1.684	0.159	203.04	0.9970	0.0028
-022	2.35	1.600	1.684	0.158	203.12	0.9964	0.0028
<u>LEU-COMP-THERM-042</u>							
-001	2.35	1.600	1.684	0.169	202.34	0.9968	0.0017
-002	2.35	1.600	1.684	0.176	201.86	0.9961	0.0016
<u>LEU-COMP-THERM-050</u>							
-001	4.738	2.032	1.300	0.200	200.35	0.9974	0.0011
-002	4.738	2.032	1.300	0.191	200.90	0.9961	0.0011

<sup>a</sup> k<sub>norm</sub> = k<sub>calc</sub> / k<sub>exp</sub>, where k<sub>calc</sub> is shown in Table 6-103 and k<sub>exp</sub> is given in Table 6-102.

<sup>b</sup> σ<sub>norm</sub> =  $\sqrt{\sigma_{calc}^2 + \sigma_{exp}^2}$ , where σ<sub>calc</sub> is shown in Table 6-103 and σ<sub>exp</sub> is given in Table 6-102.



**Figure 6-84: USL as a Function of Pin Pitch**

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### 6.12.10. Sample Input Files

Sample input files are provided for the most reactive 11x11 fuel assembly and 11x11 rod cases.

#### 6.12.10.1. 11x11 Fuel Assembly, HAC Array, Case “5wt\_13gd”

```
=csas5
      TN-B1 container, hac, 5.0wt, 13 gad rods
v7-238
read composition
uo2      1 den=10.763 1 293
                                     92235 5
                                     92238 95  end
      zr      2 0.31309 293  end
polyethylene 2 den=0.949 0.68691 293  end
h2o      3 0.01 293  end
atomuo2   4 10.763 2
                                     92000 1
                                     8016 2
                                     0.9784 293
                                     92235 5
                                     92238 95  end
atomgd2o3  4 7.407 2
                                     64000 2
                                     8016 3
                                     0.0216 293  end
h2o      5 1 293  end
ss304    6 1 293  end
h2o      7 den=1 1 293  end
polyethylene 8 den=0.08 1 293  end
polyethylene 9 den=0.16 1 293  end
atomal2o3 10 0.25 2
                                     13027 2
                                     8016 3
                                     0.49 293  end
atomsio2 10 0.25 2
                                     14000 1
                                     8016 2
                                     0.51 293  end
      zr      12 0.31309 293  end
polyethylene 12 den=0.949 0.68691 293  end
h2o      17 den=1 1 293  end
h2o      18 den=1 1 293  end
h2o      19 den=1 1 293  end
      zr      15 1 293  end
end composition
read celldata
      latticecell squarepitch fuelr=0.4100 1 gapr=0.420 18 cladr=0.5510 2 hpitch=0.6274 7 end
      latticecell squarepitch fuelr=0.4100 4 gapr=0.420 19 cladr=0.5510 12 hpitch=0.6274 17 end
end celldata
read parameter
tme=400
gen=1550
npg=2000
nsk=50
htm=no
end parameter
```

material 1, UO<sub>2</sub> (see Section 6.12.3.2 and Table 6-60)

material 4, entry 1, UO<sub>2</sub> (see Section 6.12.3.2 and Table 6-60)

material 4, entry 2, Gd<sub>2</sub>O<sub>3</sub> (see Section 6.12.3.2 and Table 6-60)

