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Discussion and Purpose

The calculation is being revised to correct the error of using gauge instead of absolute terms when calculating NPSH for the centrifugal charging pump taking suction from the volume control tank (VCT) during normal operation. The format is also updated to current convention and for clarity. Although the original calculation was in error, the error was in the conservative direction wherein the calculated NPSH available (NPSHa) was one atmosphere or 34 ft below the actual value. Therefore the actual margin is 34 ft higher than calculated.

Design Inputs and Data Sources

1. Elevation of VCT outlet pipe flange = $611'-0\frac{1}{4}" = 611.02'$, per Drawing 1-5441-12, B/8
2. Elevation of pump centerline = $590'-9" = 590.75'$, per Drawing 1-5443-23, C/5.
3. Elevation of Pump suction nozzle is 20" above the centerline, per Pacific Pump Drawing FC-45605
" " " " " " = $590'-9" + 20" = 592'-5" = 592.42'$
4. Design flow = 150 gpm, per UFSAR Table 9.2-3, however use 204 gpm, per note below
Note; Max normal operating flow is letdown flow of 120 gpm (max) plus 60 gpm miniflow, all per UFSAR table 9.2-2, plus 24 gpm seal return flow, per assumption 5 below, for a total flow of 204 gpm.
5. Temperature
Normal suction temperature = 115 ° F, per UFSAR table 9.2-3
VCT hi temp alarm QTI-451 set at 134 ° F with +/- 3 deg uncertainty, per ECP 1-CG-16B
Operating procedure limit of 185 ° F, per 01 OHP 4021.003.001, Rev 19
Use 190 ° F as bounding
6. VCT Pressure = 0 to 60 psig, per UFSAR Table 9.2-3; use 0 psig as a conservatism.
7. NPSH required (NPSHr) at 204 gpm = 12 ft, per representative NPSH curve on Pacific test curve 34617 I.

References

1. Updated Final Safety Analysis (UFSAR)
2. Hydraulic Institute Standards, 12th Edition, 1969
3. Flow of Fluids, Tech Paper No. 410, 14th printing 1974, Crane Co.
4. Procedures 01-OHP 4021.003.001, Rev 19 and 01-OHP 4022.002.001, Rev 8, C. S. 1
5. AEP Drawings 1-5441-12 and 1-5443-23, (Attachment 3, Sheets 1 & 2)
6. Pacific Pump Co Drawing FC-45605 and Test Curve 34617-I, (Attachments 4 & 5)

Method

The NPSHa is defined as the total suction head in feet of liquid absolute, determined at the suction nozzle and referred to datum, less the absolute vapor pressure of the liquid in feet of liquid pumped.

Therefore; (1) $NPSHa = Ha - Hvpa + Hs$, per Hydraulic Institute Standards page 41

Where

Ha = atmospheric pressure in ft absolute (14.7 psi or 34 ft)

$Hvpa$ = vapor pressure of liquid in ft absolute

Hs = total suction head in ft

(2) $Hs = Z + P/\gamma - Hf$

Where

Z = elevation of liquid in tank above pump suction nozzle in ft

P/γ = tank pressure in psig converted to ft of head by the density factor.

Hf = pipe friction in ft between the tank and the pump suction nozzle.

Note; the pipe friction computed for the original calculation will be used for this revision.

The friction value for flows other than the 98 gpm used in the original will be calculated based on the square relationship of flow to friction as shown in the Darcy-Weisbach equation [$Hf = f(L/D)(V^2/2g)$, eq 2-8, pg 2-8 Crane, that is;

(3) $Hf_2 = Hf_1(Q_2/Q_1)^2$.

ENS14720719 FK Rev 1
D 2 15

Acceptance Criteria

Adequate operating conditions exist when the NPSHa exceeds the NPSHr.

Assumptions

1. Based on review of the values, the friction value used in the original calc is assumed to be correct.
2. The level in the VCT is conservatively assumed to be below the low level alarm and actually out of the tank.
3. The liquid temperature is conservatively assumed to be 190 ° F, 5 ° F above the highest procedural limit.
4. It is assumed that the values calculated based on the piping configuration of the unit 1 East pump are applicable to all 4 pump installations. This assumption is made since the slight differences in the unit 1 West and both unit 2 pumps are considered insignificant relative to magnitude of the margin of 12 ft.
5. The extremely conservative assumption that the seal leakoff on all 4 RCPs is at the maximum 6 gpm instead of the nominal 3 gpm. Procedure 01-OHP 4022.002.001, Rev 8, C. S. 1, steps 3 & 15 trips the pump and the unit if the flow on any one RCP rapidly increases to 6 gpm (the max reading on the flow instrument). We are assuming the unprecedented condition where the flow on all 4 RCPs reach 6 gpm simultaneously..

Calculation

Calculate the NPSHa at a flow of 204 gpm, at 190 deg F

1. Establish the friction (Hf) at 204 gpm based on the original HFLC2 computer calc at 98 gpm.
 $Hf_1 = \text{total friction of the four series segments 1 through 4 from the VCT to the pump at 98 gpm}$
 $= 1.12 + 0.04 + 0.01 + 0.05 = \underline{1.22 \text{ ft}}$
 $Hf_2 = \text{friction of the segments at 204 gpm}$
 $= Hf_1(Q_2/Q_1)^2 \text{ [eq 3]} = 1.22\text{ft} \times (204/98)^2 = \underline{5.29 \text{ ft}}$
2. Calculate total suction head
 $H_s = Z + P/\gamma - H_f \text{ [eq 2]} = (611.02' - 592.42') + 0 - 5.29' = \underline{13.31 \text{ ft}}$
3. Calculate NPSHa
 $NPSHa = H_a - H_{vpa} + H_s \text{ [eq 1]}$
vapor pressure of water @ 190 ° F = 9.340 psia per Crane, Pg A-6
weight density of water @ 190 ° F = 60.343 #/ft³ per Crane, Pg A-6
therefore $H_{vpa} = P/\gamma = 9.340 \text{ psia} / (60.343 \text{ #/ft}^3 / 144 \text{ in}^2/\text{ft}^2) = \underline{22.29 \text{ ft}}$
 $NPSHa = 34 - 22.29 + 13.31 = 25.02 \text{ ft, use } \underline{25 \text{ ft}}$
4. Margin of NPSHa over NPSHr = 25 - 12 = 13 ft

Summary of Results and Conclusions

Adequate NPSH exists at the extreme worst case bounding conditions reviewed in this revision since the NPSHa well exceeds the NPSHr with a margin of 13 ft.

