

**ATTACHMENT 4**

**TO ENTERGY LETTER 2.14.023**

**PILGRIM RELIEF REQUEST PRR-25**

**Calculation Cover Page EC # 49514**

**Flaw Evaluation of SSW Discharge Piping Leaking Elbow**

**Structural Integrity Associates Calculation No. 1400287.302, Rev. 0**

**(20 Pages)**

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**III. CROSS REFERENCES:**

1. ASME B&PV Code , Section XI, App C, 2001 Edition w/ Add through 2003
2. ASME B31.1, Power Piping, 1967 Edition
3. ASME Code Case N-513-3
4. Flow of Fluids Through valves, Fittings and Pipe, Crane Co.,, Technical Paper No. 410

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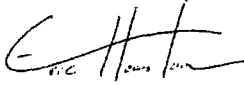



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## 1.0 INTRODUCTION

A weeping flaw, shown on Figure 1, was discovered near the extrados of a 90 degree elbow in the Salt Service Water (SSW) piping at Pilgrim Nuclear Power Station (Pilgrim). The leak is located on the JF29-8-4 pipe spool of the SSW system [1]. Ultrasonic testing has been conducted in order to characterize the flaw [1]. Allowable through-wall flaw lengths are determined using methods consistent with an upcoming revision of Code Case N-513-3 [2] as described below.

## 2.0 TECHNICAL APPROACH

The flaw evaluation herein is based on the criteria prescribed in an upcoming revision of ASME Code Case N-513-3. This Code Case allows for the temporary acceptance of through-wall flaws in moderate energy Class 2 or Class 3 piping. N-513-3 has been conditionally accepted by the NRC with the stipulation that, "The repair or replacement activity temporarily deferred under the provisions of this Code Case shall be performed during the next scheduled outage," and is published in the latest revision of Regulatory Guide 1.147 [3]. N-513-3 allows non-planar, through-wall flaws to be characterized and evaluated as planar (i.e., crack-like), through-wall flaws in the axial and circumferential directions.

The evaluation criteria provided in N-513-3 are only for straight pipe since the technical approach relies on ASME Section XI, Appendix C [4] methods. A new revision of the Code Case (N-513-4) includes rules for the evaluation of piping components such as elbows, branch tees and reducers. Flaws in these components may be evaluated as if in straight pipe provided the stresses used in the evaluation are adjusted to account for geometric differences. For elbows, hoop stress is adjusted by considering flaw location and primary stress due to elbow ovalization from axial loads. For axial stresses, the stress scaling follows the same approach given in ASME Section III, ND-3600 [5] design by rule using stress indices and stress intensification factors for the adjustment. Details are provided in N-513-4 for determining these adjusted stresses.

N-513-4 has not been approved by the ASME or reviewed by the NRC; however, it is recognized in ASME committee that the technical approach is very conservative. Simple treatment of piping component flaw evaluation using hand calculations was an important objective in the development of the approach recognizing the trade-off being conservative results. N-513-4 allows for more sophisticated analysis by the user.

As stated above, Code Case N-513-3 evaluation criteria rely on the methods given in ASME Section XI, Appendix C. Linear Elastic Fracture Mechanics (LEFM) criteria are conservatively employed as described in Article C-7000. Since a through-wall flaw is being evaluated, through-wall shape factors  $F_m$ ,  $F_b$  and  $F$  are used which are given in Appendix I of the Code Case. Allowable flaw lengths are determined through iteration comparing calculated stress intensity factors to a critical fracture toughness defined in C-7200 of Section XI, Appendix C.

This evaluation utilizes finite element methods (FEM) to calculate the primary membrane stress in the hoop direction due to ovalization from axial loads. Section 3.3 of the Code Case's new revision states

that “Alternative methods may be used to calculate the stresses used in evaluation,” which justifies the use of FEM techniques.

Details of the Code Case N-513-4 evaluation procedure for elbows are given in Appendix A.

### **3.0 DESIGN INPUTS / ASSUMPTIONS**

The SSW Code of Construction is ANSI B31.1 1967 Edition [6].

Based on information provided by Entergy, the 18 inch elbow is located on SSW spool JF29-8-4 [1]. The 90 degree elbow located on JF29-8-4 is a schedule 20, long radius elbow [7]. The design pressure and temperature are 10 psig and 100°F, respectively [8].

The elbow material is ASTM A-234 WPB [7] carbon steel. For the analysis, A106 Gr. B carbon steel is judged to have equivalent material properties. The nominal composition of the two materials is essentially the same and the minimum yield and tensile strengths are the same for both materials. In addition, the longitudinal and transverse elongations are similar between these materials.

The applied moment loadings are obtained from the ME-101 output listings in Reference [9]. Based on information provided by Entergy, the location of interest is node 22. The moments for each load case are provided in three dimensions (MA, MB, and MC), which are combined by square-root-of-the-sum-of-the-squares (SRSS). The resulting SRSS moments at each location along the elbow (beginning, middle, and end) are compared for each loading, and the bounding moment is used in this analysis.

Determination of the fracture toughness,  $J_{IC}$ , used in the evaluation is based on Section XI, Appendix C, C-8320 [4], which specifies that ‘reasonable lower bound fracture toughness data’ may be used to determine the allowable stress intensity factor,  $K_{IC}$ . The NRC’s Pipe Fracture Encyclopedia [10] contains numerous CVN test results for A106 Gr. B carbon steel at low temperature, which are reproduced in Table 1. The minimum reported value of 293 in-lb/in<sup>2</sup> is used in the analysis for both axial and circumferential flaws.