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```

read geometry
unit 1
com="container inner box"
  cuboid 6 1 0.0875 -0.0875 225.2 -228.34 8.829 -8.829
  cuboid 7 1 17.713 -17.713 225.2 -228.34 8.829 -8.829
  hole 4 -8.9002 0 0
  hole 5 8.9002 0 0
  cuboid 6 1 17.8 -17.8 225.2 -228.34 8.829 -8.9165
  cuboid 10 1 22.798 -22.798 225.2 -228.34 8.829 -13.839
  cuboid 6 1 22.798 -22.798 225.34 -228.48 8.829 -13.979
  cuboid 10 1 22.798 -22.798 225.34 -233.44 8.829 -13.979
  cuboid 6 1 22.938 -22.938 225.48 -233.58 8.829 -13.979
unit 2
com="inner box lid"
  cuboid 10 1 22.798 -22.798 229.39 -229.39 2.48 -2.48
  cuboid 6 1 22.938 -22.938 229.53 -229.53 2.62 -2.62
unit 3
com="inner box with ends and lid"
  array 1 0 0 0
unit 4
com="foam polyethylene for left assembly compartment"
  cuboid 7 1 7.1556 -7.1556 225.2 -228.34 7.1556 -7.1556
  hole 70 -6.9014 -192.5 -6.9014
  cuboid 7 1 7.6126 -7.6126 225.2 -228.34 7.629 -7.629
  cuboid 8 1 8.8126 -8.8126 225.2 -228.34 8.829 -7.629
  cuboid 9 1 8.8126 -8.8126 225.2 -228.34 8.829 -8.829
unit 5
com="foam polyethylene for right assembly compartment"
  cuboid 7 1 7.1556 -7.1556 225.2 -228.34 7.1556 -7.1556
  hole 70 -6.9014 -192.5 -6.9014
  cuboid 7 1 7.6126 -7.6126 225.2 -228.34 7.629 -7.629
  cuboid 8 1 8.8126 -8.8126 225.2 -228.34 8.829 -7.629
  cuboid 9 1 8.8126 -8.8126 225.2 -228.34 8.829 -8.829
unit 10
com="5 w/o fuel pins w/o gad - lower level"
  ycyylinder 1 1 0.4100 155.1 0
  ycyylinder 7 1 0.420 155.1 0
  ycyylinder 2 1 0.5510 155.1 0
  cuboid 7 1 0.6274 -0.6274 155.1 0 0.6274 -0.6274
unit 11
com="5 w/o fuel pins w/o gad - middle level"
  ycyylinder 1 1 0.4100 81.7 0
  ycyylinder 7 1 0.420 81.7 0
  ycyylinder 2 1 0.5510 81.7 0
  cuboid 7 1 0.6274 -0.6274 81.7 0 0.6274 -0.6274
unit 12
com="5 w/o fuel pins w/o gad - upper level"
  ycyylinder 1 1 0.4100 148.2 0
  ycyylinder 7 1 0.420 148.2 0
  ycyylinder 2 1 0.5510 148.2 0
  cuboid 7 1 0.6274 -0.6274 148.2 0 0.6274 -0.6274
unit 20
com="space within fuel assembly lattice - lower level"
  cuboid 7 1 0.6274 -0.6274 155.1 0 0.6274 -0.6274
unit 21
com="space within fuel assembly lattice - middle level"
  cuboid 7 1 0.6274 -0.6274 81.7 0 0.6274 -0.6274
unit 22
com="space within fuel assembly lattice - upper level"
  cuboid 7 1 0.6274 -0.6274 148.2 0 0.6274 -0.6274


```

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```

unit 40
com="5 w/o fuel pins w/ gad - lower level"
ycylinder 4 1 0.4100 155.1 0
ycylinder 7 1 0.420 155.1 0
ycylinder 2 1 0.5510 155.1 0
cuboid 7 1 0.6274 -0.6274 155.1 0 0.6274 -0.6274
unit 41
com="5 w/o fuel pins w/ gad - middle level"
ycylinder 4 1 0.4100 81.7 0
ycylinder 7 1 0.420 81.7 0
ycylinder 2 1 0.5510 81.7 0
cuboid 7 1 0.6274 -0.6274 81.7 0 0.6274 -0.6274
unit 42
com="5 w/o fuel pins w/ gad - upper level"
ycylinder 4 1 0.4100 148.2 0
ycylinder 7 1 0.420 148.2 0
ycylinder 2 1 0.5510 148.2 0
cuboid 7 1 0.6274 -0.6274 148.2 0 0.6274 -0.6274
unit 50
com="lower level assembly"
array 2 0 0 0
unit 51
com="middle level fuel assembly"
array 3 0 0 0
unit 52
com="upper level fuel assembly"
array 4 0 0 0
unit 70
com="complete fuel assembly"
array 5 0 0 0
replicate 15 1 0.254 0.254 0.00 0.00 0.254 0.254 1
unit 400
com="outer container body and lid"
cuboid 0 1 35.788 -35.788 247.96 -253.19 29.5 -31.9
hole 3 -22.938 -229.53 -14.024
cuboid 6 1 35.963 -35.963 248.135 -253.365 29.675 -32.075
global unit 500
com="global unit 500 references array 10"
array 10 0 0 0
replicate 5 1 30.48 30.48 30.48 30.48 30.48 30.48 1
end geometry
read array
ara=1 nux=1 nuy=1 nuz=2
com=''
fill
1
2 end fill
ara=2 nux=11 nuy=1 nuz=11
com=''
fill
10 10 10 10 10 10 10 10 10 10 10
10 10 10 10 10 10 10 10 10 10 10
10 10 10 10 10 10 10 10 10 10 10
10 10 10 10 10 10 10 10 10 10 10
10 10 10 10 20 20 20 10 10 10 10
10 10 10 10 20 20 20 10 10 10 10
10 10 10 10 20 20 20 10 40 40 10
10 10 10 10 10 10 10 40 40 40 10
10 10 10 10 10 10 40 40 40 40 10
10 10 10 10 10 10 40 40 40 40 10