DATE 7-19-72

MEMO TO: Centrifugal Charging Pp.
Suction Piping

Attached calc shows that adequate $NPSH_A$ exists from V.C.T. to CC pump "1-E" (worst case) if later pump is used for letdown & charging.

$$NPSH_R \approx (\text{at } 90 \text{ gpm}) \text{ } 10 \text{ ft}$$

$$NPSH_A \approx 18 \text{ ft (with min el. in V.C.T. of } 612' - 6" \text{ or } 1' \text{ ft. above discharge nozzle)}$$

$$\therefore NPSH_A > NPSH_R \text{ by } \sim 8 \text{ ft}$$

From F.K.

* Information phoned to Stan Farlow 7-19-72

FORM OM-14
(AD-16)

Attachment 1 sheet
ENSM 72 0719 FK (Rev)
Page 5 of 15

CENTRIFUGAL CHARGING PUMP (1-E)

FROM V.C.T.

REF. DNGS 1-5441, -5443

ASSUME DESIGN FLOW SAME RECIP. PUMP = 98 gpm

V.C.T. PRESS (MIN) 15 PSIG

FLUID TEMP 250°F → 29.83 psia

CC Pp NPSH_R (@ 98 gpm) = 10 FT From characteristic curve

FRICT = 1.22 → 2

$$NPSH_A = (V.C.T. EL - SUCT. FLG EL) + V.C.T. PRESS - FRICTION - VAP. PRESS$$

SAME WORST CASE

$$V.C.T. EL = NPSH_R + SUCT. FLG EL + FRICT. + VAP. PRESS - V.C.T. PRESS$$

$$= 10' + 592'-6" + 2'$$

$$V.C.T. EL = 604'-6" \text{ minimum el. to satisfy } NPSH_R$$

THIS COMPARES WITH V.C.T. EL (FACE OF NOZZLE) 611'-0 1/4"

TO PRECLUDE VORTEX, V.C.T. MIN EL TO BE 612'-6" → NPSH_R = 18 ft

at 150 gpm NPSH_R = 11 ft

$$FRICT = \left(\frac{150}{98}\right)^2 1.22 = 2.87 \text{ ft} \approx 3.5 \text{ ft}$$

$$V.C.T. EL = 11 + 592'-6" + 3.5 = 607'-6" \text{ (min)}$$

TO PRECLUDE VORTEX, V.C.T. MIN EL 612'-6" (1 1/2 ft above nozzle)

Attachment 1 sheet 2
ENSM 720719 FK Rev 1
Date 6-1-15

HEAD LOSSES FOR CC PUMP SUCTION PIPING (TO 11"E) FROM V.C.T.

U# = TDM47000, AEP000

EEDI VNY1-AEP
ID: ME
SYSTEM- FOR
NEW OR OLD--NEW CCPPC^S

READY
10 250.00015,4
~~20 57.81,445~~
20 98,4.026,57,.81,445
30 98,6.065,6,.2,86
40 98,7.981,17,0,60
50 98,6.065,8.5,0,139
SAVE

READY
OLD HFLC2*

READY
RUN

HFLC2* 11:29EDT 07/19/72

PIPE FRICTION CALCULATIONS-COLEBROOK EQUATION
INPUT DATAFILE?CCPPC

WATER TEMP.(F) = 250.00
DENSITY(LBM/CUFT) = 58.48
ABS.VISCOSITY(LBM/HR/FT) = 0.570931E+00
PIPE ABS.ROUGHNESS(FT) = 0.150000E-03

NO.	FLOW-GPM	DIA(IN)	VEL(FPS)	RE.NO.	HEAD(FT)	TOT.HD.(FT)
1	98.0	4.0	2.5	0.30553E+06	1.12	1.12
2	98.0	6.1	1.1	0.20282E+06	0.04	1.16
3	98.0	8.0	0.6	0.15413E+06	0.01	1.17
4	98.0	6.1	1.1	0.20282E+06	0.05	1.22

ARE THERE MORE RUNS IN THIS DATAFILE(YES-NO)?NO



PIPE FRICTION CALCULATION

00100280100

D1 ANT 2.1.10 BY FK DATE 7-1-72
 SUBJECT CC Plant System P&ID

FLUID TEMP. (°F) 55.7 PIPE REL. ROUGHNESS 0.0015 PIPE SEGMENT NUMBER 1
 FLUID FLOW (GPM) 63 PIPE DIAMETER (IN) 4.5

STRAIGHT PIPE LENGTHS FITTINGS NUMBER *K or % ΣK Σ%

0.1'		GATE VALVE	1	13		2.6
7.5'		GLOBE VALVE		340		
1'		ANGLE VALVE		145		
2.5'		BUTTERFLY VAL		40		
1'		SWING CHECK		135		135
2.5'		90° STD. ELBOW	11	30		120
3'		90° S.R. ELBOW		50		
1.5'		90° L.R. ELBOW		20		20
13.5'		90° MITRE ELBOW		57		
		45° STD. ELBOW		16		64
		45° MITRE ELBOW		15		
		45° S.R. ELBOW		26		
		180° CLOSE RETURN		50		
		* LATERAL < OUTLET		1.0		
		* LATERAL < INLET		0.5		
		* STRAIGHT RUN LATERAL		0.15		
		STD. TEE RUN		20		20
		STD. TEE BRANCH		60		60
		* PIPE ENTR. PROJ. IN WD		0.78		
		* SHARP EDGE		0.50	.5	
		* SLIGHT ROUND		0.23		
		* WELL ROUND		0.04		
		* PIPE EXIT SHARPEDED		1.0		
		* SUDDEN CONTRACTION $d_2/d_1 = 0.3$		0.42		
		* 0.5		0.33		
		* 0.75		0.17		
		* MISC SUDDEN INCREASE $-50 = 31$		(1.21)	.31	
		* MISC SUDDEN ENLARGEMENT		1		
TOTALS	57'				.81	445

* ITEMS ARE "K" VALUES ONLY.