Finite element methods are used to determine the primary membrane stress in the hoop direction due to ovalization from axial loads in Reference [11]. A unit moment of 10,000 in-lbs is applied to the FEM and linearized stresses are extracted at paths in the axial direction from the flaw. A stress of 100 psi conservatively bounds the tensile hoop stress reported in Reference [11]. This bounding stress is factored based on the ratio of the applied moment for the applicable service level to the unit moment of 10,000 in-lbs. The factored stress is used as described in Section 4.1.1 below.

The following design inputs are used in this calculation:

1. Long radius 90° elbow OD = 18 inches [7]
2. Nominal elbow thickness = 0.312 inch (based on Schedule 20 piping [7])
3. Design pressure = 10 psig [8]

4. Design temperature = 100°F [8]
5. Young's modulus = 27,900 ksi [6, Table C-1]
6. Allowable stress = 15 ksi [6, Table A-2]
7. Enveloped SRSS Deadweight Moment = 43,973 in-lbs [9]
8. Enveloped SRSS OBE Moment = 38,820 in-lbs [9]
9. Enveloped SRSS SSE Moment = 72,789 in-lbs [9]
10. Enveloped SRSS Thermal Moment = 22,047 in-lbs [9]
11. Stress intensification factor,  $i = 3.98$  [6]
12.  $J_{IC}$  for axial flaws = 293 in-lb/in<sup>2</sup> [4, 10]
13.  $J_{IC}$  for circumferential flaws = 293 in-lb/in<sup>2</sup> [4, 10]
14. Bounding primary membrane stress in the hoop direction due to unit moment load = 100 psi [11]

Note that the wall thickness surrounding the flaw is greater than the elbow nominal thickness [1]. Therefore, the use of the 0.312 inch surrounding wall thickness is considered conservative.

The following assumptions are used in this calculation:

1. Poisson's ratio is assumed to be 0.3.
2. Due to the flaw remoteness from a weld, residual stress effects are assumed negligible.
3. A corrosion allowance is not considered (the ongoing inspection requirements in Code Case N-513-3 address the possibility of flaw growth during the temporary acceptance period).

## 4.0 CALCULATIONS

### 4.1 Applied Loads

#### 4.1.1 Hoop Stress

For the allowable axial flaw length, the hoop stress,  $\sigma_h$ , due to internal pressure and elbow ovalization from the axial moments may be determined from Equation 9 of N-513-4 (see Appendix A):

$$\sigma_h = \left( \frac{pD_o}{2t} \right) \left[ \frac{2R_{bend} + R_o \sin \phi}{2(R_{bend} + R_o \sin \phi)} \right] + \left( \frac{1.95}{h^{2/3}} \right) \frac{R_o M_b}{I} \quad (1)$$

where:

- $p$  = internal pressure, psig
- $D_o$  = outside diameter, in
- $t$  = wall thickness, in
- $R_{bend}$  = elbow bend radius (27 inches)
- $R_o$  = outside radius, in
- $\phi$  = circumferential angle from elbow flank (see Figure 7 in Appendix A)



$h$  = flexibility characteristic =  $t \cdot R_{\text{bend}} / (R_{\text{mean}})^2$  [6]

$R_{\text{mean}}$  = elbow mean radius, in

$M_b$  = primary bending moment, in-lbs

$I$  = moment of inertia, in<sup>4</sup>.

Note that the first term of Equation 1 accounts for the hoop stress due to internal pressure and includes a scaling factor to account for the circumferential location of the flaw (assuming uniform thickness, pressure based hoop stress is a maximum at the elbow intrados, while a minimum at the elbow extrados). At the flank, the pressure based hoop stress is equal to that of straight pipe. For the analysis herein, it is conservative to set  $\phi = 0$  since the flaw is between the flank and extrados as shown on Figure 2.

The second term of Equation 1 accounts for the hoop stress resulting from the axial moments acting to ovalize the elbow. This term is replaced with the scaled primary membrane stress in the hoop direction as discussed in the previous section.

Finally, N-513-4 limits the use of Equation 1 for  $h \geq 0.1$ . For this elbow,  $h \approx 0.11$ .

#### 4.1.2 Axial Stresses

For the allowable circumferential flaw length, the axial stress due to pressure, deadweight, seismic, and thermal loading is presented below. For axial membrane stress due to pressure,  $\sigma_m$ , Equation 10 of N-513-4 is used:

$$\sigma_m = B_1 \left( \frac{p D_o}{2t} \right) \quad (2)$$

where  $B_1$  is an ASME Section III primary stress index for internal pressure. N-513-4 sets this value to 0.5.

For axial bending stress,  $\sigma_b$ , due to deadweight and seismic moments, Equation 11 of N-513-4 may be used:

$$\sigma_b = B_2 \left( \frac{R_o M_b}{I} \right) \quad (3)$$

where  $B_2$  is an ASME Section III primary stress index for moment loading. From Figure ND-3673.2(b)-1 of Reference [5],  $B_2 = 1.30/h^{2/3}$ . For this elbow,  $B_2 = 5.74$ .

For axial bending stress due to thermal moments,  $\sigma_e$ , Equation 12 of N-513-4 may be used:

$$\sigma_e = i \left( \frac{R_o M_e}{I} \right) \quad (4)$$

where  $i$  is the stress intensification factor. From [6, Appendix D],  $i = 3.98$ .

#### 4.2 Stress Intensity Factor Calculations

For LEFM analysis, the stress intensity factor,  $K_I$ , for an axial flaw is taken from Article C-7000 [4] as prescribed by N-513-3 and is given below:

$$K_I = K_{Im} + K_{Ir} \quad (5)$$

where:

$$K_{Im} = (SF_m)F\sigma_h(\pi a/Q)^{0.5}$$

$SF_m$  = structural factor for membrane stress (see Table 2)

$F$  = through-wall shape factor for an axial flaw under hoop stress (given in Appendix I of N-513-3)

$\sigma_h$  = hoop stress, ksi

$a$  = flaw depth (half flaw length for through-wall flaw), in

$Q$  = flaw shape parameter (unity per Appendix I of N-513-3)

$K_{Ir} = K_I$  from residual stresses at flaw location (assumed negligible).