```

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```

10 10 10 10 10 10 10 10 10 10 10 end fill
ara=3 nux=11 nuy=1 nuz=11
com=''
fill
21 11 11 11 11 11 11 11 11 11 21
11 11 11 11 11 11 11 11 11 11 11
11 11 21 11 11 11 11 11 21 11 11
11 11 11 11 11 21 11 11 11 11 11
11 11 11 11 21 21 21 11 11 11 11
11 11 11 21 21 21 21 11 41 11 11
11 11 11 11 21 21 21 11 41 41 11
11 11 11 11 11 21 11 11 41 41 11
11 11 21 11 11 11 41 41 21 41 11
11 11 11 11 11 41 41 41 41 41 11
21 11 11 11 11 11 11 11 11 11 21 end fill
ara=4 nux=11 nuy=1 nuz=11
com=''
fill
22 12 12 12 12 12 12 12 12 12 22
12 12 12 12 12 12 12 12 12 12 12
12 12 22 12 12 22 12 12 22 12 12
12 12 12 22 12 22 12 22 12 12 12
12 12 22 22 22 22 22 12 12 12 12
12 12 22 22 22 22 22 22 42 12 12
12 12 12 12 22 22 22 12 42 42 12
12 12 12 22 12 22 12 22 42 42 12
12 12 22 12 12 22 42 42 22 42 12
12 12 12 12 12 42 42 42 42 42 12
22 12 12 12 12 12 12 12 12 12 22 end fill
ara=5 nux=1 nuy=3 nuz=1
fill
52
51
50 end fill
ara=10 nux=10 nuy=1 nuz=10
com=''
fill
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400
400 400 400 400 400 400 400 400 400 400 end fill
end array
read bnds
+xb=vacuum
-xb=vacuum
+yb=vacuum
-yb=vacuum
+zb=vacuum
-zb=vacuum
end bnds
end data
end

```

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### 6.12.10.2. 11x11 Fuel Rod Analysis, HAC Array, Case “mod\_in\_pipe10\_dens010”

'Input generated by GeeWiz SCALE 6.1 Compiled on Tue Sep 6 15:23:32 2011  
=csas5

TN-B1 container, 11, hac, worst case model, 2.0484 cm pitch, rods in pipe  
v7-238

read composition

uo2 1 den=10.763 1 293

material 1, UO<sub>2</sub> (see Section 6.12.3.2  
and Table 6-60)

92235 5  
92238 95 end

polyethylene 2 den=0.949 1 293 end

h2o 3 den=1.00 1 293 end

h2o 5 1 293 end

ss304 6 1 293 end

h2o 7 den=0.10 1 293 end

h2o 8 den=1.00 1 293 end

atomal2o3 10 0.25 2

13027 2  
8016 3

atomsio2 10 0.25 2

0.49 293 end

14000 1  
8016 2

0.51 293 end

end composition

read celldata

latticecell triangpitch fuelr=0.41 1 gapr=0.465 3 cladr=0.48024 2 hpitch=1.1803 8 end  
end celldata

read parameter

tme=400

gen=1550

npg=2000

nsk=50

htm=no

end parameter

read geometry

unit 1

com="container inner box"

cuboid 6 1 0.0875 -0.0875 225.2 -228.34 8.829 -8.829