Attachment 2 sheet 2
 ENSM 720719 FK Rev 1
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PLANT 10 10

Mr. F. L.

DATE 7-2-72

D BGCY 10 APR 1978

FLUID TEMP (°F) _____ AIR LB. FLOW/HR _____ AIR SECONDARY NUMBER _____ 2

FLUID FLOW (GPM) 25 PIPE DIAMETER (IN) 2"

STRAIGHT PIPE LENGTHS

६।७।८।९

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	GATE VALVE	13	
	GLOBE VALVE	340	
	ANGLED VALVE	115	
	BUTTERFLY VAL	40	
	SWING CHECK	125	
	90° STD. ELBOW	80	
	90° S.R. ELBOW	50	
	90° L.R. ELBOW	20	
	90° MITER ELBOW	57	
	45° STD. ELBOW	16	
	45° MITER ELBOW	15	
	45° S.R. ELBOW	26	
	180° CURE RETURN	50	
*	LATERAL \angle OUTLET	1.0	
*	LATERAL \angle INLET	0.5	
*	STRAIGHT RUN LATERAL	0.15	
	STD. TEE RUN	20	
	STD. TEE BRANCH	60	
*	PIPE ENTR. PROF. INWD	0.75	
*	SHARP EDGE	0.50	
*	SLIGHT ROUND	0.23	
*	WELL ROUND	0.04	
*	PIPE EXIT SHARPEDED	1.0	
*	SUDDEN CONTRACTION $d_2/d_1 = 0.3$	0.52	
*	0.5	0.33	
*	0.25	0.17	
*	MISC SUDDEN INCREASE	(1.0)	
*	MISC SUDDEN ENLARGE	1	
17'			0
			60

Attachment 2 Sheet 4

ENSM 720719 FK Rev1

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* ITEMS ARE "K" VALUES ONLY..

0 0 1 0 0 2 6 0 7 0 8 PIPE FRICITION CALCULATION

PLANT D. C. Cook BY FLC DATE 7-19-72

SUBJECT _____

FLUID TEMP. (°F) _____ PIPE REL. ROUGHNESS _____ PIPE SEGMENT NUMBER 6

FLUID FLOW (GPM) 25 PIPE DIAMETER (IN) 1.315 1.315

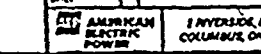
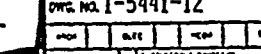
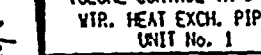
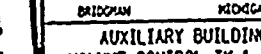
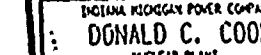
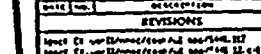
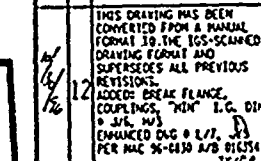
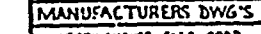
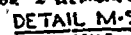
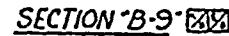
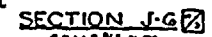
STRAIGHT PIPE LENGTHS FITTINGS NUMBER *K or % ΣK Σ%
 13 13

13			GATE VALVE	1	13		13
1			GLOBE VALVE		340		
1			ANGULAR VALVE		145		
1			BUTTERFLY VAL		40		
			SWING CHECK		135		
			90° STD. ELBOW	111	30		30
			90° S.R. ELBOW		50		
			90° L.R. ELBOW		20		
			90° MITRE ELBOW		57		
			45° STD. ELBOW	1	16		
			45° MITRE ELBOW		15		
			45° S.R. ELBOW		26		
			180° CLOSE RETURN		50		
			* LATERAL < OUTLET		1.0		
			* LATERAL < INLET		0.5		
			* STRAIGHT RUN LATERAL		0.15		
			STD. TEE RUN	1	20		20
			STD. TEE BRANCH		60		
			* PIPE ENTR PROJ. IN WD		0.78		
			* SHARP EDGE		0.50		
			* SLIGHT ROUND		0.23		
			* WELL ROUND		0.04		
			* PIPE EXIT SHARPEDED		1.0		
			* SUDDEN CONTRACTION $d_2/d_1 = 0.3$		0.42		
			* 0.5		0.33		
			* 0.75		0.17		
			* MISC SUDDEN INCREASE		(1.0)		
			* MISC SUDDEN ENLARGE		1		
TOTALS	8.5					0	139

* ITEMS ARE "K" VALUES ONLY.