Only the hoop stress influences the allowable axial flaw length which is a function of pressure and primary bending stress.

For LEFM analysis, the stress intensity factor,  $K_I$ , for a circumferential flaw is taken from Article C-7000 [4] as prescribed by N-513-3 and is given below:

$$K_I = K_{Im} + K_{Ib} + K_{Ir} \quad (6)$$

where:

$$K_{Im} = (SF_m)F_m\sigma_m(\pi a)^{0.5}$$

$F_m$  = through-wall shape factor for a circumferential flaw under membrane stress (given in Appendix I of N-513-3)

$\sigma_m$  = membrane stress, ksi

$$K_{Ib} = [(SF_b)\sigma_b + \sigma_e]F_b(\pi a)^{0.5}$$

$SF_b$  = structural factor for bending stress (see Table 2)

$\sigma_b$  = bending stress, ksi

$\sigma_e$  = thermal stress, ksi

$F_b$  = through-wall shape factor for a circumferential flaw under bending stress (given in Appendix I of N-513-3).

Note that the through-wall flaw shape factors are a function of flaw length.

Table 3 shows the specific load combinations considered herein for the allowable circumferential flaw calculations. Since the load combination for Service Level C and D are equivalent, the more limiting flaw length associated with the Service Level C structural factors are presented.

### 4.3 Critical Fracture Toughness Determination

For LEFM analysis, the static fracture toughness for crack initiation under plane strain conditions,  $K_{Ic}$ , is taken from Article C-7000 [4] as prescribed by N-513-3 and is given below:

$$K_{Ic} = \sqrt{\frac{J_{Ic} E'}{1000}} \quad (7)$$

where:

$J_{Ic}$  = material toughness, in-lb/in<sup>2</sup>

$E' = E/(1-\nu^2)$

$E$  = Young's modulus, ksi

$\nu$  = Poisson's ratio.

Based on the design input listed previously,  $K_{Ic}$  is 94.7 ksi-in<sup>0.5</sup> for both axial and circumferential flaws. The allowable flaw lengths are determined iteratively by increasing flaw length until the stress intensity factor is equal to the static fracture toughness.

## 5.0 RESULTS

Table 4 shows the allowable through-wall flaw lengths resulting from the analysis based on a surrounding nominal wall thickness. The most limiting flaw length is 8 inches in the circumferential direction. The UT results for the leaking elbow are shown in Figure 3 [1]. The leak is easily bounded in the axial and circumferential directions by 8 inches. Thus, the acceptance criteria of Code Case N-513-4 are met.

Finally, Paragraph 3.2(d) requires that N-513-3 Equation 9 be satisfied (i.e., the remaining ligament average thickness over the degraded area bounded by the limiting flaw size will resist pressure blowout). The average remaining wall thickness requirement covering the degraded area from Equation 9 is 0.07 inch (using a  $d_{adj} = 8$  inches). From the inspection data given in Figure 3, only the grids nearest to the leak are less than this value. Thus, this Code Case requirement is met.

## 6.0 CONCLUSIONS

The flaw evaluation of the weeping flaw in a 18-inch elbow of the SSW piping at Pilgrim has been evaluated using the methods of a pending revision to Code Case N-513-3 (designated N-513-4) currently in the ASME approval process (N-513-3 does not provide evaluation criteria for flaws in elbows, while N-513-4 does). N-513-4 has not been approved by the ASME or reviewed by the NRC; however, it is recognized in ASME committee that the technical approach is very conservative. Table 4 shows the axial and circumferential allowable flaw lengths based on a surrounding nominal wall thickness of 0.312 inch. The most limiting flaw size is 8 inches in the circumferential direction. The leak is easily bounded

in the axial and circumferential directions by 8 inches (as shown in Figure 3). Thus, the acceptance criteria of Code Case N-513-4 are met.

## 7.0 REFERENCES

1. Pilgrim NDE Inspection Report, File Name "JF29 4 8 0.dmsdr," February 25, 2014, SI File Number 1400287.201.
2. ASME Code Case N-513-3, "Evaluation Criteria for Temporary Acceptance of Flaws in Moderate Energy Class 2 or 3 Piping Section XI, Division 1," Cases of ASME Boiler and Pressure Vessel Code, January 26, 2009.
3. Regulatory Guide 1.147, "Inservice Inspection Code Case Acceptability, ASME Section XI, Division 1," Revision 16, Nuclear Regulatory Commission, October, 2010.
4. ASME Boiler and Pressure Vessel Code, Section XI, Appendix C, 2001 Edition with addenda through 2003.
5. ASME Boiler and Pressure Vessel Code, Section III, Subsection ND, 2004 Edition.
6. ANSI B31.1, Power Piping, 1967 Edition.
7. Entergy Drawing Number M100-7250, Revision E5, "Service Water System E209B SSW Backwash Drain Piping," SI File Number 1400287.201.
8. Pilgrim Nuclear Power Station Specification Number M300, System 29 Service Water, SI File Number 1400287.201.
9. Pilgrim Nuclear Power Station Pipe Stress Calculation 638, SI File Number 1400057.201.
10. Pipe Fracture Encyclopedia, US Nuclear Regulatory Commission, Volume 1, 1997.
11. SI Calculation Number 1400287.301, Revision 0, "Pilgrim Salt Service Water Discharge Piping Elbow (JF29-8-4 Spool) Wall Thinning Stress Analysis."