cuboid 7 1 17.713 -17.713 225.2 -228.34 8.829 -8.829

hole 4 -8.9003 0 0

hole 5 8.9003 0 0

cuboid 6 1 17.8 -17.8 225.2 -228.34 8.829 -8.9165

cuboid 10 1 22.798 -22.798 225.2 -228.34 8.829 -13.839

cuboid 6 1 22.798 -22.798 225.34 -228.48 8.829 -13.979

cuboid 10 1 22.798 -22.798 225.34 -233.44 8.829 -13.979

cuboid 6 1 22.938 -22.938 225.48 -233.58 8.829 -13.979

unit 2

com="inner box lid"

cuboid 10 1 22.798 -22.798 229.39 -229.39 2.48 -2.48

cuboid 6 1 22.938 -22.938 229.53 -229.53 2.62 -2.62

unit 3

com="inner box with ends and lid"

array 1 0 0 0

unit 4




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```

com="foam polyethylene for left assembly compartment"
  cuboid 7 1 8.8126 -5.3274 225.2 -228.34 7.07 -7.07
    hole 70 1.7426 -192.5 0
  cuboid 7 1 8.8126 -8.8126 225.2 -228.34 8.829 -8.829
unit 5
com="foam polyethylene for right assembly compartment"
  cuboid 7 1 5.3274 -8.8126 225.2 -228.34 7.07 -7.07
    hole 70 -1.7426 -192.5 0
  cuboid 7 1 8.8126 -8.8126 225.2 -228.34 8.829 -8.829
unit 10
com="5 w/o fuel pins w/o gad"
  ycylinder 1 1 0.41 385.0 0
  ycylinder 3 1 0.465 385.0 0
  ycylinder 2 1 0.48024 385.0 0
unit 70
com="5-inch ss pipe"
  ycylinder 3 1 7.065 385.0 0
    hole10 -5.9015 0 0
    hole10 -3.5409 0 0
    hole10 -1.1803 0 0
    hole10 1.18030 0
    hole10 3.54090 0
    hole10 5.90150 0
    hole10 -3.5409 0 -4.0887
    hole10 -1.1803 0 -4.0887
    hole10 1.18030 -4.0887
    hole10 3.54090 -4.0887
    hole10 -3.5409 0 4.0887
    hole10 -1.1803 0 4.0887
    hole10 1.18030 4.0887
    hole10 3.54090 4.0887
    hole10 -4.7212 0 -2.0443
    hole10 -2.3606 0 -2.0443
    hole10 0 0 -2.0443
    hole10 2.36060 -2.0443
    hole10 4.72120 -2.0443
    hole10 -4.7212 0 2.0443
    hole10 -2.3606 0 2.0443
    hole10 0 0 2.0443
    hole10 2.36060 2.0443
    hole10 4.72120 2.0443
    hole10 -2.3606 0 -6.1330
    hole10 0 0 -6.1330
    hole10 2.36060 -6.1330
    hole10 -2.3606 0 6.1330
    hole10 0 0 6.1330
    hole10 2.36060 6.1330
unit 400
com="outer container body and lid"
  cuboid 0 1 35.788 -35.788 247.96 -253.19 29.5 -31.9
    hole 3 -22.938 -229.53 -14.024
  cuboid 6 1 35.963 -35.963 248.135 -253.365 29.675 -32.075
global unit 500
  array 10 0 0 0
  replicate 5 1 30.48 30.48 30.48 30.48 30.48 30.48 1
end geometry
read array