Attachment 2 Sheet 5
 ENSM 720719FK Rev. 1.
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Attachment 3 sheet 1
ENSM 220719 FK Rev 1
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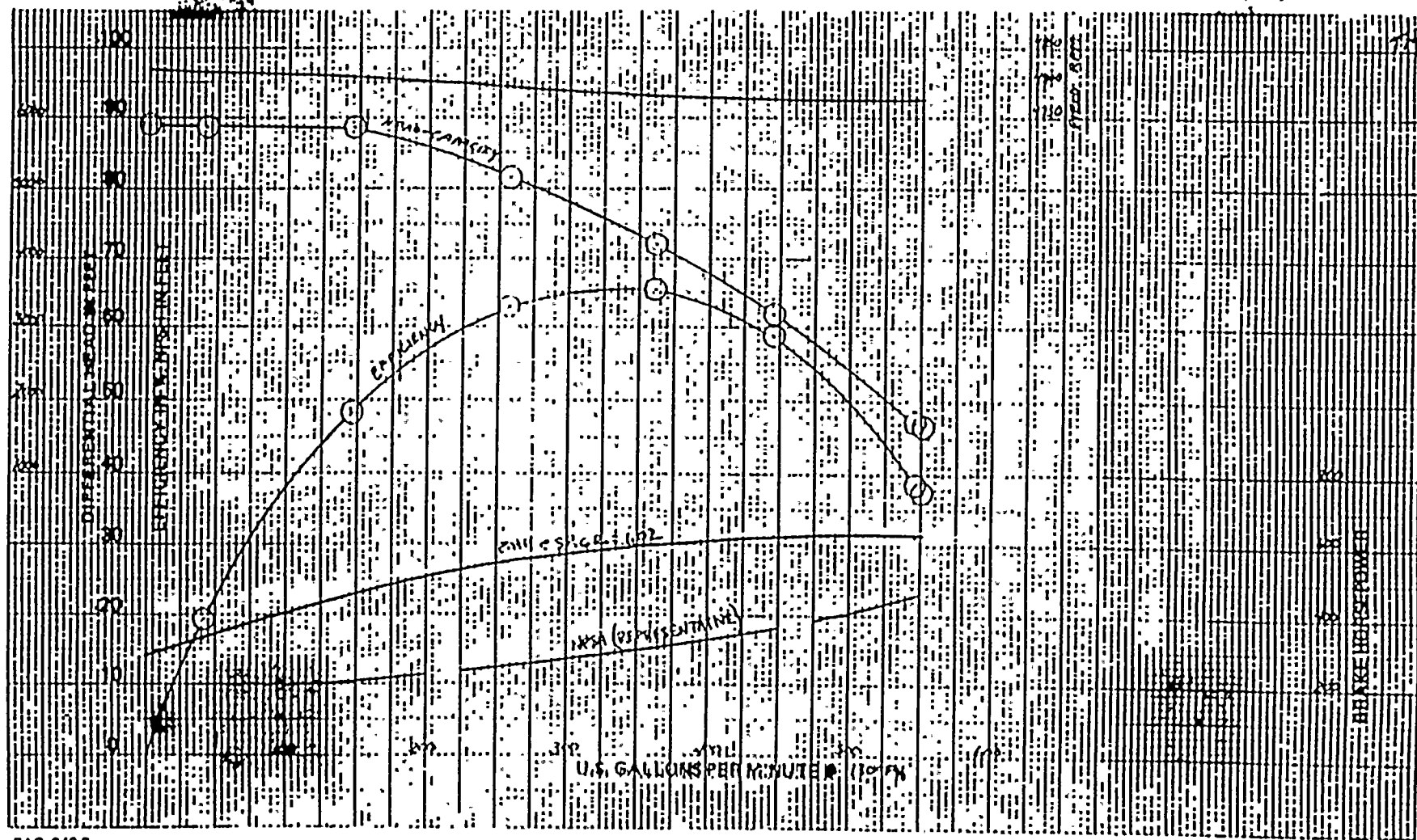
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[illegible]

CONTRACTOR WESTINGHOUSE
CUSTOMER INDIANA E Michigan Electric
ITEM NO. P.O. 11854-041-2X
IMPELLER PATTERN M-7278 M-60221
MAXIMUM DIAMETER 8 1/4 8 1/4
RATED DIAMETER 8 1/4 8 1/4
MINIMUM DIAMETER 7 1/4 7 1/4

TEST PERFORMANCE CURVE NO. 34617-I-51
SIZE 2 1/2" O.C. TYPE IS STAGES 11
R.P.M. FIELD DATE 1/26/84
PUMP NUMBER SMC ELEMENT FOR 3/4 45674-8
PERFORMANCE ALSO APPLIES TO PUMP
NUMBER NE 90973



PAC 2402

Attachment 5
EN SM 720719 FK Rev 1
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EN-015
(1/97)

DONALD C. COOK NUCLEAR PLANT

ENSM

Functional Area

VERIFICATION CHECKLIST - CALCULATIONS

Calculation Number ENSM 720719 FKRev. 1Signature of Verifier *Clark Vankum*7/6/98
Date1.0 Were the inputs/data sources correctly selected, incorporated and documented into the calculation? Yes X N/A Basis: INPUTS WERE CONSERVATIVE AND BOUNDING CONDITIONS FOR CHARGING PUMP NPSH WHILE OPERATING IN THE CIRC/NORMAL CHARGING CAPACITY.2.0 Are assumptions necessary to perform the calculation adequately described and reasonable? Yes X N/A Basis: ASSUMPTIONS HAVE BEEN IDENTIFIED AND JUSTIFICATION WAS PROVIDED AS TO WHY EACH ASSUMPTION IS CONSERVATIVE FOR THIS NPSH CALCULATION.3.0 Are the applicable codes, standards and regulatory requirements identified and requirements for design met? Yes X N/A Basis: DESIGN REQUIRED NPSH AVAILABLE TO EXCEED THE NPSH REQUIRED, EVEN UNDER WORST CASE ASSUMPTIONS. AS SHOWN BY THIS CALCULATION SYSTEM DESIGN CONTAINS ENOUGH MARGIN TO ENSURE ADEQUATE NPSH4.0 Was an appropriate design method used? Yes X N/A Basis: NPSH WAS CALCULATED IN A METHOD CONSISTENT WITH THE METHOD DESCRIBED IN THE HYDRAULIC INSTITUTE STANDARDS 12TH EDITION. THIS METHOD HAS NOT BEEN CHANGED BY SUBSEQUENT EDITIONS.5.0 Is the output reasonable compared to input? Yes X N/A Basis: THE OUTPUT WAS VERIFIED THROUGH AN ALTERNATE CALCULATION, WHICH VERIFIED THE OUTPUT TO BE CORRECT.6.0 Are the results numerically correct? Yes X N/A Basis: AN ALTERNATE CALCULATION HAS BEEN PERFORMED USING A PREVIOUSLY APPROVED PACTO-FIC COMPUTER MODEL. THE MODEL WAS PREVIOUSLY USED IN CALCULATION ENSM 7612-07/6/98 ENSM 961213AF REV 1.

RESULTS OF THE ALTERNATE CALCULATION CONFIRMED THE RESULTS OF THE CALCULATION TO BE CORRECT.

ENSM 720719FK Rev.1 Alternate Calculation Verification

INPUTS

1. Proto-Flo Version 3.04 Model for CVCS Normal charging flow path developed for approved Calculation # ENSM961213AF Rev.1, Approval Date 5/28/97.
2. Model inputs Specific to Charging Pump NPSH Calculation # ENSM 720719FK Rev.1 under worst case normal operating conditions.
 - Suction Temperature 190 ° F
 - VCT Pressure 14.7 psia
 - Seal water injection flow 44 gpm
 - Charging header flow 100 gpm
 - CCP mini-flow in service (model calculated flow at 57.24 gpm vs the 60 gpm assumed by calculation initiator)

Discussion

Charging Pump NPSH calculation was verified using a previously approved Proto-flo model of the CVCS system under calculation ENSM 961213AF Rev. 1. Model inputs were revised slightly to reflect inputs used by calculation ENSM 720719FK Rev. 1. The revised inputs are stated under item #2 of the inputs listed.