**Table 1: J<sub>IC</sub> Values for A106 Gr. B Carbon Steel from NRC's Pipe Fracture Database [10]**

<b>A106 Grade B</b>					
Database Reference	Temperature (°C)	Temperature (°F)	J <sub>IC</sub> (kJ/m <sup>2</sup> )	J <sub>IC</sub> (lb <sub>f</sub> -in/in <sup>2</sup> )	K <sub>IC</sub> (ksi-in <sup>3/2</sup> )
2	24	75	97	552	133
2	24	75	336	1919	249
16	25	77	81	464	122
16	25	77	418	2386	277
16	25	77	270	1542	223
16	25	77	193	1104	189
22	24	75	224	1278	203
22	20	68	112	641	144
22	20	68	117	668	147
22	23	73	214	1223	199
22	20	68	167	954	175
22	20	68	223	1271	202
22	20	68	108	617	141
23	52	126	116	663	146
23	23	73	103	590	138
23	23	73	105	600	139
23	23	73	93	528	131
24	23	73	76	431	118
24	23	73	82	469	123
24	57	135	51	293	97
25	23	73	77	439	119
25	23	73	70	400	114
25	57	135	62	356	107
90	20	68	235	1342	208
90	20	68	219	1251	201
90	20	68	255	1456	217
90	20	68	281	1605	228
90	20	68	281	1605	228
90	20	68	335	1913	248
90	20	68	421	2404	279
90	20	68	385	2198	266
90	20	68	175	999	180
90	20	68	172	982	178
90	20	68	178	1016	181
90	20	68	214	1222	199
90	20	68	275	1570	225
90	20	68	133	759	157
90	20	68	140	799	161
90	20	68	174	994	179
90	20	68	111	634	143
90	20	68	190	1085	187
90	20	68	71	405	114
90	20	68	110	628	142
90	20	68	104	594	138
90	20	68	104	594	138
90	20	68	97	554	134
90	20	68	89	508	128
90	20	68	88	502	127
90	20	68	267	1525	222



**Table 2: Axial and Circumferential Structural Factors [4]**

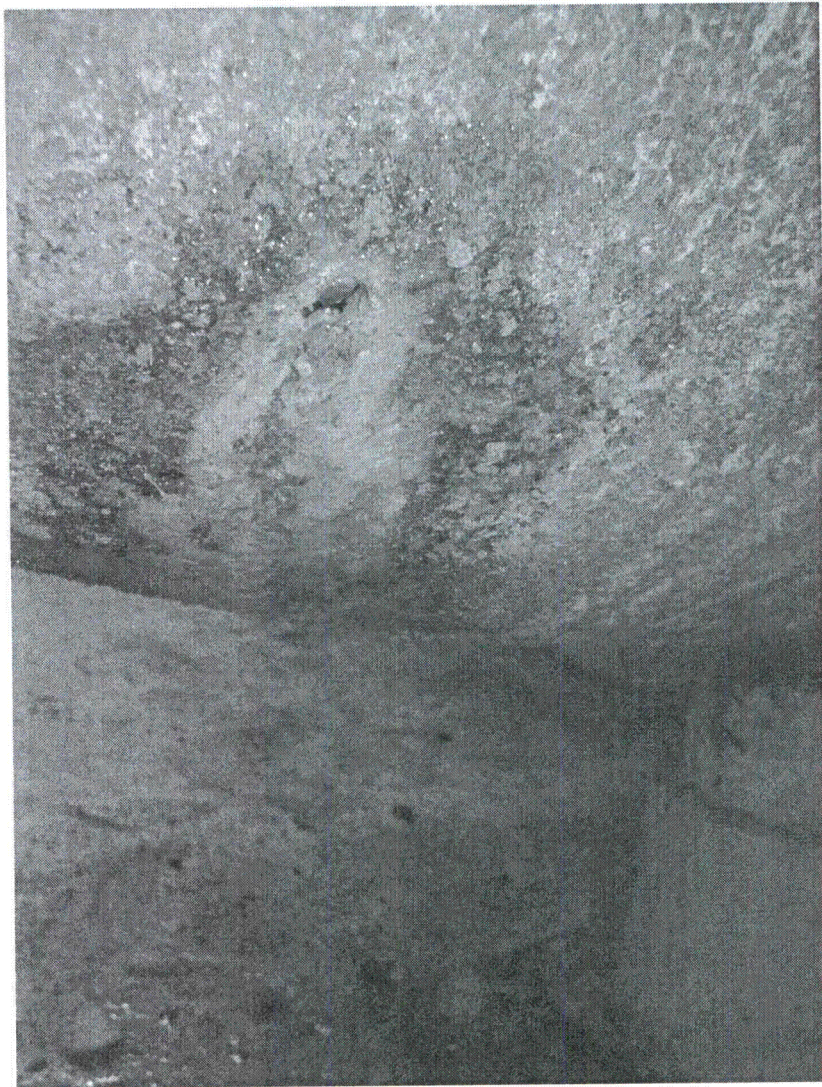
Service Level	Membrane Stress, $SF_m$	Bending Stress, $SF_b$
A	2.7	2.3
B	2.4	2.0
C	1.8	1.6
D	1.3	1.4

**Table 3: Load Combinations for Circumferential Flaw Analyses**

Load Combination	Service Level
P+DW+TH	A
P+DW+OBE+TH	B
P+DW+SSE+TH	C/D

**Table 4: Allowable Through-Wall Flaw Lengths (based on  $t = 0.312''$ )**

Service Level	Allowable Axial Flaw Length (in)	Allowable Circumferential Flaw Length (in)
A	16.0	13.2
B	16.0	8.8
C/D	16.0	8.0