```

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```

ara=1 nux=1 nuy=1 nuz=2
  fill
    1
    2 end fill
ara=10 nux=10 nuy=1 nuz=10
  fill
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400
    400 400 400 400 400 400 400 400 400 400 end fill
  end array
read bnds
+xb=vacuum
-xb=vacuum
+yb=vacuum
-yb=vacuum
+zb=vacuum
-zb=vacuum
end bnds
end data
end

```

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## 7. PACKAGE OPERATIONS

This chapter provides general instructions for loading and unloading and operation of the TN-B1 package. Specific detailed procedures based on and consistent with this application are used for the operation of the package. These procedures are maintained by the user of the package and may provide additional detail regarding the handling and operation of the package. Due to the low specific activity and low abundance of gamma emitting radionuclides, dose rates from the contents of the package when used as a Type A or Type B package are minimal. As a result of the low dose rates, there are no special handling requirements for radiation protection.

### 7.1. PACKAGE LOADING

This section delineates the procedures for loading a payload into the TN-B1 packaging. Hereafter, reference to specific TN-B1 packaging components may be found in Appendix 1.4.1.

#### 7.1.1. *Preparation for Loading*

Prior to loading the TN-B1 with fuel, the packaging is inspected to ensure that it is in unimpaired physical condition. The inspection looks for damage, dents, corrosion, and missing hardware. Acceptable conditions are defined by the drawings in Section 1.4.1 as described in Section 8.2.5. Acceptance criteria and detailed loading procedures derived from this application are specified in user written procedures. These user procedures are specific to the authorized content of the package. Since the primary containment is the sealed fuel rod, radiation and contamination surveys are not required prior to loading. There is no required moderator, neutron absorbers or gaskets that require testing or inspection.

Defects that require repair will be fixed prior to shipping in accordance with approved procedures consistent with the quality program.

When used as a Type B package, verification that the primary containment (i.e., fuel rods have been leak checked) will be performed prior to shipping.


#### 7.1.2. *Loading of Contents*

##### 7.1.2.1. **Outer Container Lid Removal**

1. Remove the lid bolts.
2. Attach slings to the four lid lift attachment points on the lid.
3. Remove the outer lid.

##### 7.1.2.2. **Inner Container Removal**

1. Release the inner clamp by removing the eight clamp bolts.

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2. Remove the inner container from the outer container, and move it onto the packing table. Ensure that the inner container is lifted using the inner container handles and not the inner container lid handles.
3. Remove the bolts of the inner container lid and take the lid off.

#### 7.1.2.3. **Loading Fuel Assemblies into the TN-B1**

1. Clamp the inner container body to the packing table or up righting device, and remove the end lid.
2. Ensure that the following preparation work for packing has been completed if required.
  - a. The separators have been inserted.
  - b. The finger spring protectors have been attached.
  - c. The foam has been put in place.
  - d. The fuel assemblies have been covered with poly bags.
3. Stand the packing table upright. (The inner container body is fixed with clamps.)
4. Lift one fuel assembly and pack it in the inner container.
5. After packing one fuel assembly into the inner container, fit the securing fixtures of the fuel assembly. Then pack the other fuel assembly in the inner container
6. Lower the packing table back to the horizontal position from the upright position.
7. Attach the end lid of the inner container.
8. Check to ensure that the fuel assemblies are packaged in the container properly.
9. Attach the inner container lid and tighten the bolts securely (wrench tight or as defined in user procedures).
10. Place the inner container into the outer container.
11. Put on hold down clamps and tighten bolts.
12. Place the outer container lid on the package, and tighten the bolts securely (wrench tight or as defined in user procedures).
13. Install tamper-indicating devices on the outer container ends.

#### 7.1.2.4. **Loading Loose Rods in the Protective Case into the TN-B1**

1. Insert poly endcap spacers over each end of the fuel rod endcap (optional).
2. Sleeve (optional) each rod to be packed with a maximum of 5 mil polyethylene sleeve/tubing.
3. Insert up to 30, 11x11 or 10x10 design rods, 26, 9x9 design rods or 22, 8x8 design rods into the protective case and fill any empty space with empty tubing.

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4. Place cushioning foam pads in protective case as needed to prevent sliding during shipment (optional).
5. Close the protective case and tighten bolts wrench tight.

#### **7.1.2.5. Loading the Protective Case into the TN-B1**

1. Loose rods may be loaded in the protective case while either in the inner container or while removed from the inner container.
2. After packing the protective case(s) into the inner container, fit the securing fixtures for the case.
3. Check to ensure that the protective cases are packaged in the container properly.
4. Attach the inner container lid and tighten the bolts securely (wrench tight or as defined in user procedures).