The revised Proto-flo model was run 7/6/98 on a Dell computer model # GXMT5133, AEP specific PC # 6738. Results of this alternate calculation along with a CVCS System Nodal Model Drawing have been attached with the Verification Check List. The Proto-flo model automatically calculates NPSHa and compares the value against the NPSHr curve which was input as part of the model design. The verification calculation confirms within one ft the results obtained under ENSM 720719FK Rev.1

I&M - C:\PROTOFLO\CVCS1.PDB -

Cook Plant - ECCS/CCP System

Pump Status Report

Pump Name: PP50W Manufacturer: Model:
Drawings:
Pump Status: OFF
Speed (RPM): Flow (gpm): Head (ft):
Hydraulic Horsepower:
Pump Impeller Datum (ft) Inlet Node Elevation (ft): 597.00
Pump Suction Temperature (°F) 190.00
NPSH Curve: PP50 NPSH
NPSH Available: NPSH Required:

Pump Name: PP50E Manufacturer: Model:
Drawings:
Pump Status: PP50
Speed (RPM): Flow (gpm): 201.25 Head (ft): 5,494.61
Hydraulic Horsepower: 270.37
Pump Impeller Datum (ft) Inlet Node Elevation (ft): 596.33
Pump Suction Temperature (°F) 190.00
NPSH Curve: PP50 NPSH
NPSH Available: 24.21 NPSH Required: 11.02

VERIFIES 25 ft OBTAINED BY
ENSM 720719 FK REV. 1

I&M - C:\PROTOFLO\CVCS1.PDB -

Cook Plant - ECCS/CCP System

Combined Output Report

Convergence: Pressure=1.0E-5 Sum Q=1.0E-2 Friction=1.0E-6 FCV=1.0E-4 PCV=1.0E-3 Temperature=5.0E-3 - Balancing Parameters Used

Pipe: 1.00	ID = 4.260	Flow = 144.01	Vel = 3.242		
	Turbulent f = 0.0177	Reynold's Number = 3.2017E+05			
N1: CS1	** Press= 14.70	Elev= 611.02	Flow= 0.00	Temp= 190.00	
N2: CS2	Press= 14.71	Elev= 610.42	Flow= 0.00	Temp= 190.00	
Pipe: 2.00	ID = 4.260	Flow = 201.25	Vel = 4.530	100.00% Vel	
	Turbulent f = 0.0173	Reynold's Number = 4.4742E+05			
N1: CS2	Press= 14.71	Elev= 610.42	Flow= 0.00	Temp= 190.00	
N2: CS3	Press= 19.39	Elev= 597.00	Flow= 0.00	Temp= 190.00	
Pipe: 4.00	ID = 6.357	Flow = 201.25	Vel = 2.034		
	Turbulent f = 0.0169	Reynold's Number = 2.9983E+05			
N1: CS3	Press= 19.39	Elev= 597.00	Flow= 0.00	Temp= 190.00	
N2: CS5	Press= 19.61	Elev= 596.33	Flow= 0.00	Temp= 190.00	
Pipe: 5.00	ID = 8.329	Flow = 201.25	Vel = 1.185		
	Turbulent f = 0.0169	Reynold's Number = 2.2884E+05			
N1: CS5	Press= 19.61	Elev= 596.33	Flow= 0.00	Temp= 190.00	
N2: CS6	Press= 19.60	Elev= 596.33	Flow= 0.00	Temp= 190.00	
Pipe: 6.00	ID = 6.357	Flow = 201.25	NPSHA = 24.21	NPSHR = 11.02	
N1: CS6	Press= 19.60	Elev= 596.33	Flow= 0.00	Temp= 190.00	
N2: CS7	Press= 2,324.70	Elev= 590.75	Flow= 0.00	Temp= 190.00	
Pipe: 8.00	ID = 3.626	Flow = 201.25	Vel = 6.253		
	Turbulent f = 0.0176	Reynold's Number = 5.2565E+05			
N1: CS7	Press= 2,324.70	Elev= 590.75	Flow= 0.00	Temp= 190.00	
N2: CS9	Press= 2,323.84	Elev= 592.00	Flow= 0.00	Temp= 190.00	
Pipe: 10.00	ID = 1.689	Flow = 57.24	Vel = 8.197	100.00% Vel	
	Turbulent f = 0.0208	Reynold's Number = 3.2095E+05			
N1: CS9	Press= 2,323.84	Elev= 592.00	Flow= 0.00	Temp= 190.00	
N2: CS10	Press= 24.26	Elev= 592.00	Flow= 0.00	Temp= 190.00	
Pipe: 11.00	ID = 2.067	Flow = 57.24	Vel = 5.473		
	Turbulent f = 0.0202	Reynold's Number = 2.6226E+05			
N1: CS10	Press= 24.26	Elev= 592.00	Flow= 0.00	Temp= 190.00	
N2: CS11	Press= 17.21	Elev= 605.58	Flow= 0.00	Temp= 190.00	
Pipe: 12.00	ID = 4.260	Flow = 57.24	Vel = 1.288		
	Turbulent f = 0.0194	Reynold's Number = 1.2725E+05			
N1: CS11	Press= 17.21	Elev= 605.58	Flow= 0.00	Temp= 190.00	
N2: CS12	Press= 16.89	Elev= 606.33	Flow= 0.00	Temp= 190.00	

!! Reverse Flow Thru Check Valve

++ Section Was Balanced

** Fixed Pressure

%% Pressure Below Vapor Pressure

?? Temperature Outside Fluid Property Range

NPSHA less than NPSHR

&& Flow Past End of Pump Curve

225



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Cook Plant - ECCS/CCP System

Combined Output Report

Convergence: Pressure=1.0E-5 Sum Q=1.0E-2 Friction=1.0E-6 FCV=1.0E-4 PCV=1.0E-3 Temperature=5.0E-3 - Balancing Parameters Used

Pipe: 13.00	ID = 4.260	Flow = 57.24	Vel = 1.288		
	Turbulent f = 0.0194		Reynold's Number = 1.2725E+05		
N1: CS12		Press= 16.89	Elev= 606.33	Flow= 0.00	Temp= 190.00
N2: CS2		Press= 14.71	Elev= 610.42	Flow= 0.00	Temp= 190.00