**Figure 1. Pinhole Leak in Service Water Piping, 18-inch Elbow**



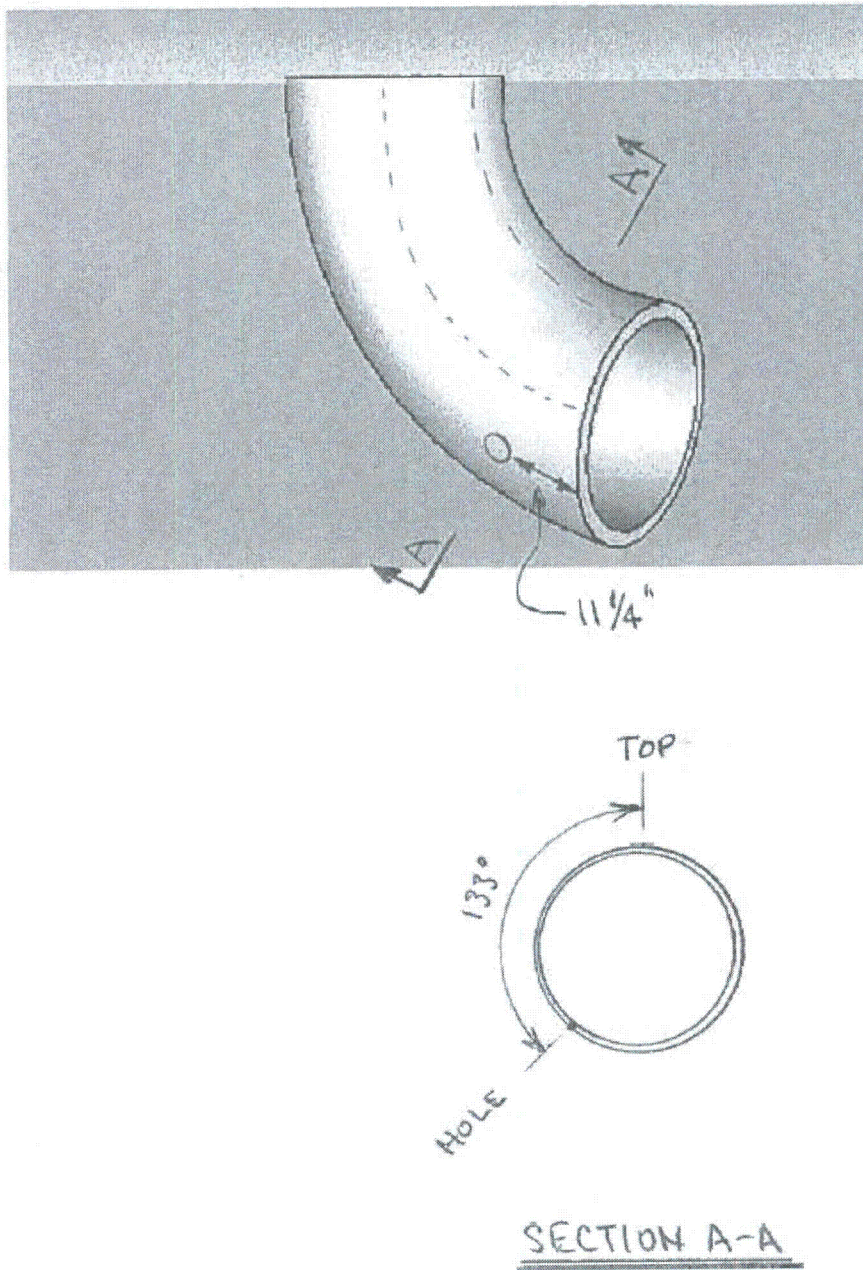


Figure 2. Sketch of Leak Location in Service Water Piping, 18-inch Elbow

LOCATION	AJ:	AK:	AL:	AM:	AN:	AO:	AP:	AQ:	AR:	AS:	AT:	AU:	AV:
1	0.396	0.397	0.397	0.380	0.369	0.352	0.348	0.344	0.342	0.335	0.328	0.315	0.328
2	0.398	0.396	0.393	0.371	0.334	0.353	0.347	0.343	0.345	0.341	0.337	0.311	0.325
3	0.393	0.390	0.393	0.369	0.309	0.361	0.347	0.338	0.343	0.335	0.327	0.308	0.326
4	0.387	0.389	0.389	0.383	0.150	0.051	0.109	0.336	0.299	0.336	0.327	0.308	0.318
5	0.386	0.390	0.394	0.192	0.087	0.064	0.083	0.334	0.313	0.328	0.328	0.315	0.320
6	0.389	0.388	0.392	0.108	0.352	0.362	0.332	0.333	0.334	0.298	0.292	0.317	0.323
7	0.390	0.390	0.393	0.390	0.380	0.343	0.342	0.339	0.341	0.296	0.337	0.321	0.323
8	0.385	0.388	0.392	0.388	0.386	0.366	0.360	0.356	0.349	0.349	0.348	0.327	0.326

**Figure 3. UT Data (3/4 Inch Grid) for Service Water Piping, 18-inch Elbow**

## **Appendix A**

### **DRAFT CODE CASE N-513-4 PROCEDURES FOR ELBOW FLAW EVALUATION**

### 3.3 Through-wall Flaws in Elbows and Bent Pipe

Through-wall flaws in elbows and bent pipe may be evaluated using the straight pipe procedures given in 3.1 or 3.2(d) provided the stresses used in the evaluation are adjusted as described below to account for the geometry differences. Alternate methods may be used to calculate the stresses used in evaluation.