5. Put on hold down clamps and tighten bolts.
6. Place the outer container lid on the package, and tighten the bolts securely (wrench tight or as defined in user procedures).
7. Install tamper-indicating devices on the outer container ends.
8. It is allowable to ship only one protective case in an TN-B1 inner.

#### **7.1.2.6. Loading Loose Rods in the 5-Inch Stainless Steel Pipe into the TN-B1**

1. Sleeve (optional) each rod to be packed with a maximum of 5 mil polyethylene sleeve/tubing. The ends of the sleeves should be closed in a manner such as knotting or taping with the excess polyethylene trimmed away.
2. Place a cushioning foam pad in the capped end of the pipe (optional).
3. Insert up to 30, 11x11 or 10X10 design rods, 26, 9x9 design rods or 22, 8x8 design rods into the pipe and fill the empty space with empty zircaloy tubing with welded end plugs on both ends.
4. Place cushioning foam pads against the rod ends to block the rods from sliding during shipment (optional).
5. Close pipe with end cap.
6. Lift each 5-inch stainless steel pipe and pack it in the inner container.
7. Check to ensure that the 5-inch stainless steel pipe(s) is packaged in the container properly.
8. Attach the inner container lid and tighten the bolts securely (wrench tight or as defined in user procedures).
9. Place the outer container lid on the package, and tighten the bolts securely (wrench tight or as defined in user procedures).

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10. Install tamper-indicating devices on the outer container ends.
11. It is allowable to ship one or two 5-inch pipes containing rods in an TN-B1 inner.

#### 7.1.2.7. **Loading Loose Rods (25 Maximum Per Side) into the TN-B1**

1. Sleeve (optional) each rod to be packed with a maximum of 5 mil polyethylene sleeve/tubing. The ends of the sleeves should be closed in a manner such as knotting or taping with the excess polyethylene trimmed away.
2. When only one rod per side is to be packed, no clamps are required. Block the rod in the lower corner of the container by evenly spacing 10 or more notched foam pads the length of the rod.
3. When 2 rods up to a maximum of 25 rods are to be packed, banding with steel clamps is not required for criticality safety purposes. If banding is chosen, position 10 or more open steel clamps evenly in each side of the inner container in which loose rods are place.
4. Place foam pads on top of the open clamps, lay the rods on top of the foam.
5. Close and tighten the clamps so the foam surrounds the array of rods. Tighten each clamp until the foam collapses slightly.
6. Place foam pads against the ends of the rods, above the rods and beside the rods to block the rods from moving during shipment.
7. Repeat the above steps for the other side of the inner container, if required.
8. Fill each side (if used) with foam pads so as to minimize movement during shipment.
9. Attach the inner container lid and tighten the bolts securely (wrench tight or as defined by user procedure).
10. Place the outer container lid on the package, and tighten the bolts securely (wrench tight as defined by user procedure).
11. Install tamper-indicating devices on the outer container ends.

#### 7.1.3. ***Preparation for Transport***

When used as a type B package leak testing of the rods (primary containment) is performed during the manufacturing process. Verification of successful leak testing is done prior to shipment. There are no surface temperature measurements required for this package.

#### **Procedure: (These steps may be performed in any sequence.)**

1. Complete the necessary shipping papers in accordance with Subpart C of 49 CFR 172.
2. Ensure that the TN-B1 package markings are in accordance with 10 CFR 71.85(c) and Subpart D of 49 CFR 172. Package labeling shall be in accordance with Subpart E of 49CFR 172. Package placarding shall be in accordance with Subpart F of 49 CFR 172.
3. Survey the surface of the package for potential contamination and dose rates.



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4. Transfer the package to the conveyance and secure using tie-downs secured to the package.

## **7.2. PACKAGE UNLOADING**

### **7.2.1. *Receipt of Package from Carrier***

Radiation and contamination surveys are performed upon receipt of the package and the packages are inspected for significant damage. There are no fission gases, coolants or solid contaminants to be removed.

### **7.2.2. *Removal of Contents***

After freeing the tie downs, the TN-B1 package is lifted from the carrier either by fork lift or by the use of lifting slings placed around the package. If lifted by forklift, the forks are placed at the designated lift locations and the package is lifted. If slings lift the package, a sling is placed under each end of the package at the lifting angles that prevent the sling from sliding. Care should be taken to ensure that the slings are placed in the correct location depending on whether the package is loaded or empty.

#### **7.2.2.1. Outer Container Lid Removal**

1. Remove the lid bolts.
2. Attach slings to the four sling fittings on the lid.
3. Remove the outer lid.

#### **7.2.2.2. Inner Container Removal**

1. Release the inner clamp by removing the eight clamp bolts.
2. Remove the inner container from the outer container, and move it onto the packing table. Ensure that the inner container is lifted using the appropriate inner container handles and not the inner container lid handles.
3. Remove the bolts of the inner container lid and take the lid off.