Pipe: 17.00	ID = 3.626	Flow = 144.01	Vel = 4.475		100.00% Vel
	Turbulent f = 0.0179		Reynold's Number = 3.7615E+05		
N1: CS9		Press= 2,323.84	Elev= 592.00	Flow= 0.00	Temp= 190.00
N2: CS16		Press= 2,324.86	Elev= 588.50	Flow= 0.00	Temp= 190.00

Pipe: 18.00	ID = 3.626	Flow = 144.01	Vel = 4.475		
	Turbulent f = 0.0179		Reynold's Number = 3.7615E+05		
N1: CS16		Press= 2,324.86	Elev= 588.50	Flow= 0.00	Temp= 190.00
N2: CS15		Press= 2,321.33	Elev= 596.17	Flow= 0.00	Temp= 190.00

Pipe: 19.00	ID = 2.626	Flow = 144.01	Vel = 8.531		
	Turbulent f = 0.0187		Reynold's Number = 5.1939E+05		
N1: CS15		Press= 2,321.33	Elev= 596.17	Flow= 0.00	Temp= 190.00
N2: CS17		Press= 2,283.15	Elev= 594.58	Flow= 0.00	Temp= 190.00

Pipe: 20.00	ID = 2.626	Flow = 144.01	Vel = 8.531		
	Turbulent f = 0.0187		Reynold's Number = 5.1939E+05		
N1: CS17		Press= 2,283.15	Elev= 594.58	Flow= 0.00	Temp= 190.00
N2: CS18		Press= 2,279.61	Elev= 596.08	Flow= 0.00	Temp= 190.00

Pipe: 21.00	ID = 2.626	Flow = 100.01	Vel = 5.925		
	Turbulent f = 0.0190		Reynold's Number = 3.6069E+05		
N1: CS18		Press= 2,279.61	Elev= 596.08	Flow= 0.00	Temp= 190.00
N2: CS19		Press= 2,282.08	Elev= 588.50	Flow= 0.00	Temp= 190.00

Pipe: 22.00	ID = 2.626	Flow = 100.01	Vel = 5.925		
	Turbulent f = 0.0190		Reynold's Number = 3.6069E+05		
N1: CS19		Press= 2,282.08	Elev= 588.50	Flow= 0.00	Temp= 190.00
N2: CS19A		Press= 2,281.67	Elev= 588.50	Flow= 0.00	Temp= 190.00

Pipe: 23.00	ID = 2.626	Flow = 100.01	++	Vel = 5.925	
	Turbulent f = 0.0190		Reynold's Number = 3.6069E+05		
N1: CS19A		Press= 2,281.67	Elev= 588.50	Flow= 0.00	Temp= 190.00
N2: CS19B		Press= 2,161.75	Elev= 594.50	Flow= 0.00	Temp= 190.00

Pipe: 24.00	ID = 2.626	Flow = 100.01	Vel = 5.925		100.00% Vel
	Turbulent f = 0.0190		Reynold's Number = 3.6069E+05		
N1: CS19B		Press= 2,161.75	Elev= 594.50	Flow= 0.00	Temp= 190.00
N2: CS19C		Press= 2,087.86	Elev= 613.75	Flow= 0.00	Temp= 190.00

!! Reverse Flow Thru Check Valve

++ Section Was Balanced

** Fixed Pressure

%% Pressure Below Vapor Pressure

?? Temperature Outside Fluid Property Range

NPSHA less than NPSHR

&& Flow Past End of Pump Curve

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Cook Plant - ECCS/CCP System

Combined Output Report

Convergence: Pressure=1.0E-5 Sum Q=1.0E-2 Friction=1.0E-6 FCV=1.0E-4 PCV=1.0E-3 Temperature=5.0E-3 - Balancing Parameters Used

Pipe: 25.00	ID = 2.626	Flow = 54.02	Vel = 3.200	100.00% Vel
	Turbulent f = 0.0198		Reynold's Number = 1.9484E+05	
N1: CS19C		Press= 2,087.86	Elev= 613.75	Flow= 0.00 Temp= 190.00
N2: CS20		** Press= 2,085.00	Elev= 615.75	Flow= 0.00 Temp= 190.00
Pipe: 26.00	ID = 2.626	Flow = 45.99	Vel = 2.724	100.00% Vel
	Turbulent f = 0.0200		Reynold's Number = 1.6585E+05	
N1: CS19C		Press= 2,087.86	Elev= 613.75	Flow= 0.00 Temp= 190.00
N2: CS21		** Press= 2,085.00	Elev= 615.75	Flow= 0.00 Temp= 190.00
Pipe: 35.00	ID = 2.626	Flow = 44.00	Vel = 2.607	
	Turbulent f = 0.0201		Reynold's Number = 1.5870E+05	
N1: CS18		Press= 2,279.61	Elev= 596.08	Flow= 0.00 Temp= 190.00
N2: CS30		Press= 2,277.04	Elev= 601.81	Flow= 0.00 Temp= 190.00
Pipe: 36.00	ID = 2.626	Flow = 44.00	Vel = 2.607	
	Turbulent f = 0.0201		Reynold's Number = 1.5870E+05	
N1: CS30		Press= 2,277.04	Elev= 601.81	Flow= 0.00 Temp= 190.00
N2: CS31		Press= 2,280.91	Elev= 591.75	Flow= 0.00 Temp= 190.00
Pipe: 37.00	ID = 2.626	Flow = 44.00	Vel = 2.607	
	Turbulent f = 0.0201		Reynold's Number = 1.5870E+05	
N1: CS31		Press= 2,280.91	Elev= 591.75	Flow= 0.00 Temp= 190.00
N2: CS32		Press= 2,280.86	Elev= 591.75	Flow= 0.00 Temp= 190.00
Pipe: 38.00	ID = 1.689	Flow = 44.00	Vel = 6.301	
	Turbulent f = 0.0211		Reynold's Number = 2.4674E+05	
N1: CS32		Press= 2,280.86	Elev= 591.75	Flow= 0.00 Temp= 190.00
N2: CS33		Press= 2,279.64	Elev= 587.84	Flow= 0.00 Temp= 190.00
Pipe: 43.00	ID = 2.626	Flow = 44.00	Vel = 2.607	
	Turbulent f = 0.0201		Reynold's Number = 1.5870E+05	
N1: CS33		Press= 2,279.64	Elev= 587.84	Flow= 0.00 Temp= 190.00
N2: CS37		Press= 2,279.55	Elev= 587.84	Flow= 0.00 Temp= 190.00
Pipe: 44.00	ID = 2.626	Flow = 44.00	Vel = 2.607	
	Turbulent f = 0.0201		Reynold's Number = 1.5870E+05	
N1: CS37		Press= 2,279.55	Elev= 587.84	Flow= 0.00 Temp= 190.00
N2: CS38		Press= 2,279.49	Elev= 587.84	Flow= 0.00 Temp= 190.00
Pipe: 45.00	ID = 2.626	Flow = 44.00	Vel = 2.607	
	Turbulent f = 0.0201		Reynold's Number = 1.5870E+05	
N1: CS38		Press= 2,279.49	Elev= 587.84	Flow= 0.00 Temp= 190.00
N2: CS39		Press= 2,276.66	Elev= 594.00	Flow= 0.00 Temp= 190.00