The hoop stress,  $\sigma_h$ , for elbow and bent pipe evaluation shall be:

$$\sigma_h = \left( \frac{pD_o}{2t} \right) \left[ \frac{2R_{bend} + R_o \sin \phi}{2(R_{bend} + R_o \sin \phi)} \right] + \left( \frac{1.95}{h^{2/3}} \right) \frac{R_o M_b}{I} \quad (9)$$

where

- $R_{bend}$  = elbow or bent pipe bend radius
- $\phi$  = circumferential angle defined in Figure 7
- $h$  = flexibility characteristic
- $M_b$  = resultant primary bending moment
- $I$  = moment of inertia based on evaluation wall thickness,  $t$

Equation 9 is only applicable for elbows and bent pipe where  $h \geq 0.1$ .

The axial membrane pressure stress,  $\sigma_m$ , for elbow and bent pipe evaluation shall be:

$$\sigma_m = B_1 \left( \frac{pD_o}{2t} \right) \quad (10)$$

where  $B_1$  is a primary stress index as defined in ASME Section III for the piping item.  $B_1$  shall be equal to 0.5 for elbows and bent pipe.

The axial bending stress,  $\sigma_b$ , for elbow and bent pipe evaluation shall be:

$$\sigma_b = B_2 \left( \frac{R_o M_b}{I} \right) \quad (11)$$

where  $B_2$  is a primary stress index as defined in ASME Section III for the piping item.

The thermal expansion stress,  $\sigma_e$ , for elbow and bent pipe evaluation shall be:

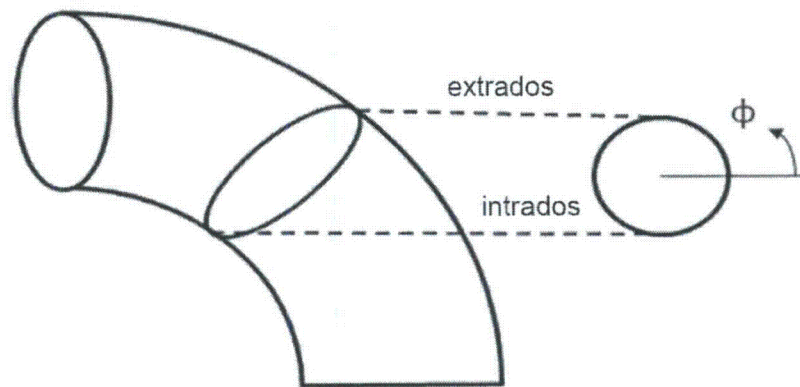
$$\sigma_e = i \left( \frac{R_o M_e}{I} \right) \quad (12)$$

where

- $i$  = stress intensification factor as defined in the Code of Record for the piping item
- $M_e$  = resultant thermal expansion moment

Figure 7 from N-513-4:

**FIG. 7 CIRCUMFERENTIAL ANGLE DEFINED**



**ATTACHMENT 5**

**TO ENTERGY LETTER 2.14.023**

**PILGRIM RELIEF REQUEST PRR-25**

**SSW Spool JF 29-8-4 NDE Data Sheet**

**(4 pages)**





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# UT Erosion/Corrosion Examination

Site/Unit PNPS / 1  
Summary No. SSW Pipe Spool  
Workscope: BOP

Procedure: CEP-NDE-0505  
Procedure Rev.: 004  
Work Order No.: 375247-04

Outage No.: N/A  
Report No.: BOP-UT-14-001  
Page: 1 of 24

Code ASME Sec-XI, 2001-2003 Ada. Cat./Item: C-H/C7.10 Location: "B" Aux Bay  
Drawing No.: M100-7250 Description: 18" Elbow  
System ID: Service Water System (29)  
Component ID: Pipe Spool JF29-8-4 Size/Length: 18" / 6"-12" Thickness/Diameter Sch.-20/18"  
Limitations: N/A Component File No.: N/A Start Time 9:45 Finish Time 14:10

Calibration Information			
Calibration Thickness (In)		Calibration Times / Initials	
Actual	Measured		
0.100	0.100	Start: 9:40	RDA
0.200	0.199	Verify: 12:00	RDA
0.300	0.299	Verify: N/A	N/A
0.400	0.400	Verify: N/A	N/A
0.500	0.500	Final: 14:15	RDA

Partitioning Information		
Component	Begin/Col/Row	Ending/Col/Row
M. UPST Ext.	N/A	N/A
Main UPST.	N/A	N/A
Main	1 A	8 BI
Main DNST.	N/A	N/A
M. DNST Ext.	N/A	N/A
Branch	N/A	N/A
Branch Ext.	N/A	N/A

Component Information	
Component Geometry:	Pipe Elbow
Outside Diameter:	18"
Grid Size:	3/4"
Max. Thickness:	0.457
Min. Thickness:	0.051
Nominal Thickness:	0.312
Tmin.:	0.270
Min. Thickness Location:	4 AO
Max. Thickness Location:	8 Z
Surface Condition:	SMOOTH

Instrument:  
Manufacturer: GE  
Model: USM-GO  
Serial No.: USMGO12915119  
Gain: 66  
Range: 0.500

Transducer:  
Manufacturer: KBA  
Serial No.: 01550W  
Size: 0.375" Freq.: 5.0 MHz  
Model: 113-550-001  
# of Elements: 2

Reference/Simulator Block:  
Serial No.: 94-5570  
Type: 0.1"-0.5"  
Ref./Simulator Block Temp.: 70 °F  
Material/Component Temp.: 73 °F

Temp. Tool:  
Manufacturer: Elcometer  
Serial No.: PNPTEM-288  
Couplant:  
Type: Ultragel II  
Batch No.: 05125

Comments/Obstructions: UT performed do to a through wall hole. See CR-PNP-2014-00815. This is not a Code required exam.