#### **7.2.2.3. Unloading Fuel Assemblies from the TN-B1**

1. Clamp the inner container body to the packing table or up righting device, and remove the end lid.
2. Stand the packing table upright. (The inner container body is fixed with clamps.)
3. Attach the lifting device to the assembly and remove the securing fixture.
4. Lift one fuel assembly at a time from the package.
5. Repeat for other assembly.

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#### **7.2.2.4. Removing / Unloading Protective Case or 5-Inch Stainless Steel Pipe from the TN-B1**

1. Remove the outer container and inner container lids as described in Sections 7.2.2.1 and 7.2.2.2.
2. The inner container may be removed or left in place while removing the protective case or 5- inch pipe.
3. Remove the 5-inch stainless steel pipe with a sling or remove the cover from the protective case.
4. Remove the rods from the 5-inch pipe or protective case.

### **7.3. PREPARATION OF EMPTY PACKAGE FOR TRANSPORT**

Empty TN-B1's are prepared and transported per the requirements of 49 CFR 173.428. Prior to shipping as an empty TN-B1, the packaging is surveyed to assure that contamination levels are less than the 49 CFR 173.433(a) limit. The TN-B1 is visually verified as being empty. The packaging is inspected to assure that it is in an unimpaired condition and is securely closed so that there will be no leakage of material under conditions normally incident to transportation. Any labels previously applied in conformance with subpart E of part 172 of this subchapter are removed, obliterated, or covered and the "Empty" label prescribed in 49 CFR 172.450 of this subchapter is affixed to the packaging.

### **7.4. OTHER OPERATIONS**

The following are considered normal routine maintenance items and do not require QA or Engineering evaluation for replacement. Material must be of the same type as original equipment parts.

- a. Wooden Bolster Assemblies
- b. Bolster Bolting
- c. Delrin Inserts
- d. Polyethylene Container Guides
- e. Gaskets
- f. Shock Absorbers (Paper Honeycomb)
- g. Fork Pocket Rubber Protective Pads
- h. Outer Container Stopper #2 (Rubber Pad)
- i. Safety Walk
- j. Plastic Plugs
- k. Lid Tightening Bolts (Outer, Inner and End Lid)

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- l. Inner Container End Face Lumber (Upper)
- m. Inner Container End Face Lumber (Lower “Y” Block)
- n. Inner Container Polyethylene Foam
- o. Heliserts

When deviations to items other than those listed above are identified, the TN-B1 shall be removed from service, and the item(s) shall be identified as non-conforming material, and dispositioned in accordance with written procedures including the 10 CFR 71, Subpart H approved QA Plan.

## 7.5. **APPENDIX**

No additional information is required. Loading and unloading this package is a relatively simple and routine operation. The weights, contamination levels and radiation dose rates do not impose significant hazards or operations outside normal material handling.

**Note: The regulatory references provided, such as 49 CFR and 10 CFR, are the current requirements. If regulatory references change, the new references are applicable. This applies throughout the SAR.**

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## 8. ACCEPTANCE TESTS AND MAINTENANCE PROGRAM

### 8.1. ACCEPTANCE TESTS

Per the requirements of subpart G of 10 CFR 71, this section discusses the inspections and tests to be performed prior to first use of the TN-B1. The TN-B1 is manufactured under a Quality Assurance Program meeting the requirements of 10 CFR 71 subpart H.

#### 8.1.1. *Visual Inspections and Measurements*

Prior to the first use of the TN-B1 for the shipment of licensed material, the TN-B1 will be inspected to ensure that it is conspicuously and durably marked with its model number, serial number, gross weight and package identification number assigned by NRC. Prior to applying the model number, it will be determined that the TN-B1 was fabricated in accordance with the drawings reference in the NRC Certificate of Compliance. Critical dimensions related to quality are called out in the Appendix 1.4.1 drawings as Critical to Quality (CTQ). Data for these dimensions is recorded and verified in accordance with the quality plan. Documentation of these measurements is compiled in a data pack. This data pack will be checked for completeness for each TN-B1 as part of the acceptance program. TN-B1's are inspected to ensure that there are no missing parts (nuts, bolts, gaskets, plugs, etc.) or components and that there is no shipping damage on receipt.

#### 8.1.2. *Weld Examinations*

TN-B1 packaging materials of construction and welds shall be examined in accordance with the requirements delineated on the drawings in Appendix 1.4.1, per the requirements of 10 CFR 71.85(a). This includes 100% liquid penetrant examination of the specified areas of the first ten (10) production units. (Note that the first 10 TN-B1 production units were manufactured as RAJ-II units in 2004 and had the required 100% liquid penetrant examination performed on them, no other production units require liquid penetrant testing.)

The non-destructive examination personnel qualification and certification shall be in accordance with either The American Society for Non-destructive Testing (ASNT) SNT-TC-1A (recommended practice) or Japanese Society for Non-destructive Inspection (JSND) Japanese Industrial Standard (JIS) JIS Z 2305 latest revision.