!! Reverse Flow Thru Check Valve

++ Section Was Balanced

** Fixed Pressure

%% Pressure Below Vapor Pressure

?? Temperature Outside Fluid Property Range

NPSHA less than NPSHR

&& Flow Past End of Pump Curve

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Cook Plant - ECCS/CCP System

Combined Output Report

Convergence: Pressure=1.0E-5 Sum Q=1.0E-2 Friction=1.0E-6 FCV=1.0E-4 PCV=1.0E-3 Temperature=5.0E-3 - Balancing Parameters Used

Pipe: 46.00	ID = 1.689	Flow = 11.00	++	Vel = 1.575	100.00% Vel
	Turbulent f = 0.0236		Reynold's Number = 6.1685E+04		
N1: CS39		Press= 2,276.66	Elev= 594.00	Flow= 0.00	Temp= 190.00
N2: CS40		Press= 2,085.38	Elev= 616.50	Flow= 0.00	Temp= 190.00
Pipe: 47.00	ID = 1.689	Flow = 11.00		Vel = 1.575	100.00% Vel
	Turbulent f = 0.0236		Reynold's Number = 6.1685E+04		
N1: CS40		Press= 2,085.38	Elev= 616.50	Flow= 0.00	Temp= 190.00
N2: CS41		Press= 2,085.30	Elev= 616.50	Flow= 0.00	Temp= 190.00
Pipe: 48.00	ID = 1.338	Flow = 11.00		Vel = 2.510	
	Turbulent f = 0.0238		Reynold's Number = 7.7867E+04		
N1: CS41		Press= 2,085.30	Elev= 616.50	Flow= 0.00	Temp= 190.00
N2: CS42		** Press= 2,085.00	Elev= 616.94	Flow= 0.00	Temp= 190.00
Pipe: 49.00	ID = 2.626	Flow = 33.00		Vel = 1.955	
	Turbulent f = 0.0207		Reynold's Number = 1.1902E+05		
N1: CS39		Press= 2,276.66	Elev= 594.00	Flow= 0.00	Temp= 190.00
N2: CS43		Press= 2,276.32	Elev= 594.75	Flow= 0.00	Temp= 190.00
Pipe: 50.00	ID = 1.689	Flow = 11.00	++	Vel = 1.575	100.00% Vel
	Turbulent f = 0.0236		Reynold's Number = 6.1685E+04		
N1: CS43		Press= 2,276.32	Elev= 594.75	Flow= 0.00	Temp= 190.00
N2: CS44		Press= 2,085.13	Elev= 616.94	Flow= 0.00	Temp= 190.00
Pipe: 51.00	ID = 1.689	Flow = 11.00		Vel = 1.575	100.00% Vel
	Turbulent f = 0.0236		Reynold's Number = 6.1685E+04		
N1: CS44		Press= 2,085.13	Elev= 616.94	Flow= 0.00	Temp= 190.00
N2: CS45		Press= 2,085.09	Elev= 616.94	Flow= 0.00	Temp= 190.00
Pipe: 52.00	ID = 1.338	Flow = 11.00		Vel = 2.510	
	Turbulent f = 0.0238		Reynold's Number = 7.7867E+04		
N1: CS45		Press= 2,085.09	Elev= 616.94	Flow= 0.00	Temp= 190.00
N2: CS46		** Press= 2,085.00	Elev= 616.94	Flow= 0.00	Temp= 190.00
Pipe: 53.00	ID = 2.626	Flow = 22.00		Vel = 1.303	
	Turbulent f = 0.0217		Reynold's Number = 7.9350E+04		
N1: CS43		Press= 2,276.32	Elev= 594.75	Flow= 0.00	Temp= 190.00
N2: CS47		Press= 2,276.00	Elev= 595.50	Flow= 0.00	Temp= 190.00
Pipe: 54.00	ID = 2.626	Flow = 11.00	++	Vel = 0.652	100.00% Vel
	Turbulent f = 0.0240		Reynold's Number = 3.9675E+04		
N1: CS47		Press= 2,276.00	Elev= 595.50	Flow= 0.00	Temp= 190.00
N2: CS48		Press= 2,084.23	Elev= 619.00	Flow= 0.00	Temp= 190.00

Reverse Flow Thru Check Valve

++ Section Was Balanced

** Fixed Pressure

% Pressure Below Vapor Pressure

?? Temperature Outside Fluid Property Range

NPSHA less than NPSHR

&& Flow Past End of Pump Curve

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Cook Plant - ECCS/CCP System

Combined Output Report

Convergence: Pressure=1.0E-5 Sum Q=1.0E-2 Friction=1.0E-6 FCV=1.0E-4 PCV=1.0E-3 Temperature=5.0E-3 - Balancing Parameters Used