Results: Accept ☐ Reject ☐ Eval ☒

Examiner	Level	III	Signature	Date	Reviewer	Signature	Date
Avery, Richard D. (Rick)			<i>R.D. Avery</i>	2/25/2014	<i>D.B. King</i>	<i>UT Level III</i>	2/27/14
Examiner	Level	N/A	Signature	Date	Site Review	Signature	Date
N/A					N/A		
Other	Level	N/A	Signature	Date	ANII Review	Signature	Date
N/A					N/A		



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## Supplemental Report

Report No.: BOP-UT-14-001

Page: 2 of 4

Summary No.: SSW Pipe Spool

File Name: Spool JF29-8-4  
Description: 18" Elbow  
Creation Date: 2/25/2014  
Probe: See UT Report  
Cal Comment: See report  
Inspector: R. AVERY Company: Entergy  
Instrument Type: DMS Go Instrument S.N.: USMGO12015119

Units: INCH Velocity(in/us): 0.2360

Number of Readings: 488 Number of Empties es: 0  
Number of Obstructs: 0 Number of Attachments: 0  
Range: 0.406 Points Below MinAlarm: 0  
Mean: 0.368 Standard Deviation: 0.047  
Minimum Value: 0.051  
Minimum Value Loc.: 4:AQ:1  
Maximum Value: 0.457  
Maximum Value Loc.: 8:Z:1

100% CRIP SCANS, LOWEST READING RECORDED

LOCATION	A:	B:	C:	D:	E:	F:	G:	H:	I:	J:	K:	L:	M:	N:
1	0.344	0.348	0.365	0.372	0.374	0.375	0.377	0.373	0.372	0.364	0.361	0.364	0.358	0.359
2	0.346	0.356	0.365	0.370	0.372	0.376	0.373	0.377	0.375	0.370	0.367	0.367	0.364	0.365
3	0.348	0.358	0.363	0.366	0.373	0.376	0.375	0.377	0.376	0.374	0.369	0.367	0.366	0.366
4	0.354	0.352	0.365	0.369	0.375	0.379	0.381	0.380	0.380	0.380	0.372	0.367	0.366	0.368
5	0.349	0.354	0.366	0.368	0.375	0.378	0.378	0.382	0.382	0.380	0.377	0.372	0.369	0.372
6	0.353	0.357	0.364	0.367	0.378	0.378	0.381	0.382	0.366	0.381	0.375	0.372	0.373	0.372
7	0.354	0.357	0.365	0.374	0.375	0.380	0.381	0.385	0.381	0.379	0.379	0.371	0.372	0.371
8	0.360	0.359	0.363	0.374	0.373	0.363	0.369	0.382	0.380	0.378	0.377	0.377	0.374	0.372

TOP C/L of PIPE

Direction of Flow



*PAK-8* 2/27/14





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## Supplemental Report

Report No.: **BOP-UT-14-001**

Page: **3** of **4**

Summary No: **SSW Pipe Spool**

	O:	P:	Q:	R:	S:	T:	U:	V:	W:	X:	Y:	Z:	AA:	AB:	AC:	AD:	AE:
1	0.367	0.371	0.380	0.386	0.398	0.400	0.373	0.413	0.419	0.425	0.427	0.421	0.417	0.415	0.417	0.413	0.404
2	0.372	0.378	0.385	0.392	0.398	0.401	0.409	0.411	0.419	0.431	0.428	0.428	0.421	0.422	0.417	0.414	0.405
3	0.373	0.380	0.383	0.394	0.400	0.405	0.410	0.415	0.420	0.429	0.437	0.428	0.422	0.417	0.418	0.410	0.404
4	0.374	0.382	0.383	0.396	0.400	0.405	0.409	0.413	0.423	0.427	0.429	0.423	0.421	0.418	0.419	0.415	0.403
5	0.378	0.384	0.385	0.400	0.403	0.407	0.412	0.416	0.426	0.428	0.429	0.431	0.427	0.423	0.423	0.421	0.409
6	0.380	0.389	0.393	0.404	0.407	0.414	0.421	0.419	0.429	0.437	0.433	0.430	0.428	0.425	0.422	0.426	0.408
7	0.380	0.387	0.390	0.400	0.406	0.412	0.416	0.419	0.430	0.433	0.430	0.430	0.427	0.425	0.423	0.426	0.414
8	0.383	0.389	0.395	0.401	0.407	0.412	0.418	0.422	0.426	0.428	0.427	0.457	0.421	0.419	0.422	0.421	0.406

	AF:	AG:	AH:	AI:	AJ:	AK:	AL:	AM:	AN:	AO:	AP:	AQ:	AR:	AS:	AT:	AU:	AV:
1	0.400	0.395	0.391	0.396	0.396	0.397	0.397	0.380	0.369	0.352	0.348	0.344	0.342	0.335	0.328	0.315	0.328
2	0.393	0.394	0.389	0.397	0.398	0.396	0.393	0.371	0.334	0.353	0.347	0.343	0.345	0.341	0.337	0.311	0.325
3	0.394	0.390	0.388	0.389	0.393	0.390	0.393	0.369	0.309	0.361	0.347	0.338	0.343	0.335	0.327	0.308	0.326
4	0.391	0.385	0.383	0.383	0.387	0.389	0.389	0.383	0.150	0.051	0.109	0.336	0.299	0.336	0.327	0.308	0.318
5	0.378	0.388	0.380	0.380	0.386	0.390	0.394	0.182	0.087	0.064	0.083	0.334	0.313	0.328	0.328	0.315	0.320
6	0.390	0.380	0.382	0.383	0.389	0.388	0.392	0.188	0.352	0.362	0.332	0.333	0.334	0.298	0.292	0.317	0.323
7	0.398	0.385	0.385	0.380	0.390	0.390	0.393	0.390	0.380	0.343	0.342	0.339	0.341	0.296	0.337	0.321	0.323
8	0.401	0.392	0.388	0.381	0.385	0.388	0.392	0.388	0.386	0.366	0.360	0.356	0.349	0.349	0.348	0.327	0.326