Subsequent production units will be tested as defined in the manufacturing quality plan.

#### 8.1.3. *Structural and Pressure Tests*

The TN-B1 is not pressurized and is structurally the same to the test units. There are no additional structural or pressure tests required.

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#### **8.1.4. *Leakage Tests***

No leak tests of the packaging are required. The fuel rod weld joints are examined at the time of fuel fabrication and leak tested to ensure they are sealed. The welding and leak testing of fuel rods is performed during manufacturing using a qualified process. This process assures that the fuel is acceptable for use in a nuclear reactor core and is tightly controlled. The acceptable leak rate is less than  $1 \times 10^{-7}$  atm-cc/s. The inner and outer container are not relied on for containment, and do not require leak testing.

#### **8.1.5. *Component and Material Tests***

The TN-B1 packaging does not contain gaskets that perform a safety function or pressure boundary, and as such, do not require testing. The packaging does not contain neutron absorbers that would require testing. No component tests are required.

Material testing or certifications from the suppliers of material for this container must show compliance to the properties found in Tables 2-2 and 2-3, or to other properties that satisfactorily indicate compliance to the properties found in these tables and that are approved by the licensee.

#### **8.1.6. *Shielding Tests***

The TN-B1 packaging does not contain shielding and therefore shielding tests are not required.

#### **8.1.7. *Thermal Tests***

The alumina silicate thermal properties will be assured by procuring this material with a certified pedigree. This procurement is done consistent with the QA program.

#### **8.1.8. *Miscellaneous Tests***

There are no additional or miscellaneous tests are required prior to the use of the TN-B1 packaging.

### **8.2. MAINTENANCE PROGRAM**

#### **8.2.1. *Structural and Pressure Tests***

Prior to each use of the TN-B1, the packaging is visually inspected to assure that the packaging is not damaged and that the components parts are in place. The packagings are constructed primarily from stainless steel making it corrosion resistant. Since the packaging is not relied on for containment, there are no pressure test requirements for the inner or outer containers that comprise the packaging. When used as a Type B package, each fuel rod is leak checked and the successful results of the test are checked before shipment.

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The TN-B1 packaging is maintained consistent with a 10 CFR 71 subpart H QA program. Packagings that do not conform to the license drawings are removed from service until they are brought back into compliance. Repairs are performed in accordance with the approved procedures and consistent with the quality assurance program.

### **8.2.2. *Leakage Tests***

Containment is provided by the fuel rod for Type B shipments. The fuel rods are manufactured under a Quality Assurance Program meeting the requirements of 10 CFR 71 Subpart H. Welds of the fuel rod end caps to the cladding are conducted under a qualified process and verified for integrity by such means as X-ray inspection, ultrasonic testing, or process control. For 11x11 fuel rods, the integrity of the closure welds for the fuel rods are periodically assessed using burst testing. This testing is considered successful if the ultimate hoop strength at room temperature obtained is equal or greater than the minimum ultimate strength established from room temperature, longitudinal tensile tests for the same lot. Each loaded fuel rod (of any design) is leak tested after fabrication to assure that the rod is leak tight ( $<1 \times 10^{-7}$  atm-cc/s). Neither the inner or outer container is credited with providing leak protection. Therefore, no leak test of the packaging is required.

### **8.2.3. *Component and Material Tests***

There are no prescribed component tests or replacement requirements for this packaging. The packaging does not use neutron absorbers or shielding that would require testing or maintenance.

### **8.2.4. *Thermal Tests***

The alumina silicate thermal material is sealed within the stainless steel plates of the container wall. The packaging is visually inspected prior to use to assure that the alumina silicate is contained.

### **8.2.5. *Miscellaneous Tests***

There are no additional or miscellaneous tests are required for the use of this packaging. The TN-B1 packaging is inspected prior to each use and maintained consistent with the license drawings. The package is inspected to verify that the package remains within the tolerances specified on the drawings in Section 1.4.1. As noted on the drawings localized deformation in the shell is permitted up to 25mm if the internal structure of the package remains within tolerance. The packaging is repaired in accordance with drawings found in Section 1.4.1.

Foam cushioning material may have up to 2% of the total volume removed for packing purposes, handling or as a result of tears or punctures to the foam.

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Small dents, tears and rounding of corners on paper honeycomb are acceptable providing the area is less than 2%. The corners of the individual pieces of paper honeycomb may be rounded to approximately a radius of 3 inches.

### 8.3. **APPENDIX**

No appendix for this section