Pipe: 55.00	ID = 2.626	Flow = 11.00	Vel = 0.652	100.00% Vel
	Turbulent f = 0.0240	Reynold's Number = 3.9675E+04		
N1: CS48		Press= 2,084.23	Elev= 619.00	Flow= 0.00 Temp= 190.00
N2: CS49		Press= 2,085.63	Elev= 615.58	Flow= 0.00 Temp= 190.00
Pipe: 56.00	ID = 1.338	Flow = 11.00	Vel = 2.510	
	Turbulent f = 0.0238	Reynold's Number = 7.7867E+04		
N1: CS49		Press= 2,085.63	Elev= 615.58	Flow= 0.00 Temp= 190.00
N2: CS50		** Press= 2,085.00	Elev= 616.94	Flow= 0.00 Temp= 190.00
Pipe: 57.00	ID = 2.626	Flow = 11.00	Vel = 0.652	
	Turbulent f = 0.0240	Reynold's Number = 3.9675E+04		
N1: CS47		Press= 2,276.00	Elev= 595.50	Flow= 0.00 Temp= 190.00
N2: CS51		Press= 2,275.68	Elev= 596.25	Flow= 0.00 Temp= 190.00
Pipe: 58.00	ID = 2.626	Flow = 11.00 ++	Vel = 0.652	100.00% Vel
	Turbulent f = 0.0240	Reynold's Number = 3.9675E+04		
N1: CS51		Press= 2,275.68	Elev= 596.25	Flow= 0.00 Temp= 190.00
N2: CS52		Press= 2,085.33	Elev= 616.50	Flow= 0.00 Temp= 190.00
Pipe: 59.00	ID = 2.626	Flow = 11.00	Vel = 0.652	100.00% Vel
	Turbulent f = 0.0240	Reynold's Number = 3.9675E+04		
N1: CS52		Press= 2,085.33	Elev= 616.50	Flow= 0.00 Temp= 190.00
N2: CS53		Press= 2,085.30	Elev= 616.50	Flow= 0.00 Temp= 190.00
Pipe: 60.00	ID = 1.338	Flow = 11.00	Vel = 2.510	
	Turbulent f = 0.0238	Reynold's Number = 7.7867E+04		
N1: CS53		Press= 2,085.30	Elev= 616.50	Flow= 0.00 Temp= 190.00
N2: CS54		** Press= 2,085.00	Elev= 616.94	Flow= 0.00 Temp= 190.00

Reverse Flow Thru Check Valve

++ Section Was Balanced

** Fixed Pressure

%% Pressure Below Vapor Pressure

?? Temperature Outside Fluid Property Range

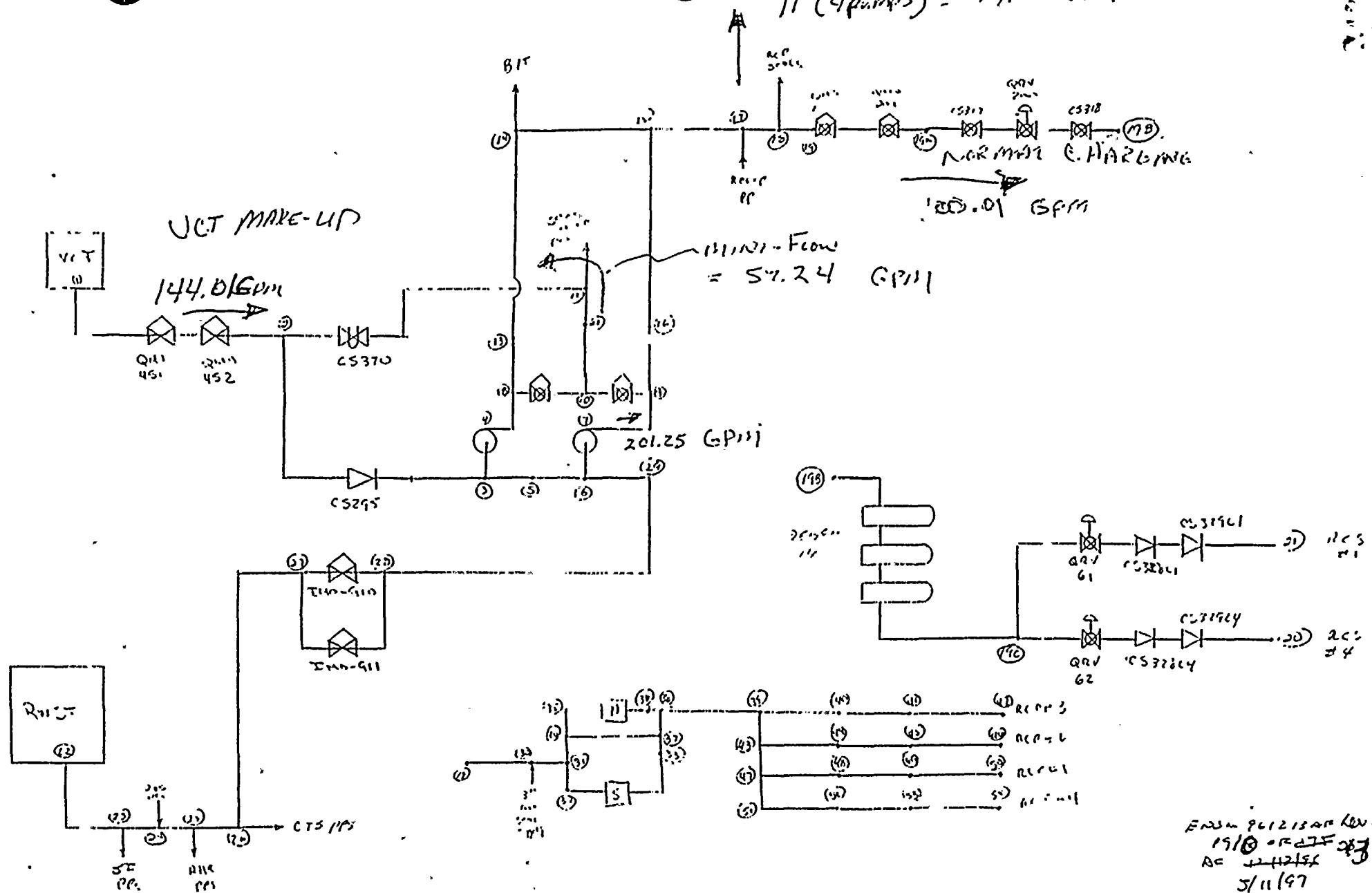
NPSHA less than NPSHR

&& Flow Past End of Pump Curve

2000



11 (4 pumps) = 44,000 GPM



20