Areas that are below 0.300. Area of hole

	AW:	AX:	AY:	AZ:	BA:	BB:	BC:	BD:	BE:	BF:	BG:	BH:	BI:
1	0.336	0.345	0.346	0.340	0.339	0.339	0.335	0.338	0.333	0.332	0.338	0.341	0.338
2	0.337	0.341	0.344	0.345	0.341	0.337	0.332	0.328	0.331	0.330	0.336	0.339	0.343
3	0.332	0.336	0.343	0.346	0.339	0.337	0.327	0.335	0.334	0.336	0.336	0.339	0.344
4	0.326	0.338	0.344	0.343	0.339	0.331	0.336	0.336	0.334	0.336	0.336	0.340	0.349
5	0.331	0.337	0.343	0.345	0.341	0.339	0.333	0.339	0.335	0.335	0.330	0.346	0.343
6	0.336	0.341	0.347	0.348	0.349	0.346	0.339	0.335	0.338	0.343	0.345	0.339	0.345
7	0.337	0.347	0.350	0.380	0.348	0.347	0.339	0.335	0.344	0.340	0.339	0.341	0.377
8	0.332	0.343	0.351	0.356	0.351	0.348	0.338	0.343	0.343	0.341	0.339	0.343	0.351

*[Signature]* 2/27/14





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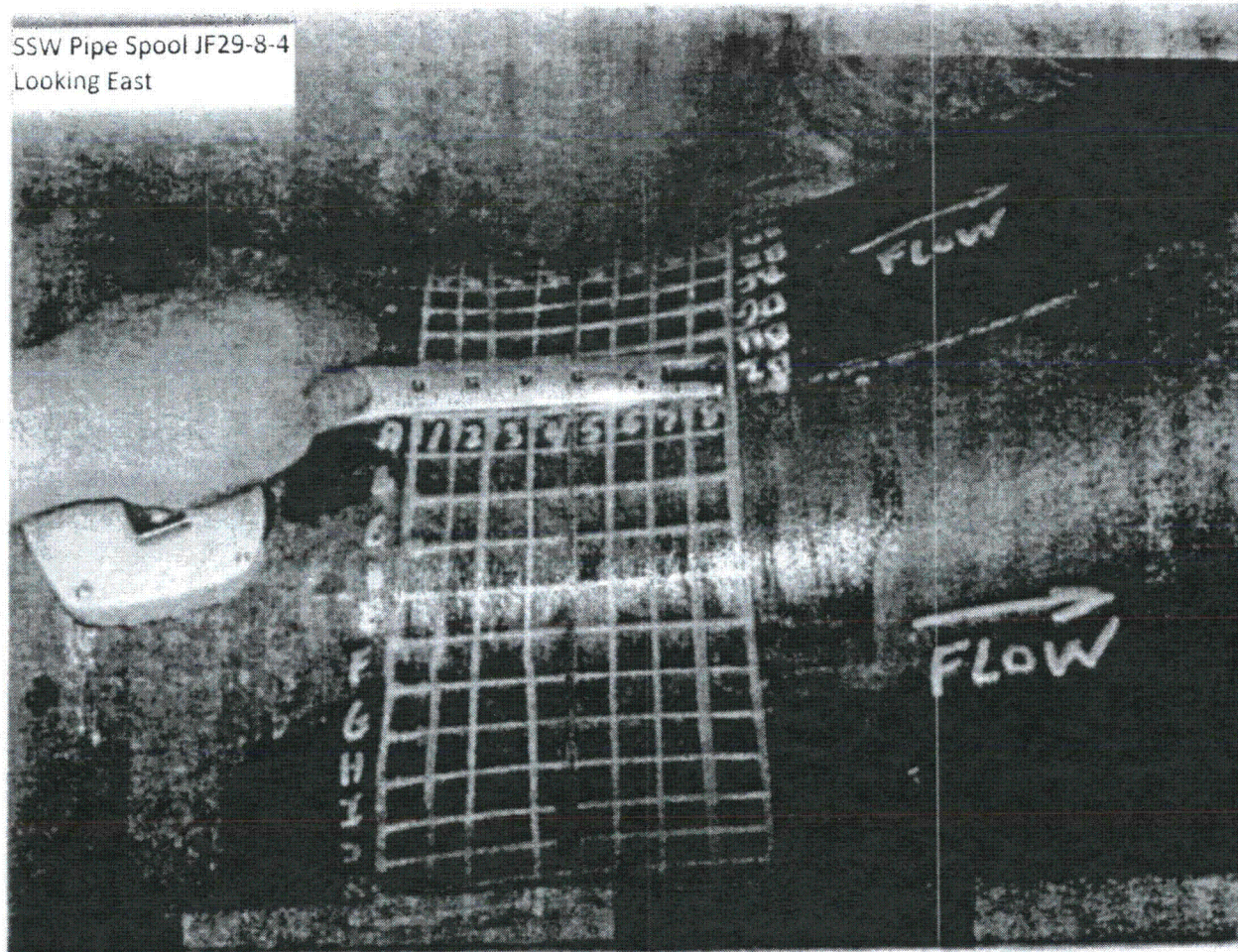
## Supplemental Report

Report No. BOP-UT-14-001

Page 4 of 4

Summary No. SSW Pipe Spool

SSW Pipe Spool JF29-8-4  
Looking East



*AK* 2/27/14